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# Heat Transfer

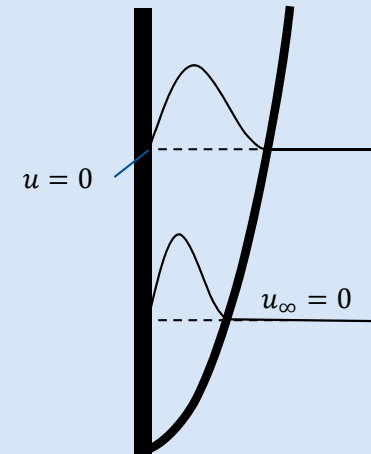
## Boundary Layer equations – Natural convection

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# Learning Goals

- Boundary Layer in Natural Convection

- Understanding the Boundary Layer profile (Temperature and Velocity) on a flat plate with natural (free) convection
- Derivation and meaning of the Grashof number
- Knowledge of the differences between the Boundary Layer profiles for forced and free convection



(Note: "natural" or "free" convection, both terms are identical and can be used equivalently.)

# Classification convection according to flow regime

## Forced Convection

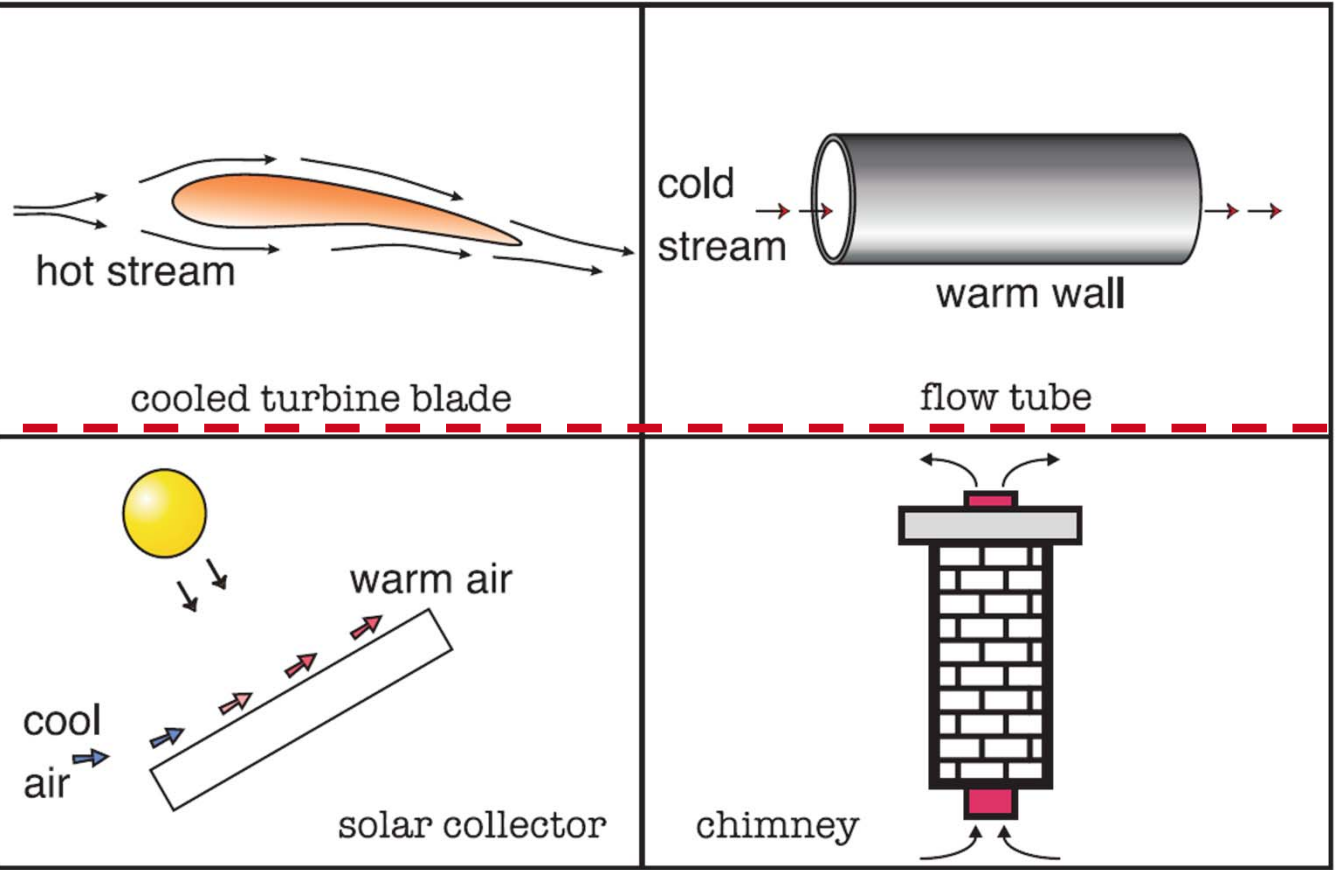
- Driven by externally generated movement of the fluid/object

