



# AHSANULLAH UNIVERSITY OF SCIENCE AND TECHNOLOGY

## Department of Electrical and Electronic Engineering

### Assignment

Course No : EEE 4105

Course Name: Control System

Project Name: Water Level Control System

### Prepared By

Mohsin Islam (Rifat)

Student ID: 19.01.05.046

Section: A-2

Year: 4<sup>th</sup>

Semester: 1st

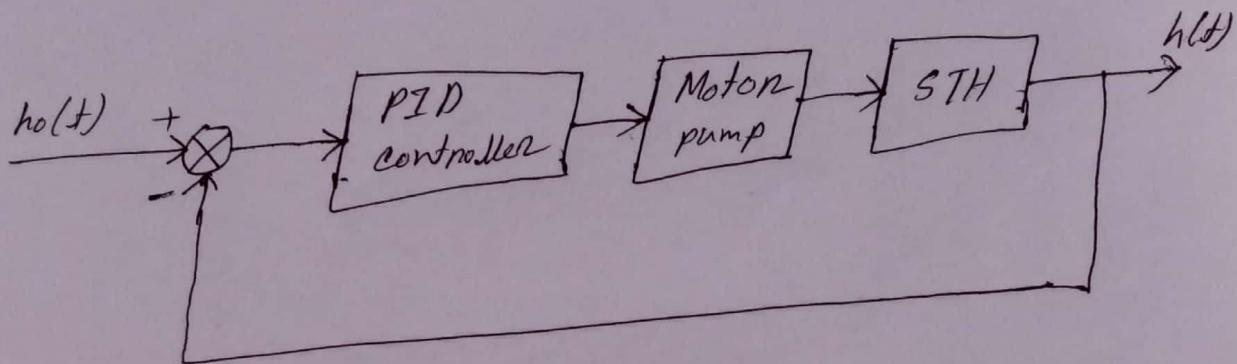
(i)

System: A system is a combination of elements acting together. For example: Electrical, Mechanical, Chemical etc.

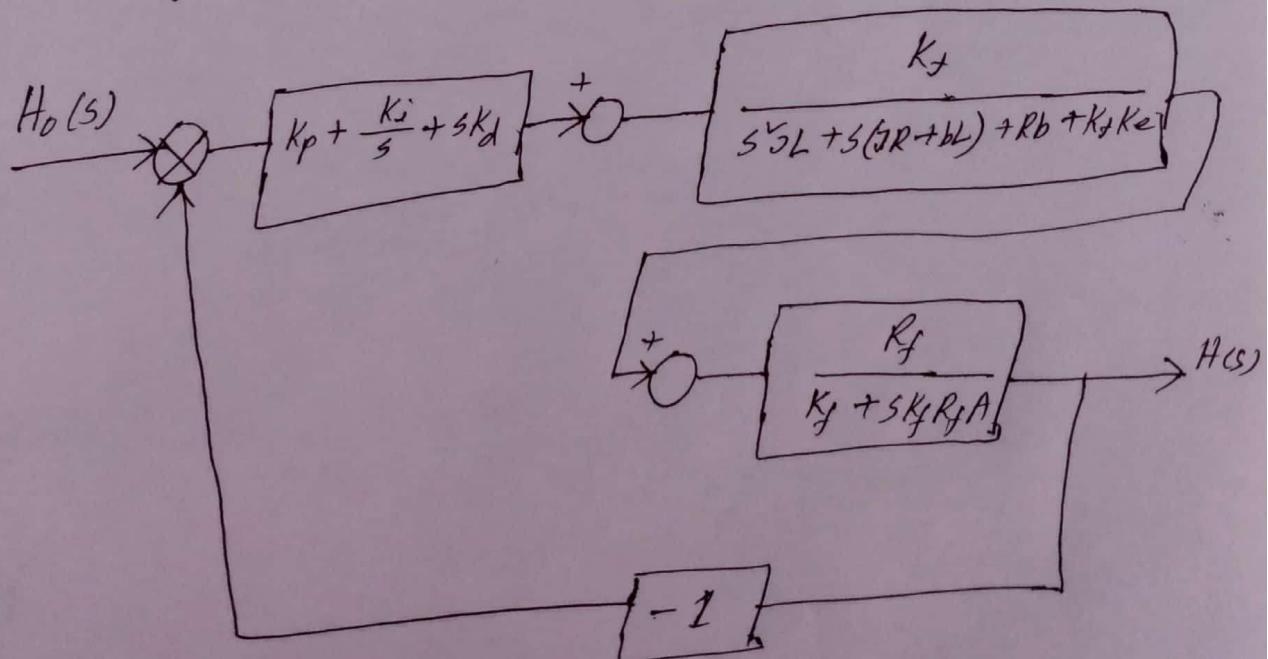
I have chosen "Water level control System". This system is chosen as it has practical applications in many industries such as agriculture, water treatment plants, flood control, irrigation system and so on. The system to be designed is a closed-loop control system for an automated water level control in a storage tank. That's why I chose a PID controlled "Water level control System" that can provide precise and stable control of water.

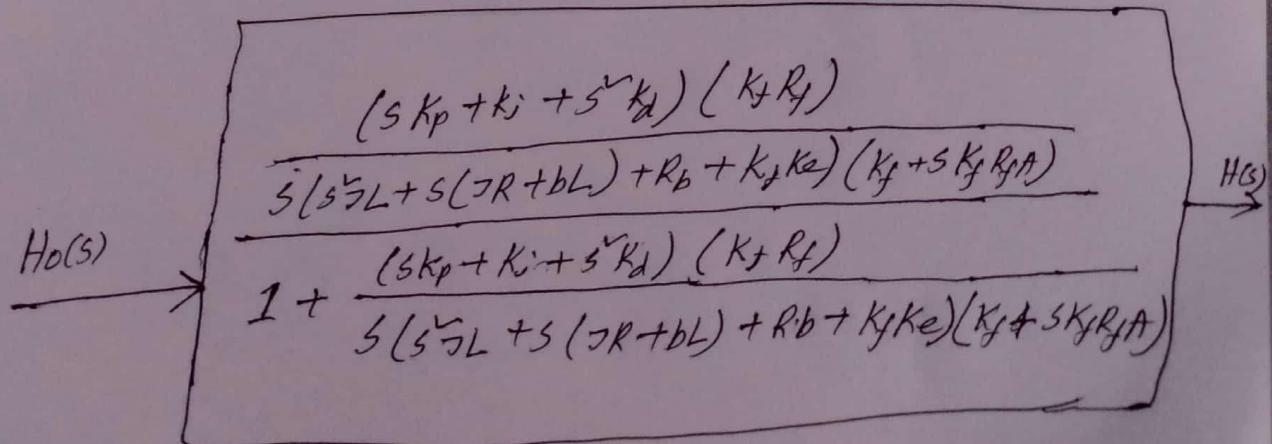
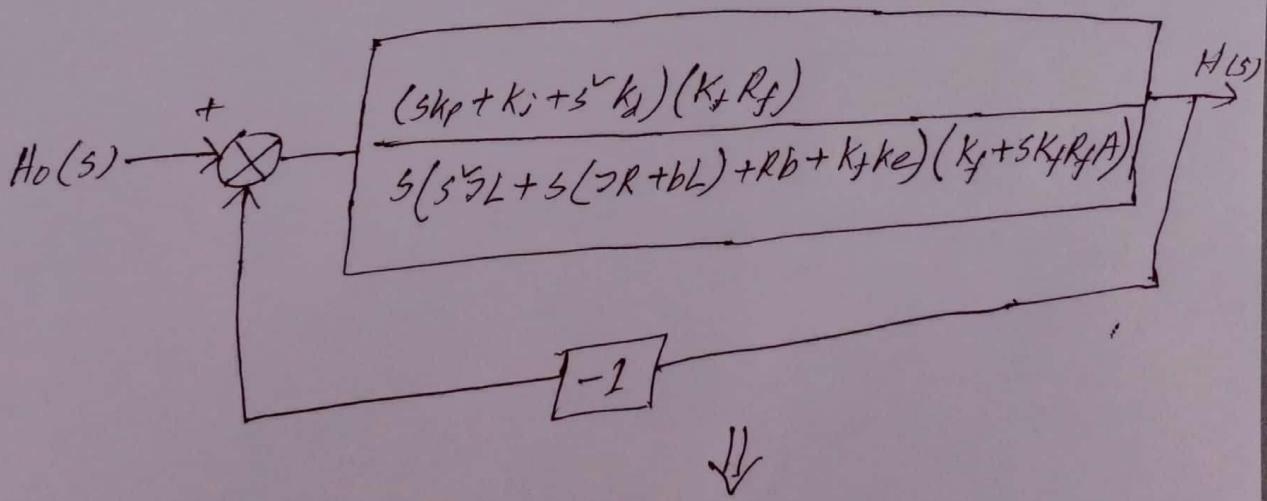
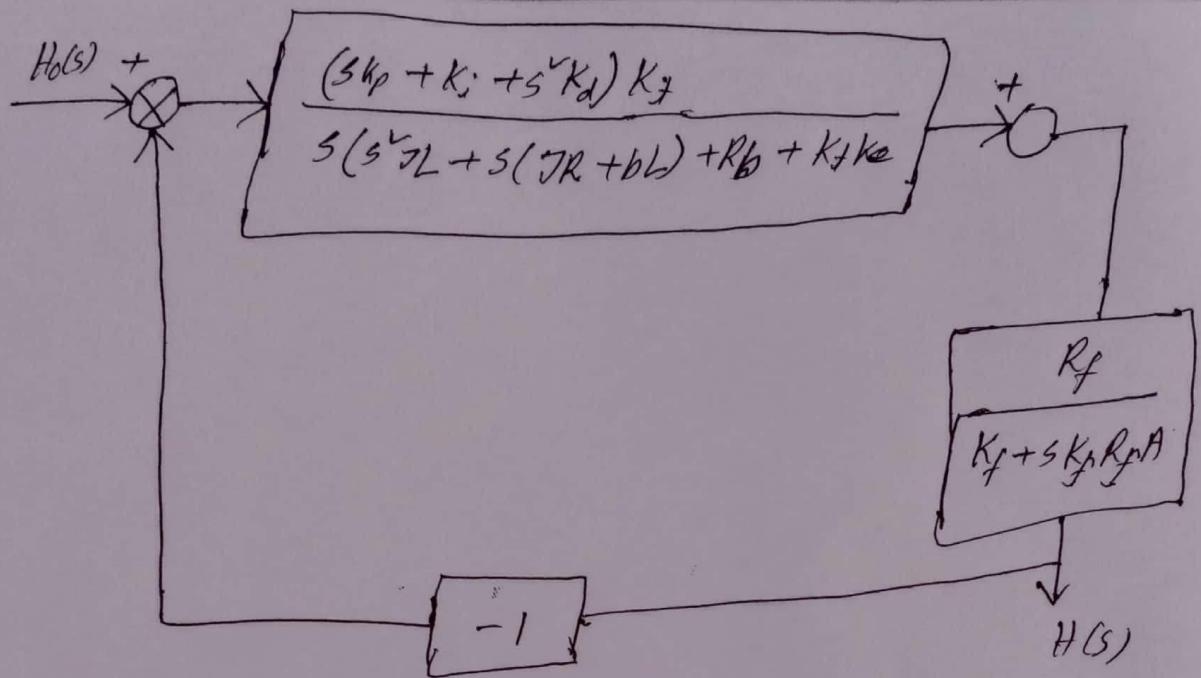
(ii)

