

---

# Heat Transfer

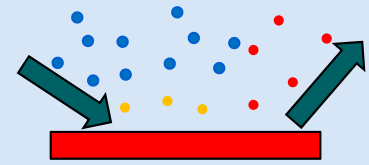
## Introduction to convection and the conservation equations

Prof. Dr.-Ing. Reinhold Kneer  
Prof. Dr.-Ing. Dr. rer. pol. Wilko Rohlf's  
Prof. Dr. ir. Kees Venner

# Learning Goals

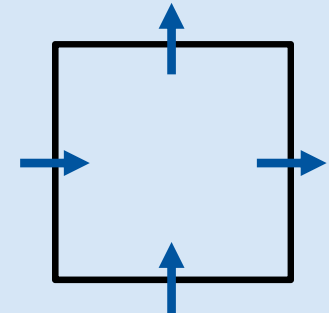
- Classification

- Understanding Convection and the distinction from Advection
- Convection as the interaction of heat Conduction and Advection
- Classification of convection problems



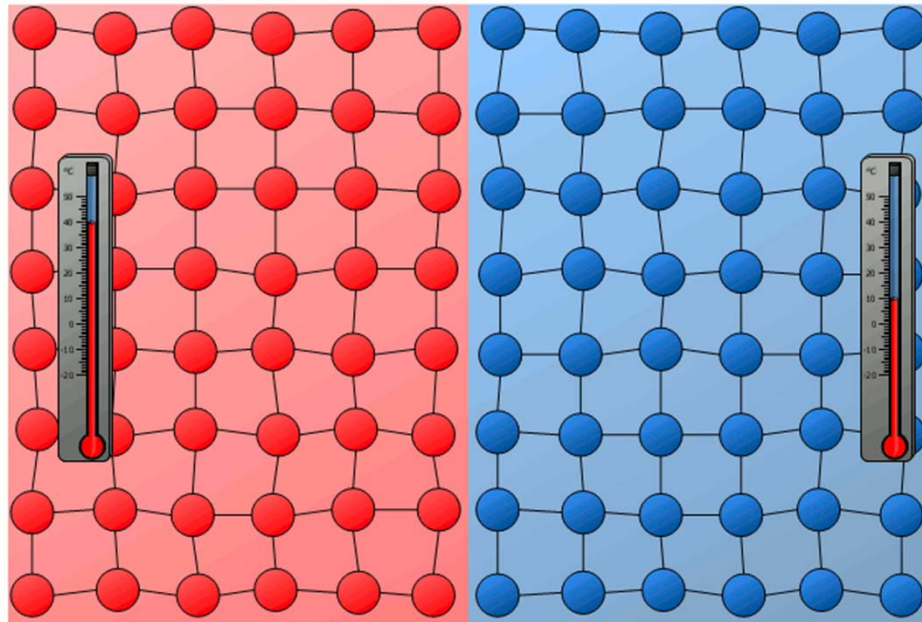
- Conservation Equation

- Derive the conservation equations for mass, momentum and energy
- Understand the similarity between momentum and energy transport



# How is the heat transferred?

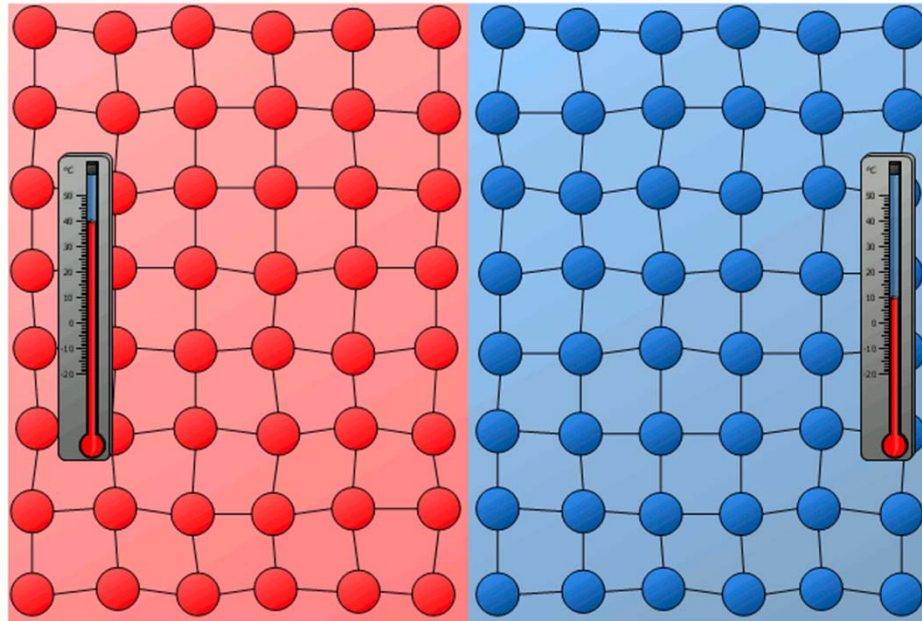
## Heat Conduction (conduction/diffusion)



Source: [www.tec-science.com/de/thermodynamik-waermelehre/waerme/warme-und-thermodynamisches-gleichgewicht/](http://www.tec-science.com/de/thermodynamik-waermelehre/waerme/warme-und-thermodynamisches-gleichgewicht/)  
[www.tec-science.com/de/thermodynamik-waermelehre/waerme/warum-befinden-sich-heizkorper-meist-unter-einem-fenster/](http://www.tec-science.com/de/thermodynamik-waermelehre/waerme/warum-befinden-sich-heizkorper-meist-unter-einem-fenster/)

