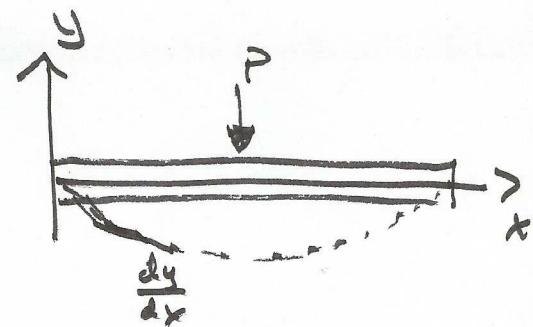


# Deflection of Beams

Curvature:  $\frac{1}{e} = \frac{M}{EI}$

$$\frac{1}{e} = \frac{d^2y / dx^2}{[1 + (\frac{dy}{dx})^2]^{3/2}}$$



$$\frac{1}{e} = \frac{d^2y}{dx^2}$$

$\frac{dy}{dx}$  is small

$$\frac{dy}{dx} \approx 0$$

$$\Theta = \frac{dx}{dy}$$

slope  
of  
the  
curve

$$\frac{M}{EI} = \frac{d^2y}{dx^2}$$

$$\frac{V}{EI} = \frac{d^3y}{dx^3}$$

$$\frac{q}{EI} = \frac{d^4y}{dx^4}$$

①

Given  $q(x)$

$$V = \int q(x) dx + C_v$$

$$M = \int V(x) dx + C_m$$

$C_v$  and  $C_m$   
obtained  
from static  
equilibrium condition

$$EI\Theta = \int M(x) dx + C_1$$

$$EIy = \int \Theta dx + C_2$$

$C_1 + C_2$   
obtained from  
boundary conditions

Multiple Ways to Solve the eqs.

- 
- Sectioning the beam and integrating the sections
  - super position \*
  - moment area method
  - singularity funcs \*
  - numerical integration

# Superposition

- Many simple beam cases have been pre-solved and the results can be algebraically combined.

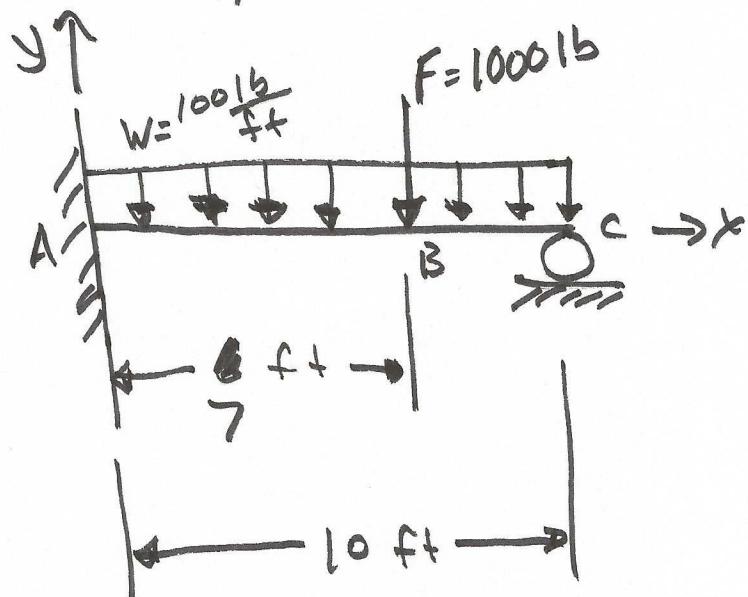
Table A-9

- Roark's Formulas for stress and strain

SP can be used if:

- each effect must be linearly related to the load that produced it, e.g.  $\delta = \frac{PL}{EA}$
- a load does not create a condition that affects the result of another load
- deformation from any specific load is not large enough to appreciably alter the geometric relations of the structural system

