# Imperial College London

# PLAN OF INVESTIGATION

IMPERIAL COLLEGE LONDON

DEPARTMENT OF EARTH SCIENCE AND ENGINEERING

# Modelling the Effects of Projectile Impact on Laminated Window Glass

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#### 1 Introduction

Deliberate acts of terrorism, criminal activity and malicious behaviour pose significant threat to buildings and infrastructure. A recent state-of the arts review by EU decision makers [1] revealed that current building guidelines tend to not take these threats into account. Appropriate regulations and guidelines require new experimental and numerical testing methods to accurately quantify the resilience of building elements against explosions.

The Applied Modelling and Computation Group (AMCG) at Imperial College London [2] recently developed the novel coupled dynamic gas/solid FEMDEM code Solidity [3], which had successfully been used for geological and biological applications in the past.

The software is expected to realistically simulate shock wave impact on secondary laminated glass. In this paper, the code is provisionally applied to simulate 2D and 3D projectile impact on a single laminate. The shock wave impact is analysed in a future paper.

# 2 Physics Theory

#### 2.1 Laminated Glass Model

Laminated glass is a sandwich structure consisting of two brittle glass plies and an adhered visco-elastic polymer inter-layer (or inter-face) in between. Secondary laminated glass consists of two such laminates, separated by a layer of air.

Advantageous properties of laminated glass include a relatively high penetration resistance [4], low weight [5] and the adherence of fractured glass fragments to the structure to reduce the risk of injuries[4, 6, 7, 8]. Breakage of the inner ply significantly reduces strength and facilitates a full collapse of the glass [7]. An optional back layer (usually poly-carbonate (PC) [9, 10]) improves structural stability and additional energy absorption [9, 11].

The prediction of crack initiation and propagation poses a significant challenge and requires ongoing research effort. Local stress intensification is provided by pre-existing micro-structural material flaws (inhomogeneities or discontinuities) such as micro-cracks and voids [12]. Application of stress higher or equal than the fracture stress,  $\sigma \geq \sigma_{\rm f}$ , causes these flaws to grow in size [7, 13].

Many fracture models have been proposed. This paper uses the combined single and smeared crack model approach [3, 14]. According to the model, increasing stress leads to strain-hardening, followed by strain-softening. Strain-hardening can be modelled by continuum constitutive laws, as fracturing only occurs in the strain-softening part [15]. The constitutive law for the strain-hardening part is given by

$$\tau = c + \sigma \tan \phi \,, \tag{1}$$

with shear stress  $\tau$ , normal stress  $\sigma$ , cohesion c and internal friction angle  $\phi$ . The stresses  $\sigma$  and  $\tau$  correspond to normal and shear displacements  $\delta_{\rm n}$  and  $\delta_{\rm s}$  and tensile and shear strengths  $f_{\rm t}$  and  $f_{\rm s}$ . The material strengths are defined as the maximum strength in the stress-displacement diagram (Fig. 1), with maximum elastic displacement  $\delta_{\rm p}$  and critical displacement  $\delta_{\rm s}$ .

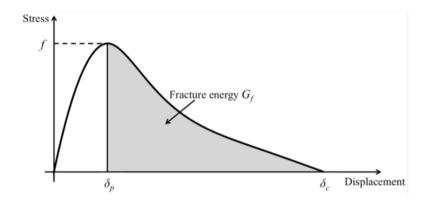


Figure 1: Stress-displacement curve according to the single and smeared crack model. [3]

The fracturing in the strain-softening part is modelled using joint elements which are generated between two neighbouring shell elements. Detailed descriptions are given by Latham et al [3], Munjiza et al [16], Lei et al [17] and Chen and Chang [18].

## 2.2 Inter-layer Model

The task of the inter-layer is the absorption of impact energy and the maintenance of adhesion to the plies [5]. Common inter-layer materials include polymers such as polyvinyl butyral (PVB), thermoplastic polyurethane (TPU), and most recently SentryGlas@Plus (SGP) [19, 20].

