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Computer Assignment 4.1.1

MMP Finite Element Methods (WI4243AP-FE)

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### 1 Introduction

The following is a report containing the evaluation of the heat equation with the finite element method. To solve the two-dimensional heat equation numerically, the finite element method is applied. To that end the minimization problem corresponding to the partial differential in question is derived. The solution of the minimization problem is approximated with the finite element method by deriving the element stiffness matrix and element stiffness vector using triangular and line elements.

## 2 The Heat Equation

schem.png

Figure 1: Rectangular plate made of two materials, A and B.

Consider the rectangular plate shown in figure 1, with width, b, and height, h. In the center of the plate is a rectangular part, of a different material, with

width,  $b_0$  and height,  $h_0$ . A denotes the outer part of the plate while B denotes the inner part.

The situation is stationary and the effects of convection and sources are absent. Thus the heat equation is simply defined by the conduction (diffusion) term as shown in (1)

$$-\nabla \cdot (k\nabla T) = 0 \tag{1}$$

with heat conductivity k and temperature T. The thermal conductivity is different in each material, i.e.  $k|_A = k_A$  and  $k|_B = k_B$ .

The boundary of A,  $\partial A$ , is divided in four segments such that  $\partial A = \Gamma_1 \cup \Gamma_2 \cup \Gamma_3 \cup \Gamma_4$ . On these boundaries the following conditions hold:

$$T|_{\Gamma_1 \cup \Gamma_3} = T_0 \tag{2}$$

$$\nabla T|_{\Gamma_2} = 0 \tag{3}$$

$$k\frac{\partial T}{\partial n}|_{\Gamma_4} = -\alpha(T_w - T_\infty) \tag{4}$$

In words, the temperature at boundaries  $\Gamma_1$  and  $\Gamma_3$  are held constant at  $T=T_0$ . The flux at  $\Gamma_2$  is zero.  $\Gamma_4$  obeys Newton's heat transfer relation with the environment temperature  $T_{\infty}$  and the temperature at the wall  $T_w$ .

In this problem a symmetry plane is present. This plane runs along the midsection of the plate in the vertical direction. Here, to ensure symmetry, the gradient of the temperature must be zero.

## 3 Minimization problem

A partial differential equation can be related to an equivalent minimization problem with the following theorem[?]:

Let L be a linear, symmetric, positive differential operator defined of space  $\Sigma$  and let

$$Lu = f (5)$$

Then the solution u minimizes the functional

$$I(u) = \int_{\Omega} \left\{ \frac{1}{2}uLu - uf \right\} d\Omega \tag{6}$$

over the space  $\Sigma$ . On the other hand, if u minimizes (6) then u satisfies (5)

Another requirement u should obey is homogeneous boundary conditions, a requirement T fails. Even so, this theorem can still be utilized by making the boundary conditions homogeneous.

Let Lu = f with non-homogeneous boundary equations and suppose that there is another smooth function w that also satisfies the same non-homogeneous boundary conditions. Now let

$$v = T - w \tag{7}$$

T and w satisfy the same non-homogeneous boundary conditions, therefore v satisfies homogeneous boundary conditions. Now the theorem can be applied to v as it meets all the requirements.

$$Lv = LT - Lw = f - Lw \tag{8}$$

so the corresponding minimization problem is given by

$$\min_{v \in \Sigma} I(v) = \frac{1}{2} \int_{\Omega} v L v d\Omega - \int_{\Omega} v (f - L w) d\Omega \tag{9}$$

