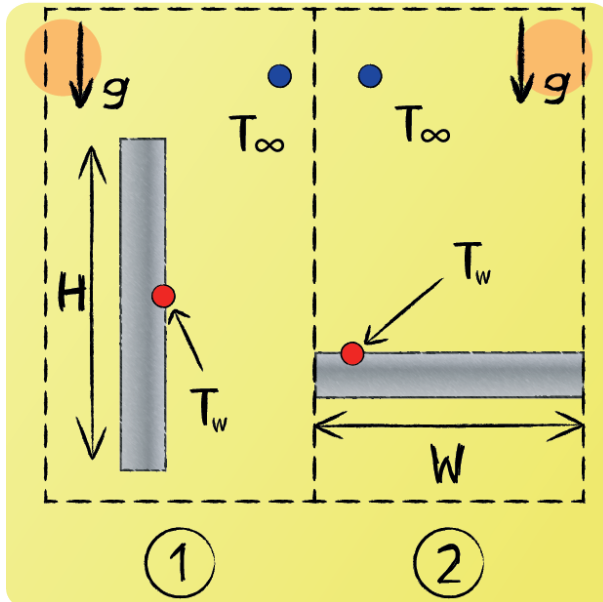


Exam Preparation Convection 02



Two heat-emitting surfaces (case 1: H height, case 2: W width) with the respective wall temperatures T_H and T_W are given. The quiescent environment has a temperature T_∞ . Determine the ratio of the two heat transfer coefficients α_H and α_W of the surfaces.

For the mean heat transfer at natural convection (vertical plate (H), laminar boundary layer, isothermal surface (HTC 17))

$$\overline{Nu}_H = 0,535 \cdot (Gr_H Pr)^{1/4}$$

and the mean heat transfer at natural convection (horizontal plate (B), laminar boundary layer, isothermal surface (HTC 22a),

$$\overline{Nu}_B = 0,54 \cdot (Gr_B Pr)^{1/4}$$

under the condition

$$Gr_L Pr < 1 \cdot 10^6 < Gr_{L,krit.} Pr$$

with the Grashof-Numbers Gr_H und Gr_B

$$Gr_H = \frac{1}{T_\infty} g(T_H - T_\infty) H^3 \nu^2$$



and

$$Gr_B = \frac{1}{T_\infty} g(T_B - T_\infty) B^3 \nu^2$$

,a relation for the heat transfer α at both surfaces can be developed

$$\overline{\alpha}_L = \frac{\overline{Nu}_L \lambda_\infty}{L}$$

The ratio of both heat transfer coefficients leads to:

$$\frac{\overline{Nu}_B}{\overline{Nu}_H} = \frac{0,54 (Gr_B Pr)^{(\frac{1}{4})}}{0,535 (Gr_H Pr)^{(\frac{1}{4})}}$$

simplified to

$$\left(\frac{\overline{\alpha}_B}{\overline{\alpha}_H} \right)^4 = \frac{0,54^4 \cdot (T_B - T_\infty) \cdot H}{0,535^4 \cdot (T_H - T_\infty) \cdot B}$$

$$\overline{\alpha}_B = 0,645 \cdot \overline{\alpha}_H$$