## Free Convection

- Inherently driven due to heat transfer (density differences)

### External

### Internal

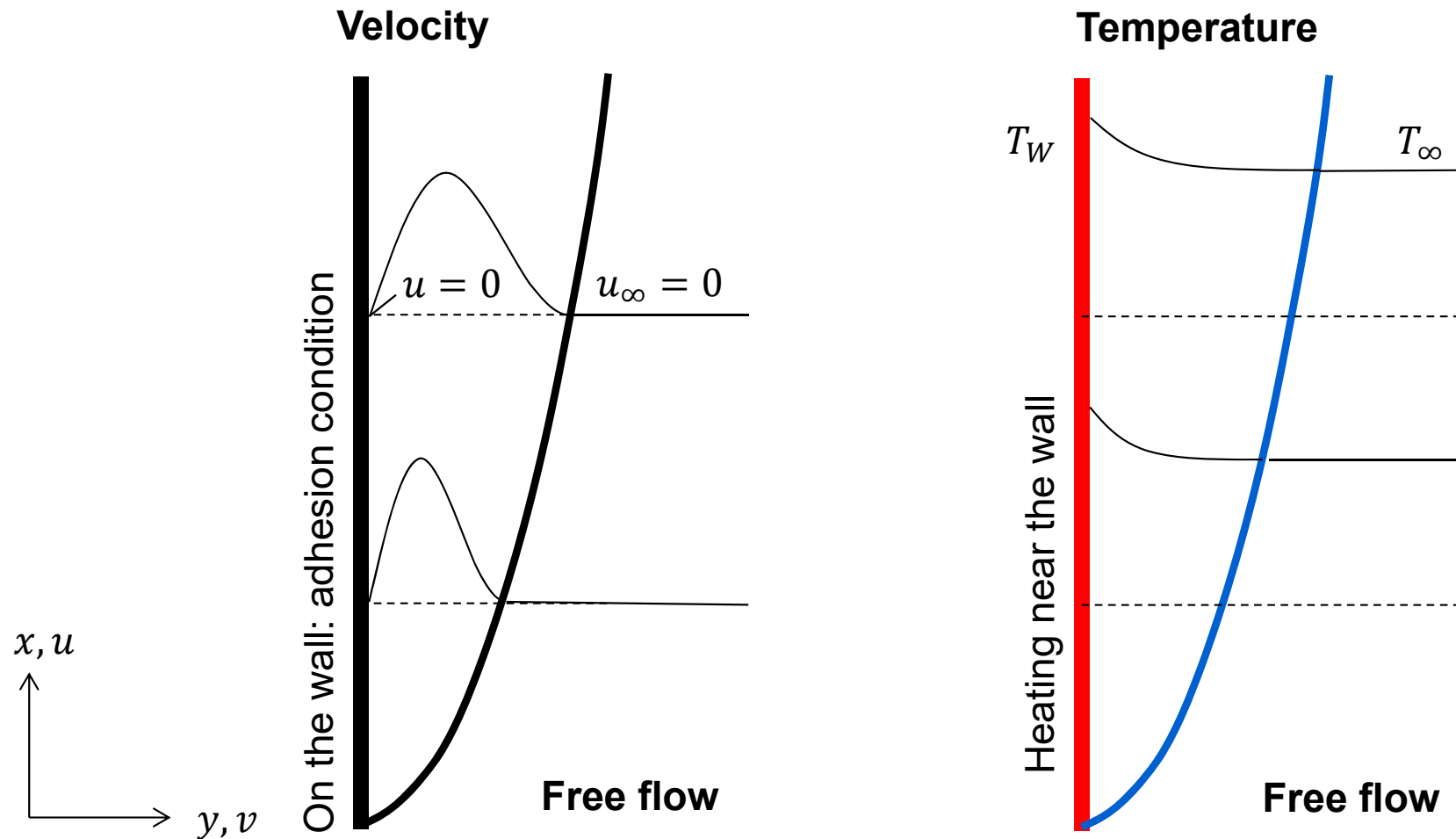


# Free Convection

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# Free Convection: Boundary Layers