The block model of my system:



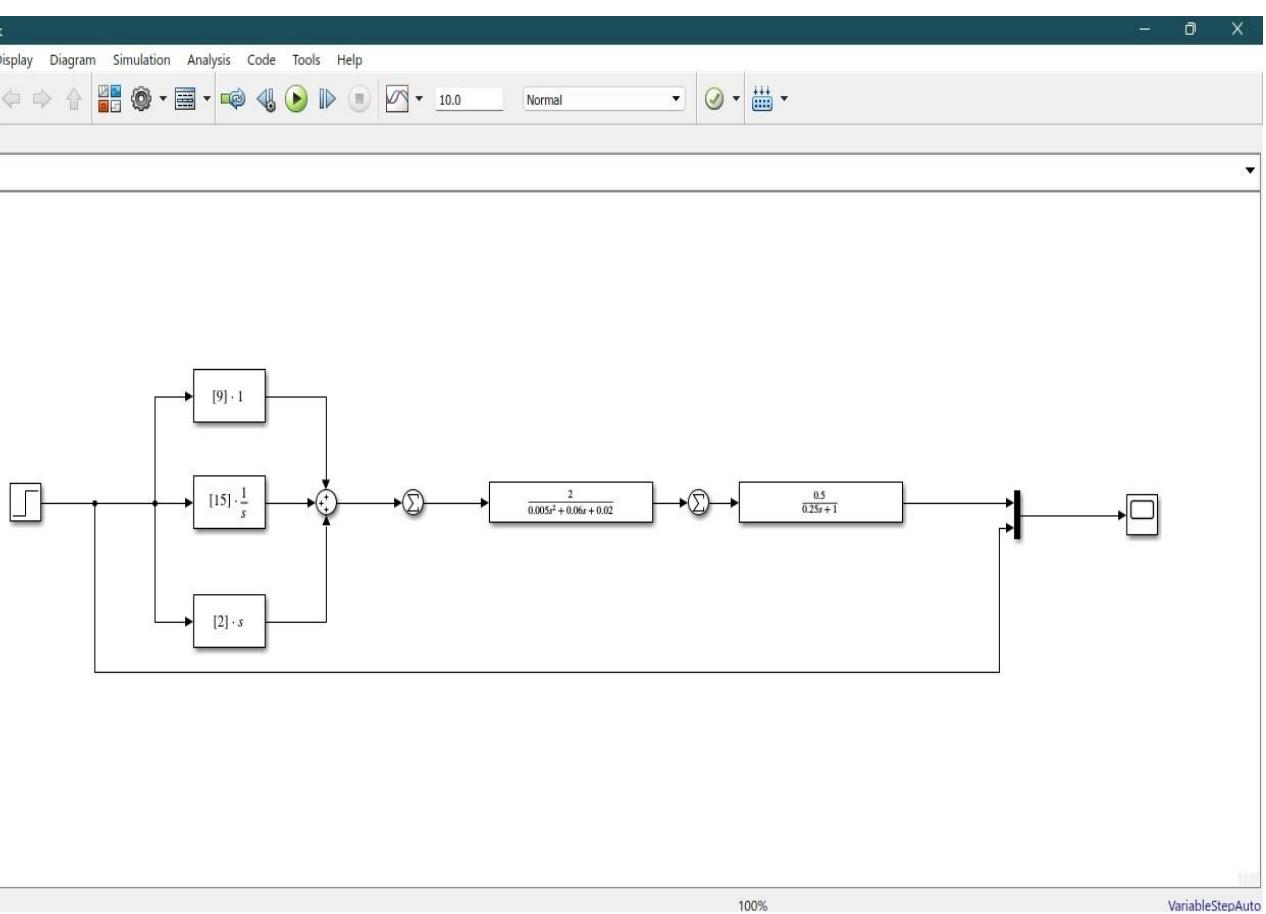
Now, putting the value of PID controller, motor pump and STH.

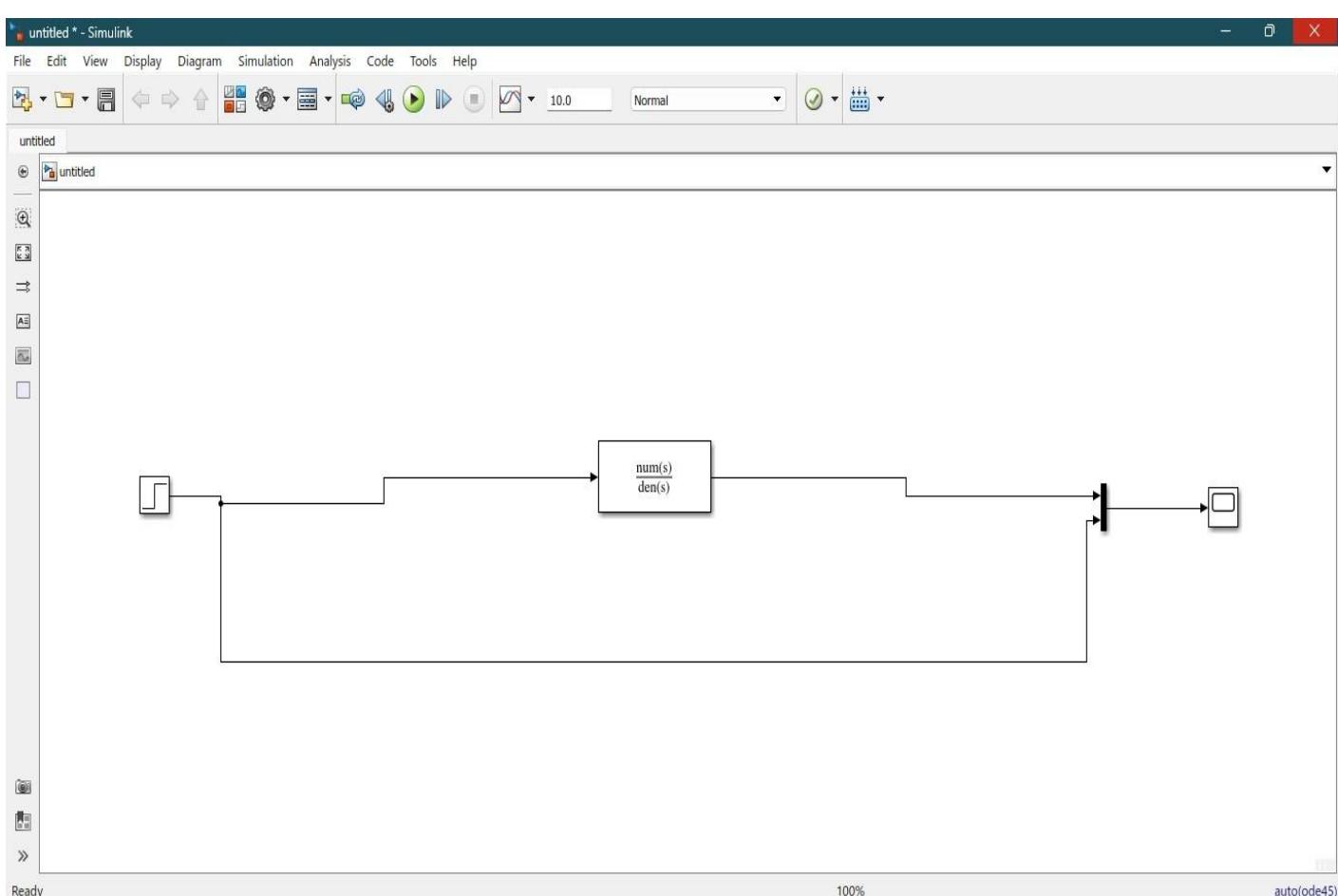
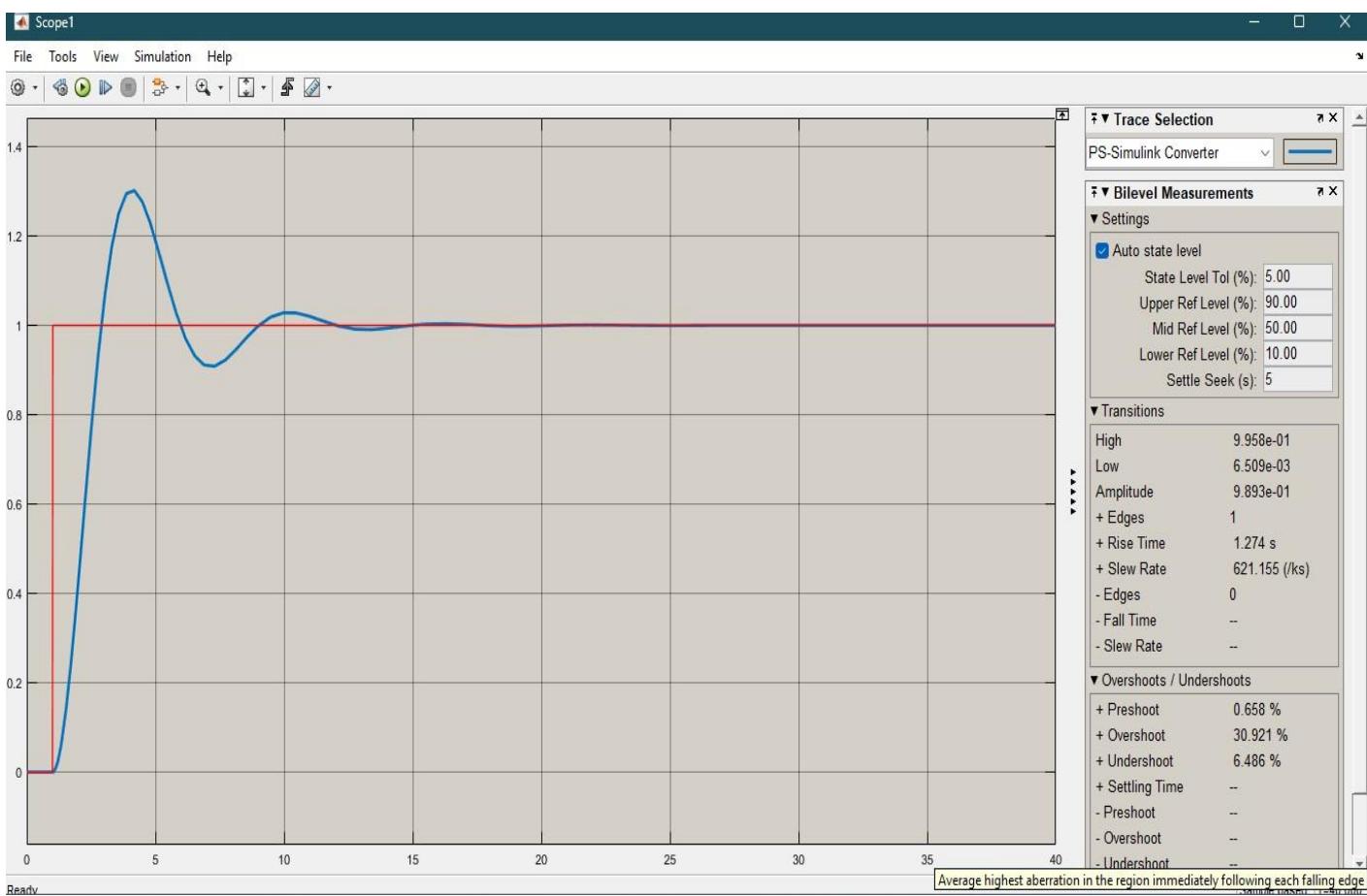


