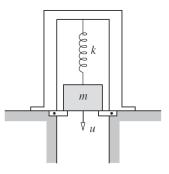
## Homework #2

Due: Friday, September 12

1) A mass m is initially at rest, partially supported by a spring and partially by stops (see Figure 1). In the position shown, the spring force is mg/4. At time t = 0 the stops are rotated, suddenly releasing the mass. Determine the motion of the mass.

Main concept: definition of the coordinate system and relation with the form of the equation of motion.



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Figure 1

2) The weight of the wooden block shown in Fig. 2 is 20 lb and the spring stiffness is 120 lb/in. The block is initially at rest. A bullet weighing 0.4 lb is fired at a speed of 50 ft/sec into the block and becomes embedded in the block. Determine the resulting motion u(t) of the block.

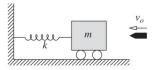


Figure 2

\*Hint: Use conservation of momentum.

- 3) The vertical suspension system of an automobile is idealized as a viscously damped SDF system. Under the 3500-lb weight of the car the suspension system deflects 2.1 inches. The suspension is designed to have a damping ratio of  $\zeta = 0.7$  with no one in the car.
- (a) Calculate the damping and stiffness coefficients of the suspension.
- (b) With four 150-lb passengers in the car, what is the effective damping ratio?
- (c) Calculate the natural vibration frequency for case (b).

Main concept: static equilibrium position and the vibration properties of a system with damping.

4) Find a linear elastic oscillator, measure its damped natural frequency and determine its percentage of critical damping. Submit a sketch of the system, your measurements, and calculations.

Note: It is up to you how you measure the response. One option is to use your cell phone as an accelerometer.