# How is the heat transferred?

## Heat Conduction (conduction/diffusion)



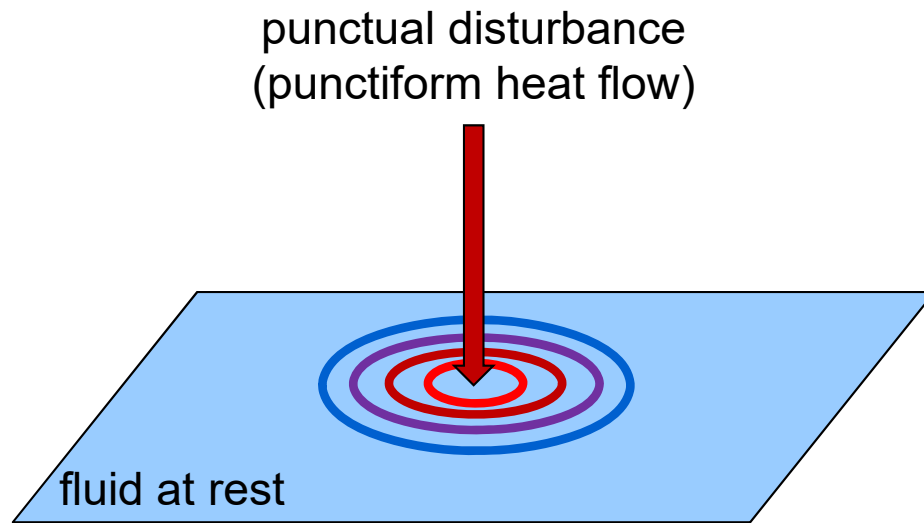
Source: [www.tec-science.com/de/thermodynamik-waermelehre/waerme/waerme-und-thermodynamisches-gleichgewicht/](http://www.tec-science.com/de/thermodynamik-waermelehre/waerme/waerme-und-thermodynamisches-gleichgewicht/)  
[www.tec-science.com/de/thermodynamik-waermelehre/waerme/warum-befinden-sich-heizkorper-meist-unter-einem-fenster/](http://www.tec-science.com/de/thermodynamik-waermelehre/waerme/warum-befinden-sich-heizkorper-meist-unter-einem-fenster/)

## Convection



# How is the heat transferred?

## Heat Conduction (conduction/diffusion)

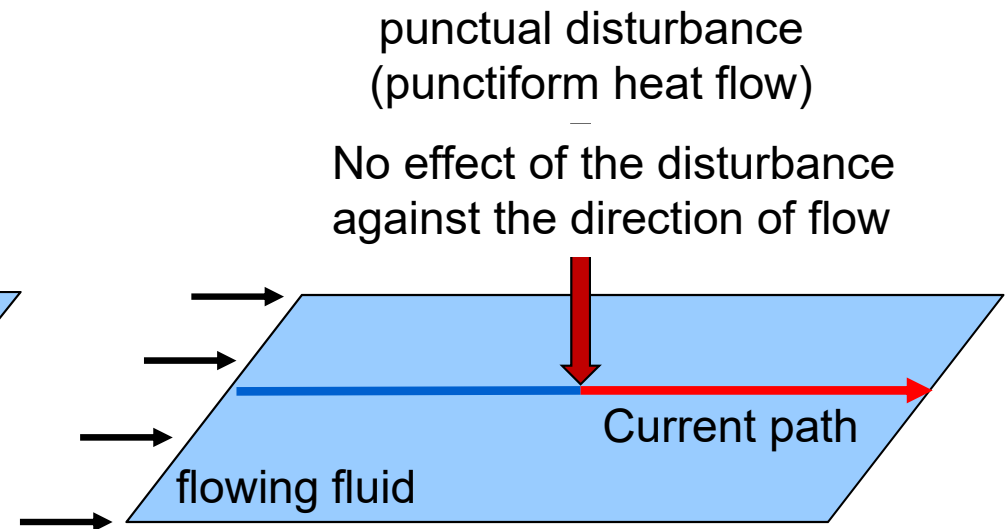


Heat flow in radial directional  
along the gradients

**Fourier Law**

$$\dot{q}'' = -\lambda \nabla T$$

## Advection



Heat is transported by **fluid movement**  
along a current path

**Enthalpy flow density**

$$\dot{h}'' = \rho u c_p T$$



# How is the heat transferred?

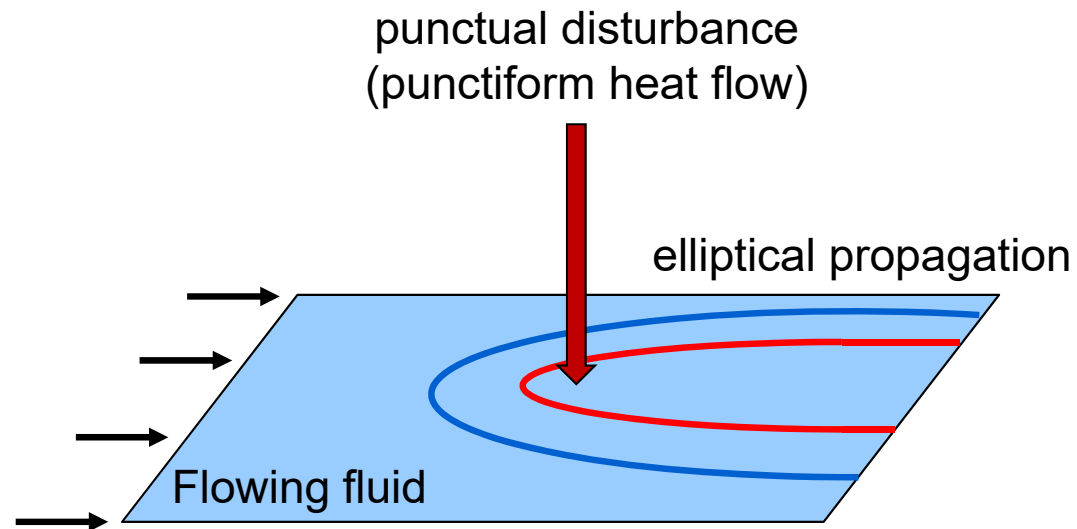
Heat Conduction (conduction/diffusion)



