LECTURE NOTES

ON

TH-4 (ENGINEERING MECHANICS)
SCTE&VT, ODISHA, BHUBANESWAR

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1. FUNDAMENTALS OF ENGINEERING MECHANICS

ENGINEERING MECHANICS: The subject of Engineering Mechanics is that branch of Applied Science, which deals with the laws and principles of Mechanics, along with their applications to engineering problems.

The subject of Engineering Mechanics may be divided into the following two main groups:
1. Statics, and 2. Dynamics

STATICS: It is that branch of Engineering Mechanics, which deals with the forces and their effects, while acting upon the bodies at rest.

DYNAMICS: It is that branch of Engineering Mechanics, which deals with the forces and their effects, while acting upon the bodies in motion. The subject of Dynamics may be further sub-divided into the following two branches:
1. Kinetics, and 2. Kinematics

KINETICS: It is the branch of Dynamics, which deals with the bodies in motion due to the application of forces.

KINEMATICS: It is that branch of Dynamics, which deals with the bodies in motion, without any reference to the forces which are responsible for the motion.

RIGID BODY: A rigid body (also known as a rigid object) is a solid body in which deformation is zero or so small it can be neglected. The distance between any two given points on a rigid body remains constant in time regardless of external forces exerted on it. A rigid body is usually considered as a continuous distribution of mass.

FORCE: It is defined as an agent which produces or tends to produce, destroys or tends to destroy motion. *e.g.*, a horse applies force to pull a cart and to set it in motion. Force is also required to work on a bicycle pump. In this case, the force is supplied by the muscular power of our arms and shoulders.

SYSTEM OF FORCES: When two or more forces act on a body, they are called to form a system of forces. Following systems of forces are important from the subject point of view;
1. **Coplanar forces**: The forces, whose lines of action lie on the same plane, are known as coplanar forces.
2. **Collinear forces**: The forces, whose lines of action lie on the same line, are known as collinear forces
3. **Concurrent forces:** The forces, which meet at one point, are known as concurrent forces. The concurrent forces may or may not be collinear.
4. **Coplanar concurrent forces**: The forces, which meet at one point and their lines of action also lie on the same plane, are known as coplanar concurrent forces.
5. **Coplanar non-concurrent forces:** The forces, which do not meet at one point, but their lines of action lie on the same plane, are known as coplanar non-concurrent forces.
6. **Non-coplanar concurrent forces**: The forces, which meet at one point, but their lines of

action do not lie on the same plane, are known as non-coplanar concurrent forces.
7. **Non-coplanar non-concurrent forces**: The forces, which do not meet at one point and their lines of action do not lie on the same plane, are called non-coplanar non-concurrent forces.

CHARACTERISTIC OF A FORCE: In order to determine the effects of a force, acting on a body, we must know the following characteristics of a force:
1. Magnitude of the force (*i.e.*, 100 N, 50 N, 20 kN, 5 kN, etc.)
2. The direction of the line, along which the force acts (*i.e.*, along *OX*, *OY*, at 30° North of East etc.). It is also known as line of action of the force.
3. Nature of the force (*i.e.*, whether the force is push or pull). This is denoted by placing an arrow head on the line of action of the force.
4. The point at which (or through which) the force acts on the body

EFFECTS OF A FORCE: A force may produce the following effects in a body, on which it acts:
1. It may change the motion of a body. i.e. if a body is at rest, the force may set it in motion. And if the body is already in motion, the force may accelerate it.
2. It may retard the motion of a body.
3. It may retard the forces, already acting on a body, thus bringing it to rest or in equilibrium.
4. It may give rise to the internal stresses in the body, on which it acts.

PRINCIPLE OF TRANSMISSIBILITY: It states, "If a force acts at any point on a rigid body, it may also be considered to act at any other point on its line of action, provided this point is rigidly connected with the body."

PRINCIPLE OF SUPERPOSITION: This principle states that the combined effect of force system acting on a particle or a rigid body is the sum of effects of individual forces.

ACTION AND REACTION FORCE: Forces always act in pairs and always act in opposite directions. When you push on an object, the object pushes back with an equal force. Think of a pile of books on a table. The weight of the books exerts a downward force on the table. This is the action force. The table exerts an equal upward force on the books. This is the reaction force.

FREE BODY DIAGRAM: A free body diagram is a graphical illustration used to visualize the applied forces, moments, and resulting reactions on a body in a given condition. They depict a body or connected bodies with all the applied forces and moments, and reactions, which act on the body. The body may consist of multiple internal members (such as a truss), or be a compact body (such as a beam). A series of free bodies and other diagrams may be necessary to solve complex problems.

REOSLUTION OF A FORCE: The process of splitting up the given force into a number of components, without changing its effect on the body is called resolution of a force. A force is, generally, resolved along two mutually perpendicular directions.

COMPOSITION OF FORCES: The process of finding out the resultant force, of a number of given forces, is called composition of forces or compounding of forces.

RESULTANT FORCE: If a number of forces, P, Q, R ... etc. are acting simultaneously on a particle, then it is possible to find out a single force which could replace them i.e., which would produce the same effect as produced by all the given forces. This single force is called resultant force and the given forces R ...etc. are called component forces

METHODS FOR THE RESULTANT FORCE:
Though there are many methods for finding out the resultant force of a number of given forces, yet the following are important from the subject point of view :
1. Analytical method. 2. Method of resolution.

ANALYTICAL METHOD FOR RESULTANT FORCE:
The resultant force, of a given system of forces, may be found out analytically by the following methods :
1. Parallelogram law of forces. 2. Method of resolution.

PARALLELOGRAM LAW OF FORCES:
It states, "If two forces, acting simultaneously on a particle, be represented in magnitude and direction by the two adjacent sides of a parallelogram ; their resultant may be represented in magnitude and direction by the diagonal of the parallelogram, which passes through their point of intersection."
Mathematically, resultant force,

$$R = \sqrt{F_1^2 + F_2^2 + 2F_1F_2 \cos\theta}$$

and
$$\tan\alpha = \frac{F_2 \sin\theta}{F_1 + F_2 \cos\theta}$$

where
F_1 and F_2 = Forces whose resultant is required to be found out,
θ = Angle between the forces F_1 and F_2, and
α = Angle which the resultant force makes with one of the forces (say F_1).

EXAMPLE: Two forces of 100 N and 150 N are acting simultaneously at a point. What is the resultant of these two forces, if the angle between them is 45°?

Solution. Given : First force (F_1) = 100 N; Second force (F_2) = 150 N and angle between F_1 and F_2 (θ) = 45°.
We know that the resultant force,

$$R = \sqrt{F_1^2 + F_2^2 + 2F_1F_2 \cos\theta}$$

$$= \sqrt{(100)^2 + (150)^2 + 2 \times 100 \times 150 \cos 45°} \text{ N}$$

$$= \sqrt{10\,000 + 22\,500 + (30\,000 \times 0.707)} \text{ N}$$

$$= 232 \text{ N} \quad \textbf{Ans.}$$

EXAMPLE: Find the magnitude of the two forces, such that if they act at right angles, their resultant is 10 N . But if they Act at 60°, their resultant is 13 N .

Solution. Given : Two forces $= F_1$ and F_2.

First of all, consider the two forces acting at right angles. We know that when the angle between the two given forces is 90°, then the resultant force (R)

$$\sqrt{10} = \sqrt{F_1^2 + F_2^2}$$

or
$$10 = F_1^2 + F_2^2 \qquad \text{...(Squaring both sides)}$$

Similarly, when the angle between the two forces is 60°, then the resultant force (R)

$$\sqrt{13} = \sqrt{F_1^2 + F_2^2 + 2F_1 F_2 \cos 60°}$$

$\therefore \qquad 13 = F_1^2 + F_2^2 + 2F_1 F_2 \times 0.5 \qquad \text{...(Squaring both sides)}$

or
$\qquad F_1 F_2 = 13 - 10 = 3 \qquad \text{...(Substituting } F_1^2 + F_2^2 = 10)$

We know that $(F_1 + F_{2)})^2 = F_1^2 + F_2^2 + 2F_1 F_2 = 10 + 6 = 16$

$\therefore \qquad F_1 + F_2 = \sqrt{16} = 4 \qquad \qquad \text{...(i)}$

Similarly $\qquad (F_1 - F_2)^2 = F_1^2 + F_2^2 - 2F_1 F_2 = 10 - 6 = 4$

$\therefore \qquad F_1 - F_2 = \sqrt{4} = 2 \qquad \qquad \text{...(ii)}$

Solving equations (i) and (ii),

$\qquad F_1 = 3 \text{ N} \quad \text{and} \quad F_2 = 1 \text{ N} \quad \textbf{Ans.}$

RESOLUTION OF A FORCE: The process of splitting up the given force into a number of components, without changing its effect on the body is called resolution of a force. A force is, generally, resolved along two mutually perpendicular directions. In fact, the resolution of a force is the reverse action of the addition of the component vectors.

PRINCIPLE OF RESOLUTION: It states, "The algebraic sum of the resolved parts of a no. of forces, in a given direction, is equal to the resolved part of their resultant in the same direction."

Note: In general, the forces are resolved in the vertical and horizontal directions.

METHOD OF RESOLUTION:
- Resolve all the forces horizontally and find the algebraic sum of all the horizontal components .
- Resolve all the forces vertically and find the algebraic sum of all the vertical components
- The resultant R of the given forces will be given by the equation;

$$R = \sqrt{(\Sigma H)^2 + (\Sigma V)^2}$$

- The resultant force will be inclined at an angle , with the horizontal, such that

$$\tan \theta = \frac{\Sigma V}{\Sigma H}$$

EXAMPLE: A triangle ABC has its side AB = 40 mm along positive x-axis and side BC = 30 mm along positive y-axis. Three forces of 40 N, 50 N and 30 N act along the sides AB, BC and CA respectively. Determine magnitude of the resultant of such a system of forces.

Solution. The system of given forces is shown in Fig. 2.3.

From the geometry of the figure, we find that the triangle *ABC* is a right angled triangle, in which the *side *AC* = 50 mm. Therefore

$$\sin \theta = \frac{30}{50} = 0.6$$

and

$$\cos \theta = \frac{40}{50} = 0.8$$

Resolving all the forces horizontally (*i.e.*, along *AB*),
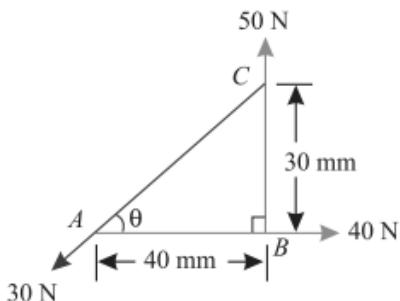
$$\Sigma H = 40 - 30 \cos \theta$$

$$= 40 - (30 \times 0.8) = 16 \text{ N}$$

and now resolving all the forces vertically (*i.e.*, along *BC*)

$$\Sigma V = 50 - 30 \sin \theta$$

$$= 50 - (30 \times 0.6) = 32 \text{ N}$$

We know that magnitude of the resultant force,

$$R = \sqrt{(\Sigma H)^2 + (\Sigma V)^2} = \sqrt{(16)^2 + (32)^2} = 35.8 \text{ N} \quad \textbf{Ans.}$$

LAWS FOR THE RESULTANT FORCE:
The resultant force, of a given system of forces, may also be found out by the following laws
1. Triangle law of forces. **2.** Polygon law of forces.

TRIANGLE LAW OF FORCES:
It states, "If two forces acting simultaneously on a particle, be represented in magnitude and direction by the two sides of a triangle, taken in order ; their resultant may be represented in magnitude and direction by the third side of the triangle, taken in opposite order."

POLYGON LAW OF FORCES:
It is an extension of Triangle Law of Forces for more than two forces, which states, "If a number of forces acting simultaneously on a particle, be represented in magnitude and direction, by the sides of a polygon taken in order ; then the resultant of all these forces may be represented, in magnitude and direction, by the closing side of the polygon, taken in opposite order."

GRAPHICAL (VECTOR) METHOD FOR THE RESULTANT FORCE:
It is another name for finding out the magnitude and direction of the resultant force by the polygon law of forces. It is done as discussed below:
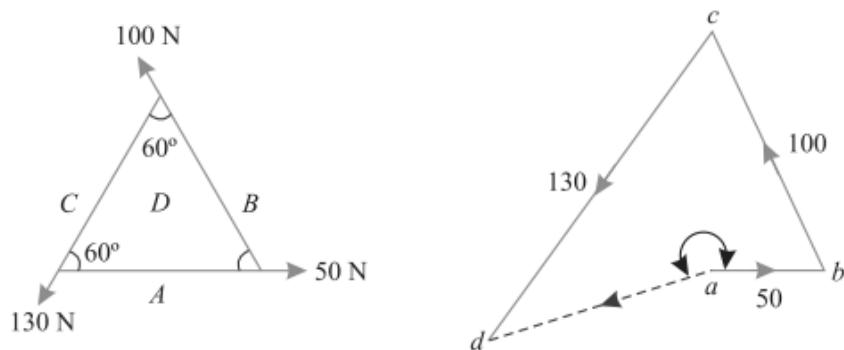
> ➢ **Construction of space diagram (position diagram):** It means the construction of a diagram showing the various forces (or loads) along with their magnitude and lines of action.

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➢ **Use of Bow's notations:** All the forces in the space diagram are named by using the Bow's notations. It is a convenient method in which every force (or load) is named by two capital letters, placed on its either side in the space diagram.

➢ **Construction of vector diagram (force diagram):** It means the construction of a diagram starting from a convenient point and then go on adding all the forces vectorially one by one (keeping in view the directions of the forces) to some suitable scale. Now the closing side of the polygon, taken in opposite order, will give the magnitude of the resultant force (to the scale) and its direction.

EXAMPLE: A particle is acted upon by three forces equal to 50 N, 100 N and 130 N, along the three sides of an equilateral triangle, taken in order. Find graphically the Magnitude and direction of the resultant force.

Solution. The system of given forces is shown in Fig. 2.8 (*a*)

First of all, name the forces according to Bow's notations as shown in Fig. 2.8 (*a*). The 50 N force is named as *AD*, 100 N force as *BD* and 130 N force as *CD*.

(*a*) Space diagram (*b*) Vector diagram

Now draw the vector diagram for the given system of forces as shown in Fig. 2.8 (*b*) and as discussed below :

1. Select some suitable point *a* and draw *ab* equal to 50 N to some suitable scale and parallel to the 50 N force of the space diagram.
2. Through *b*, draw *bc* equal to 100 N to the scale and parallel to the 100 N force of the space diagram.
3. Similarly through *c*, draw *cd* equal to 130 N to the scale and parallel to the 130 N force of the space diagram.
4. Join *ad*, which gives the magnitude as well as direction of the resultant force.
5. By measurement, we find the magnitude of the resultant force is equal to 70 N and acting at an angle of 200° with *ab*. **Ans.**

MOMENT OF A FORCE: It is the turning effect produced by a force, on the body, on which it acts. The moment of a force is equal to the product of the force and the perpendicular distance of the point, about which the moment is required and the line of action of the force.

Mathematically, moment,

$M = P \times l$

where P = Force acting on the body, and
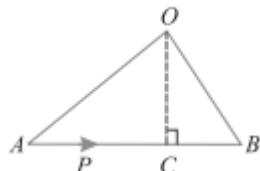
l = Perpendicular distance between the point, about which the moment is required and the line of action of the force.

GRAPHICAL REPRESENTATION OF A MOMENT: Consider a force *P* represented, in magnitude and direction, by the line *AB*. Let *O* be a point, about which the moment of this force is required to be found out, as shown in Fig. From *O*, draw *OC* perpendicular to *AB*. Join *OA* and *OB*.

Now moment of the force *P* about *O*

$= P \times OC = AB \times OC$

But *AB* × *OC* is equal to twice the area of triangle *ABO*. Thus the moment of a force, about any point, is equal to twice the area of the triangle, whose base is the line to some scale representing the force and whose vertex is the point about which the moment is taken.

UNITS OF MOMENT:
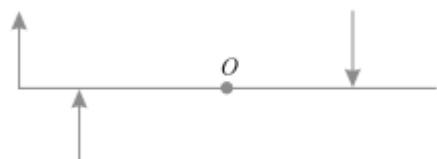
Since the moment of a force is the product of force and distance, therefore the units of the moment will depend upon the units of force and distance. Thus, if the force is in Newton and the distance is in meters, then the units of moment will be Newton-meter (briefly written as N-m). Similarly, the units of moment may be kN-m (*i.e.* kN × m), N-mm (*i.e.* N × mm) etc.

TYPES OF MOMENTS:
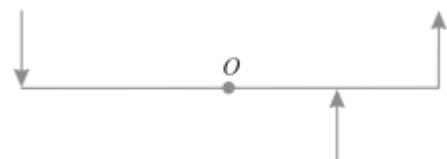
Broadly speaking, the moments are of the following two types:

1. Clockwise moments. 2. Anticlockwise moments.

CLOCKWISE MOMENT:

(*a*) Clockwise moments   (*b*) Anticlockwise moments

It is the moment of a force, whose effect is to turn or rotate the body, about the point in the same direction in which hands of a clock move as shown in Fig.
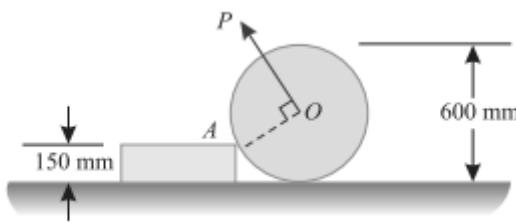
ANTICLOCKWISE MOMENT:
It is the moment of a force, whose effect is to turn or rotate the body, about the point in the opposite direction in which the hands of a clock move as shown in Fig.(*b*).
Note. The general convention is to take clockwise moment as positive and anticlockwise moment as negative.

VARIGNON'S PRINCIPLE OR LAW OF MOMENTS:
It states, "If a number of coplanar forces are acting simultaneously on a particle, the algebraic sum of the moments of all the forces about any point is equal to the moment of their resultant force about the same point."

EXAMPLE: A uniform wheel of 600 mm diameter, weighing 5 kN rests against a rigid rectangular block of 150 mm height as shown in Fig.

Find the least pull, through the centre of the wheel, required just to turn the wheel over the corner A of the block. Also find the reaction on the block. Take all the surfaces to be smooth.

   Solution. Given : Diameter of wheel = 600 mm; Weight of wheel = 5 kN and height of the block = 150 mm.

   Least pull required just to turn the wheel over the corner.

   Let $\qquad\qquad P$ = Least pull required just to turn the wheel in kN.

   A little consideration will show that for the least pull, it must be applied normal to AO. The system of forces is shown in Fig. 3.9. From the geometry of the figure, we find that

$$\sin \theta = \frac{150}{300} = 0.5 \quad \text{or} \quad \theta = 30°$$

and $\qquad\qquad AB = \sqrt{(300)^2 - (150)^2} = 260 \text{ mm}$

   Now taking moments about A and equating the same,
$$P \times 300 = 5 \times 260 = 1300$$

$\therefore \qquad\qquad P = \dfrac{1300}{300} = 4.33 \text{ kN} \quad \textbf{Ans.}$
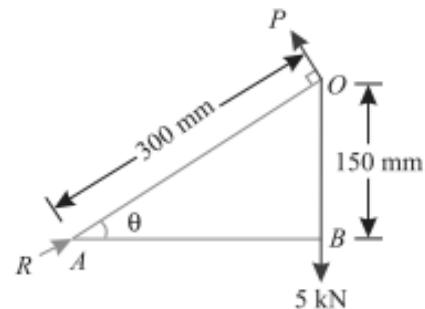
Reaction on the block

   Let $\qquad\qquad R$ = Reaction on the block in kN.

   Resolving the forces horizontally and equating the same,
$$R \cos 30° = P \sin 30°$$

$\therefore \qquad\qquad R = \dfrac{P \sin 30°}{\cos 30°} = \dfrac{4.33 \times 0.5}{0.866} = 2.5 \text{ kN} \quad \textbf{Ans.}$

EXAMPLE: Four forces equal to P, 2P, 3P and 4P are respectively acting along the four sides of square ABCD taken in order. Find the magnitude, direction and position of the resultant force.

Solution. The system of given forces is shown in Fig. 3.12.

Magnitude of the resultant force

Resolving all the forces horizontally,

$$\Sigma H = P - 3P = -2P \qquad \qquad ...(i)$$

and now resolving all forces vertically,

$$\Sigma V = 2P - 4P = -2P \qquad \qquad ...(ii)$$

We know that magnitude of the resultant forces,

$$R = \sqrt{(\Sigma H)^2 + (\Sigma V)^2} = \sqrt{(-2P)^2 + (-2P)^2}$$

$$= 2\sqrt{2}P \quad \textbf{Ans.}$$

Direction of the resultant force

Let $\qquad \theta =$ Angle, which the resultant makes with the horizontal.

$$\therefore \qquad \tan \theta = \frac{\Sigma V}{\Sigma H} = \frac{-2P}{-2P} = 1 \quad \text{or} \quad \theta = 45°$$

Since ΣH as well as ΣV are –ve, therefore resultant lies between 180°and 270°. Thus actual angle of the resultant force = 180° + 45° = 225° **Ans.**

Position of the resultant force

Let $\qquad x =$ Perpendicular distance between A and the line of action of the resultant force.

