



# Modeling Contact and Collisions for Robotic Assembly

2020 American Modelica Conference

Scott A. Bortoff

21-September-2020



MITSUBISHI ELECTRIC RESEARCH LABORATORIES (MERL)  
Cambridge, Massachusetts, USA  
<http://www.merl.com>

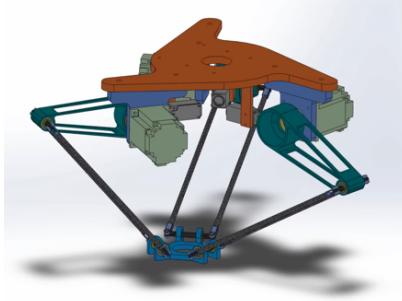
© MERL



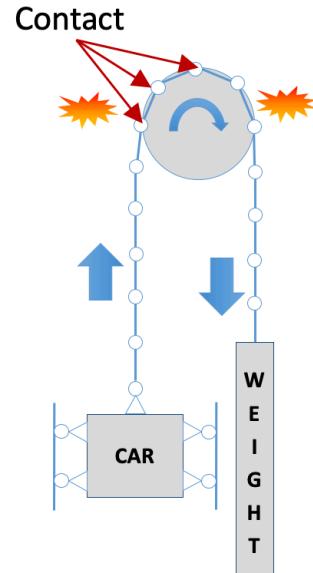
# Motivation: Robust Robotic Assembly, Mechanical Contact Problems

- Propose implicit, event-driven, penalty-based model of rigid body contact / collision
- Hybrid: Continuous-time DAE model + finite state machine
- Uses: Not just simulation. Mathematical analysis, model-based control design
- Implementation: Native Modelica, variable-step, stiff (implicit) solver

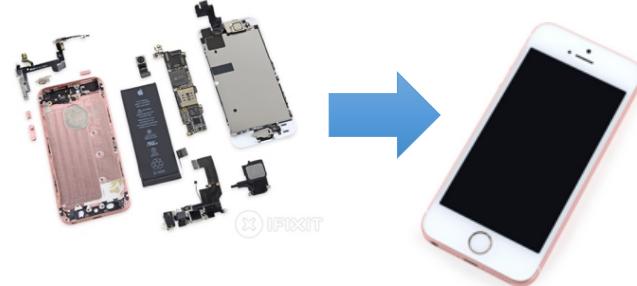
MERL Delta Robot



Elevator Motion



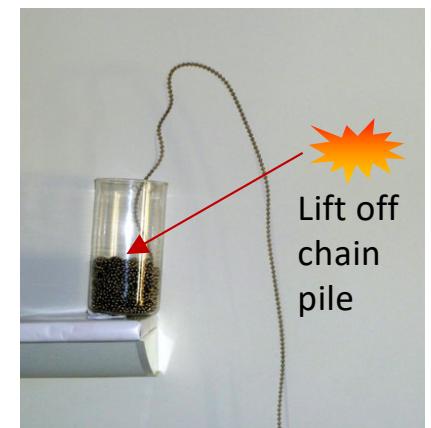
Mechanical Assembly



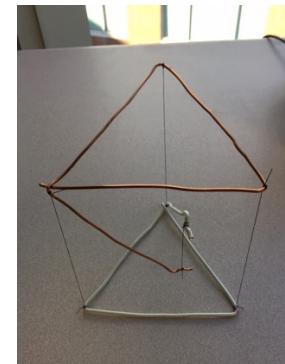
Ball Maze



Chain Fountain

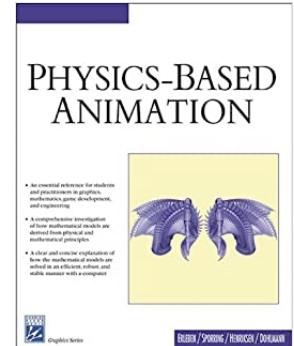
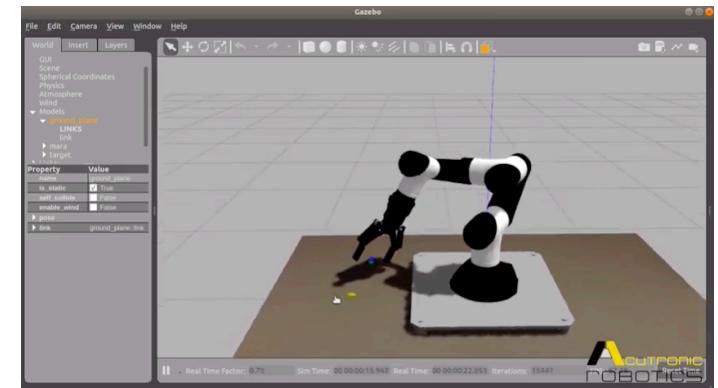
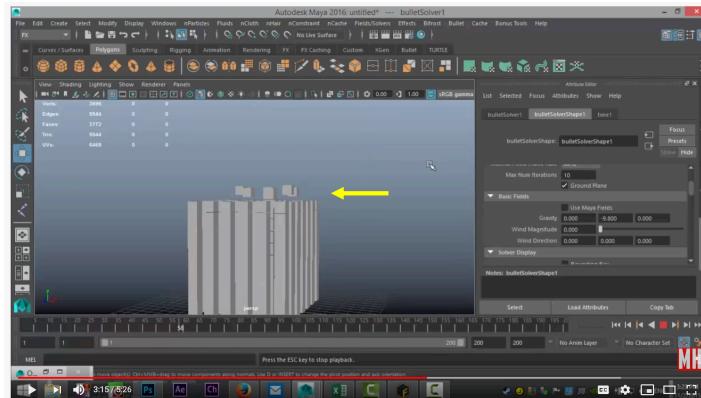


Tensegrity



# Collisions and Contact: Lots of Previous Work

- Physics-Based Animation: Bullet, PhyX, Gazebo (with different engines such as DART), etc.
- State-of-the-art: Represent as Nonlinear or Linear Complementary Problem (LCP), solve
  - Solve a QP problem at each discrete time step
  - Fixed time step, usually first-order (Euler) symplectic (to approx. conserve energy) integrator
  - Simulation is the only purpose
- Documented limitations and problems...
  - Energy Conservation
  - Difficulty with widely ranging object sizes
  - Collision detection requires non-zero margin
  - Tolerances need tuning
  - Not intended for numerical analysis – only to simulate
  - [https://youtu.be/k6nKC\\_DCh3o?t=188](https://youtu.be/k6nKC_DCh3o?t=188)



# Delta Robot Differential-Algebraic Model (2018)

- Forward Kinematics of each arm...

$$\psi(q_i) = \begin{bmatrix} l_2 \sin(q_{i2}) \sin(q_{i3}) \\ l_1 \cos(q_{i1}) + l_2 \cos(q_{i2}) \\ l_1 \sin(q_{i1}) + l_2 \sin(q_{i2}) \cos(q_{i3}) \end{bmatrix}$$

Arm 1:  $q_1 = [q_{11}, q_{12}, q_{13}]^T$   
 Arm 2:  $q_2 = [q_{21}, q_{22}, q_{23}]^T$   
 Arm 3:  $q_3 = [q_{31}, q_{32}, q_{33}]^T$

- Constraint at wrist flange (6 equations)...

