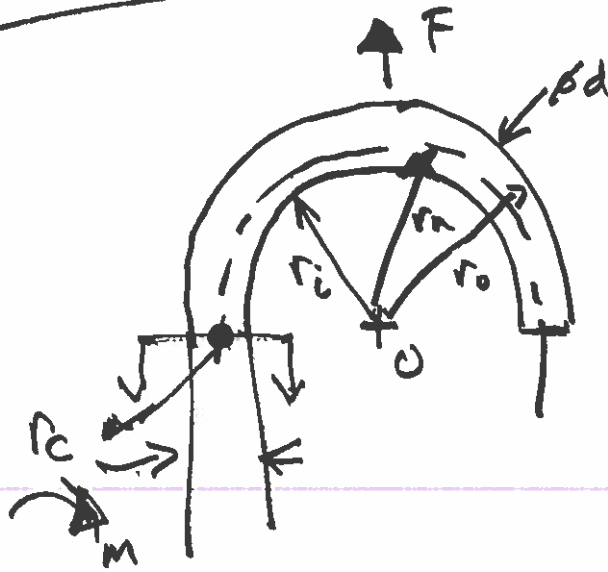
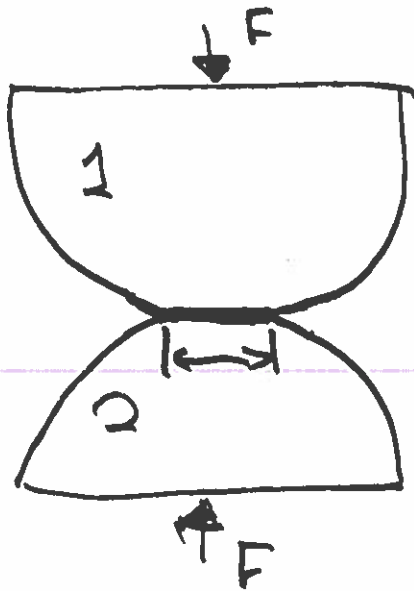


HW 3 PROB #4

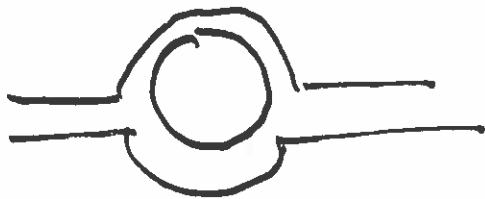
$$M = F \underbrace{r_c}$$

if $d \ll r_c$
 then r_c, r_n , or r_i or r_o
 are sufficient, ~~if~~
 Using r_o is most
 conservative

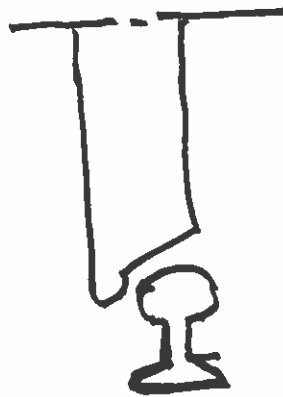
Contact Stress



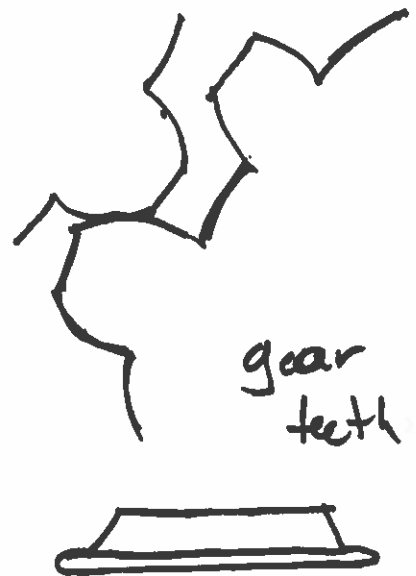
When two objects are pressed together a region of area contact develops.



