Heat Transfer: Conduction

Dimensionless Numbers and Heisler Diagrams

Prof. Dr.-Ing. Reinhold Kneer

Prof. Dr.-Ing. Dr. rer. pol. Wilko Rohlfs









Learning goals

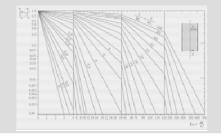
Dimensionless numbers

Importance of dimensionless numbers, especially Fourier and Biot numbers for transient heat transfer.

$$\Theta^* = \Theta^*(x^*, y^*, z^*, t^*, Fo, Bi)$$

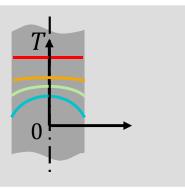
Heisler diagrams

Understanding of the Heisler diagrams for the determination of the body core temperature, the local temperature profile and the heat flow.



Example: Quenching of a steel plate

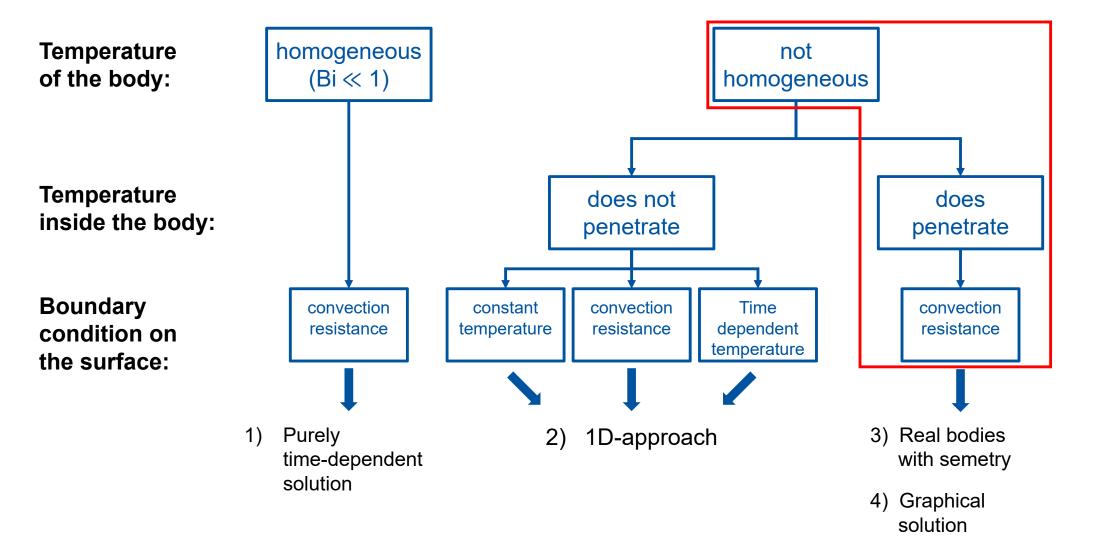
Application of the Heisler diagrams.







How to simplify the problem?