Substituting (7) into (9) yields

$$\min_{v \in \Sigma} I(v) = \frac{1}{2} \int_{\Omega} (T - w)L(T - w)d\Omega - \int_{\Omega} (T - w)(f - Lw)d\Omega$$
 (10)

f = 0 and  $L = -\nabla \cdot (k\nabla)$  according to (1), so (10) is rewritten as

$$\begin{aligned} \min_{v \in \Sigma} & I(v) = -\frac{1}{2} \int_{\Omega} (T - w) \nabla \cdot (k \nabla (T + w)) d\Omega \\ &= \frac{1}{2} \int_{\Omega} \nabla (T - w) (k \nabla (T + w)) - \nabla \cdot \{ (T - w) (k \nabla (T + w)) \} d\Omega \\ &= \frac{1}{2} \int_{\Omega} \nabla (T - w) (k \nabla (T + w)) d\Omega - \frac{1}{2} \int_{\Gamma} (T - w) (k \nabla (T + w)) \cdot \hat{\mathbf{n}} d\Gamma \\ &= \frac{1}{2} \int_{\Omega} \nabla T \cdot k \nabla T d\Omega - \frac{1}{2} \int_{\Omega} \nabla w \cdot k \nabla w d\Omega \\ &- \frac{1}{2} \int_{\Gamma} (T - w) (k \nabla (T + w)) \cdot \hat{\mathbf{n}} d\Gamma \end{aligned}$$

$$(11)$$

where the product rule and subsequent Gauss theorem is applied to expand the integral in two volume integrals and one boundary integral. The second integral is independent on T and thus has no influence on the minimization and can be neglected.

The boundary integral is evaluated over four boundaries by taking in account the boundary conditions (2)-(4).

$$\frac{1}{2} \int_{\Gamma_1 \cup \Gamma_3} (T_0 - T_0)(k\nabla(T + w)) \cdot \hat{\mathbf{n}} d\Gamma = 0$$
(12)

$$\frac{1}{2} \int_{\Gamma_2} (T - w)(k(0+0)) \cdot \hat{\mathbf{n}} d\Gamma = 0$$
(13)

$$\frac{1}{2} \int_{\Gamma_4} (T - w)(k\nabla(T + w)) \cdot \hat{\mathbf{n}} d\Gamma = \frac{1}{2} \int_{\Gamma_4} Tk \frac{\partial T}{\partial n} d\Gamma - \frac{1}{2} \int_{\Gamma_4} wk \frac{\partial w}{\partial n} d\Gamma$$

$$= -\frac{1}{2} \int_{\Gamma_4} \alpha T(T_w - T_\infty) d\Gamma \tag{14}$$

where  $\frac{1}{2} \int_{\Gamma_4} w k \frac{\partial w}{\partial n} d\Gamma$  in (14) is neglected as it too is independent of T.

Finally, the general minimization problem is given by

$$\min_{T \in \Sigma} I(v) = \frac{1}{2} \int_{\Omega} k |\nabla T|^2 d\Omega + \frac{1}{2} \int_{\Gamma_4} \alpha T(T_w - T_\infty) d\Gamma$$
 (15)

with  $\Sigma : \{T \mid T|_{\Gamma_1 \cup \Gamma_3} = T_0 \mid \nabla T|_{\Gamma_2} = 0\}$ . The boundary conditions (2) and (3) are essential boundary conditions.

# 4 Element stiffness matrix and element stiffness vector

This problem can be solved by computing the element stiffness matrix and the element stiffness vector. To do this, the solution is approximated with piecewise linear functions  $\phi_i(x, y)$  such that

$$T(x,y) \simeq T^n(x,y) = \sum_{j=1}^n c_j \phi_j(x,y)$$
(16)

One might derive the matrix and vector by substituting this approximation into the minimization problem  $\min_{T \in \Sigma} I(v)$  and then set the derivative of  $c_j$  to zero. This is called the Ritz method.

The second method is by deriving the Galerkin equations. To that end the partial differential equation is multiplied with a test function  $\phi$  to derive these equations. Afterwards the approximation of the solution is substituted into the weak form which then results in the element stiffness matrix and element stiffness vector.