Polymers are commonly modelled as hyper-elastic materials [21, 22]. Work done by stresses onto such materials only depends on the reference state X and the current state x, but not on the load path (Fig. 2). Deformation from X to x is described by a deformation gradient [23]

$$F = \frac{\mathrm{d}\varphi}{\mathrm{d}X} \tag{2}$$

with mapping function  $\varphi$  mapping from X to x.

Hyper-elastics are mathematically described by a characteristic strain energy density function W. One of the simplest hyper-elastic models is the Neo-Hookean model [21] whose characteristic function is given by

$$W = \frac{\mu_0}{2} (I_1 - 3) - \mu_0 \ln J + \frac{\lambda_0}{2} \ln^2 J, \qquad (3)$$

with Lamé constants  $\lambda_0$  and  $\mu_0$  from the linearised theory, J=|F| and first invariant  $I_1=C_{\rm H}$  with right Cauchy stress tensor [21]

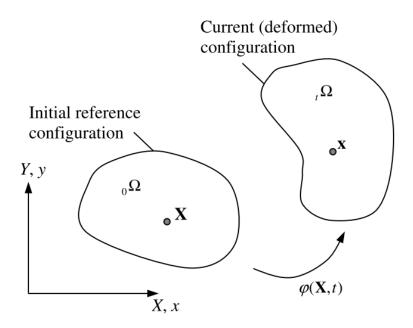


Figure 2: Deformation from reference to current state. [23]

$$C_{\rm ii} = F_{\rm Ii} F_{\rm Ii} \,. \tag{4}$$

Upper case indices refer to the reference configuration, while lower case indices refer to the current configuration.

Another common, simple hyper-elastic model is the Mooney - Rivlin model [24, 25]. The characteristic strain energy function for the compressible 2-Parameter Mooney - Rivlin model [25] is given by

$$W_2 = C_{10} \left( \bar{I}_1 - 1 \right) + C_{01} \left( \bar{I}_2 - 1 \right) + \frac{1}{d} \left( J - 1 \right), \tag{5}$$

where  $C_{10}$  and  $C_{01}$  are adjustable parameters, d=2/K with bulk modulus K and  $\bar{I}_1=J^{-\frac{1}{3}}\,I_1$  and  $\bar{I}_2=J^{-\frac{1}{3}}\,I_2$  are deviatoric invariants [24]. The second invariant is given by

$$I_2 = \frac{1}{2} \left( C_{JJ}^2 - C_{IK} C_{KI} \right) \,. \tag{6}$$

Other hyper-elastic models [24] include a more general polynomial model, Arruda-Boyce, Ogden and Yeoh. For the polynomial model, customized coefficients [26] already exist in the literature, specifically for laminated glass.

Modelling the potential occurrence of fracture needs to be permitted for the inter-layer as well, as fracturing of the inter-layer is possible. This consideration necessitates the extension of the smeared single and combined fracture model to the inter-layer.

# 3 Numerics Theory

#### 3.1 FEMDEM

The combined discrete finite element method FEMDEM [4, 14, 15, 16, 18, 20, 27, 28, 29, 30] is able to realistically model the detection, interaction, deformation and fracturing of matrix bodies. These bodies are modelled as a discrete element, which consists of a cluster of finite elements. Fracturing results in the formation of new discrete elements [15].

Based on previous works by Munjiza et al [14, 16, 27], Xiang et al [31] developed the 3D FEMDEM code Y3D created the model Solidity. This model is capable of created realistic coupled multi-physics simulations. In contrast to conventional models, Solidity is not reliant element deformability restriction constraints (locking) and uses simpler triangular, quadratic and tetrahedral elements [3].

### 3.2 Governing Equations

The governing equations for the finite element calculations in the FEMDEM method are the equations of motion. The equations are given by

$$M\ddot{x} + \mu \dot{x} + f_{\text{int}} = f_{\text{ext}} = f_{\text{l}} + f_{\text{b}} + f_{\text{c}},$$
 (7)

with lumped nodal mass matrix M, nodal displacements x, viscosity  $\mu$ , internal nodal forces  $f_{\rm int}$  and external nodal forces  $f_{\rm ext}$ . External forces contain consist of external loads  $f_{\rm l}$ , bonding forces  $f_{\rm b}$  and contact forces  $f_{\rm c}$ . Internal forces  $f_{\rm int}$  are generated by element deformation. FEMDEM systems solve these equations via explicit time integration using the forward Euler method [17].