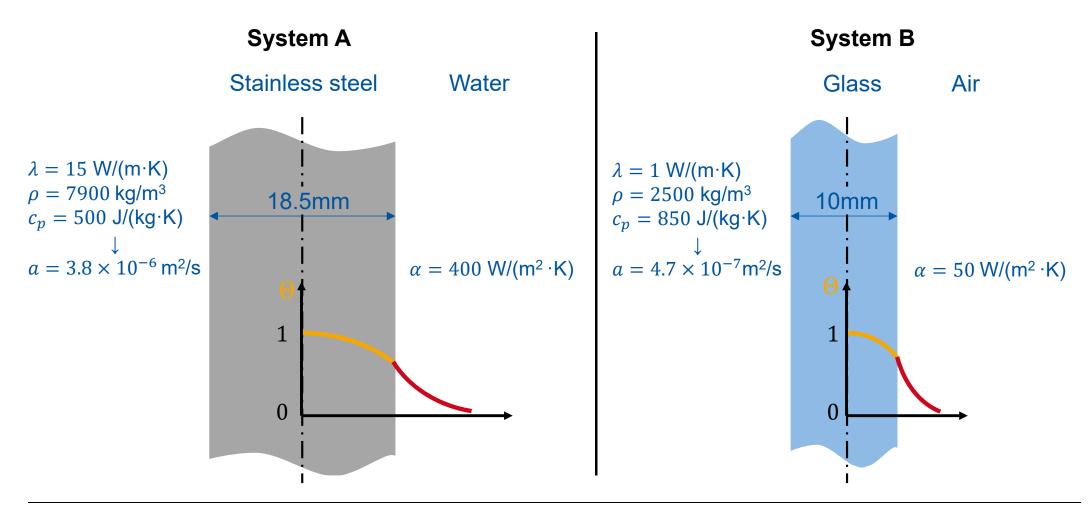








Can the temperature distribution in two different cooled or heated systems look similar?









Dimensionless form

Transient heat conduction

3-D Conservation Equation without advection and source

$$\rho c_p \frac{\partial T}{\partial t} = \frac{\partial}{\partial x} \left(\lambda \frac{\partial T}{\partial x} \right) + \frac{\partial}{\partial y} \left(\lambda \frac{\partial T}{\partial y} \right) + \frac{\partial}{\partial z} \left(\lambda \frac{\partial T}{\partial z} \right)$$

Dimensionless equation

$$\frac{\partial \Theta^*}{\partial t^*} = F_0 \left(\frac{\partial^2 \Theta^*}{\partial x^{*2}} + \frac{\partial^2 \Theta^*}{\partial y^{*2}} + \frac{\partial^2 \Theta^*}{\partial z^{*2}} \right)$$

Solution

 $T = T(x, y, z, t, \rho, c_p, \lambda, Initial - and Boundary Conditions)$

 T_0

 α , T_A

Dimensionless solution

$$\Theta^* = \Theta^*(x^*, y^*, z^*, t^*, Fo, Bi)$$

Example: Quenching of a steel plate

$$x^* = \frac{x}{\delta_x}$$
 $y^* = \frac{y}{\delta_y}$ $z^* = \frac{z}{\delta_z}$ $t^* = \frac{t}{\tau}$ $\Theta^* = \frac{T - T_A}{T_0 - T_A}$

$$Bi = \frac{\alpha\delta}{\lambda} \quad Fo = \frac{a\tau}{\delta^2} = \frac{\lambda}{\rho c_p} \frac{\tau}{\delta^2}$$





Time evolution of the core body temperature

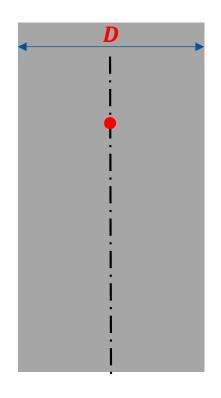
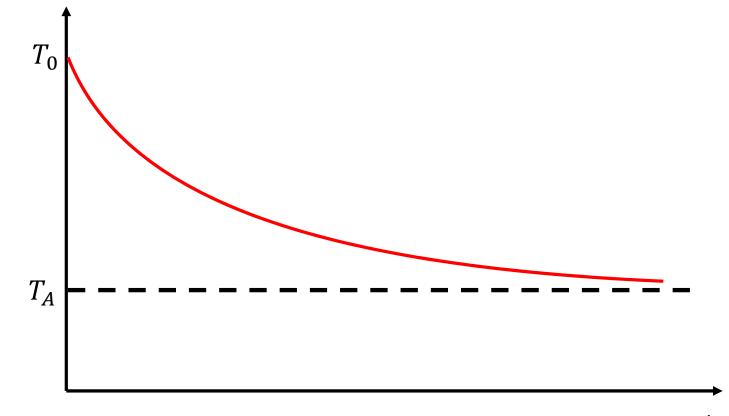


Plate (infinite expansion)