$$h(q) = \begin{bmatrix} \psi(q_1) - R_z(2\pi/3) \cdot \psi(q_2) \\ \psi(q_1) - R_z(-2\pi/3) \cdot \psi(q_3) \end{bmatrix} \quad h(q) : \mathcal{R}^9 \rightarrow \mathcal{R}^6$$

- Robot Dynamics...Index 3 DAE 😞

$$\begin{aligned} \dot{q} &= v \\ M(q)\dot{v} + C(q, v) + G(q) &= \lambda^T H(q) + Bu \\ h(q) &= 0 \end{aligned}$$

$H = \frac{\partial h}{\partial q}$

- Robot Dynamics ... Index 1 DAE 😊

$$\begin{aligned} \dot{q} &= v \\ M(q)\dot{v} + C(q, v) + G(q) &= \lambda^T H(q) + Bu \\ h''(q, v, \lambda) + \alpha_1 h'(q, v) + \alpha_0 h(q) &= 0 \end{aligned}$$

- 24 equations, 24 states ( $q, v, \lambda$ )

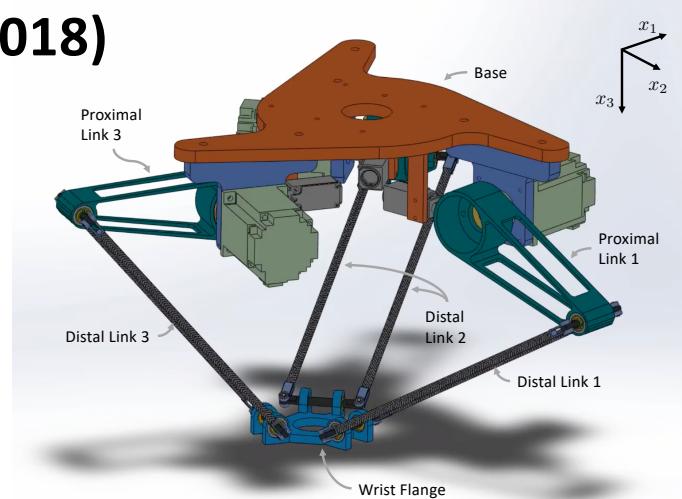
$$\begin{aligned} M(q) &= \text{diag}(m(q_1), m(q_2), m(q_3)) \in \mathbb{R}^{9 \times 9}, \\ C(q, v) &= \text{diag}(c(q_1, v_1), c(q_2, v_2), c(q_3, v_3)) \in \mathbb{R}^9, \\ G(q) &= \text{diag}(g(q_1), g(q_2), g(q_3)) \in \mathbb{R}^9, \\ B &= \text{diag}(b, b, b) \in \mathbb{R}^{9 \times 3}. \end{aligned}$$

- Can be expressed

$$\begin{aligned} \dot{\eta} &= f(\eta, \xi, u) & \eta \in \mathcal{R}^6 \\ \dot{\xi} &= A\xi, & \xi \in \mathcal{R}^{12} \end{aligned}$$

- Robot dynamics are zero dynamics

$$\dot{\eta} = f(\eta, 0, u)$$



# Contact and Collision Model Idea

- Dynamics

$\dot{q} = v$

$\lambda = 0$  Free  
 $\lambda > 0$  Contact

$$M(q)\dot{v} + C(q, v) + G(q) = \lambda^T H(q) + Bu$$

$$h''(q, v, \lambda) + \alpha_1 h'(q, v) + \alpha_0 h(q) = 0$$

This can model elastic collision

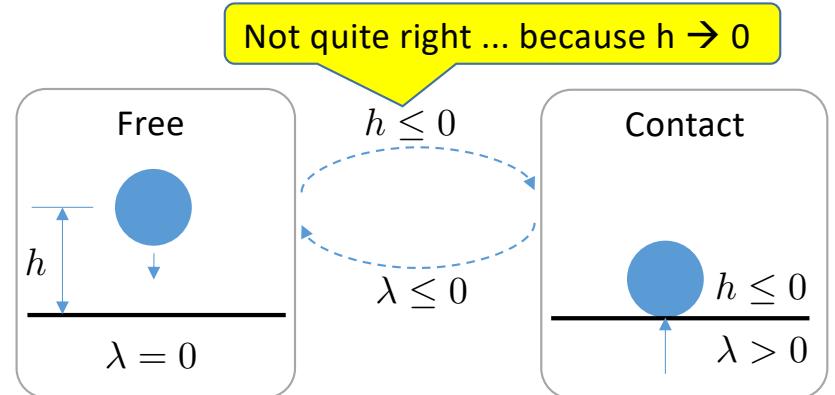
- Elastic collision...  $\ddot{h} + \alpha_1 \dot{h} + \alpha_0 h = 0$
- Tune  $\alpha_1, \alpha_0$
- Properties...
  - $h \rightarrow 0$  after collision transient – enforces penetration constraint
  - Numerically stiff for large  $\alpha_0$

So add nonlinear feedback

- Replace stabilized constraint with ...

$$\ddot{h} + \alpha_1 \dot{h} + \alpha_0 h + \alpha_3 h^3 = 0$$

- This is stiff during collision, but soft in steady-state
- Replace switching condition with FSM
  - Because  $h \rightarrow 0$



# Contact and Collision Model Idea

- Dynamics

$\dot{q} = v$

$\lambda = 0 \text{ Free}$   
 $\lambda > 0 \text{ Contact}$

$$M(q)\dot{v} + C(q, v) + G(q) = \lambda^T H(q) + Bu$$

$$h''(q, v, \lambda) + \alpha_1 h'(q, v) + \alpha_0 h(q) = 0$$

This can model elastic collision

- Elastic collision...  $\ddot{h} + \alpha_1 \dot{h} + \alpha_0 h = 0$

Tune  $\alpha_1, \alpha_0$

- Properties...

- $h \rightarrow 0$  after collision transient – enforces penetration constraint

- Numerically stiff for large  $\alpha_0$

So add nonlinear feedback

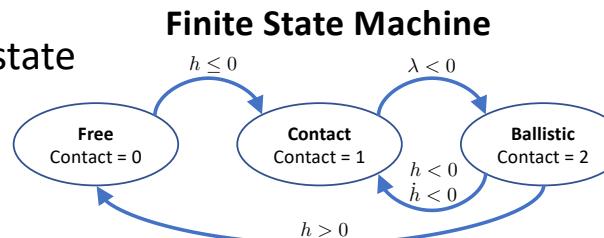
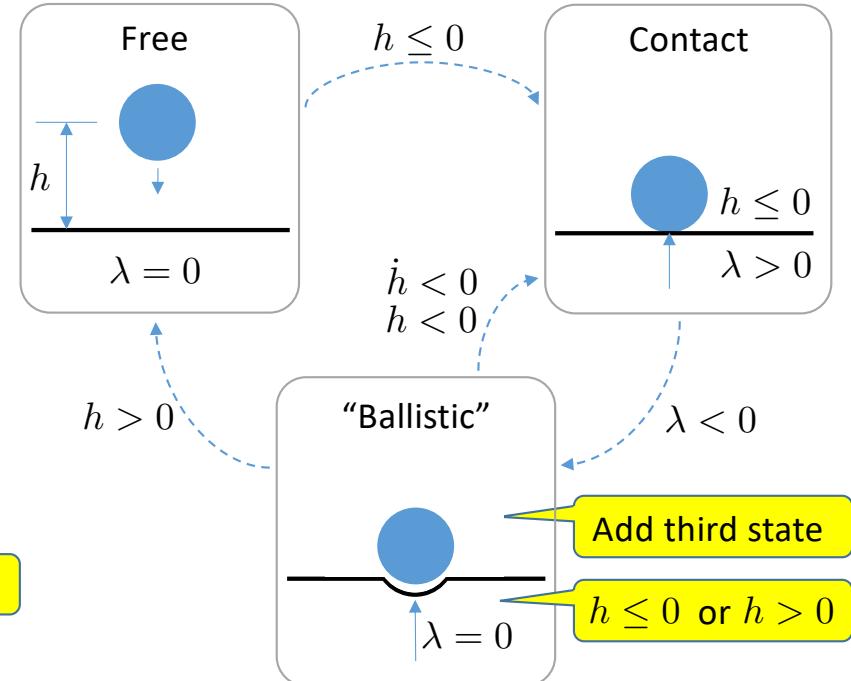
- Replace stabilized constraint with ...

