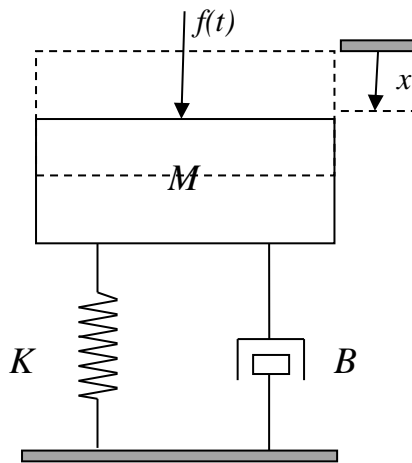

4. State-space representation

4.1 Introduction

- When the differential equations governing the system are linear, the model can be described in terms of a Laplace transfer function.
- Consider for example the spring-mass damper system:



Physical model

$$M \frac{d^2 x}{dt^2} + B \frac{dx}{dt} + Kx = f(t)$$

$$G(s) = \frac{X(s)}{F(s)} = \frac{1}{Ms^2 + Bs + K}$$

- While transfer functions are compact and have several advantages in relation to analysis of dynamical systems, they also have a number of weaknesses as follows:
 - They do not handle initial conditions (assumed to be zero).
 - Information about internal variables is lost ($\frac{dx}{dt}$ in the case of the above example).
 - For general m -input, p -output systems, we would need a total of $m \times p$ transfer functions to fully describe the system.
- An alternative model representation, known as a **state-space model**, overcomes these weaknesses.
- The basic idea behind state-space modelling is to write down a set of first-order differential equations in terms of the system state(s) and input(s).

- **Example 4.1:** Develop a state-space model for the second-order differential equation model of the spring-mass damper system:

$$M \frac{d^2 x}{dt^2} + B \frac{dx}{dt} + Kx = f(t)$$

Solution

We proceed as follows:

Firstly we have a **second-order** differential equation; hence we define two states x_1 and x_2 :

$$x_1 = x, \quad x_2 = \frac{dx}{dt}$$

When we do it this way we get the CONTROLLABLE CANONICAL FORM state-space model.

We then obtain expressions for $\frac{dx_1}{dt}$ and $\frac{dx_2}{dt}$:

$$\frac{dx_1}{dt} = \frac{dx}{dt} \quad \frac{dx_2}{dt} = \frac{d^2 x}{dt^2} = \frac{1}{M} \left(f(t) - B \frac{dx}{dt} - Kx \right)$$

Note that the latter expression is obtained from the original mathematical model.

We can rewrite these equations in terms of the states x_1 and x_2 only, by replacing x and $\frac{dx}{dt}$ with their state equivalent. Hence:

$$\frac{dx_1}{dt} = x_2 \quad \frac{dx_2}{dt} = \frac{1}{M} (f(t) - Bx_2 - Kx_1)$$

Now, we combine these into matrix form as follows:

$$\frac{d}{dt} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -\frac{K}{M} & -\frac{B}{M} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{M} \end{bmatrix} f(t)$$

Finally, let us define the output as $y = x$. Hence $y = x_1$ giving:

$$[y] = [1 \quad 0] \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + [0]f(t) \Rightarrow [y] = [1 \quad 0] \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$$

Our state-space model is expressed, in full, as:

$$\frac{d}{dt} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -\frac{K}{M} & -\frac{B}{M} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{M} \end{bmatrix} f(t)$$

$$[y] = [1 \quad 0] \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$$

4.2 Formal definitions

- The commonly used terms associated with state-space representation are as follows:
- **State** – the state of a dynamic system is the smallest set of variables (called state variables) such that knowledge of these variables at $t = t_0$, together with knowledge of the input for $t \geq t_0$, completely determines the behaviour of the system for $t \geq t_0$.
- **State variables** – the variables that make up the state as defined above.
- **State vector** – when there is more than one state variable, they are normally collected together into a vector called a state vector.
- **State-space** – the n -dimensional state vector can be viewed as a point moving around in n -dimensional space. This n -dimensional space is known as state-space.