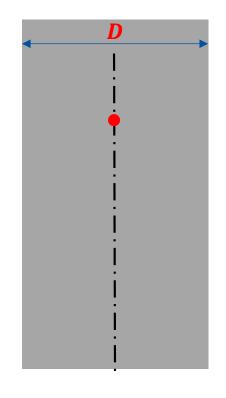
Relevant dependencies:
$$Bi = \frac{\alpha\delta}{\lambda}$$
 $Fo = \frac{a\tau}{\delta^2} = \frac{\lambda}{\rho c_p} \frac{\tau}{\delta^2}$







Time evolution of the core body temperature



Relevant dependencies:
$$Bi = \frac{\alpha \delta}{\lambda}$$
 $Fo = \frac{a\tau}{\delta^2} = \frac{\lambda}{\rho c_p} \frac{\tau}{\delta^2}$

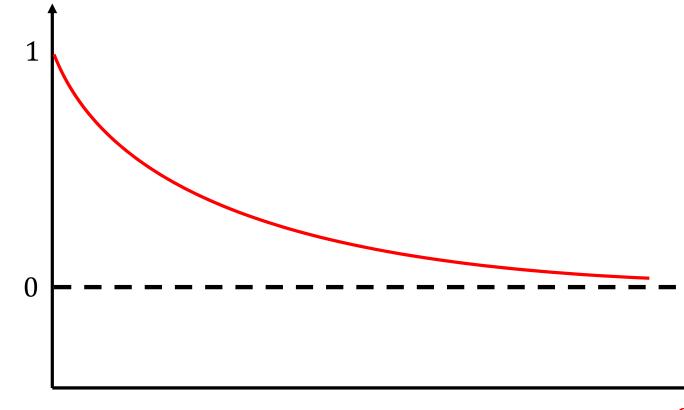


Plate (infinite expansion)

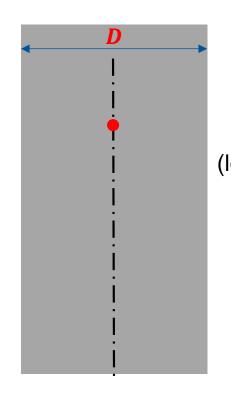
$$Fo = \frac{at}{D/2}$$







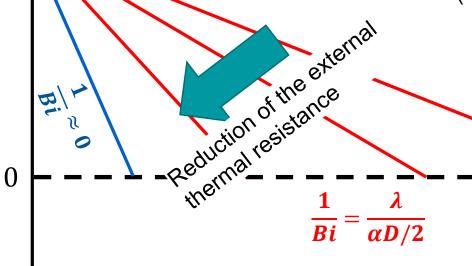
Time evolution of the core body temperature



Relevant dependencies:
$$Bi = \frac{\alpha\delta}{\lambda}$$
 $Fo = \frac{a\tau}{\delta^2} = \frac{\lambda}{\rho c_p} \frac{\tau}{\delta^2}$

 $\frac{T_c - T_A}{T_0 - T_A} \qquad 1$ Valid for a given ratio of internal and (logarithmic) external resistance (Biot number)





 $\frac{1}{Ri} \approx 0 \rightarrow \text{imposed wall temperature}$

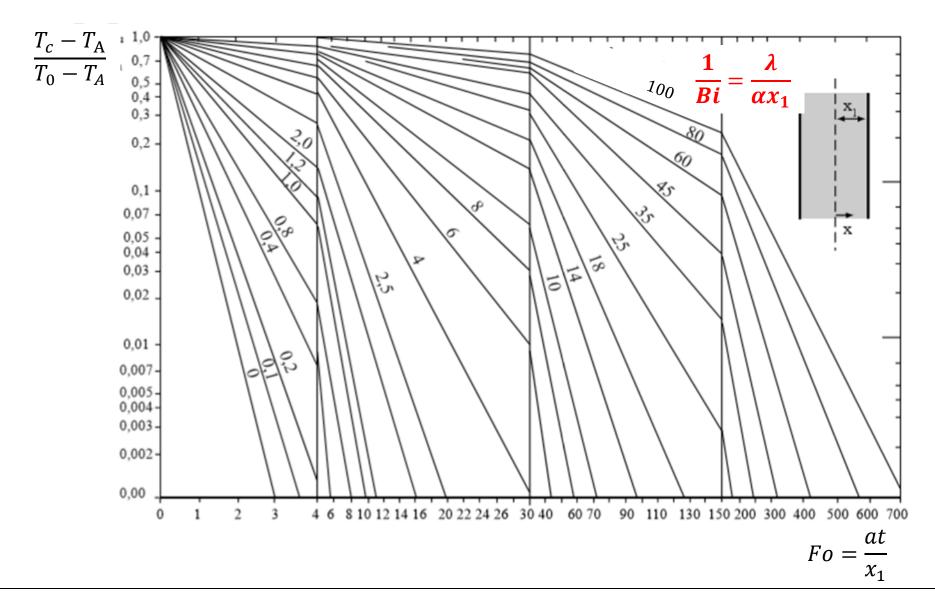
$$Fo = \frac{at}{D/2}$$
 (logarithmic)