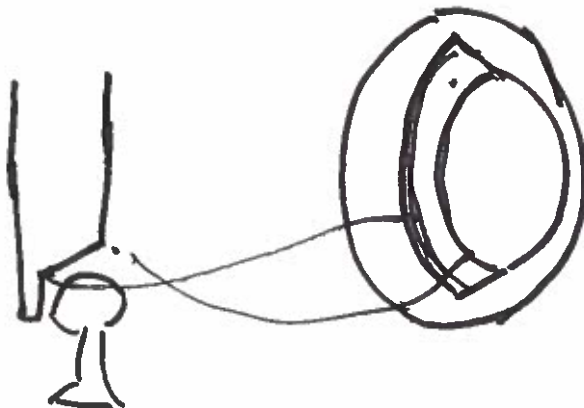
roller bearing



train wheel



gear teeth



Theory developed 1882 Hertz

Hertzian

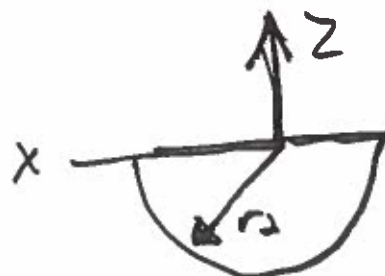
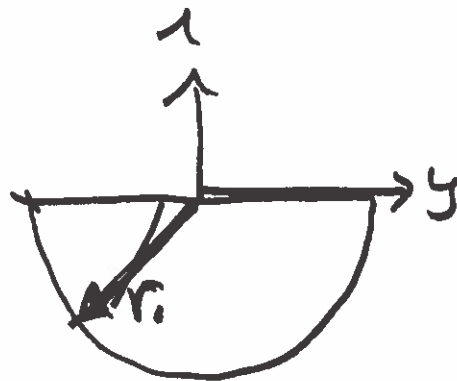
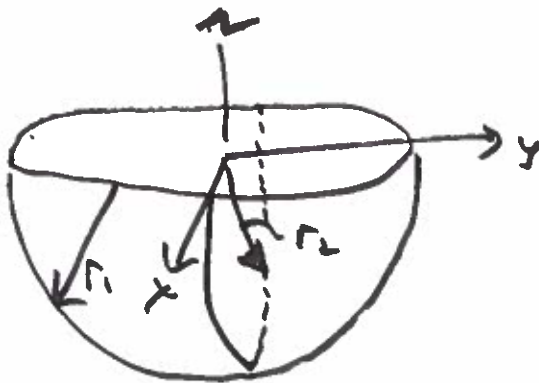
Hertzian contact stresses

Assumptions:

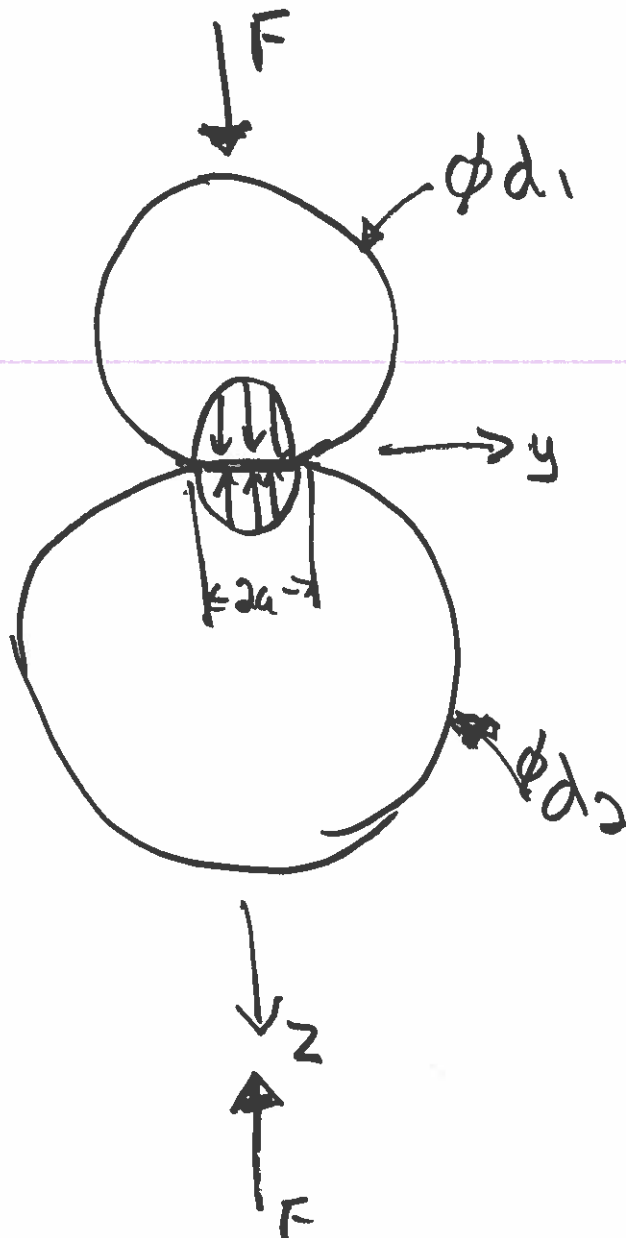
- loads are perpendicular to the surface
- no friction, smooth, continuous surfaces
- small strain theory

General Case

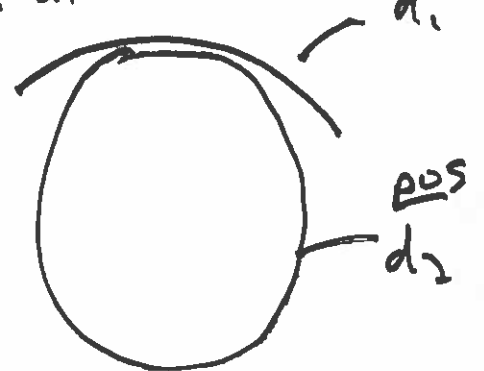
two radii of curvature



Special Case : Two Spheres



- circular area of contact
- hemispherical stress dis. at area

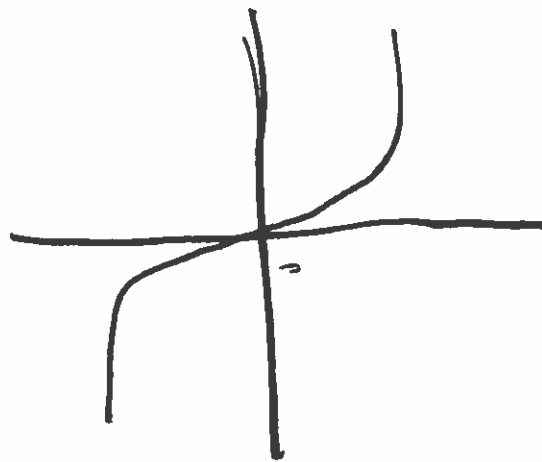
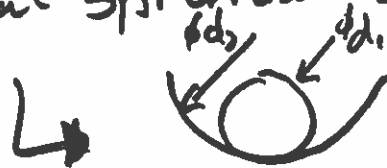


$$a = \sqrt[3]{\frac{3F}{8} \frac{(1-\nu_1^2)/E_1 + (1-\nu_2^2)/E_2}{1/d_1 + 1/d_2}} \quad (3-68)$$

subscripts match to potentially different diameter spheres

$d = \infty \Rightarrow$ flat surface

$d < 0 \Rightarrow$ internal spherical shape



$$P_{max} = \frac{3F}{2\pi a^2} \quad (3-69)$$

Along the z axis is max stress:

$$\sigma_1 = \sigma_2 = \sigma_x = \sigma_y =$$

$$-P_{\max} \left[\left(1 - \left| \frac{z}{a} \right| \tan^{-1} \frac{1}{\left| z/a \right|} \right) (1+\nu) - \frac{1}{2 \left(1 + \frac{z^2}{a^2} \right)} \right]$$

