# Adaptative balancing techniques applied to parallel mechanisms

Tarcisio Antonio Hess Coelho <sup>a</sup>, Renato Maia Matarazzo Orsino <sup>b</sup>, André Garnier Coutinho <sup>a</sup>

#### **SUMMARY**

#### **KEYWORDS:**

#### 1 Introduction and literature review

## 1.1 Dynamic Models

ℜ Modelo do mecanismo RR

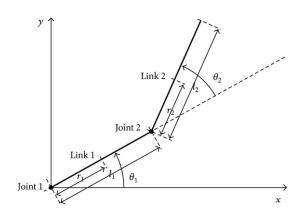


Figure 1: Robô <u>RR</u>

i) Primeiro definimos  $\nu_q$  coordenadas q. Estas podem ser subdivididas em  $\nu^{\#}$  coordenadas independentes  $\mathfrak{q}^{\#}$  e  $\nu_q^{\circ}$  coordenadas redudantes  $\mathfrak{q}^{\circ}$ .

$$\mathbf{q} = \begin{bmatrix} \mathbf{q}^\# \\ \mathbf{q}^\circ \end{bmatrix}$$

No caso do mecanismo RR, temos:

$$q^{\#} = \begin{bmatrix} \theta_1 & \theta_2 \end{bmatrix}^T \tag{1}$$

$$q^{\circ} = \begin{bmatrix} x_1 & y_1 & x_2 & y_2 \end{bmatrix}^T \tag{2}$$

Com  $\nu^{\#}=2$  e  $\nu_{q}^{\circ}=4$ . Neste caso, as componentes de  $\mathfrak{q}^{\circ}$  são as coordenadas dos centros de massa das barras, escritas no referencial inercial  $O_{xy}$ .

<sup>&</sup>lt;sup>a</sup> Department of Mechatronics and Mechanical Systems Engineering, Escola Politecnica, University of Sao Paulo, Brazil. E-mail: tarchess@usp.br

<sup>&</sup>lt;sup>b</sup> Department of Mechanical Engineering, Escola Politecnica, University of Sao Paulo, Brazil.

ii) Depois definimos os vetores de velocidades absolutas:

$$\mathbf{w} = \begin{bmatrix} \mathbf{w}_{\omega} \\ \mathbf{w}_{v} \end{bmatrix}$$

$$\mathbf{w}_{\omega} = \begin{bmatrix} \omega_{z_1} & \omega_{z_2} \end{bmatrix}^T \tag{3}$$

$$\mathbf{w}_v = \begin{bmatrix} v_{x_1} & v_{y_1} & v_{x_2} & v_{y_2} \end{bmatrix}^T \tag{4}$$

Sendo  $w_v$  as componentes das velocidades absolutas dos centros de massa das barras, escritas nas bases presas s barras, e  $w_\omega$  as componentes das velocidades angulares absolutas, escritas nas bases presas s barras.

iii) Definimos  $\nu_p$  coordenadas  $\mathbb{p}$ . Estas podem ser subdivididas em  $\nu^{\#}$  coordenadas independentes  $\mathbb{p}^{\#}$  e  $\nu_p^{\circ}$  coordenadas redudantes  $\mathbb{p}^{\circ}$ . As coordenadas  $\mathbb{p}^{\#}$  podem ser subdividas em  $\nu_{\omega}^{\#}$  velocidades angulares  $\omega^{\#}$  e  $\nu_v^{\#}$  velocidades lineares  $\nu^{\#}$ . As coordenadas  $\mathbb{p}^{\circ}$  podem ser subdividas em  $\nu_{\omega}^{\circ}$  velocidades angulares  $\omega^{\circ}$  e  $\nu_v^{\circ}$  velocidades lineares  $\nu^{\circ}$ .

$$\mathbb{p} = \begin{bmatrix} \mathbb{p}^{\#} \\ \mathbb{p}^{\circ} \end{bmatrix} \qquad \qquad \mathbb{p}^{\#} = \begin{bmatrix} \omega^{\#} \\ \mathbb{p}^{\#} \end{bmatrix} \qquad \qquad \mathbb{p}^{\circ} = \begin{bmatrix} \omega^{\circ} \\ \mathbb{p}^{\circ} \end{bmatrix}$$

Como é conveniente que as velocidades generalizadas p sejam velocidades absolutas, escolhemos as componentes de p como sendo as mesmas componentes de p, respeitando a ordenação indicada acima.

No caso do mecanismo RR, temos:

$$\omega^{\#} = \begin{bmatrix} \omega_{z_1} \\ \omega_{z_2} \end{bmatrix} \tag{5}$$

$$\nu^{\#} = \emptyset \tag{6}$$

$$\omega^{\circ} = \emptyset \tag{7}$$

$$\nu^{\circ} = \begin{bmatrix} v_{x_1} & v_{y_1} & v_{x_2} & v_{y_2} \end{bmatrix}^T \tag{8}$$

Com  $\nu_{\omega}^{\#} = 2$ ,  $\nu_{v}^{\#} = 0$ ,  $\nu_{\omega}^{\circ} = 0$ ,  $\nu_{v}^{\circ} = 4$  e  $\nu_{p}^{\circ} = \nu_{\omega}^{\circ} + \nu_{v}^{\circ} = 4$ .

iv) Realizamos a cinemtica de posição para os centros de massa das barras, de modo a relacionar as coordenadas q° com as coordenadas q#. Para isso, utilizamos matrizes de transformação homogênea.

$$[\boldsymbol{H}]_{\mathsf{B}_{0} \mid \mathsf{B}_{1}} = \begin{bmatrix} Rot(\theta_{1}, z_{0}) & [\overrightarrow{\mathsf{0}_{0}\mathsf{0}_{1}}]_{\mathsf{B}_{0}} \\ \mathbb{O}_{2x1} & 1 \end{bmatrix} = \begin{bmatrix} \mathsf{c}_{1} & -\mathsf{s}_{1} & \mathsf{0} \\ \mathsf{s}_{1} & \mathsf{c}_{1} & \mathsf{0} \\ \mathsf{0} & \mathsf{0} & 1 \end{bmatrix}; \begin{bmatrix} [\overrightarrow{\mathsf{0}_{1}\mathsf{G}_{1}}]_{\mathsf{B}_{1}} \\ 1 \end{bmatrix} = \begin{bmatrix} l_{1g} \\ \mathsf{0} \\ 1 \end{bmatrix}$$