$$\ddot{h} + \alpha_1 \dot{h} + \alpha_0 h + \alpha_3 h^3 = 0$$

- This is stiff during collision, but soft in steady-state

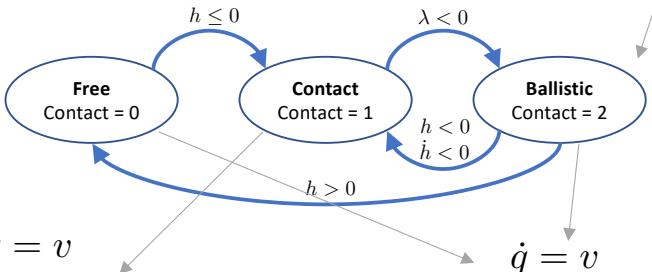
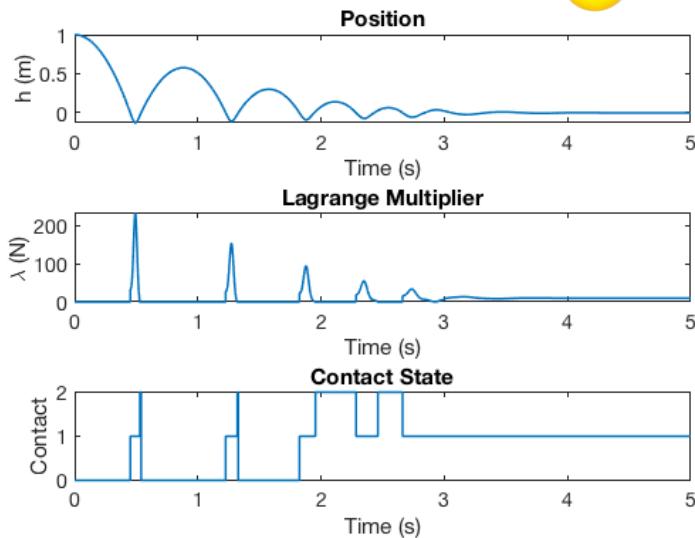
- Replace switching condition with FSM

- Because  $h \rightarrow 0$



# Bouncing Ball Example

## Example 1: Positive Damping



$$\begin{aligned} \dot{q} &= v \\ m\dot{v} &= -mg + \lambda \\ 0 &= \ddot{h} + \alpha_1 \dot{h} + \alpha_0 h + \alpha_3 h^3 \end{aligned}$$

$$\begin{aligned} \dot{q} &= v \\ m\dot{v} &= -mg \\ \lambda &= 0 \end{aligned}$$

```
model myBouncingBall
```

```

Real q(start=1.0), v(start=0.0), f;
Real h, hDot, hDotDot, lambda(start=0);
discrete Integer contact(start=0);
Boolean b1, b2, b3, b4;
parameter Real g=9.81, m=1.0;
parameter Real a0=100, a1=20, a2=1e6;
parameter Boolean linFlag = false;

algorithm
    b1 := h <= 0;
    b2 := lambda < 0.0;
    b3 := h > 0.0;
    b4 := h <= 0 and hDot < 0;
    if edge(b1) and contact == 0 then
        contact := 1;
    end if;
    if contact == 1 and edge(b2) then
        contact := 2;
    end if;
    if contact == 2 and edge(b3) then
        contact := 0;
    end if;
    if contact == 2 and edge(b4) then
        contact := 1;
    end if;

equation
    if contact == 1 or linFlag then
        0 = hDotDot + a1*hDot + a0*h + a2*h^3;
    else
        lambda = 0.0;
    end if;

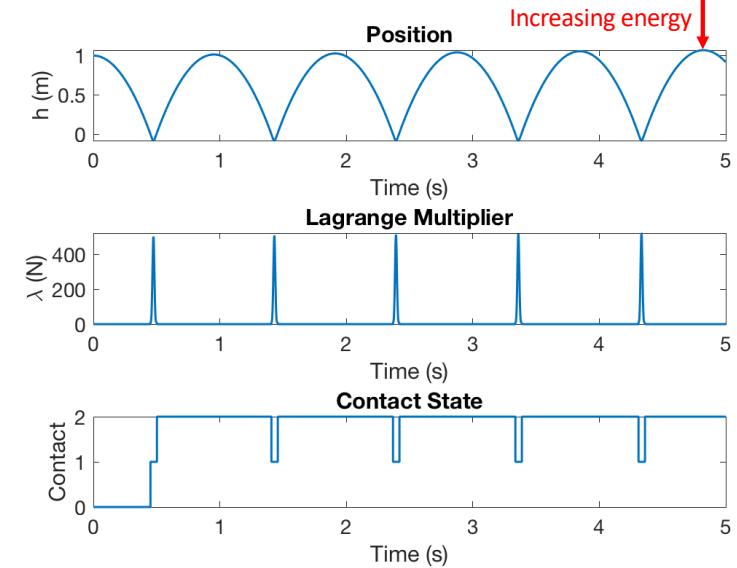
    f = if linFlag then lambda
        else contact*lambda;

    der(q) = v;
    m * der(v) = -m * g + f;

    h = q;
    hDot = der(h);
    hDotDot = der(hDot);

end myBouncingBall;
```

## Example 2: No Damping



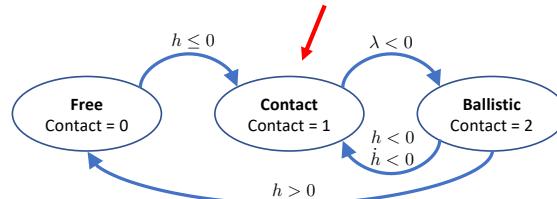
# What's Going On? A Failure to Conserve Energy. D' Oh!