Heisler Diagram: Time evolution of the core body temperature







Local distribution of body temperature

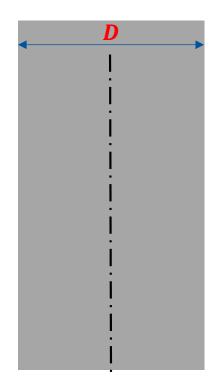
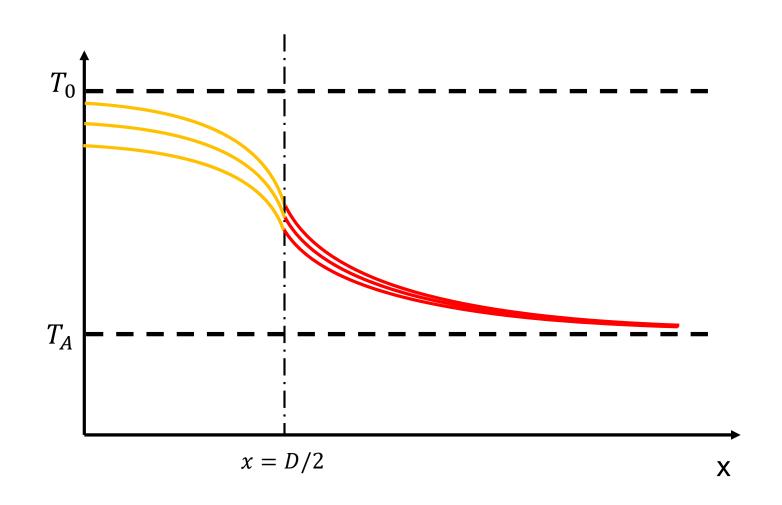


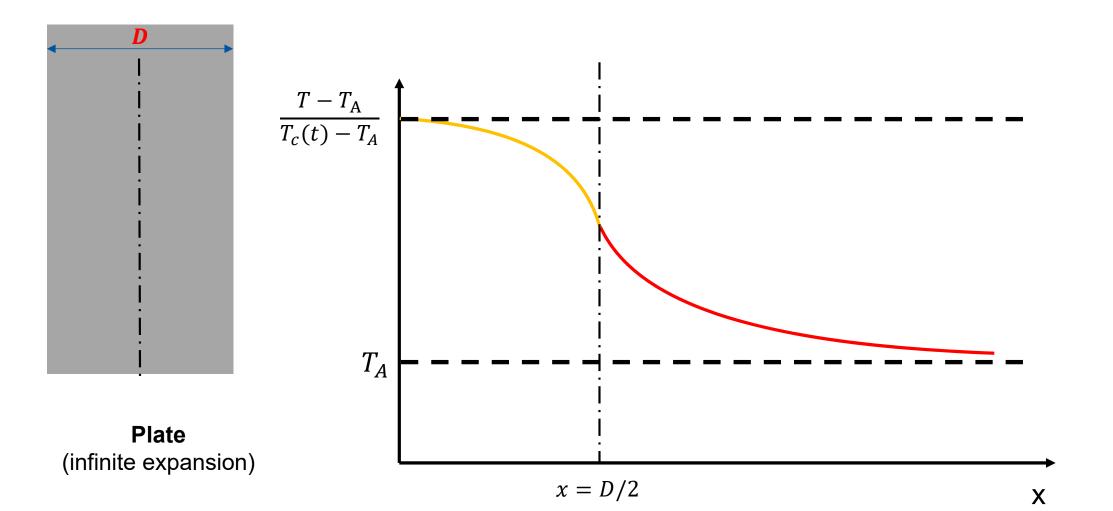
Plate (infinite expansion)