Advection



**Convective Heat Transfer (convection)**



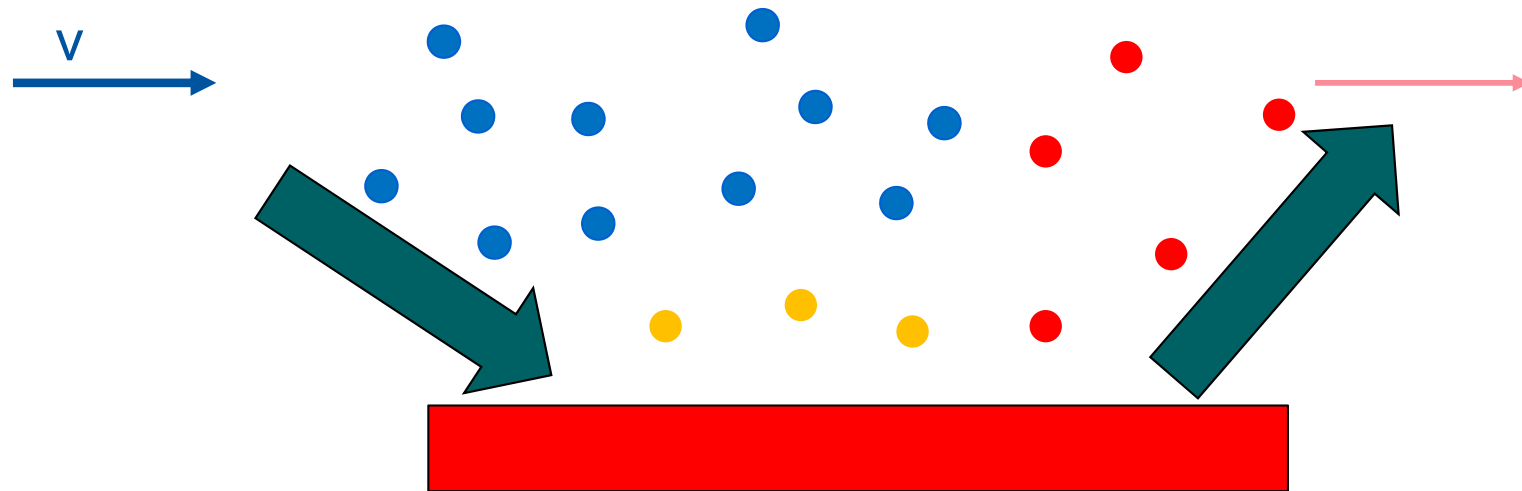
Transport along the current (flow) paths:  
Transport perpendicular to the current paths:

**Convection (and Conduction)**  
**only Conduction**



# Mechanism of convective heat transfer

What is the difference in comparison to pure heat conduction?

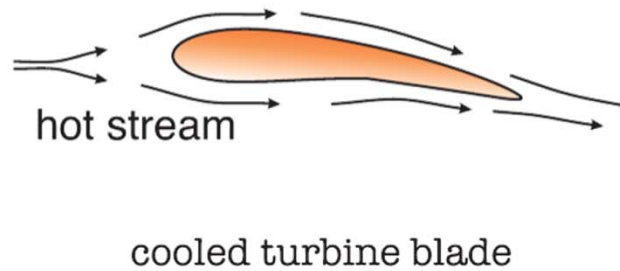


# Classifications according to flow condition

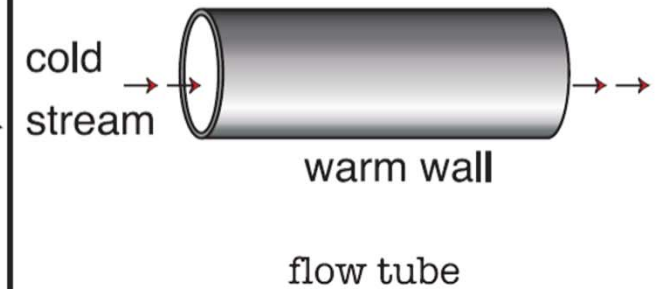
## Forced Convection

- Driven by externally generated movement of the fluid/object

### External



### Internal



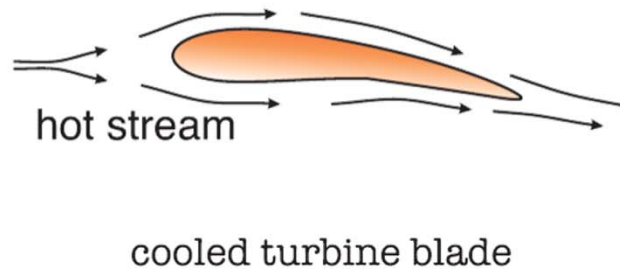


# Classifications according to flow condition

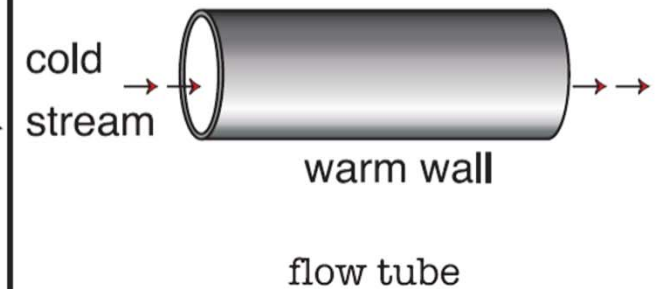
## Forced Convection

- Driven by externally generated movement of the fluid/object

### External



### Internal



## Free Convection

- Inherently driven due to heat transfer (density differences)



