# Equations Of Motion of Krang on Fixed Wheels

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In this report we attempt to find the dynamic model of Golem Krang with its wheels fixed. So it is reduced to a serial robot with a tree-structure (due to two arms branching out). Figure 1 shows the frames of references we will be using to determine the transforms and the coordinates on the robot. We denote these frames using symbol  $R_i$  where  $i \in \mathbb{F} = \{0, 1, 2, 3, 4l, 5l, 6l, 7l, 8l, 9l, 10l, 4r, 5r, 6r, 7r, 8r, 9r, 10r\}$ .  $R_0$  is the world frame fixed in the middle of the two wheels.  $R_1, R_2, R_3$  are fixed on the base, spine and torso with their rotations represented by  $q_{imu}$ ,  $q_w$  and  $q_{torso}$  respectively. Frames  $R_{4l}, ...R_{10l}$  are frames fixed on the links left 7-DOF arm with their motion represented by  $q_{1l}, ...q_{7l}$ . Similarly, frames  $R_{4r}, ...R_{10r}$  are frames fixed on the links right 7-DOF arm with their motion represented by  $q_{1r}, ...q_{7r}$ . All equations in the following text that do not show r or l in the subscript where they are supposed to, will mean that the respective equations are valid for both subscripts.

We will be using the Kane's formulation. This is done so that our current analysis can easily be merged with the dynamic modelling of wheeled inverted pendulum which is found in terms of quasi-velocities, that prohibit the use of Lagrange for analytical modelling of the robot. Kane's method however is applicable for this problem.

### 1 Introduction to Kane's Formulation

$$\sum_{k} \left[ m_{k} \bar{a}_{Gk} \cdot (\bar{v}_{Gk})_{j} + \left( \frac{d\bar{H}_{Gk}}{dt} \right) \cdot (\bar{\omega}_{k})_{j} \right] = \sum_{n} \bar{F}_{n} \cdot (\bar{v}_{n})_{j} + \sum_{n} \bar{M}_{m} \cdot (\bar{\omega}_{m})_{j} \quad j = 1...K$$

$$\tag{1}$$

where

j is the unique number identifying each generalized co-ordinate in the system

k is the unique number identifying each rigid body in the system

n is the unique number identifying each external force acting on the system

m is the unique number identifying each external torque acting on the system

 $m_k$  is the mass of the kth body

 $\bar{a}_{Gk}$  is the acceleration of the center of mass of kth body

 $\bar{v}_{Gk}$  is the velocity of the center of mass of the kth body

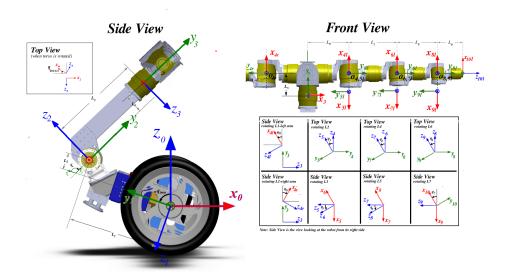


Figure 1: Frames of references on the robot

 $\bar{H}_{Gk}$  is the angular momentum of body k about its center of mass

 $\bar{\omega}_k$  is the angular velocity of the body k

 $F_n$  is the *n*th external force

 $M_m$  is the mth external moment

 $\bar{v}_n$  is the velocity of the point at which external Force  $F_n$  is acting

 $\bar{\omega}_m$  is the angular velocity of the body on which torque is acting relative to the actuator applying the torque

()<sub>j</sub> =  $\frac{\partial()}{\partial \dot{q}_j}$  the partial derivative of the quantity in brackets () with respect to the generalized velocity  $\dot{q}_j$ 

## 2 Transformations

The transformation of frame  $R_i$  into frame  $R_j$  is represented by the homogeneous transformation matrix  ${}^iT_j$  such that.

