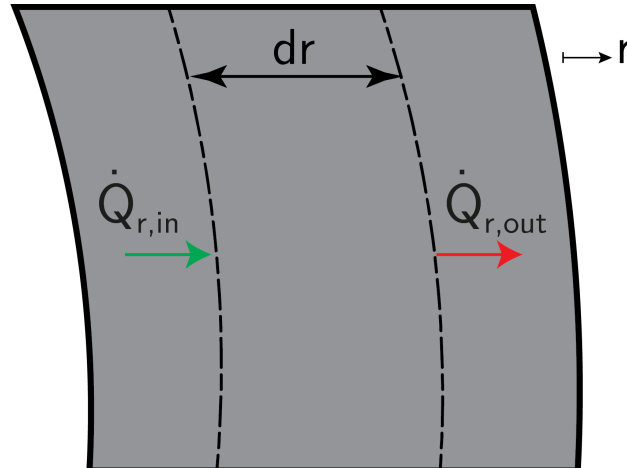


## EB - Cond. - IE 9

Hot water flows through a long pipe of length  $L$ . The water temperature and external surface temperature of the pipe are constant and equal to  $T_\infty$  and  $T_1$  respectively. Set up the energy balance for radial heat conduction in the pipe wall and give the appropriate boundary conditions.



**Energy balance:**

$$\dot{Q}_{r,in} - \dot{Q}_{r,out} = 0$$

Since the type of heat transfer is steady-state, the sum of the in- and outgoing heat fluxes of the control volume should equal zero.

**Heat fluxes:**

$$\dot{Q}_{r,in} = -\lambda A(r) \frac{\partial T}{\partial r} = -\lambda 2\pi r L \frac{\partial T}{\partial r}$$

$$\dot{Q}_{r,out} = \dot{Q}_{r,in} + \frac{\partial \dot{Q}_{r,in}}{\partial r} dr = -\lambda 2\pi r L \frac{\partial T}{\partial r} + \frac{\partial}{\partial r} \left( -\lambda 2\pi r L \frac{\partial T}{\partial r} \right) dr$$

The ingoing flux can be described by use of Fourier's law and the outgoing flux can be approximated by use of the Taylor series expansion.

**Substituting and rewriting:**

$$\dot{Q}_{r,in} - \dot{Q}_{r,out} = 0$$

$$-\lambda 2\pi r L \frac{\partial T}{\partial r} + \lambda 2\pi r L \frac{\partial T}{\partial r} - \frac{\partial}{\partial r} \left( -\lambda 2\pi r L \frac{\partial T}{\partial r} \right) dr = 0 \quad (1)$$

$$\Rightarrow \frac{\partial}{\partial r} \left( \lambda 2\pi r L \frac{\partial T}{\partial r} \right) = 0 \quad (2)$$

Even further simplifying:

$$\Rightarrow \frac{\partial}{\partial r} \left( r \frac{\partial T}{\partial r} \right) = 0 \quad (3)$$