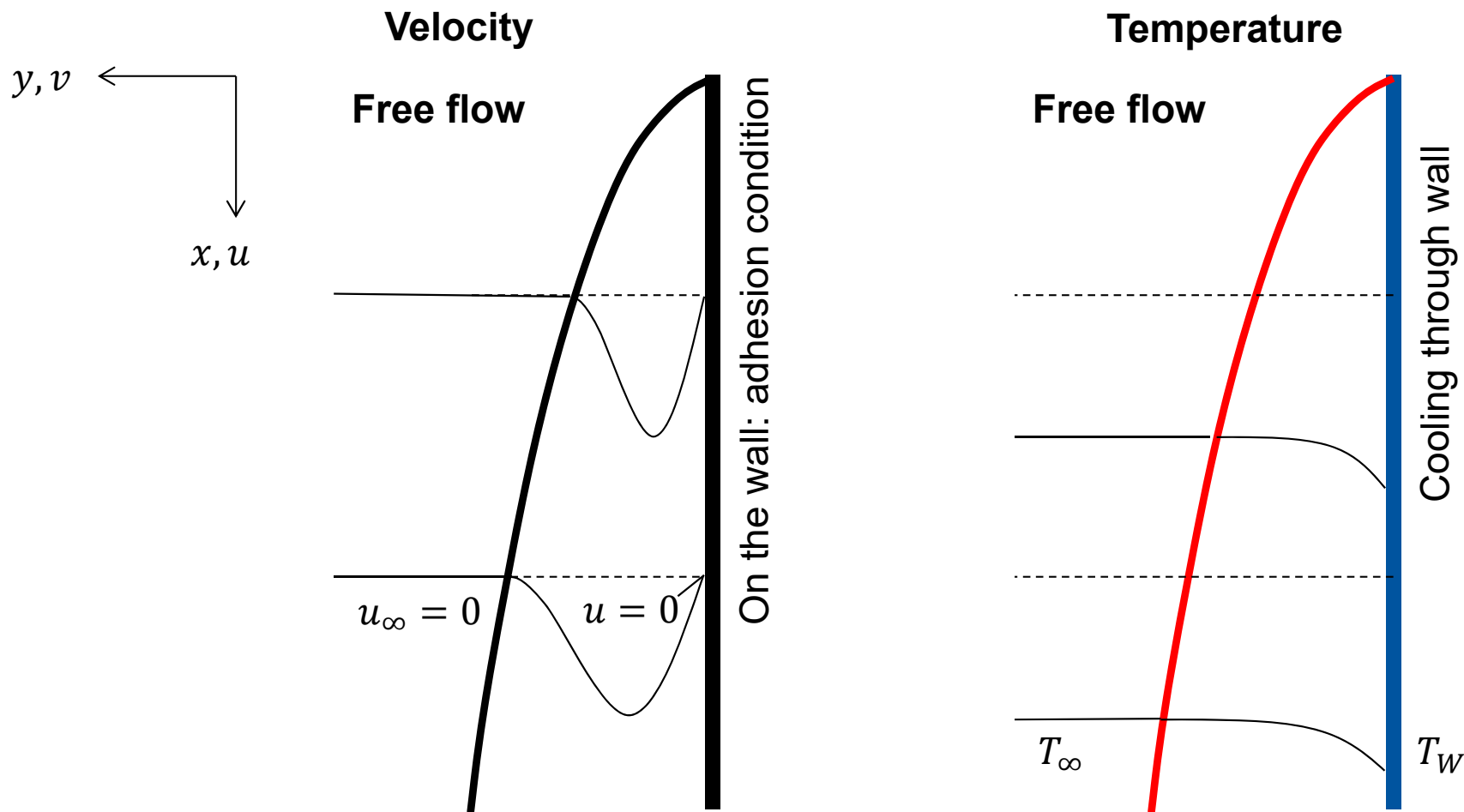


Heating near the wall leads to a decrease in density

**Natural Convection = Upward flow**



# Free Convection: Initial Situation



Cooling near the wall leads to an increase in density  
**Natural Convection = Downward flow**



# Review: Conservation equations (2D, steady state, incompressible)

Continuity  
equation

Mass Flow

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

$$\delta \ll L, \quad u \gg v \rightarrow \frac{\partial}{\partial x} \ll \frac{\partial}{\partial y}$$

Momentum  
equation

Momentum Flow

$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} =$$

Pressure

Shear stresses

Gravity

$$u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial y} + \nu \left( \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right)$$

$$+ \frac{g}{\rho} (\rho_{\infty} - \rho)$$

No pressure variation  
across boundary layer

$$\frac{\partial p}{\partial y} \ll \frac{\partial p}{\partial x} \approx 0$$

Energy  
equation

Enthalpy Flow

$$u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} =$$

Heat Conduction

$$\frac{\nu}{Pr} \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right)$$



# Conservation equations (2D, steady state, incompressible)

Continuity  
equation

Mass Flow

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

Coefficient of volumetric expansion

$$\beta \equiv \frac{1}{V} \left( \frac{\partial V}{\partial T} \right)_p = -\frac{1}{\rho} \left( \frac{\partial \rho}{\partial T} \right)_p = \frac{\rho_\infty - \rho}{\rho(T - T_\infty)}$$