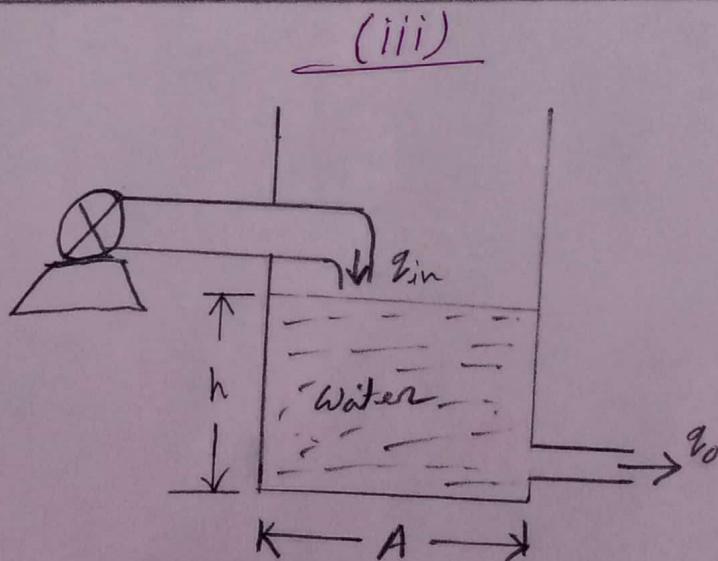
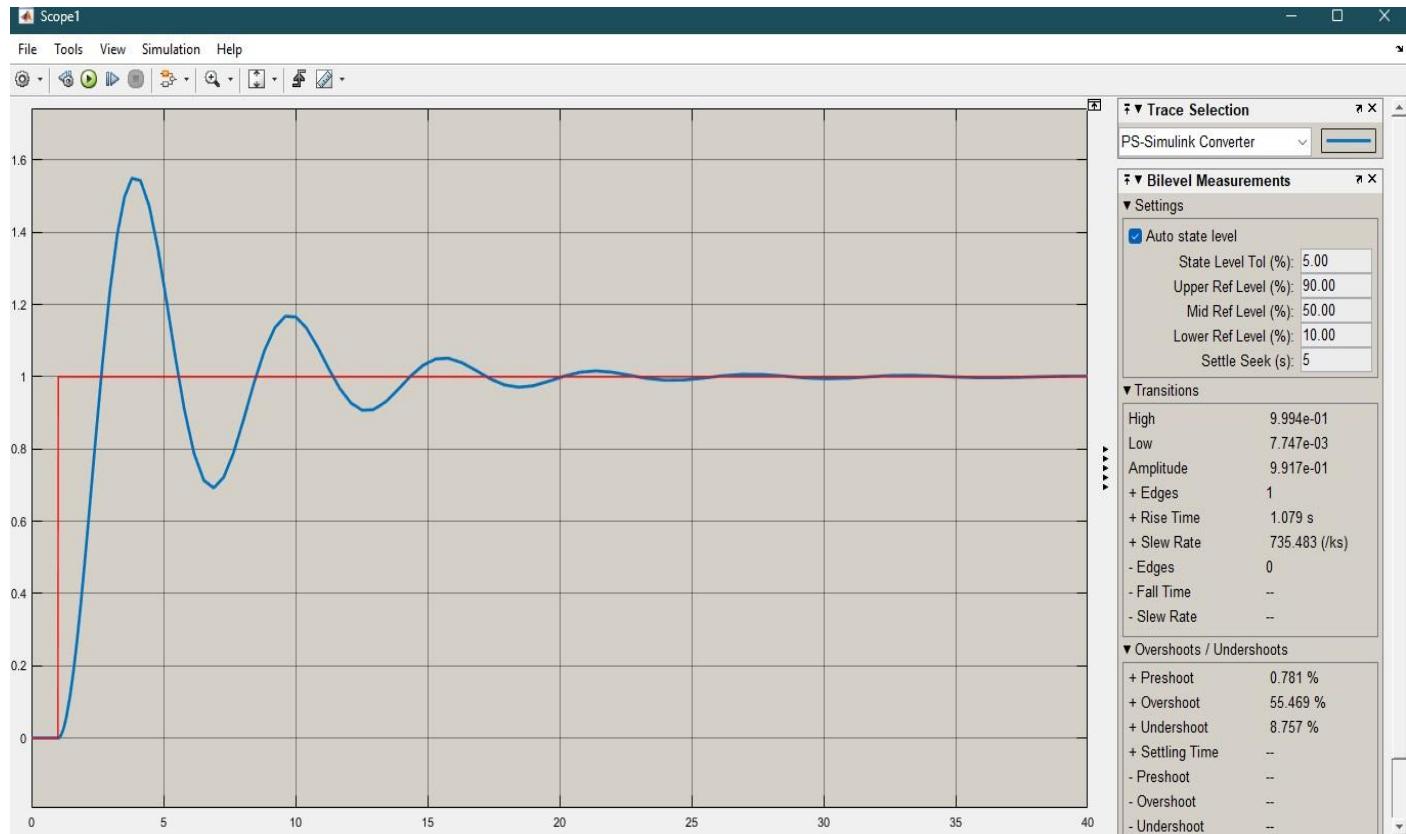


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$$\boxed{H_o(s) \rightarrow \frac{s(K_p + K_i + sK_d)(K_f R_f)}{s(s^2 L + s(JR + bL) + R_B + K_f K_e)(K_d + sK_f R_f A) + (sK_p + K_i + sK_d)(K_f R_f)}}$$







Here,  
Fig. Single Tank container filled to level  $h$   
 Water entering the container =  $q_{in}$

Water leaving " " " " =  $q_{out}$

Cross sectional Area =  $A$

Water level =  $h$

Now, using the balance of the flows into and out of the tank, the height 'h' is related to  $q_{in}$  and  $q_{out}$

$$q_{in} - q_{out} = A \frac{dh}{dt} \quad \dots \dots (1)$$

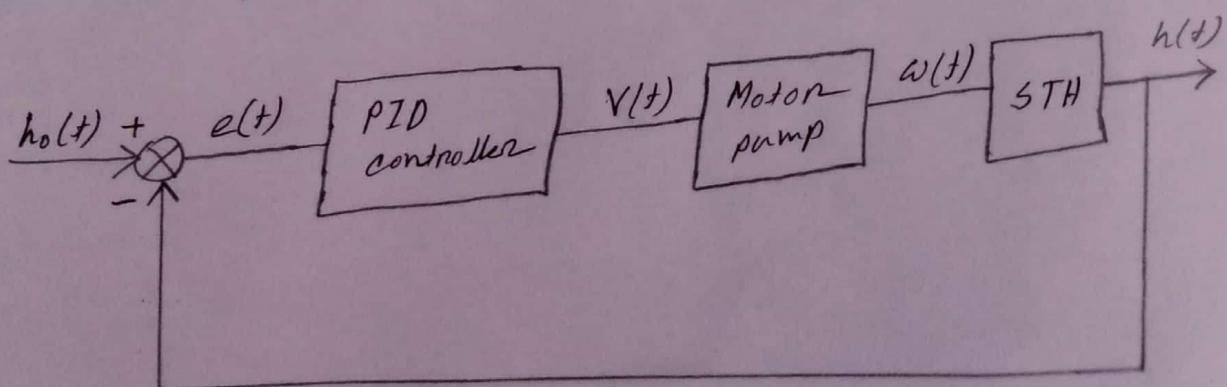
The out flow from the tank and the height can also be related assuming linear resistance to flow for simplicity of analysis and it gives at,

$$q_{out} = \frac{h}{R_f} \quad \text{where, } R_f = \text{Flow resistance} \quad \dots \dots (2)$$

We assume a simple linear relationship between the speed and the incoming flow rate to the given as,

$$\omega(t) = k_f q_{in}(t) \quad \dots \dots (3)$$

The block diagram of water controlling system,



Here, STH means Speed to Height transformation.