General state-space form

- The dynamics of a general n^{th} order linear dynamical system, with m inputs, is completely described by a n^{th} order state-space equation of the form:

$$\frac{d}{dt} \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_n \end{bmatrix} = \begin{bmatrix} a_{11} & a_{12} & \cdots & a_{1n} \\ a_{21} & a_{22} & \cdots & a_{2n} \\ \vdots & \vdots & \ddots & \vdots \\ a_{n1} & a_{n2} & \cdots & a_{nn} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_n \end{bmatrix} + \begin{bmatrix} b_{11} & \cdots & b_{1m} \\ b_{21} & \cdots & b_{2m} \\ \vdots & \ddots & \vdots \\ b_{n1} & \cdots & b_{nm} \end{bmatrix} \begin{bmatrix} u_1 \\ \vdots \\ u_m \end{bmatrix}$$
$$\equiv \frac{d\mathbf{x}}{dt} = \mathbf{Ax} + \mathbf{Bu} \quad \text{state equation}$$

with a set of initial conditions (one for each state):

$$\mathbf{x}(0) = \begin{bmatrix} x_1(0) \\ x_2(0) \\ \vdots \\ x_n(0) \end{bmatrix} = \mathbf{x}_0 \quad \text{initial conditions}$$

- The model output(s) are given by a linear combination of the states and the inputs. Given p outputs, we obtain:

$$\begin{bmatrix} y_1 \\ y_2 \\ \vdots \\ y_p \end{bmatrix} = \begin{bmatrix} c_{11} & c_{12} & \cdots & c_{1n} \\ c_{21} & c_{22} & \cdots & c_{2n} \\ \vdots & \vdots & \ddots & \vdots \\ c_{p1} & c_{p2} & \cdots & c_{pn} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_n \end{bmatrix} + \begin{bmatrix} d_{11} & \cdots & d_{1m} \\ d_{21} & \cdots & d_{2m} \\ \vdots & \ddots & \vdots \\ d_{p1} & \cdots & d_{pm} \end{bmatrix} \begin{bmatrix} u_1 \\ \vdots \\ u_m \end{bmatrix}$$

$$\equiv \quad \mathbf{y} = \mathbf{C}\mathbf{x} + \mathbf{D}\mathbf{u} \quad \text{output equation}$$

- Together, these two equations describe the state-space model of a system.

$$\frac{d\mathbf{x}}{dt} = \mathbf{A}\mathbf{x} + \mathbf{B}\mathbf{u} \quad , \mathbf{x}(0) = \mathbf{x}_0$$

$$\mathbf{y} = \mathbf{C}\mathbf{x} + \mathbf{D}\mathbf{u}$$

- The matrices \mathbf{A} , \mathbf{B} , \mathbf{C} and \mathbf{D} are called the state matrix, input matrix, output matrix and direct transmission matrix respectively:

\mathbf{A}	-	state matrix ($n \times n$)
\mathbf{B}	-	input matrix ($n \times m$)
\mathbf{C}	-	output matrix ($p \times n$)
\mathbf{D}	-	direct transmission matrix ($p \times m$)

4.3 State – space representations are not unique

State-space representations are NOT unique.

Consider the following system

$$\frac{d^2y(t)}{dt^2} + 4\frac{dy(t)}{dt} + 3y(t) = 2f(t)$$

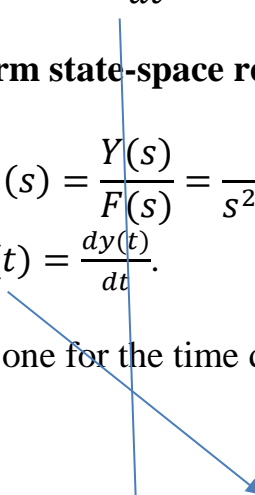
Controllable canonical form state-space representation

Form the transfer function

$$G(s) = \frac{Y(s)}{F(s)} = \frac{2}{s^2 + 4s + 3}$$

Let $x_{1c}(t) = y(t)$ and $x_{2c}(t) = \frac{dy(t)}{dt}$.