$${}^{i}T_{j} = \begin{bmatrix} {}^{i}s_{j} & {}^{i}n_{j} & {}^{i}a_{j} & {}^{i}P_{j} \end{bmatrix} = \begin{bmatrix} {}^{i}A_{j} & {}^{i}P_{j} \\ 0 & 0 & 0 & 1 \end{bmatrix} = \begin{bmatrix} s_{x} & n_{x} & a_{x} & P_{x} \\ s_{y} & n_{y} & a_{y} & P_{y} \\ s_{z} & n_{z} & a_{z} & P_{z} \\ 0 & 0 & 0 & 1 \end{bmatrix}$$
 (2)

where  ${}^{i}s_{j}$ ,  ${}^{i}n_{j}$  and  ${}^{i}a_{j}$  contain the components of the unit vectors along the  $x_{j}$ ,  $y_{j}$  and  $z_{j}$  axes respectively expressed in frame  $R_{i}$ , and where  ${}^{i}P_{j}$  is the vector representing the coordinates of the origin of frame  $R_{j}$  expressed in frame  $R_{i}$ .

The transformation matrix  ${}^{i}T_{j}$  can be interpreted as: (a) the transformation from frame  $R_{i}$  to frame  $R_{j}$  and (b) the representation of frame  $R_{j}$  with respect

to frame  $R_i$ . Using figure 1, we can write down these transformation matrices for our system as follows:

$${}^{0}T_{1} = \begin{bmatrix} 0 & sq_{imu} & -cq_{imu} & 0 \\ -1 & 0 & 0 & 0 & 0 \\ 0 & cq_{imu} & sq_{imu} & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}, {}^{1}T_{2} = \begin{bmatrix} 1 & 0 & 0 & 0 & 0 \\ 0 & cq_{w} & sq_{w} & L_{1} \\ 0 & -sq_{w} & cq_{w} & -L_{2} \\ 0 & 0 & 0 & 1 \end{bmatrix}, {}^{2}T_{3} = \begin{bmatrix} -cq_{torso} & 0 & sq_{torso} & 0 \\ 0 & 1 & 0 & L_{3} \\ -sq_{torso} & 0 & -cq_{torso} & L_{4} \\ 0 & 0 & 0 & 1 \end{bmatrix}, {}^{2}T_{3} = \begin{bmatrix} -cq_{torso} & 0 & sq_{torso} & 0 \\ 0 & 1 & 0 & L_{3} \\ -sq_{torso} & 0 & -cq_{torso} & L_{4} \\ 0 & 0 & 0 & 1 \end{bmatrix}, {}^{2}T_{3} = \begin{bmatrix} -cq_{torso} & 0 & sq_{torso} & 0 \\ 0 & 1 & 0 & L_{3} \\ -sq_{torso} & 0 & -cq_{torso} & L_{4} \\ 0 & 0 & 0 & 1 \end{bmatrix}, {}^{2}T_{3} = \begin{bmatrix} -cq_{torso} & 0 & sq_{torso} & 0 \\ 0 & 1 & 0 & L_{3} \\ -sq_{torso} & 0 & -cq_{torso} & L_{4} \\ 0 & 0 & 0 & 1 \end{bmatrix}, {}^{2}T_{3} = \begin{bmatrix} -cq_{torso} & 0 & sq_{torso} & 0 \\ 0 & 1 & 0 & L_{3} \\ -sq_{torso} & 0 & -cq_{torso} & L_{4} \\ 0 & 0 & 0 & 1 \end{bmatrix}, {}^{2}T_{3} = \begin{bmatrix} -cq_{torso} & 0 & sq_{torso} & 0 \\ 0 & 1 & 0 & L_{3} \\ -sq_{torso} & 0 & -cq_{torso} & L_{4} \\ 0 & 0 & 0 & 1 \end{bmatrix}, {}^{2}T_{3} = \begin{bmatrix} -cq_{torso} & 0 & sq_{torso} & 0 \\ 0 & 1 & 0 & L_{3} \\ -sq_{torso} & 0 & -cq_{torso} & L_{4} \\ 0 & 0 & 0 & 1 \end{bmatrix}, {}^{2}T_{3} = \begin{bmatrix} -cq_{torso} & 0 & sq_{torso} & 0 \\ 0 & 1 & 0 & L_{3} \\ -sq_{torso} & 0 & -cq_{torso} & L_{4} \\ 0 & 0 & 0 & 1 \end{bmatrix}, {}^{2}T_{3} = \begin{bmatrix} -cq_{torso} & 0 & sq_{torso} & 0 \\ -cq_{torso} & 0 & -cq_{torso} & 0 \\ -cq_{torso} & 0 & -cq_{torso} & L_{4} \\ 0 & 0 & 0 & 1 \end{bmatrix}, {}^{2}T_{3} = \begin{bmatrix} -cq_{torso} & 0 & sq_{torso} & 0 \\ -cq_{torso} & 0 & cq_{torso} & 0 \\ -cq_{torso} & 0 & -cq_{torso} & 0 \\ -cq_{torso} & 0 & -cq_{torso} & -cq_{torso} & 0 \\ -cq_{torso} & 0 & -cq_{torso} & -cq_{tor$$