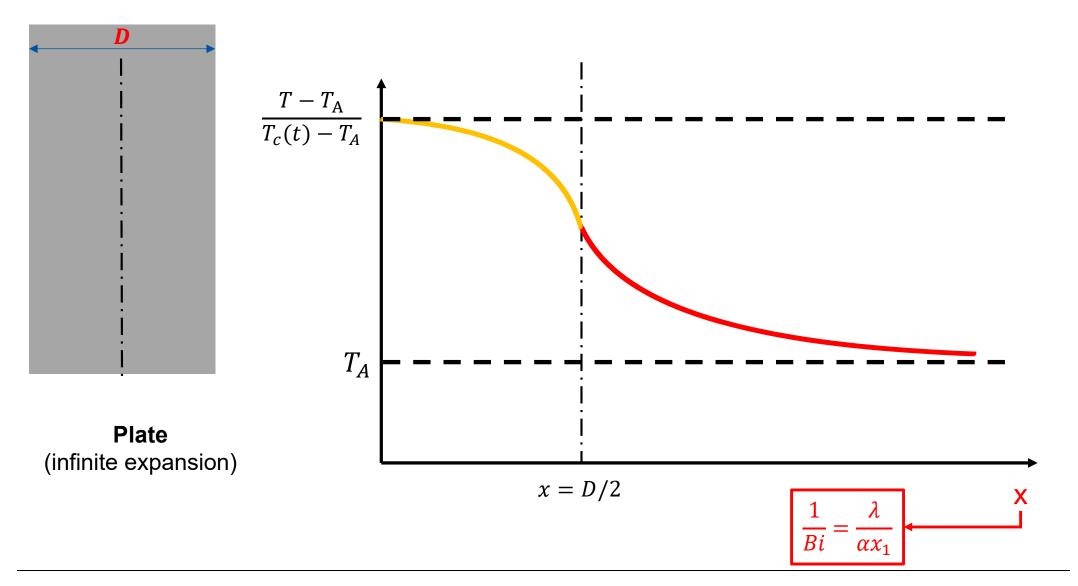
Local distribution of body temperature







Local distribution of body temperature

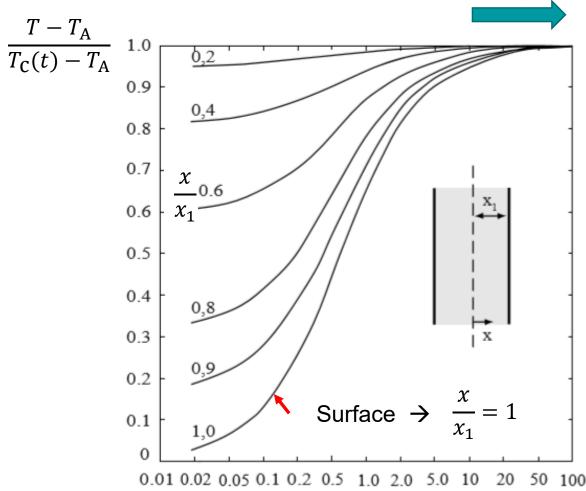






Heisler diagram: Local temperature distribution

Low Bi → homogeneous temperature

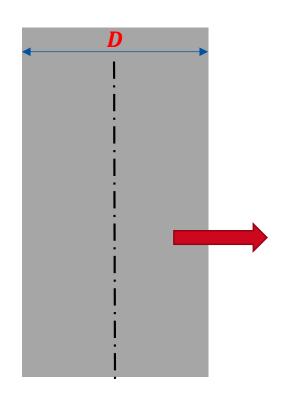


Body center:
$$\rightarrow \frac{x}{x_1} = 0$$

$$\frac{1}{Bi} = \frac{\lambda}{\alpha x_1}$$
 (logarithmic)