$$\sigma_3 = \sigma_z = - \frac{P_{\max}}{1 + \frac{z^2}{a^2}}$$



Fig 3-37

Sphere contact stress
as function of
depth into sphere

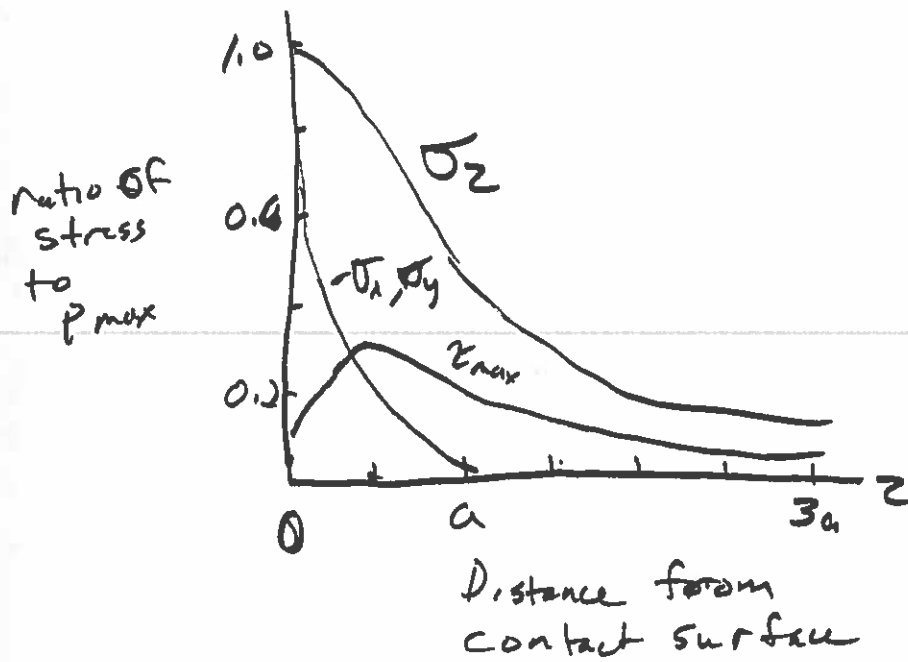
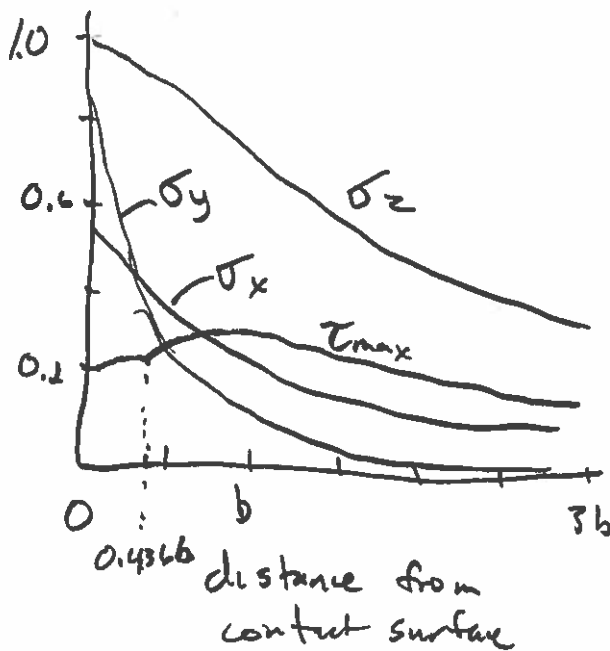