# Example: Superposition



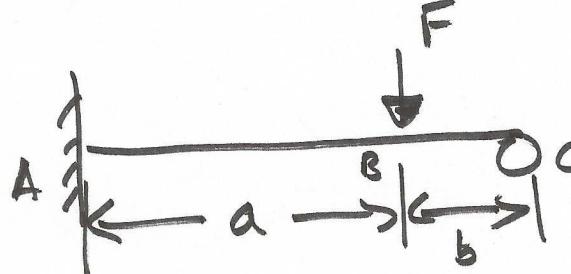
$$E = 30 \times 10^6 \text{ psi}$$

$$I = 5 \text{ in}^4$$

~~What~~ What is  
 $y_{\max}$ ?

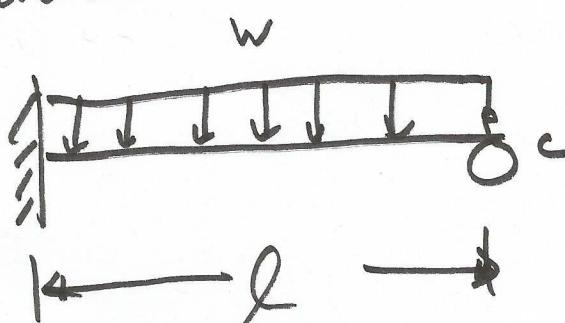
Table A-9 12

$$y_{AB} = \frac{F b x^2}{12 E I l^3} [3l(b^2 - l^2) + x(3l^2 - b^2)]$$



$$y_{BC} = y_{AB} - \frac{F(x-a)^3}{6 EI}$$

Table A-9 13



$$y = \frac{w x^2}{48 EI} (l-x)(2x-3l)$$

$$\theta = \frac{dy}{dx} = 0 \Rightarrow y_{\max} =$$

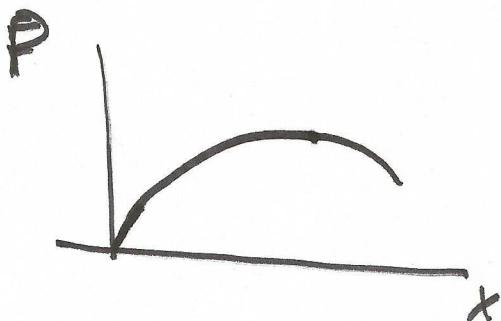
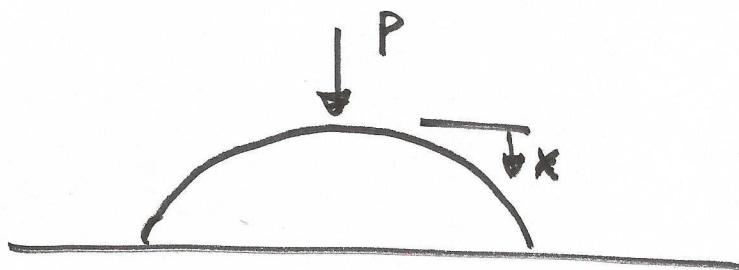
(4)

$y_{AB}$

$$y_{\substack{\text{tot} \\ AB}} = y_{AB} + y \quad \left. \right\}$$

$$y_{\substack{\text{tot} \\ BC}} = y_{BC} + y \quad \left. \right\}$$

super imposed  
the two results



$$y_{AB} = \frac{Fx^2}{12EIe^3} [3e(b^2 - e^2) + x(3e^2 - b^2)] + \frac{wx^3}{48EI} (e-x)(2x-3e)$$

$$y_{AB} = C_1 x^4 + C_2 x^3 + C_3 x^2 + C_4 x^1 + C_5$$

$$\frac{dy_{AB}}{dx} = \text{cubic polynomial} = 0$$

roots of cubic polynomial

(5)

root, fzero, feval  $\Rightarrow$  some functions  
in matlab

$$X = \begin{pmatrix} 156 & 73'' & -2 \\ - & \frac{1}{156} & - \\ z & & \end{pmatrix}$$