Momentum  
equation

Momentum Flow

$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} =$$

Pressure

Shear stresses

Gravity

$$u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial y} + \nu \left( \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right)$$

$$+ \beta g (T - T_\infty)$$

negligible

Energy  
equation

Enthalpy Flow

$$u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} =$$

Heat Conduction

$$\frac{\nu}{Pr} \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right)$$



# Coefficient of volumetric expansion of an ideal Gas

$$p \cdot V = nRT$$

## Coefficient of volumetric expansion

$$\beta \equiv \frac{1}{V} \left( \frac{\partial V}{\partial T} \right)_p = -\frac{1}{\rho} \left( \frac{\partial \rho}{\partial T} \right)_p = \frac{\rho_\infty - \rho}{\rho(T - T_\infty)}$$

- $n$  Amount of substance
- $R = 8,314 \frac{\text{J}}{\text{mol} \cdot \text{K}}$  Universal gas constant

$$\frac{V}{T} = \frac{nR}{p} = \text{const.} \rightarrow \left( \frac{\partial V}{\partial T} \right)_p = \frac{V}{T}$$

## For ideal Gases

$$\beta = \frac{1}{T} \approx \frac{1}{T_{\text{mean}}} = \frac{2}{T_W + T_\infty}$$



# Conservation equations (2D, steady state, incompressible, plane boundary layer)

Continuity  
equation

Mass Flow

$$\frac{\partial u^*}{\partial x^*} + \frac{\partial v^*}{\partial y^*} = 0$$

Scaling (dimensionless variables)

$$x^* = \frac{x}{L}, y^* = \frac{y}{L}, u^* = \frac{u}{u_\infty}, v^* = \frac{v}{u_\infty}, \Theta^* = \frac{T - T_\infty}{T_W - T_\infty}$$