$${}^{3}T_{4l} = \begin{bmatrix} 0 & 1 & 0 & L_6 \\ cq_{1l} & 0 & -sq_{1l} & L_5 \\ -sq_{1l} & 0 & -cq_{1l} & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}, {}^{3}T_{4r} = \begin{bmatrix} 0 & -1 & 0 & -L_6 \\ cq_{1r} & 0 & -sq_{1r} & L_5 \\ sq_{1r} & 0 & cq_{1r} & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}, {}^{4}T_5 = \begin{bmatrix} -1 & 0 & 0 & 0 \\ 0 & -cq_2 & -sq_2 & 0 \\ 0 & -sq_2 & cq_2 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix},$$

$${}^{5}T_{6} = \begin{bmatrix} -cq_{3} & 0 & sq_{3} & 0 \\ 0 & -1 & 0 & -L_{7} \\ sq_{3} & 0 & cq_{3} & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}, {}^{6}T_{7} = \begin{bmatrix} -1 & 0 & 0 & 0 \\ 0 & -cq_{4} & -sq_{4} & 0 \\ 0 & -sq_{4} & cq_{4} & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}, {}^{7}T_{8} = \begin{bmatrix} -cq_{5} & 0 & sq_{5} & 0 \\ 0 & -1 & 0 & -L_{8} \\ sq_{5} & 0 & cq_{5} & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix},$$

$${}^8T_9 = \begin{bmatrix} -1 & 0 & 0 & 0 \\ 0 & -cq_6 & -sq_6 & 0 \\ 0 & -sq_6 & cq_6 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}, {}^9T_{10} = \begin{bmatrix} -cq_7 & -sq_7 & 0 & 0 \\ 0 & 0 & -1 & -L_9 \\ sq_7 & -cq_7 & 0 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

## 3 Velocities and Accelerations of Frames

The angular and linear velocities of the frames can be calculated using the recursive formulation:

$${}^{j}\omega_{i} = {}^{j}A_{i}{}^{i}\omega_{i} + \dot{q}_{i}{}^{j}e_{i} \tag{3}$$

$${}^{j}\alpha_{i} = {}^{j}A_{i}{}^{i}\alpha_{i} + \ddot{q}_{i}{}^{j}e_{i} + \dot{q}_{i}{}^{j}({}^{j}\omega_{i} \times {}^{j}e_{i})$$

$$\tag{4}$$

$${}^{j}V_{i} = {}^{j}A_{i} \left( {}^{i}V_{i} + {}^{i}\omega_{i} \times {}^{i}P_{i} \right) \tag{5}$$

$${}^{j}a_{j} = {}^{j}A_{i} \left( {}^{i}a_{i} + {}^{i}\alpha_{i} \times {}^{i}P_{j} + {}^{i}\omega_{i} \times ({}^{i}\omega_{i} \times {}^{i}P_{j}) \right)$$
 (6)

where  ${}^{i}\omega_{j}$ ,  ${}^{i}\alpha_{j}$ ,  ${}^{i}a_{j}$  and  ${}^{i}V_{j}$  denote the angular velocity, linear velocity, angular acceleration and linear acceleration repectively of frame j measured with respect to the world frame and represented in frame i.  ${}^{j}e_{j}$  denotes the direction of local angular velocity of frame j represented in frame j.  $i, j \in \mathbb{F}$  identify the frames and i identifies the antecedent frame of j. So, the rotation  ${}^{j}A_{i}$  and the translation  ${}^{j}P_{i}$  that appear in these equations can not be directly deduced from the transformations listed in the previous section, as the they all represent  ${}^{i}T_{j}$  (note the position of i and j). Rather, we need to use following expressions to deduce our matrices:

$$^{j}A_{i} = {}^{i}A_{j}^{T}$$
 
$$^{j}P_{i} = -{}^{i}A_{j}^{T} \, {}^{i}P_{j}$$

Since frame  $R_0$  is fixed  ${}^0\omega_0$  and  ${}^0V_0$  are both  $\begin{bmatrix} 0 & 0 & 0 \end{bmatrix}^T$ . We can deduce directions of local angular velocities of the frames using figure 1 as follows.

$${}^{1}e_{1} = \begin{bmatrix} -1 & 0 & 0 \end{bmatrix}^{T}, {}^{2}e_{2} = \begin{bmatrix} -1 & 0 & 0 \end{bmatrix}^{T}, {}^{3}e_{3} = \begin{bmatrix} 0 & -1 & 0 \end{bmatrix}^{T}, {}^{4}e_{4} = \begin{bmatrix} 0 & -1 & 0 \end{bmatrix}^{T},$$

$${}^{5}e_{5} = \begin{bmatrix} -1 & 0 & 0 \end{bmatrix}^{T}, {}^{6}e_{6} = \begin{bmatrix} 0 & -1 & 0 \end{bmatrix}^{T}, {}^{7}e_{7} = \begin{bmatrix} -1 & 0 & 0 \end{bmatrix}^{T}, {}^{8}e_{8} = \begin{bmatrix} 0 & -1 & 0 \end{bmatrix}^{T},$$
 
$${}^{9}e_{9} = \begin{bmatrix} -1 & 0 & 0 \end{bmatrix}^{T}, {}^{10}e_{10} = \begin{bmatrix} 0 & 0 & -1 \end{bmatrix}^{T}$$

This information can now be used to derive expressions for the velocities and accelerations of the frames.

## 4 Kane's Left-Hand Side

The inertial forces i.e. the term inside the brackets on the LHS in Kane's formulation can be simplified by expansion and manipulation that results in cancelation of terms leading to a simplified expression. We show the details of this manipulation here. The final expression is the outcome in the end which we will use in our code to find the dynamic model.

$$\bar{v}_{Gk} = \bar{v}_k + \bar{\omega}_k \times \bar{S}_k$$

$$\bar{a}_{Gk} = \bar{a}_k + \bar{a}_k \times \bar{S}_k + \bar{\omega}_k \times (\bar{\omega}_k \times \bar{S}_k)$$

$$(8)$$

$$\bar{H}_{Gk} = \mathbf{J}_{Gk}\bar{\omega}_k$$

$$(9)$$

$$\frac{d\bar{H}_{Gk}}{dt} = \mathbf{J}_{Gk}\bar{\omega}_k + \bar{\omega}_k \times \mathbf{J}_{Gk}\bar{\omega}_k$$

$$= (\mathbf{J}_k + m_k \mathbf{S}_k^{\times} \mathbf{S}_k^{\times})\bar{a}_k + \bar{\omega}_k \times (\mathbf{J}_k + m_k \mathbf{S}_k^{\times} \mathbf{S}_k^{\times})\bar{\omega}_k$$

$$= \mathbf{J}_k\bar{a}_k + \bar{\omega}_k \times \mathbf{J}_k\bar{\omega}_k + m_k \mathbf{S}_k^{\times} \mathbf{S}_k^{\times} \bar{a}_k + \bar{\omega}_k \times (m_k \mathbf{S}_k^{\times} \mathbf{S}_k^{\times} \bar{\omega}_k)$$

$$(10)$$

$$m_k\bar{a}_{Gk} \cdot (\bar{v}_{Gk})_j = m_k\bar{a}_{Gk} \cdot (\bar{v}_k + \bar{\omega}_k \times \bar{S}_k)_j$$