# Classifications according to flow condition

## Forced Convection

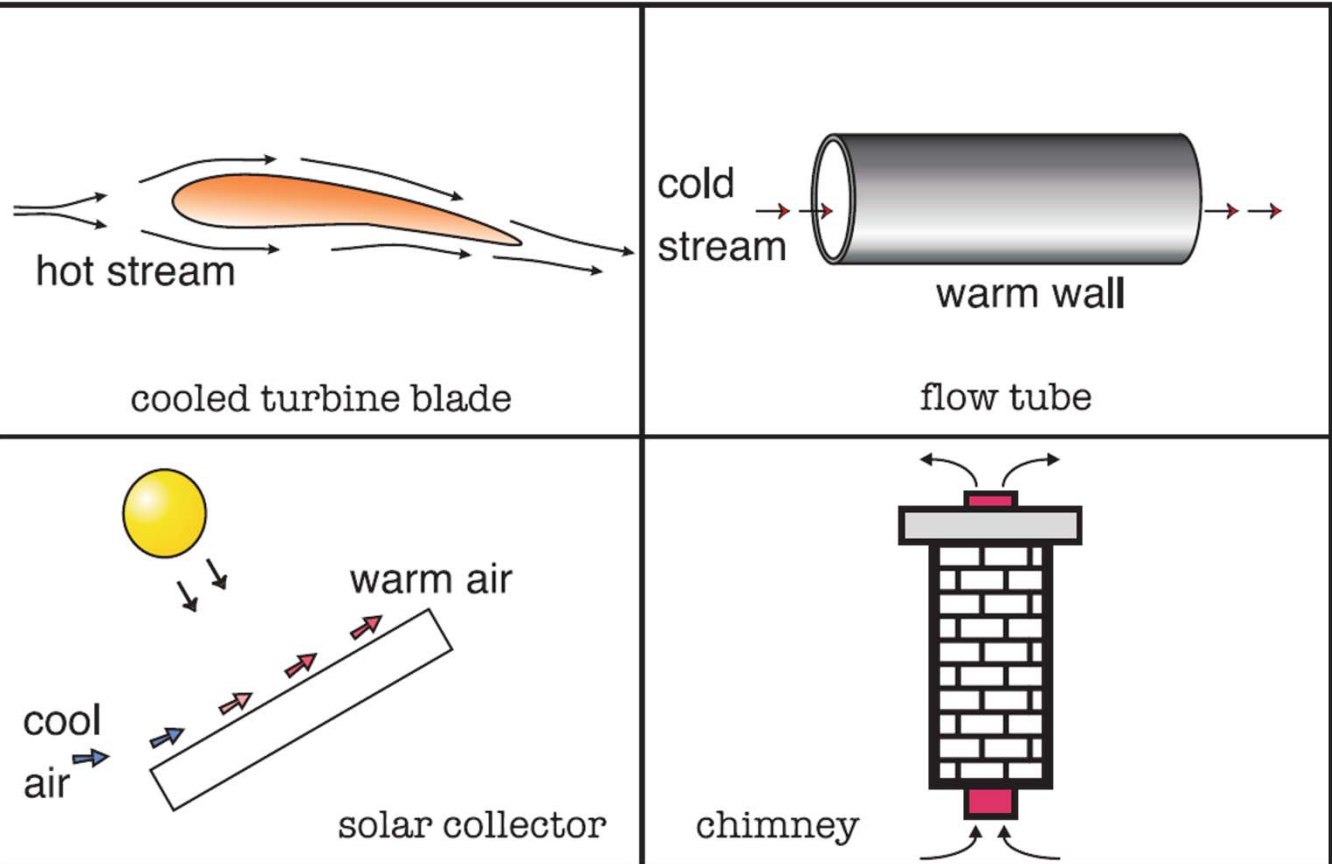
- Driven by externally generated movement of the fluid/object

## Free Convection

- Inherently driven due to heat transfer (density differences)

### External

### Internal



# Empirical description by the heat transfer coefficient

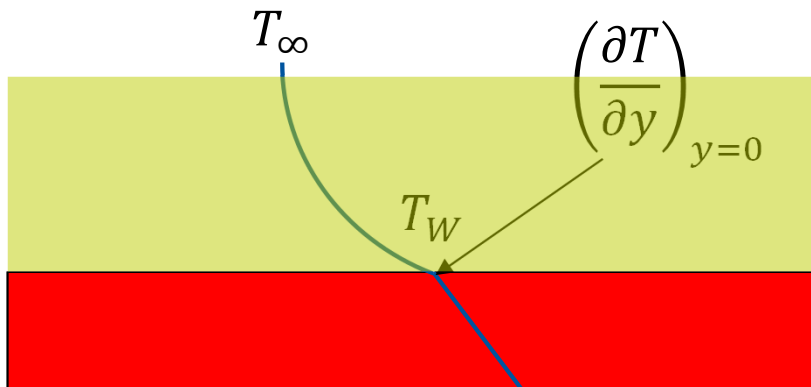
$$\dot{Q} = \alpha A (T_W - T_\infty)$$

Fourier's Heat  
Conduction Law

$$\dot{Q} = -A\lambda_f \left( \frac{\partial T}{\partial y} \right)_{y=0,f}$$

The heat transfer coefficient  $\alpha$  describes the approximately linear relationship between the amount of heat transferred and the temperature gradient.  $\alpha$  is a SYSTEM parameter, not a material property !

$$\alpha \equiv - \frac{\lambda_f \left( \frac{\partial T}{\partial y} \right)_{y=0,f}}{(T_W - T_\infty)}$$



## Nusselt number

- Dimensionless heat transfer coefficient with the reference length  $L$

$$Nu = \frac{\alpha L}{\lambda} = L \frac{-\left( \frac{\partial T}{\partial y} \right)_{y=0,f}}{(T_W - T_\infty)}$$

