Design of a Fuzzy-PID Controller for Controlling Quadrotor Attitude and Altitude

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Overview

- ✓ Motivation
- **✓** Challenges
- ✓ Mathematical modeling of quadrotor
- ✓ PID controller design
- ✓ Fuzzy-PID controller design
- ✓ Simulation
- **✓** Conclusion

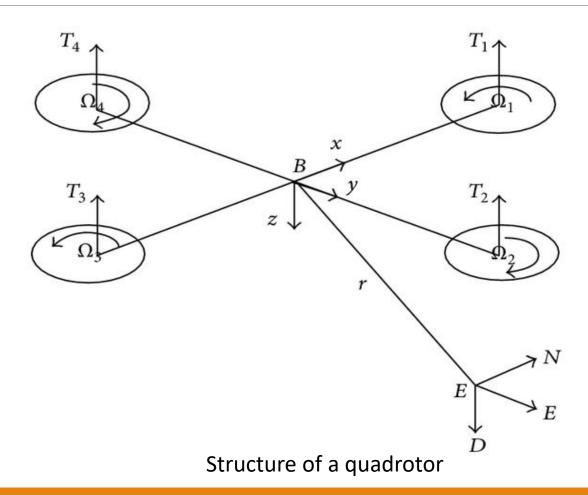
Motivation

- ✓ Fuzzy PID Controller solves the problem of fixed gain
- ✓ Fuzzy PID Controller is cheaper to develop
- ✓ Fuzzy PID Controller covers a wider range of operating conditions
- ✓ Fuzzy PID Controllers are more readily customizable
- ✓ Fuzzy PID Controller is robust in terms of uncertainty

Challenges

- ✓ Nonlinear system
- ✓ Under actuated system
- ✓ Low on board processing capability
- ✓ Low operation time
- ✓ Low efficiency in power consumption

Mathematical modeling of quadrotor



$$R(x,\phi) = \begin{bmatrix} 1 & 0 & 0 \\ 0 & \cos\phi & \sin\phi \\ 0 & -\sin\phi & \cos\phi \end{bmatrix}$$

$$R(y,\theta) = \begin{bmatrix} \cos\theta & 0 & -\sin\theta \\ 0 & 1 & 0 \\ \sin\theta & 0 & \cos\theta \end{bmatrix}$$

$$R(z, \psi) = \begin{bmatrix} \cos \psi & \sin \psi & 0 \\ -\sin \psi & \cos \psi & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

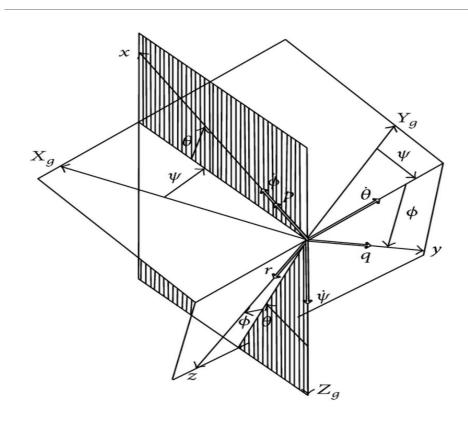
$$R_{E \to B} = R(x, \phi) R(y, \theta) R(z, \psi)$$

$$R_{B \to E} = R_{E \to B}^T.$$

$$R_{B \to E} = \begin{bmatrix} c_{\theta}c_{\psi} & s_{\phi}s_{\theta}c_{\psi} - c_{\phi}s_{\psi} & c_{\phi}s_{\theta}c_{\psi} + s_{\phi}s_{\psi} \\ c_{\theta}s_{\psi} & s_{\phi}s_{\theta}s_{\psi} + c_{\phi}c_{\psi} & c_{\phi}s_{\theta}s_{\psi} - s_{\phi}c_{\psi} \\ -s_{\theta} & s_{\phi}c_{\theta} & c_{\phi}c_{\theta} \end{bmatrix}$$

where $c_{\cdot} = \cos(\cdot)$ and $s_{\cdot} = \sin(\cdot)$.

roll angle ϕ , pitch angle θ , and yaw angle ψ around x-, y-, and z-axes, respectively.



$$\begin{bmatrix} p \\ q \\ r \end{bmatrix} = R_r \begin{bmatrix} \dot{\phi} \\ \dot{\theta} \\ \dot{\psi} \end{bmatrix}$$

where

$$R_r = \begin{bmatrix} 1 & 0 & -\sin\theta \\ 0 & \cos\phi & \sin\phi\cos\theta \\ 0 & -\sin\phi & \cos\phi\cos\theta \end{bmatrix}$$

The relationships between angular velocity components and the attitude angle change rate.

<u>Dynamic Model:</u> The dynamics model is composed of the rotational and translational motions. The rotational motion is fully actuated, while the translational motion is underactuated. In the body coordinate system, the rotational motion equations are derived according to the law of momentum theorem and gyroscopic effect of quadrotor, and they are given by,

$$J\dot{\omega}+\omega\times J\omega+\omega\times \left[0\ 0\ J_r\Omega_r\right]=M_B$$

$$F_i=b\Omega_i^2$$

$$M_i=d\Omega_i^2$$
 where Ω_i $(i=1,2,3,4)$ represents the i th rotor speed.

$$M_{B} = \begin{bmatrix} l \cdot b \left(-\Omega_{2}^{2} + \Omega_{4}^{2} \right) \\ l \cdot b \left(\Omega_{1}^{2} - \Omega_{3}^{2} \right) \\ d \left(\Omega_{1}^{2} - \Omega_{2}^{2} + \Omega_{3}^{2} - \Omega_{4}^{2} \right) \end{bmatrix} \qquad m\ddot{r} = \begin{bmatrix} 0 & 0 & mg \end{bmatrix}^{T} + RF_{B}, \qquad F_{B} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ -b \left(\Omega_{1}^{2} + \Omega_{2}^{2} + \Omega_{3}^{2} + \Omega_{4}^{2} \right) \end{bmatrix}$$