~~0x~~ ~~x~~ 84"

$$\boxed{y_{AB}(73') = -0.164''}$$

# Singularity Functions

$$\left\{ \begin{array}{l} \langle x-a \rangle^n = \begin{cases} 0 & x < a \\ (x-a)^n & x \geq a \end{cases} \\ n \geq 0 \end{array} \right.$$

$$\int \langle x-a \rangle^n dx = \frac{\langle x-a \rangle^{n+1}}{n+1} + C$$

$$\begin{array}{c} \langle x-a \rangle^{-1} \quad \langle x-a \rangle^{-2} \\ \uparrow \quad \curvearrowleft \\ \bullet \end{array} \quad \begin{array}{ll} 0 & x \neq a \\ \infty & x=a \end{array}$$

$$EI\theta = \int M(x)dx$$

$$EIy = \int \theta(x)dx$$

$$EI\theta = \frac{R}{a} \langle x \rangle^2 - \frac{W}{c} \langle x-a \rangle^3 + \frac{W}{c} \langle x-e \rangle^3 + \frac{E}{2} \langle x-e \rangle^2 + C_1$$

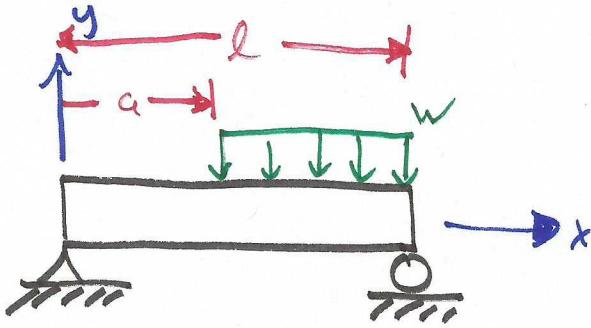
$$\langle x \rangle^2 = x^2$$



$$M(x) = \begin{cases} f_1(x) & 0 < x \leq a \\ f_2(x) & a < x \leq b \\ f_3(x) & b < c \\ f_4(x) & c < d \end{cases}$$

(8)

## Example



$$l = 10"$$

$$a = 4"$$

$$W = 100 \frac{lb}{in}$$

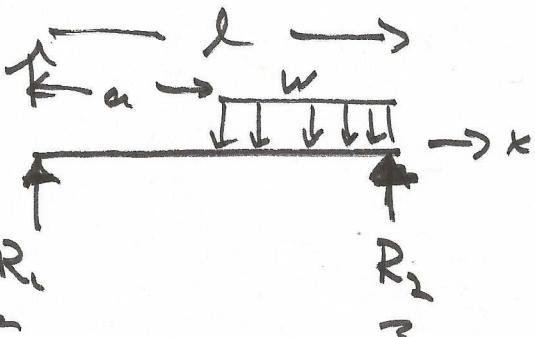
$$E = 30 \text{ Mpsi}$$

$$I = 0.163 \text{ in}^4$$

Find slope and deflection functions of  $x$

and the maximum deflection

$$q(x) = R_1 \langle x \rangle^0 + -w \langle x-a \rangle^0 \\ + w \langle x-l \rangle^0 + R_2 \langle x-l \rangle^0$$



$$V(x) = R_1 \langle x \rangle^0 - w \langle x-a \rangle^1 \\ + w \langle x-l \rangle^1 + R_2 \langle x-l \rangle^0$$

$$M(x) = R_1 \langle x \rangle^1 - \frac{w}{2} \langle x-a \rangle^2 + \frac{w}{2} \langle x-l \rangle^2 + R_2 \langle x-l \rangle^1$$