$$= m_k\bar{a}_{Gk} \cdot (\bar{v}_k)_j + m_k\bar{a}_{Gk} \cdot (\bar{\omega}_k \times \bar{S}_k)_j$$

$$= m_k\bar{a}_{Gk} \cdot (\bar{v}_k)_j - m_k (\bar{a}_k + \bar{a}_k \times \bar{S}_k + \bar{\omega}_k \times (\bar{\omega}_k \times \bar{S}_k)) \cdot (\bar{S}_k \times \bar{\omega}_k)_j$$

$$= m_k\bar{a}_{Gk} \cdot (\bar{v}_k)_j - m_k (\bar{a}_k - \mathbf{S}_k^{\times} \bar{\alpha}_k - \bar{\omega}_k \times (\bar{S}_k \times \bar{\omega}_k)) \cdot (\bar{S}_k \times (\bar{\omega}_k)_j)$$

$$= m_k\bar{a}_{Gk} \cdot (\bar{v}_k)_j - m_k (\bar{a}_k - \mathbf{S}_k^{\times} \bar{\alpha}_k - \bar{\omega}_k \times (\bar{S}_k \times \bar{\omega}_k)) \cdot (\bar{S}_k \times (\bar{\omega}_k)_j)$$

$$= m_k\bar{a}_{Gk} \cdot (\bar{v}_k)_j - m_k (\bar{a}_k - \mathbf{S}_k^{\times} \bar{\alpha}_k - \bar{\omega}_k \times (\bar{S}_k \times \bar{\omega}_k)) \cdot (\bar{S}_k \times (\bar{\omega}_k)_j)$$

$$= m_k\bar{a}_{Gk} \cdot (\bar{v}_k)_j - m_k (\bar{S}_k^{\times} \bar{a}_k - \mathbf{S}_k^{\times} \bar{S}_k^{\times} \bar{\omega}_k) \cdot \bar{\mathbf{S}}_k^{\times} \bar{\omega}_k)^T (\bar{\omega}_k)_j$$

$$= m_k\bar{a}_{Gk} \cdot (\bar{v}_k)_j - m_k (\bar{S}_k^{\times} \bar{a}_k - \mathbf{S}_k^{\times} \mathbf{S}_k^{\times} \bar{\omega}_k) \cdot \bar{\mathbf{S}}_k^{\times} \bar{\omega}_k)^T (\bar{\omega}_k)_j$$

$$= m_k\bar{a}_{Gk} \cdot (\bar{v}_k)_j + m_k (\mathbf{S}_k^{\times} \bar{a}_k - \mathbf{S}_k^{\times} \mathbf{S}_k^{\times} \bar{\omega}_k)^T (\bar{\omega}_k)_j$$

$$= m_k\bar{a}_{Gk} \cdot (\bar{v}_k)_j + m_k (\bar{S}_k \times \bar{a}_k - \bar{S}_k \times \bar{S}_k \bar{\omega}_k - \bar{S}_k^{\times} \bar{\omega}_k)^T (\bar{\omega}_k)_j$$

$$= m_k\bar{a}_{Gk} \cdot (\bar{v}_k)_j + m_k (\bar{S}_k \times \bar{a}_k - \bar{S}_k \times \bar{S}_k \bar{\omega}_k - \bar{S}_k \times \bar{\omega}_k) \times \bar{S}_k) + (\bar{S}_k \times \bar{\omega}_k) \times (\bar{S}_k \times \bar{\omega}_k)$$

$$= m_k\bar{a}_{Gk} \cdot (\bar{v}_k)_j + m_k (\bar{S}_k \times \bar{a}_k - \bar{S}_k \times \bar{S}_k \bar{\omega}_k - \bar{\omega}_k \times (\bar{S}_k \times \bar{\omega}_k)) + (\bar{\omega}_k)_j$$

$$= m_k\bar{a}_{Gk} \cdot (\bar{v}_k)_j + (m_k\bar{S}_k \times \bar{a}_k - m_k \mathbf{S}_k^{\times} \mathbf{S}_k^{\times} \bar{\alpha}_k - \bar{\omega}_k \times (m_k \mathbf{S}_k^{\times} \mathbf{S}_k^{\times} \bar{\omega}_k)) + (\bar{\omega}_k)_j$$

$$= m_k\bar{a}_{Gk} \cdot (\bar{v}_k)_j + (m_k\bar{S}_k \times \bar{a}_k - m_k \mathbf{S}_k^{\times} \mathbf{S}_k^{\times} \bar{\alpha}_k - \bar{\omega}_k$$