The DC motor equations relating the motor speed to the physical parameter of the motor can be written as,

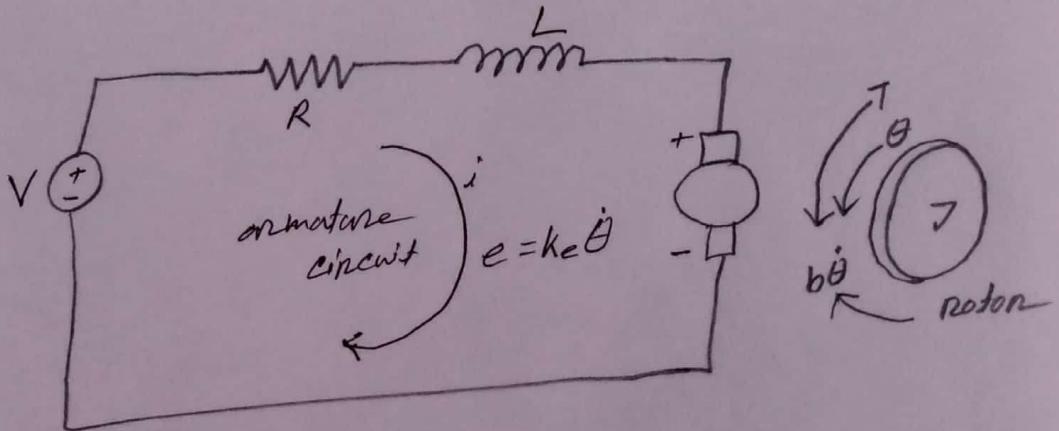
$$T\ddot{\theta} + b\dot{\theta} = k_t i \quad \text{--- (4)}$$

$$\Rightarrow L \frac{di}{dt} + R i = e = V - k_e \dot{\theta} \quad \text{--- (5)}$$

Here,  
 $k_e$  = electromotive force constant  
 $k_t$  = motor torque constant

$$\dot{\theta} = \frac{d\theta}{dt} = \omega$$

Equivalent circuit of the DC motor,



Using Laplace transformation,

$$s^2 T\theta(s) + s b\theta(s) = k_t I(s)$$

$$\Rightarrow sL I(s) + RI(s) = V(s) - s k_e \theta(s) \quad \text{--- (6)}$$

Now, the motor rotational speed output  $\omega(s)$  is related to the armature voltage input,  $V(s)$ ,

$$P_m(s) = \frac{\omega(s)}{V(s)} = \frac{k_f}{s^2 JL + s(JR + bL) + Rb + k_f k_e} \quad \dots \dots (7)$$

In general design of the PID, the error  $e(t)$  is related to the output  $V(t)$ ,

$$V(t) = K_p e(t) + k_i \int e(t) dt + k_d \frac{de(t)}{dt} \quad \dots \dots (8)$$

Using Laplace domain the input-output relation for the controller is,

$$C(s) = \frac{V(s)}{e(s)} = K_p + \frac{k_i}{s} + s k_d \quad \dots \dots (9)$$

Combining eqn (1), (2). and (3),  $\omega(t)$  is related to  $h(t)$ ,

$$R_f \omega(t) - k_f h = k_f R_f A \frac{dh}{dt} \quad \dots \dots (10)$$

Eqn (10) in Laplace domain, we have .

$$R_f \omega(s) - k_f h(s) = k_f R_f A s h(s) \quad \dots \dots (10)$$

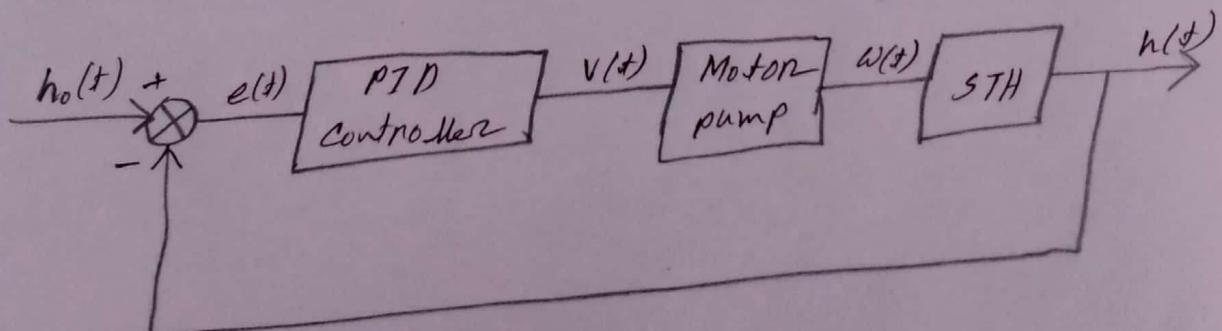
So, the transfer function is,

$$G_2(s) = \frac{h(s)}{\omega(s)} = \frac{R_f}{K_f + s K_f R_f A} \quad \dots \dots (12)$$

The overall transfer function of the forward path,

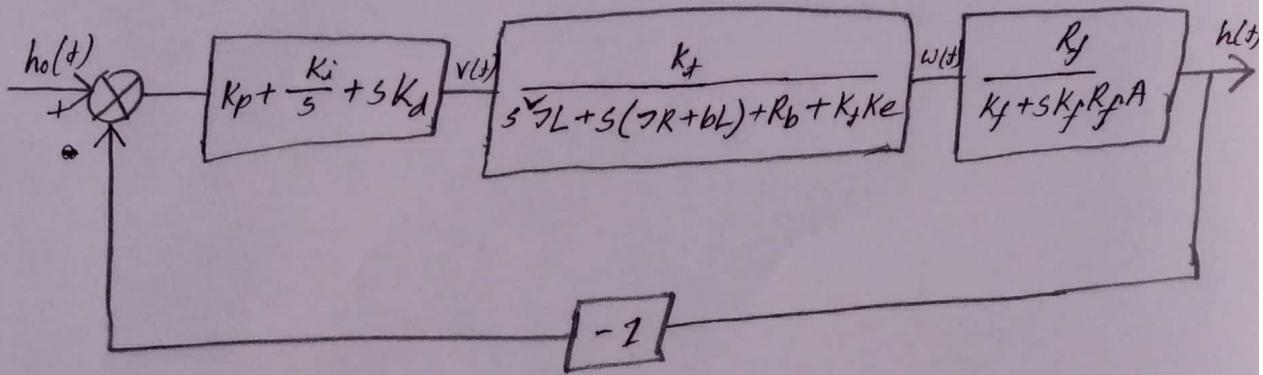
$$\begin{aligned} F(s) &= \frac{h(s)}{e(s)} = C(s) P_m(s) G_2(s) \\ &= \left( K_p + \frac{K_i}{s} + s K_d \right) \left( \frac{K_f}{s^2 J_L + s(JR + bL) + R_b + K_f K_e} \right) \\ &\quad \times \left( \frac{R_f}{K_f + s K_f R_f A} \right) \end{aligned} \quad \dots \dots (13)$$

The block diagram of water controlling system:

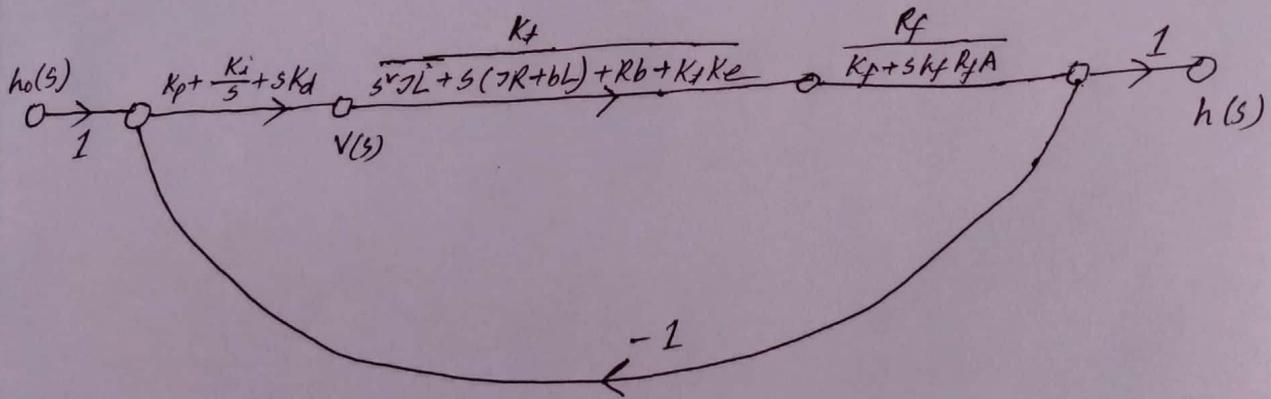


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After putting the value of PID controller, Motor pump, and STH. Then the block diagram will be