$$[\boldsymbol{H}]_{\mathsf{B}_{1} \mid \mathsf{B}_{2}} = \begin{bmatrix} Rot(\theta_{2}, z_{1}) & [\overrightarrow{\mathsf{0}_{1}\mathsf{0}_{2}}]_{\mathsf{B}_{1}} \\ \mathbb{O}_{2x1} & 1 \end{bmatrix} = \begin{bmatrix} \mathsf{c}_{2} & -\mathsf{s}_{2} & l_{1} \\ \mathsf{s}_{2} & \mathsf{c}_{2} & \mathsf{0} \\ \mathsf{0} & \mathsf{0} & 1 \end{bmatrix}; \begin{bmatrix} [\overrightarrow{\mathsf{0}_{2}\mathsf{G}_{2}}]_{\mathsf{B}_{2}} \\ 1 \end{bmatrix} = \begin{bmatrix} l_{2g} \\ \mathsf{0} \\ 1 \end{bmatrix}$$

$$[\boldsymbol{H}]_{\mathsf{B}_{0} \mid \mathsf{B}_{2}} = [\boldsymbol{H}]_{\mathsf{B}_{0} \mid \mathsf{B}_{1}} [\boldsymbol{H}]_{\mathsf{B}_{1} \mid \mathsf{B}_{2}} = \begin{bmatrix} \mathsf{c}_{1+2} & -\mathsf{s}_{1+2} & l_{1}\mathsf{c}_{1} \\ \mathsf{s}_{1+2} & \mathsf{c}_{1+2} & l_{1}\mathsf{s}_{1} \\ \mathsf{0} & \mathsf{0} & 1 \end{bmatrix}$$

$$\therefore \begin{bmatrix} x_1 \\ y_1 \\ 1 \end{bmatrix} = [\boldsymbol{H}]_{\mathbf{B}_0 \mid \mathbf{B}_1} \begin{bmatrix} [\overrightarrow{\mathbf{0}_1 \mathbf{G}_1}]_{\mathbf{B}_1} \\ 1 \end{bmatrix} = \begin{bmatrix} l_{1g} \mathbf{c}_1 \\ l_{1g} \mathbf{s}_1 \\ 1 \end{bmatrix}$$
(9)

$$\begin{bmatrix} x_2 \\ y_2 \\ 1 \end{bmatrix} = [\boldsymbol{H}]_{\mathsf{B}_0 \mid \mathsf{B}_2} \begin{bmatrix} [\overrightarrow{\mathsf{0}_2\mathsf{G}_2}]_{\mathsf{B}_2} \\ 1 \end{bmatrix} = \begin{bmatrix} l_1\mathsf{c}_1 + l_{2g}\mathsf{c}_{1+2} \\ l_1\mathsf{s}_1 + l_{2g}\mathsf{s}_{1+2} \\ 1 \end{bmatrix}$$
(10)

Repare que a partir das matrizes de transformação homognea encontradas, encontramos tambm as seguintes matrizes de mudança de base:

$$\mathbb{R}_1 = \begin{bmatrix} \mathbf{1} \end{bmatrix}_{\mathsf{B}_0 \mid \mathsf{B}_1} = \begin{bmatrix} \mathsf{c}_1 & -\mathsf{s}_1 \\ \mathsf{s}_1 & \mathsf{c}_1 \end{bmatrix} \tag{11}$$

$$\mathbb{R}_2 = \begin{bmatrix} \mathbf{1} \end{bmatrix}_{\mathsf{B}_0 \mid \mathsf{B}_2} = \begin{bmatrix} \mathsf{c}_{1+2} & -\mathsf{s}_{1+2} \\ \mathsf{s}_{1+2} & \mathsf{c}_{1+2} \end{bmatrix} \tag{12}$$

Com a cinemtica de posição, conseguimos obter  $\nu_q^\circ=4$  equações vinculares de posição. Sendo assim, o vetor dos vínculos de posição dado por:

$$\phi(\mathbf{q}) = \begin{bmatrix} x_1 - l_{1g} \mathbf{c}_1 \\ y_1 - l_{1g} \mathbf{s}_1 \\ x_2 - l_1 \mathbf{c}_1 - l_{2g} \mathbf{c}_{1+2} \\ y_2 - l_1 \mathbf{s}_1 - l_{2g} \mathbf{s}_{1+2} \end{bmatrix}$$
(13)

v) Utilizamos as matrizes de rotação para calcular as velocidades angulares em função de q# e q#:

$$[\boldsymbol{\omega}_1]_{\mathsf{B}_1|\mathsf{B}_1}^{\mathsf{S}} = \mathbb{R}_1^T \dot{\mathbb{R}}_1 = \begin{bmatrix} 0 & -\dot{\theta}_1 \\ \dot{\theta}_1 & 0 \end{bmatrix} \Rightarrow [\boldsymbol{\omega}_1]_{\mathsf{B}_1} = \dot{\theta}_1 \hat{k}$$
(14)

$$[\boldsymbol{\omega}_2]_{\mathsf{B}_2|\mathsf{B}_2}^{\mathsf{S}} = \mathbb{R}_2^T \dot{\mathbb{R}}_2 = \begin{bmatrix} 0 & -\dot{\theta}_1 - \dot{\theta}_2 \\ \dot{\theta}_1 + \dot{\theta}_2 & 0 \end{bmatrix} \Rightarrow [\boldsymbol{\omega}_2]_{\mathsf{B}_2} = (\dot{\theta}_1 + \dot{\theta}_2)\hat{k}$$
(15)

vi) Derivamos as equações de posição ((29) e (30)) para encontrar as velocidades dos centros de massa:

$$[\mathbf{v}_1]_{\mathbf{B}_0} = \begin{bmatrix} \dot{x}_1 \\ \dot{y}_1 \end{bmatrix} = \begin{bmatrix} -l_{1g} \mathbf{s}_1 \dot{\theta}_1 \\ l_{1g} \mathbf{c}_1 \dot{\theta}_1 \end{bmatrix}$$
(16)

$$[\mathbf{v}_2]_{\mathbf{B}_0} = \begin{bmatrix} \dot{x}_2 \\ \dot{y}_2 \end{bmatrix} = \begin{bmatrix} -l_1 \mathbf{s}_1 \dot{\theta}_1 - l_{2g} \mathbf{s}_{1+2} (\dot{\theta}_1 + \dot{\theta}_2) \\ l_1 \mathbf{c}_1 \dot{\theta}_1 + l_{2g} \mathbf{c}_{1+2} (\dot{\theta}_1 + \dot{\theta}_2) \end{bmatrix}$$
(17)