One needs two equations – one for the time derivative of state 1 and one for the time derivative of state 2.


$$\frac{dx_{1c}}{dt} = \frac{dy(t)}{dt} = x_{2c}$$

$$\begin{aligned}\frac{d^2y}{dt^2} + 4\frac{dy}{dt} + 3y &= 2f \\ \frac{d^2y}{dt^2} &= -4\frac{dy}{dt} - 3y + 2f\end{aligned}$$

Now put in the states

$$\frac{d^2y}{dt^2} = \frac{dx_{2c}}{dt} = -4x_{2c} - 3x_{1c} + 2f$$

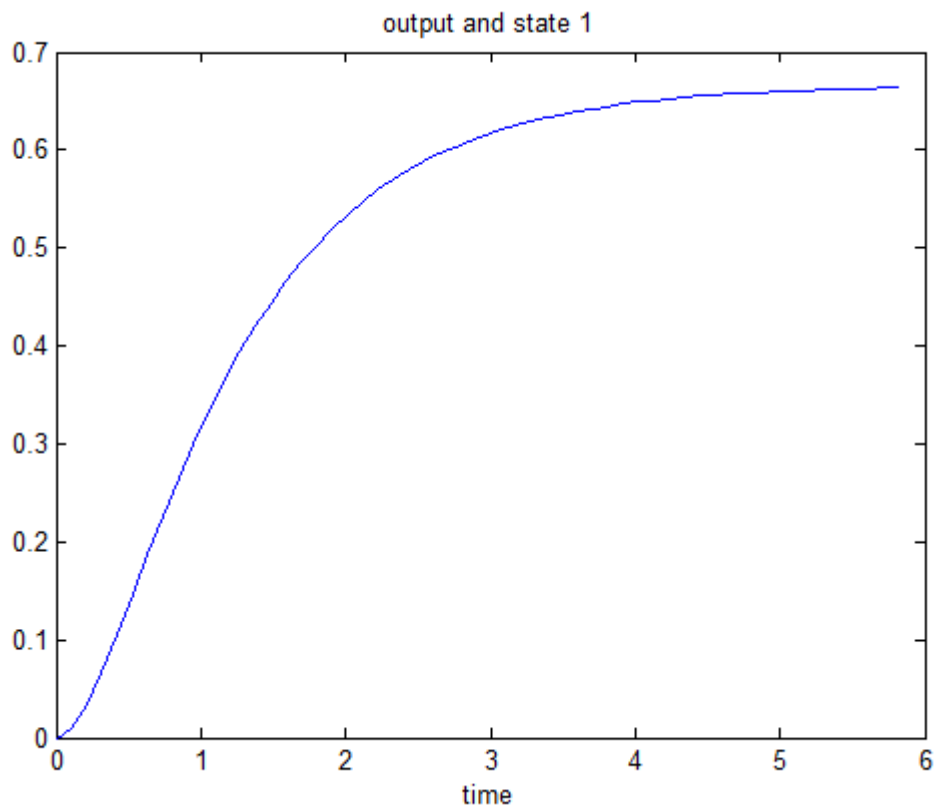
Put in matrix formation

$$\frac{d}{dt} \begin{bmatrix} x_{1c}(t) \\ x_{2c}(t) \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -3 & -4 \end{bmatrix} \begin{bmatrix} x_{1c}(t) \\ x_{2c}(t) \end{bmatrix} + \begin{bmatrix} 0 \\ 2 \end{bmatrix} f(t)$$

The output is $y(t)$ which is the first state

$$y(t) = \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} x_{1c}(t) \\ x_{2c}(t) \end{bmatrix}$$

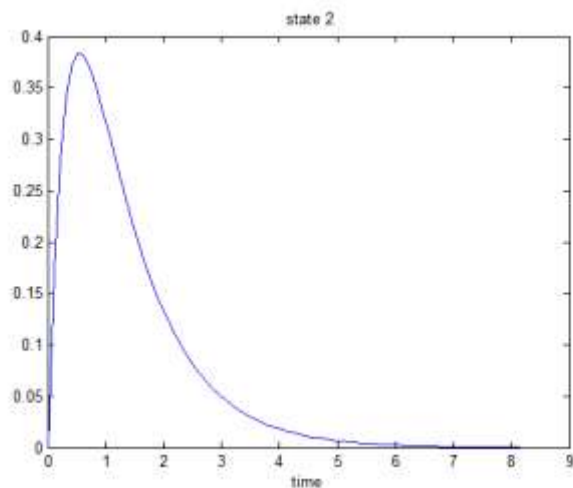
The unit-step response is as follows



Suppose you wanted to see what the second state looks like

$$o(t) = \begin{bmatrix} 0 & 1 \end{bmatrix} \begin{bmatrix} x_{1c}(t) \\ x_{2c}(t) \end{bmatrix}$$

Then you could plot this.