Now taking moments of the resultant force about A and equating the same,

$$2\sqrt{2}P \times x = (2P \times a) + (3P \times a) = 5P \times a$$

$$\therefore \qquad x = \frac{5a}{2\sqrt{2}} \quad \textbf{Ans.}$$

Note. The moment of the forces P and $4P$ about the point A will be zero, as they pass through it.

COUPLE: A pair of two equal and unlike parallel forces (*i.e.*forces equal in magnitude, with lines of action parallel to each other and acting in opposite directions) is known as a couple.

As a matter of fact, a couple is unable to produce any translatory motion (*i.e.*, motion in a straight line). But it produces a motion of rotation in the body, on which it acts. The simplest example of a couple is the forces applied to the key of a lock, while locking or unlocking it.

ARM OF A COUPLE:

The perpendicular distance (*a*), between the lines of action of the two equal and opposite parallel forces, is known *as arm of the couple* as shown in Fig.

MOMENT OF A COUPLE: The moment of a couple is the product of the force (*i.e.*, one of the forces of the two equal and opposite parallel forces) and the arm of the couple.
Mathematically:
Moment of a couple = $P \times a$
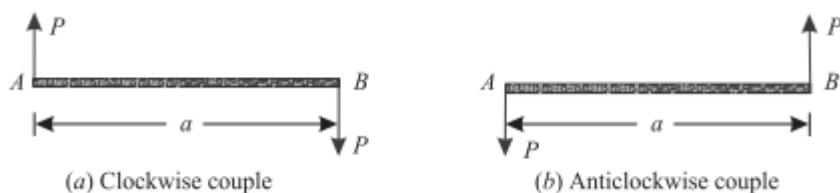
where
P = Magnitude of the force, and
a = Arm of the couple.

CLASSIFICATION OF COUPLES:
The couples may be, broadly, classified into the following two categories, depending upon their direction, in which the couple tends to rotate the body, on which it acts:

1. Clockwise couple, and 2. Anticlockwise couple

(a) Clockwise couple   (b) Anticlockwise couple

CLOCKWISE COUPLE:
A couple, whose tendency is to rotate the body, on which it acts, in a clockwise direction, is known as a clockwise couple as shown in Fig. (*a*). Such a couple is also called positive couple.

ANTICLOCKWISE COUPLE:
 A couple, whose tendency is to rotate the body, on which it acts, in an anticlockwise direction, is known as an anticlockwise couple as shown in Fig. (*b*). Such a couple is also called a negative couple.

CHARACTERISTICS OF A COUPLE:
 A couple (whether clockwise or anticlockwise) has the following characteristics:

1. The algebraic sum of the forces, constituting the couple, is zero.
2. The algebraic sum of the moments of the forces, constituting the couple, about any point is the same, and equal to the moment of the couple itself.
3. A couple cannot be balanced by a single force. But it can be balanced only by a couple of opposite sense.
4. Any no. of co-planer couples can be reduced to a single couple, whose magnitude will be equal to the algebraic sum of the moments of all the couples.

EXAMPLE: A square ABCD has forces acting along its sides as shown in Fig. 4.13. Find the values of P and Q, if the system reduces to a couple. Also find magnitude of the couple, if the side of the square is 1 m.

Solution. Given : Length of square = 1 m

Values of P and Q

We know that if the system reduces to a couple, the resultant force in horizontal and vertical directions must be zero. Resolving the forces horizontally,

$$100 - 100 \cos 45° - P = 0$$

$$\therefore \qquad P = 100 - 100 \cos 45° \text{ N}$$

$$= 100 - (100 \times 0.707) = 29.3 \text{ N } \textbf{ Ans.}$$

Now resolving the forces vertically,

$$200 - 100 \sin 45° - Q = 0$$

$$\therefore \qquad Q = 200 - (100 \times 0.707) = 129.3 \text{ N } \textbf{ Ans.}$$
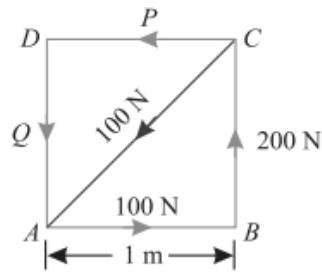
Magnitude of the couple

We know that moment of the couple is equal to the algebraic sum of the moments about any point. Therefore moment of the couple (taking moments about *A*)

$$= (-200 \times 1) + (-P \times 1) = -200 - (29.3 \times 1) \text{ N-m}$$

$$= -229.3 \text{ N-m } \textbf{Ans.}$$

Since the value of moment is negative, therefore the couple is anticlockwise.

2. EQUILIBRIUM

EQUILIBRIUM:
 If the resultant of a number of forces, acting on a particle is zero, the particle will be in equilibrium. Such a set of forces, whose resultant is zero, are called equilibrium forces. The force, which brings the set of forces in equilibrium is called an equilibrant.
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PRINCIPLES OF EQUILIBRIUM:
Though there are many principles of equilibrium, yet the following three are important from the subject point of view :
1. **Two force principle:**
 As per this principle, if a body in equilibrium is acted upon by two
forces, then they must be equal, opposite and collinear.
2. **Three force principle**:
As per this principle, if a body in equilibrium is acted upon by three
forces, then the resultant of any two forces must be equal, opposite and collinear with the third force.
3. **Four force principle:**
 As per this principle, if a body in equilibrium is acted upon by four
forces, then the resultant of any two forces must be equal, opposite and collinear with the resultant of the other two forces.

METHODS FOR THE EQUILIBRIUM OF COPLANAR FORCES:

Though there are many methods of studying the equilibrium of forces, yet the following are important from the subject point of view :
1. Analytical method. 2. Graphical method.

LAMI'S THEOREM:
It states, "If three coplanar forces acting at a point be in equilibrium, then each force is proportional to the sine of the angle between the other two."
 Mathematically,

$$\frac{P}{\sin \alpha} = \frac{Q}{\sin \beta} = \frac{R}{\sin \gamma}$$

Proof

Consider three coplanar forces P, Q, and R acting at a point O. Let the opposite angles to three forces be α, β and γ as shown in Fig. 5.2.

Now let us complete the parallelogram $OACB$ with OA and OB as adjacent sides as shown in the figure. We know that the resultant of two forces P and Q will be given by the diagonal OC both in magnitude and direction of the parallelogram $OACB$.

Since these forces are in equilibrium, therefore the resultant of the forces P and Q must be in line with OD and equal to R, but in opposite direction.

From the geometry of the figure, we find

$$BC = P \text{ and } AC = Q$$

$\therefore$
$$\angle AOC = (180° - \beta)$$

and
$$\angle ACO = \angle BOC = (180° - \alpha)$$

$\therefore$
$$\angle CAO = 180° - (\angle AOC + \angle ACO)$$
$$= 180° - [(180° - \beta) + (180° - \alpha)]$$
$$= 180° - 180° + \beta - 180° + \alpha$$
$$= \alpha + \beta - 180°$$

But
$$\alpha + \beta + \gamma = 360°$$

Subtracting $180°$ from both sides of the above equation,

$$(\alpha + \beta - 180°) + \gamma = 360° - 180° = 180°$$
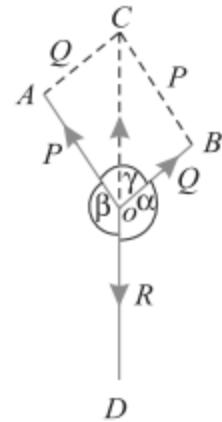
or
$$\angle CAO = 180° - \gamma$$

We know that in triangle AOC,
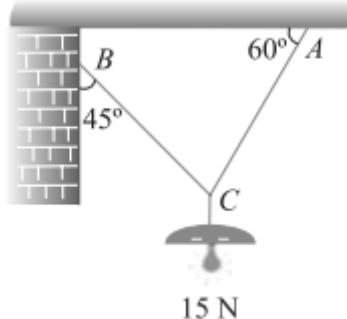
$$\frac{OA}{\sin \angle ACO} = \frac{AC}{\sin \angle AOC} = \frac{OC}{\sin \angle CAO}$$

$$\frac{OA}{\sin (180° - \alpha)} = \frac{AC}{\sin (180° - \beta)} = \frac{OC}{\sin (180° - \gamma)}$$

or
$$\frac{P}{\sin \alpha} = \frac{Q}{\sin \beta} = \frac{R}{\sin \gamma}$$

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EXAMPLE: An electric light fixture weighting 15 N hangs from a point C, by two strings AC and BC. The string AC is inclined at 60° to the horizontal and BC at 45° to the horizontal as shown in Fig. Using Lami's theorem, or otherwise, determine the force in the strings AC.

15 N

Solution. Given : Weight at $C = 15$ N

Let T_{AC} = Force in the string AC, and
T_{BC} = Force in the string BC.

The system of forces is shown in Fig. 5.4. From the geometry of the figure, we find that angle between T_{AC} and 15 N is 150° and angle between T_{BC} and 15 N is 135°.
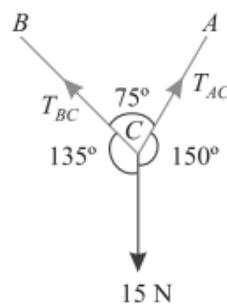
∴ $\angle ACB = 180° - (45° + 60°) = 75°$

Applying Lami's equation at C,

$$\frac{15}{\sin 75°} = \frac{T_{AC}}{\sin 135°} = \frac{T_{BC}}{\sin 150°}$$

or

$$\frac{15}{\sin 75°} = \frac{T_{AC}}{\sin 45°} = \frac{T_{BC}}{\sin 30°}$$

∴ $$T_{AC} = \frac{15\sin 45°}{\sin 75°} = \frac{15 \times 0.707}{0.9659} = 10.98\,\text{N} \quad \textbf{Ans.}$$

EXAMPLE: Two equal heavy spheres of 50 mm radius are in equilibrium within a smooth cup of 150 mm radius. Show that the reaction between the cup of one sphere is double than that between the two spheres.

Solution. Given : Radius of spheres = 50 mm and radius of the cup = 150 mm.
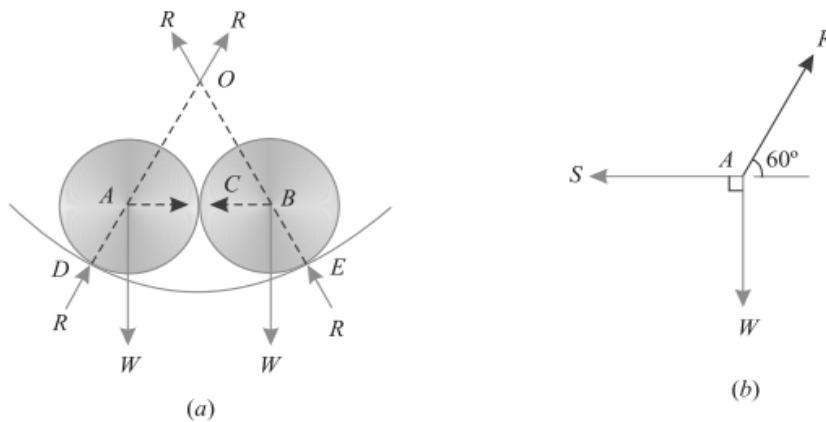

Fig. 5.11.

The two spheres with centres A and B, lying in equilibrium, in the cup with O as centre are shown in Fig. 5.11 (a). Let the two spheres touch each other at C and touch the cup at D and E respectively.

Let R = Reactions between the spheres and cup, and
S = Reaction between the two spheres at C.

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From the geometry of the figure, we find that $OD = 150$ mm and $AD = 50$ mm. Therefore $OA = 100$ mm. Similarly $OB = 100$ mm. We also find that $AB = 100$ mm. Therefore OAB is an equilateral triangle. The system of forces at A is shown in Fig. 5.11 (b).

Applying Lami's equation at A,

$$\frac{R}{\sin 90°} = \frac{W}{\sin 120°} = \frac{S}{\sin 150°}$$

$$\frac{R}{1} = \frac{W}{\sin 60°} = \frac{S}{\sin 30°}$$

$$\therefore \quad R = \frac{S}{\sin 30°} = \frac{S}{0.5} = 2S$$

Hence the reaction between the cup and the sphere is double than that between the two spheres. **Ans.**

GRAPHICAL METHOD FOR THE EQUILIBRIUM OF COPLANAR FORCES:

We have studied that the equilibrium of forces by analytical method. Sometimes, the analytical method is too tedious and complicated. The equilibrium of such forces may also be studied, graphically, by drawing the vector diagram. This may also be done by studying the

1. Converse of the Law of Triangle of Forces
2. Converse of the Law of Polygon of Forces
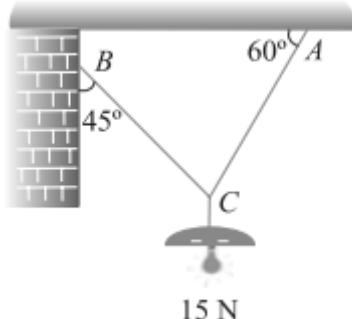
CONVERSE OF THE LAW OF TRIANGLE OF FORCES:

If three forces acting at a point be represented in magnitude and direction by the three sides a triangle, taken in order, the forces shall be in equilibrium.

CONVERSE OF THE LAW† OF POLYGON OF FORCES:

If any number of forces acting at a point be represented in magnitude and direction by the sides of a closed polygon, taken in order, the forces shall be in equilibrium.

EXAMPLE: An electric light fixture weighing 15 N hangs from a point C, by two strings AC and BC. The string AC is inclined at 60° to the horizontal and BC at 45° to the horizontal as shown in Fig.


15 N
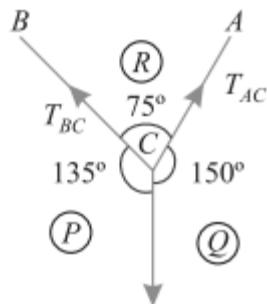
SOLUTION: Given. Weight at *C* = 15 N
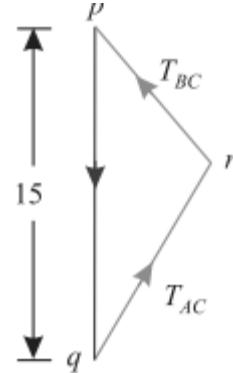
T_{AC} = Force in the string *AC*, and

T_{BC} = Force in the string *BC*.

First of all, draw the space diagram for the joint *C* and name the forces according to Bow's notations as shown in Fig. The force *TAC* is named as *RQ* and the force *TBC* as *PR*.

(*a*) Space diagram   (*a*) Vector diagram

Now draw the vector diagram for the given system of forces as shown in Fig. (*b*) and as discussed below;
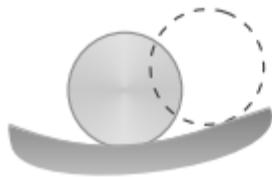
- Select some suitable point p and draw a vertical line pq equal to 15 N to some suitable scale representing weight (PQ) of the electric fixture.
- Through p draw a line pr parallel to PR and through q, draw a line qr parallel to QR. Let these two lines meet at r and close the triangle pqr, which means that joint C is in equilibrium.
- By measurement, we find that the forces in strings AC (TAC) and BC (TPC) is equal to 1.0 N and 7.8 N respectively.

CONDITIONS OF EQUILIBRIUM: If the body is completely at rest, it necessarily means that there is neither a resultant force nor a couple acting on it. A little consideration will show, that in this case the following conditions are already satisfied:
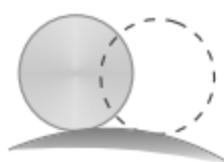
$$\Sigma H = 0 \ \Sigma V = 0 \text{ and } \Sigma M = 0$$

The above mentioned three equations are known as the conditions of equilibrium.
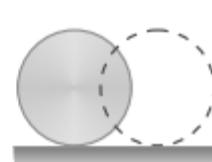
TYPES OF EQUILIBRIUM:

(*a*) Stable   (*b*) Unstable   (*c*) Neutral

1. **STABLE EQUILIBRIUM:** A body is said to be in stable equilibrium, if it returns back to its original position, after it is slightly displaced from its position of rest. This happens when some additional force sets up due to displacement and brings the body back to its original position. A smooth cylinder, lying in a curved surface, is in stable equilibrium. If we slightly displace the cylinder from its position of rest (as shown by dotted lines), it will tend to return back to its original position in order to bring its weight normal to horizontal axis as shown in Fig. (*a*).

2. **UNSTABLE EQUILIBRIUM:** A body is said to be in an unstable equilibrium, if it does not return back to its original position, and heels farther away, after slightly displaced from its position of rest. This happens when the additional force moves the body away from its position of rest. This happens when the additional force moves the body away from its position of rest. A smooth cylinder lying on a convex surface is in unstable equilibrium. If we slightly displace the cylinder from its position of rest (as shown by dotted lines) the body will tend to move away from its original position as shown in Fig. (*b*).

3. **NEUTRAL EQUILIBRIUM:** A body is said to be in a neutral equilibrium, if it occupies a new position (and remains at rest in this position) after slightly displaced from its position of rest. This happens when no additional force sets up due to the displacement. A smooth cylinder lying on a horizontal plane is in neutral equilibrium as shown in Fig. (*c*).

3. FRICTION

INTRODUCTION:
    If a block of one substance is placed over the level surface of the same or different material, a certain degree of interlocking of the minutely projecting particles takes place. This does not involve any force, so long as the block does not move or tends to move. But whenever one of the blocks moves or tends to move tangentially with respect to the surface, on which it rests, the interlocking property of the projecting particles opposes the motion. This opposing force, which acts in the opposite direction of the movement of the block, is called *force of friction* or simply *friction.* It is of the following two types:
1. Static friction. **2.** Dynamic friction

STATIC FRICTION:
    It is the friction experienced by a body when it is at rest. Or in other words, it is the friction when the body tends to move.

DYNAMIC FRICTION:
    It is the friction experienced by a body when it is in motion. It is also called kinetic friction. The dynamic friction is of the following two types:
1. **Sliding friction**: It is the friction, experienced by a body when it slides over another body.
2. **Rolling friction**: It is the friction, experienced by a body when it rolls over another body.

LIMITING FRICTION: The maximum value of frictional force, which comes into play, when a body just begins to slide over the surface of the other body, is known as limiting friction. It may be noted that when the applied force is less than the limiting friction, the body remains at rest, and the friction is called static friction, which may have any value between zero and limiting friction.

COEFFICIENT OF FRICTION:
It is the ratio of limiting friction to the normal reaction, between the two bodies, and is generally denoted by μ.
Mathematically, coefficient of friction,

$$\mu = \frac{F}{R} = \tan \phi \qquad \text{or} \qquad F = \mu R$$
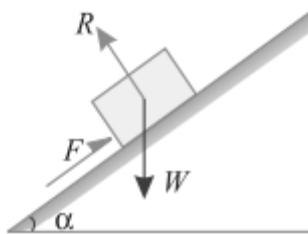
ϕ = Angle of friction,

F = Limiting friction, and

R = Normal reaction between the two bodies.

ANGLE OF FRICTION: Consider a body of weight *W* resting on an inclined plane as shown in Fig. We know that the body is in equilibrium under the action of the following forces:
1. Weight (*W*) of the body, acting vertically downwards,
2. Friction force (*F*) acting upwards along the plane, and
3. Normal reaction (*R*) acting at right angles to the plane.

Let the angle of inclination (α) be gradually increased, till the body just starts sliding down the plane. This angle of inclined plane, at which a body just begins to slide down the plane, is called the angle of friction. This is also equal to the angle, which the normal reaction makes with the vertical.

ANGLE OF REPOSE: Angle of repose is defined as the angle of the inclined plane with horizontal such that a body placed on it just begins to slide.

LAWS OF FRICTION:
Prof. Coulomb, after extensive experiments, gave some laws of friction, which may be grouped under the following heads :
1. Laws of static friction, and
2. Laws of kinetic or dynamic friction

LAWS OF STATIC FRICTION:

Following are the laws of static friction :
1. The force of friction always acts in a direction, opposite to that in which the body tends to move, if the force of friction would have been absent.
2. The magnitude of the force of friction is exactly equal to the force, which tends to move the body.
3. The magnitude of the limiting friction bears a constant ratio to the normal reaction between the two surfaces. Mathematically:
$$F/R = CONSTANT$$
4. The force of friction is independent of the area of contact between the two surfaces.
5. The force of friction depends upon the roughness of the surfaces

LAWS OF KINETIC OR DYNAMIC FRICTION:

Following are the laws of kinetic or dynamic friction:
1. The force of friction always acts in a direction, opposite to that in which the body is moving.
2. The magnitude of kinetic friction bears a constant ratio to the normal reaction between the two surfaces. But this ratio is slightly less than that in case of limiting friction.
3. For moderate speeds, the force of friction remains constant. But it decreases slightly with the increase of speed.

ADVANTAGES OF FRICTION:
- Friction is responsible for many types of motion
- It helps us walk on the ground
- Brakes in a car make use of friction to stop the car

- Asteroids are burnt in the atmosphere before reaching Earth due to friction.
- It helps in the generation of heat when we rub our hands.

DISADVANTAGES OF FRICTION:
- Friction produces unnecessary heat leading to the wastage of energy.
- The force of friction acts in the opposite direction of motion, so friction slows down the motion of moving objects.
- A lot of money goes into preventing friction and the usual wear and tear caused by it by using techniques like greasing and oiling.