Total heat stored in the object: $Q_0 = mc_p(T_0 - T_A)$

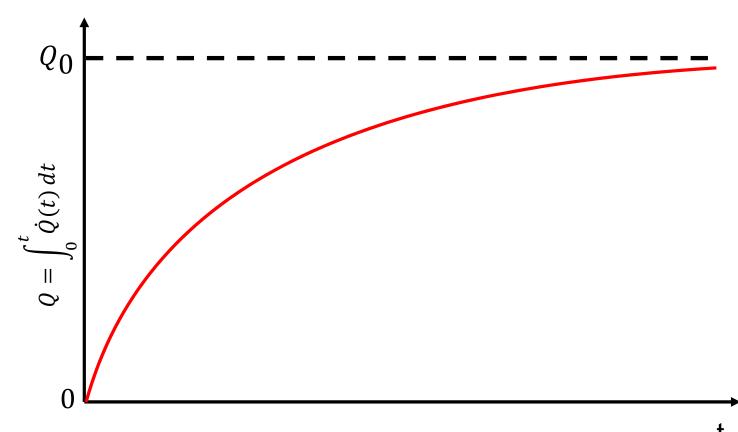
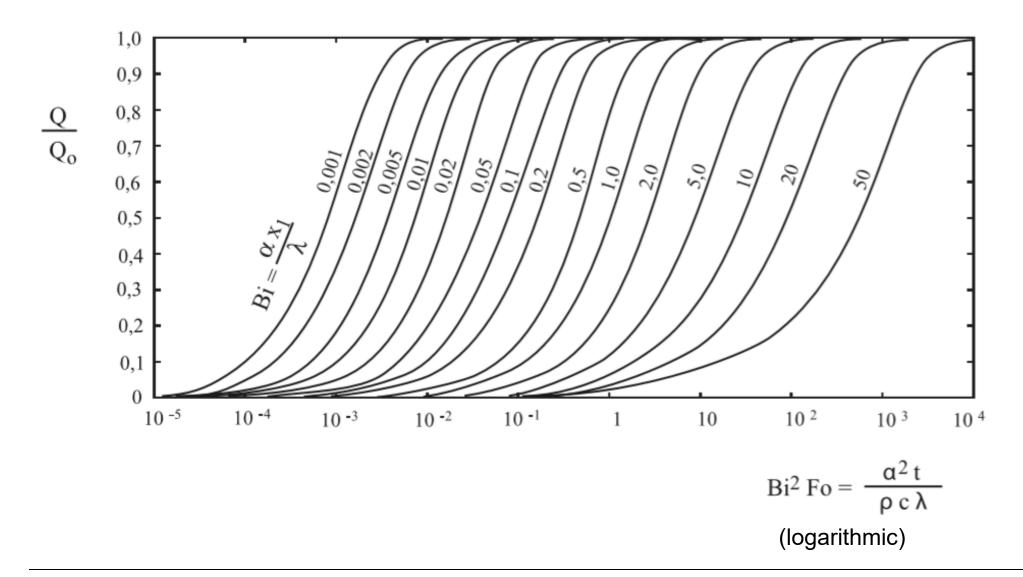


Plate (infinite expansion)





Heisler diagram: Time evolution of the emitted heat



UNIVERSITY OF TWENTE.