The Motion Equations of Quadrotor:

$$\begin{split} \ddot{\phi} &= \dot{\theta} \dot{\psi} \left(\frac{I_y - I_z}{I_x} \right) - \frac{J_r}{I_x} \dot{\theta} \Omega_r + \frac{L}{I_x} U_2, \\ \ddot{\theta} &= \dot{\phi} \dot{\psi} \left(\frac{I_z - I_x}{I_y} \right) + \frac{J_r}{I_y} \dot{\phi} \Omega_r + \frac{L}{I_y} U_3, \\ \ddot{\psi} &= \dot{\phi} \dot{\theta} \left(\frac{I_x - I_y}{I_z} \right) + \frac{1}{I_z} U_4, \\ \ddot{x} &= -\frac{U_1}{m} \left(\cos \phi \sin \theta \cos \psi + \sin \phi \sin \psi \right), \\ \ddot{y} &= -\frac{U_1}{m} \left(\cos \phi \sin \theta \sin \psi - \sin \phi \cos \psi \right), \\ \ddot{z} &= g - \frac{U_1}{m} \left(\cos \phi \cos \theta \right), \end{split}$$

where U_1 , U_2 , U_3 , and U_4 are the control input variables, which can be calculated by $U_1 = b(\Omega_1^2 + \Omega_2^2 + \Omega_3^2 + \Omega_4^2)$, $U_2 = b(-\Omega_2^2 + \Omega_4^2)$, $U_3 = b(\Omega_1^2 - \Omega_3^2)$, and $U_4 = d(\Omega_1^2 - \Omega_2^2 + \Omega_3^2 - \Omega_4^2)$, respectively.

Cont. (State space model)

The state space model adopted by the control system is $\dot{X} = f(X, U)$, where X is the state vector and U is the control input vector. The state vector is chosen as $X = [x \ y \ z \ \dot{x} \ \dot{y} \ \dot{z} \ \phi \ \theta \ \psi \ \dot{\phi} \ \dot{\theta} \ \dot{\psi}]^T$. In the design of controller, the state variables are chosen as $x_1 = x, x_2 = y, x_3 = z, x_4 = \dot{x}_1 = \dot{x}, x_5 = \dot{x}_2 = \dot{y}, x_6 = \dot{x}_3 = \dot{z}, x_7 = \phi, x_8 = \theta, x_9 = \psi, x_{10} = \dot{x}_7 = \dot{\phi}, x_{11} = \dot{x}_8 = \dot{\theta}, \text{ and } x_{12} = \dot{x}_9 = \dot{\psi}.$

Synthesizing the motion equations of quadrotor, the state vector, and the control input variables, the state equations can be described as

$$f(X,U) = \begin{bmatrix} \frac{U_1}{m} (\cos x_7 \sin x_8 \cos x_9 + \sin x_7 \sin x_9) \\ -\frac{U_1}{m} (\cos x_7 \sin x_8 \sin x_9 - \sin x_7 \cos x_9) \\ g - \frac{U_1}{m} (\cos x_7 \cos x_8) \\ x_{10} \\ x_{11} \\ x_{12} \end{bmatrix}$$

$$x_{11} x_{12} \frac{I_y - I_z}{I_x} - \frac{I_r}{I_x} x_{11} \Omega_r + \frac{L}{I_x} U_2 \\ x_{10} x_{12} \frac{I_z - I_x}{I_y} + \frac{I_r}{I_y} x_{10} \Omega_r + \frac{L}{I_y} U_3 \\ x_{10} x_{11} \frac{I_x - I_y}{I_z} + \frac{1}{I_z} U_4$$

Underactuation

Underactuation is a technical term used in robotics and control theory to describe mechanical systems that cannot be commanded to follow arbitrary trajectories in configuration space. This condition can occur for a number of reasons, the simplest of which is when the system has a lower number of actuators than degrees of freedom. In this case, the system is said to be trivially underactuated.

The class of underactuated mechanical systems is very rich and includes such diverse members as automobiles, airplanes, and even animals.

$$\ddot{q} = f(q, \dot{q}, u, t)$$

Where:

 $q \in \mathbb{R}^n$ is the position state vector

 $u \in \mathbb{R}^m$ is the vector of control inputs

t is time.

Furthermore, in many cases the dynamics for these systems can be rewritten to be affine in the control inputs:

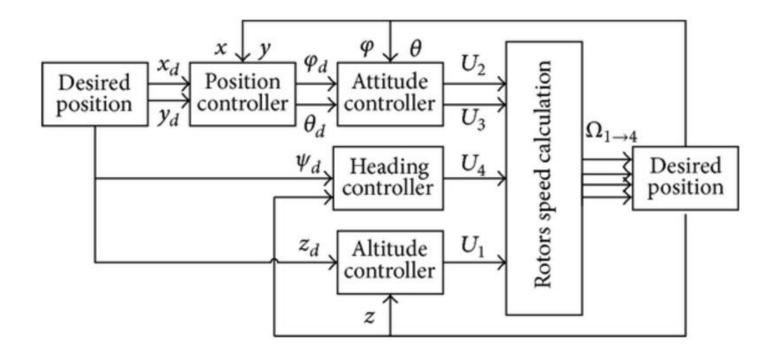
$$\ddot{q}=f_1(q,\dot{q},t)+f_2(q,\dot{q},t)u$$

When expressed in this form, the system is said to be underactuated if:[1]

$$rank[f_2(q,\dot{q},t)] < dim[q]$$

When this condition is met, there are acceleration directions that can not be produced no matter what the control vector is.

PID Controller Design



Quadrotor control structure

Hovering Condition

```
For
hovering,
```

$$\Phi = 0;$$
 Φ " = 0;

$$\theta = 0;$$
 Θ `` = 0; $U_3 = 0$

$$\Psi = 0; \qquad \begin{array}{c} \Psi = 0; \\ X^{"} = 0; \end{array}$$

$$\Phi' = 0; Y'' = 0;$$

$$\Theta' = 0;$$
 $Z'' = 0;$

$$\Psi$$
 = 0;

$$X' = 0;$$

$$Y' = 0;$$

$$Z' = 0;$$

So, for hovering Condition the control inputs are,

$$U_1 = mg$$

$$U_2 = 0$$

$$U_3 = 0$$

$$\Psi$$
`` = 0; $U_4 = 0$

where
$$a_1 = (I_y - I_z)/I_x$$
, $a_2 = J_r/I_x$, $a_3 = (I_z - I_x)/I_y$, $a_4 = J_r/I_y$, $a_5 = (I_x - I_y)/I_z$, $b_1 = L/I_x$, $b_2 = L/I_y$, and $b_3 = 1/I_z$.