EQUILIBRIUM OF A BODY ON A ROUGH HORIZONTAL PLANE:

We know that a body, lying on a rough horizontal plane will remain in equilibrium. But whenever a force is applied on it, the body will tend to move in the direction of the force. In such cases, equilibrium of the body is studied first by resolving the forces horizontally and then vertically.
Now the value of the force of friction is obtained from the relation:

$$F = Mr$$

EXAMPLE: A body of weight 300 N is lying on a rough horizontal plane having a coefficient of friction as 0.3. Find the magnitude of the force, which can move the body, while acting at an angle of 25° with the horizontal.

Solution. Given: Weight of the body $(W) = 300$ N; Coefficient of friction $(\mu) = 0.3$ and angle made by the force with the horizontal $(\alpha) = 25°$

Let $\qquad P =$ Magnitude of the force, which can move the body, and

$\qquad F =$ Force of friction.

Resolving the forces horizontally,

$$F = P \cos \alpha = P \cos 25° = P \times 0.9063$$

and now resolving the forces vertically,
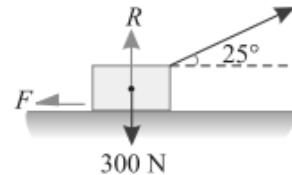
$$R = W - P \sin \alpha = 300 - P \sin 25°$$
$$= 300 - P \times 0.4226$$

We know that the force of friction (F),

$$0.9063 \, P = \mu R = 0.3 \times (300 - 0.4226 \, P) = 90 - 0.1268 \, P$$
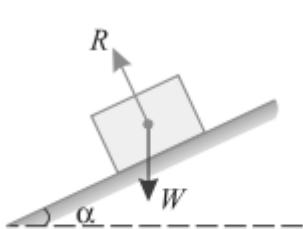
or $\qquad 90 = 0.9063 \, P + 0.1268 \, P = 1.0331 \, P$

$\therefore \qquad P = \dfrac{90}{1.0331} = 87.1 \text{ N} \qquad$ **Ans.**
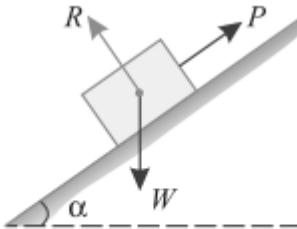
EQUILIBRIUM OF A BODY ON A ROUGH INCLINED PLANE:
Consider a body, of weight W, lying on a rough plane inclined at an angle α with the horizontal as shown in Fig.(a) and (b).

(*a*) Angle of inclination less than the angle of friction

(*b*) Angle of inclination more than the angle of friction

A little consideration will show, that if the inclination of the plane, with the horizontal, is less the angle of friction, the body will be automatically in equilibrium as shown in Fig. (*a*). If in this condition, the body is required to be moved upwards or downwards, a corresponding force is required, for the same. But, if the inclination of the plane is more than the angle of friction, the body will move down. And an upward force (*P*) will be required to resist the body from moving down the plane as shown in Fig. (b).

Though there are many types of forces, for the movement of the body, yet the following are important from the subject point of view :
1. Force acting along the inclined plane.
2. Force acting horizontally.
3. Force acting at some angle with the inclined plane.

EQUILIBRIUM OF A BODY ON A ROUGH INCLINED PLANE SUBJECTED TO A FORCE ACTING ALONG THE INCLINED PLANE:
Consider a body lying on a rough inclined plane subjected force acting along the inclined plane, which keeps it in equilibrium as shown in Fig.(a) and (b).

Let W = Weight of the body,
α = Angle, which the inclined plane makes with the horizontal,
R = Normal reaction,
μ = Coefficient of friction between the body and the inclined plane, and
ϕ = Angle of friction, such that $\mu = \tan \phi$.

A little consideration will show that if the force is not there, the body will slide down the plane. Now we shall discuss the following two cases:

1. Minimum force (P1) which will keep the body in equilibrium, when it is at the point of sliding downwards:

(*a*) Body at the point of sliding downwards

(*b*) Body at the point of sliding upwards

In this case, the force of friction ($F_1 = \mu.R_1$) will act upwards, as the body is at the point of sliding downwards as shown in Fig. 8.8 (a). Now resolving the forces along the plane,

$$P_1 = W \sin \alpha - \mu.R_1 \qquad \qquad ...(i)$$

and now resolving the forces perpendicular to the plane.

$$R_1 = W \cos \alpha \qquad \qquad ...(ii)$$

Substituting the value of R_1 in equation (i),

$$P_1 = W \sin \alpha - \mu \, W \cos \alpha = W (\sin \alpha - \mu \cos \alpha)$$

and now substituting the value of $\mu = \tan \phi$ in the above equation,

$$P_1 = W (\sin \alpha - \tan \phi \cos \alpha)$$

Multiplying both sides of this equation by $\cos \phi$,

$$P_1 \cos \phi = W (\sin \alpha \cos \phi - \sin \phi \cos \alpha) = W \sin (\alpha - \phi)$$

$$\therefore \qquad P_1 = W \times \frac{\sin (\alpha - \phi)}{\cos \phi}$$

2. *Maximum force (P_2) which will keep the body in equilibrium, when it is at the point of sliding upwards.*

In this case, the force of friction ($F_2 = \mu.R_2$) will act downwards as the body is at the point of sliding upwards as shown in Fig. 8.8 (b). Now resolving the forces along the plane,

$$P_2 = W \sin \alpha + \mu.R_2 \qquad \qquad ...(i)$$

and now resolving the forces perpendicular to the plane,

$$R_2 = W \cos \alpha \qquad \qquad ...(ii)$$

Substituting the value of R_2 in equation (i),

$$P_2 = W \sin \alpha + \mu \, W \cos \alpha = W (\sin \alpha + \mu \cos \alpha)$$

and now substituting the value of $\mu = \tan \phi$ in the above equation,
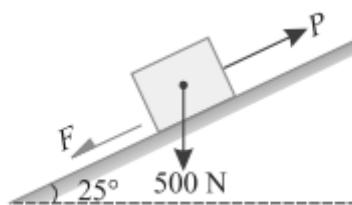
$$P_2 = W (\sin \alpha + \tan \phi \cos \alpha)$$

Multiplying both sides of this equation by $\cos \phi$,

$$P_2 \cos \phi = W (\sin \alpha \cos \phi + \sin \phi \cos \alpha) = W \sin (\alpha + \phi)$$

$$\therefore \qquad P_2 = W \times \frac{\sin (\alpha + \phi)}{\cos \phi}$$

EXAMPLE: A body of weight 500 N is lying on a rough plane inclined at an angle of 25° with the horizontal. It is supported by an effort (P) parallel to the plane as shown in Fig. Determine the minimum and maximum values of P, for which the equilibrium can exist, if the angle of friction is 20°.

Solution. Given: Weight of the body $(W) = 500$ N ; Angle at which plane is inclined (α) = $25°$ and angle of friction $(\phi) = 20°$.

Minimum value of P

We know that for the minimum value of P, the body is at the point of sliding downwards. We also know that when the body is at the point of sliding downwards, then the force

$$P_1 = W \times \frac{\sin(\alpha - \phi)}{\cos \phi} = 500 \times \frac{\sin(25° - 20°)}{\cos 20°} \text{ N}$$

$$= 500 \times \frac{\sin 5°}{\cos 20°} = 500 \times \frac{0.0872}{0.9397} = 46.4 \text{ N} \qquad \textbf{Ans.}$$

Maximum value of P

We know that for the maximum value of P, the body is at the point of sliding upwards. We also know that when the body is at the point of sliding upwards, then the force

$$P_2 = W \times \frac{\sin(\alpha + \phi)}{\cos \phi} = 500 \times \frac{\sin(25° + 20°)}{\cos 20°} \text{ N}$$

$$= 500 \times \frac{\sin 45°}{\cos 20°} = 500 \times \frac{0.7071}{0.9397} = 376.2 \text{ N} \qquad \textbf{Ans.}$$

EQUILIBRIUM OF A BODY ON A ROUGH INCLINED PLANE SUBJECTED TO A FORCE ACTING HORIZONTALLY:

Consider a body lying on a rough inclined plane subjected to a force acting horizontally, which keeps it in equilibrium as shown in Fig.(*a*) and (*b*).

W = Weight of the body,
α = Angle, which the inclined plane makes with the horizontal,
R = Normal reaction,
μ = Coefficient of friction between the body and the inclined plane, and
ϕ = Angle of friction, such that $\mu = \tan \phi$.

A little consideration will show that if the force is not there, the body will slide down on the plane.
Now we shall discuss the following two cases:

1. *Minimum force* (P_1) *which will keep the body in equilibrium, when it is at the point of sliding downwards.*

(a) Body at the point of
sliding downwards

(b) Body at the point of
sliding upwards
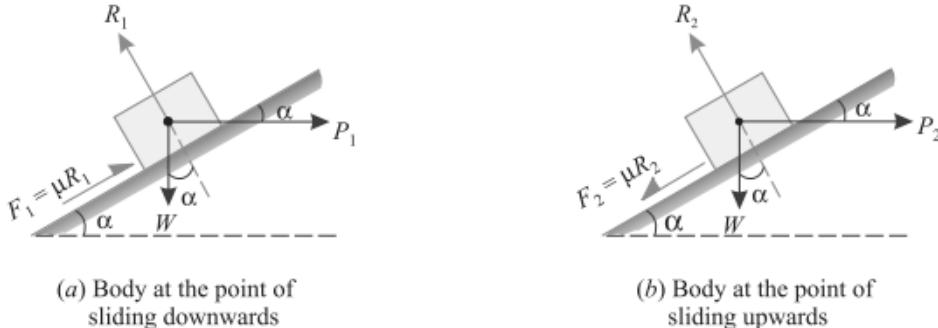
Fig. 8.13.

In this case, the force of friction $(F_1 = \mu.R_1)$ will act upwards, as the body is at the point of sliding downwards as shown in Fig. 8.13. (*a*). Now resolving the forces along the plane,

$$P_1 \cos \alpha = W \sin \alpha - \mu R_1 \qquad \qquad ...(i)$$

and now resolving the forces perpendicular to the plane,

$$R_1 = W \cos \alpha + P_1 \sin \alpha \qquad \qquad ...(ii)$$

Substituting this value of R_1 in equation (*i*),

$$P_1 \cos \alpha = W \sin \alpha - \mu(W \cos \alpha + P_1 \sin \alpha)$$
$$= W \sin \alpha - \mu W \cos \alpha - \mu P_1 \sin \alpha$$
$$P_1 \cos \alpha + \mu P_1 \sin \alpha = W \sin \alpha - \mu W \cos \alpha$$

$$P_1(\cos \alpha + \mu \sin \alpha) = W (\sin \alpha - \mu \cos \alpha)$$

$$\therefore \qquad P_1 = W \times \frac{(\sin \alpha - \mu \cos \alpha)}{(\cos \alpha + \mu \sin \alpha)}$$

Now substituting the value of $\mu = \tan \phi$ in the above equation,

$$P_1 = W \times \frac{(\sin \alpha - \tan \phi \cos \alpha)}{(\cos \alpha + \tan \phi \sin \alpha)}$$

Multiplying the numerator and denominator by $\cos \phi$,

$$P_1 = W \times \frac{\sin \alpha \cos \phi - \sin \phi \cos \alpha}{\cos \alpha \cos \phi + \sin \alpha \sin \phi} = W \times \frac{\sin (\alpha - \phi)}{\cos (\alpha - \phi)}$$

$$= W \tan (\alpha - \phi) \qquad \qquad ...(\text{when } \alpha > \phi)$$
$$= W \tan (\phi - \alpha) \qquad \qquad ...(\text{when } \phi > \alpha)$$

2. *Maximum force* (P_2) *which will keep the body in equilibrium, when it is at the point of sliding upwards*

In this case, the force of friction $(F_2 = \mu R_2)$ will act downwards, as the body is at the point of sliding upwards as shown in Fig.8.12. (*b*). Now resolving the forces along the plane,

$$P_2 \cos \alpha = W \sin \alpha + \mu R_2 \qquad \qquad ...(iii)$$

and now resolving the forces perpendicular to the plane,

$$R_2 = W \cos \alpha + P_2 \sin \alpha \qquad \qquad ...(iv)$$

Substituting this value of R_2 in the equation (*iii*),

$$P_2 \cos \alpha = W \sin \alpha + \mu (W \cos \alpha + P_2 \sin \alpha)$$
$$= W \sin \alpha + \mu W \cos \alpha + \mu P_2 \sin \alpha$$

$$P_2 \cos \alpha - \mu P_2 \sin \alpha = W \sin \alpha + \mu W \cos \alpha$$
$$P_2 (\cos \alpha - \mu \sin \alpha) = W (\sin \alpha + \mu \cos \alpha)$$

$$\therefore \qquad P_2 = W \times \frac{(\sin \alpha + \mu \cos \alpha)}{(\cos \alpha - \mu \sin \alpha)}$$

Now substituting the value of $\mu = \tan \phi$ in the above equation,

$$P_2 = W \times \frac{\sin \alpha + \tan \phi \cos \alpha}{\cos \alpha - \tan \phi \sin \alpha}$$

Multiplying the numerator and denominator by $\cos \phi$,

$$P_2 = W \times \frac{\sin \alpha \cos \phi + \sin \phi \cos \alpha}{\cos \alpha \cos \phi - \sin \phi \sin \alpha} = W \times \frac{\sin (\alpha + \phi)}{\cos (\alpha + \phi)}$$

$$= W \tan (\alpha + \phi)$$
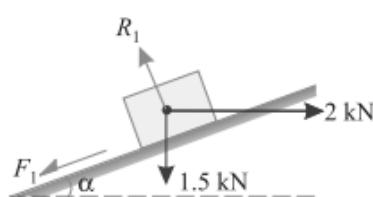
EXAMPLE: A load of 1.5 kN, resting on an inclined rough plane, can be moved up the plane by a force of 2 kN applied horizontally or by a force 1.25 kN applied parallel to the plane. Find the inclination of the plane and the coefficient of friction.

Solution. Given: Load $(W) = 1.5$ kN; Horizontal effort $(P_1) = 2$ kN and effort parallel to the inclined plane $(P_2) = 1.25$ kN.
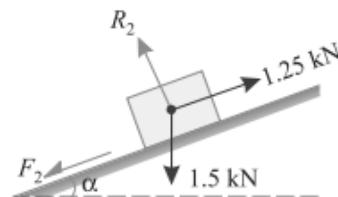
Inclination of the plane

Let $\qquad \alpha =$ Inclination of the plane, and
$\qquad \qquad \phi =$ Angle of friction.

(*a*) Horizontal force   (*b*) Force parallel to the plane

First of all, consider the load of 1.5 kN subjected to a horizontal force of 2 kN as shown in Fig. 8.14 (a). We know that when the force is applied horizontally, then the magnitude of the force, which can move the load up the plane.

$$P = W \tan (\alpha + \phi)$$

or
$$2 = 1.5 \tan (\alpha + \phi)$$

∴
$$\tan (\alpha + \phi) = \frac{2}{1.5} = 1.333 \quad \text{or} \quad (\alpha + \phi) = 53.1°$$

Now consider the load of 1.5 kN subjected to a force of 1.25 kN along the plane as shown in Fig. 8.14 (b). We Know that when the force is applied parallel to the plane, then the magnitude of the force, which can move the load up the plane,

$$P = W \times \frac{\sin (\alpha + \phi)}{\cos \phi}$$

or
$$1.25 = 1.5 \times \frac{\sin 53.1°}{\cos \phi} = 1.5 \times \frac{0.8}{\cos \phi} = \frac{1.2}{\cos \phi}$$

∴
$$\cos \phi = \frac{1.2}{1.25} = 0.96 \quad \text{or} \quad \phi = 16.3°$$

and
$$\alpha = 53.1° - 16.3° = 36.8° \quad \textbf{Ans.}$$

Coefficient of friction

We know that the coefficient of friction,

$$\mu = \tan \phi = \tan 16.3° = 0.292 \quad \textbf{Ans.}$$

EQUILIBRIUM OF A BODY ON A ROUGH INCLINED PLANE SUBJECTED TO A FORCE ACTING AT SOME ANGLE WITH THE INCLINED PLANE:

Consider a body lying on a rough inclined plane subjected to a force acting at some angle with the inclined plane, which keeps it in equilibrium as shown in Fig.(a) and (b).

Let W = Weight of the body,
α = Angle which the inclined plane makes with the horizontal,
θ = Angle which the force makes with the inclined surface,
R = Normal reaction,
μ = Coefficient of friction between the body and the inclined plane, and
ϕ = Angle of friction, such that $\mu = \tan \phi$.
A little consideration will show that if the force is not there, the body will slide down the plane.
Now we shall discuss the following two cases :

1. *Minimum force* (P_1) *which will keep the body in equilibrium when it is at the point of sliding downwards.*

(a) Body at the point of sliding downwards

(b) Body at the point of sliding upwards
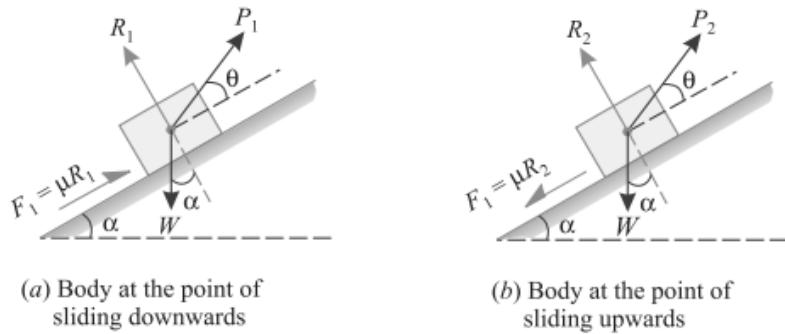
Fig. 8.17.

In this case, the force of friction $(F_1 = \mu R_1)$ will act upwards, as the body is at the point of sliding downwards as shown in Fig. 8.17 (a). Now resolving the forces along the plane,

$$P_1 \cos \theta = W \sin \alpha - \mu R_1 \qquad \qquad ...(i)$$

and now resolving the forces perpendicular to the plane,

$$R_1 = W \cos \alpha - P_1 \sin \theta \qquad \qquad ...(ii)$$

Substituting the value of R_1 in equation (i),

$$P_1 \cos \theta = W \sin \alpha - \mu (W \cos \alpha - P_1 \sin \theta)$$
$$= W \sin \alpha - \mu W \cos \alpha + \mu P_1 \sin \theta$$
$$P_1 \cos \theta - \mu P_1 \sin \theta = W \sin \alpha - \mu W \cos \alpha$$
$$P_1(\cos \theta - \mu \sin \theta) = W (\sin \alpha - \mu \cos \alpha)$$

$$\therefore \qquad P_1 = W \times \frac{(\sin \alpha - \mu \cos \alpha)}{(\cos \theta - \mu \sin \theta)}$$

and now substituting the value of $\mu = \tan \phi$ in the above equation,

$$P_1 = W \times \frac{(\sin \alpha - \tan \phi \cos \alpha)}{(\cos \theta - \tan \phi \sin \theta)}$$

Multiplying the numerator and denominator by $\cos \phi$,

$$P_1 = W \times \frac{(\sin \alpha \cos \phi - \sin \phi \cos \alpha)}{(\cos \theta \cos \phi - \sin \phi \sin \theta)} = W \times \frac{\sin (\alpha - \phi)}{\cos (\theta + \phi)}$$

2. Maximum force (P_2) *which will keep the body in equilibrium, when it is at the point of sliding upwards.*

In this case, the force of friction ($F_2 = \mu\,R_2$) will act downwards as the body is at the point of sliding upwards as shown in Fig. 8.17 (*b*). Now resolving the forces along the plane.

$$P_2 \cos\theta = W\sin\alpha + \mu\,R_2 \qquad\qquad ...(iii)$$

and now resolving the forces perpendicular to the plane,

$$R_2 = W\cos\alpha - P_2\sin\theta \qquad\qquad ...(iv)$$

Substituting the value of R_2 in equation (*iii*),

$$P_2\cos\theta = W\sin\alpha + \mu\,(W\cos\alpha - P_2\sin\theta)$$
$$= W\sin\alpha + \mu\,W\cos\alpha - \mu\,P_2\sin\theta$$
$$P_2\cos\theta + \mu\,P_2\sin\theta = W\sin\alpha + \mu\,W\cos\alpha$$
$$P_2(\cos\theta + \mu\sin\theta) = W(\sin\alpha + \mu\cos\alpha)$$

$$\therefore \qquad P_2 = W \times \frac{(\sin\alpha + \mu\cos\alpha)}{(\cos\theta + \mu\sin\theta)}$$

and now substituting the vaue of $\mu = \tan\phi$ in the above equation,

$$P_2 = W \times \frac{(\sin\alpha + \tan\phi\cos\alpha)}{(\cos\theta + \tan\phi\sin\theta)}$$

Multiplying the numerator and denominator by $\cos\phi$,
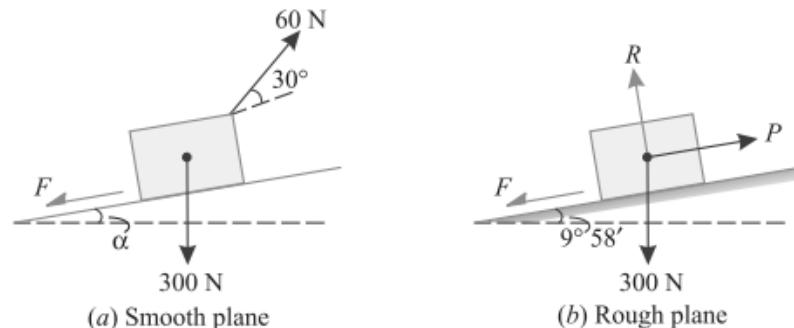
$$P_2 = W \times \frac{(\sin\alpha\cos\phi + \sin\phi\cos\alpha)}{(\cos\theta\cos\phi + \sin\phi\sin\theta)} = W \times \frac{\sin(\alpha+\phi)}{\cos(\theta-\phi)}$$

EXAMPLE: Find the force required to move a load of 300 N up a rough plane, the force being applied parallel to the plane. The inclination of the plane is such that when the same load is kept on a perfectly smooth plane inclined at the same angle, a force of 60 N applied at an inclination of 30° to the plane, keeps the same load in equilibrium. Assume coefficient of friction between the rough plane and the load to be equal to 0.3.