Momentum  
equation

Momentum Flow

$$u^* \frac{\partial u^*}{\partial x^*} + v^* \frac{\partial u^*}{\partial y^*} = \frac{1}{Re} \frac{\partial^2 u^*}{\partial y^{*2}}$$

Shear stresses

Gravity

$$+ \underbrace{\frac{\beta g L (T_W - T_\infty)}{u_\infty^2}}_{\frac{\beta g \rho^2 (T_W - T_\infty) L^3}{\eta^2} \cdot \left(\frac{\eta}{\rho u_\infty L}\right)^2} \Theta^*$$

$$\Rightarrow Gr \cdot \left(\frac{1}{Re}\right)^2$$

Energy  
equation

Enthalpy flow

$$u^* \frac{\partial \Theta^*}{\partial x^*} + v^* \frac{\partial \Theta^*}{\partial y^*} = \underbrace{\frac{1}{Re Pr}}_{Pe} \frac{\partial^2 \Theta^*}{\partial y^{*2}}$$

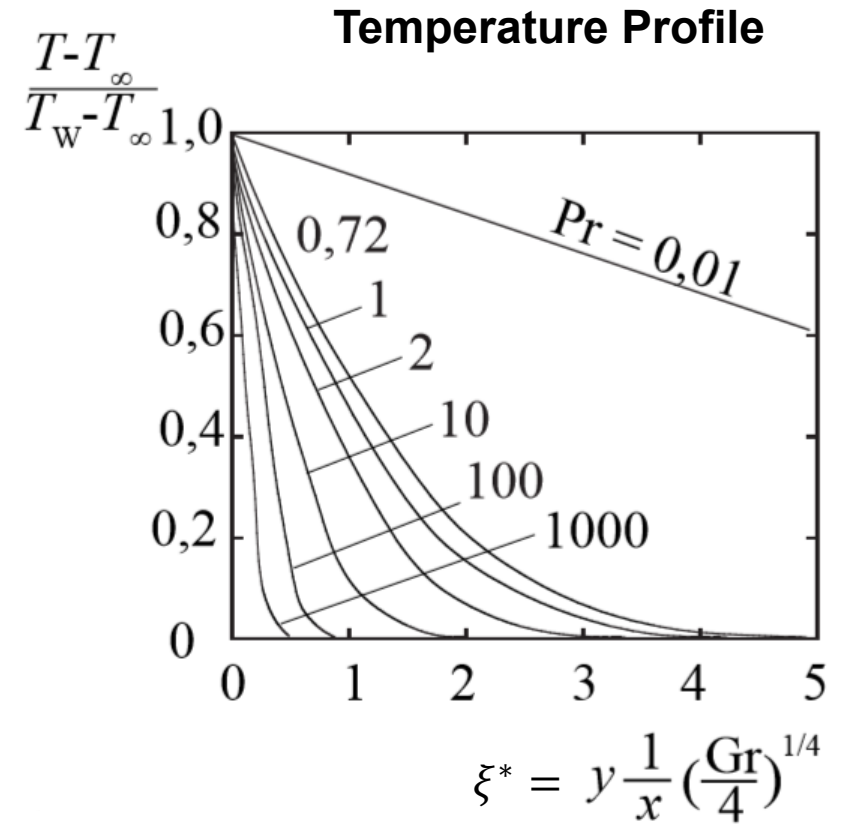
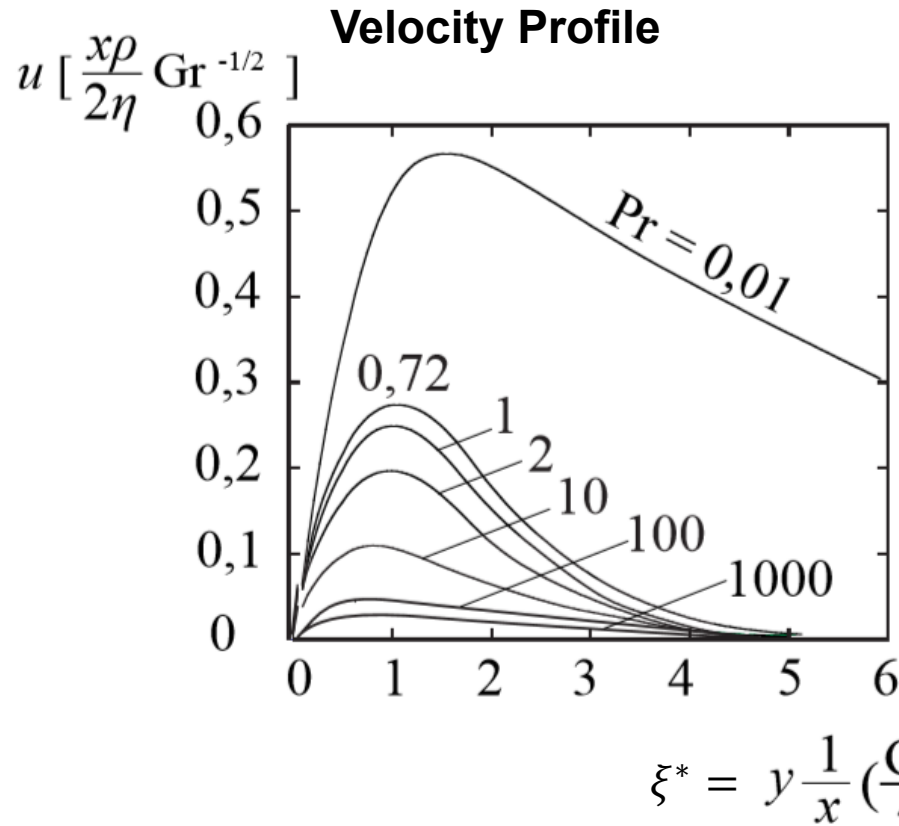
Heat Conduction