- Contact State

$$\dot{q} = v$$

$$m\dot{v} = -mg + \lambda$$

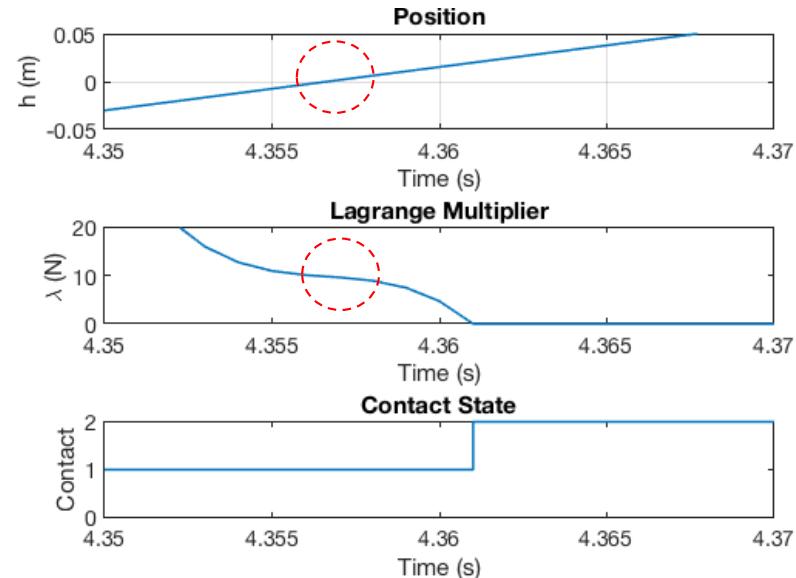
$$0 = \ddot{h} + \alpha_1 \dot{h} + \alpha_0 h + \alpha_3 h^3$$



- Solve for Lagrange Multiplier...

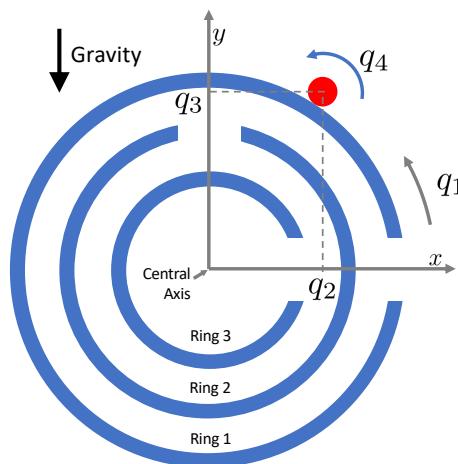
$$\lambda(t) = (\alpha_0 q(t) + \alpha_2 q^3(t) + g)m.$$

- If this were a “spring force,” the  $g$  would not appear
- This force adds energy to the ball when  $\lambda > 0$  and  $h > 0$
- This force is responsible for
  - $h \rightarrow 0$  (no penetration in steady state) ... Good
  - Failure to conserve energy ... Bad
- But, its not *that* bad
  - Energy added only when  $\lambda > 0$  and  $\dot{h} \neq 0$
  - In typical applications,  $\alpha_1 > 0$  (and  $\alpha_1 = 0$  isn't stable)
  - Many “physics engines” also fail to conserve energy!



# Ball Maze Game

- Strategy: Tip maze to constant angle, rotate maze about axis
- Open-loop unstable against rings. Stabilize with feedback.



## Dynamics

$$M\ddot{q} + G = H^T(q, i)\lambda + Bu$$

$$\ddot{h}(q, \dot{q}, \lambda, i) + \alpha_1 \dot{h}(q, \dot{q}, i) + \alpha_0 h(q, i) = 0$$

$$M = \text{diag}(J_m, m_b, m_b, J_b),$$

$$G = [0 \ 0 \ gm_b \ 0]^T,$$

$$B = [1 \ 0 \ 0 \ 0]^T,$$

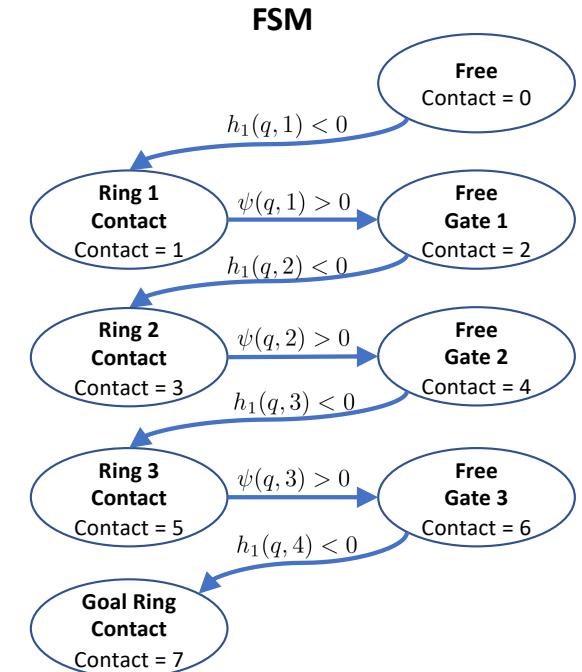
$$H(q, i) = \frac{\partial h(q, i)}{\partial q},$$

## Constraints

$$h_1(q, i) = q_1^2 + q_2^2 + (r_{mi} + r_b)^2 \quad \leftarrow \textbf{Ring}$$

$$h_2(q, i) = r_b(q_4 - q_{40}) + r_{mi}(q_1 - q_{10}) \quad \leftarrow \textbf{Rolling}$$

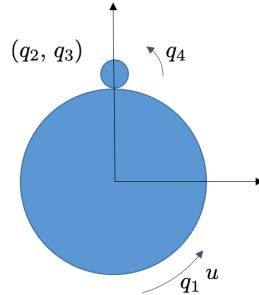
$$+ r_{mi}(\text{atan}(q_2/q_3) - \text{atan}(q_{20}/q_{30}))$$



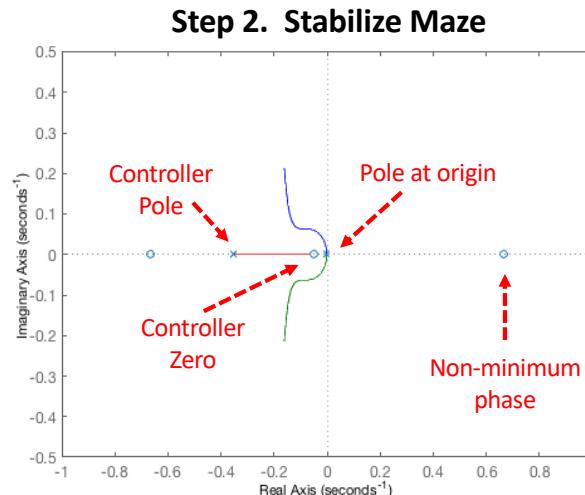
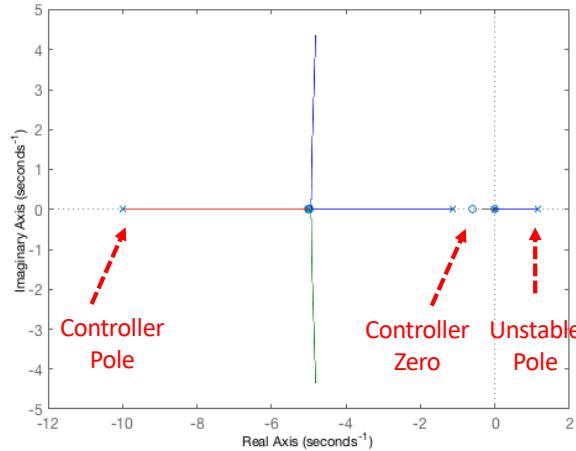
# Ball Maze Lead-Compensator Control Design