vii) Passamos as velocidades dos centros de massa para as bases presas nas barras:

$$\begin{aligned} [\mathbf{v}_1]_{\mathsf{B}_1} &= [\mathbf{1}]_{\mathsf{B}_1 \mid \mathsf{B}_0} [\mathbf{v}_1]_{\mathsf{B}_0} = \mathbb{R}_1^T [\mathbf{v}_1]_{\mathsf{B}_0} \\ [\mathbf{v}_2]_{\mathsf{B}_2} &= [\mathbf{1}]_{\mathsf{B}_2 \mid \mathsf{B}_0} [\mathbf{v}_2]_{\mathsf{B}_0} = \mathbb{R}_2^T [\mathbf{v}_2]_{\mathsf{B}_0} \end{aligned}$$

Definindo:

$$\mathbb{R}_{\Diamond} = \begin{bmatrix} \mathbb{R}_1 & \mathbb{O}_{2x2} \\ \mathbb{O}_{2x2} & \mathbb{R}_2 \end{bmatrix} \tag{18}$$

Temos:

$$\mathbf{w}_v = \mathbb{R}_{\Diamond}^T \dot{\mathbf{q}}^{\circ} \tag{19}$$

$$\therefore \begin{bmatrix} v_{x_1} \\ v_{y_1} \\ v_{x_2} \\ v_{y_2} \end{bmatrix} = \begin{bmatrix} c_1 & -s_1 & 0 & 0 \\ s_1 & c_1 & 0 & 0 \\ 0 & 0 & c_{1+2} & -s_{1+2} \\ 0 & 0 & s_{1+2} & c_{1+2} \end{bmatrix}^T \begin{bmatrix} \dot{x}_1 \\ \dot{y}_1 \\ \dot{x}_2 \\ \dot{y}_2 \end{bmatrix} = \begin{bmatrix} 0 \\ l_{1g}\dot{\theta}_1 \\ l_{1s_2}\dot{\theta}_1 \\ (l_1c_2 + l_{2g})\dot{\theta}_1 + l_{2g}\dot{\theta}_2 \end{bmatrix} \tag{20}$$

viii) Montamos os vetores p# e p° em função de q# e q#:

$$\mathbb{p}^{\#} = \begin{bmatrix} \omega_{z_1} \\ \omega_{z_2} \end{bmatrix} = \mathbb{p}_{\star}^{\#}(\mathbb{q}^{\#}, \dot{\mathbb{q}}^{\#}) = \begin{bmatrix} \dot{\theta}_1 \\ \dot{\theta}_1 + \dot{\theta}_2 \end{bmatrix}$$
(21)

$$p^{\circ} = \begin{bmatrix} v_{x_1} \\ v_{y_1} \\ v_{x_2} \\ v_{y_2} \end{bmatrix} = p^{\circ}_{\star}(q^{\#}, \dot{q}^{\#}) = \begin{bmatrix} 0 \\ l_{1g}\dot{\theta}_{1} \\ l_{1s_2}\dot{\theta}_{1} \\ (l_{1}c_2 + l_{2g})\dot{\theta}_{1} + l_{2g}\dot{\theta}_{2} \end{bmatrix}$$
(22)

ix) Utilizando o fato de que  $p_{\star}^{\#}(q^{\#},\dot{q}^{\#})$  e  $p_{\star}^{\circ}(q^{\#},\dot{q}^{\#})$  são lineares em  $\dot{q}^{\#}$ , encontramos as transformações lineares  $\Psi(q)$  e  $\Upsilon(q)$  e o vetor dos vínculos de velocidades  $\Lambda(q,p)$ :

$$p^{\#} = p_{\star}^{\#}(q^{\#}, \dot{q}^{\#}) = \frac{\partial p_{\star}^{\#}}{\partial \dot{q}^{\#}} \dot{q}^{\#} = \Psi \dot{q}^{\#}$$

$$(23)$$

$$p^{\circ} = p_{\star}^{\circ}(q^{\#}, \dot{q}^{\#}) = \frac{\partial p_{\star}^{\circ}}{\partial \dot{q}^{\#}} \dot{q}^{\#} = \Upsilon \dot{q}^{\#}$$

$$(24)$$

No caso do mecanismo RR, temos:

$$\Psi = \frac{\partial \mathbb{p}_{\star}^{\#}}{\partial \dot{q}^{\#}} = \begin{bmatrix} 1 & 0 \\ 1 & 1 \end{bmatrix} \tag{25}$$

$$\mathcal{F} = \frac{\partial p_{\star}^{\circ}}{\partial \dot{q}^{\#}} = \begin{bmatrix} 0 & 0 \\ l_{1g} & 0 \\ l_{1}s_{2} & 0 \\ l_{1}c_{2} + l_{2g} & l_{2g} \end{bmatrix}$$
(26)

Como  $p^{\#}$  e  $\dot{q}^{\#}$  são independentes e tem o mesmo tamanho:

$$\dot{q}^{\#} = \Psi^{-1} p^{\#}$$

$$\Rightarrow p^{\circ} = \Upsilon \Psi^{-1} p^{\#}$$

$$\therefore \Lambda(\mathbf{q}, \mathbf{p}) = \Upsilon \Psi^{-1} \mathbf{p}^{\#} - \mathbf{p}^{\circ} = 0 \tag{27}$$

x) A partir de  $\mathbb{A}(q,p)$ , encontramos o jacobiano  $\mathbb{A}$  dos vínculos de velocidade e a matriz  $\mathbb{C}$  dos vínculos cinemáticos:

$$\mathbb{A}(\mathbf{q},\mathbf{p}) = \begin{bmatrix} \mathbb{Y}\mathbb{\Psi}^{-1} & -\mathbb{1} \end{bmatrix} \mathbf{p}$$

$$\mathbb{A} = \frac{\partial \mathbb{A}}{\partial \mathbb{P}} = \begin{bmatrix} \mathbb{Y} \Psi^{-1} & -\mathbb{1} \end{bmatrix} = \begin{bmatrix} 0 & 0 & -1 & 0 & 0 & 0 \\ l_{1g} & 0 & 0 & -1 & 0 & 0 \\ l_{1}s_{2} & 0 & 0 & 0 & -1 & 0 \\ l_{1}c_{2} & l_{2g} & 0 & 0 & 0 & -1 \end{bmatrix}$$
(28)