$$V=0, M=0 @ x=l^+$$

$$V(l^+) = 0 = R_1 - w(l-a) + w(l-l) + R_2$$

$$M(l^+) = 0 = R_1 l - \frac{w}{2}(l-a)^2 + \frac{w}{2}(l-l)^2 + R_2(l-l)$$

$$R_1 = \frac{w}{2e}(l-a)^2$$

~~R<sub>2</sub>~~  $R_1 = 180 \text{ lb}, R_2 = 420 \text{ lb}$  ①

$$\left\{ \begin{array}{l} EI\theta = \frac{R_1}{2}x^2 - \frac{W}{6}(x-a)^3 + C_1 \\ EIy = \frac{R_1}{6}x^3 - \frac{W}{24}(x-a)^4 + C_1x + C_2 \end{array} \right.$$

$$y(0) = 0 \quad y(l) = 0$$

boundary conditions

$$@ x=0 : 0 = C_2$$

$$@ x=l : 0 = \frac{R_1 l^3}{6} - \frac{W}{24}(l-a)^4 + C_1 l$$

$$C_1 = -2460 \text{ lb/in}^2$$

$$\text{Look } a < x < l$$

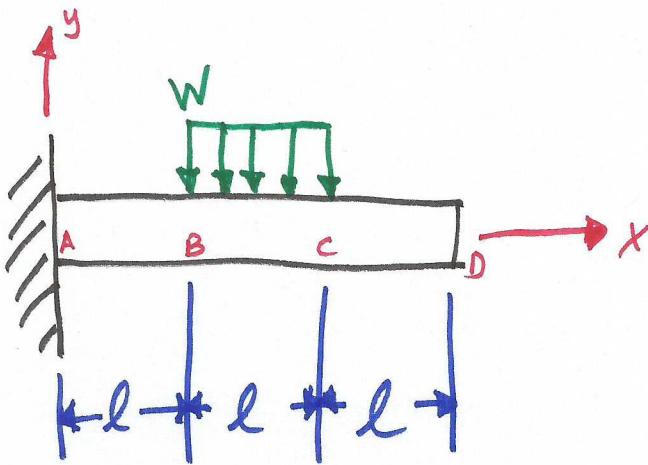
$$\cancel{EI\theta = 0 = \frac{1}{EI} (90x^2 - 16\frac{2}{3}(x-a)^3 - 2460)}$$

Cubic polynomial

$$x = (-1.191, \underbrace{5.264}_{\text{bad}}, 13.33)$$

$$y_{\max} = -1.76 \times 10^{-3} \text{ in}$$

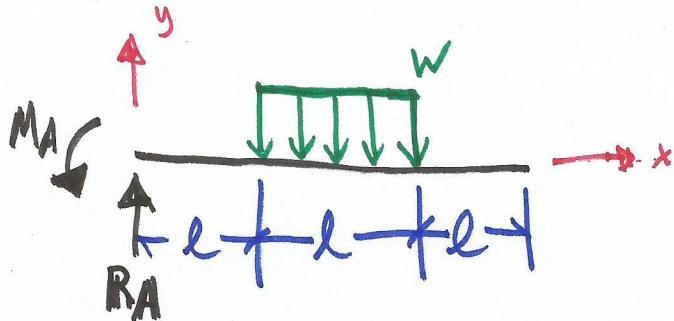
(10)



## Goal

Find a function for both  $\Theta$  and  $y$ .

FBD



$$q(x) = R_A \langle x \rangle^{-1} - M_A \langle x \rangle^{-2} - W \langle x-l \rangle^0 + W \langle x-2l \rangle^0$$

$$V(x) = R_A \langle x \rangle^0 - M_A \langle x \rangle^{-1} - W \langle x-l \rangle^1 + W \langle x-2l \rangle^1$$