1. Stabilize ball with lead compensator  $k_b \frac{1 + s/\omega_1}{1 + s/\omega_2}$

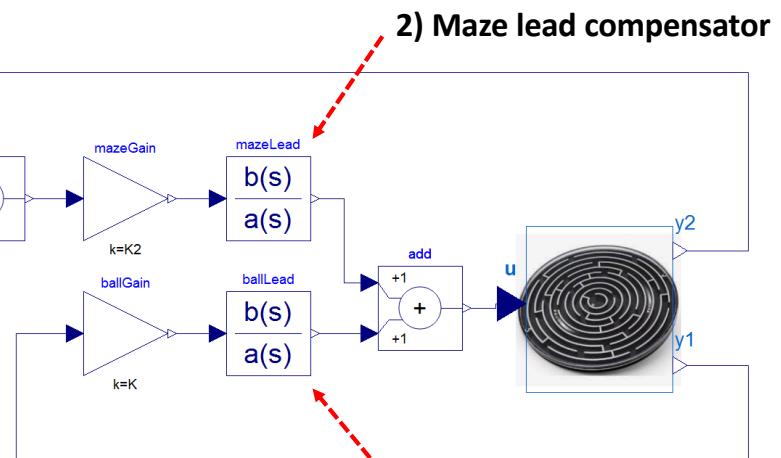
2. Stabilize maze with lead compensator  $k_m \frac{1 + s/\omega_3}{1 + s/\omega_4}$



**Step 1. Stabilize Ball**



Note Fundamental Limits:  $\begin{cases} \text{Lower bound on } k_b \\ \text{Upper bound on } k_m \end{cases}$



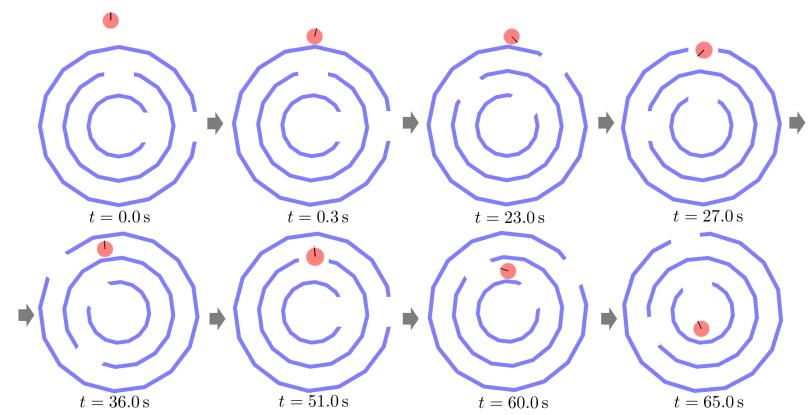
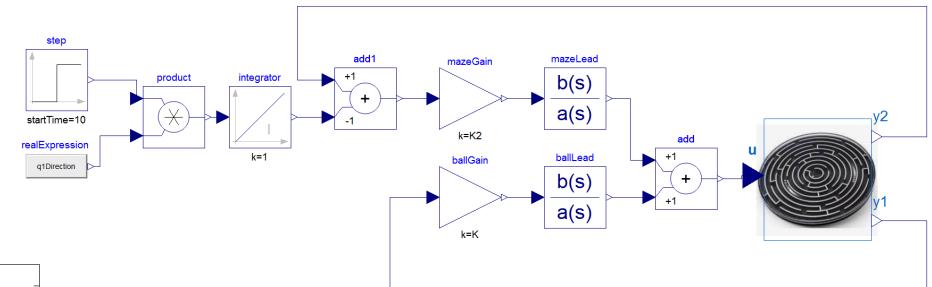
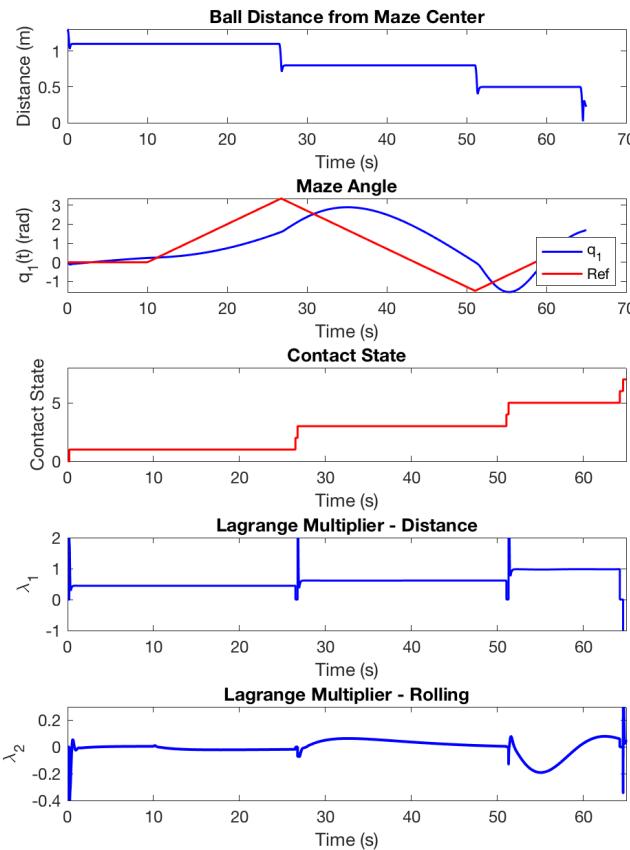
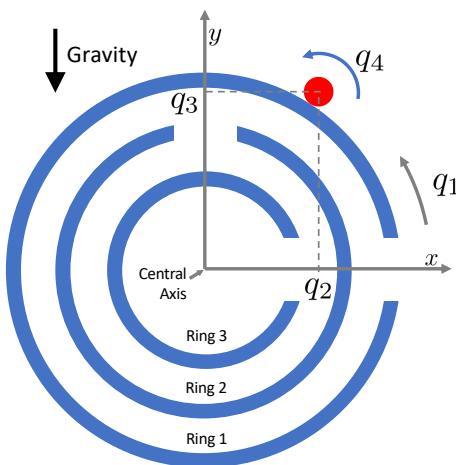
**2) Maze lead compensator**