Solution. Given: Load (W) = 300 N; Force (P_1) = 60 N and angle at which force is inclined (θ) = 30°.

Let α = Angle of inclination of the plane.

First of all, consider the load lying on a smooth plane inclined at an angle (α) with the horizontal and subjected to a force of 60 N acting at an angle 30° with the plane as shown in Fig. 8.18 (*a*).

(*a*) Smooth plane   (*b*) Rough plane

We know that in this case, because of the smooth plane $\mu = 0$ or $\phi = 0$. We also know that the force required, when the load is at the point of sliding upwards (P),

$$60 = W \times \frac{\sin(\alpha + \phi)}{\cos(\theta - \phi)} = 300 \times \frac{\sin \alpha}{\cos 30°} = 300 \times \frac{\sin \alpha}{0.866} = 346.4 \sin \alpha$$

$$...(\because \phi = 0)$$

or $\quad\quad\quad \sin \alpha = \dfrac{60}{346.4} = 0.1732 \quad\quad$ or $\quad\quad \alpha = 10°$

Now consider the load lying on the rough plane inclined at an angle of 10° with the horizontal as shown in Fig. 8.18. (b). We know that in this case, $\mu = 0.3 = \tan \phi$ or $\phi = 16.7°$.
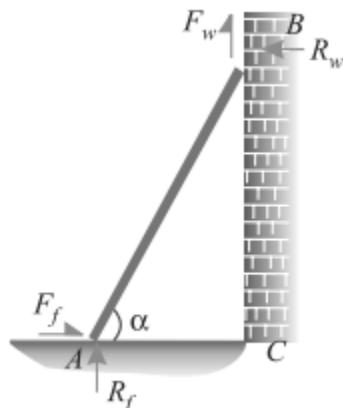
We also know that force required to move the load up the plane,

$$P = W \times \frac{\sin(\alpha + \phi)}{\cos \phi} = 300 \times \frac{\sin(10° + 16.7°)}{\cos 16.7°} \text{ N}$$

$$= 300 \times \frac{\sin 26.7°}{\cos 16.7°} = 300 \times \frac{0.4493}{0.9578} = 140.7 \text{ N} \quad\quad \textbf{Ans.}$$

APPLICATIONS OF FRICTION

LADDER FRICTION: The ladder is a device for climbing or scaling on the roofs or walls. It consists of two long uprights of wood, iron or rope connected by a number of cross pieces called rungs. These runing serve as steps.

Consider a ladder *AB* resting on the rough ground and leaning against a wall, as shown in Fig.

As the upper end of the ladder tends to slip downwards, therefore the direction of the force of friction between the ladder and the wall (*Fw*) will be upwards as shown in the figure. Similarly, as the lower end of the ladder tends to slip away from the wall, therefore the direction of the force of friction between the ladder and the floor (*Ff*) will be towards the wall as shown in the figure.

Since the system is in equilibrium, therefore the algebraic sum of the horizontal and vertical components of the forces must also be equal to zero.

Note: The normal reaction at the floor (*Rf*) will act perpendicular of the floor. Similarly, normal reaction of the wall (*Rw*) will also act perpendicular to the wall.

EXAMPLE: A ladder 5 meters long rests on a horizontal ground and leans against a smooth vertical wall at an angle 70° with the horizontal. The weight of the ladder is 900 N and acts at its middle. The ladder is at the point of sliding, when a man weighing 750N stands on a rung 1.5 metre from the bottom of the ladder. Calculate the coefficient of friction between the ladder and the floor.

Solution. Given: Length of the ladder (l) = 5 m; Angle which the ladder makes with the horizontal (α) = 70°; Weight of the ladder (w_1) = 900 N; Weight of man (w_2) = 750 N and distance between the man and bottom of ladder = 1.5 m.

Forces acting on the ladder are shown in Fig. 9.3.

Let μ_f = Coefficient of friction between ladder and floor and

R_f = Normal reaction at the floor.

Resolving the forces vertically,

$$R_f = 900 + 750 = 1650 \text{ N} \qquad ...(i)$$

∴ Force of friction at A

$$F_f = \mu_f \times R_f = \mu_f \times 1650 \qquad ...(ii)$$

Fig. 9.3.

Now taking moments about B, and equating the same,

$$R_f \times 5 \sin 20° = (F_f \times 5 \cos 20°) + (900 \times 2.5 \sin 20°)$$
$$+ (750 \times 3.5 \sin 20°)$$
$$= (F_f \times 5 \cos 20°) + (4875 \sin 20°)$$
$$= (\mu_f \times 1650 \times 5 \cos 20°) + 4875 \sin 20°$$

and now substituting the values of R_f and F_f from equations (i) and (ii)

$$1650 \times 5 \sin 20° = (\mu_f \times 1650 \times 5 \cos 20°) + (4875 \sin 20°)$$

Dividing both sides by 5 sin 20°,

$$1650 = (\mu_f \times 1650 \cot 20°) + 975$$
$$= (\mu_f \times 1650 \times 2.7475) + 975 = 4533 \,\mu_f + 975$$

∴ $$\mu_f = \frac{1650 - 975}{4533} = 0.15 \qquad \textbf{Ans.}$$
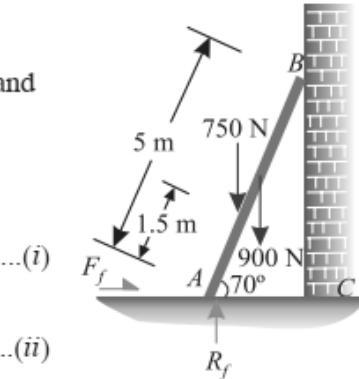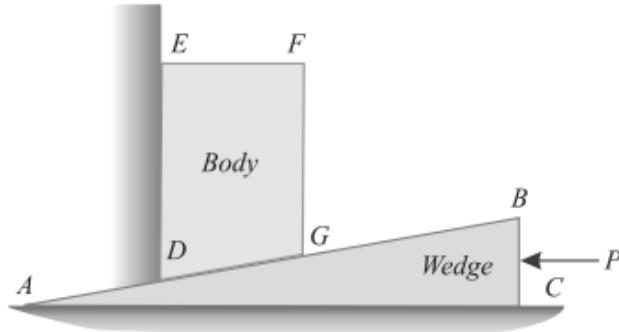
WEDGE FRICTION: A wedge is, usually, of a triangular or trapezoidal in cross-section. It is, generally, used for slight adjustements in the position of a body *i.e.* for tightening fits or keys for shafts. Sometimes, a wedge is also used for lifting heavy weights as shown in Fig.

It will be interesting to know that the problems on wedges are basically the problems of equilibrium on inclined planes. Thus these problems may be solved either by the equilibrium method or by applying Lami's theorem. Now consider a wedge *ABC*, which is used to lift the body *DEFG*.
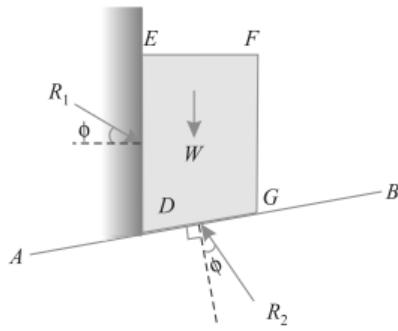
Let W = Weight fo the body *DEFG*,

P = Force required to lift the body, and
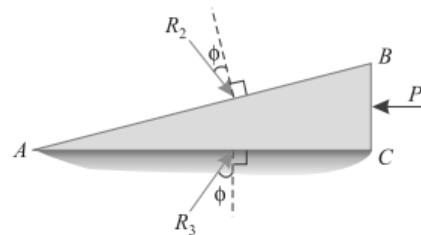
μ = Coefficient of friction on the planes *AB*, *AC* and *DE* such that

tan ϕ = μ.

A little consideration will show that when the force is sufficient to lift the body, the sliding will take place along three planes *AB*, *AC* and *DE* will also occur as shown in Fig.(*a*) and (*b*).

(*a*) Forces on the body *DEFG*   (*a*) Forces on the wedge *ABC*

The three reactions and the horizontal force (*P*) may now be found out by analytical method as discussed below:

Analytical method:
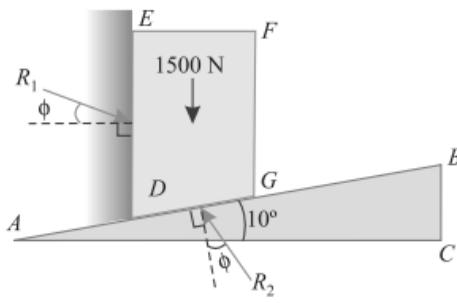
1. First of all, consider the equilibrium of the body DEFG. And resolve the forces W, R1 and R2 horizontally as well as vertically.

2. Now consider the equilibrium of the wedge ABC. And resolve the forces P, R2 and R3 horizontally as well as vertically.
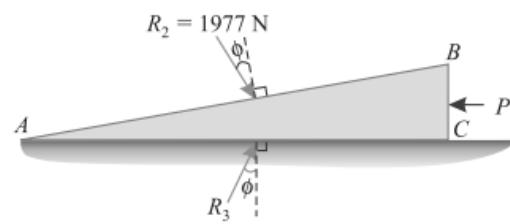
EXAMPLE: A block weighing 1500 N, overlying a 10° wedge on a horizontal floor and leaning against a vertical wall, is to be raised by applying a horizontal force to the wedge. Assuming the coefficient of friction between all the surface in contact to be 0.3, determine the minimum horizontal force required to raise the block.

SOLUTION: Given: Weight of the block (*W*) = 1500 N; Angle of the wedge (α) = 10° and coefficient of friction between all the four surfaces of contact (μ) = 0.3 = tan ϕ or ϕ = 16.7°

Let *P* = Minimum horizontal force required to raise the block.

(*a*) Block *DEFG*   (*b*) Wedge *ABC*

First of all, consider the equilibrium of the block. We know that it is in equilibrium under the action of the following forces as shown in Fig. (*a*).

1. Its own weight 1500 N acting downwards.
2. Reaction $R1$ on the face *DE*.
3. Reaction $R2$ on the face *DG* of the block.

Resolving the forces horizontally,

$$R_1 \cos (16.7°) = R_2 \sin (10 + 16.7°) = R_2 \sin 26.7°$$

$$R_1 \times 0.9578 = R_2 \times 0.4493$$

or $\qquad R_2 = 2.132 \ R_1$

and now resolving the forces vertically,

$$R_1 \times \sin (16.7°) + 1500 = R_2 \cos (10° + 16.7°) = R_2 \cos 26.7°$$

$$R_1 \times 0.2874 + 1500 = R_2 \times 0.8934 = (2.132 \ R_1)0.8934$$

$$= 1.905 \ R_1 \qquad\qquad ...(R_2 = 2.132 \ R_1)$$

$$R_1(1.905 - 0.2874) = 1500$$

∴ $\qquad R_1 = \dfrac{1500}{1.6176} = 927.3 \text{ N}$

and $\qquad R_2 = 2.132 \ R_1 = 2.132 \times 927.3 = 1977 \text{ N}$

Now consider the equilibrium of the wedge. We know that it is in equilibrium under the action of the following forces as shown in Fig. 9.14 (*b*).

1. Reaction R_2 of the block on the wedge.
2. Force (*P*) acting horizontally, and
3. Reaction R_3 on the face *AC* of the wedge.

Resolving the forces vertically,

$$R_3 \cos 16.7° = R_2 \cos (10° + 16.7°) = R_2 \cos 26.7°$$

$$R_3 \times 0.9578 = R_2 \times 0.8934 = 1977 \times 0.8934 = 1766.2$$

∴ $\qquad R_3 = \dfrac{1766.2}{0.9578} = 1844 \text{ N}$

and now resolving the forces horizontally,

$$P = R_2 \sin (10° + 16.7°) + R_3 \sin 16.7° = 1977 \sin 26.7° + 1844 \sin 16.7° \text{ N}$$

$$= (1977 \times 0.4493) + (1844 \times 0.2874) = 1418.3 \text{ N} \qquad \textbf{Ans.}$$

4. CENTROID AND MOMENT OF INERTIA
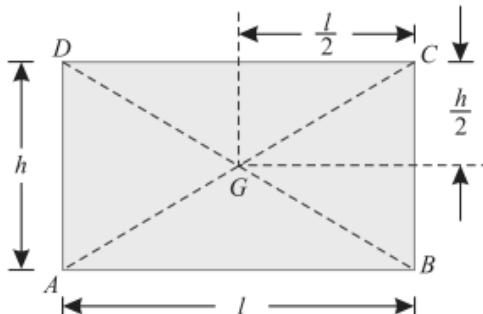
CENTRE OF GRAVITY: The point, through which the whole weight of the body acts, irrespective of its position, is known as centre of gravity (briefly written as C.G.). It may be noted that everybody has one and only one centre of gravity.

CENTROID: The plane figures (like triangle, quadrilateral, circle etc.) have only areas, but no mass. The centre of area of such figures is known as centroid. The method of finding out the centroid of a figure is the same as that of finding out the centre of gravity of a body.
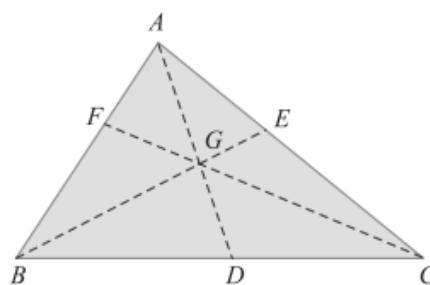
CENTRE OF GRAVITY BY GEOMETRICAL CONSIDERATIONS:
The centre of gravity of simple figures may be found out from the geometry of the figure as given below.
1. The centre of gravity of uniform rod is at its middle point.
2. The centre of gravity of a rectangle (or a parallelogram) is at the point, where its diagonals meet each other. It is also a middle point of the length as well as the breadth of the rectangle as shown in Fig.
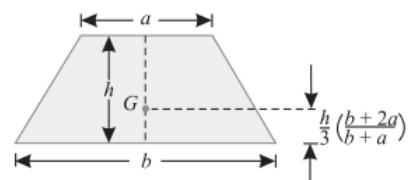
3. The centre of gravity of a triangle is at the point, where the three medians (a median is a line connecting the vertex and middle point of the opposite side) of the triangle meet as shown in Fig.
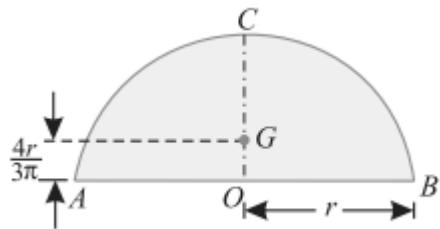
4. The centre of gravity of a trapezium with parallel sides a and b is at a distance of

$$\frac{h}{3} \times \left(\frac{b + 2a}{b + a} \right)$$
measured form the side b as shown in Fig.

5. The centre of gravity of a semicircle is at a distance of 4r/3 π from its base measured along the vertical radius as shown in Fig.
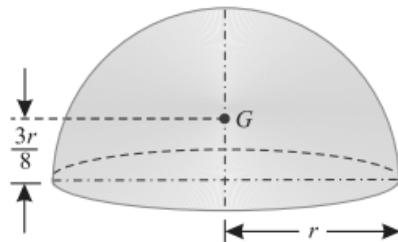
6. The centre of gravity of a circular sector making semi-vertical angle α is at a distance of

$$\frac{2r}{3} \frac{\sin \alpha}{\alpha}$$
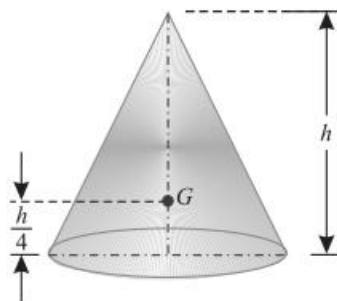
7. The centre of gravity of a cube is at a distance of $l/2$ from every face (where l is the length of each side).

8. The centre of gravity of a sphere is at a distance of $d/2$ from every point (where d is the diameter of the sphere).

9. The centre of gravity of a hemisphere is at a distance of $3r/8$ from its base, measured along the vertical radius as shown in Fig.

10. The centre of gravity of right circular solid cone is at a distance of $h/4$ from its base, measured along the vertical axis as shown in Fig.

AXIS OF REFERENCE:

The centre of gravity of a body is always calculated with reference to some assumed axis known as axis of reference (or sometimes with reference to some point of reference). The axis of reference, of plane figures, is generally taken as the lowest line of the figure for calculating $\bar{y}$ and the left line of the figure for calculating $\bar{x}$.

CENTRE OF GRAVITY OF PLANE FIGURES:

Let $\bar{x}$ and $\bar{y}$ be the co-ordinates of the centre of gravity with respect to some axis of reference, then

$$\bar{x} = \frac{a_1 x_1 + a_2 x_2 + a_3 x_3 + \text{........}}{a_1 + a_2 + a_3}$$

and

$$\bar{y} = \frac{a_1 y_1 + a_2 y_2 + a_3 y_3 + \text{........}}{a_1 + a_2 + a_3 + ...}$$

where a_1, a_2, a_3........ etc., are the areas into which the whole figure is divided x_1, x_2, x_3 ..... etc., are the respective co-ordinates of the areas a_1, a_2, a_3....... on X-X axis with respect to same axis of reference.

y_1, y_2, y_3....... etc., are the respective co-ordinates of the areas a_1, a_2, a_3....... on Y-Y axis with respect to same axis of the reference.

CENTRE OF GRAVITY OF SYMMETRICAL SECTIONS:

Sometimes, the given section, whose centre of gravity is required to be found out, is symmetrical about X-X axis or Y-Y axis. In such cases, the procedure for calculating the centre of gravity of the body is very much simplified; as we have only to calculate either $\bar{x}$ or $\bar{y}$. This is due to the reason that the centre of gravity of the body will lie on the axis of symmetry.

EXAMPLE: Find the centre of gravity of a 100 mm × 150 mm × 30 mm T-section.

Solution. As the section is symmetrical about Y-Y axis, bisecting the web, therefore its centre of gravity will lie on this axis. Split up the section into two rectangles ABCH and DEFG as shown in Fig 6.10.

Let bottom of the web FE be the axis of reference.

(i) Rectangle ABCH

$$a_1 = 100 \times 30 = 3000 \text{ mm}^2$$

and

$$y_1 = \left(150 - \frac{30}{2}\right) = 135 \text{ mm}$$

(ii) Rectangle DEFG
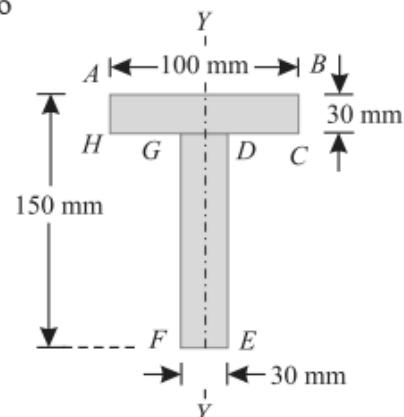
$$a_2 = 120 \times 30 = 3600 \text{ mm}^2$$

and

$$y_2 = \frac{120}{2} = 60 \text{ mm}$$

We know that distance between centre of gravity of the section and bottom of the flange FE,

$$\bar{y} = \frac{a_1 y_1 + a_2 y_2}{a_1 + a_2} = \frac{(3000 \times 135) + (3600 \times 60)}{3000 + 3600} \text{ mm}$$

$$= 94.1 \text{ mm} \quad \textbf{Ans.}$$

EXAMPLE: An I-section has the following dimensions in mm units:
Bottom flange = 300 × 100
Top flange = 150 × 50
Web = 300 × 50
Determine mathematically the position of centre of gravity of the section.

Solution. As the section is symmetrical about Y-Y axis, bisecting the web, therefore its centre of gravity will lie on this axis. Now split up the section into three rectangles as shown in Fig.

Let bottom of the bottom flange be the axis of reference.