$$\wedge(\mathbf{q},\mathbf{p})=0\Rightarrow \mathbf{p}^\circ=\Upsilon\Psi^{-1}\mathbf{p}^\#\Rightarrow \mathbf{p}=\begin{bmatrix}\mathbb{1}\\ \Upsilon\Psi^{-1}\end{bmatrix}\mathbf{p}^\#$$

$$\therefore \mathbb{C} = \begin{bmatrix} \mathbb{1} \\ \mathbb{Y}\Psi^{-1} \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \\ 0 & 0 \\ l_{1g} & 0 \\ l_{1}s_{2} & 0 \\ l_{1}c_{2} & l_{2g} \end{bmatrix} \tag{29}$$

xi) Como (19) e (23) são transformações inversíveis, encontramos a transformação linear  $\mathbb{F}(\mathfrak{q})$ :

$$\dot{\mathbf{q}} = \begin{bmatrix} \dot{\mathbf{q}}^{\#} \\ \dot{\mathbf{q}}^{\circ} \end{bmatrix} = \dot{\mathbf{q}}_{\star}(\mathbf{q}, \mathbf{p}) = \begin{bmatrix} \Psi^{-1}(\mathbf{q})\mathbf{p}^{\#} \\ \mathbb{R}_{\Diamond}(\mathbf{q})\mathbf{w}_{v}(\mathbf{p}) \end{bmatrix} = \begin{bmatrix} \omega_{z_{1}} \\ \omega_{z_{2}} - \omega_{z_{1}} \\ v_{x_{1}} \, \mathbf{c}_{1} - v_{y_{1}} \, \mathbf{s}_{1} \\ v_{x_{1}} \, \mathbf{s}_{1} + v_{y_{1}} \, \mathbf{c}_{1} \\ v_{x_{2}} \, \mathbf{c}_{1+2} - v_{y_{2}} \, \mathbf{s}_{1+2} \\ v_{x_{2}} \, \mathbf{s}_{1+2} + v_{y_{2}} \, \mathbf{c}_{1+2} \end{bmatrix}$$

$$(30)$$

$$\Gamma(\mathbf{q}) = \frac{\partial \dot{\mathbf{q}}_{\star}}{\partial \mathbf{p}} = \begin{bmatrix}
1 & 0 & 0 & 0 & 0 & 0 \\
-1 & 1 & 0 & 0 & 0 & 0 \\
0 & 0 & c_{1} & -s_{1} & 0 & 0 \\
0 & 0 & s_{1} & c_{1} & 0 & 0 \\
0 & 0 & 0 & 0 & c_{1+2} & -s_{1+2} \\
0 & 0 & 0 & 0 & s_{1+2} & c_{1+2}
\end{bmatrix}$$
(31)

xii) Aplicamos o mtodos de Gibbs-Appel extendido:

O mtodo de Gibbs-Appell apresenta certa simularidade com o mtodo de Lagrange, pois utiliza derivadas de uma função energia para encontrar a equações de movimento do sistema. Porém, a função energia utilizada no a energia cintica, mas sim a energia de acelerações. A energia de acelerações para um corpo rígido é dada pela seguinte expressão:

$$S = \frac{1}{2}m(\boldsymbol{a}_G\cdot\boldsymbol{a}_G) + \frac{1}{2}(\dot{\boldsymbol{\omega}}\cdot\boldsymbol{I}\dot{\boldsymbol{\omega}} + 2\dot{\boldsymbol{\omega}}(\boldsymbol{\omega}\wedge\boldsymbol{I}\boldsymbol{\omega}))$$

Sendo m a massa do corpo rígido, I seu tensor de inércia,  $\mathbf{a}_G$  o vetor aceleração absoluta de seu centro de massa e  $\boldsymbol{\omega}$  o vetor velocidade angular absoluta.

O modelo dinmico utilizando o mtodo de Gibbs-Appel extendido, dado pela seguinte expresso:

$$\mathbb{C}(q)^{T}(\mathbb{M}(\mathfrak{q})\dot{\mathbb{p}} + \mathbb{V}(\mathfrak{q}, \mathbb{p}) + \mathfrak{g}(\mathfrak{q})) = (\Psi^{T})^{-1}\mathbb{I}_{\dot{\mathfrak{q}}^{\#}}$$
(32)

Com:

$$\mathbb{M}(\mathbf{q}) = \frac{\partial^2 S}{\partial \dot{\mathbf{p}}^2} \tag{33}$$

$$v(\mathbf{q}, \mathbf{p}) = \frac{\partial \mathbf{S}}{\partial \dot{\mathbf{p}}} - \frac{\partial^2 \mathbf{S}}{\partial \dot{\mathbf{p}}^2} \dot{\mathbf{p}}$$
(34)

$$g(q) = \mathbb{F}^T \frac{\partial E_p}{\partial q} \tag{35}$$

Sendo  $E_p$  a energia potencial do sistema e  $\mathbb{f}_{\dot{q}^{\#}}$  os esforços nas direções de  $\dot{q}^{\#}$ .

Como já calculamos os vetores de velocidades absolutas dos centros de massa e de velocidades angulares absolutas, escritos em bases presas s barras, as acelerações absolutas dos centros de massa so dadas por:

$$[\pmb{a}_i]_{\mathtt{B}_i} = rac{\mathsf{d}}{\mathsf{d}t}[\pmb{v}_i]_{\mathtt{B}_i} + [\pmb{\omega}_i]_{\mathtt{B}_i \mid \mathtt{B}_i}^{\mathsf{S}}[\pmb{v}_i]_{\mathtt{B}_i}$$

Como o mecanismo plano:

$$[\mathbf{a}_i]_{\mathbf{B}_i} = \begin{bmatrix} \dot{v}_{x_i} \\ \dot{v}_{y_i} \end{bmatrix} + \begin{bmatrix} \mathbf{0} & -\omega_{z_i} \\ \omega_{z_i} & \mathbf{0} \end{bmatrix} \begin{bmatrix} v_{x_i} \\ v_{y_i} \end{bmatrix} = \begin{bmatrix} \dot{v}_{x_i} - \omega_{z_i} v_{y_i} \\ \dot{v}_{y_i} + \omega_{z_i} v_{x_i} \end{bmatrix}$$