**1) Ball lead compensator**

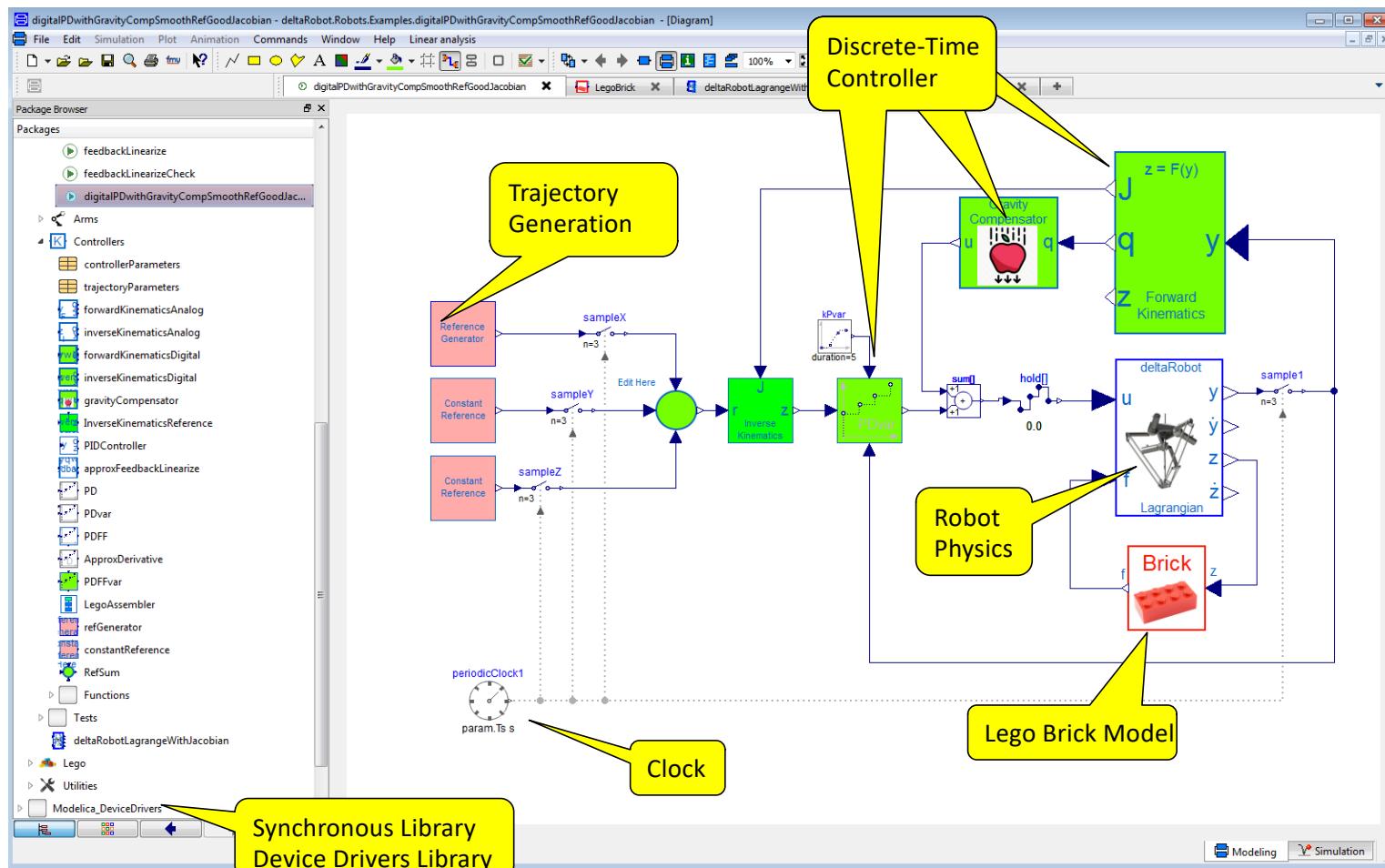


# Ball – Maze Simulation Results

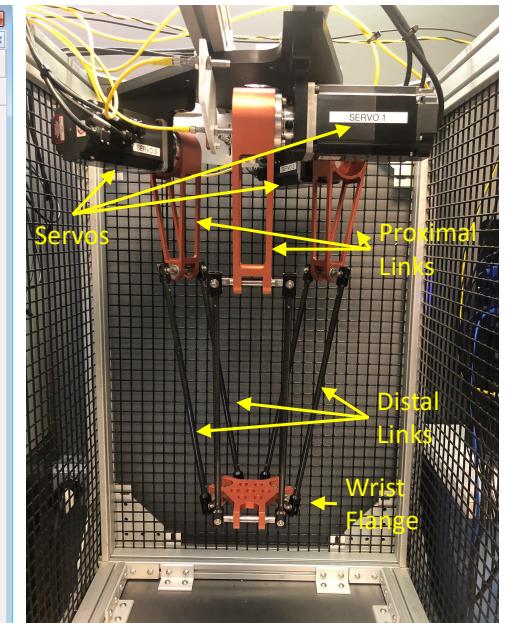
- Feedback control to stabilize ball
- Rotate maze to bring gate sequence under ball



# Soft-Touch Robotic Control

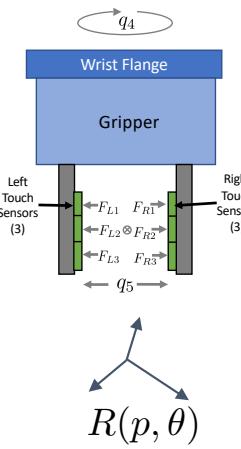
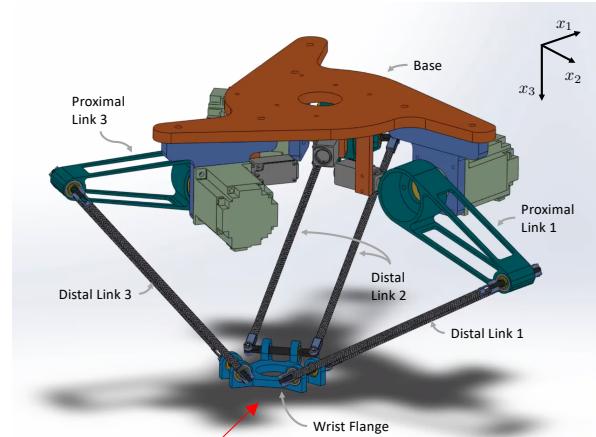
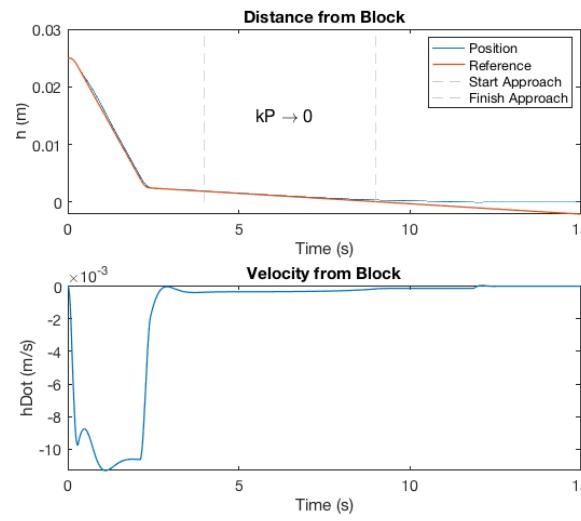
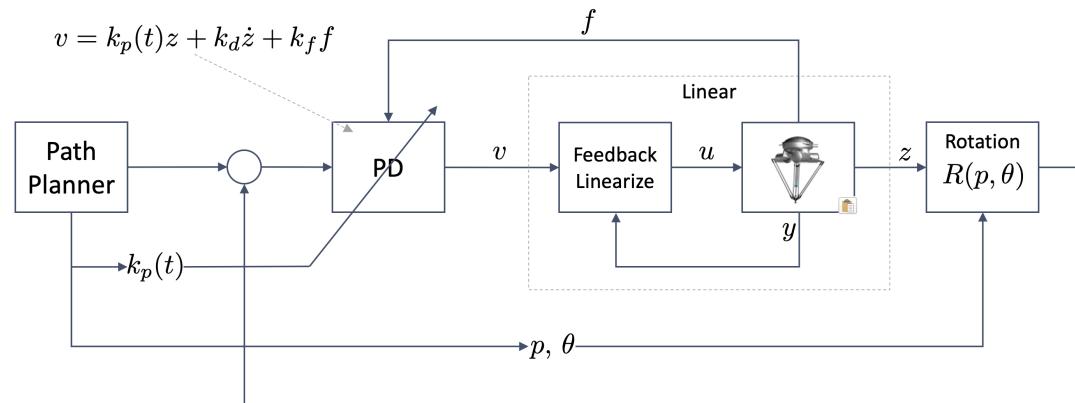