Feedback Linearization

$$\begin{split} \dot{x}_{10} &= a_1 x_{11} x_{12} + b_1 U_2, \\ \dot{x}_{11} &= a_3 x_{10} x_{12} + b_2 U_3, \\ \dot{x}_{12} &= a_5 x_{10} x_{11} + b_3 U_4. \end{split}$$

$$\dot{x}_{10} = a_1 x_{11} x_{12} + b_1 U_2,
\dot{x}_{11} = a_3 x_{10} x_{12} + b_2 U_3,
\dot{x}_{12} = a_5 x_{10} x_{11} + b_3 U_4.$$

$$U_2 = f_2 (x_{10}, x_{11}, x_{12}) + U_2^{\#},
U_3 = f_3 (x_{10}, x_{11}, x_{12}) + U_3^{\#},
U_4 = f_4 (x_{10}, x_{11}, x_{12}) + U_4^{\#},$$

$$a_1 x_{11} x_{12} + b_1 f_2 (x_{10}, x_{11}, x_{12}) = K_2 x_{10},$$

$$a_3 x_{10} x_{12} + b_2 f_3 (x_{10}, x_{11}, x_{12}) = K_3 x_{11},$$

$$a_5 x_{10} x_{11} + b_3 f_4 (x_{10}, x_{11}, x_{12}) = K_4 x_{10},$$

$$\dot{x}_{10} = K_2 x_{10} + b_1 U_2^{\#},$$

$$\dot{x}_{11} = K_3 x_{11} + b_2 U_3^{\#},$$

$$\dot{x}_{12} = K_4 x_{12} + b_3 U_4^{\#}.$$

$$f_2(x_{10}, x_{11}, x_{12}) = \frac{1}{b_1} (K_2 x_{10} - a_1 x_{11} x_{12}),$$

$$f_3(x_{10}, x_{11}, x_{12}) = \frac{1}{b_2} (K_3 x_{11} - a_3 x_{10} x_{12}),$$

$$f_4(x_{10}, x_{11}, x_{12}) = \frac{1}{b_3} (K_4 x_{10} - a_5 x_{10} x_{11}).$$

where K_2 , K_3 , and K_4 are undetermined parameters.

Laplace Transformation of the Linear Model and PID Controller

$$G_{1}(s) = \frac{X_{7}(s)}{U_{2}^{\#}(s)} = \frac{b_{1}}{s^{2} - K_{2}s},$$

$$G_{2}(s) = \frac{X_{8}(s)}{U_{3}^{\#}(s)} = \frac{b_{2}}{s^{2} - K_{3}s},$$

$$G_{3}(s) = \frac{X_{9}(s)}{U_{4}^{\#}(s)} = \frac{b_{3}}{s^{2} - K_{4}s},$$

$$K(s) = k_p + \frac{k_i}{s} + k_d s$$
$$= \frac{k_p s + k_i + k_d s^2}{s}$$

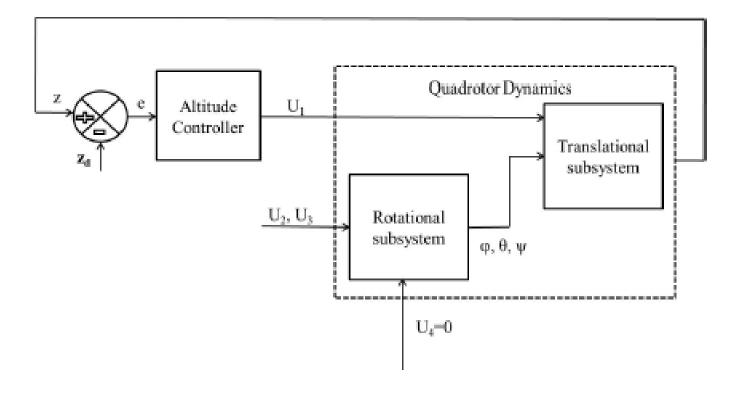
Where,

 k_p = proportional gain

k_i = integral gain

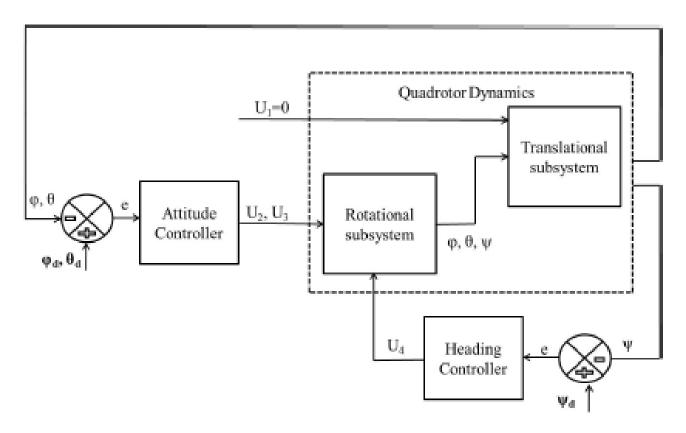
kd = differential gain

Altitude Controller



Block Diagram for Altitude Controller

Attitude and Heading Controller



Block Diagram for Attitude and Heading Controller

Position Controller

Position can not be controlled directly for a quadcopter.