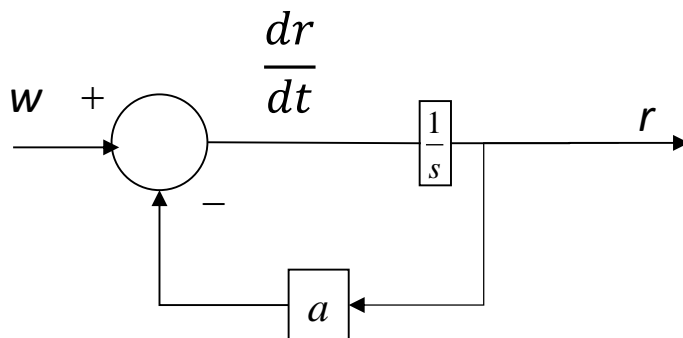


Series form representation

Express $G(s)$ as the product of $\frac{1}{s+a}$ terms.

$$G(s) = \frac{2}{s^2 + 4s + 3} = 2 \left(\frac{1}{s+1} \right) \left(\frac{1}{s+3} \right)$$

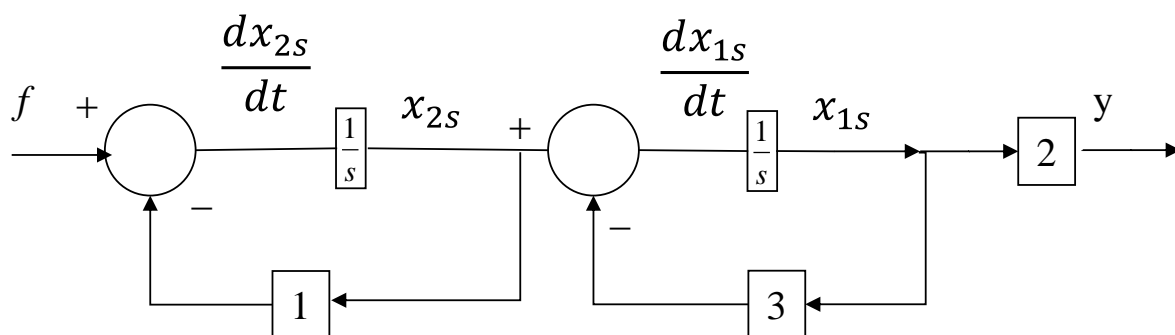
The block diagram for $\frac{1}{s+a}$ is



$$\begin{aligned} \frac{dr}{dt} &= w - ar \\ sR(s) &= W(s) - aR(s) \\ \frac{R(s)}{W(s)} &= \frac{1}{s+a} \end{aligned}$$

So $G(s)$ can be represented in block diagram form as follows

$$G(s) = \frac{2}{s^2 + 4s + 3} = 2 \left(\frac{1}{s+1} \right) \left(\frac{1}{s+3} \right)$$



$$\frac{dx_{1s}}{dt} = -3x_{1s} + x_{2s}$$

$$\begin{aligned}\frac{dx_{2s}}{dt} &= -x_{2s} + f \\ y &= 2x_{1s}\end{aligned}$$