So the signal flow graph (SFG) of the system will be,



Now,

$$\begin{aligned}
 P_1 &= \left( K_p + \frac{K_i}{s} + sK_d \right) \left( \frac{K_f}{s^2JL + s(\gamma R + bL) + R_b + K_f K_e} \right) \left( \frac{R_f}{K_f + sK_f R_f A} \right) \\
 &= \left( \frac{sK_p + K_i + s^2K_d}{s} \right) \left( \frac{K_f R_f}{(s^2JL + s(\gamma R + bL) + R_b + K_f K_e)(K_f + sK_f R_f A)} \right) \\
 &= \frac{(sK_p + K_i + s^2K_d)(K_f R_f)}{s(s^2JL + s(\gamma R + bL) + R_b + K_f K_e)(K_f + sK_f R_f A)}
 \end{aligned}$$

$$\Delta = 1 + \frac{(sK_p + K_i + s^v K_d)(K_f R_f)}{s(s^v \tau_L + s(\tau_R + bL) + R_b + K_f K_e)(K_f + sK_f R_f A)}$$

$$= \frac{s(s^v \tau_L + s(\tau_R + bL) + R_b + K_f K_e)(K_f + sK_f R_f A) + (sK_p + K_i + s^v K_d)(K_f R_f)}{s(s^v \tau_L + s(\tau_R + bL) + R_b + K_f K_e)(K_f + sK_f R_f A)}$$

$$\Delta_1 = 1$$

Now the transfer function will be,

$$T.F = \frac{h(s)}{h_0(s)} = \frac{P_1 \Delta_1}{\Delta}$$

$$= \frac{(sK_p + K_i + s^v K_d)(K_f R_f)}{s(s^v \tau_L + s(\tau_R + bL) + R_b + K_f K_e)(K_d + sK_f R_f A) + (sK_p + K_i + s^v K_d)(K_f R_f)}$$

--- (14)

let,

$$J = 0.01 \text{ kg m}^2$$

$$b = 0.1 \text{ N.m.s}$$

$$k_f = 0.1 \text{ N.m/A}$$

$$K_e = 0.02 \text{ V/rad/s}$$

$$k_f = 1$$

$$k_p = 9$$

$$K_i = 15$$

$$K_d = 2$$

(iv)

$$R = 2 \Omega$$

$$L = 0.5 H$$

$$A = 0.5 \text{ m}^2$$

$$R_f = 0.5 \text{ s/m}^2$$

From eqn (14),

$$T.F = \frac{H(s)}{H_0(s)} = \frac{0.15 + 0.45s + 0.75}{0.00125s^4 + 0.02s^3 + 0.0015s^2 + 0.101s}$$

$$= \frac{0.15 + 0.45s^{-3} + 0.75s^{-4}}{0.00125 + 0.02s^{-1} + 0.0015s^{-2} + 0.101s^{-3}}$$

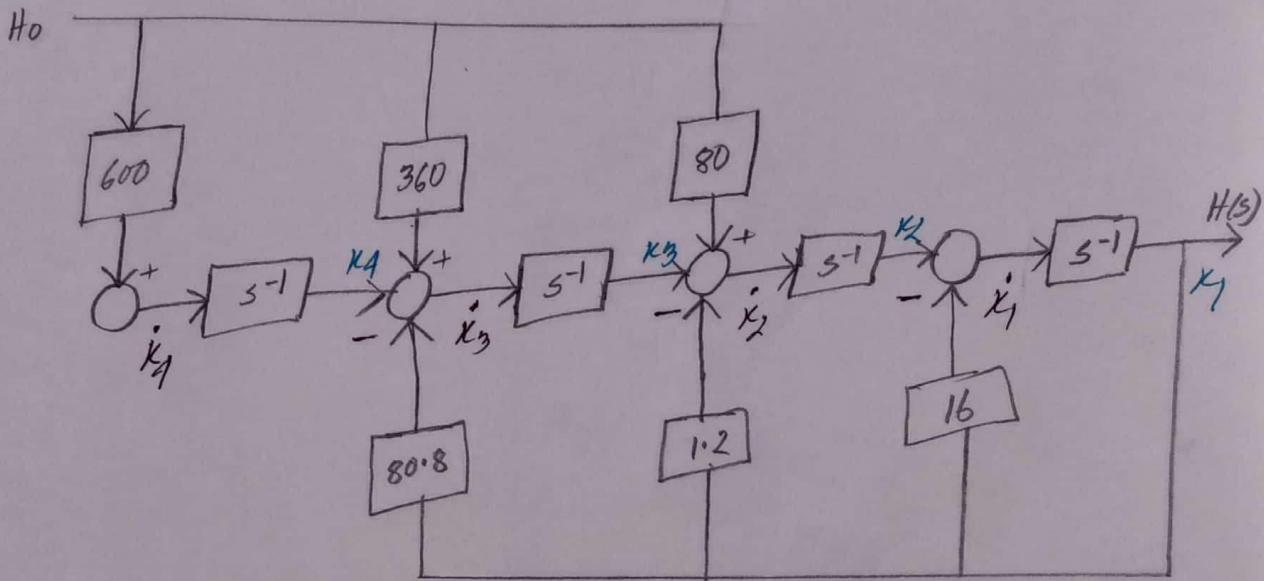
[Multiply by  $s^{-4}$ ]

$$\frac{H(s)}{H_0(s)} = \frac{80s^{-2} + 360s^{-3} + 600s^{-4}}{1 + 16s^{-1} + 12s^{-2} + 80.8s^{-3}}$$

[Multiply by 800]

$$\Rightarrow H(s) = 80s^{-2}H_0(s) + 360s^{-3}H_0(s) + 600s^{-4}H_0(s) - 16s^{-1}H(s) - 12s^{-2}H(s) \\ + 80.8s^{-3}H(s)$$

Now,



$$\dot{x}_1 = -16x_1 + x_2$$

$$\dot{x}_2 = -1.2x_1 + x_3 + 80H_0$$

$$\dot{x}_3 = -80.8x_1 + x_4 + 360H_0$$

$$\dot{x}_4 = 600H_0$$

Now,

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \begin{bmatrix} -16 & 1 & 0 & 0 \\ -1.2 & 0 & 1 & 0 \\ -80.8 & 0 & 0 & 1 \\ 0 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} + \begin{bmatrix} 0 \\ 80 \\ 360 \\ 600 \end{bmatrix} H_0(t)$$

$$y = x_1$$

$$\therefore y = \begin{bmatrix} 1 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix}$$

(V)

From eqn (13),

$$F(s) = \left( K_p + \frac{K_i}{s} + sK_d \right) \left( \frac{K_f}{s^2 J L + s(JR + bL) + R_b + K_f K_e} \right) X \left( \frac{R_f}{K_f + sK_f R_f A} \right)$$

Now, assumed the parameter values  
for the motor, the tank and the STH system:

$$\begin{aligned} J &= 0.01 \text{ kg m}^2 \\ b &= 0.1 \text{ Nms} \\ K_f &= 2 \text{ N.m/A} \\ A &= 0.5 \text{ m}^2 \end{aligned} \quad \left\{ \begin{aligned} K_e &= 0.01 \text{ V/rad/s} \\ R &= 1 \Omega \\ L &= 0.5 \text{ H} \\ R_f &= 0.5 \text{ n/m} \end{aligned} \right.$$

$$\begin{aligned} \therefore F(s) &= \left( K_p + \frac{K_i}{s} + sK_d \right) \left( \frac{2}{0.005s^2 + 0.06s + 0.02} \right) \left( \frac{0.05}{1 + 0.25s} \right) \\ &= \underbrace{\left( K_p + \frac{K_i}{s} + sK_d \right)}_{\text{PID controller}} \left( \frac{1}{0.00125s^3 + 0.02s^2 + 0.0015s + 0.02} \right) \end{aligned}$$

Initially consider an uncompensated system,

$$\text{So, } G_2(s)H(s) = \frac{K}{0.00125s^3 + 0.02s^2 + 0.0015s + 0.02} \quad (15)$$

(V) Continue

Now,

$$G(s)H(s) = \frac{K}{0.00125s^3 + 0.025s^2 + 0.0015s + 0.02}$$

Now, the characteristic eqn,

$$1 + G(s)H(s) = 0$$

$$\Rightarrow 1 + \frac{K}{0.00125s^3 + 0.025s^2 + 0.0015s + 0.02} = 0$$

$$\Rightarrow 0.00125s^3 + 0.025s^2 + 0.0015s + 0.02 + K = 0$$

Now, Routhian Array,

$s^3$	0.00125	0.0015
$s^2$	0.02	$0.02 + K$
$s^1$	$\frac{3 \times 10^{-5} - 2.5 \times 10^{-3} - 0.00125K}{0.02}$	0
$s^0$	$0.02 + K$	0

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Here,

$$\frac{3 \times 10^{-5} - 2.5 \times 10^{-3} - 0.00125K}{0.02} = 0$$

$$\Rightarrow 3 \times 10^{-5} - 2.5 \times 10^{-3} - 0.00125K = 0$$

$$\therefore K = -1.976$$

Here, all the elements in the first column  
are always positive if the value of  $K < -1.976$   
positive, negative and zero can be depending  
upon the value of  $K$ .

If  $K < -1.976$  then all the terms in the first  
column will be positive and since there  
are no sign changes, then the system will  
be stable.

If  $K > -1.976$  then 5' term in the first column  
is negative. There are two sign changes, indicating  
that the system has two right half plane poles.

which makes the system unstable.