(i) *Bottom flange*

$$a_1 = 300 \times 100 = 30\ 000 \text{ mm}^2$$

and

$$y_1 = \frac{100}{2} = 50 \text{ mm}$$

(ii) *Web*

$$a_2 = 300 \times 50 = 15\ 000 \text{ mm}^2$$

and

$$y_2 = 100 + \frac{300}{2} = 250 \text{ mm}$$

(iii) *Top flange*
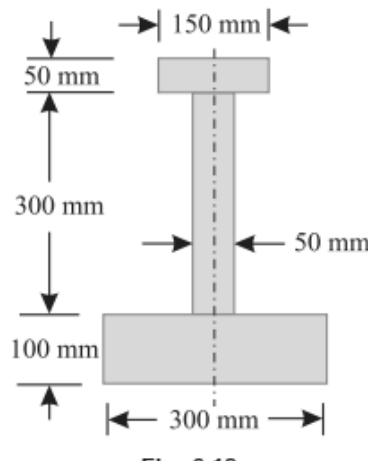
$$a_3 = 150 \times 50 = 7500 \text{ mm}^2$$

and

$$y_3 = 100 + 300 + \frac{50}{2} = 425 \text{ mm}$$

We know that distance between centre of gravity of the section and bottom of the flange,

$$\bar{y} = \frac{a_1 y_1 + a_2 y_2 + a_3 y_3}{a_1 + a_2 + a_3}$$

$$= \frac{(30\ 000 \times 50) + (15\ 000 \times 250) + (7500 \times 425)}{30\ 000 + 15\ 000 + 7500} = 160.7 \text{ mm} \qquad \textbf{Ans.}$$

CENTRE OF GRAVITY OF UNSYMMETRICAL SECTIONS:

Sometimes, the given section, whose centre of gravity is required to be found out, is not symmetrical either about X-X axis or Y-Y axis. In such cases, we have to find out both the values of $\bar{x}$ and $\bar{y}$

EXAMPLE: Find the centroid of an unequal angle section 100 mm × 80 mm × 20 mm.

Solution. As the section is not symmetrical about any axis, therefore we have to find out the values of $\bar{x}$ and $\bar{y}$ for the angle section. Split up the section into two rectangles as shown in Fig.

Let left face of the vertical section and bottom face of the horizontal section be axes of reference.

Let left face of the vertical section and bottom face of the horizontal section be axes of reference.

(i) Rectangle 1

$$a_1 = 100 \times 20 = 2000 \text{ mm}^2$$

$$x_1 = \frac{20}{2} = 10 \text{ mm}$$

and

$$y_1 = \frac{100}{2} = 50 \text{ mm}$$

(ii) Rectangle 2

$$a_2 = (80 - 20) \times 20 = 1200 \text{ mm}^2$$

$$x_2 = 20 + \frac{60}{2} = 50 \text{ mm}$$

and

$$y_2 = \frac{20}{2} = 10 \text{ mm}$$

We know that distance between centre of gravity of the section and left face,

$$\bar{x} = \frac{a_1 x_1 + a_2 x_2}{a_1 + a_2} = \frac{(2000 \times 10) + (1200 \times 50)}{2000 + 1200} = 25 \text{ mm} \qquad \textbf{Ans.}$$

Similarly, distance between centre of gravity of the section and bottom face,

$$\bar{y} = \frac{a_1 y_1 + a_2 y_2}{a_1 + a_2} = \frac{(2000 \times 50) + (1200 \times 10)}{2000 + 1200} = 35 \text{ mm} \qquad \textbf{Ans.}$$

EXAMPLE: A body consists of a right circular solid cone of height 40 mm and radius 30 mm placed on a solid hemisphere of radius 30 mm of the same material. Find the position of centre of gravity of the body.

Solution. As the body is symmetrical about Y-Y axis, therefore its centre of gravity will lie on this axis as shown in Fig.   Let bottom of the hemisphere (D) be the point of reference.

(i) Hemisphere

$$v_1 = \frac{2\pi}{3} \times r^3 = \frac{2\pi}{3}(30)^3 \text{ mm}^3$$

$$= 18\,000\,\pi \text{ mm}^3$$

and

$$y_1 = \frac{5r}{8} = \frac{5 \times 30}{8} = 18.75 \text{ mm}$$

(ii) Right circular cone
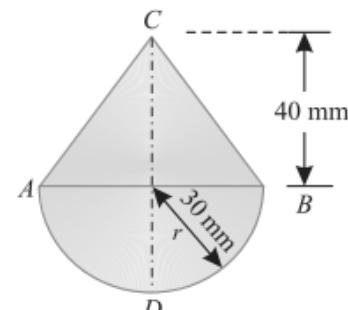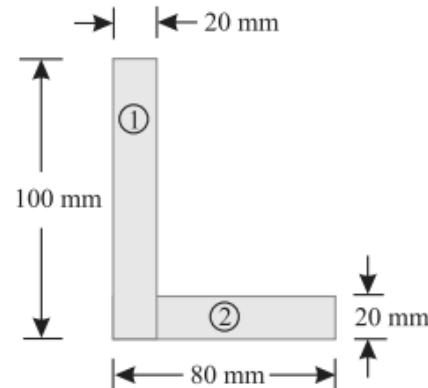
$$v_2 = \frac{\pi}{3} \times r^2 \times h = \frac{\pi}{3} \times (30)^2 \times 40 \text{ mm}^3$$

$$= 12\,000\,\pi \text{ mm}^3$$

and

$$y_2 = 30 + \frac{40}{4} = 40 \text{ mm}$$

We know that distance between centre of gravity of the body and bottom of hemisphere D,

$$\bar{y} = \frac{v_1 y_1 + v_2 y_2}{v_1 + v_2} = \frac{(18\,000\pi \times 18.75) + (12\,000\pi \times 40)}{18\,000\pi + 12\,000\pi} \text{ mm}$$

$$= 27.3 \text{ mm} \qquad \textbf{Ans.}$$

MOMENT OF INERTIA: The moment of a force (P) about a point, is the product of the force and perpendicular distance (x) between the point and the line of action of the force (*i.e.* $P.x$). This moment is also called first moment of force. If this moment is again multiplied by the perpendicular distance (x) between the point and the line of action of the force *i.e.* $P.x$ (x) = $Px2$, then this quantity is called moment of the moment of a force or second moment of force or moment of inertia (briefly written as M.I.).

MOMENT OF INERTIA OF A PLANE AREA:

Consider a plane area, whose moment of inertia is required to be found out. Split up the whole area into a number of small elements.

Let $a_1, a_2, a_3, ...$ = Areas of small elements, and

$r_1, r_2, r_3, ...$ = Corresponding distances of the elements from the line about which the moment of inertia is required to be found out.

Now the moment of inertia of the area,

$$I = a_1 r_1^2 + a_2 r_2^2 + a_3 r_3^2 + ...$$
$$= \Sigma\, a\, r^2$$

UNITS OF MOMENT OF INERTIA:

As a matter of fact the units of moment of inertia of a plane area depend upon the units of the area and the length. e.g.

1. If area is in m^2 and the length is also in m, the moment of inertia is expressed in m^4
2. If area in mm^2 and the length is also in mm, then moment of inertia is expressed in mm^4.

MOMENT OF INERTIA BY INTEGRATION:
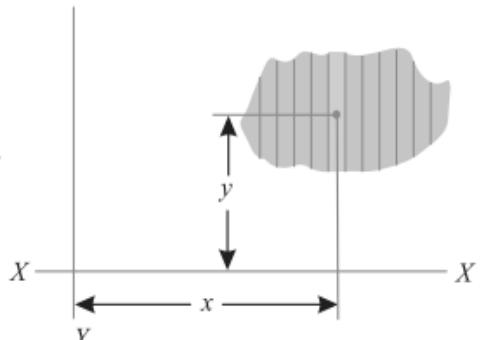
The moment of inertia of an area may also be found out by the method of integration as discussed below:

Consider a plane figure, whose moment of inertia is required to be found out about *X-X* axis and *Y-Y* axis as shown in Fig   Let us divide the whole area into a no. of strips. Consider one of these strips.

Let dA = Area of the strip

x = Distance of the centre of gravity of the strip on *X-X* axis and

y = Distance of the centre of gravity of the strip on *Y-Y* axis.

. Moment of inertia by integration.

We know that the moment of inertia of the strip about *Y-Y* axis

$$= dA \cdot x^2$$

Now the moment of inertia of the whole area may be found out by integrating above equation. *i.e.*,

$$I_{YY} = \Sigma\, dA \cdot x^2$$

Similarly $I_{XX} = \Sigma\, dA \cdot y^2$

MOMENT OF INERTIA OF A RECTANGULAR SECTION:

Consider a rectangular section $ABCD$ as shown in Fig. whose moment of inertia is required to be found out.

Let $\qquad b$ = Width of the section and

$\qquad\qquad d$ = Depth of the section.

Now consider a strip PQ of thickness dy parallel to X-X axis and at a distance y from it as shown in the figure

∴ Area of the strip

$$= b.dy$$
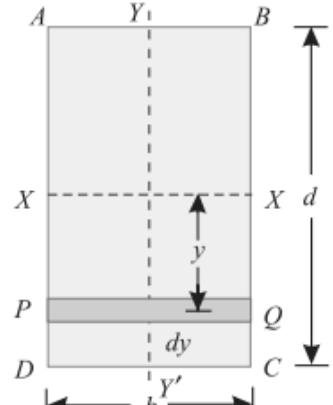
We know that moment of inertia of the strip about X-X axis,

$$= \text{Area} \times y^2 = (b.\ dy)\ y^2 = b.\ y^2.\ dy$$

Now *moment of inertia of the whole section may be found out by integrating the above equation for the whole length of the lamina $i.e.$ from $-\dfrac{d}{2}$ to $+\dfrac{d}{2}$,

$$I_{xx} = \int_{-\frac{d}{2}}^{+\frac{d}{2}} b.y^2.dy = b \int_{-\frac{d}{2}}^{+\frac{d}{2}} y^2.dy$$

$$= b\left[\frac{y^3}{3}\right]_{-\frac{d}{2}}^{+\frac{d}{2}} = b\left[\frac{(d/2)^3}{3} - \frac{(-d/2)^3}{3}\right] = \frac{bd^3}{12}$$

Similarly, $\qquad I_{YY} = \dfrac{db^3}{12}$

Rectangular section.

MOMENT OF INERTIA OF A HOLLOW RECTANGULAR SECTION:

Consider a hollow rectangular section, in which $ABCD$ is the main section and $EFGH$ is the cut out section as shown in Fig

Let $\qquad\qquad b$ = Breadth of the outer rectangle,

$\qquad\qquad d$ = Depth of the outer rectangle and

$\qquad b_1, d_1$ = Corresponding values for the cut out rectangle.

We know that the moment of inertia, of the outer rectangle $ABCD$ about X-X axis

$$= \frac{bd^3}{12} \qquad\qquad ...(i)$$

and moment of inertia of the cut out rectangle $EFGH$ about X-X axis
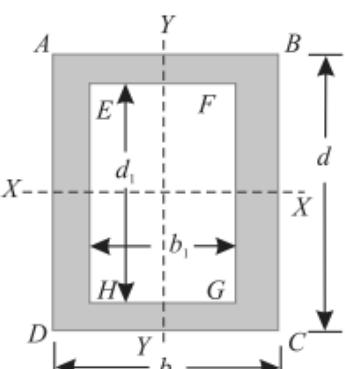
$$= \frac{b_1 d_1^3}{12} \qquad\qquad ...(ii)$$

∴ M.I. of the hollow rectangular section about X-X axis,

$$I_{XX} = \text{M.I. of rectangle } ABCD - \text{M.I. of rectangle } EFGH$$

$$= \frac{bd^3}{12} - \frac{b_1 d_1^3}{12}$$

Similarly, $\qquad I_{yy} = \dfrac{db^3}{12} - \dfrac{d_1 b_1^3}{12}$

Hollow rectangular section.

THEOREM OF PERPENDICULAR AXIS:

It states, *If I_{XX} and I_{YY} be the moments of inertia of a plane section about two perpendicular axis meeting at O, the moment of inertia I_{ZZ} about the axis Z-Z, perpendicular to the plane and passing through the intersection of X-X and Y-Y is given by:*

$$I_{ZZ} = I_{XX} + I_{YY}$$

Proof :

Consider a small lamina (P) of area da having co-ordinates as x and y along OX and OY two mutually perpendicular axes on a plane section as shown in Fig.

Now consider a plane OZ perpendicular to OX and OY. Let (r) be the distance of the lamina (P) from Z-Z axis such that $OP = r$.

From the geometry of the figure, we find that

$$r^2 = x^2 + y^2$$

We know that the moment of inertia of the lamina P about X-X axis,

$$I_{XX} = da. \, y^2 \qquad \qquad ...[\because I = \text{Area} \times (\text{Distance})^2]$$
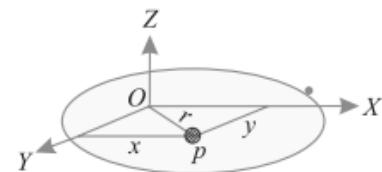
Similarly, $\qquad I_{YY} = da. \, x^2$

and $\qquad I_{ZZ} = da. \, r^2 = da \, (x^2 + y^2) \qquad ...(\because r^2 = x^2 + y^2)$

$$= da. \, x^2 + da. \, y^2 = I_{YY} + I_{XX}$$

Theorem of perpendicular axis.

MOMENT OF INERTIA OF A CIRCULAR SECTION:

Consider a circle $ABCD$ of radius (r) with centre O and X-X' and Y-Y' be two axes of reference through O as shown in Fig.

Now consider an elementary ring of radius x and thickness dx. Therefore area of the ring,
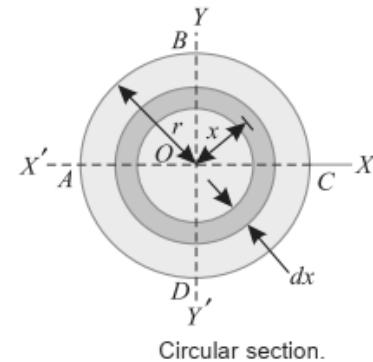
$$da = 2\pi x. \, dx$$

and moment of inertia of ring, about X-X axis or Y-Y axis

$$= \text{Area} \times (\text{Distance})^2$$
$$= 2\pi x. \, dx \times x^2$$
$$= 2\pi x^3. \, dx$$

Circular section.

Now moment of inertia of the whole section, about the central axis, can be found out by integrating the above equation for the whole radius of the circle *i.e.*, from 0 to r.

$$\therefore \qquad I_{ZZ} = \int_0^r 2\pi x^3 . \, dx = 2\pi \int_0^r x^3 . \, dx$$

$$I_{ZZ} = 2\pi \left[\frac{x^4}{4} \right]_0^r = \frac{\pi}{2} (r)^4 = \frac{\pi}{32} (d)^4 \qquad ...\left(\text{substituting } r = \frac{d}{2} \right)$$

We know from the Theorem of Perpendicular Axis that

$$I_{XX} + I_{YY} = I_{ZZ}$$

$$* \, I_{XX} = I_{YY} = \frac{I_{ZZ}}{2} = \frac{1}{2} \times \frac{\pi}{32} (d)^4 = \frac{\pi}{64} (d)^4$$

MOMENT OF INERTIA OF A HOLLOW CIRCULAR SECTION:

Consider a hollow circular section as shown in Fig. whose moment of inertia is required to be found out.

Let D = Diameter of the main circle, and

d = Diameter of the cut out circle.

We know that the moment of inertia of the main circle about X-X axis

$$= \frac{\pi}{64}(D)^4$$

and moment of inertia of the cut-out circle about X-X axis
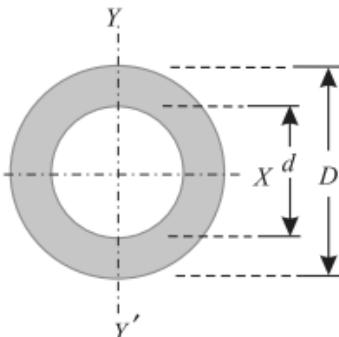
$$= \frac{\pi}{64}(d)^4$$

∴ Moment of inertia of the hollow circular section about X-X axis,

I_{XX} = Moment of inertia of main circle – Moment of inertia of cut out circle,

$$= \frac{\pi}{64}(D)^4 - \frac{\pi}{64}(d)^4 = \frac{\pi}{64}(D^4 - d^4)$$

Similarly, $$I_{YY} = \frac{\pi}{64}(D^4 - d^4)$$

Hollow circular section.

THEOREM OF PARALLEL AXIS:

It states, *If the moment of inertia of a plane area about an axis through its centre of gravity is denoted by I_G, then moment of inertia of the area about any other axis AB, parallel to the first, and at a distance h from the centre of gravity is given by*:

$$I_{AB} = I_G + ah^2$$

where I_{AB} = Moment of inertia of the area about an axis AB,

I_G = Moment of Inertia of the area about its centre of gravity

a = Area of the section, and

h = Distance between centre of gravity of the section and axis AB.

Proof

Consider a strip of a circle, whose moment of inertia is required to be found out about a line AB as shown in Fig.
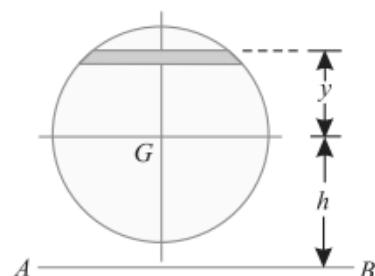
Let δa = Area of the strip

y = Distance of the strip from the centre of gravity the section and

h = Distance between centre of gravity of the section and the axis AB.

We know that moment of inertia of the whole section about an axis passing through the centre of gravity of the section

$$= \delta a.\, y^2$$

Theorem of parallel axis.

and moment of inertia of the whole section about an axis passing through its centre of gravity,

$$I_G = \Sigma\, \delta a.\, y^2$$

∴   Moment of inertia of the section about the axis *AB*,

$$I_{AB} = \Sigma\, \delta a\, (h + y)^2 = \Sigma\, \delta a\, (h^2 + y^2 + 2\, h\, y)$$
$$= (\Sigma\, h^2.\, \delta a) + (\Sigma\, y^2.\, \delta a) + (\Sigma\, 2\, h\, y.\, \delta a)$$
$$= a\, h^2 +\ I_G + 0$$

It may be noted that $\Sigma\, h^2.\, \delta a = a\, h^2$ and $\Sigma\, y^2.\, \delta a = I_G$ [as per equation (*i*) above] and $\Sigma\, \delta a.y$ is the algebraic sum of moments of all the areas, about an axis through centre of gravity of the section and is equal to $a.\bar{y}$, where $\bar{y}$ is the distance between the section and the axis passing through the centre of gravity, which obviously is zero.

MOMENT OF INERTIA OF A TRIANGULAR SECTION:

Consider a triangular section *ABC* whose moment of inertia is required to be found out.

Let   *b* = Base of the triangular section and

h = Height of the triangular section.

Now consider a small strip *PQ* of thickness *dx* at a distance of *x* from the vertex *A* as shown in Fig.   . From the geometry of the figure, we find that the two triangles *APQ* and *ABC* are similar. Therefore

. Triangular section.

$$\frac{PQ}{BC} = \frac{x}{h} \quad \text{or} \quad PQ = \frac{BC\,.x}{h} = \frac{bx}{h} \qquad (\because BC = \text{base} = b)$$

We know that area of the strip *PQ*
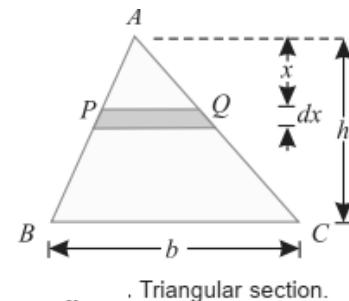
$$= \frac{bx}{h}\,.\,dx$$

and moment of inertia of the strip about the base *BC*

$$= \text{Area} \times (\text{Distance})^2\ = \frac{bx}{h}\,dx\,(h-x)^2 = \frac{bx}{h}\,(h-x)^2\,dx$$

Now moment of inertia of the whole triangular section may be found out by integrating the above equation for the whole height of the triangle *i.e.*, from 0 to *h*.

$$I_{BC} = \int_0^h \frac{b\,x}{h}\,(h-x)^2\,dx$$

$$= \frac{b}{h} \int_0^h x\,(h^2 + x^2 - 2hx)\,dx$$

$$= \frac{b}{h} \int_0^h (xh^2 + x^3 - 2hx^2)\,dx$$

$$= \frac{b}{h} \left[\frac{x^2 h^2}{2} + \frac{x^4}{4} - \frac{2hx^3}{3} \right]_0^h = \frac{bh^3}{12}$$

We know that distance between centre of gravity of the triangular section and base BC,

$$d = \frac{h}{3}$$

∴ Moment of inertia of the triangular section about an axis through its centre of gravity and parallel to X-X axis,

$$I_G = I_{BC} - ad^2 \qquad\qquad \dots(\because I_{XX} = I_G + a\,h^2)$$

$$= \frac{bh^3}{12} - \left(\frac{bh}{2}\right)\left(\frac{h}{3}\right)^2 = \frac{bh^3}{36}$$

MOMENT OF INERTIA OF A SEMICIRCULAR SECTION:

Consider a semicircular section ABC whose moment of inertia is required to be found out as shown in Fig. ___.

Let $\quad r$ = Radius of the semicircle.