No caso do mecanismo RR, temos:

$$S = \frac{1}{2} \Big( m_1 ((\dot{v}_{x_1} - \omega_{z_1} v_{y_1})^2 + (\dot{v}_{y_1} + \omega_{z_1} v_{x_1})^2) + m_2 ((\dot{v}_{x_2} - \omega_{z_2} v_{y_2})^2 + (\dot{v}_{y_2} + \omega_{z_2} v_{x_2})^2) + J_{z_1} \dot{\omega}_{z_1}^2 + J_{z_2} \dot{\omega}_{z_2}^2 \Big)$$
(36)

$$E_p = m_1 g y_1 + m_2 g y_2 (37)$$

Calculando as derivadas:

$$\mathbb{M}(\mathbf{q}) = \begin{bmatrix} J_{z_1} & 0 & 0 & 0 & 0 & 0 \\ 0 & J_{z_2} & 0 & 0 & 0 & 0 \\ 0 & 0 & m_1 & 0 & 0 & 0 \\ 0 & 0 & 0 & m_1 & 0 & 0 \\ 0 & 0 & 0 & 0 & m_2 & 0 \\ 0 & 0 & 0 & 0 & 0 & m_2 \end{bmatrix}$$

$$(38)$$

$$v(\mathbf{q}, \mathbf{p}) = \begin{bmatrix} 0 \\ 0 \\ -m_1 \omega_{z_1} v_{y_1} \\ m_1 \omega_{z_1} v_{x_1} \\ -m_2 \omega_{z_2} v_{y_2} \\ m_2 \omega_{z_2} v_{x_2} \end{bmatrix}$$
(39)

$$g = g \begin{bmatrix} 0 \\ 0 \\ m_1 s_1 \\ m_1 c_1 \\ m_2 s_{1+2} \\ m_2 c_{1+2} \end{bmatrix}$$
(40)

Sendo assim, o modelo dinmico para o mecanismo <u>RR</u> dado por:

$$\begin{bmatrix}
1 & 0 \\
0 & 1 \\
0 & 0 \\
l_{1g} & 0 \\
l_{1s_{2}} & 0 \\
l_{1c_{2}} & l_{2g}
\end{bmatrix}^{T} \left\{ \begin{bmatrix}
J_{z_{1}}\dot{\omega}_{z_{1}} \\
J_{z_{2}}\dot{\omega}_{z_{2}} \\
m_{1}\dot{v}_{x_{1}} \\
m_{1}\dot{v}_{y_{1}} \\
m_{2}\dot{v}_{x_{2}} \\
m_{2}\dot{v}_{y_{2}}
\end{bmatrix} + \begin{bmatrix}
0 \\
0 \\
-m_{1}\omega_{z_{1}}v_{y_{1}} \\
m_{1}\omega_{z_{1}}v_{y_{1}} \\
-m_{2}\omega_{z_{2}}v_{y_{2}} \\
m_{2}\omega_{z_{2}}v_{y_{2}}
\end{bmatrix} + g \begin{bmatrix}
0 \\
0 \\
m_{1}s_{1} \\
m_{1}c_{1} \\
m_{1}s_{1+2} \\
m_{1}c_{1+2}
\end{bmatrix} \right\} = \begin{bmatrix}
1 & 1 \\
0 & 1
\end{bmatrix}^{-1} \begin{bmatrix}
\tau_{1} \\
\tau_{2}
\end{bmatrix}$$
(41)

Repare que o modelo no depende das coordenadas q°. Elas foram uteis para a dedução do modelo, mas com o modelo deduzido elas no tem mais utilidade.

#### • Massa pontual:

$$\mathbf{q}_n = \begin{bmatrix} x_n \\ y_n \end{bmatrix} = \begin{bmatrix} q_{n,1} \\ q_{n,2} \end{bmatrix} \tag{42}$$

$$\mathbb{p}_n = \begin{bmatrix} p_{n,1} \\ p_{n,2} \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} \dot{q}_{n,1} \\ \dot{q}_{n,2} \end{bmatrix} \tag{43}$$

$$\begin{cases}
\begin{bmatrix} \dot{q}_{n,1} \\ \dot{q}_{n,2} \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} p_{n,1} \\ p_{n,2} \end{bmatrix} \\
\begin{bmatrix} M_n \dot{p}_{n,1} \\ M_n \dot{p}_{n,2} \end{bmatrix} + g \begin{bmatrix} 0 \\ M_n \end{bmatrix} = \begin{bmatrix} f_{n,1} \\ f_{n,2} \end{bmatrix}
\end{cases}$$
(44)

Que pode ser reescrito como:

$$\begin{bmatrix} M_n & \mathbf{0} \\ \mathbf{0} & M_n \end{bmatrix} \begin{bmatrix} \ddot{q}_{n,1} \\ \ddot{q}_{n,2} \end{bmatrix} + g \begin{bmatrix} \mathbf{0} \\ M_n \end{bmatrix} = \begin{bmatrix} f_{n,1} \\ f_{n,2} \end{bmatrix}$$

#### • RR:

$$q_n = \begin{bmatrix} \theta_{1\,n} \\ \theta_{2\,n} \end{bmatrix} = \begin{bmatrix} q_{n,1} \\ q_{n,2} \end{bmatrix} \tag{45}$$

$$\mathbb{p}_{n} = \begin{bmatrix} p_{n,1} \\ p_{n,2} \\ p_{n,3} \\ p_{n,4} \\ p_{n,5} \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ 1 & 1 \\ l_{g1} & 0 \\ l_{1}s_{n,2} & 0 \\ l_{g2} + l_{1}c_{n,2} & l_{g2} \end{bmatrix} \begin{bmatrix} \dot{q}_{n,1} \\ \dot{q}_{n,2} \end{bmatrix}$$
(46)