## 3.3 Contact Algorithm

The combination of discrete and finite elements is established via an interaction algorithm [17]. This algorithm consists of contact detection [32] and contact interaction [15]. Contact detection is carried out using search algorithms. Upon contact detection, the contact forces between a contractor and target solid are given by

$$f_c = \int_{\Gamma_c} n \left( \varphi_c - \varphi_t \right) d\Gamma_c , \qquad (8)$$

with outward unit normal n to penetration boundary  $\Gamma_c$ , and potential functions  $\varphi_c$  and  $\varphi_t$  for contractor and target. The contractor applies to the target a normal contact force

$$f_n = -n \int_0^{L_p} p\varphi(l) dl, \qquad (9)$$

with penetration length  $L_{\rm p}$ , potential function  $\varphi$  along the target edge and penalty term p. It also applies a tangential friction force

$$f_{\rm t} = \mu_{\rm mob} \|f_{\rm n}\| \frac{v_{\rm r}}{\|v_{\rm r}\|},$$
 (10)

with relative velocity  $v_{\rm r}$  at the Gauss point and mobilised friction coefficient  $\mu_{mob}$  which varies during the shearing process. A more detailed explanation is given by Lei et al [17].

# 4 Numerical Model Approach

#### 4.1 Software Setup

A 2D input Y file is generated using the pre-processor GID [33]. The geometry could also be imported from common CAD software such as AutoCAD.

The input file is compiled using three binary files Yf, m2vtk and m2vtk\_crack, which constitute the Y2D code. The program may be obtained from the AMCG at Imperial College. The high performance computing (HPC) system at Imperial College London is applied to generate time series texttt.vtu files for the simulation.

The series is analysed using the post-processing software Hyperview [34]. For the 3D simulation setup, which is is not considered for now, a similar approach will be applied.

#### 4.2 Model Setup

The input file contains the model including the geometry, constraints, materials, as well as the simulation parameters. The majority of modelling parameters are adduced from the literature. A provisional list of material parameters is given in appendix A. A provisional list of input file simulation parameters is provided in appendix B.



Figure 3: 2D geometry of the laminated glass structure and projectile [18]

Fig. 3 illustrates the preliminary 2D geometric setup. The figure shows the initial state of the spherical projectile (yellow) immediately before impact on the laminated glass (impactor glass plies in red, inner glass ply in blue, inter-layer in green). The glass is fixed via a support system (brown). The support structure is not modelled for now and is replaced with no-velocity boundary conditions acting on the sides of the glass plies.

In real-world applications, the dimensions of the inter-layer are smaller compared to the glass plies. Therefore, a translational degree of freedom in y-direction exists for the inter-layer. This effect will also need to be considered in this paper.

Critical flaws, which cause complete structural failure, are usually found on the cut and machined glass edges [13]. Based on this consideration, the boundary structure layout requires special care. A project task remains to determine a realistic boundary structure to be applied.

Modifications to this preliminary geometric setup will include changes to the shape of the projectile, as well as changes to the thickness of the layers. A potential consideration is to model the projectile as firearm ammunition instead of a circle (or sphere). A symmetry boundary condition may be applicable and only half of the laminate may need to be modeled.

The dimensions of the glass plate is set to  $2000\,\mathrm{mm} \times 2000\,\mathrm{mm}$  in order to avoid anisotropic effects. The thickness of the glass plies is set to  $2\,\mathrm{mm}$  and  $0.76\,\mathrm{mm}$  for the inter-layer. For now, a PVB material is chosen for the inter-layer.