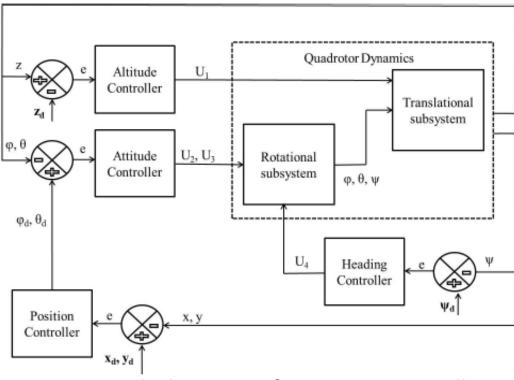
$$\ddot{x} = \frac{-U_1}{m} (\sin \phi_d \sin \psi + \cos \phi_d \sin \theta_d \cos \psi)$$
$$\ddot{y} = \frac{-U_1}{m} (\cos \phi_d \sin \theta_d \sin \psi - \sin \phi_d \cos \psi)$$

 $\ddot{x} = \frac{-U_1}{m} (\phi_d \sin \psi + \theta_d \cos \psi)$ $\ddot{y} = \frac{-U_1}{m} (\theta_d \sin \psi - \phi_d \cos \psi)$

small angle assumption ($\sin \phi_d \equiv \phi_d$, $\sin \theta_d \equiv \theta_d$ and $\cos \phi_d = \cos \theta_d = 1$)

$$\begin{bmatrix} \phi_d \\ \theta_d \end{bmatrix} = \begin{bmatrix} -\sin\psi & -\cos\psi \\ \cos\psi & -\sin\psi \end{bmatrix}^{-1} \frac{m}{U_1} \begin{bmatrix} \ddot{x}_d \\ \ddot{y}_d \end{bmatrix}$$
$$= \frac{m}{U_1} \begin{bmatrix} -\sin\psi & \cos\psi \\ -\cos\psi & -\sin\psi \end{bmatrix} \begin{bmatrix} \ddot{x}_d \\ \ddot{y}_d \end{bmatrix}$$
$$= \frac{m}{U_1} \begin{bmatrix} -\ddot{x}_d\sin\psi + \ddot{y}_d\cos\psi \\ -\ddot{x}_d\cos\psi - \ddot{y}_d\sin\psi \end{bmatrix}$$

Position Controller (cont.)



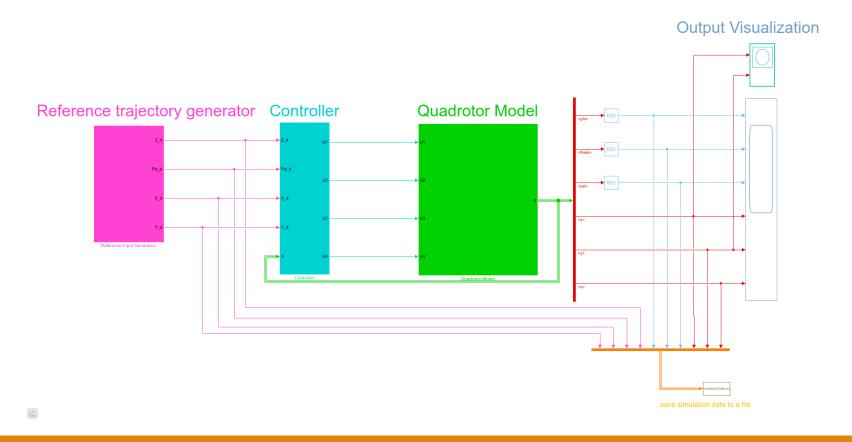
Block Diagram for Position Controller

Simulation

Parameters	Description	Values
m	Quadrotor mass	0.4794 kg
g	Gravitational acceleration	9.8 m/s ²
I	Moment arm	0.225 m
l _x	MOI about body frame's x-axis	0.0086 kg.m ²
$I_{\mathbf{y}}$	MOI about body frame's y-axis	0.0086 kg.m ²
l _z	MOI about body frame's z-axis 0.0172 kg.m²	
J_r	Rotor inertia	3.7404 x 10 ⁻⁵ kg.m ²
b	Aerodynamic force constant	3.13 x 10 ⁻⁵ N.s ²
d	Aerodynamic moment constant	9 x 10 ⁻⁷ Nm.s ²
K ₂	Linearization constant	-14.6211
K ₃	Linearization constant	-14.6211
K ₄	Linearization constant	-32

Simulink Model

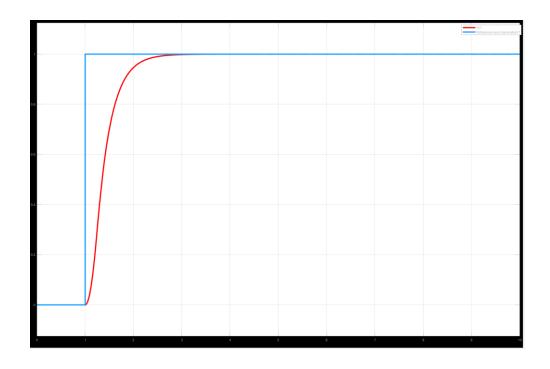
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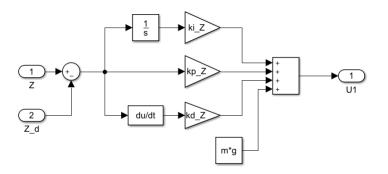
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Altitude Controller(Simulation Result)

$$U_1 = k_p(z - z_d) + k_d(\dot{z} - \dot{z}_d) + k_i \int (z - z_d) dt$$

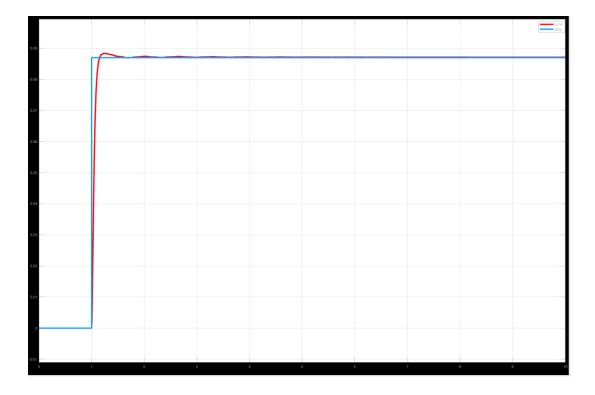


Parameter	Description	Value
k_p	Proportional gain	30
k _i	Integral gain	0
k_d	Derivative gain	10
t _r	Rising time	702.5 ms
t_s	Settling time	1.5 s
%OS	Overshoot	0