If  $K = -1.976$  the entire row is zero<sup>o</sup> so,

$$\begin{array}{c|cc|c}
 s^3 & 0.00125 & 0.0015 \\
 \hline
 s^2 & 0.02 & 0.02 - 1.976 \rightarrow 0.02s^2 - 1.956 = u(s) \\
 \hline
 s^1 & \varnothing = 0.04 & 0 \\
 \hline
 s^0 & 0.02 - 1.976 &
 \end{array}$$

$$\frac{d(s)}{ds} = 0.04$$

Now,

$$0.02s^2 - 1.956 = 0$$

$$\Rightarrow 0.02s^2 = 1.956$$

$$\therefore s = \pm 9.89j$$

Here, roots are not repeated and placed on the imaginary axis. So the system is marginal stable.

(vi)

From eqn (15),

$$G(s) H(s) = \frac{K}{0.00125s^3 + 0.02s^2 + 0.0015s + 0.02}$$

$$= \frac{K}{(s+15.99)(s+6.23 \pm j1)}$$

Here,

3 open-loop poles at,

$$s = -15.99$$

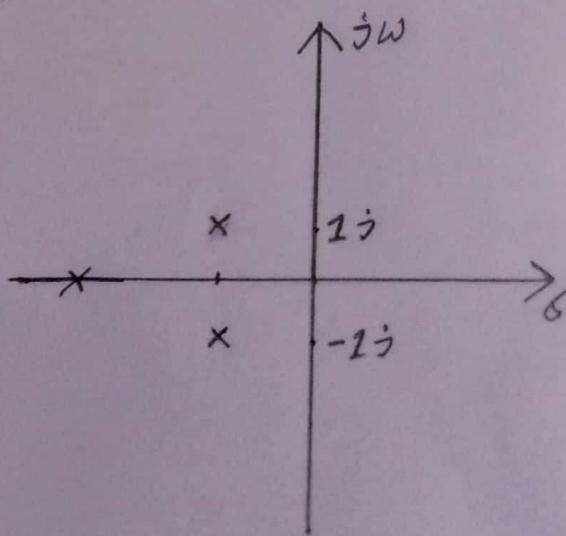
$$s = -6.23 + j1$$

$$s = -6.23 - j1$$

and, no open-loop zeros.

So, No of poles,  $n_p = 3$

No of zeros,  $n_z = 0$



Now the characteristic equation,

$$1 + G(s)H(s) = 0$$

$$\Rightarrow 1 + \frac{K}{0.00125s^3 + 0.02s^2 + 0.0015s + 0.02} = 0$$

$$\Rightarrow 0.00125s^3 + 0.02s^2 + 0.0015s + 0.02 + K = 0 \quad \dots (16)$$

$$\Rightarrow K = - (0.00125s^3 + 0.02s^2 + 0.0015s + 0.02)$$

$$\therefore \frac{dK}{ds} = 0$$

$$\Rightarrow \frac{d}{ds} \left\{ - (0.00125s^3 + 0.02s^2 + 0.0015s + 0.02) \right\} = 0$$

$$\Rightarrow -0.00375s^2 - 0.045 - 0.0015 = 0$$

$$\Rightarrow 0.00375s^2 + 0.045 + 0.0015 = 0$$

$$\therefore s = -0.0376, -19.629$$

Centre of asymptotes,

$$G_H = \frac{\sum_p - \sum_z}{n_p - n_z} = \frac{-15.99 - 6.23 + j1 - 6.23 - j1.0}{3 - 0}$$

$$= -9.48$$

Angle of asymptotes,

$$\varphi_A = \frac{2q+1}{np-nz} \times 180^\circ$$

$$\begin{aligned} q &= 0 \text{ to } (np-nz-1) \\ &= 0 \text{ to } (3-0-1) \\ &= 0, 1, 2 \end{aligned}$$

$$\therefore \varphi_0 = \frac{2 \times 0 + 1}{3 - 0} \times 180^\circ = 60^\circ$$

$$\varphi_1 = \frac{2 \times 1 + 1}{3 - 0} \times 180^\circ = 180^\circ$$

$$\varphi_2 = \frac{2 \times 2 + 1}{3 - 0} \times 180^\circ = 300^\circ$$

Now, from eqn (16),

$$0.00125s^3 + 0.02s^2 + 0.0015s + 0.02 + k = 0$$

Routhian Array,

$s^3$	0.00125	0.0015
$s^2$	0.02	(0.02 + k)
$s^1$	$\frac{3 \times 10^{-5} - 2.5 \times 10^{-3} - 0.00125k}{0.02}$	0
$s^0$	0.02 + k	0

Here,

$$\frac{3 \times 10^{-5} - 2.5 \times 10^{-3} - 0.00125K}{0.02} = 0$$

$$\therefore K = 4.166$$

Here,

$$0.02s^2 + 0.02 + K = 0$$

$$\Rightarrow 0.02s^2 + 0.02 + 4.166 = 0$$

$$\therefore s = \pm j 14.47$$

Angle of departure at the complex pole,

$$\sum \angle Z - \sum \angle P = -180^\circ$$

$$\Rightarrow 0 - (\theta_d + 90^\circ + 5.85) = -180^\circ$$

$$\therefore \theta_d = 84.15^\circ$$

Gain at operating point,

$$\left| \frac{K}{(s+15.99)(s+6.23+j)} \right|_{s=4.85+j14.47} = 1$$

$$\therefore K = 510.37$$

$$\tan \theta = \frac{15.99 - 6.23}{9.76}$$

$$\theta = 5.85^\circ$$

*operating point*

$$-1.69 + j6.18$$

$$j6.18$$

$$-15.97$$

$$-9.62$$

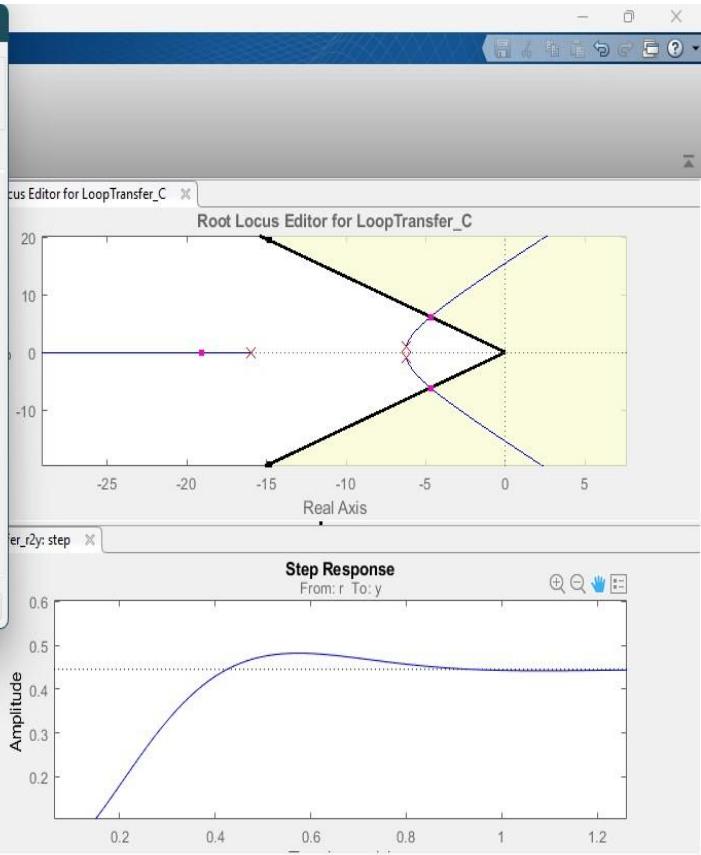
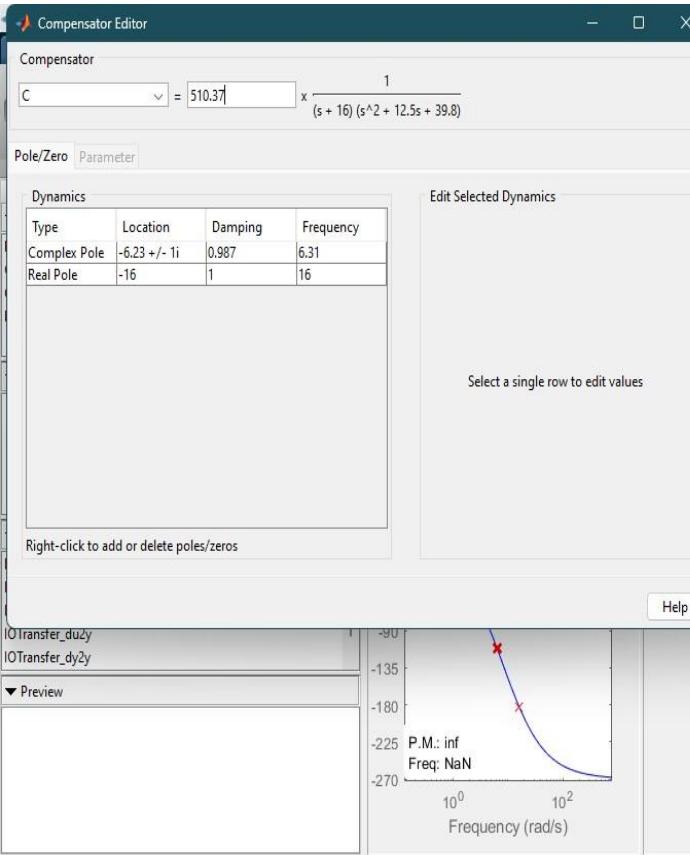
$$-1.69$$

$$-1j$$

$$6$$

$$-j14.17$$

Scanned with CamScanner



(Vii)

For PID compensator, first of all I will design a PD compensator.