$$\begin{cases}
\begin{bmatrix} \dot{q}_{n,1} \\ \dot{q}_{n,2} \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ -1 & 1 \end{bmatrix} \begin{bmatrix} p_{n,1} \\ p_{n,2} \end{bmatrix} \\
\begin{bmatrix} 1 & 0 \\ 0 & 1 \\ l_{g1} & 0 \\ l_{1}s_{n,2} & 0 \\ l_{1}c_{n,2} & l_{g2} \end{bmatrix}^{T} \begin{bmatrix} J_{z1}\dot{p}_{n,1} \\ J_{z2}\dot{p}_{n,2} \\ m_{1}\dot{p}_{n,3} \\ m_{2}\dot{p}_{n,4} \\ m_{2}\dot{p}_{n,5} \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 0 \\ -m_{1}p_{n,2}p_{n,5} \\ m_{1}p_{n,2}p_{n,4} \end{bmatrix} + g \begin{bmatrix} 0 \\ 0 \\ m_{1}c_{n,1} \\ m_{2}s_{n,1+2} \\ m_{2}c_{n,1+2} \end{bmatrix} \right\} = \begin{bmatrix} u_{n,1} \\ u_{n,2} \end{bmatrix} \\
\begin{bmatrix} l_{g1} & 0 & -1 & 0 & 0 \\ l_{1}s_{n,2} & 0 & 0 & -1 & 0 \\ l_{1}c_{n,2} & l_{g2} & 0 & 0 & -1 \end{bmatrix} \begin{bmatrix} \dot{p}_{n,1} \\ \dot{p}_{n,2} \\ \dot{p}_{n,3} \\ \dot{p}_{n,4} \\ \dot{p}_{n,5} \end{bmatrix} = - \begin{bmatrix} 0 \\ l_{1}c_{n,2}p_{n,1}(-p_{n,1} + p_{n,2}) \\ l_{1}s_{n,2}p_{n,1}(p_{n,1} - p_{n,2}) \end{bmatrix}$$

$$(47)$$

Que pode ser reescrito como:

$$\begin{bmatrix} J_{z1} + J_{z2} + m_1 l_{g1}^2 + m_2 (l_1^2 + 2 l_1 l_{g2} \mathsf{c}_{n,2} + l_{g2}^2) & J_{z2} + m_2 l_{g2} (l_1 \mathsf{c}_{n,2} + l_{g2}) \end{bmatrix} \begin{bmatrix} \ddot{q}_{n,1} \\ J_{z2} + m_2 l_{g2} (l_1 \mathsf{c}_{n,2} + l_{g2}) & J_{z2} + m_2 l_{g2}^2 \end{bmatrix} \begin{bmatrix} \ddot{q}_{n,1} \\ \ddot{q}_{n,2} \end{bmatrix} \\ + \begin{bmatrix} -m_2 l_1 l_{g2} \mathsf{s}_{n,2} \dot{q}_{n,2}^2 - 2 m_2 l_1 l_{g2} \mathsf{s}_{n,2} \dot{q}_{n,1} \dot{q}_{n,2} \\ m_2 l_1 l_{g2} \mathsf{s}_{n,2} \dot{q}_{n,1}^2 \end{bmatrix} + g \begin{bmatrix} m_1 l_{g1} \mathsf{c}_{n,1} + m_2 (l_{g2} \mathsf{c}_{n,1+2} + l_1 \mathsf{c}_{n,1}) \\ m_2 l_{g2} \mathsf{c}_{n,1+2} \end{bmatrix} = \begin{bmatrix} u_{n,1} \\ u_{n,2} \end{bmatrix}$$

• RR (0) com 2 massas acopladas (1 e 2):

$$\begin{cases} \begin{bmatrix} \dot{q}_{0,1} \\ \dot{q}_{0,2} \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ -1 & 1 \end{bmatrix} \begin{bmatrix} p_{0,1} \\ p_{0,2} \end{bmatrix} \\ \begin{bmatrix} 1 & 0 \\ 0 & 1 \\ l_{1}s_{0,2} & 0 \\ l_{1}c_{0,2} & l_{g2} \\ -L_{1}s_{0,1} & 0 \\ l_{1}s_{0,2} & 0 \end{bmatrix}^{T} \begin{cases} \begin{bmatrix} J_{z_{1}}\dot{p}_{0,1} \\ J_{z_{2}}\dot{p}_{0,2} \\ m_{1}\dot{p}_{0,3} \\ m_{2}\dot{p}_{0,4} \\ m_{2}\dot{p}_{0,5} \\ M_{1}\dot{p}_{1,1} \\ M_{2}\dot{p}_{2,1} \end{bmatrix}^{T} \begin{pmatrix} 0 \\ 0 \\ 0 \\ -m_{1}p_{0,2}p_{0,5} \\ m_{1}p_{0,2}p_{0,4} \\ 0 \\ 0 \\ 0 \end{bmatrix} + g \begin{bmatrix} 0 \\ 0 \\ m_{1}c_{0,1} \\ m_{2}s_{0,1+2} \\ m_{2}c_{0,1+2} \\ 0 \\ M_{1} \\ 0 \\ M_{2} \end{bmatrix} \\ = \begin{bmatrix} u_{0,1} \\ u_{0,2} \end{bmatrix}$$
 
$$\begin{bmatrix} l_{g1} & 0 & -1 & 0 & 0 & 0 & 0 & 0 & 0 \\ l_{1}s_{i,2} & 0 & 0 & -1 & 0 & 0 & 0 & 0 & 0 \\ l_{1}s_{i,2} & 0 & 0 & -1 & 0 & 0 & 0 & 0 & 0 \\ l_{1}s_{0,1} & l_{2}s_{0,1+2} & 0 & 0 & 0 & 0 & 1 & 0 & 0 \\ l_{1}s_{0,1} & l_{2}s_{0,1+2} & 0 & 0 & 0 & 0 & 0 & 1 & 0 & 0 \\ l_{1}s_{0,1} & l_{2}s_{0,1+2} & 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 \\ l_{1}s_{0,1} & l_{2}s_{0,1+2} & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 \\ l_{1}s_{0,1} & l_{2}s_{0,1+2} & 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 \\ l_{1}s_{0,1} & l_{2}s_{0,1+2} & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 \\ l_{1}s_{0,1} & l_{2}s_{0,1+2} & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 \\ l_{1}s_{0,1} & l_{2}s_{0,1+2} & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 \\ l_{1}s_{0,1} & l_{2}s_{0,1+2} & l_{2}s_{0,1+2} & l_{2}s_{0,1+2} & l_{2}s_{0,1+2} \\ l_{1}s_{0,1}p_{0,1}^{2} & l_{1}s_{0,1+2}p_{0,2}^{2} \\ l_{1}s_{0,1}p_{0,1}^{2} & l_{1}$$

Que pode ser reescrito como:

$$\mathbb{M}^{\#}\ddot{q}_{0} + \mathbf{v}^{\#} + \mathbf{g}^{\#} = \mathbf{u}_{0} \tag{49}$$