We know that moment of inertia of the semicircular section about the base AC is equal to half the moment of inertia of the circular section about AC. Therefore moment of inertia of the semicircular section ABC about the base AC,

$$I_{AC} = \frac{1}{2} \times \frac{\pi}{64} \times (d)^4 = 0.393\,r^4$$

We also know that area of semicircular section,

$$a = \frac{1}{2} \times \pi r^2 \quad \frac{\pi r^2}{2}$$
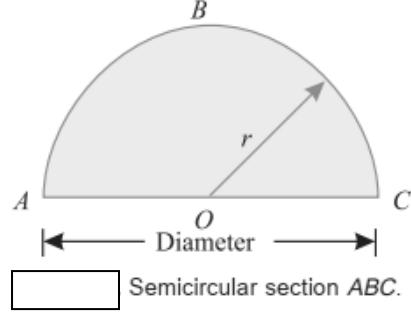
and distance between centre of gravity of the section and the base AC,

$$h = \frac{4r}{3\pi}$$

∴ Moment of inertia of the section through its centre of gravity and parallel to x-x axis,

$$I_G = I_{AC} - ah^2 = \left[\frac{\pi}{8} \times (r)^4 \right] - \left[\frac{\pi r^2}{2} \left(\frac{4r}{3\pi} \right)^2 \right]$$

$$= \left[\frac{\pi}{8} \times (r)^4 \right] - \left[\frac{8}{9\pi} \times (r)^4 \right] = 0.11\,r^4$$

Note. The moment of inertia about y-y axis will be the same as that about the base AC i.e., $0.393\,r^4$.

Semicircular section ABC.

MOMENT OF INERTIA OF A COMPOSITE SECTION:

The moment of inertia of a composite section may be found out by the following steps :
1. First of all, split up the given section into plane areas (*i.e.*, rectangular, triangular, circular etc., and find the centre of gravity of the section).
2. Find the moments of inertia of these areas about their respective centres of gravity.
3. Now transfer these moment of inertia about the required axis (*AB*) by the Theorem of Parallel Axis, *i.e.*,

$$I_{AB} = I_G + ah^2$$

where I_G = Moment of inertia of a section about its centre of gravity and parallel to the axis.
    a = Area of the section,
    h = Distance between the required axis and centre of gravity of the section.
4. The moments of inertia of the given section may now be obtained by the algebraic sum of the moment of inertia about the required axis.

EXAMPLE: Find the moment of inertia of a T-section with flange as 150 mm × 50 mm and web as 150 mm × 50 mm about X-X and Y-Y axes through the centre of gravity of the section.

Solution. The given *T*-section is shown in Fig.

First of all, let us find out centre of gravity of the section. As the section is symmetrical about *Y-Y* axis, therefore its centre of gravity will lie on this axis. Split up the whole section into two rectangles *viz.*, 1 and 2 as shown in figure. Let bottom of the web be the axis of reference.

(*i*) *Rectangle* (1)

$$a_1 = 150 \times 50 = 7500 \text{ mm}^2$$

and $$y_1 = 150 + \frac{50}{2} = 175 \text{ mm}$$

(*ii*) *Rectangle* (2)

$$a_2 = 150 \times 50 = 7500 \text{ mm}^2$$

and $$y_2 = \frac{150}{2} = 75 \text{ mm}$$

We know that distance between centre of gravity of the section and bottom of the web,

$$\bar{y} = \frac{a_1 y_1 + a_2 y_2}{a_1 + a_2} = \frac{(7500 \times 175) + (7500 \times 75)}{7500 + 7500} = 125 \text{ mm}$$
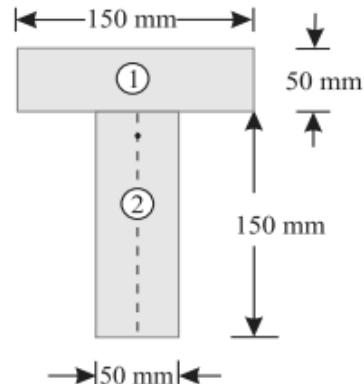
Moment of inertia about X-X axis

We also know that M.I. of rectangle (1) about an axis through its centre of gravity and parallel to *X-X* axis.

$$I_{G1} = \frac{150 (50)^3}{12} = 1.5625 \times 10^6 \text{ mm}^4$$

and distance between centre of gravity of rectangle (1) and *X-X* axis,

$$h_1 = 175 - 125 = 50 \text{ mm}$$

∴   Moment of inertia of rectangle (1) about X-X axis

$$I_{G1} + a_1 h_1^2 = (1.5625 \times 10^6) + [7500 \times (50)^2] = 20.3125 \times 10^6 \ mm^4$$

Similarly, moment of inertia of rectangle (2) about an axis through its centre of gravity and parallel to X-X axis,

$$I_{G2} = \frac{50 \ (150)^3}{12} = 14.0625 \times 10^6 \ mm^4$$

and distance between centre of gravity of rectangle (2) and X-X axis,

$$h_2 = 125 - 75 = 50 \ mm$$

∴   Moment of inertia of rectangle (2) about X-X axis

$$= I_{G2} + a_2 h_2^2 = (14.0625 \times 10^6) + [7500 \times (50)^2] = 32.8125 \times 10^6 \ mm^4$$

Now moment of inertia of the whole section about X-X axis,

$$I_{XX} = (20.3125 \times 10^6) + (32.8125 \times 10^6) = 53.125 \times 10^6 \ mm^4 \qquad \textbf{Ans.}$$

Moment of inertia about Y-Y axis

We know that M.I. of rectangle (1) about Y-Y axis

$$= \frac{50 \ (150)^3}{12} = 14.0625 \times 10^6 \ mm^4$$

and moment of inertia of rectangle (2) about Y-Y axis,

$$= \frac{150 \ (50)^3}{12} = 1.5625 \times 10^6 \ mm^4$$

Now moment of inertia of the whole section about Y-Y axis,

$$I_{YY} = (14.0625 \times 10^6) + (1.5625 \times 10^6) = 15.625 \times 10^6 \ mm^4 \qquad \textbf{Ans.}$$

EXAMPLE: An I-section is made up of three rectangles as shown in Fig. Find the moment of inertia of the section about the horizontal axis passing through the centre of gravity of the section.

Solution. First of all, let us find out centre of gravity of the section. As the section is symmetrical about Y-Y axis, therefore its centre of gravity will lie on this axis. Split up the whole section into three rectangles 1, 2 and 3 as shown in Fig.   . Let bottom face of the bottom flange be the axis of reference.

(i) *Rectangle 1*

$$a_1 = 60 \times 20 = 1200 \ mm$$

and $$y_1 = 20 + 100 + \frac{20}{2} = 130 \ mm$$

(ii) *Rectangle 2*

$$a_2 = 100 \times 20 = 2000 \ mm^2$$

and $$y_2 = 20 + \frac{100}{2} = 70 \ mm$$

(iii) *Rectangle 3*
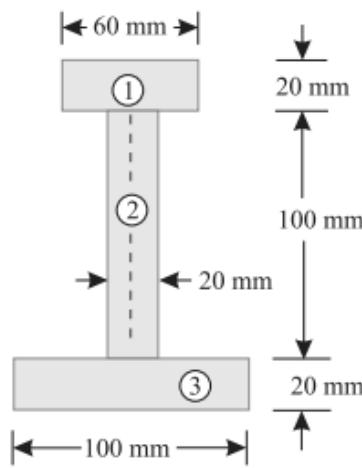
$$a_3 = 100 \times 20 = 2000 \ mm^2$$

and $$y_3 = \frac{20}{2} = 10 \ mm$$

We know that the distance between centre of gravity of the section and bottom face,

$$\bar{y} = \frac{a_1 y_1 + a_2 y_2 + a_3 y_3}{a_1 + a_2 + a_3} = \frac{(1200 \times 130) + (2000 \times 70) + (2000 \times 10)}{1200 + 2000 + 2000} \ mm$$

$$= 60.8 \ mm$$

We know that moment of inertia of rectangle (1) about an axis through its centre of gravity and parallel to X-X axis,

$$I_{G1} = \frac{60 \times (20)^3}{12} = 40 \times 10^3 \text{ mm}^4$$

and distance between centre of gravity of rectangle (1) and X-X axis,

$$h_1 = 130 - 60.8 = 69.2 \text{ mm}$$

∴ Moment of inertia of rectangle (1) about X-X axis,

$$= I_{G1} + a_1 h_1^2 = (40 \times 10^3) + [1200 \times (69.2)^2] = 5786 \times 10^3 \text{ mm}^4$$

Similarly, moment of inertia of rectangle (2) about an axis through its centre of gravity and parallel to X-X axis,

$$I_{G2} = \frac{20 \times (100)^3}{12} = 1666.7 \times 10^3 \text{ mm}^4$$

and distance between centre of gravity of rectangle (2) and X-X axis,

$$h_2 = 70 - 60.8 = 9.2 \text{ mm}$$

∴ Moment of inertia of rectangle (2) about X-X axis,

$$= I_{G2} + a_2 h_2^2 = (1666.7 \times 10^3) + [2000 \times (9.2)^2] = 1836 \times 10^3 \text{ mm}^4$$

Now moment of inertia of rectangle (3) about an axis through its centre of gravity and parallel to X-X axis,

$$I_{G3} = \frac{100 \times (20)^3}{12} = 66.7 \times 10^3 \text{ mm}^4$$

and distance between centre of gravity of rectangle (3) and X-X axis,

$$h_3 = 60.8 - 10 = 50.8 \text{ mm}$$

∴ Moment of inertia of rectangle (3) about X-X axis,

$$= I_{G3} + a_3 h_3^2 = (66.7 \times 10^3) + [2000 \times (50.8)^2] = 5228 \times 10^3 \text{ mm}^4$$

Now moment of inertia of the whole section about X-X axis,

$$I_{XX} = (5786 \times 10^3) + (1836 \times 10^3) + (5228 \times 10^3) = 12\,850 \times 10^3 \text{ mm}^4 \quad \textbf{Ans.}$$

5. SIMPLE MACHINES

SIMPLE MACHINE:
A simple machine may be defined as a device, which enables us to do some useful work at some point or to overcome some resistance, when an effort or force is applied to it, at some other convenient point.

COMPOUND MACHINE:
A compound machine may be defined as a device, consisting of a number of simple machines, which enables us to do some useful work at a faster speed or with a much less effort as compared to a simple machine.

LIFTING MACHINE:
It is a device, which enables us to lift a heavy load (W) by applying a comparatively smaller effort (P).

MECHANICAL ADVANTAGE:
The mechanical advantage (briefly written as M.A.) is the ratio of weight lifted (W) to the effort applied (P) and is always expressed in pure number. Mathematically, mechanical advantage, M.A. = *W/P*

INPUT OF A MACHINE:
The input of a machine is the work done on the machine. In a lifting machine, it is measured by the product of effort and the distance through which it has moved.

OUTPUT OF A MACHINE:
The output of a machine is the actual work done by the machine. In a lifting machine, it is measured by the product of the weight lifted and the distance through which it has been lifted.

EFFICIENCY OF A MACHINE:
It is the ratio of output to the input of a machine and is generally expressed as a percentage. Mathematically, efficiency

$$\eta = \frac{\text{Output}}{\text{Input}} \times 100$$

IDEAL MACHINE:
If the efficiency of a machine is 100% *i.e.*, if the output is equal to the input, the machine is called as a perfect or an *ideal machine.*

VELOCITY RATIO:

The velocity ratio (briefly written as V.R.) is the ratio of distance moved by the effort (y) to the distance moved by the load (x) and is always expressed in pure number. Mathematically, velocity ratio,

$$V.R. = \frac{y}{x}$$

RELATION BETWEEN EFFICIENCY, MECHANICAL ADVANTAGE AND VELOCITY RATIO OF A LIFTING MACHINE:

Let W = Load lifted by the machine,

P = Effort required to lift the load,

Y = Distance moved by the effort, in lifting the load, and

x = Distance moved by the load.

We know that $M.A. = \dfrac{W}{P} = W/P$ and $V.R. = \dfrac{y}{x} = y/x$

We also know that input of a machine

= Effort applied × Distance through which the effort has moved

= $P \times y$ ...(i)

and output of a machine = Load lifted × Distance through which the load has been lifted

= $W \times x$ ...(ii)

∴ Efficiency, $\eta = \dfrac{\text{Output}}{\text{Input}} = \dfrac{W \times x}{P \times y} = \dfrac{W/P}{y/x} = \dfrac{M.A.}{V.R.}$

Note. It may be seen from the above relation that the values of M.A. and V.R. are equal only in case of a machine whose efficiency is 100%. But in actual practice, it is not possible.

EXAMPLE: In a certain weight lifting machine, a weight of 1 kN is lifted by an effort of 25 N. While the weight moves up by 100 mm, the point of application of effort moves by 8 m. Find mechanical advantage, velocity ratio and efficiency of the machine.

Solution. Given: Weight (W) = 1 kN = 1000 N ; Effort (P) = 25 N ; Distance through which the weight is moved (x) = 100 mm = 0.1 m and distance through which effort is moved (y) = 8 m.

Mechanical advantage of the machine.

We know that mechanical advantage of the machine

$$M.A. = \frac{W}{P} = \frac{1000}{25} = 40 \quad \textbf{Ans.}$$

Velocity ratio of the machine

We know that velocity ratio of the machine

$$V.R. = \frac{y}{x} = \frac{8}{0.1} = 80 \quad \textbf{Ans.}$$

Efficiency of the machine

We also know that efficiency of the machine,

$$\eta = \frac{M.A.}{V.R.} = \frac{40}{80} = 0.5 = 50\% \quad \textbf{Ans.}$$

REVERSIBILITY OF A MACHINE:

Sometimes, a machine is also capable of doing some work in the reversed direction, after the effort is removed. Such a machine is called a reversible machine and its action is known as *reversibility of the machine.*

CONDITION FOR THE REVERSIBILITY OF A MACHINE:

Consider a reversible machine, whose condition for the reversibility is required to be found out.

Let W = Load lifted by the machine,
P = Effort required to lift the load,
y = Distance moved by the effort, and
x = Distance moved by the load

We know that input of the machine

$$= P \times y \qquad \qquad ...(i)$$

and output of the machine $= W \times x \qquad \qquad ...(ii)$

We also know that machine friction

$$= \text{Input} - \text{Output} = (P \times y) - (W \times x) \qquad ...(iii)$$

A little consideration will show that in a reversible machine, the *output of the machine should be more than the machine friction, when the effort (P) is zero. *i.e.,*

$$W \times x > P \times y - W \times x$$

or $\qquad 2\,W \times x > P \times y$

or $\qquad \dfrac{W \times x}{P \times y} > \dfrac{1}{2}$

or $\qquad \dfrac{\dfrac{W}{P}}{\dfrac{y}{x}} > \dfrac{1}{2}$

or $\qquad \dfrac{\text{M.A.}}{\text{V.R.}} > \dfrac{1}{2}$

$$\cdots \left(\because \ \frac{W}{P} = \text{M.A.} \quad \text{and} \quad \frac{y}{x} = \text{V.R.} \right)$$

$\therefore \qquad \eta > \dfrac{1}{2} = 0.5 = 50\%$

Hence the condition for a machine, to be reversible, is that its efficiency should be *more than* 50%.

SELF-LOCKING MACHINE:

Sometimes, a machine is not capable of doing any work in the reversed direction, after the effort is removed. Such a machine is called a non-reversible or *self-locking machine.* A little consideration will show, that the condition for a machine to be non-reversible or self-locking is that its efficiency should *not be more than* 50%.

EXAMPLE: In a lifting machine, whose velocity ratio is 50, an effort of 100 N is required to lift a load of 4 kN. Is the machine reversible ? If so, what effort should be applied, so that the machine is at the point of reversing?

Solution. Given: Velocity ratio (V.R.) = 50 ; Effort (P) = 100 N and load (W) = 4 kN = 4000 N.

Reversibility of the machine

We know that M.A. $= \dfrac{W}{P} = \dfrac{4000}{100} = 40$

and efficiency, $\eta = \dfrac{\text{M.A.}}{\text{V.R.}} = \dfrac{40}{50} = 0.8 = 80\%$

Since efficiency of the machine is more than 50%, therefore the machine is reversible.   **Ans.**

Effort to be applied

A little consideration will show that the machine will be at the point of reversing, when its efficiency is 50% or 0.5.

Let $\quad\quad\quad\quad\quad\quad\quad\quad\quad P_1$ = Effort required to lift a load of 4000 N when the machine is at the point of reversing.

We know that $\quad\quad\quad$ M.A. $= \dfrac{W}{P_1} = \dfrac{4000}{P_1} = 4000/P_1$
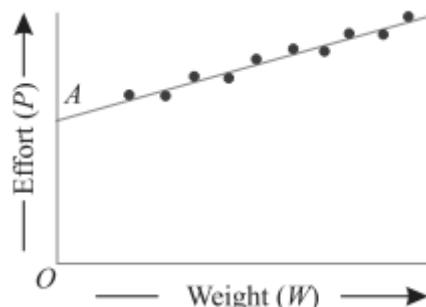
and efficiency, $\quad\quad\quad 0.5 = \dfrac{\text{M.A.}}{\text{V.R.}} = \dfrac{4000/P_1}{50} = \dfrac{80}{P_1}$

∴ $\quad\quad\quad\quad\quad\quad\quad\quad P_1 = \dfrac{80}{0.5} = 160$ N   **Ans.**

LAW OF A MACHINE: The term 'law of a machine' may be defined as relationship between the effort applied and the load lifted. Thus for any machine, if we record the various efforts required to raise the corresponding loads, and plot a graph between effort and load, we shall get a straight line *AB* as shown in Fig.

We also know that the intercept *OA* represents the amount of friction offered by the machine. Or in other words, this is the effort required by the machine to overcome the friction, before it can lift any load.
Mathematically, the law of a lifting machine is given by
the relation:
P = mW + C
where *P* = Effort applied to lift the load,
m = A constant (called coefficient of friction) which is equal to the slope of the line *AB*
W = Load lifted, and
C = Another constant, which represents the machine friction, (i.e. OA)

EXAMPLE: What load can be lifted by an effort of 120 N, if the velocity ratio is 18 and efficiency of the machine at this load is 60%?
Determine the law of the machine, if it is observed that an effort of 200 N is required to lift a load of 2600 N and find the effort required to run the machine at a load of 3.5 kN.

Solution. Given: Effort $(P) = 120\,N$; Velocity ratio (V.R.) $= 18$ and efficiency $(\eta) = 60\% = 0.6$.

Load lifted by the machine.

Let $\qquad W = $ Load lifted by the machine.

We know that M.A. $= \dfrac{W}{P} = \dfrac{W}{120} = W/120$

and efficiency, $\qquad 0.6 = \dfrac{M.A.}{V.R.} = \dfrac{W/120}{18} = \dfrac{W}{2160}$

$\therefore \qquad W = 0.6 \times 2160 = 1296\,N$ **Ans.**

Law of the machine

In the second case, $P = 200\,N$ and $W = 2600\,N$

Substituting the two values of P and W in the law of the machine, *i.e.*, $P = m\,W + C$,

$$120 = m \times 1296 + C \qquad\qquad ...(i)$$

and $\qquad\qquad 200 = m \times 2600 + C \qquad\qquad ...(ii)$

Subtracting equation (i) from (ii),

$$80 = 1304\,m \qquad \text{or} \qquad m = \dfrac{80}{1304} = 0.06$$

and now substituting the value of m in equation (ii)

$$200 = (0.06 \times 2600) + C = 156 + C$$
$$C = 200 - 156 = 44$$

Now substituting the value of $m = 0.06$ and $C = 44$ in the law of the machine,

$$P = 0.06\,W + 44 \qquad \text{**Ans.**}$$

Effort required to run the machine at a load of 3.5 kN.

Substituting the value of $W = 3.5\,kN$ or $3500\,N$ in the law of machine,

$$P = (0.06 \times 3500) + 44 = 254\,N \qquad \text{**Ans.**}$$

MAXIMUM MECHANICAL ADVANTAGE OF A LIFTING MACHINE:

We know that mechanical advantage of a lifting machine,

$$M.A. = \dfrac{W}{P}$$

For maximum mechanical advantage, substituting the value of $P = mW + C$ in the above equation,

$$\text{Max. M.A.} = \dfrac{W}{mW + C} = \dfrac{1}{m + \dfrac{C}{W}} = \dfrac{1}{m} \qquad ...\left(\text{Neglecting } \dfrac{C}{W} \right)$$

MAXIMUM EFFICIENCY OF A LIFTING MACHINE:
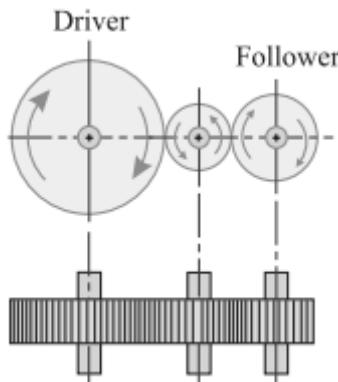
We know that efficiency of a lifting machine,

$$\eta = \frac{\text{Mechanical advantage}}{\text{Velocity ratio}} = \frac{\dfrac{W}{P}}{\text{V.R.}} = \frac{W}{P \times \text{V.R.}}$$

For *maximum efficiency, substituting the value of $P = mW + C$ in the above equation,

$$\text{Max. } \eta = \frac{W}{(mW + C) \times \text{V.R}} = \frac{1}{\left(m + \dfrac{C}{W}\right) \times \text{V.R.}} = \frac{1}{m \times \text{V.R.}} \quad \dots \left(\text{Neglecting } \frac{C}{W}\right)$$

SIMPLE GEAR TRAIN:

Driver

Follower

Now consider a simple train of wheels with one intermediate wheel as shown in Fig.

Let
       N1 = Speed of the driver
       T1= No. of teeth on the driver,

N2, T2 = Corresponding values for the intermediate wheel, and

N3, T3 = Corresponding values for the follower.