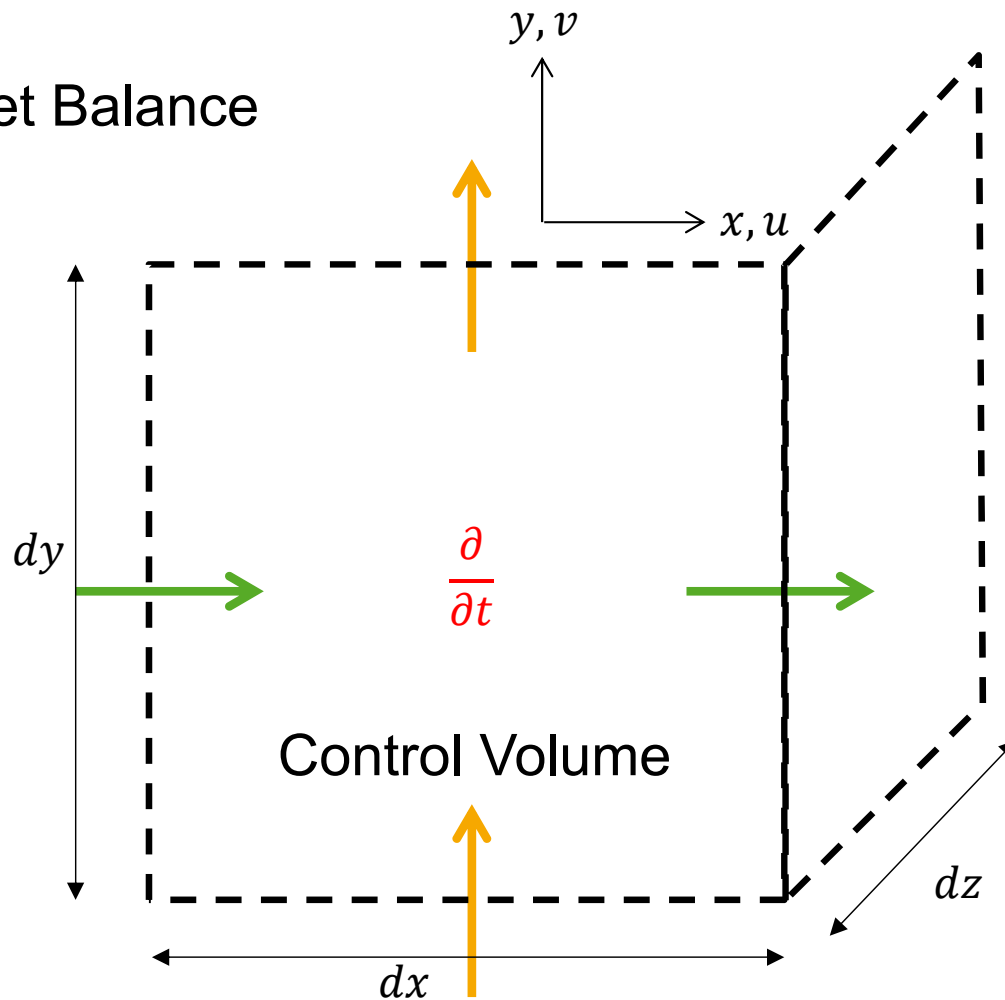
## Boundary Layer

- Near-wall layer with significant gradient of Velocity and Temperature
- What happens here? → Conservation Equation

# Conservation Equation

For Mass  $\dot{m}$ , Momentum  $\dot{I}$ , Energy  $h$ ,  $\dot{q}''$ .

Set Balance



## General Balance

Temporal change of a quantity inside the control volume

=

Net transport of the quantity across the boundaries of the control volume

+

External forces  
(for momentum equation)

+

Work output of the external forces  
(for energy equation)

# Continuity Equation

## Set Balance

$$\begin{aligned} & \dot{m}_y(y + dy) \\ &= \rho v(y + dy) dx dz \\ &= \left[ \rho v(y) + \frac{\partial \rho v}{\partial y} dy \right] dx dz \end{aligned}$$

$$\dot{m}_x(x) = \rho u(x) dy dz = \dot{V}_x$$

$$\dot{m}_x(x + dx) = \rho u(x + dx) dy dz = \left[ \rho u(x) + \frac{\partial \rho u}{\partial x} dx \right] dy dz$$

$$\dot{m}_y(y) = \rho v(y) dx dz$$

$\frac{\partial m}{\partial t}$

Control Volume

## Mass Flows

$$\begin{aligned} \frac{\partial m}{\partial t} &= \dot{m}_x(x) - \dot{m}_x(x + dx) \\ &\quad + \dot{m}_y(y) - \dot{m}_y(y + dy) \end{aligned}$$

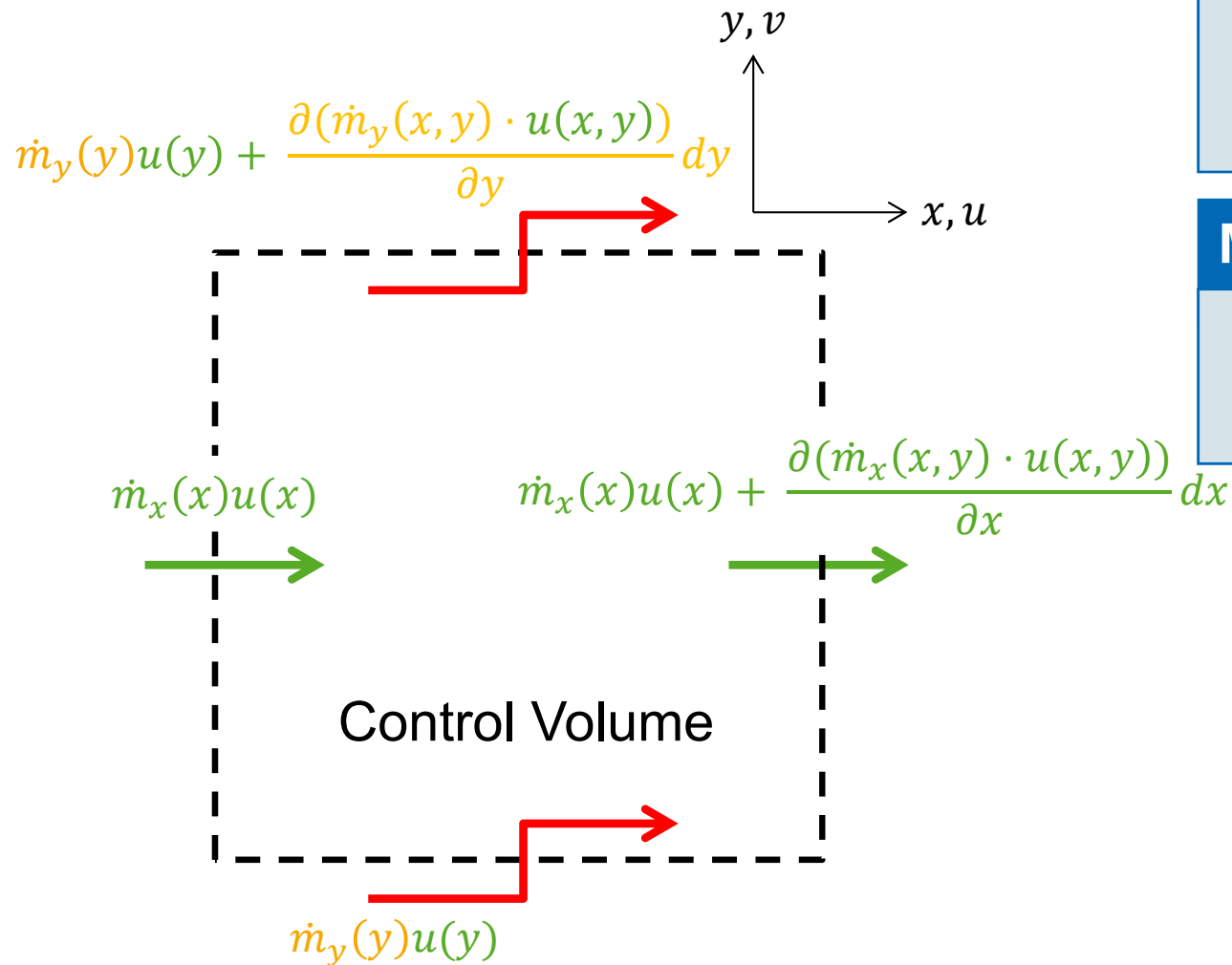
$$\begin{aligned} & \cancel{\frac{\partial \rho}{\partial t} dV} \\ &= -\cancel{\frac{\partial \rho u}{\partial x} dx dy dz} - \cancel{\frac{\partial \rho v}{\partial y} dx dy dz} \end{aligned}$$

