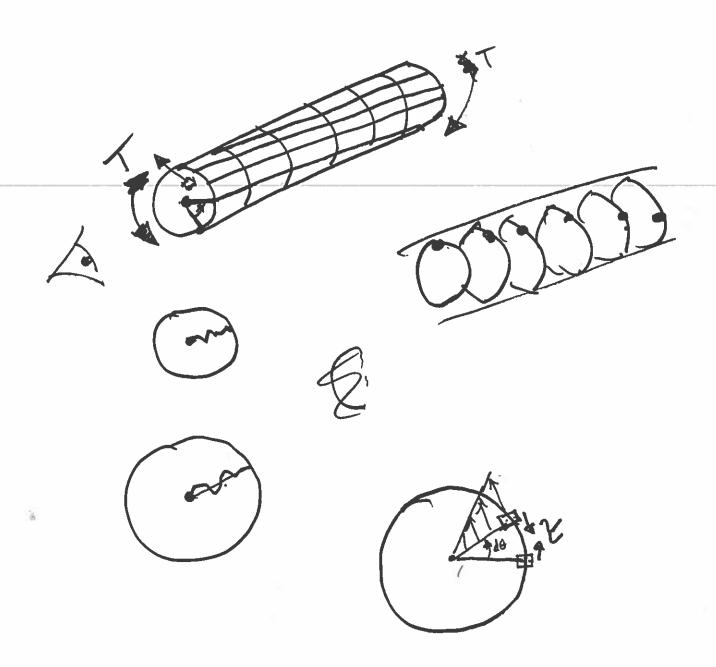
Complementary Shear

N. e. Kry

Transiterse Shear in Bending

Torsion and Shear



computing stress on an arbitrary 54re55 $\hat{n} = l\hat{i} + m\hat{j} + n\hat{k}$ 12=1

X

SF=0: AG, -A'X Gx - A'y Gy - A'2 G2

Project A onto xy, xz, yz planes A'x=RA A'y=MA A'z=nA

/ .. Jn = lox + moy + noz

The stress associated with the face A (defined by i), contains the normal and shear stress

マーグメ(プ×ダ)

Example Find a) To as function of 0 b) 100 and 121 $\hat{N} = \cos\theta \hat{i} + \sin\theta \hat{j}$ 5n=15x+mgy+n=2 JE = PI 可= lox= |coso 子仁| 可, n = P cos 20 12/+5=15,13 |T|= /15,12-0,2 = $\sqrt{\cos^2 \Theta \frac{p^2}{A^2} - \frac{p^2}{A^2} \cos^4 \Theta}$

 $= \frac{P}{A} \cos \theta \sqrt{1 - \cos^2 \theta} = \sin^2 \theta$ $= \frac{P}{A} \cos \theta \sin \theta$

Principal Stresses

From the theory of Elasticity

At any point with a general State of stress, an element can be oriented such that the shear components vanish.

principal stresses: 5,75,753

principal directions: 1,15,13

Eigenvalue and eignenvector Problem

symmetric

A X = 0