Since the driver gears with the intermediate wheel, therefore

$$\frac{N_2}{N_1} = \frac{T_1}{T_2}$$
---------------------------(I)

Similarly, as the intermediate wheel gears with the follower, therefore

$$\frac{N_3}{N_2} = \frac{T_2}{T_3}$$
---------------------------(II)
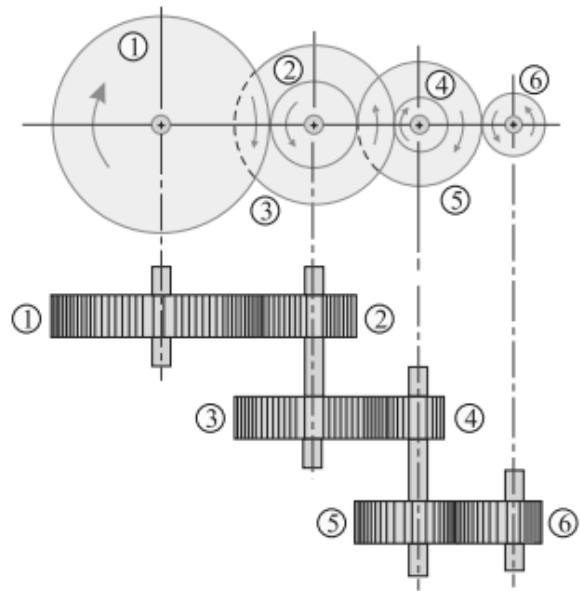
Multiplying equation (ii) by (i),

$$\frac{N_3}{N_2} \times \frac{N_2}{N_1} = \frac{T_2}{T_3} \times \frac{T_1}{T_2}$$

or

$$\frac{N_3}{N_1} = \frac{T_1}{T_3}$$

$$\therefore \quad \frac{\text{Speed of the follower}}{\text{Speed of the driver}} = \frac{\text{No. of teeth on the driver}}{\text{No. of teeth on the follower}}$$

COMPOUND GEAR TRAIN:

Let $\qquad$ N_1 = Speed of the driver 1

$\qquad$ T_1 = No. of teeth on the driver 1,

Similarly $\qquad$ $N_2, N_3, ...N_6$ = Speed of the respective wheels

$\qquad$ $T_2, T_3, ...T_6$ = No. of teeth on the respective wheels.

Since the wheel 1 gears with the wheel 2, therefore

$$\frac{N_2}{N_1} = \frac{T_1}{T_2} \qquad ...(i)$$

Similarly

$$\frac{N_4}{N_3} = \frac{T_3}{T_4} \qquad ...(ii)$$

and

$$\frac{N_6}{N_5} = \frac{T_5}{T_6} \qquad ...(iii)$$

Multiplying equations (i), (ii) and (iii),
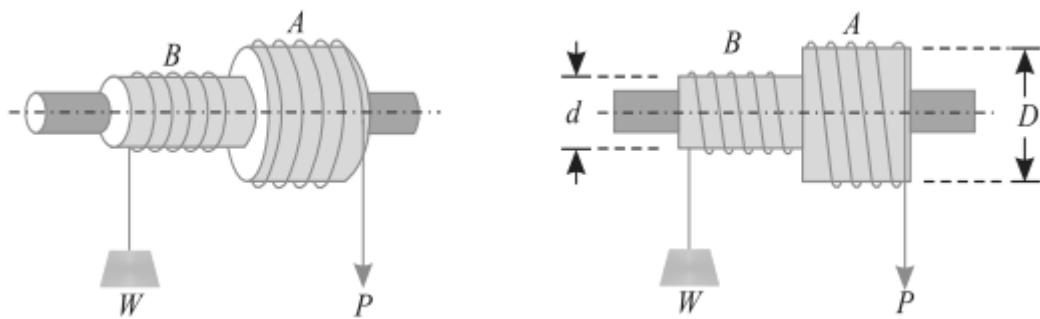
$$\frac{N_2}{N_1} \times \frac{N_4}{N_3} \times \frac{N_6}{N_5} = \frac{T_1}{T_2} \times \frac{T_3}{T_4} \times \frac{T_5}{T_6}$$

$\therefore$

$$\frac{N_6}{N_1} = \frac{T_1 \times T_3 \times T_5}{T_2 \times T_4 \times T_6} \qquad (\because \quad N_2 = N_3 \text{ and } N_4 = N_5)$$

$$= \frac{\text{Product of the teeth on the drivers}}{\text{Product of the teeth on the followers}}$$

SIMPLE LIFTING MACHINES:

1. SIMPLE WHEEL AND AXLE:

In Fig. is shown a simple wheel and axle, in which the wheel A and axle B are keyed to the same shaft. The shaft is mounted on ball bearings, order to reduce the frictional resistance to a minimum. A string is wound round the axle B, which carries the load to be lifted. A second string is wound round the wheel A in the opposite direction to that of the string on B.

Let D = Diameter of effort wheel,

d = Diameter of the load axle,

W = Load lifted, and

P = Effort applied to lift the load.

One end of the string is fixed to the wheel, while the other is free and the effort is applied to this end. Since the two strings are wound in opposite directions, therefore a downward motion of the effort (P) will raise the load (W).

Since the wheel as well as the axle are keyed to the same shaft, therefore when the wheel rotates through one revolution, the axle will also rotate through one revolution. We know that displacement of the effort in one revolution of effort wheel A,

$$= \pi D \ ...(i)$$

and displacement of the load in one revolution

$$= \pi d \ ...(ii)$$

$$\therefore \qquad \text{V.R.} = \frac{\text{Distance moved by the effort}}{\text{Distance moved by the load}} = \frac{\pi D}{\pi d} = \frac{D}{d}$$

Now $$\qquad \text{M.A.} = \frac{\text{Load lifted}}{\text{Effort applied}} = \frac{W}{P}$$

and efficiency $$\qquad \eta = \frac{\text{M.A}}{\text{V.R.}}$$

EXAMPLE: A simple wheel and axle has wheel and axle of diameters of 300 mm and 30 mm respectively. What is the efficiency of the machine, if it can lift a load of 900 N by an effort of 100 N.

Solution. Given: Diameter of wheel (D) = 300 mm; Diameter of axle (d) = 30 mm; Load lifted by the machine (W) = 900 N and effort applied to lift the load (P) = 100 N
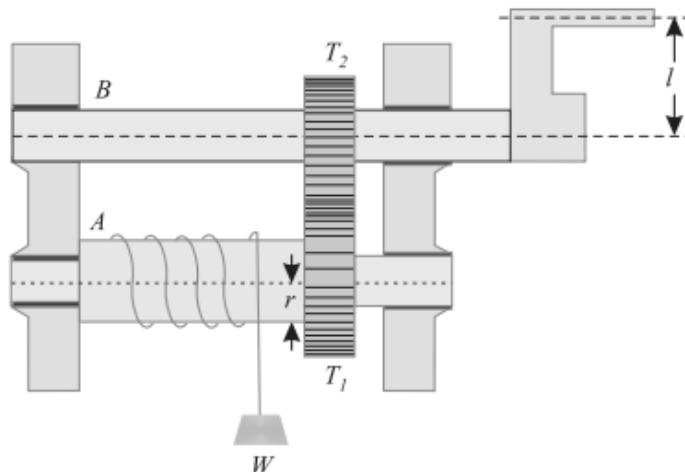
We know that velocity ratio of the simple wheel and axle,

$$V.R. = \frac{D}{d} = \frac{300}{30} = 10$$

and mechanical advantage $\quad M.A. = \frac{W}{P} = \frac{900}{100} = 9$

∴ $\quad$ Efficiency, $\eta = \frac{M.A.}{V.R.} = \frac{9}{10} = 0.9 = 90\%$ $\quad$ **Ans.**

2. SINGLE PURCHASE CRAB WINCH:

In single purchase crab winch, a rope is fixed to the drum and is wound a few turns round it. The free end of the rope carries the load W. A toothed wheel A is rigidly mounted on the load drum. Another toothed wheel B, called pinion, is geared with the toothed wheel A as shown in Fig. The effort is applied at the end of the handle to rotate it.

Let $\quad\quad\quad\quad\quad\quad\quad$ T_1 = No. of teeth on the main gear (or spur wheel) A,

$\quad\quad\quad\quad\quad\quad\quad\quad\quad$ T_2 = No. of teeth on the pinion B,

$\quad\quad\quad\quad\quad\quad\quad\quad\quad$ l = Length of the handle,

$\quad\quad\quad\quad\quad\quad\quad\quad\quad$ r = Radius of the load drum.

$\quad\quad\quad\quad\quad\quad\quad\quad\quad$ W = Load lifted, and

$\quad\quad\quad\quad\quad\quad\quad\quad\quad$ P = Effort applied to lift the load.

We know that distance moved by the effort in one revolution of the handle,

$$= 2\pi l \quad\quad\quad\quad\quad\quad\quad ...(i)$$

∴ No. of revolutions made by the pinion B

$$= 1$$

and no. of revolutions made by the wheel A

$$= \frac{T_2}{T_1}$$

∴ No. of revolutions made by the load drum

$$= \frac{T_2}{T_1}$$

and distance moved by the load $\qquad = 2\pi r \times \frac{T_2}{T_1}$ ...(ii)

∴ $\qquad$ V.R. $= \dfrac{\text{Distance moved by the effort}}{\text{Distance moved by the load}} = \dfrac{2\pi l}{2\pi r \times \dfrac{T_2}{T_1}} = \dfrac{l}{r} \times \dfrac{T_1}{T_2}$ ...(iii)

Now $\qquad$ M.A. $= \dfrac{W}{P}$ $\qquad$ ...as usual

and efficiency, $\qquad \eta = \dfrac{\text{M.A.}}{\text{V.R.}}$ $\qquad$ ...as usual

EXAMPLE: In a single purchase crab winch, the number of teeth on pinion is 25 and that on the spur wheel 100. Radii of the drum and handle are 50 mm and 300 mm respectively. Find the efficiency of the machine and the effect of friction, if an effort of 20 N can lift a load of 300 N.

Solution. Given: No. of teeth on pinion $(T_2) = 25$; No. of teeth on the spur wheel $(T_1) = 100$; Radius of daum $(r) = 50$ mm; Radius of the handle or length of the handle $(l) = 300$ mm; Effort $(P) = 20$ N and load lifted $(W) = 300$ N.

Efficiency of the machine

We know that velocity ratio

$$\text{V.R.} = \frac{1}{r} \times \frac{T_1}{T_2} = \frac{300}{50} \times \frac{100}{25} = 24$$

and $\qquad$ M.A. $= \dfrac{W}{P} = \dfrac{300}{20} = 15$

∴ Efficiency, $\qquad \eta = \dfrac{\text{M.A.}}{\text{V.R.}} = \dfrac{15}{24} = 0.625 = 62.5\%$ $\qquad$ **Ans.**

Effect of friction

We know that effect of friction in terms of load,

$$F_{(load)} = (P \times \text{V.R.}) - W = (20 \times 24) - 300 = 180 \text{ N}$$

and effect of friction in terms of effort,

$$F_{(effort)} = P - \frac{W}{\text{V.R.}} = 20 - \frac{300}{24} = 7.5 \text{ N}$$

It means that if the machine would have been ideal (*i.e.* without friction) then it could lift an extra load of 180 N with the same effort of 20 N. Or it could have required 7.5 N less force to lift the same load of 300 N. $\qquad$ **Ans.**

EXAMPLE:

A single purchase crab winch, has the following details:

Length of lever   = 700 mm

Number of pinion teeth = 12

Number of spur gear teeth = 96

Diameter of load axle = 200 mm

It is observed that an effort of 60 N can lift a load of 1800 N and an effort of 120 N can lift a load of 3960 N.

What is the law of the machine? Also find efficiency of the machine in both the cases.

Solution. Given: Length of lever $(l) = 700$ *mm*; No. of pinion teeth $(T_2) = 12$; No. of spur geer teeth $(T_1) = 96$ and dia of load axle $= 200$ mm or radius $(r) = 200/2 = 100$ mm.

(i) Law of the machine

When $P_1 = 60$ N, $W_1 = 1800$ N and when $P_2 = 120$ N, $W_2 = 3960$ N.

Substituting the values of P and W in the law of the machine *i.e.,* $P = mW + C$

$$60 = (m \times 1800) + C \qquad \qquad ...(i)$$

and
$$120 = (m \times 3960) + C \qquad \qquad ...(ii)$$

Subtracting equation (*i*) from equation (*ii*)

$$60 = m \times 2160$$

or
$$m = \frac{60}{2160} = \frac{1}{36}$$

Now substituting this value of *m* in equation (*i*),

$$60 = \left(\frac{1}{36} \times 1800 \right) + C = 50 + C$$

∴
$$C = 60 - 50 = 10$$

and now substituting the value of $m = 1/36$ and $C = 10$ in the law of machine,

$$P = \frac{1}{36}W + 10 \qquad \textbf{Ans.}$$

(ii) *Efficiencies of the machine in both the cases*

We know that velocity ratio

$$V.R. = \frac{l}{r} \times \frac{T_1}{T_2} = \frac{700}{100} \times \frac{96}{12} = 56$$

and mechanical advantage in the first case

$$M.A. = \frac{W_1}{P_1} = \frac{1800}{60} = 30$$

∴ Efficiency $\quad \eta_1 = \dfrac{M.A}{V.R.} = \dfrac{30}{56} = 0.536 = 53.6\%$ **Ans.**
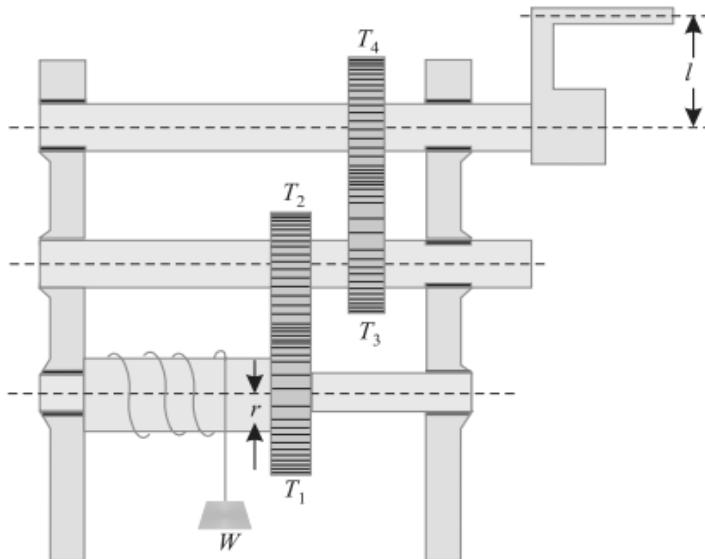
Similarly, mechanical advangate in the second case,

$$M.A. = \frac{W_2}{P_2} = \frac{3960}{120} = 33$$

∴ Efficiency $\quad \eta_2 = \dfrac{M.A.}{V.R.} = \dfrac{33}{56} = 0.589 = 58.9\%$ **Ans.**

DOUBLE PURCHASE CRAB WINCH:

A double purchase crab winch is an improved form of a single purchase crab winch, in which the velocity ratio is intensified with the help of one more spur wheel and a pinion. In a double purchase crab winch, there are two spur wheels of teeth $T1$ and $T2$ and $T3$ as well as two pinions of teeth $T2$ and $T4$.

The arrangement of spur wheels and pinions are such that the spur wheel with $T1$ gears with the pinion of teeth $T2$. Similarly, the spur wheel with teeth $T3$ gears with the pinion of the teeth $T4$, The effort is applied to a handle as shown in Fig.

Let
$$T_1 \text{ and } T_3 = \text{No. of teeth of spur wheels,}$$
$$T_2 \text{ and } T_4 = \text{No. of teeth of the pinions}$$
$$l = \text{Length of the handle,}$$
$$r = \text{Radius of the load drum,}$$
$$W = \text{Load lifted, and}$$
$$P = \text{Effort applied to lift the load, at the end of the handle.}$$

We know that distance moved by the effort in one revolution of the handle,
$$= 2\pi l \qquad \qquad \qquad ...(i)$$

∴ No. of revolutions made by the pinion 4
$$= 1$$

and no. of revolutions made by the wheel 3
$$= \frac{T_4}{T_3}$$

∴ No. of revolutions made by the pinion 2
$$= \frac{T_4}{T_3}$$

and no. of revolutions made by the wheel 1
$$= \frac{T_2}{T_1} \times \frac{T_4}{T_3}$$

∴ Distance moved by the load
$$= 2\pi r \times \frac{T_2}{T_1} \times \frac{T_4}{T_3} \qquad \qquad \qquad ...(ii)$$

∴ $\qquad \qquad$ V.R. $= \dfrac{\text{Distance moved by the effort}}{\text{Distance moved by the load}}$

$$= \frac{2\pi l}{2\pi r \times \dfrac{T_2}{T_1} \times \dfrac{T_4}{T_3}} = \frac{l}{r}\left(\frac{T_1}{T_2} \times \frac{T_3}{T_4}\right)$$

Now $\qquad \qquad$ M.A. $= \dfrac{W}{P} \qquad \qquad \qquad$ ...as usual

and efficiency, $\qquad \qquad \eta = \dfrac{\text{M.A.}}{\text{V.R.}} \qquad \qquad \qquad$ ...as usual

EXAMPLE:

In a double purchase crab winch, teeth of pinions are 20 and 25 and that of spur wheels are 50 and 60. Length of the handle is 0.5 metre and radius of the load drum is 0.25 metre. If efficiency of the machine is 60%, find the effort required to lift a load of 720 N.

Solution. Given: No. of teeth of pinion $(T_2) = 20$ and $(T_4) = 25$; No. of teeth of spur wheel $(T_1) = 501$ and $(T_3) = 60$; Length of the handle $(l) = 0.5$ m; Radius of the load drum $(r) = 0.25$ m; Efficiency $(\eta) = 60\% = 0.6$ and load to be lifted $(W) = 720$ N.

Let $\qquad\qquad\qquad P$ = Effort required in newton to lift the load.

We know that velocity ratio

$$\text{V.R.} = \frac{1}{r}\left(\frac{T_1}{T_2} \times \frac{T_3}{T_4}\right) = \frac{0.5}{0.25}\left(\frac{50}{20} \times \frac{60}{25}\right) = 12$$
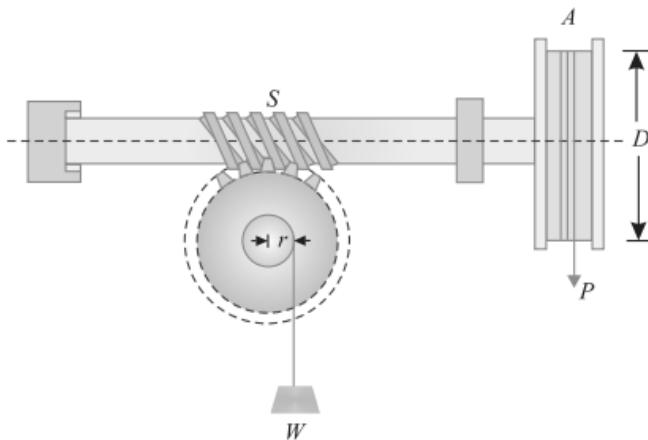
and $\qquad\qquad\qquad \text{M.A.} = \frac{W}{P} = \frac{720}{P}$

$\therefore$ Efficiency $\qquad 0.6 = \dfrac{\text{M.A.}}{\text{V.R.}} = \dfrac{\dfrac{720}{P}}{12} = \dfrac{60}{P}$

or $\qquad\qquad\qquad P = \dfrac{60}{0.6} = 100$ N $\qquad$ **Ans.**

WORM AND WORM WHEEL:

It consists of a square threaded screw, *S* (known as worm) and a toothed wheel (known as worm wheel) geared with each other, as shown in Fig. A wheel *A* is attached to the worm, over which passes a rope as shown in the figure. Sometimes, a handle is also fixed to the worm (instead of the wheel). A load drum is securely mounted on the worm wheel.

Let;
D = Diameter of the effort wheel,
r = Radius of the load drum
W = Load lifted,
P = Effort applied to lift the load, and
T = No. of teeth on the worm wheel.

We know that distance moved by the effort in one revolution of the wheel (or handle)

$\qquad = \pi D \;...(i)$

If the worm is single-threaded (*i.e.*, for one revolution of the wheel *A*, the screw *S* pushes the worm wheel through one teeth), then the load drum will move through

$$= \frac{1}{T} \text{ revolution}$$

and distance, through which the load will move

$$= \frac{2\pi r}{T} \qquad \qquad \text{...(ii)}$$

∴ $$\text{V.R.} = \frac{\text{Distance moved by the effort}}{\text{Distance moved by the load}}$$

$$= \frac{\pi D}{\frac{2\pi r}{T}} = \frac{DT}{2r} \qquad \qquad \text{...(iii)}$$

Now $$\text{M.A.} = \frac{W}{P} \qquad \qquad \text{...as usual}$$

and efficiency, $$\eta = \frac{\text{M.A.}}{\text{V.R.}} \qquad \qquad \text{...as usual}$$

Notes : 1. If the worm is double-threaded *i.e.*, for one revolution of wheel *A*, the screw *S* pushes the worm wheel through two teeths, then

$$\text{V.R.} = \frac{DT}{2 \times 2r} = \frac{DT}{4r}$$

2. In general, if the worm is *n* threaded, then

$$\text{V.R.} = \frac{DT}{2nr}$$

EXAMPLE: A worm and worm wheel with 40 teeth on the worm wheel has effort wheel of 300 mm diameter and load drum of 100 mm diameter. Find the efficiency of the machine, if it can lift a load of 1800 N with an effort of 24 N.