$$M(x) = R_A \langle x \rangle^1 - M_A \langle x \rangle^0$$

$$-\frac{W}{2} \langle x-l \rangle^2 + W \langle x-2l \rangle^2$$

Find  $R_A$  and  $M_A$

$$V=0 \text{ and } M=0 \text{ at } x=3l^+$$

$$V(3l^+) = 0 = R_A - W(3l-l) + W(3l-2l)$$

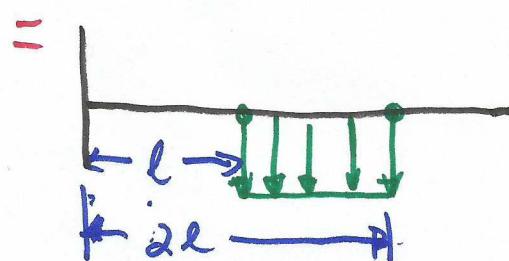
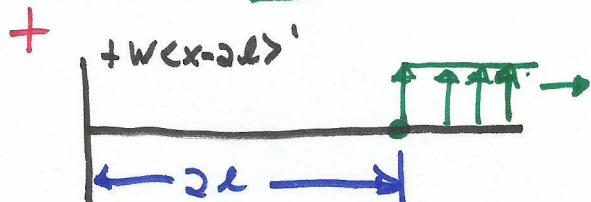
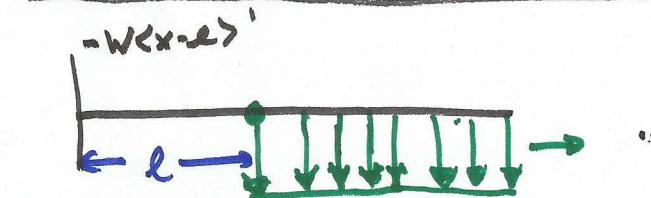
$$\boxed{R_A = WL}$$

$$M(3l^+) = 0 = 3R_A l - M_A - \frac{W}{2}(3l-l)^2 + \frac{W}{2}(3l-2l)^2$$

$$0 = 3WL - M_A - 2Wl^2 + \frac{W}{2}l^2$$

$$\boxed{M_A = \frac{3}{2}WL^2}$$

Why  $-W \langle x-l \rangle^1 + W \langle x-2l \rangle^1$ ?



11

$$M(x) = R_A x - M_A - \frac{W}{2} (x-l)^2 + W(x-2l)^2$$

$\downarrow$        $\downarrow$   
 $\langle x \rangle = x$     $\langle x^2 \rangle = l^2 \Rightarrow$  for whole length of beam

$$EI\theta = \int M(x) dx = \frac{R_A x^2}{2} - M_A x - \frac{W}{6} (x-l)^3 + \frac{W}{6} (x-2l)^5 + C_1$$

$$EIy = \int \theta(x) dx = \frac{R_A x^3}{6} - \frac{M_A x^2}{2} - \frac{W}{24} (x-l)^4 + \frac{W}{24} (x-2l)^4 + C_1 x + C_2$$

$$\theta(0) = 0 \quad y(0) = 0$$

$$0 = C_1 \quad 0 = C_1(0) + C_2$$

$$C_2 = 0$$

Apply boundary conditions  
to find  $C_1$  and  $C_2$ .

$$\theta = \frac{1}{EI} \left[ \frac{wl}{2} x^2 - \frac{3}{2} wl^2 x - \frac{W}{6} (x-l)^3 + \frac{W}{6} (x-2l)^5 \right]$$

$$y = \frac{1}{EI} \left[ \frac{wl}{6} x^3 - \frac{3}{4} wl^2 x^2 - \frac{W}{24} (x-l)^4 + \frac{W}{24} (x-2l)^4 \right]$$

The maximum deflection will be at the end of the cantilever so we can evaluate the singularity function @  $x=3l$  to find maximum deflection.

$$y(3l) = \frac{1}{EI} \left[ \frac{1}{2} wl^4 - \frac{27}{4} wl^4 - \frac{2}{3} wl^4 + \frac{W}{24} l^4 \right]$$

$$= \boxed{-\frac{23}{8} \frac{wl^4}{EI} = y_{max}}$$