incompressible  $\rho = \text{const.}$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

# Momentum Equation: x-direction

Set Balance



## Temporal Change

Steady state  $\frac{\partial I_x}{\partial t} dV = 0$

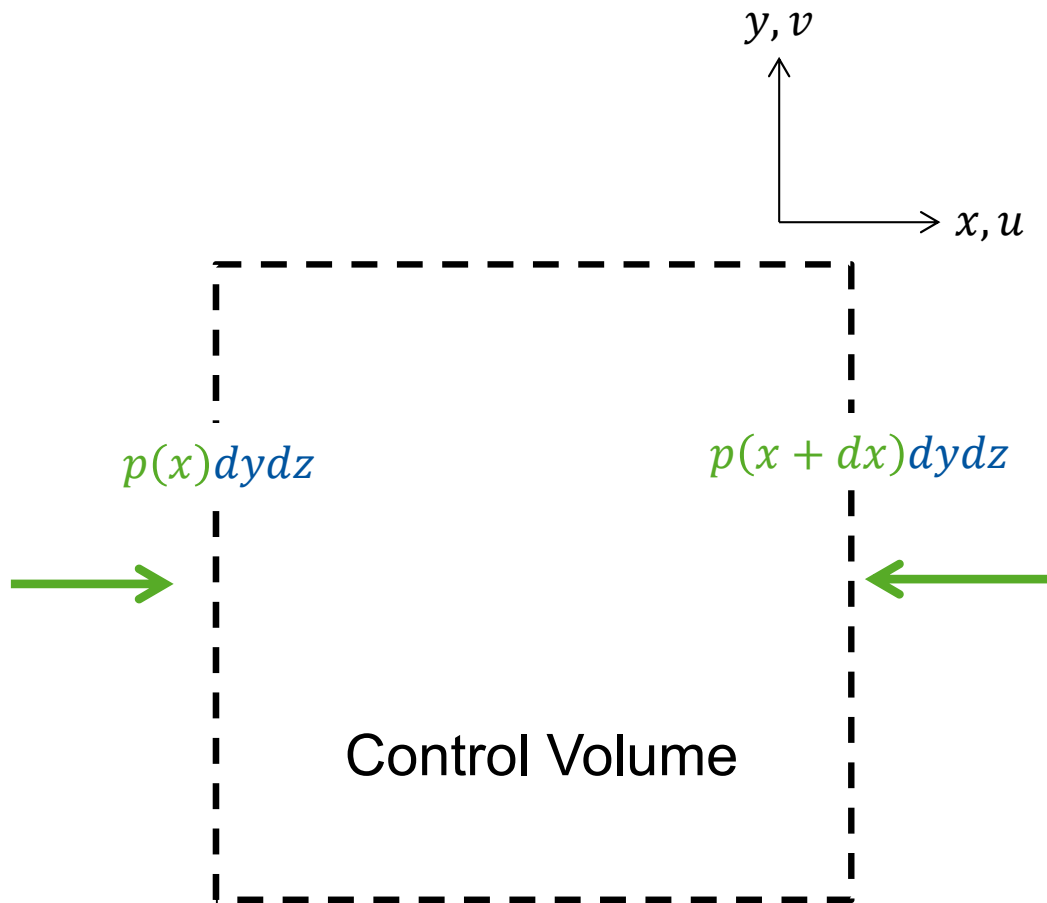
## Momentum Flow

$$-\left(\rho u \frac{\partial u}{\partial x} + \rho v \frac{\partial u}{\partial y}\right) dx dy dz$$



# Momentum Equation: x-direction

## Set Balance



## Temporal Change

Steady state  $\frac{\partial I_x}{\partial t} dV = 0$

## Momentum Flow

$$-\left(\rho u \frac{\partial u}{\partial x} + \rho v \frac{\partial u}{\partial y}\right) dx dy dz$$