Heisler Diagrams of different symmetrical bodies

Dimensionless solution

$$\Theta^* = \Theta^*(x^*, y^*, z^*, t^*, Fo, Bi)$$

Geometry	Plate	Cylinder	Sphere
Temperature in the center of the object	1	9 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 -	1. T.
Temperature distribution	T-T _c T _a ·T _c 10 0.8 0.8 0.7 2.0.6 1.0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0	7. T _a 1.0 0.9 0.9 0.4 0.7 0.5 0.4 0.3 0.4 0.3 0.4 0.3 0.4 0.3 0.4 0.3 0.4 0.3 0.4 0.3 0.4 0.3 0.4 0.3 0.4 0.3 0.4 0.3 0.4 0.3 0.4 0.3 0.4 0.3 0.4 0.3 0.4 0.3 0.4 0.3 0.4 0.3 0.4 0.4 0.4 0.4 0.4 0.4 0.4 0.4	7-1, 18 02 03 04 04 05 19 20 10 22 79 10 100 10 10 10 10 10 10 10 10 10 10 10
Heat flux fraction	$\begin{array}{c} Q \\ Q_0 \\ Q_0 \\ 0.5 \\ 0.5 \\ 0.5 \\ 0.4 \\ 0.3 \\ 0.2 \\ 0.1 \\ 0.0 \\ 0.1 \\ 0.0 \\ 0.1 \\ 0.1 \\ 0.0 \\ 0.1 \\ 0.0 \\ 0$	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	$\begin{array}{c} 1.0 \\ 0.9 \\ 0.0 \\ 0.6 \\ 0.4 \\ 0.3 \\ 0.2 \\ 0.1 \\ 0.0 \\$



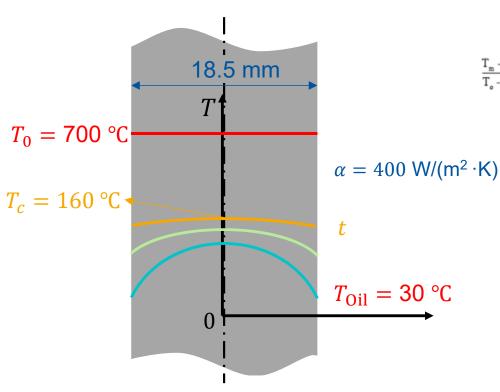


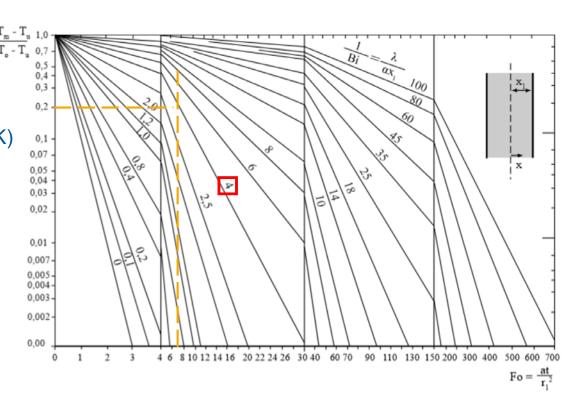
Example: Quenching of a steel plate

Stainless steel

Oil

a) After what time has the center temperature T_c cooled down to 160 °C?





$$\lambda = 15 \text{ W/(m·K)}$$
 $\rho = 7900 \text{ kg/m}^3$
 $c_p = 500 \text{ J/(kg·K)}$
 \downarrow
 $a = 3.8 \times 10^{-6} \text{ m²/s}$

$$\frac{1}{Bi} = \frac{\lambda}{\alpha x_1} = 4.05$$
$$x_1 = D/2$$

$$\frac{1}{Bi} = \frac{\lambda}{\alpha x_1} = 4.05 \qquad \frac{T_c - T_{Oil}}{T_0 - T_{Oil}} = 0.19 \quad \Longrightarrow F_0 = \frac{at}{x_1^2} = 7 \qquad \Longrightarrow t = 158s$$