The preliminary mesh for all bodies consists of triangular elements. For the 3D model, tetrahedral elements [18] are to be considered. The preliminary element size is  $1\,\mathrm{mm}$ . Necessary modifications include reducing the number of elements for the projectile, as it is not of interest for the analysis and increasing the element size for far-field mesh elements for the plies and inter-layer. Potentially, differently shaped elements are more optimal for the results. The glass and inter-layers need to consist of several element layers to enable a more precise analysis.

The radius of the steel sphere (or circle) is set to  $2.5 \, \mathrm{mm}$ . The volume of the spherical projectile is given by

$$V = \frac{4\pi}{3}r^3 = \frac{4\pi}{3}(2.5\,\text{mm})^3 \approx 65.4\,\text{mm}^3. \tag{11}$$

The time step [35]1 is set accordingly to

$$\Delta t = \sqrt{\frac{V^2 \,\rho^3}{\text{penalty number}}} = \sqrt{\frac{65.4 \cdot 7.8 \cdot 10^3}{4 \cdot 10^{11}}} \approx 1 \cdot 10^{-6} \text{s}$$
 (12)

To simulate real time  $t = 2 \,\mathrm{s}$ , the maximum number of time steps required [35]<sup>1</sup> is given by

$$n_{\rm t} = \frac{t}{\Delta t} = \frac{2\,\mathrm{s}}{10^{-6}\,\mathrm{s}} = 2\cdot 10^6$$
 (13)

#### 4.3 Verification

Meeting specified accuracy standards [1] requires verification of the numerical model by use of data from numerical experiments. Physical experiments involving the breakage of glass by projectiles or shock blasts require special safety arrangements. Air blast impact

<sup>&</sup>lt;sup>1</sup> for inquiries concerning this reference please contact the AMCG at Imperial College London

 $<sup>^{2}</sup>$ volume  $(in mm^{3})$  of the smallest element, without units

<sup>&</sup>lt;sup>3</sup>density  $(in kg/m^3)$  of the material of the smallest element, without units

experiments are being conducted outdoors by service company Jabisupra [36] in cooperation with Imperial College London. The company is active in the field of envelope security and specialises in protecting infrastructure from certain threats. The data from Jabisupra is not expected to yet be ready and applicable for this project. Many other researchers have already conducted impact experiments in the past and a majority of the findings from these experiments are likely to be applicable to verify the model for this project.

Dynamic impact on laminated glass comprises hard and soft body impact [37]. Hard body impact such as ballistic impact [9] causes minimal deformation to the projectile, while soft body impact such as bird impact [37] causes the projectile to undergo extensive deformation.

Relevant parameters of the impact projectile include the normal velocity [5, 38, 39, 40], the mass [39, 40], the angle [38, 39, 40], the shape [40] and the size [5]. Relevant parameters for the outer glass ply include its dimensions [20], its mass, the support conditions [20] and the make-up [20]. For the inter-layer, the material [10, 19, 20], thickness [8, 20, 39] and temperature [19, 41] are relevant.

Low velocity ( $\approx 20\,\mathrm{m/s}$ ) hard impact experiments include the use of projectiles in form of road construction chippings [38], ballistics [10], drop-down weights [20, 32, 42], aluminum projectiles [42] and steel balls [7, 20, 43]. High velocity (around  $180\,\mathrm{m/s}$ ) soft impact experiments include the use of silicon rubber projectiles [37] and gas guns [19].

Wang et al [20] found that the panel size had an inferior effect on the breakage resistance [20]. Similarly, Monteleone et al [10] found that only a local area of the ply around the impact absorbed the impact energy for high velocities.

Karunarathna [39] found that impact velocity and plate thickness contributed significantly towards the impact resistance, compared to impact mass and inter-layer thickness. Wang et al [20] found an increased inter-layer thickness to have a negative effect on energy absorption. Liu et al [44] established that the inter-layer thickness did not contribute towards energy absorption. Behr and Kremer [43] found an increased inter-layer thickness to better protect the inner ply. Kim et al [45] numerically optimised the PVB inter-layer constitution to prevent all damage to the inner glass ply.

Liu et al [44] numerically investigated the optimisability of the inter-layer in terms of energy absorption by simulating the impact of a human head. Zhang et al [41] investigated the influence of temperature on the inter-layer and found that a hybrid TPU/SGP/TPU inter-layer performed best over the entire range of tested temperatures.