Now write the state-space representation in matrix form

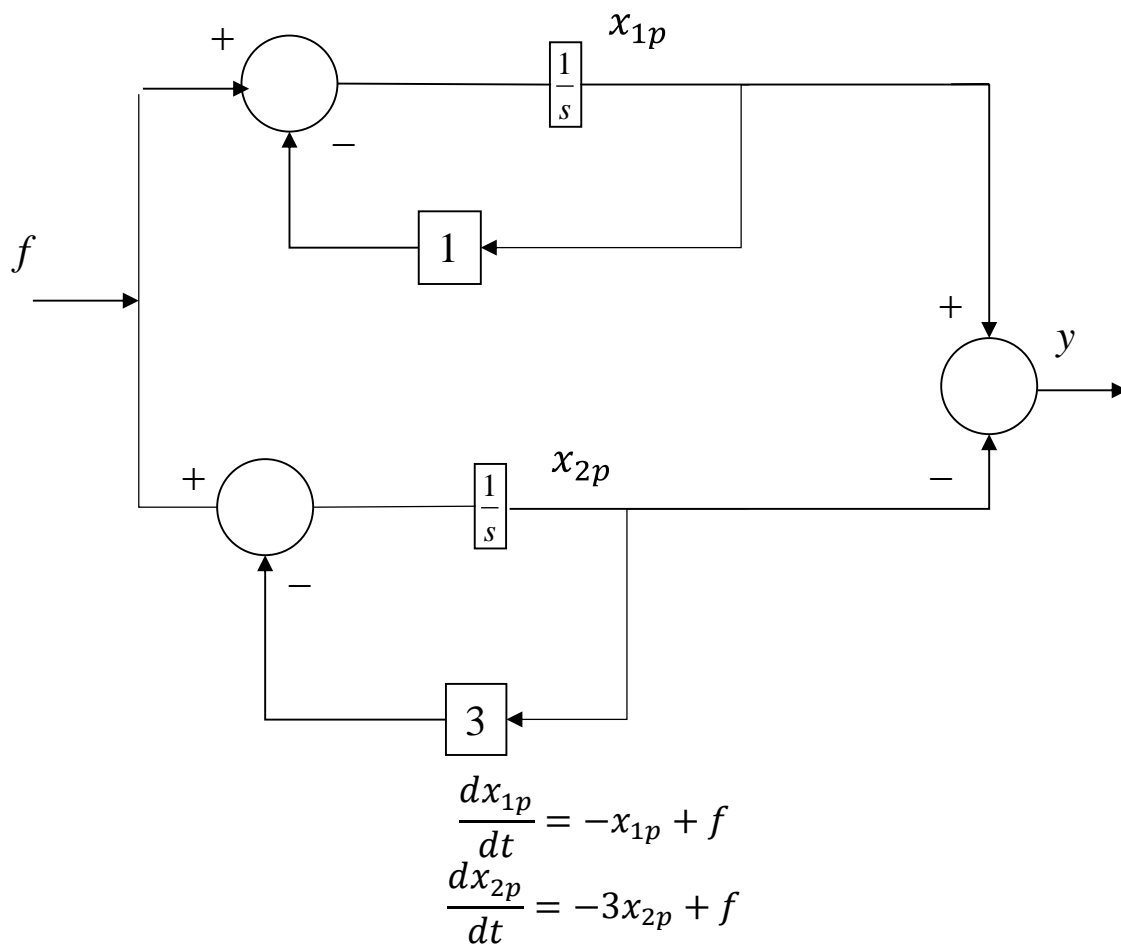
$$\begin{aligned}\frac{d}{dt} \begin{bmatrix} x_{1s}(t) \\ x_{2s}(t) \end{bmatrix} &= \begin{bmatrix} -3 & 1 \\ 0 & -1 \end{bmatrix} \begin{bmatrix} x_{1s}(t) \\ x_{2s}(t) \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} f(t) \\ y(t) &= [2 \quad 0] \begin{bmatrix} x_{1s}(t) \\ x_{2s}(t) \end{bmatrix}\end{aligned}$$

Parallel form state-space representation

Express $G(s)$ as the sum or difference of $\frac{1}{s+a}$ terms.

$$G(s) = \frac{2}{s^2 + 4s + 3} = \left(\frac{1}{s+1} \right) - \left(\frac{1}{s+3} \right)$$

This can be shown in block diagram format as follows



$$\frac{d}{dt} \begin{bmatrix} x_{1p}(t) \\ x_{2p}(t) \end{bmatrix} = \begin{bmatrix} -1 & 0 \\ 0 & -3 \end{bmatrix} \begin{bmatrix} x_{1p}(t) \\ x_{2p}(t) \end{bmatrix} + \begin{bmatrix} 1 \\ 1 \end{bmatrix} f(t)$$
$$y(t) = \begin{bmatrix} 1 & -1 \end{bmatrix} \begin{bmatrix} x_{1p}(t) \\ x_{2p}(t) \end{bmatrix}$$

Note that the state matrix is diagonal. This is important for computations as solution of the state equations (to be dealt with in a later lecture) involves exponentials of the state matrix and it is much easier to get the exponential of a diagonal matrix.

4.4 Transforming system models to state-space form

Derivation of the state-space model from the transfer function model of the system.