MERL's Kamaji



# Simulation Example – Soft-Contact

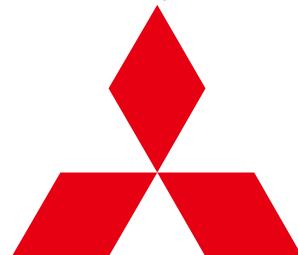
Task Planner





## Summary & Conclusions

- Hybrid DAE – FSM model of rigid body contact and collision
- Consistent mathematics – modeling with separation of concerns e.g. solver
- Native Modelica enables analysis beyond simulation. But can FSM be improved?
- Useful for some types of contact
  - Low dimensions
  - Positive damping, but not too stiff
- Caveat: Event based method – critically dependent on event detection. (Can fail.)
- Code for Ball Maze, bouncing ball available. Email: bortoff@merl.com

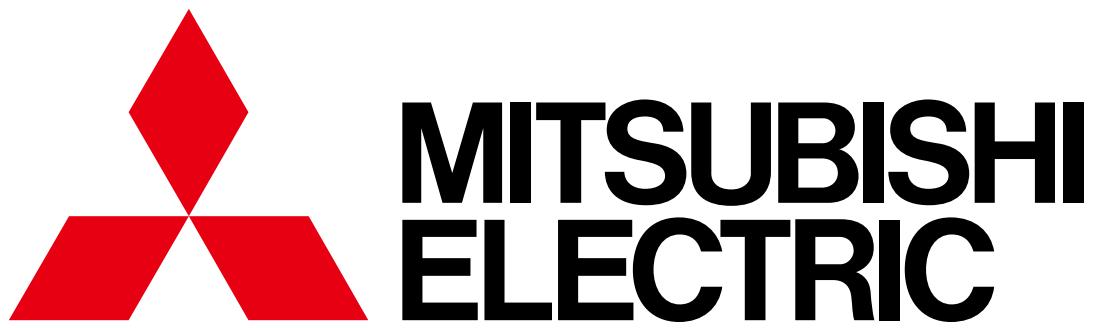


**MITSUBISHI**  
**ELECTRIC**

*Changes for the Better*

Thank – you!

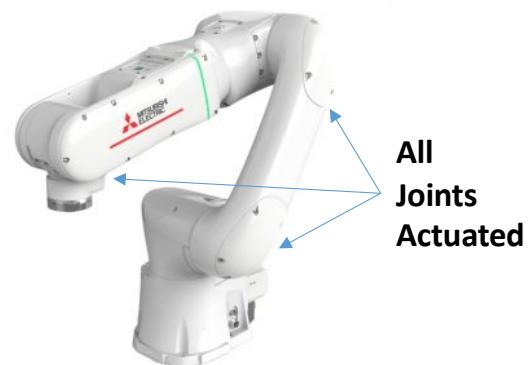




*Changes for the Better*

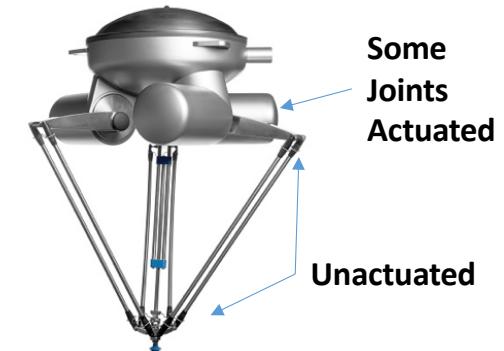
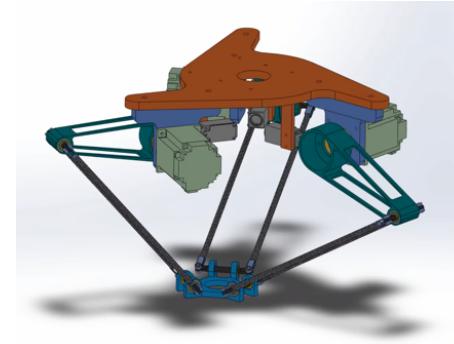
# Serial-Link Vs. Delta Robots – Mechanical Design

## Not So Good for Assembly (Contact) Applications



**Open Chain**  
**High Mass, High Inertia → Slow**  
**High Ratio Gears**  
**Small Servos, High Friction, Not Back-Drivable, although...**  
**High Impedance – Stiff**  
**Low payload to robot mass ratio ~ O(0.1)**  
**Coupled Force / Torque**  
**High Precision Mechanical Position Control**  
**High Joint & Link Bending**  
**Larger work volume**  
**More expensive**  
**Pick and Place, Assembly Applications**

## Good for Assembly (Contact) Applications

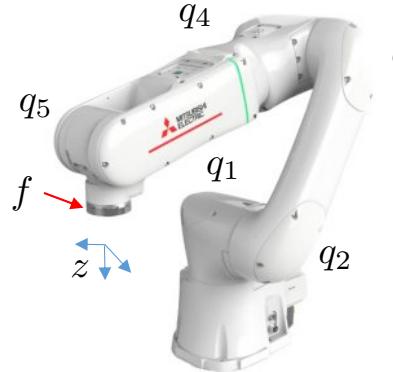


**Complex Closed Chain**  
**Low Mass, Inertia → Fast**  
**Direct Drive, although...**  
**Big Servos, Low Friction, Back-Drivable**  
**Low Impedance – Soft**  
**High payload to robot mass ratio ~ O(10)**  
**Decoupled Force / Torque**  
**High Precision Mechanical Position Control**  
**Low Joint & Link Bending**  
**Smaller work volume, although...**  
**Less expensive**  
**Pick and Place Applications**

- Volume: 1.5 m x 0.4 m
- Payload < 100kg
- Max. 250 ops / min

# Serial-Link Vs. Delta Robot – Kinematics, Dynamics

## Open Chain – Conventional Calculations



$q \in \mathcal{R}^5$  Joint Angles  
 $y = q$  Measurements  
 $u \in \mathcal{R}^5$  Inputs (Co-located)  
 $z \in \mathcal{R}^3$  End effector

## Forward Kinematics – Explicit (analytic, closed-form)

$$z = F(q)$$

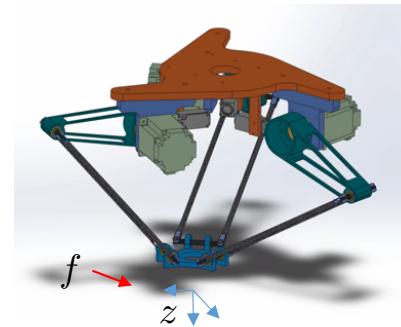
## Jacobian – Explicit (analytic closed form)

$$J(q) = \frac{\partial F(q)}{\partial q} \quad \rightarrow \quad \tau = J^T(q)f$$