## 5 Hypothesis

The software Solidity is expected to realistically predict and quantify the solid-solid interaction, deformation and fracturing behavior of the laminated window glass upon impact with the projectile. Experimental data from previous research will be adduced to further verify this hypothesis.

## 6 Milestones

#### due 28 June 2019

- 1. Compose Plan of Investigation (01-28 June)
- 2. Review of Current Theory sections 2, 3 (01-21 June)
- 3. Software Setup section 4.1 (21-28 June)
- 4. First Prototype Programme sections 4.2 (21-28 June)
- 5. Review of Experimental Research sections 4.3 (01-21 June)

#### due 30 August 2019

- 6. 2D Simulations (28 June 12 July)
- 7. Validation/Verification of Results (12 July 19 July)
- 8. 3D Simulations (19 July 16 August)
- 9. Validation/Verification of Results (16 August 30 August)
- 10. Outline of Report (28 June 16 August)
- 11. Final Draft of Report (01 June 30 August)

#### due early-mid September

12. Oral presentation

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# **A Material Parameters**

Property	Glass	PVB	Steel Projectile
Density $ ho \ [\mathrm{kg/m^3}]$	2500 [ <mark>46</mark> ]	1100 [ <mark>46</mark> ]	7800 [18]
Young's Modulus $E~[\mathrm{Pa}]$	7e10 [46]	2.2e2 [47]	2e11 [18]
Poisson's Ratio $\nu$	0.22 [46]	0.49 [ <mark>32</mark> ]	0.3 [18]
Mass damping coefficient $\mu$	0	0	0
Elastic penalty term	7e10	2.2e2	2e11
Contact penalty	1.4e11	4.4e2	4e11
Mode I energy rate $G_{ m I}~[{ m J/m^2}]$	10 [ <mark>46</mark> ]	2.8e3 [48]	1.9e5 [49]
Mode II energy rate $G_{ m II}~[{ m J/m^2}]$	50 [ <mark>46</mark> ]		
Mode III energy rate $G_{ m III}~[{ m J/m^2}]$	50 [ <mark>46</mark> ]		
Tensile Strength $\sigma\mathrm{[Pa]}$	6e7 [ <mark>32</mark> ]	2e7 [50]	1e7 [ <mark>5</mark> ]
Internal friction coefficient	0.1 [ <mark>32</mark> ]	0.7 [51]	0.15 [ <mark>52</mark> ]
Internal cohesion $[Pa]$	7e10	2.2e2	2e11
Pore fluid pressure [Pa]	0	0	0
Joint friction coefficient			
Joint roughness coefficient JRC <sub>0</sub> <sup>4</sup>			
Joint compressive strength JCS <sub>0</sub> <sup>4</sup>			
Joint sample size $[\mathrm{m}]$			
Interface friction	0.1 [ <mark>32</mark> ]	0.62 [ <mark>53</mark> ]	0.44 [ <mark>53</mark> ]
2D Problems	plane stress		

 Table 1: List of preliminary Y2 input material parameter values.

<sup>&</sup>lt;sup>4</sup>at laboratory conditions

# **B** Simulation Parameters

Parameter	Value
Maximum number of timesteps	2e6
Current number of timesteps	0
Restart file saving frequency	1e2
Gravity in X Direction (m/s2)	0
Gravity in Y Direction (m/s2)	0
Gravity in Z Direction (m/s2)	-9.8
Timestep (s)	1e-6
Output frequency	10
Current number of iterations	0
Gravity setting stage (s)	0
Load ramping stage (s)	0
Maximum dimension (m)	10
Maximum force (N)	1e6
Maximum velocity (m/s)	100
Maximum stress (Pa)	1e8
Maximum displacement (m)	0.1
Minimum joint aperture (m)	1e-7
Maximum contacting couples	1e7
Accuracy (bit)	32
Joint friction model	Coulomb
Initial aperture correlation	roughness

**Table 2:** List of preliminary Y2D input simulation parameter values.