The terms  $\bar{\omega}_k$ ,  $\bar{\alpha}_k$ ,  $\bar{v}_k$ ,  $\bar{a}_k$  will be found using the recursive formulations in eqs. 3-6. And the term  $\bar{a}_{Gk}$  will be found using eq.8.  $\bar{s}_k = \begin{bmatrix} \mathbf{X}_k & \mathbf{Y}_k & \mathbf{Z}_k \end{bmatrix}^T$  is the center of mass of the joint represented in the local frame. We will use the symbol  $\mathbf{MS}_k$  to represent mass times center of mass  $(m_k \mathbf{S}_k)$  which is the vector the

components of which are  $MS = \begin{bmatrix} MX & MY & MZ \end{bmatrix}^T$ . Finally,  $J_k = \begin{bmatrix} XX_k & XY_k & XZ_k \\ XY_k & YY_k & YZ_k \\ XZ_k & YZ_k & ZZ_k \end{bmatrix}$  is the inertia matrix of the joint represented in the local frame.

## 5 Kane's Right-Hand Side

The right hand side of the Kane's forumulation 1 is the sum of some dot product terms. Each term is either the dot product of:

- force applied on the system  $\bar{F}_n$
- the linear velocity  $\bar{v}_n$  of the point differentiated partially wrt the the unique gerneralized speed  $\dot{q}_j$  corresponding to each equation i.e.  $\frac{\partial \bar{v}_n}{\partial \dot{q}_i}$

or the dot product of:

- torque applied on the system  $\bar{\tau}_n$
- the angular velocity  $\bar{\omega}_n$  of the body differentiated partially wrt the the unique gerneralized speed  $\dot{q}_j$  corresponding to each equation i.e.  $\frac{\partial \bar{\omega}_n}{\partial \dot{q}_i}$

So, in order to analyse the right-hand side of the equation, we need to list down all the forces and torques applied to the system and the points at which they are being applied. They are as follows.

 $\bar{\tau}_j = \tau_j \bar{e}_j$  The torque applied by each joint motor fixed on the antecedent joint moving the current joint. There are 17 such torques as

$$\tau_j \in \{\tau_{imu}, \tau_w, \tau_{torso}, \tau_{1l}, ..., \tau_{7l}, \tau_{1r}, ..., \tau_{7r}\}$$

Note that  $\tau_{imu} = -\tau_R - \tau_L$  where  $\tau_R$  and  $\tau_L$  are torques applied to the right and left wheel respectively, and  $\tau_{imu}$  is the sum of reactions torques experienced by the base in response.

 $-\bar{\tau}_{j} = -\tau_{j}\bar{e}_{j}$  The reaction torque experienced by each antecedent joint.

 $\bar{F}_{qk} = m_k \bar{g}$  is the weight of each joint k

 $\bar{F}_{el}$ ,  $\bar{\tau}_{el}$  Force and torque applied by the environment on the left hand end-effector at point  $E_l$ 

 $\bar{F}_{er}, \bar{\tau}_{er}$  Force and torque applied by the environment on the right hand endeffector at point  $E_r$ 

#### 5.1 Joint Torques

The contribution of motor torque of joint k and its reaction, on the right-hand side of Kane's equation corresponding to generalized speed  $\dot{q}_j$  is:

$$\tau_k \bar{e}_k \cdot \frac{\partial \bar{\omega}_k}{\partial \dot{q}_i} - \tau_k \bar{e}_k \cdot \frac{\partial \bar{\omega}_{a(k)}}{\partial \dot{q}_i} \tag{13}$$

where a(k) = antecedent frame of k. The first term is the contribution of action torques and the second term is the contribution of the reaction torques. Since

$$\bar{\omega}_k = \dot{q}_1 \bar{e}_1 + \dot{q}_2 \bar{e}_1 + \dot{q}_3 \bar{e}_3 + \dots + \dot{q}_k \bar{e}_k$$

$$\frac{\partial \bar{\omega}_k}{\partial \dot{q}_j} = \begin{cases} 0 & \text{if } k < j \\ \bar{e}_j & \text{if } k \ge j \end{cases}$$
(14)