Grashof number

$$Gr \equiv \frac{\beta g \rho^2 (T_W - T_\infty) L^3}{\eta^2} = \frac{\text{Buoyancy forces}}{\text{Viscosity forces}}$$



# Exact Solutions



## Dimensionless Heat transfer coefficient

$$Nu = \frac{\alpha x}{\lambda} = \left( \lambda \frac{\partial T}{\partial y} \Big|_{y=0} \right) \frac{x}{\lambda} = \left( \frac{\text{Gr}}{4} \right)^{1/4} \frac{\partial \Theta^*}{\partial \xi^*} \Big|_{\xi^*=0} = Nu(\text{Gr}, \text{Pr})$$

# Comparison between Forced and Free convection

Convection	Forced (laminar $Re_x < 2 \cdot 10^5$ isothermal $0,6 < Pr < 10$ )	Free (laminar, isothermal $GrPr < 4 \cdot 10^9$ )
Dependence	$Nu(Re, Pr)$	$Nu(Gr, Pr)$
Local	$Nu = 0,332 Re_x^{\frac{1}{2}} Pr^{\frac{1}{3}}$	$Nu = 0,508 \left( \frac{Pr}{0,952 + Pr} \right)^{\frac{1}{4}} (Gr_x Pr)^{\frac{1}{4}}$
Average	$\overline{Nu} = \frac{\bar{\alpha}L}{\lambda} = \int_0^L \frac{Nu}{x} dx$ $= 0,664 Re_L^{\frac{1}{2}} Pr^{\frac{1}{3}}$	$\overline{Nu} = \frac{\bar{\alpha}L}{\lambda} = \int_0^L \frac{Nu}{x} dx$ $= \underbrace{0,677 \left( \frac{Pr}{0,952 + Pr} \right)^{\frac{1}{4}}}_{\bar{C}} (Gr_L Pr)^{\frac{1}{4}}$



# Comprehension Questions

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**What is the driving potential of Natural Convection?**

**Why are buoyancy forces negligible in Forced Convection?**