4.4.1 Transfer function → State space

Case 1: when the input does *not* involve derivatives

- Consider the following third-order transfer function:

$$G(s) = \frac{Y(s)}{U(s)} = \frac{k}{s^3 + a_2 s^2 + a_1 s + a_0}$$

- Converting this to the time domain gives:

$$\frac{d^3 y}{dt^3} + a_2 \frac{d^2 y}{dt^2} + a_1 \frac{dy}{dt} + a_0 y = ku$$

or

$$\frac{d^3 y}{dt^3} = -a_2 \frac{d^2 y}{dt^2} - a_1 \frac{dy}{dt} - a_0 y + ku$$

- We simply define the states as $x_1 = y$, $x_2 = \frac{dy}{dt}$ and $x_3 = \frac{d^2 y}{dt^2}$, leading to the following state model:

$$\frac{dx_1}{dt} = \frac{dy}{dt} = x_2$$

$$\frac{dx_2}{dt} = \frac{d^2 y}{dt^2} = x_3$$

$$\frac{dx_3}{dt} = \frac{d^3 y}{dt^3} = -a_2 x_3 - a_1 x_2 - a_0 x_1 + ku$$

$$y = x_1$$

giving:

$$\frac{d}{dt} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -a_0 & -a_1 & -a_2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ k \end{bmatrix} u$$

$$y = \begin{bmatrix} 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

4.4.2 Transfer function → State space

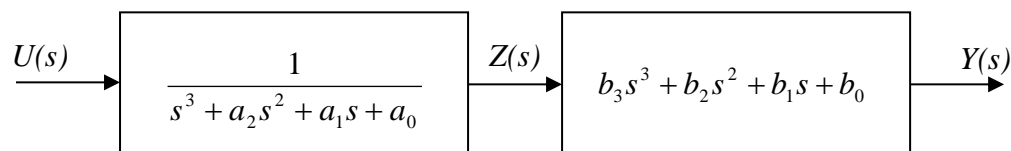
Case 2: when the input does involve derivatives

- Consider the following third-order transfer function:

$$G(s) = \frac{Y(s)}{U(s)} = \frac{b_3 s^3 + b_2 s^2 + b_1 s + b_0}{s^3 + a_2 s^2 + a_1 s + a_0}$$

- Firstly, we split the transfer function into two parts by defining an intermediate variable $Z(s)$ as follows:

$$\frac{Z(s)}{U(s)} = \frac{1}{s^3 + a_2 s^2 + a_1 s + a_0} \quad \text{and} \quad \frac{Y(s)}{Z(s)} = b_3 s^3 + b_2 s^2 + b_1 s + b_0$$



- Converting to the time domain gives:

$$\frac{d^3 z}{dt^3} + a_2 \frac{d^2 z}{dt^2} + a_1 \frac{dz}{dt} + a_0 z = u$$

and

$$b_3 \frac{d^3 z}{dt^3} + b_2 \frac{d^2 z}{dt^2} + b_1 \frac{dz}{dt} + b_0 z = y$$


-
- Setting the states as $x_1 = z$, $x_2 = \frac{dz}{dt}$ and $x_3 = \frac{d^2z}{dt^2}$, we get the state equation as follows:

$$\frac{dx_1}{dt} = x_2$$

$$\frac{dx_2}{dt} = x_3$$

$$\frac{dx_3}{dt} = -a_2x_3 - a_1x_2 - a_0x_1 + u$$

Once again, we have the **controllable canonical form**.



giving:

$$\frac{d}{dt} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -a_0 & -a_1 & -a_2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} u$$

- In terms of the output equation:

- if $b_3 = 0$ then: $y = b_2 \frac{d^2z}{dt^2} + b_1 \frac{dz}{dt} + b_0z = b_2x_3 + b_1x_2 + b_0x_1$

giving: $y = \begin{bmatrix} b_0 & b_1 & b_2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$

(Note, the D matrix is 0 in this case)