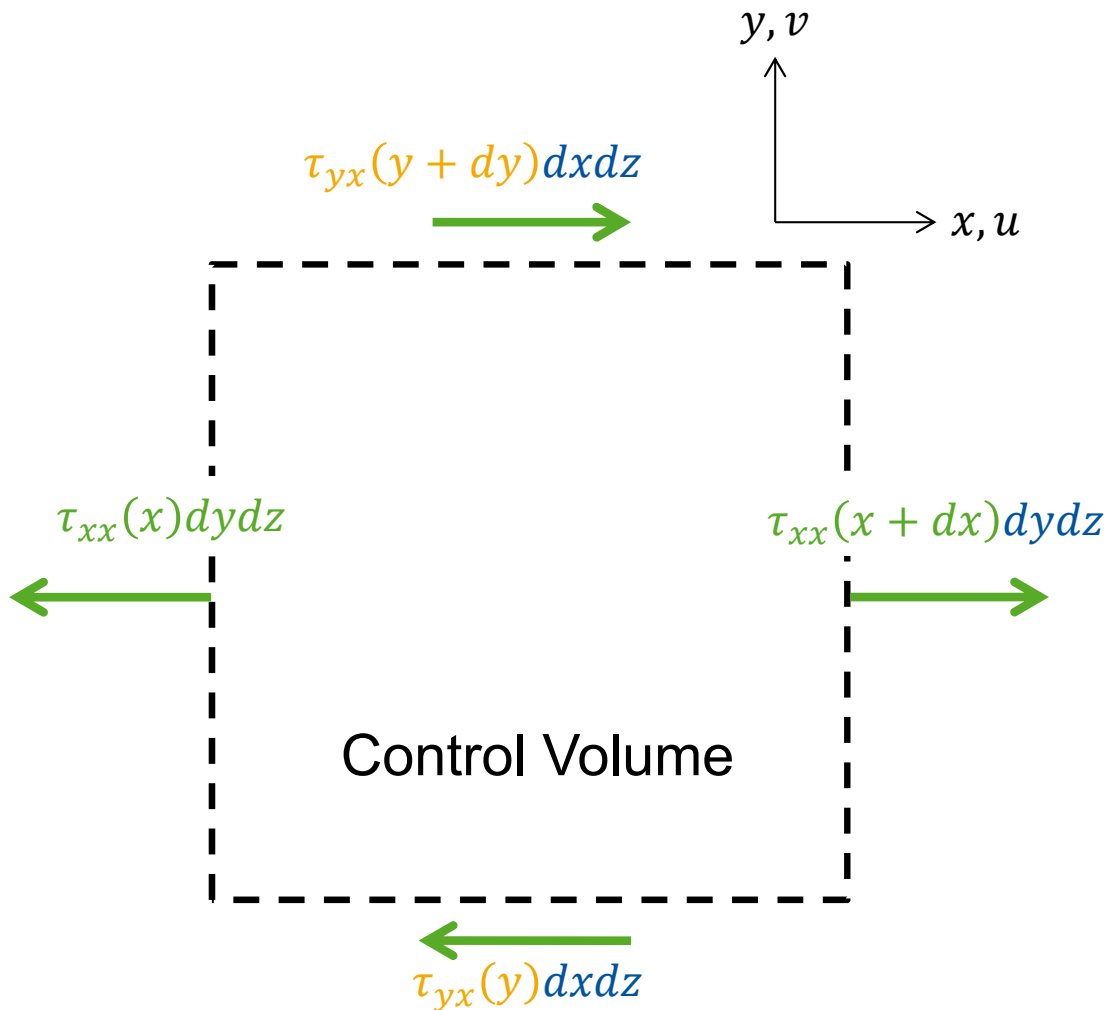
## External Forces acting ON the volume

Pressure Change  $-\frac{\partial p}{\partial x} dx dy dz$



# Momentum Equation: x-direction

## Set Balance



## Temporal Change

Steady state  $\frac{\partial I_x}{\partial t} dV = 0$

## Momentum Flow

$$-\left(\rho u \frac{\partial u}{\partial x} + \rho v \frac{\partial u}{\partial y}\right) dx dy dz$$

## External Forces acting ON the volume

Pressure Change  $-\frac{\partial p}{\partial x} dx dy dz$

Shear Stress

(if incompressible)

$$\eta \left( \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right) dx dy dz$$





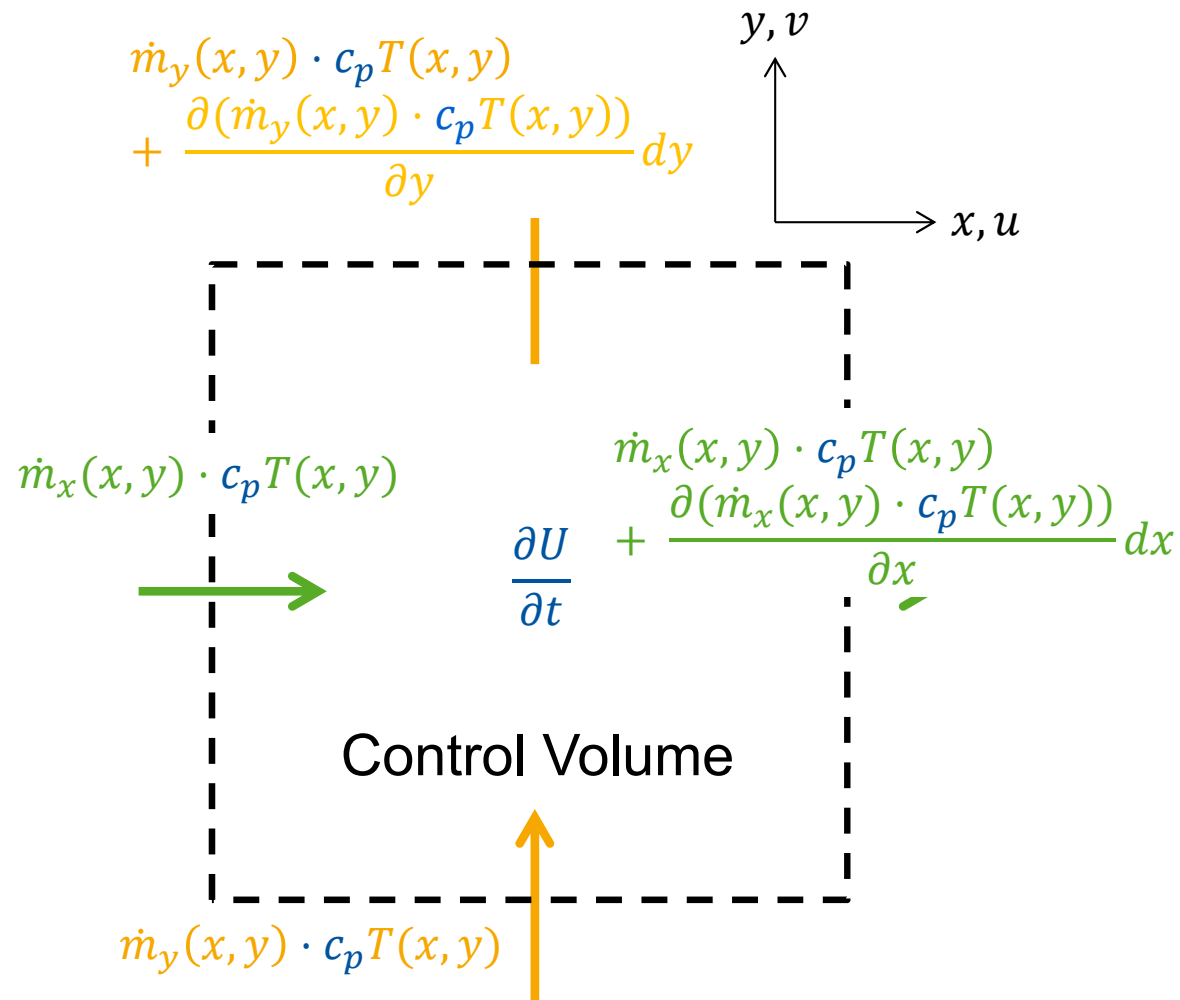
# Momentum Equation (steady state, incompressible)