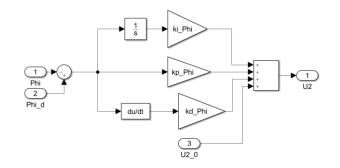


Roll Controller

$$U_2 = k_p(\phi_d - \phi) + k_d(\dot{\phi}_d - \dot{\phi}) + k_i \int (\phi_d - \phi)dt$$

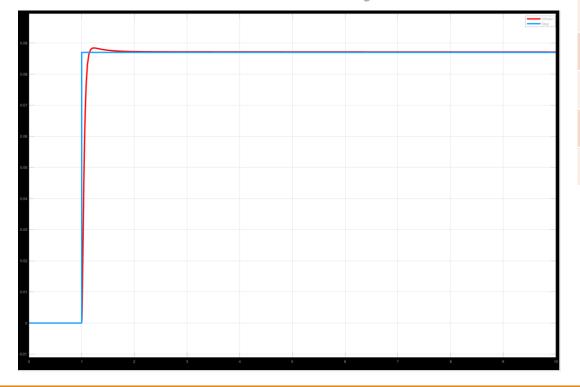


Parameter	Description	Value
k_p	Proportional gain	2.9996
k _i	Integral gain	0.17
k_{d}	Derivative gain	0.87
t _r	Rising time	110.1 ms
t_s	Settling time	597.5 ms
%OS	Overshoot	1.37

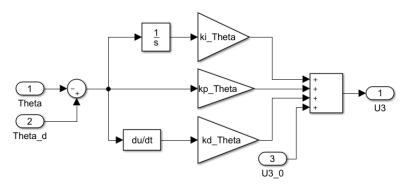


Pitch Controller

$$U_3 = k_p(\theta_d - \theta) + k_d(\dot{\theta}_d - \dot{\theta}) + k_i \int (\theta_d - \theta) dt$$



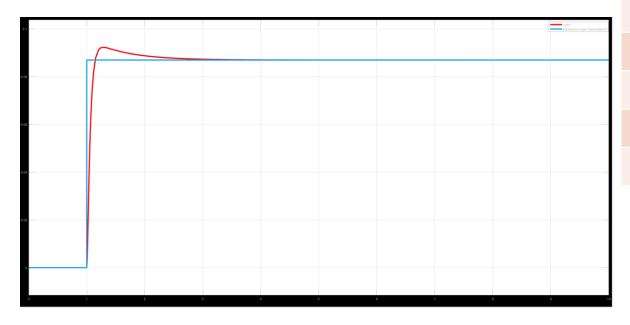
Parameter	Description	Value
k_p	Proportional gain	2.9996
k_{i}	Integral gain	0.17
k_d	Derivative gain	0.87
t_r	Rising time	110.1 ms
t_s	Settling time	597.5 ms
%OS	Overshoot	1.37



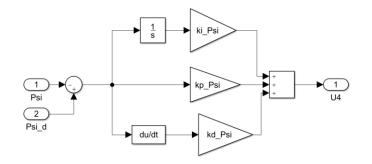
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Yaw Controller

$$U_4 = k_p(\psi_d - \psi) + k_d(\dot{\psi}_d - \dot{\psi}) + k_i \int (\psi_d - \psi) dt$$

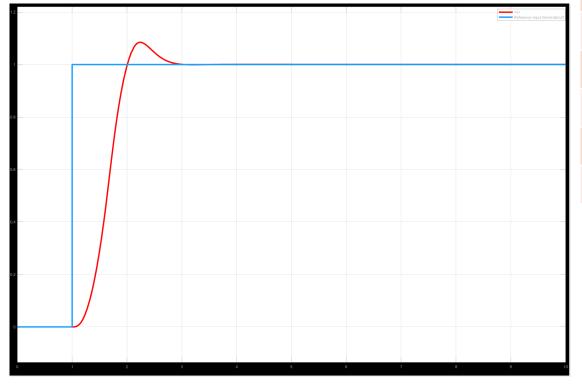


Parameter	Description	Value
k_p	Proportional gain	3.9
k_{i}	Integral gain	0
k_{d}	Derivative gain	2.57
t_r	Rising time	94 ms
t_s	Settling time	1.5s
%OS	Overshoot	6.09

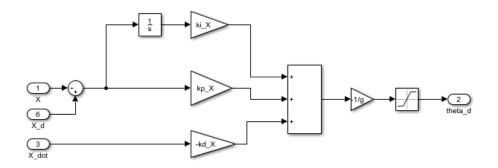


Position Controller (X)

$$\ddot{x}_d = k_p(x_d - x) + k_d(\dot{x}_d - \dot{x}) + k_i \int (x_d - x) dt$$

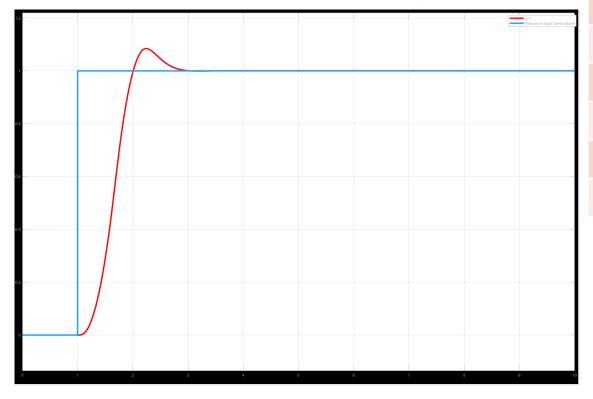


Parameter	Description	Value
k_p	Proportional gain	19.5
k _i	Integral gain	0
k_d	Derivative gain	5.7
t _r	Rising time	592.8 ms
t_s	Settling time	1.83 s
%OS	Overshoot	8.42