Sendo:

$$\mathbb{M}_{1,1}^{\#} = J_{z1} + J_{z2} + M_1 L_1^2 + M_2 L_2^2 + m_1 l_{g1}^2 + m_2 l_{g2}^2 + \left(M_2 + m_2\right) l_1^2 + 2 l_1 \mathsf{c}_{0,2} \left(L_2 M_2 + m_2 l_{g2}\right) \ (50)$$

$$\mathbb{M}_{1,2}^{\#} = \mathbb{M}_{2,1}^{\#} = J_{z2} + M_2 L_2 (l_1 c_{0,2} + L_2) + m_2 l_{g2} (l_1 c_{0,2} + l_{g2})$$
(51)

$$\mathbb{M}_{2,2}^{\#} = J_{z2} + M_2 L_2^2 + m_2 l_{g2}^2 \tag{52}$$

$$\mathbf{v}_{1}^{\#} = -(M_{2}L_{2} + m_{2}l_{q2})l_{1}\mathbf{s}_{0,2}\dot{q}_{0,2}(2\dot{q}_{0,1} + \dot{q}_{0,2}) \tag{53}$$

$$\mathbf{v}_{2}^{\#} = (M_{2}L_{2} + m_{2}l_{g2})l_{1}\mathbf{s}_{0,2}\dot{q}_{0,1}^{2} \tag{54}$$

$$g_1^{\#} = g(M_1L_1c_{0,1} + m_1l_{g1}c_{0,1} + (M_2 + m_2)l_1c_{0,1} + (M_2L_2 + m_2l_{g2})c_{0,1+2})$$
(55)

$$g_2^{\#} = g(M_2 L_2 + m_2 l_{q2}) c_{0,1+2} \tag{56}$$

## • RR balanceado:

Escolhendo  $L_1$  e  $L_2$  de modo que  $\mathfrak{g}^{\#}=\mathbb{O}$ :

$$\begin{cases}
L_1 = -\frac{M_2 l_1 + m_1 l_{g1} + m_2 l_2}{M_1} \\
L_2 = -\frac{m_2 l_{g2}}{M_2}
\end{cases}$$
(57)

Obtemos o seguinte sistema:

$$\mathbb{M}_{1,1}^{\#} = J_{z1} + J_{z2} + m_1 l_{g2}^2 + l_1^2 (M_2 + m_2) + \frac{m_2^2 l_{g2}^2}{M_2} + \frac{(m_1 l_{g1} + (M_2 + m_2) l_1)^2}{M_1}$$
(58)

$$\mathbb{M}_{1,2}^{\#} = \mathbb{M}_{2,1}^{\#} = \mathbb{M}_{2,2}^{\#} = J_{z2} + \frac{m_2(M_2 + m_2)l_{g2}^2}{M_2}$$
(59)

$$v_1^{\#} = 0$$
 (60)

$$v_2^{\#} = 0 \tag{61}$$

$$\mathfrak{g}_1^{\#} = \mathbf{0} \tag{62}$$

$$g_2^{\#} = 0 \tag{63}$$

Ou seja:

$$\begin{bmatrix} J_{z1} + J_{z2} + m_1 l_{g2}^2 + l_1^2 (M_2 + m_2) + \frac{m_2^2 l_{g2}^2}{M_2} + \frac{(m_1 l_{g1} + (M_2 + m_2) l_1)^2}{M_1} & J_{z2} + \frac{m_2 (M_2 + m_2) l_{g2}^2}{M_2} \\ J_{z2} + \frac{m_2 (M_2 + m_2) l_{g2}^2}{M_2} & J_{z2} + \frac{m_2 (M_2 + m_2) l_{g2}^2}{M_2} \end{bmatrix} \begin{bmatrix} \ddot{q}_{n,1} \\ \ddot{q}_{n,2} \end{bmatrix} = \begin{bmatrix} u_{n,1} \\ u_{n,2} \end{bmatrix}$$
(64)

• 5R balanceado:

$$\begin{cases}
\begin{bmatrix}
1 & 0 & 1 \\
0 & 1 \\
-\frac{c_{1,1+2}}{l_1s_{1,2}} & \frac{s_{1,1+2}}{l_1s_{1,2}} \\
-\frac{l_{1}c_{1,1}+l_2c_{1,1+2}}{l_1l_2s_{1,2}} & -\frac{l_{1}s_{1,1}+l_2s_{1,1+2}}{l_1l_2s_{1,2}} \\
-\frac{l_{1}c_{1,1}+l_2c_{1,1+2}}{l_1l_2s_{2,2}} & -\frac{l_{1}s_{1,1}+l_2s_{1,1+2}}{l_1l_2s_{2,2}}
\end{bmatrix}
\begin{bmatrix}
0 & 0 & 0 & 0 & 0 & 0 & 0 \\
0 & 0 & M_{1,1}^{\#} & M_{1,2}^{\#} & 0 & 0 \\
0 & 0 & M_{1,2}^{\#} & M_{1,2}^{\#} & 0 & 0 \\
0 & 0 & 0 & M_{1,1}^{\#} & M_{1,2}^{\#} & 0 \\
0 & 0 & 0 & M_{1,1}^{\#} & M_{1,2}^{\#} \\
-\frac{l_{1}c_{2,1}+l_2c_{2,1+2}}{l_1l_2s_{2,2}} & -\frac{l_{1}s_{2,1}+l_2s_{2,1+2}}{l_1l_2s_{2,2}}
\end{bmatrix}
\begin{bmatrix}
u_{1,1} \\
u_{2,1} \\
u_{2,1}
\end{bmatrix}$$

$$\begin{bmatrix}
1 & 0 & l_{1}s_{1,1} + l_{2}s_{1,1+2} & l_{2}s_{1,1+2} & 0 & 0 \\
0 & 1 & -l_{1}c_{1,1} - l_{2}c_{1,1+2} & -l_{2}c_{1,1+2} & 0 & 0 \\
0 & 1 & 0 & 0 & 0 & l_{1}s_{2,1} + l_{2}s_{2,1+2} & l_{2}s_{2,1+2} \\
1 & 0 & 0 & 0 & -l_{1}c_{2,1} - l_{2}c_{2,1+2} & -l_{2}c_{2,1+2}
\end{bmatrix}
\begin{bmatrix}
\ddot{q}_{0,1} \\
\ddot{q}_{0,2} \\
\ddot{q}_{1,1} \\
\ddot{q}_{0,2} \\
\ddot{q}_{1,1} \\
\ddot{q}_{1,2} \\
\ddot{q}_{2,2}
\end{bmatrix} =$$