	Momentum Flows	Pressure	Shear Stress	
x-direction	$\rho u \frac{\partial u}{\partial x} + \rho v \frac{\partial u}{\partial y} + \rho w \frac{\partial u}{\partial z}$	$= -\frac{\partial p}{\partial x}$	$+ \eta \left( \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right)$	+ Volume forces (e.g. Gravitation)
y-direction	$\rho u \frac{\partial v}{\partial x} + \rho v \frac{\partial v}{\partial y} + \rho w \frac{\partial v}{\partial z}$	$= -\frac{\partial p}{\partial y}$	$+ \eta \left( \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right)$	
z-direction	$\rho u \frac{\partial w}{\partial x} + \rho v \frac{\partial w}{\partial y} + \rho w \frac{\partial w}{\partial z}$	$= -\frac{\partial p}{\partial z}$	$+ \eta \left( \frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right)$	



# Energy Conservation: Enthalpy Flows

## Set Balance



## Temporal Change

$$\frac{\partial U}{\partial t} = \rho c_p \frac{\partial T}{\partial t} dV \quad (\text{steady state } \frac{\partial U}{\partial t} = 0)$$

## Enthalpy Flows

$$- \left( \rho u c_p \frac{\partial T}{\partial x} + \rho v c_p \frac{\partial T}{\partial y} \right) dx dy dz$$



# Energy Conservation: Heat conduction / diffusion

## Set Balance

$$-\lambda \left. \frac{\partial T}{\partial y} \right|_{y+dy} \cdot dxdz$$

$$= - \left[ \lambda \left. \frac{\partial T}{\partial y} \right|_y + \frac{\partial}{\partial y} \left( \lambda \frac{\partial T}{\partial y} \right) dy \right] dxdz$$

$$-\lambda \left. \frac{\partial T}{\partial x} \right|_x \cdot dydz$$

$$-\lambda \left. \frac{\partial T}{\partial x} \right|_{x+dx} \cdot dydz$$

$$-\lambda \left. \frac{\partial T}{\partial y} \right|_y \cdot dxdz$$

Control Volume

## Temporal Change

$$\frac{\partial U}{\partial t} = \rho c_p \frac{\partial T}{\partial t} dV \quad (\text{stationär } \frac{\partial U}{\partial t} = 0)$$

## Enthalpy Flows

$$- \left( \rho u c_p \frac{\partial T}{\partial x} + \rho v c_p \frac{\partial T}{\partial y} \right) dxdydz$$

## Heat Conduction

(if  $\lambda$  homogeneous)

$$\lambda \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right) dxdydz$$



# Energy Conservation (steady state, incompressible, $\lambda$ homogeneous)

Enthalpy Flows

Heat Conduction

$$\cancel{\rho} \cancel{u} \cancel{c_p} \frac{\partial T}{\partial x} + \cancel{\rho} \cancel{v} \cancel{c_p} \frac{\partial T}{\partial y} + \cancel{\rho} \cancel{w} \cancel{c_p} \frac{\partial T}{\partial z} = \lambda \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right)$$

$\downarrow$   
 $a = \frac{\lambda}{\rho c_p}$

+ Work against pressure,  
shear stresses,  
volume forces

Compared to Conservation of Momentum

Impulse Flows

Pressure    Shear Stresses

$$\cancel{\rho} \cancel{u} \frac{\partial u}{\partial x} + \cancel{\rho} \cancel{v} \frac{\partial u}{\partial y} + \cancel{\rho} \cancel{w} \frac{\partial u}{\partial z} = - \frac{\partial p}{\partial x} + \eta \left( \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right)$$

$\downarrow$   
 $\frac{1}{\rho}$

$\downarrow$   
 $\nu = \frac{\eta}{\rho}$

+ Volume forces  
(e.g. Gravitation)



# Similarity between Momentum and Energy transport

Momentum Flows

Pressure

Shear Stresses

$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \left( \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right)$$

$$u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} + w \frac{\partial T}{\partial z} =$$

$$\frac{\nu}{Pr} \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right)$$

Enthalpy Flows  
(advective transport)

Heat Conduction

## Prandtl number

$$Pr = \frac{\nu}{a} = \frac{\text{Diffusive Momentum transport}}{\text{Diffusive Heat transport}}$$



# Comprehension questions

---

**What is meant by a heat transfer coefficient and what does it describe?**

**Why does the Fourier's law of heat conduction also apply on the fluid side in the immediate vicinity of the wall?**

**What does the dimensionless Nusselt number mean?**

**What is the difference between natural and forced convection?**