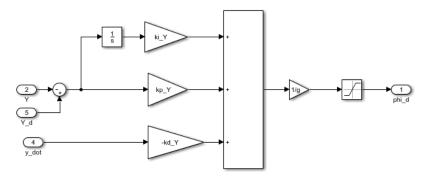


Position Controller (Y)

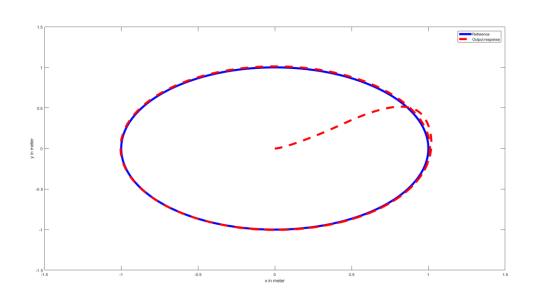
$$\ddot{y}_d = k_p(y_d - y) + k_d(\dot{y}_d - \dot{y}) + k_i \int (y_d - y) dt$$

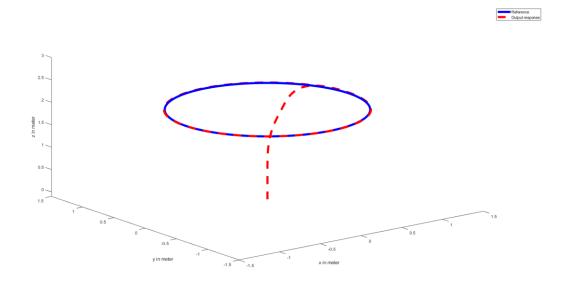


Parameter	Description	Value
k_p	Proportional gain	19.5
k_{i}	Integral gain	0
k_d	Derivative gain	5.7
t_r	Rising time	592.8 ms
t_s	Settling time	1.83 s
%OS	Overshoot	8.42



Simulation of Quadrotor on a Circular Trajectory

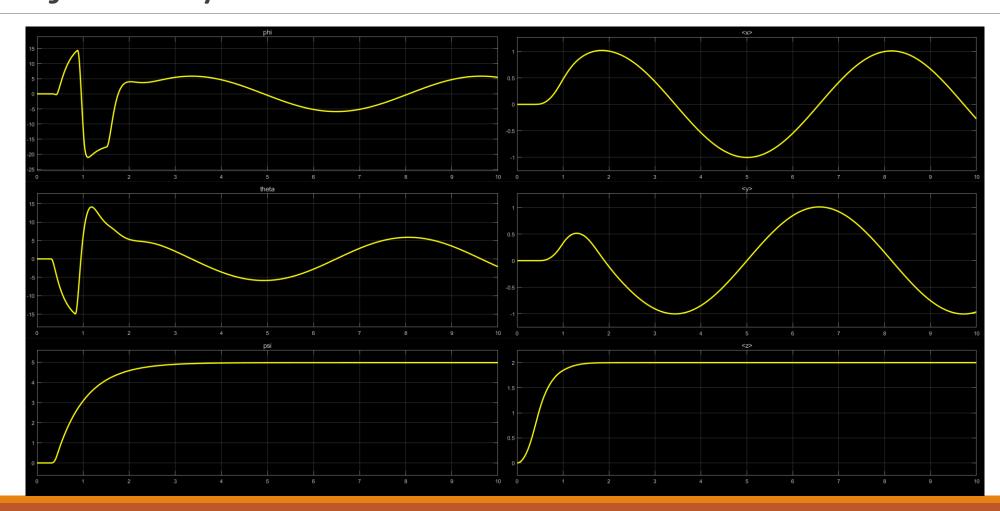




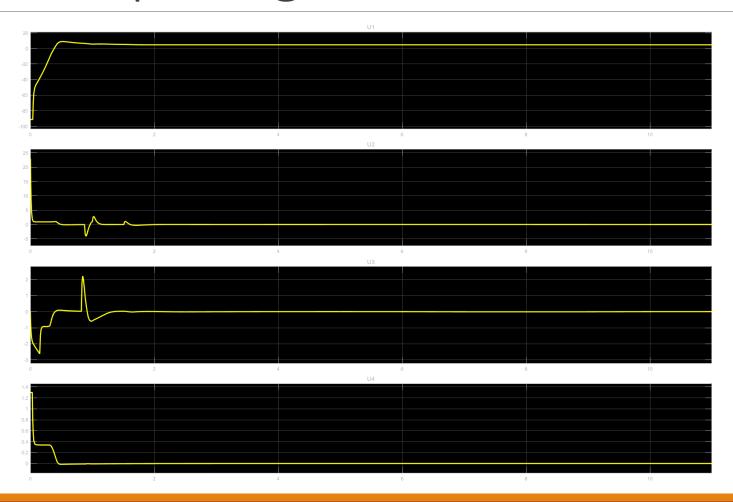
Plot of the Trajectory in 2d

Plot of the Trajectory in 3d

Simulation of Quadrotor on a Circular Trajectory



Control Input Signals



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Tuning of Conventional PID Controller

$$u(t) = K_p \ e(t) + K_i \int_0^t e(x) . \, dx + K_d \frac{de(t)}{dt}$$

formulas based on ultimate gain (K_u) and ultimate period (T_u)

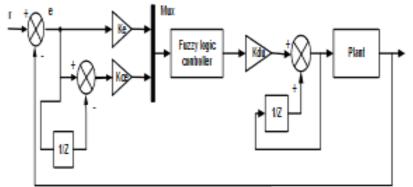
Controller	Gain (K _P)	Integral time(T _i)	Derivative time(T _d)
Р	0.5K _u	-	-
PI	0.45K _u	0.8T _u	-
PID	0.6K _u	0.5T _u	0.125T _u

Principle of Fuzzy Logic Controller (FLC)

FLC implementations require the following:

- 1) Fuzzification
- 2) Knowledge Base
 - a. Data Base
 - b. Rule Base
- 3) Fuzzy inference system
- 4) Defuzzifcation