Fig 3-39

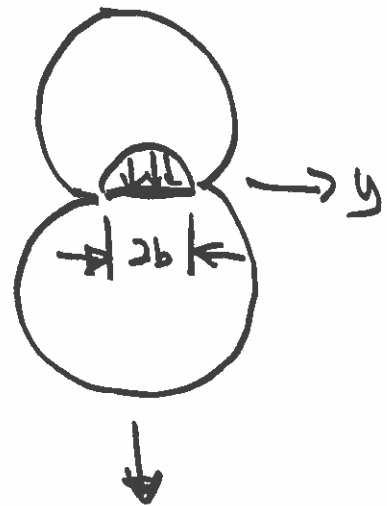
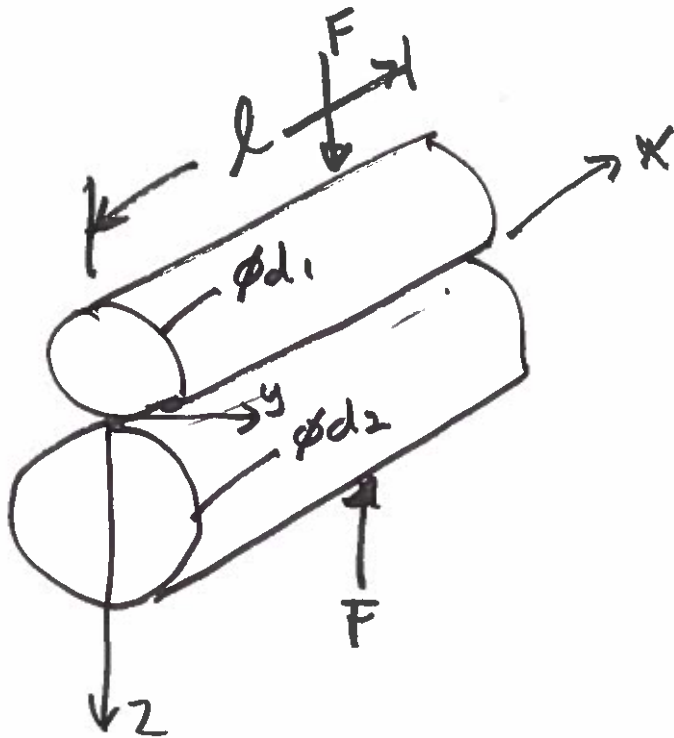
Cylindrical



$$\sigma_1 = \sigma_2 \Rightarrow \chi_{1/2} = 0$$

$$\chi_{\max} = \chi_{1/3} = \chi_{2/3} = \frac{\sigma_1 - \sigma_2}{2} = \frac{\sigma_2 - \sigma_3}{2}$$

Special Case Cylindrical Contact



$$b = \sqrt{\frac{\frac{2F}{\pi l} \frac{(1-\nu_1^2)/E_1 + (1-\nu_2^2)/E_2}{1/d_1 + 1/d_2}} \quad \underline{3-73}$$

$$P_{max} = \frac{2F}{\pi b l}$$

$$\sigma_x = -2\nu P_{max} \left(\sqrt{1 + \frac{z^2}{b^2}} - \left| \frac{z}{b} \right| \right) \quad 3-75$$

$$\sigma_y = -P_{max} \left(\frac{\sqrt{1 + 2\frac{z^2}{b^2}} - 2\left|\frac{z}{b}\right|}{\sqrt{1 + \frac{z^2}{b^2}}} \right) \quad 3-76$$

$$\sigma_3 = \sigma_2 = \frac{-P_{max}}{\sqrt{1 + \frac{z^2}{b^2}}}$$

$$0 \leq z \leq 0.436b$$

$$\sigma_1 = \sigma_x$$

$$\gamma_{max} = \frac{\sigma_1 - \sigma_3}{2}$$

$$z > 0.436b$$

$$\sigma_1 = \sigma_y$$

$$\gamma_{max} = \frac{(\sigma_y - \sigma_2)}{2}$$

⑨

pitting: spalling due to fatigue in roller bearings

brinelling: permanent indentation of hard surface

spalling: flaking of material from surface

See course website for links to
photos and more info