If we take the summation of all joint torque contibutions each described by expression 13 i.e. for k = 1...K, to get get the right-hand side contribution by all joint torques in the kane's equaiton corresponding to a generalized speed j, we will get a very simplified solution

$$\sum_{k=1}^{K} \left( \tau_{k} \bar{e}_{k} \cdot \frac{\partial \bar{\omega}_{k}}{\partial \dot{q}_{j}} - \tau_{k} \bar{e}_{k} \cdot \frac{\partial \bar{\omega}_{k-1}}{\partial \dot{q}_{j}} \right)$$

$$= \sum_{k=1}^{j-1} \left( \tau_{k} \bar{e}_{k} \cdot \frac{\partial \bar{\omega}_{k}}{\partial \dot{q}_{j}} - \tau_{k} \bar{e}_{k} \cdot \frac{\partial \bar{\omega}_{k-1}}{\partial \dot{q}_{j}} \right)$$

$$+ \tau_{j} \bar{e}_{j} \cdot \frac{\partial \bar{\omega}_{j}}{\partial \dot{q}_{j}} - \tau_{j} \bar{e}_{j} \cdot \frac{\partial \bar{\omega}_{j-1}}{\partial \dot{q}_{j}}$$

$$+ \sum_{k=j+1}^{K} \left( \tau_{k} \bar{e}_{k} \cdot \frac{\partial \bar{\omega}_{k}}{\partial \dot{q}_{j}} - \tau_{k} \bar{e}_{k} \cdot \frac{\partial \bar{\omega}_{k-1}}{\partial \dot{q}_{j}} \right)$$

$$= \sum_{k=1}^{j-1} (0-0) + \tau_{j} \bar{e}_{j} \cdot \bar{e}_{j} - 0 + \sum_{k=j+1}^{K} \left( \tau_{k} \bar{e}_{k} \cdot \bar{e}_{j} - \tau_{k} \bar{e}_{k} \cdot \bar{e}_{j} \right)$$

$$= \tau_{j} \tag{15}$$

So the total joint torques contributions on the RHS of each Kane's equation is just the joint torque actuating the generalized speed wrt which the equaiton is being evaluated.

#### 5.2 End-effector Forces and Torques

Let  $\bar{F}_e$  and  $\bar{\tau}_e$  be the force and torque being applied by the environment on the end-effector. The contribution on the RHS of Kane's equaiton corresponding to generalized speed  $\dot{q}_i$  will be:

$$\bar{F}_e \cdot \frac{\partial \bar{v}_e}{\partial \dot{q}_i} + \bar{\tau}_e \cdot \frac{\partial \bar{\omega}_K}{\partial \dot{q}_i} \tag{16}$$

where  $\bar{v}_e$  is the linear velocity of the point E on the end-effector on which the force is being applied. And  $\bar{\omega}_K$  is the angular velocity of the last joint on which the end-effector is mounted.

$$\bar{v}_{e} = \bar{v}_{K} + \bar{\omega}_{K} \times \bar{r}_{e/O_{K}} \\
= \bar{v}_{K-1} + \bar{\omega}_{K-1} \times \bar{r}_{O_{K}/O_{K-1}} + \bar{\omega}_{K} \times \bar{r}_{E/O_{K}} \\
... \\
= \bar{v}_{0} + \bar{\omega}_{0} \times \bar{r}_{O_{1}/O_{0}} + \bar{\omega}_{1} \times \bar{r}_{O_{2}/O_{1}} + \bar{\omega}_{2} \times \bar{r}_{O_{3}/O_{2}} + ... + \bar{\omega}_{K-1} \times \bar{r}_{O_{K}/O_{K-1}} + \bar{\omega}_{K} \times \bar{r}_{O_{K+1}/O_{K}} \\
= \bar{v}_{0} + \sum_{k=0}^{K} \left( \bar{\omega}_{k} \times \bar{r}_{O_{k+1}/O_{k}} \right) \tag{17}$$