Design and Tuning Sample



Meaning of the linguistic variables in the fuzzy inference system

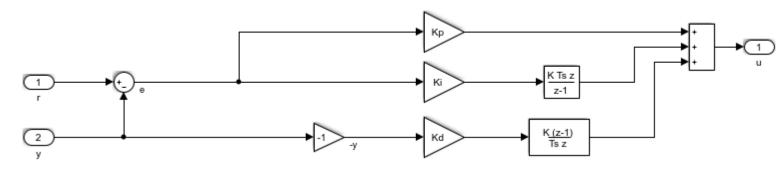
Basic rules table for fuzzy inference system

e	NB	NS	Z	PS	РВ
NB	NVB	NB	NM	NS	Z
NS	NB	NM	NS	Z	PS
Z	NM	NS	Z	PS	PM
PS	NS	Z	PS	PM	PB
PB	Z	PS	PM	PB	PVB

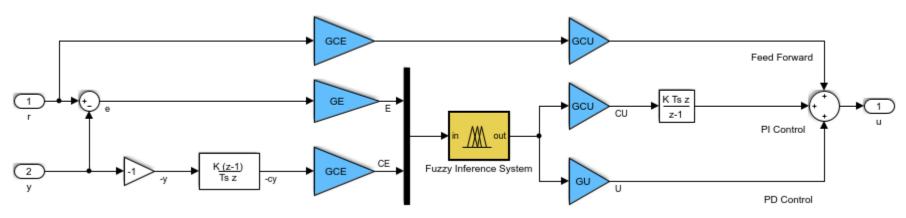
NVB	Negative very big
NB	Negative big
NM	Negative medium
NS	Negative small
Z	Zero
PS	Positive small
PM	Positive medium
PB	Positive big
PVB	Positive very big

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Conventional PID vs Fuzzy PID (Structure)



Conventional PID structure



Fuzzy PID structure

Relation between Conventional PID gain and Fuzzy PID gain

$$GE = \frac{1}{max.error}$$

$$GU = \frac{K_{df}}{GCE}$$

$$GCE = GE * (K_{pf} - \sqrt{K_{pf}^2 - 4K_{if}K_{df}} \frac{K_{if}}{2})$$

$$GCU = \frac{K_{if}}{GE}$$

Where,

GE -> Error Normalization factor

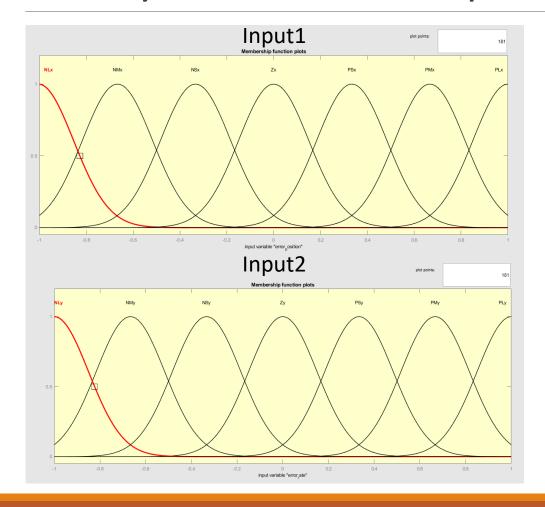
GCE -> Change in measurement normalization factor

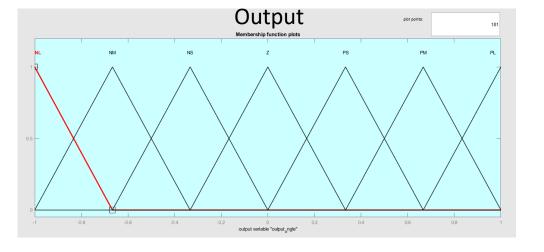
GU -> response de-normalization factor

GCU -> change in the response de-normalization factor

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Fuzzy Membership Function





Fuzzy Inference Rules

In1/In2	NLy	NMy	NSy	Zy	PSy	PMy	PLy
NLx	NL	NL	NL	NL	NM	NS	Z
NMx	NL	NL	NL	NM	NS	Z	PS
NSx	NL	NL	NM	NS	Z	PS	PM
Zx	NL	NM	NS	Z	PS	PM	PL
PSx	NM	NS	Z	PS	PM	PL	PM
PMx	NS	Z	PS	PM	PL	PL	PL
PLx	Z	PS	PM	PL	PL	PL	PL

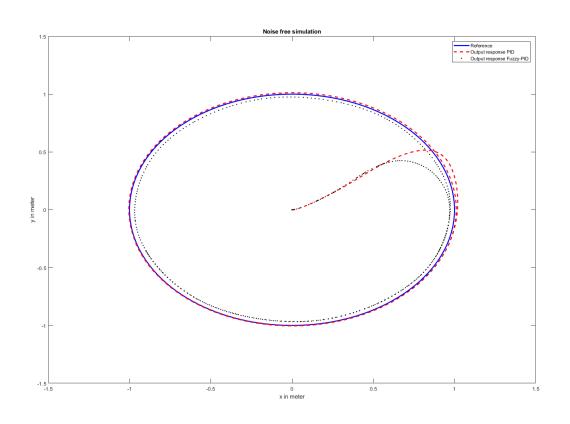
In1 = derivative value/error position; In2 = error rate

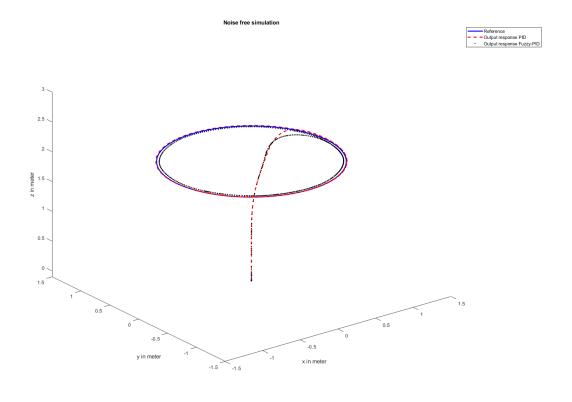
Linguistic Variable	Meaning
NL	Negative Large
NM	Negative Medium
NS	Negative Small
Z	Zero
PS	Positive Small
PM	Positive Medium
PL	Positive Large