Solution. Given: No. of teeth on the worm wheel (*T*) = 40 ; Diameter of effort wheel = 300 mm Diameter of load drum = 100 mm or radius (*r*) = 50 mm; Load lifted (*W*) 1800 N and effort (*P*) = 24 N.

We know that velocity ratio of worm and worm wheel,

$$\text{V.R.} = \frac{DT}{2r} = \frac{300 \times 40}{2 \times 50} = 120$$

and $$\text{M.A.} = \frac{W}{P} = \frac{1800}{24} = 75$$

∴ Efficiency, $$\eta = \frac{\text{M.A.}}{\text{V.R.}} = \frac{75}{120} = 0.625 = 62.5\% \quad \textbf{Ans.}$$

SCREW JACK:
It consists of a screw, fitted in a nut, which forms the body of the jack. The principle, on which a screw jack works, is similar to that of an inclined plane.
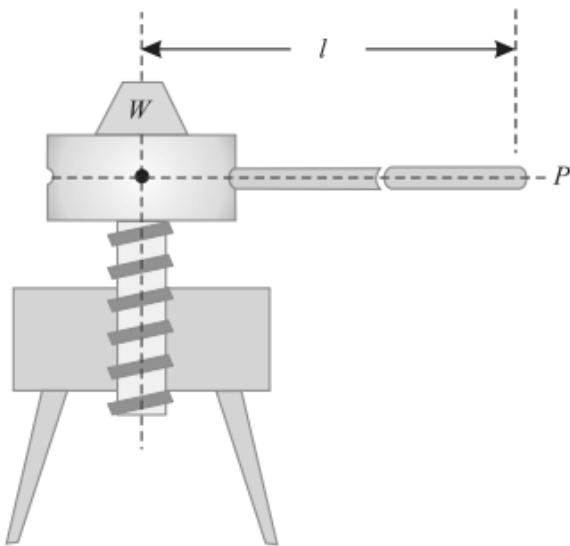

Fig.　 shows a simple screw jack, which is rotated by the application of an effort at the end of the lever, for lifting the load. Now consider a single threaded simple screw jack.

Let $\qquad$ l = Length of the effort arm,

p = Pitch of the screw,

W = Load lifted, and

P = Effort applied to lift the load at the end of teh lever.

We know that distance moved by the effort in one revolution of screw,

$$= 2\pi l \qquad \qquad ...(i)$$

and distance moved by the load　　 $= p$ ...(ii)

$\therefore$ 　Velocity ratio $= \dfrac{\text{Distance moved by the effort}}{\text{Distance moved by the load}} = \dfrac{2\pi l}{p}$ ...(iii)

Now 　　M.A. $= \dfrac{W}{P}$ ...as usual

and efficeincy, 　　 $\eta = \dfrac{\text{M.A.}}{\text{V.R.}}$ ...as usual

Note: The value of *P i.e.*, the effort applied may also found out by the relation :

$$*P = W \tan (\alpha + \phi)$$

where 　　 W = Load lifted

$$\tan \alpha = \dfrac{p}{\pi d}$$

and 　　 $\tan \phi = \mu$ = Coefficient of friction.

EXAMPLE:

A screw jack has a thread of 10 mm pitch. What effort applied at the end of a handle 400 mm long will be required to lift a load of 2 kN, if the efficiency at this load is 45%.

SOLUTION: Given: Pitch of thread (p) = 10 mm;
Length of the handle (l) = 400 mm;
Load lifted (W) = 2 kN = 2000 N and efficiency (n) = 45% = 0.45.
Let P = Effort required to lift the load

We know that velocity ratio

$$V.R. = \frac{2\pi l}{p} = \frac{2\pi \times 400}{10} = 251.3$$

and

$$M.A. = \frac{W}{P} = \frac{2000}{P}$$

We also know that efficiency,

$$0.45 = \frac{M.A.}{V.R.} = \frac{\dfrac{2000}{P}}{251.3} = \frac{7.96}{P}$$

$$P = \frac{7.96}{0.45} = 17.7 \text{ N} \qquad \textbf{Ans.}$$

HOISTING MACHINE:

Mechanisms for raising and lowering material with intermittent motion while holding the material freely suspended. Hoisting machines are capable of picking up loads at one location and depositing them at another anywhere within a limited area. In contrast, elevating machines move their loads only in a fixed vertical path, and monorails operate on a fixed horizontal path rather than over a limited area.

The principal components of hoisting machines are: sheaves and pulleys, for the hoisting mechanisms; winches and hoists, for the power units; and derricks and cranes, for the structural elements.

TYPES:
- Pulley and sheave block
- Chain hoists
- Mobile cranes
- Winch
- Jack
- Shear leg
- Tower cranes
- Whirler cranes
- Derrick cranes
- Gantry cranes

DERRICK:

A derrick is a lifting device composed at minimum of one guyed mast, as in a gin pole, which may be articulated over a load by adjusting its guys. Most derricks have at least two components, either a guyed mast or self-supporting tower, and a boom hinged at its base to provide articulation, as in a stiffleg derrick.

The most basic type of derrick is controlled by three or four lines connected to the top of the mast, which allow it both to move laterally and cant up and down. To lift a load, a separate line runs up and over the mast with a hook on its free end, as with a crane.

Forms of derricks are commonly found aboard ships and at docking facilities. Some large derricks are mounted on dedicated vessels, and known as floating derrick and sheerlegs.

The term derrick is also applied to the framework supporting a drilling apparatus in an oil rig.

6. DYNAMICS

KINETICS: It is the branch of Dynamics, which deals with the bodies in motion due to the application of forces.

KINEMATICS: It is that branch of Dynamics, which deals with the bodies in motion, without any reference to the forces which are responsible for the motion.

PRINCIPLE OF DYNAMICS:
1. A body can posses acceleration only when some force is applied on it. Or in other words, if no force is applied on the body, then there will be no acceleration, and the body will continue to move with the existing uniform velocity.
2. The force applied on a body is proportional to the product of the mass of the body and the acceleration produced in it.

NEWTON'S LAWS OF MOTION:
Following are the three laws of motion, which were enunciated by Newton,
1. Newton's First Law of Motion states, "Everybody continues in its state of rest or of uniform motion, in a straight line, unless it is acted upon by some external force."

2. Newton's Second Law of Motion states, "The rate of change of momentum is directly proportional to the impressed force, and takes place in the same direction, in which the force acts."

$$F = ma = \text{Mass} \times \text{Acceleration}$$

3. Newton's Third Law of Motion states, "To every action, there is always an equal and opposite reaction."

D'ALEMBERT'S PRINCIPLE:

It states, "If a rigid body is acted upon by a system of forces, this system may be reduced to a single resultant force whose magnitude, direction and the line of action may be found out by the methods of graphic statics."
We know that force acting on a body.
$P = ma$ ...(i)
The equation (i) may also be written as :
$P - ma = 0$ ...(ii)

It may be noted that equation (i) is the equation of dynamics whereas the equation (ii) is the equation of statics. The equation (ii) is also known as the equation of dynamic equilibrium under the action of the real force P. This principle is known as D' Alembert's principle.

EQUATIONS OF MOTION:

Let
      u = Initial velocity,
      v = Final velocity,

t = Time (in seconds) taken by the particle to change its velocity from u to v.
a = Uniform positive acceleration, and
s = Distance travelled in t seconds.

Since in t seconds, the velocity of the particle has increased steadily from (u) to (v) at the rate of a, therefore total increase in velocity = a t

$$v = u + at \qquad \qquad \text{...(i)}$$

We know that distance travelled by the particle,
s = Average velocity × Time

$$= \left(\frac{u+v}{2}\right) \times t \qquad \qquad \text{...(ii)}$$

$$s = \left(\frac{u+u+at}{2}\right) \times t = ut + \frac{1}{2}at^2 \qquad \qquad \text{...(iii)}$$

$$v^2 = u^2 + 2as$$

EXAMPLE:
A scooter starts from rest and moves with a constant acceleration of 1.2 m/s2. Determine its velocity, after it has travelled for 60 meters.

Solution. Given : Initial velocity $(u) = 0$ (because it starts from rest) Acceleration $(a) = 1.2$ m/s^2 and distance travelled $(s) = 60$ m.

Let $v =$ Final velocity of the scooter.

We know that $v^2 = u^2 + 2as = (0)^2 + 2 \times 1.2 \times 60 = 144$

$$v = 12 \text{ m/s} = \frac{12 \times 3600}{1000} = 43.2 \text{ km.p.h. } \textbf{Ans.}$$

EXAMPLE: A motor car takes 10 seconds to cover 30 meters and 12 seconds to cover 42 meters. Find the uniform acceleration of the car and its velocity at the end of 15 seconds.

Solution. Given : When $t = 10$ seconds, $s = 30$ m and when $t = 12$ seconds, $s = 42$ m.

Uniform acceleration

Let $\qquad\qquad\qquad\qquad u$ = Initial velocity of the car, and

$\qquad\qquad\qquad\qquad\qquad a$ = Uniform acceleration.

We know that the distance travelled by the car in 10 seconds,

$$30 = ut + \frac{1}{2}at^2 = u \times 10 + \frac{1}{2} \times a\,(10)^2 = 10u + 50\,a$$

Multiplying the above equation by 6,

$$180 = 60u + 300a \qquad\qquad\qquad ...(i)$$

Similarly, distance travelled by the car in 12 seconds,

$$42 = u \times 12 + \frac{1}{2} \times a\,(12)^2 = 12u + 72a$$

Mulitiplying the above equation by 5,

$$210 = 60u + 360a \qquad\qquad\qquad ...(ii)$$

Subtracting equation (i) from (ii),

$$30 = 60a \qquad \text{or} \qquad a = \frac{30}{60} = 0.5 \text{ m/s}^2 \text{ **Ans.**}$$

Velocity at the end of 15 seconds

Substituting the value of a in equation (i)

$$180 = 60u + (300 \times 0.5) = 60u + 150$$

$$\therefore \qquad\qquad u = \frac{(180 - 150)}{60} = 0.5 \text{ m/s}$$

We know that the velocity of the car after 15 seconds,

$$v = u + at = 0.5 + (0.5 \times 15) = 8 \text{ m/s} \quad \text{**Ans.**}$$

WORK: Whenever a force acts on a body, and the body undergoes some displacement, then work is said to be done. *e.g.*, if a force *P*, acting on a body, causes it to move through a distance *s* as shown in Fig.(*a*). Then work done by the force *P*

= Force × Distance
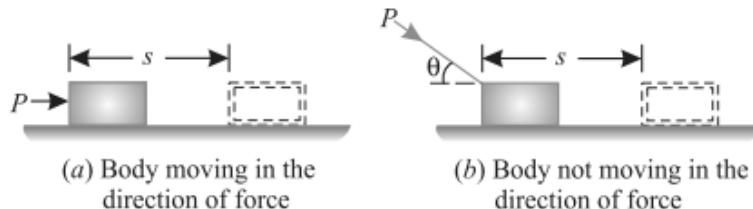
= *P* × *s*

Sometimes, the force *P* does not act in the direction of motion of the body, or in other words, the body does not move in the direction of the force as shown in Fig.(*b*).

Then work done by the force *P*

= Component of the force in the direction of motion × Distance

= *P* cos θ × *s*

(*a*) Body moving in the direction of force

(*b*) Body not moving in the direction of force

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UNITS OF WORK:

The units of work (or work done) are :
1**. One N-m**: It is the work done by a force of 1 N, when it displaces the body through 1 m. It is called joule (briefly written as J), Mathematically.
<div align="center">

1 joule = 1 N-m
</div>

2. **One kN-m:** It is the work done by a force of 1 kN, when it displaces the body through 1 m. It is also called kilojoule (briefly written as kJ). Mathematically.
<div align="center">

1 kilo-joule = 1 kN-m
</div>

POWER:
The power may be defined as the rate of doing work. It is thus the measure of performance of engines. e.g. an engine doing a certain amount of work, in one second, will be twice as powerful as an engine doing the same amount of work in two seconds.

UNITS OF POWER:
In S.I. units, the unit of power is watt (briefly written as W) which is equal to 1 N-m/s or 1 J/s. Generally, a bigger unit of power (kW) is used, which is equal to 10^3 W. Sometimes, a still bigger unit of power (MW) is also used, which is equal to 10^6 W.

ENERGY:
The energy may be defined as the capacity to do work. It exists in many forms i.e., mechanical, electrical chemical, heat, light etc. But in this subject, we shall deal in mechanical energy only.

UNITS OF ENERGY:
Since the energy of a body is measured by the work it can do, therefore the units of energy will be the same as those of the work.

POTENTIAL ENERGY:
It is the energy possessed by a body, for doing work, by virtue of its position. e.g.,
1. A body, raised to some height above the ground level, possesses some potential energy, because it can do some work by falling on the earth's surface.
2. Compressed air also possesses potential energy because it can do some work in expanding, to the volume it would occupy at atmospheric pressure.
3. A compressed spring also possesses potential energy, because it can do some work in recovering to its original shape.

Now consider a body of mass (m) raised through a height (h) above the datum level. We know that work done in raising the body
= Weight × Distance = (mg) h = mgh
This work (equal to m.g.h) is stored in the body as potential energy.

KINETIC ENERGY:
It is the energy, possessed by a body, for doing work by virtue of its mass and velocity of motion.

$$KE = \frac{mv^2}{2}$$

LAW OF CONSERVATION OF ENERGY:
It states "The energy can neither be created nor destroyed, though it can be transformed from one form into any of the forms, in which the energy can exist."

From the above statement, it is clear, that no machine can either create or destroy energy, though it can only transform from one form into another. We know that the output of a machine is always less than the input of the machine. This is due to the reason that a part of the input is utilized in overcoming friction of the machine. This does not mean that this part of energy, which is used in overcoming the friction, has been destroyed. But it reappears in the form of heat energy at the bearings and other rubbing surfaces of the machine, though it is not available to us for useful work.

The above statement may be exemplified as below :
1. In an electrical heater, the electrical energy is converted into heat energy.
2. In an electric bulb, the electrical energy is converted into light energy.
3. In a dynamo, the mechanical energy is converted into electrical energy.

IMPULSE AND MOMENTUM:
Impulse is the change of momentum of an object when the object is acted upon by a force for an interval of time. So, with impulse, you can calculate the change in momentum, or you can use impulse to calculate the average impact force of a collision.
Impulse = Force X time

Momentum is the quantity of motion of a moving body, measured as a product of its mass and velocity.
Momentum = mass x velocity

PHENOMENON OF COLLISION:
Whenever two elastic bodies collide with each other, the phenomenon of collision takes place as given below :
1. The bodies, immediately after collision, come momentarily to rest.
2. The two bodies tend to compress each other, so long as they are compressed to the maximum value.
3. The two bodies attempt to regain its original shape due to their elasticity. This process of regaining the original shape is called **restitution**.
The time taken by the two bodies in compression, after the instant of collision, is called the time of compression and time for which restitution takes place is called the time of restitution. The sum of the two times of collision and restitution is called time of collision, period of collision, or period of impact.

LAW OF CONSERVATION OF MOMENTUM:
It states, "The total momentum of two bodies remains constant after their collision or any other mutual action." Mathematically

$$m_1u_1 + m_2u_2 = m_1 v_1 + m_2 v_2$$
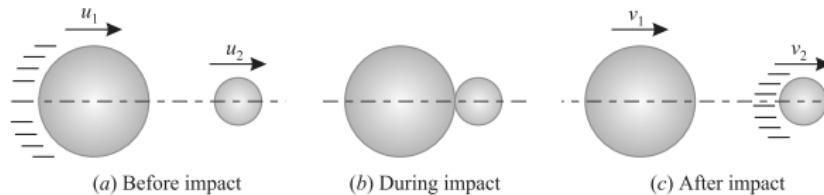
Where; m_1 = Mass of the first body,
u_1 = Initial velocity of the first body,
v_1 = Final velocity of the first body, and
m_2, u_2, v_2 = Corresponding values for the second body.

COEFFICIENT OF RESTITUTION:

(a) Before impact   (b) During impact   (c) After impact

Consider two bodies A and B having a direct impact as shown in Fig. (a).
Let
$u1$ = Initial velocity of the first body,
$v1$ = Final velocity of the first body, and
$u2, v2$ = Corresponding values for the second body.

The impact will take place only if $u1$ is greater than $u2$.
Therefore, the velocity of approach will be equal to $(u1 – u2)$. After impact, the separation of the two bodies will take place, only if $v2$ is greater than $v1$. Therefore the velocity of separation will be equal to $(v2 – v1)$.
Now as per Newton's Law of Collision of Elastic Bodies:
Velocity of separation = e × Velocity of approach
**$(v2 – v1) = e (u1 – u2)$**
where e is a constant of proportionality, and is called the **coefficient of restitution**.
- Its value lies between 0 and 1. It may be noted that if e = 0, the two bodies are inelastic. But if e = 1, the two bodies are perfectly elastic.
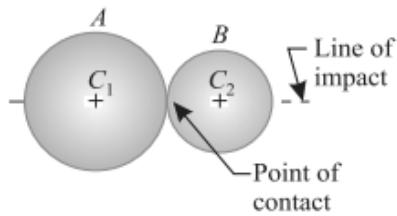
NOTE:
- If the two bodies are moving in the same direction, before or after impact, then the velocity of approach or separation is the difference of their velocities. But if the two bodies are moving in the opposite directions, then the velocity of approach or separation is the algebraic sum of their velocities.

DIRECT COLLISION OF TWO BODIES:

The line of impact, of the two colliding bodies, is the line joining the centres of these bodies and passes through the point of contact or point of collision as shown in Fig. If the two bodies, before impact, are moving along the line of impact, the collision is called as direct impact as shown in Fig.

$$\text{Now; } m_1u_1 + m_2u_2 = m_1v_1 + m_2v_2$$

NOTES: 1. Since the velocity of a body is a vector quantity, therefore its direction should always be kept in view while solving the examples.

  2. If velocity of a body is taken as + ve in one direction, then the velocity in opposite direction should be taken as – ve.

  3. If one of the bodies is initially at rest, then such a collision is also called impact.

EXAMPLE: A ball of mass 1 kg moving with a velocity of 2 m/s impinges directly on a ball of mass 2 kg at rest. The first ball, after impinging, comes to rest. Find the velocity of the second ball after the impact and the coefficient of restitution.

Solution. Given : Mass of first ball $(m_1) = 1$ kg ; Initial velocity of first ball $(u_1) = 2$ m/s ; Mass of second ball $(m_2) = 2$ kg ; Initial velocity of second ball $(u_2) = 0$ (because it is at rest) and final velocity of first ball after impact $(v_1) = 0$ (because, it comes to rest)

Velocity of the second ball after impact.

  Let $v_2 = $ Velocity of the second ball after impact.

  We know from the law of conservation of momentum that

$$m_1u_1 + m_2u_2 = m_1v_1 + m_2v_2$$
$$(1 \times 2) + (2 \times 0) = (1 \times 0) + (2 \times v_2)$$

$\therefore$ $2 = 2v_2$

or $v_2 = 1$ m/s **Ans.**

Coefficient of restitution

  Let $e = $ Coefficient of restitution.

  We also know from the law of collision of elastic bodies that

$$(v_2 - v_1) = e\,(u_1 - u_2)$$
$$(1 - 0) = e\,(2 - 0)$$

or $e = \dfrac{1}{2} = 0.5$ **Ans.**

EXAMPLE: The masses of two balls are in the ratio of 2: 1 and their velocities are in the ratio of 1: 2, but in the opposite direction before impact. If the coefficient of restitution be 5/6, prove that after the impact, each ball will move back with 5/6th of its original velocity.

Solution. Given : Mass of first ball $(m_1) = 2\,M$; Mass of second ball $(M_2) = M$; Initial velocity of first ball $(u_1) = U$; Initial velocity of second ball $(u_2) = -2U$ (Minus sign due to opposite direction) and coefficient of restitution $(e) = \dfrac{5}{6}$

Let v_1 = Final velocity of the first ball, and
v_2 = Final velocity of the second ball.

We know from the law of conservation of momentum that

$$m_1 u_1 + m_2 u_2 = m_1 v_1 + m_2 v_2$$
$$2M \times U + M\,(-2U) = 2Mv_1 + Mv_2$$

or
$$0 = 2Mv_1 + Mv_2$$

∴
$$v_2 = -2v_1 \qquad \qquad ...(i)$$

We also know from the law of collision of elastic bodies that

$$(v_2 - v_1) = e\,(u_1 - u_2) = \frac{5}{6}\big[U - (-2U)\big] = \frac{5U}{2} \qquad ...(ii)$$

Substituting the value of v_2 from equation (i)

$$\big[-2v_1 - (v_1)\big] = \frac{5U}{2} \qquad \text{or} \qquad v_1 = -\frac{5}{6} \times U$$

Minus sign indicates that the direction of v_1 is opposite to that of U. Thus the first ball will move back with $\dfrac{5}{6}$ th of its original velocity. **Ans.**

Now substituting the value of v_1 in equation (i),

$$v_2 = -2\left(-\frac{5}{6} \times U\right) = +\frac{5}{6} \times 2U$$

Plus sign indicates that the direction of v_2 is the same as that of v_1 or opposite to that of u_2. Thus the second ball will also move back with $\dfrac{5}{6}$ th of its original velocity. **Ans.**