$$\begin{bmatrix}
1 & 0 & l_{1}s_{1,1} + l_{2}s_{1,1+2} & l_{2}s_{1,1+2} & 0 & 0 \\
0 & 1 & -l_{1}c_{1,1} - l_{2}c_{1,1+2} & -l_{2}c_{1,1+2} & l_{2}s_{2,1+2} \\
0 & 1 & 0 & 0 & -l_{1}c_{2,1} - l_{2}c_{2,1+2} & -l_{2}c_{2,1+2}
\end{bmatrix}
\begin{bmatrix}
\ddot{q}_{0,1} \\
\ddot{q}_{0,2} \\
\ddot{q}_{1,1} \\
\ddot{q}_{1,2} \\
\ddot{q}_{2,1} \\
\ddot{q}_{2,1} \\
\ddot{q}_{2,1} \\
\ddot{q}_{2,2}
\end{bmatrix} =$$

$$\begin{bmatrix}
l_{1}c_{1,1}\dot{q}_{1,1}^{2} + l_{2}c_{1,1+2}(\dot{q}_{1,1} + \dot{q}_{1,2})^{2} \\
l_{1}s_{1,1}\dot{q}_{1,1}^{2} + l_{2}s_{1,1+2}(\dot{q}_{1,1} + \dot{q}_{1,2})^{2} \\
l_{1}c_{2,1}\dot{q}_{2,1}^{2} + l_{2}c_{2,1+2}(\dot{q}_{2,1} + \dot{q}_{2,2})^{2} \\
l_{1}s_{2,1}\dot{q}_{2,1}^{2} + l_{2}s_{2,1+2}(\dot{q}_{2,1} + \dot{q}_{2,2})^{2}
\end{bmatrix}$$

1.2 Control

$$\begin{bmatrix} \mathbb{C}^T \mathbb{M} \\ \mathbb{A} \end{bmatrix} \ddot{\mathbf{q}} = \begin{bmatrix} \mathbb{Q} \mathbf{u} \\ -\mathbb{b} \end{bmatrix}$$

$$\mathbf{u} = \mathbb{Q}^{-1} \mathbb{C}^T \mathbb{M} \mathbf{u}'$$

$$\mathbf{u}' = \ddot{\mathbf{q}}_d + \lambda \dot{\mathbf{e}} - k \operatorname{sign}(\mathbf{s})$$

$$\mathbf{s} = -\dot{\mathbf{e}} - \lambda \mathbf{e}$$

$$\dot{\mathbf{s}} = -\ddot{\mathbf{e}} - \lambda \dot{\mathbf{e}} = \ddot{\mathbf{q}} - \ddot{\mathbf{q}}_d - \lambda \dot{\mathbf{e}}$$

$$\dot{\mathbf{s}} = \begin{bmatrix} \mathbb{C}^T \mathbb{M} \\ \mathbb{A} \end{bmatrix}^{-1} \begin{bmatrix} \mathbb{Q} \mathbf{u} \\ -\mathbb{b} \end{bmatrix} - \ddot{\mathbf{q}}_d - \lambda \dot{\mathbf{e}}$$

Aplicando a lei de controle:

$$\begin{split} \dot{\mathbf{s}} &= \begin{bmatrix} \mathbb{C}^T \mathbb{M} \\ \mathbb{A} \end{bmatrix}^{-1} \begin{bmatrix} \mathbb{C}^T \mathbb{M} (\ddot{\mathbf{q}}_d + \lambda \dot{\mathbf{e}} - k \operatorname{sign}(\mathbf{s})) \\ -\mathbb{b} \end{bmatrix} - \ddot{\mathbf{q}}_d - \lambda \dot{\mathbf{e}} \\ \dot{\mathbf{s}} &= \begin{bmatrix} \mathbb{C}^T \mathbb{M} \\ \mathbb{A} \end{bmatrix}^{-1} \begin{bmatrix} \mathbb{C}^T \mathbb{M} (\ddot{\mathbf{q}}_d + \lambda \dot{\mathbf{e}} - k \operatorname{sign}(\mathbf{s})) - \mathbb{C}^T \mathbb{M} (\ddot{\mathbf{q}}_d + \lambda \dot{\mathbf{e}}) \\ -\mathbb{b} - \mathbb{A} (\ddot{\mathbf{q}}_d + \lambda \dot{\mathbf{e}}) \end{bmatrix} \\ \dot{\mathbf{s}} &= \begin{bmatrix} \mathbb{C}^T \mathbb{M} \\ \mathbb{A} \end{bmatrix}^{-1} \begin{bmatrix} -\mathbb{C}^T \mathbb{M} k \operatorname{sign}(\mathbf{s}) \\ -\mathbb{b} - \mathbb{A} (\ddot{\mathbf{q}}_d + \lambda \dot{\mathbf{e}}) \end{bmatrix} \end{split}$$

Definindo:

$$\begin{bmatrix} \mathbb{C}^T \mathbb{M} \\ \mathbb{A} \end{bmatrix}^{-1} = \begin{bmatrix} (\mathbb{C}^T \mathbb{M})^\dagger & \mathbb{A}^\dagger \end{bmatrix}$$

Temos:

$$\dot{\mathbf{s}} = -(\mathbb{C}^T \mathbb{M})^{\dagger} \mathbb{C}^T \mathbb{M} k \operatorname{sign}(\mathbf{s}) - \mathbb{A}^{\dagger} \mathbb{b} - \mathbb{A}^{\dagger} \mathbb{A} (\ddot{\mathbf{q}}_d + \lambda \dot{\mathbf{e}})$$

Sendo assim, se a seguinte inequa<br/>o for respeitada para pelo menos  $\nu^{\scriptscriptstyle\#}$  componentes de <br/> <br/>  $\dot{s},$ o erro vai a zero:

$$\dot{\mathbf{s}} = -(\mathbb{C}^T \mathbb{M})^{\dagger} \mathbb{C}^T \mathbb{M} k \operatorname{sign}(\mathbf{s}) - \mathbb{A}^{\dagger} \mathbb{b} - \mathbb{A}^{\dagger} \mathbb{A} (\ddot{\mathbf{q}}_d + \lambda \dot{\mathbf{e}}) \leq \mathbb{0}$$

# Acknowledgments