## Dynamics – ODE (all analytic, closed-form)

$$M(q)\ddot{q} + C(q, \dot{q}) + D(\dot{q}) + G(q) = u + \tau$$

## Kinematic Loop – Conventional Becomes Difficult Unless we are unconventional !



$q \in \mathcal{R}^9$  Joint Angles  
 $y \in \mathcal{R}^3$  Measurements  
 $u \in \mathcal{R}^3$  Inputs

## Forward Kinematics – Implicit (no Closed Form)

$$F(q, z) = 0 \quad (6 \text{ equations, 6 unknowns, solved numerically})$$

## Conventional Jacobian – Implicit if expressed in $y$ ...

$$J(y) = \frac{\partial f(y)}{\partial y} \quad (\text{computed by Implicit Function Theorem, evaluated numerically. Used for control.})$$

## BUT, Jacobian is explicit if expressed in $q$

$$J(q) = \frac{\partial f}{\partial q} \quad (\text{maps forces at end effector to virtual torques at all 9 joints. Used for simulation.})$$

## So Dynamics – Loops in Loops - are NOT a problem

$$M(y)\ddot{y} + C(y, \dot{y}) + D(y) + G(y) = u + \tau \quad \tau = J^T(q)f$$

# Modelica Realization of the DAE Model

Declare 3 arms

$$\dot{q}_i = v_i$$

$$m(q_i)\ddot{v}_i + c(q_i, v_i) + g(q_i) = b\tau$$

Declare Lagrange Multiplier

Declare Constraint

Constant Rotation Matrices

## model deltaRobotLagrange

```
Arms.deltaRobotArmLagrange arm1, arm2, arm3;
Real lambda[6]; // Lagrange multiplier
Real h0[6], h1[6], h2[6];
Input Real u[3], f[3]; // torque inputs, force inputs
parameter Real POLE = 5.0;

constant Real Rot2[3,3] = Utilities.RotZ(2.0*PI/3.0);
constant Real Rot3[3,3] = Utilities.RotZ(-2.0*PI/3.0);
constant Real B[3] = {1, 0, 0}; // input torque vector
```

## equation

```
arm1.tau = transpose(arm1.dh) * lambda[1:3] + transpose(arm1.dh) * lambda[4:6] + transpose(dz1) * f + B * u[1];
arm2.tau = -transpose(Rot2 * arm2.dh) * lambda[1:3] + B * u[2] + transpose(dz2) * f;
arm3.tau = -transpose(Rot3 * arm3.dh) * lambda[4:6] + B * u[3] + transpose(dz3) * f;

h0 = cat(1,arm1.psi-Rot2*arm2.psi,arm1.ps -Rot3*arm3.psi);
h1 = der(h0);
h2 = der(h1);
zeros(6) = h2 + 2.0 * POLE * h1 + POLE^2 * h0;
```

$$\dot{q} = v$$

$$M(q)\ddot{v} + C(q, v) + D(v) + G(q) = H^T(q)\lambda + B(u + \tau_u) + \tau_v$$

$$\ddot{h}(q, v, \dot{v}) + \alpha_1 \dot{h}(q, v) + \alpha_0 h(q) = 0$$

Servo Angle Measurements  $y, \dot{y}$

```
y = cat(1, vector(arm1.q[1]), vector(arm2.q[1]), vector(arm3.q[1]));
yDot = cat(1, vector(arm1.v[1]), vector(arm2.v[1]), vector(arm3.v[1]));
```

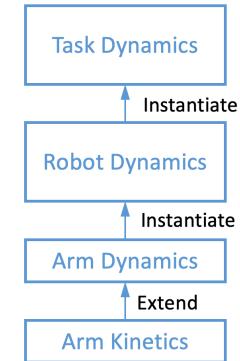
```
dz1 = Controllers.Functions.armJacobian(arm1.q) / 3.0;
dz2 = R2*Controllers.Functions.armJacobian(arm2.q) / 3.0;
dz3 = R3*Controllers.Functions.armJacobian(arm3.q) / 3.0;
```

```
Jv = cat(2, dz1, dz2, dz3);
```

Virtual Jacobian  $J_v$

```
z = arm1.h;
zDot = Jv * cat(1, vector(arm1.v), vector(arm2.v), vector(arm3.v));
```

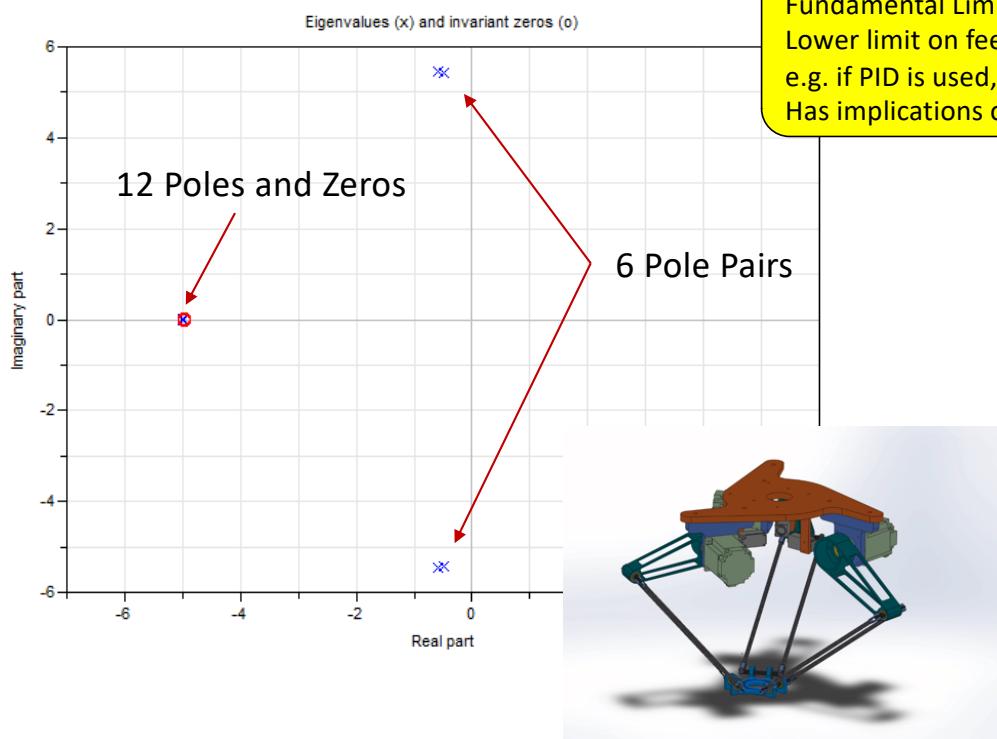
end deltaRobotLagrange;



# Simple Example of Dynamic Analysis (Robot is Open-Loop Unstable)

## Stable Configuration

$$q_{i1} = \frac{\pi}{4} \text{ rad} = 45^\circ$$



## Unstable Configuration

$$q_{i1} = 0 \text{ rad} = 0^\circ$$