From the graph,

$$\omega_n = 4.69$$

$$\Rightarrow \omega_n = \frac{4.69}{0.607} \quad \left[ \begin{array}{l} \text{As assumed} \\ \zeta = 0.607 \end{array} \right]$$

$$= 7.72$$

Now,

$$T_s = \frac{4}{\zeta \omega_n}$$

$$= \frac{4}{0.607 \times 7.72}$$

$$= 0.853 \text{ sec}$$

$$K_p = \lim_{s \rightarrow 0} K G(s) H(s)$$

$$= \lim_{s \rightarrow 0} \frac{K}{(s + 15.99)(s + 6.23 \pm j)}$$

$$= \frac{510.37}{15.99 \times (6.23 \pm j)}$$

$$= 4.995$$

$$\therefore E_{ss} = \frac{1}{1 + K_p}$$

$$= \frac{1}{1 + 4.995}$$

$$= 0.167$$

So the error for the uncompensated system  
is 0.167

Now, we know,

$$\text{Peak time, } T_p = \frac{\pi}{\omega_n \sqrt{1 - \zeta^2}}$$

$$= \frac{\pi}{7.72 \sqrt{1 - (0.607)^2}}$$

$$= 0.512$$

The imaginary part of the compensated dominant

$$\omega_d = \frac{\pi}{T_p}$$

$$= \frac{\pi}{\frac{1}{4} \times 0.512}$$

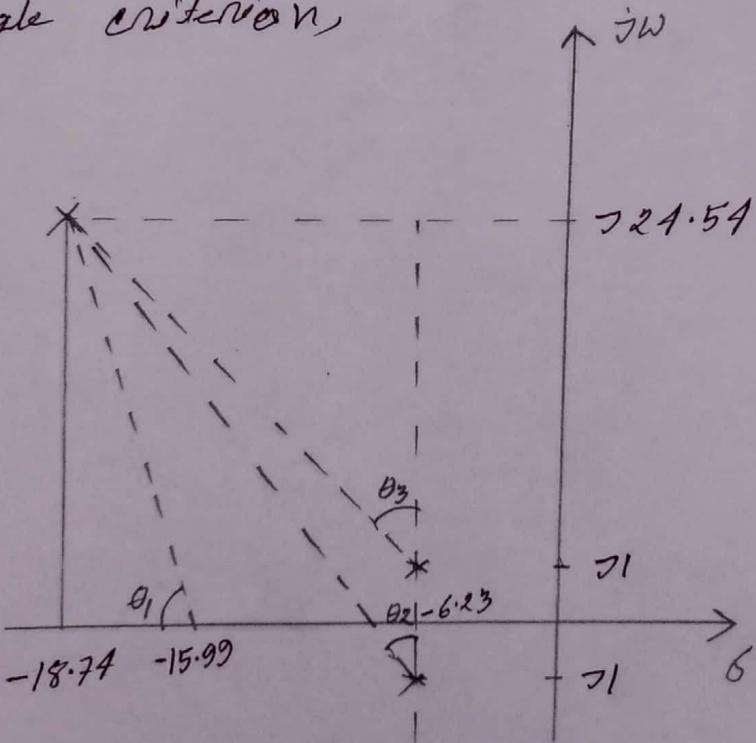
Assume, peak time is  
 $\frac{1}{4}$  times of the uncompensated  
system

$$= 24.54$$

The real part of compensated dominant pole is,

$$\begin{aligned}\sigma &= \frac{\omega_d}{\tan(127.37^\circ)} \\ &= \frac{24.54}{\tan(127.37^\circ)} \\ &= -18.74\end{aligned}$$

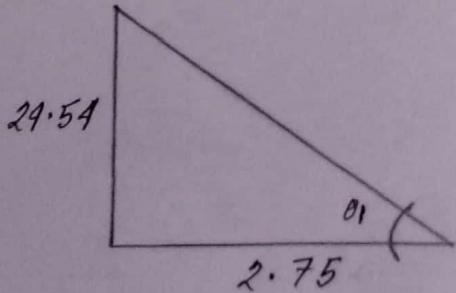
Now, Angle criterion,



For  $\theta_1$ ,

$$\tan \theta_1 = \frac{24.54}{2.75}$$

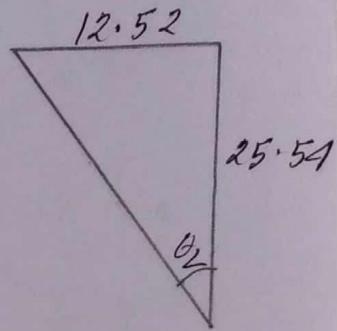
$$\therefore \theta_1 = 83.6^\circ$$



For  $\theta_2$ :

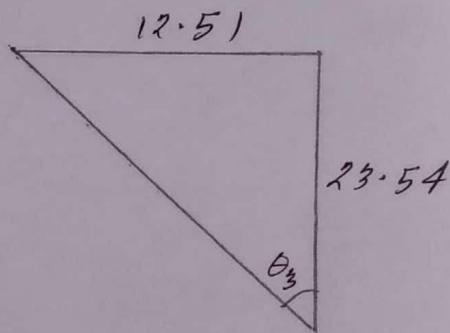
$$\tan \theta_2 = \frac{12.52}{25.54}$$

$$\theta = 26.11^\circ$$

For  $\theta_3$ :

$$\tan \theta_3 = \frac{12.52}{23.54}$$

$$\therefore \theta_3 = 28^\circ$$



Now, Applying the angle criterion,

$$\sum \angle Z - \sum \angle P = -180^\circ$$

$$\Rightarrow \varphi - (83.6 + 26.11 + 28) = -180^\circ$$

$$\therefore \varphi = 42.29^\circ$$

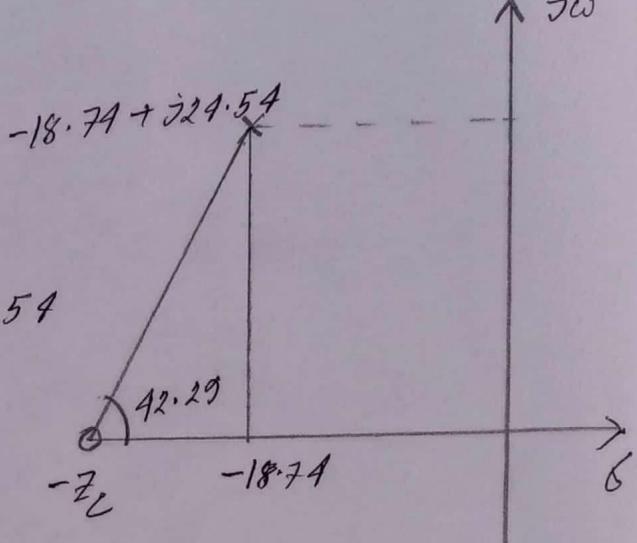
Assume that, the compensator zero is located at  $-Z_c$  as shown in figure,

From the figure,

$$\tan(42.29) = \frac{24.54}{Z_c - 18.74}$$

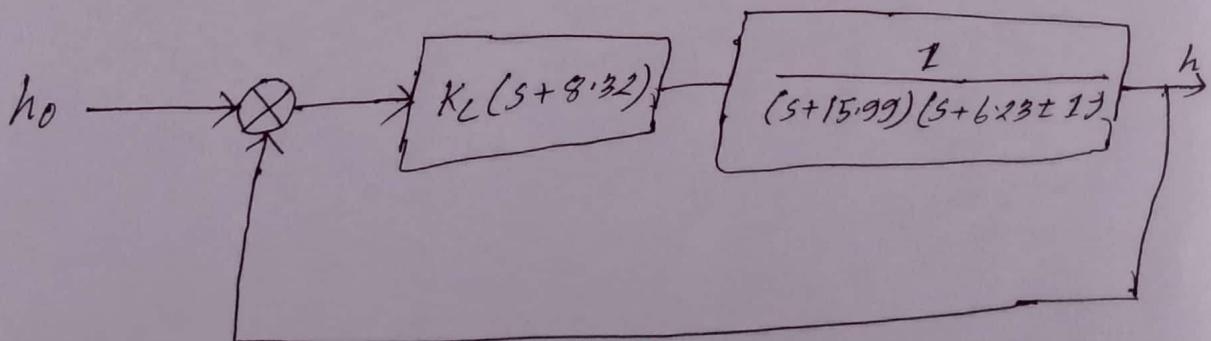
$$\Rightarrow 0.9 Z_c - (18.74 \times 0.9) = 24.54$$