$$F_0 = \frac{at}{x_1^2} = 7$$
 $\Rightarrow t = 1588$

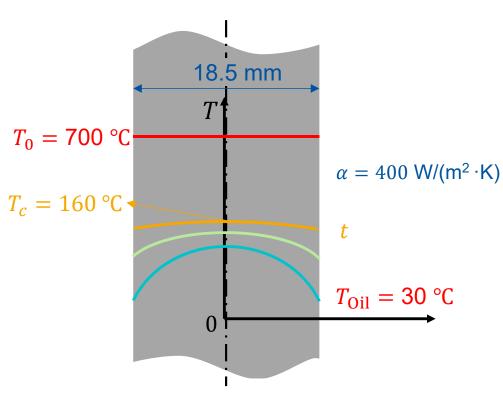




Example: Quenching of a steel plate

Stainless steel

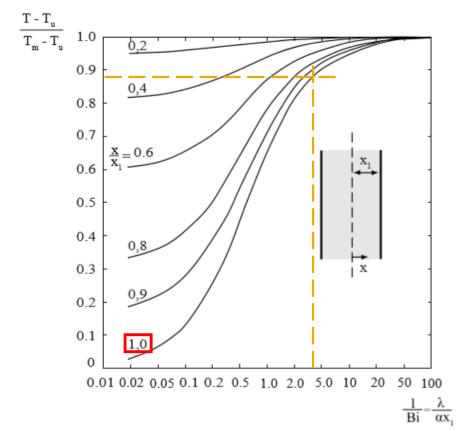
Oil



$$\lambda = 15 \text{ W/(m·K)}$$
 $\rho = 7900 \text{ kg/m}^3$
 $c_p = 500 \text{ J/(kg·K)}$
 \downarrow
 $a = 3.8 \times 10^{-6} \text{ m²/s}$

$$\frac{1}{Bi} = \frac{\lambda}{\alpha x_1} = 4.05$$
$$x_1 = D/2$$

b) What is the value of the temperature T at the plate surface after t = 158s?



$$\frac{1}{Bi} = \frac{\lambda}{\alpha x_1} \approx 4 \implies \frac{T - T_{\text{Oil}}}{T_c - T_{\text{Oil}}} = 0.88 \implies T = 144^{\circ}\text{C}$$

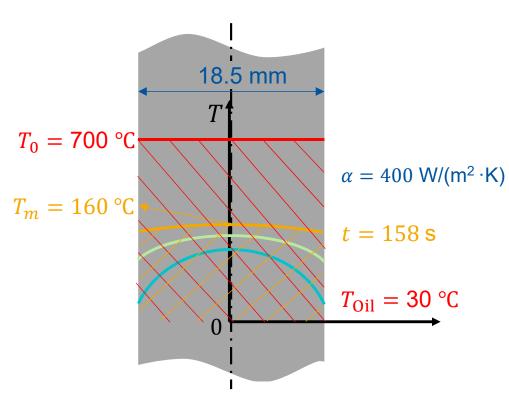




Example: Quenching of a steel plate

Stainless steel

Oil

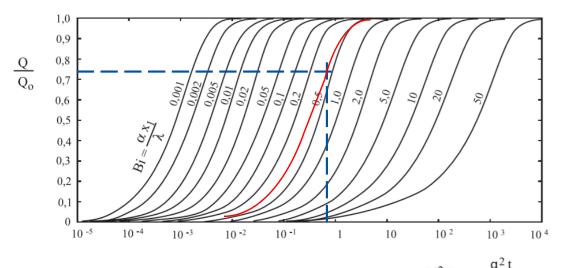


c) How much heat has the plate dissipated after t = 158 s?

total heat Q_0

remaining heat Q_t

dissipated Heat $Q = Q_0 - Q_t$



$$Bi^{2}Fo = \frac{\alpha^{2}t}{\rho c_{p}\lambda} = 0.43 \quad \Longrightarrow \frac{Q}{Q_{0}} = 0.74 \quad \Longrightarrow \frac{Q}{m} = 247.9 \frac{\text{kJ}}{\text{kg}}$$

with
$$Q_0 = mc_p(T_0 - T_{Oil})$$

$$Bi = 0.25$$
$$Fo = 7$$





Comprehension questions

Which two dimensionless numbers are used to describe a transient heat transfer problem of a body with additional external thermal resistance?

Which tool allows the determination of the temperature profile or the amount of heat transferred for extended plates, long cylinders or spheres?