- if $b_3 \neq 0$ then: $y = b_2x_3 + b_1x_2 + b_0x_1 + b_3 \frac{d^3z}{dt^3}$

but: $\frac{d^3z}{dt^3} = \frac{dx_3}{dt} = -a_2x_3 - a_1x_2 - a_0x_1 + u$

giving:

$$y = (b_2 - b_3a_2)x_3 + (b_1 - b_3a_1)x_2 + (b_0 - b_3a_0)x_1 + b_3u$$

therefore:

$$y = \begin{bmatrix} (b_0 - b_3 a_0) & (b_1 - b_3 a_1) & (b_2 - b_3 a_2) \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + [b_3] u$$

- Note that the state matrix (i.e. matrix A) is exactly the same for both types of transfer function models.

$$G(s) = \frac{Y(s)}{U(s)} = \frac{k}{s^3 + a_2 s^2 + a_1 s + a_0}$$

$$G(s) = \frac{Y(s)}{U(s)} = \frac{b_3 s^3 + b_2 s^2 + b_1 s + b_0}{s^3 + a_2 s^2 + a_1 s + a_0}$$

- This implies that the dynamics depend on the transfer function denominator only.

4.5 Obtaining transfer functions from state-space models

- In section 4.4, we saw how to go from a transfer function model to a state-space model (known as the **controllable canonical form**).
- It is also possible to go from a state-space model to a transfer function model as follows:

4.5.1 Continuous-time state-space model \rightarrow transfer function

- Consider the following single-input-single-output continuous-time state-space model:

$$\begin{aligned}\dot{\mathbf{x}}(t) &= \mathbf{A}\mathbf{x}(t) + \mathbf{B}u(t) \\ y(t) &= \mathbf{C}\mathbf{x}(t) + \mathbf{D}u(t)\end{aligned}$$

- Taking the Laplace transform gives:

$$\begin{aligned}s\mathbf{X}(s) &= \mathbf{A}\mathbf{X}(s) + \mathbf{B}U(s) \\ Y(s) &= \mathbf{C}\mathbf{X}(s) + \mathbf{D}U(s)\end{aligned}$$

- Rearranging the state equation gives:

$$\begin{aligned}(s\mathbf{I} - \mathbf{A})\mathbf{X}(s) &= \mathbf{B}U(s) \\ \Rightarrow \mathbf{X}(s) &= (s\mathbf{I} - \mathbf{A})^{-1}\mathbf{B}U(s)\end{aligned}$$

- Substituting this equation into the output equation gives:

$$Y(s) = \mathbf{C}(s\mathbf{I} - \mathbf{A})^{-1}\mathbf{B}U(s) + \mathbf{D}U(s)$$

- Hence, the transfer function is defined in terms of the state-space equation matrices as:

$$G(s) = \frac{Y(s)}{U(s)} = \mathbf{C}(s\mathbf{I} - \mathbf{A})^{-1}\mathbf{B} + \mathbf{D}$$

-
- **Example 4.2:** Determine the transfer function for the following system:

$$\dot{\mathbf{x}}(t) = \begin{bmatrix} 0 & 2 \\ -3 & 5 \end{bmatrix} \mathbf{x}(t) + \begin{bmatrix} 1 \\ 0 \end{bmatrix} u(t), \quad y(t) = \begin{bmatrix} 1 & 0 \end{bmatrix} \mathbf{x}(t)$$

Solution

$$\begin{aligned} G(s) &= \mathbf{C}(s\mathbf{I} - \mathbf{A})^{-1} \mathbf{B} \\ &= \begin{bmatrix} 1 & 0 \end{bmatrix} \left(\begin{bmatrix} s & 0 \\ 0 & s \end{bmatrix} - \begin{bmatrix} 0 & 2 \\ -3 & 5 \end{bmatrix} \right)^{-1} \begin{bmatrix} 1 \\ 0 \end{bmatrix} \\ &= \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} s & -2 \\ 3 & s-5 \end{bmatrix}^{-1} \begin{bmatrix} 1 \\ 0 \end{bmatrix} \\ &= \frac{\begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} s-5 & 2 \\ -3 & s \end{bmatrix} \begin{bmatrix} 1 \\ 0 \end{bmatrix}}{s^2 - 5s + 6} \\ &= \frac{\begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} s-5 \\ -3 \end{bmatrix}}{s^2 - 5s + 6} = \frac{s-5}{s^2 - 5s + 6} \end{aligned}$$

Hence: $G(s) = \frac{Y(s)}{U(s)} = \frac{s-5}{s^2 - 5s + 6}$