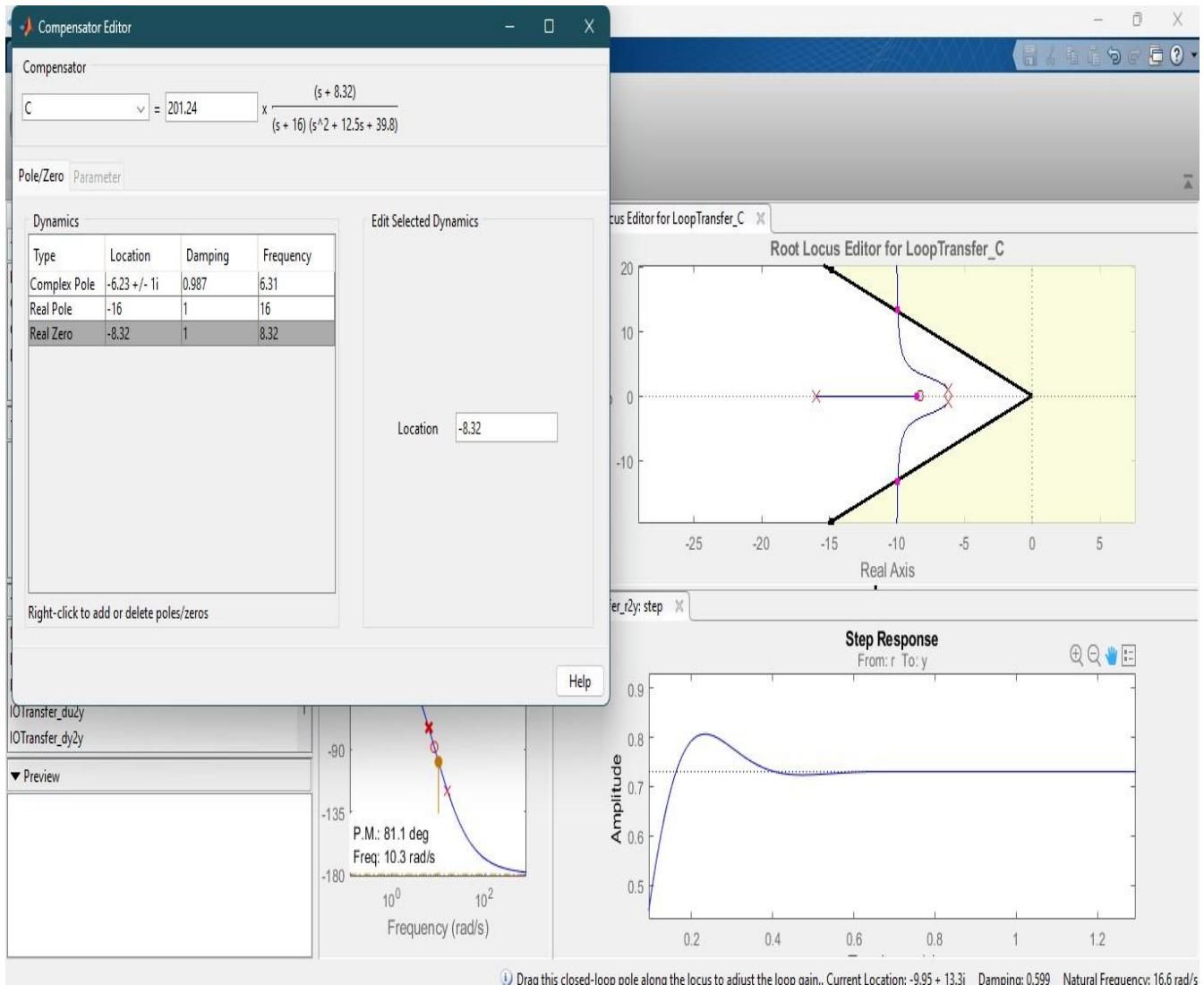
$$Z_c = 8.32$$



So, the PD controller is  $C_{PD}(s) = (s + 8.32)$

For PD compensator system diagram is -





Here,

Dominant Pole:  $-9.95+13.3j$

From the root locus, we have,

dominant poles,

$$s = -9.95 + 13.3j$$

Now,

$$|K(G(s)H(s))| = 1$$

$$\Rightarrow \left| \frac{K_C (s+8.74)}{(s+15.99)(s+6.23+j)} \right| = 1$$

$$\Rightarrow K_C = \left| \frac{(s+15.99)(s+6.23+j)}{(s+8.74)} \right|_{s = -9.95 + 13.3j}$$

$$= 201.25$$

Here,

$$\omega_n \gamma = 9.95$$

$$\omega_n = \frac{9.95}{0.607}$$

$$= 16.39$$

$$T_p = \frac{\pi}{\omega_n \sqrt{1-\gamma^2}}$$

$$= \frac{\pi}{16.39 \sqrt{1-(0.607)^2}} = 0.241$$

$$T_s = \frac{4}{\Im \omega_n} = \frac{4}{0.607 \times 16.39}$$

$$= 0.402$$

$$\begin{aligned}\therefore K_p &= \lim_{s \rightarrow 0} K_c G(s) H(s) \\ &= \lim_{s \rightarrow 0} \frac{K_c (s+8.74)}{(s+15.99)(s+6.23+j)} \\ &= \frac{201.25 \times 8.74}{15.99 \times 6.23} \\ &= 17.65\end{aligned}$$

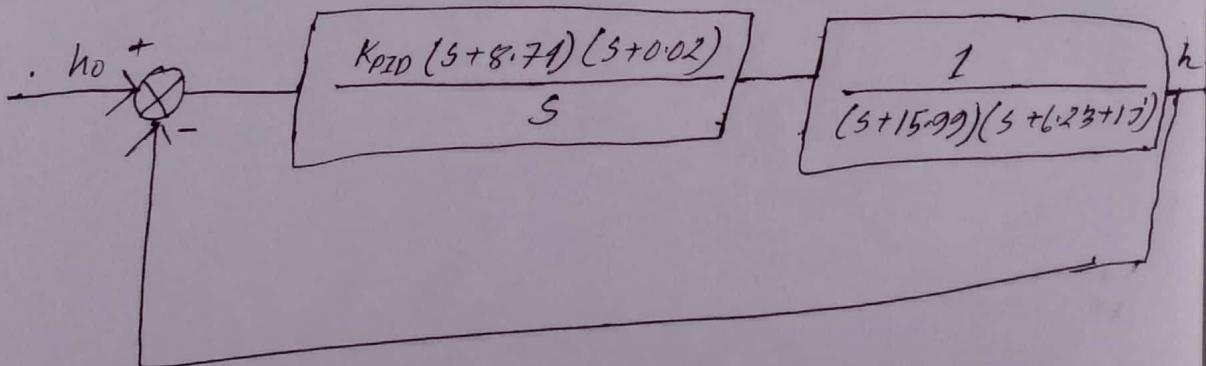
$$\begin{aligned}E_{ss} &= \frac{1}{1 + K_p} \\ &= \frac{1}{1 + 17.65} \\ &= 0.05\end{aligned}$$

Here, we can see that, after using PD compensation we have steady state error 0.05. whereas for un compensated system was 0.167

After design PD controller, now design a PID controller that reduce the steady state error to zero for a step input. So we placed zero close to the origin.

$$G_{PD}(s) = \frac{s+0.02}{s}$$

So the PID controller system is,



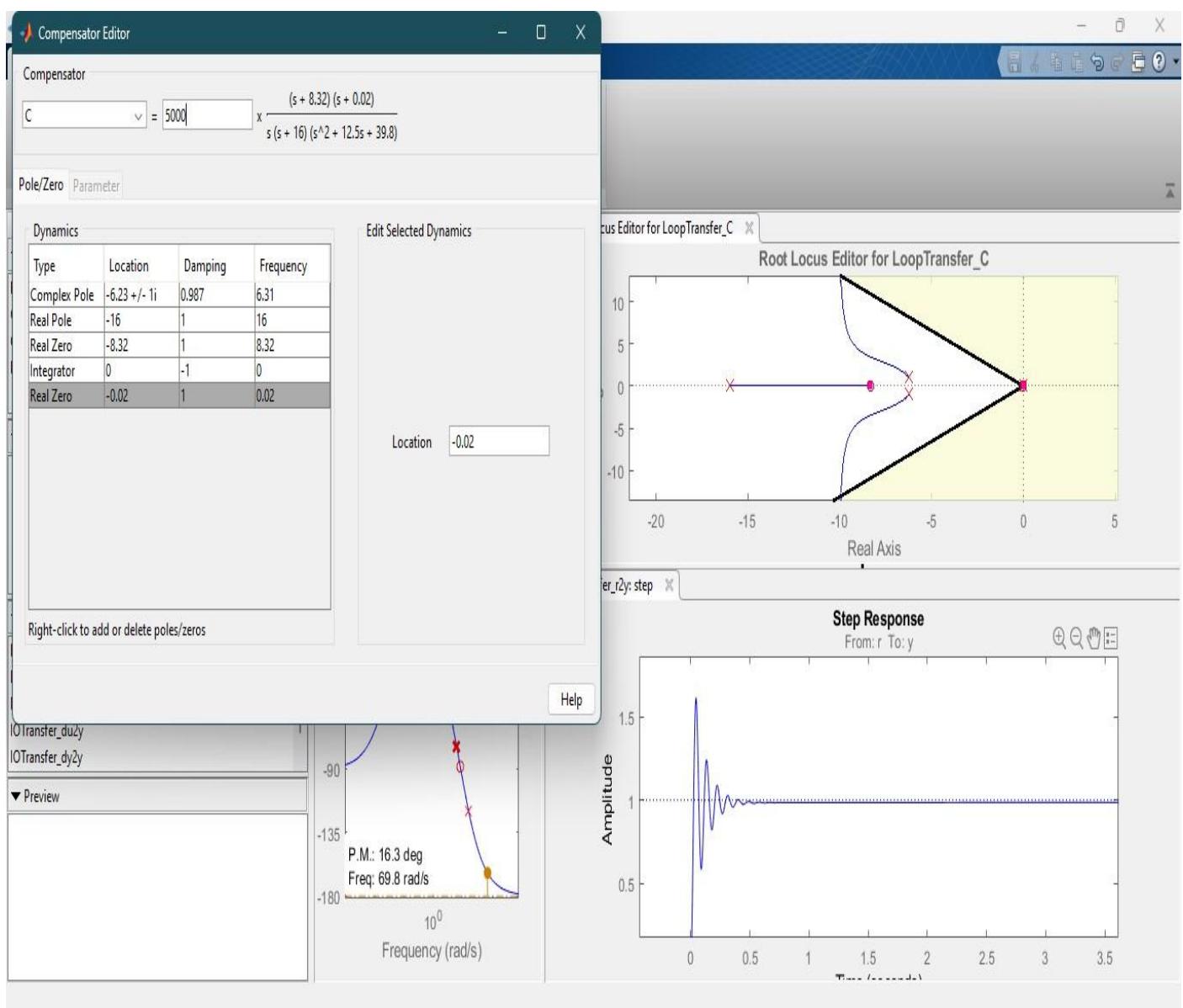
$$K_v = \lim_{s \rightarrow 0} K_{PID}$$

$$= \lim_{s \rightarrow 0} \frac{K_{PID} (s + 8.74)(s + 0.02)}{s(s + 15.99)(s + 6.23 + j)}$$

$$= \alpha$$

$$\therefore E_{ss} = \frac{1}{K_v} = 0$$

So, For using PID compensator, the system become error free.



(viii)

I have chosen 'water level control system' using PID controller. The compensator is justified based on its low environmental impact. The compensator is designed to minimize water waste by ensuring that the correct amount of water is supplied. The compensator is designed to minimize the energy consumption of the system, thereby reducing system's carbon footprint.

Overall a water level control system creates a positive environmental impact by reducing water waste and maintain optimal water quality.