Both methods yield the same results. The following shows how the matrix and vector are acquired using Galerkin's method. First, the Galerkin equations:

$$-\int_{\Omega} \phi \nabla \cdot (k \nabla T) d\Omega = 0$$

$$-\int_{\Omega} \nabla \cdot (\phi k \nabla T) - \nabla \phi k \nabla T d\Omega = 0$$

$$-\int_{\Gamma} \phi k \frac{\partial T}{\partial n} d\Gamma + \int_{\Omega} \nabla \phi k \nabla T d\Omega = 0$$

$$\int_{\Gamma} \phi \alpha \left( T_w - T_{\infty} \right) d\Gamma + \int_{\Omega} \nabla \phi k \nabla T d\Omega = 0$$
(17)

where the boundary integral in the penultimate line is split up over the four boundaries like in (12) - (14) yielding (17) as a result.

The weak form is thus formulated as follows:

$$\int_{\Gamma_4} \phi \alpha \left( T_w - T_\infty \right) d\Gamma + \int_{\Omega} \nabla \phi k \nabla T d\Omega = 0$$

Find  $T \in \Sigma \{ T \text{ sufficiently smooth } | T_{\Gamma_1 \cup \Gamma_3} = T_0 \}, \forall \phi \in \Sigma \}$ 

Now the approximation (16) is applied.  $\phi$  is approximated in the same way i.e.

$$\phi \simeq \sum_{i=1}^{n} b_i \phi_i = \phi_i, i = 1...n$$

As  $\phi$  is chosen arbitrarily, it is useful to make one of the of the coefficients  $b_i$  equal to one and the others to zero.

Substituting these approximations into the weak form results in

$$\int_{\Gamma_4} \alpha \phi_i \left( \sum_{j=1}^n c_j \phi_j - T_\infty \right) d\Gamma + \int_{\Omega} \nabla \phi_i k \nabla \sum_{j=1}^n c_j \phi_j d\Omega = 0$$

$$\sum_{j=1}^n c_j \cdot \alpha \int_{\Gamma_4} \phi_i \phi_j d\Gamma + \sum_{j=1}^n c_j \cdot \int_{\Omega} \nabla \phi_i k \nabla \phi_j d\Omega = \sum_{j=1}^n c_j \cdot \alpha \int_{\Gamma_4} \phi_i T_\infty d\Gamma$$

$$(i = 1...n)$$

(18) can be written more compactly as

$$\sum_{j=1}^{n} c_j S_{ij} = f_i$$

with the element stiffness vector

$$f_i = \alpha \int_{\Gamma_4} \phi_i T_{\infty} d\Gamma$$

and the element stiffness matrix that consists of a boundary matrix

$$S_{ij}^B = \alpha \int_{\Gamma_A} \phi_i \phi_j d\Gamma$$

and a volume matrix

$$S_{ij}^{V} = \int_{\Omega} k \nabla \phi_i \nabla \phi_j d\Omega$$

The matrices and vector need to be evaluated and are done so using the theorems of Newton-Cotes and Holland & Bell.

The set of functions  $\phi_i$  are piecewise linear functions and therefore can be

$$\phi_i = \alpha_i + \beta_i x + \gamma_i y$$

According to Holland & Bell, the  $S_{ij}^B$  is evaluated as follows

$$S_{ij}^{B} = \alpha \int_{\Gamma_{4}} \phi_{i} \phi_{j} d\Gamma = \alpha \|\mathbf{x}_{l1} - \mathbf{x}_{l2}\| \cdot \begin{cases} \frac{2!0!}{(1+2+0)!} = \frac{1}{3} & i = j\\ \frac{1!1!}{(1+1+1)!} = \frac{1}{6} & i \neq j \end{cases}$$
(19)

where  $\|\mathbf{x}_{l1} - \mathbf{x}_{l2}\|$  is the distance between two points. Holland & Bell are applied to  $S_{ij}^V$  as well

$$S_{ij}^{V} = \int_{\Omega} k \nabla \phi_i \nabla \phi_j d\Omega = k \left( \beta_i \beta_j + \gamma_i \gamma_j \right) \frac{\|\Delta\|}{2}$$
 (20)

where k can be  $k_A$  or  $k_B$  depending on the position on the plate.

### 5 Computation

Before computing the element stiffness matrix and vector, a mesh has to be created. This mesh consists of a grid of triangular elements along the surface of the plates and line elements on the boundaries.

#### References 6