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S<u>oluk</u>an.
Problem #1.
                                                            @ time = t
                                        state equilibrium
                     6 f=0
 Position of no torce
                                           position
 in the sping.
                                                  T rust
                                                              u = stretch in
                                               displacement
A free body stagram gives the £0.4 @ time t.
                      mü + ku = mg
                                              (see picture above)
                        ud = u -
     ne intradice
4
                       üd = ü
      ud the top is:
                       mid + kud = 0
          solution to this
                                 FOH
                                        is :
and
```

 $ud(t) = A \cdot cos(unt) + B \cdot so(unt)$ 

where A and B can be found based on the initial conditions.

$$vd(0) = -\frac{3m9}{4k}$$
  $vd(0) = 0$  (starts from rest).

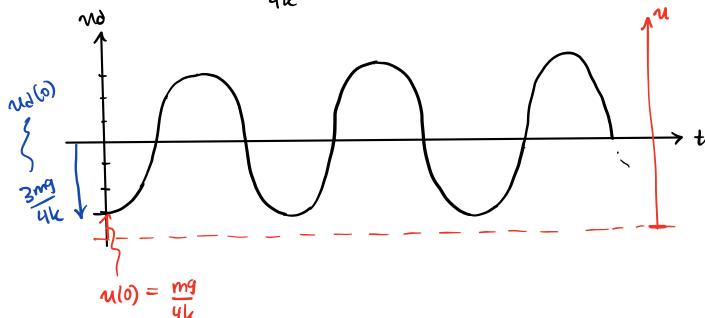
$$n_{0}(0) = 0 \Rightarrow B \cdot \omega_{0} \cdot (1) = 0 \Rightarrow B = 0$$

and 
$$u_{\delta}(0) = u_{\delta} \Rightarrow A \cdot (1) = u_{\delta} \Rightarrow A = u_{\delta}$$

thus, the motion:

$$nud(t) = -\frac{3mg}{4k} \cdot cos(\omega_n t)$$

and 
$$u(t) = -\frac{3mg}{4k}$$
. (because  $u(t) = ust + ud(t)$ )



where 
$$u(0) = 0 \Rightarrow A = 0$$
  
 $\dot{u}(0) = \dot{u}_0$ 

$$\dot{u}(t) = B \cdot \omega_n \cdot \cos(\omega_n t)$$

$$\dot{u}(0) = \dot{u}_0 \Rightarrow B \cdot \omega_n = \dot{u}_0 \Rightarrow \omega_n$$

and 
$$u(t) = \frac{\dot{u}_0}{\dot{u}_0}$$
 sin (unt)

So, need to get no - conservation of linear momentum!

(mb is the mass of the blief).

$$m.(0) + mb.v_0 = (m+mb) \dot{u}_0$$

before impact. Billet now it part of the block

$$\Rightarrow$$
  $\dot{v}_0 = -\frac{mb}{m+mb} \cdot v_0$ 

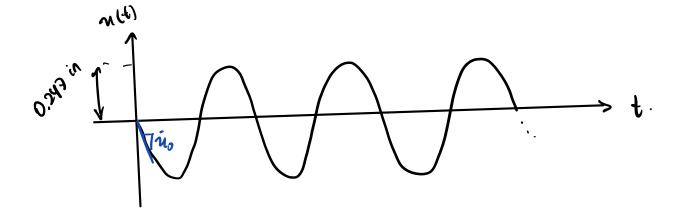
Plugging the values: 
$$\dot{u}_0 = \frac{0.4}{20.4 \text{ lb}}$$
. 50 ft/sec = -0.98 (ft/sec)  
Now,  $w_n = \sqrt{k/M} \rightarrow \text{here } M = 20.4 \text{ lb}$ !

$$\omega_n = \sqrt{\frac{|20 \text{ lb/in}}{204 \text{ lb}}} \cdot \frac{386 \text{ in}}{5^2} = 47.65 \text{ (rad/sec)}.$$

So: 
$$\frac{\dot{u}_0}{u_n} = \frac{11.76}{47.65} \left(\frac{in/5}{115}\right) = -0.247 \text{ (in)}$$

and the motion is:

$$M(t) = 0.247 \cdot Sin(47.65t)$$
 (in).



Problem # 3.

$$fst = 2.1 (in)$$

$$dst = \frac{mg}{k} \Rightarrow$$

$$k = \frac{mg}{6st} = \frac{3,800 (lb)}{2.1 (in)} = (667 (lb(in)).$$

$$\omega_n = \sqrt{m} = \sqrt{3,500 \text{ (lb)}}$$
 $\omega_n = \sqrt{m} = \sqrt{3,500 \text{ (lb)}}$ 
 $\omega_n = \sqrt{m} = \sqrt{m} = \sqrt{3,500 \text{ (lb)}}$ 
 $\omega_n = \sqrt{m} = \sqrt{m} = \sqrt{3,500 \text{ (lb)}}$ 
 $\omega_n = \sqrt{m} =$ 

$$C = 2.3500 \text{ (lb)}. |3.56 \left(\frac{50}{6}\right).0.7$$

however, if we want  $[C.\tilde{v}] = [Force]$  we can get  $C$  in  $C$  in.

however, if we want 
$$[c. \pi] = [+oices]$$
  
 $c = 64,444 (\frac{eb}{s^2}) \cdot \frac{1}{386} \cdot (\frac{s^2}{in}) = 172.13 (\frac{eb \cdot s}{in}).$ 

$$\begin{cases} \ell b \cdot \frac{s}{i} \end{cases}$$

$$w_{n}^{NEN} = \sqrt{\frac{k}{M}} = \sqrt{\frac{|b67|(|b|in)}{150.4 + 3500 (lb)}} = 12.53 (rad/s)$$

$$6 \left( in/\delta_3 \right) = 15.23 \left( cag/e \right)$$

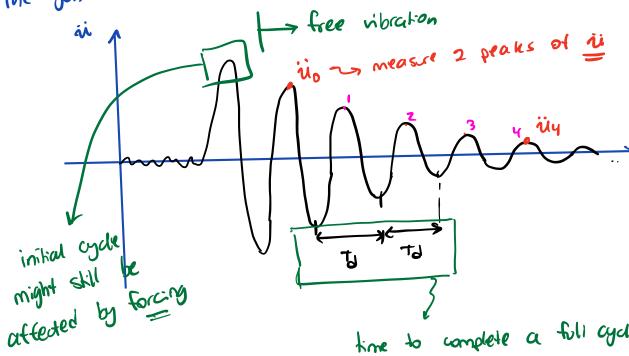
now 
$$C = 2m\omega_n \cdot 3 \Rightarrow 5 = \frac{C}{2m\omega_n}$$

$$\Rightarrow \int_{N^{EW}} = \frac{c}{2 \, \text{M} \cdot \omega_n^{NEW}} = \frac{172.13 \, (\text{lb} \cdot \text{S/in})}{2. \, (3500 + 150.4 \, \text{lb}) \cdot 12.53 \, (\text{1/s})}.386 \, (\text{in/s}^2)$$

(c) 
$$w_n^{\text{NEW}} = 12.53 \, (rad/sec)$$
.

Problem # 4.

the general idea was to measure ii(t) for free vibration -- free vibration



time to complete a full cycle = Td  $ud = \frac{2\pi}{Td}$ then

From the two peaks of ii, get 3 using log-decrement:

 $\ln\left(\frac{u_{i}}{u_{i+j}}\right) = 2\pi \xi j \Rightarrow \text{ for my example plot}:$ 

$$\ln\left(\frac{no}{ny}\right) = 9\pi3 \Rightarrow 3 = \frac{\ln\left(\frac{no}{ny}\right)}{8\pi}.$$