Parameters for fuzzy-PID controller

Fuzzy-PID Gain	Value
GE	1.15
GCE	0.45
GU	1.97
GCU	0

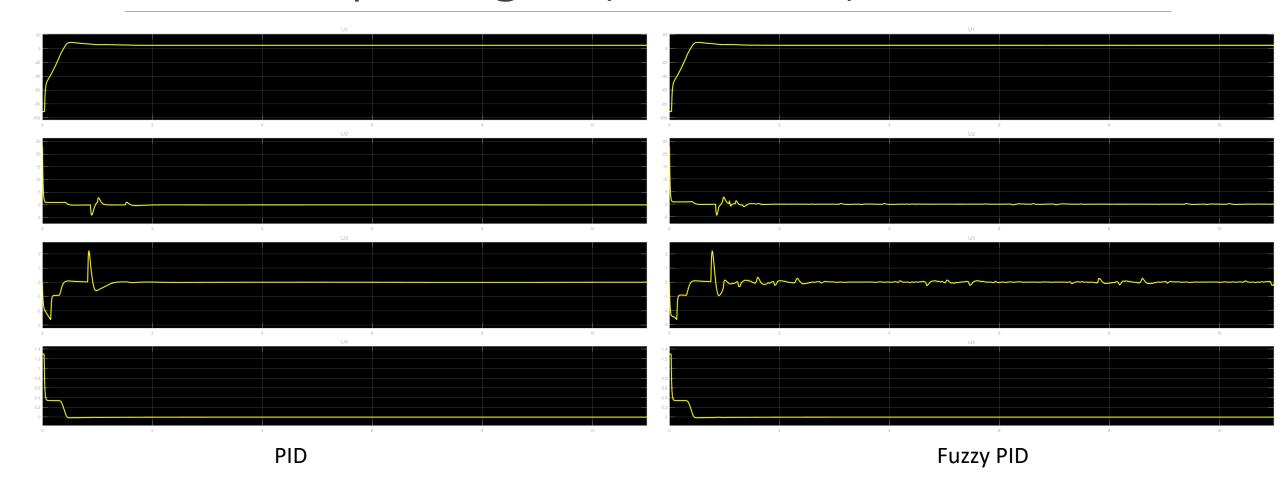
Plot of Trajectory (No Noise)





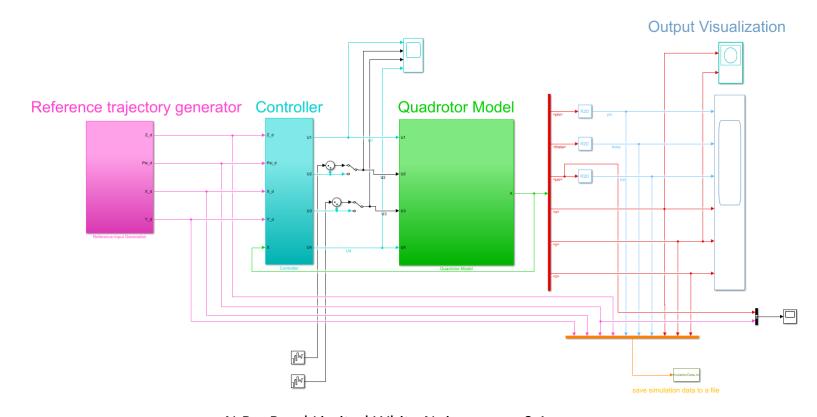
11/21/2017

Plot of Input Signal(No Noise)



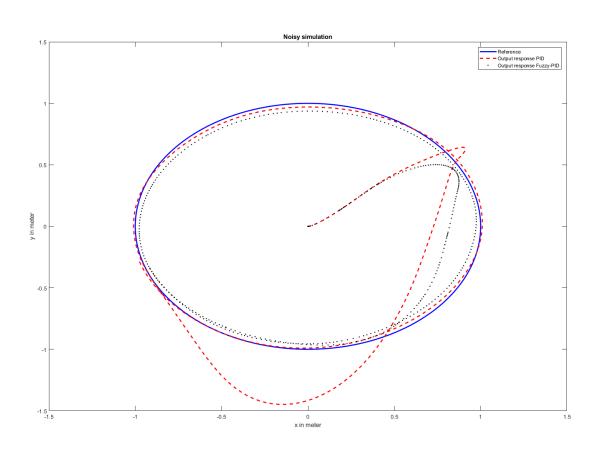
Noisy Simulation Block Diagram

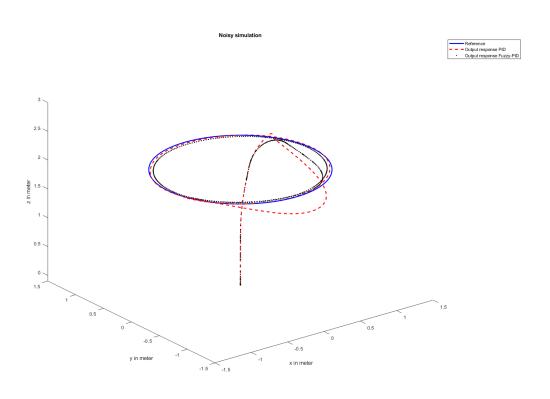
Design of a Fuzzy-PID Controller for Controlling Quadrotor Attitude and Altitude



N.B.: Band Limited White Noise power 0.1

Plot of Trajectory (Noisy)





Plot of Input Signal(Noisy)



Conclusion

- Studied the mathematical modelling of quadcopter
- Developed conventional PID controller for controlling attitude and altitude of a quadcopter
- Simulated the system with conventional PID controller in Simulink
- Developed Fuzzy-PID controller to control the position of the quadcopter
- Simulated the system with Fuzzy-PID controller
- After adding noise, the system is simulated using both PID and fuzzy-PID controller

Remarks:

- With proper tuning of gains in noise free system the performance of conventional PID and Fuzzy PID is same
- But in noisy system Fuzzy PID is better than conventional PID

Thank You