where we have replace E with  $O_{K+1}$  for the convenience of writing the closedform expression. When partially differentiated wrt  $\dot{q}_j$  we can use eq. 14 to get:

$$\frac{\partial \bar{v}_{e}}{\partial \dot{q}_{j}} = \frac{\partial \bar{v}_{0}}{\partial \dot{q}_{j}} + \sum_{k=0}^{K} \left( \frac{\partial \bar{\omega}_{k}}{\partial \dot{q}_{j}} \times \bar{r}_{O_{k+1}/O_{k}} \right)$$

$$= 0 + \sum_{k=0}^{j-1} \left( \frac{\partial \bar{\omega}_{k}}{\partial \dot{q}_{j}} \times \bar{r}_{O_{k+1}/O_{k}} \right) + \sum_{k=j}^{K} \left( \frac{\partial \bar{\omega}_{k}}{\partial \dot{q}_{j}} \times \bar{r}_{O_{k+1}/O_{k}} \right)$$

$$= \sum_{k=0}^{j-1} \left( 0 \times \bar{r}_{O_{k+1}/O_{k}} \right) + \sum_{k=j}^{K} \left( \bar{e}_{j} \times \bar{r}_{O_{k+1}/O_{k}} \right)$$

$$= \bar{e}_{j} \times \sum_{k=j}^{K} \bar{r}_{O_{k+1}/O_{k}}$$

$$= \bar{e}_{j} \times \left( \bar{r}_{O_{j+1}/O_{j}} + \bar{r}_{O_{j+2}/O_{j+1}} + \bar{r}_{O_{j+3}/O_{j+2}} + \dots + \bar{r}_{O_{K+1}/O_{K}} \right)$$

$$= \bar{e}_{j} \times \bar{r}_{O_{K+1}/O_{j}}$$

$$= \bar{e}_{j} \times \bar{r}_{E/O_{j}}$$
(18)

Substituting eqs. 18 and 14 in expression 16, we get

$$\begin{split} \bar{F}_e \cdot \bar{e}_j \times \bar{r}_{E/O_j} + \bar{\tau}_e \cdot \bar{e}_j \\ &= \begin{bmatrix} \left[ \bar{e}_j \times \bar{r}_{E/O_j} \right]^T & \bar{e}_j^T \end{bmatrix} \begin{bmatrix} \bar{F}_e \\ \bar{\tau}_e \end{bmatrix} \end{split}$$

If we write all kane's equations in the form of a vector then the right hand side contribution of end-effector force and torque will become,

$$\begin{bmatrix} \left[\bar{e}_{1} \times \bar{r}_{E/O_{1}}\right]^{T} & \bar{e}_{1}^{T} \\ \left[\bar{e}_{2} \times \bar{r}_{E/O_{2}}\right]^{T} & \bar{e}_{1}^{T} \\ \left[\bar{e}_{3} \times \bar{r}_{E/O_{3}}\right]^{T} & \bar{e}_{3}^{T} \\ & \cdots \\ \left[\bar{e}_{K} \times \bar{r}_{E/O_{K}}\right]^{T} & \bar{e}_{K}^{T} \end{bmatrix} \begin{bmatrix} \bar{F}_{e} \\ \bar{\tau}_{e} \end{bmatrix}$$

$$= \mathbb{J}^{T} \mathbf{f}_{e} \tag{19}$$

where

$$\begin{split} \mathbb{J} &= \begin{bmatrix} \bar{e}_1 \times \bar{r}_{E/O_1} & \dots & \bar{e}_K \times \bar{r}_{E/O_K} \\ \bar{e}_1 & \dots & \bar{e}_K \end{bmatrix} \\ \mathbb{f} &= \begin{bmatrix} \bar{F}_e \\ \bar{\tau}_e \end{bmatrix} \end{split}$$

The matrix  $\mathbb{J}$  is referred to as the Jacobian. And  $\mathbb{f}$  is the wrench (formal term to refer to a force/torque couple). This whole theory was assuming a single serial chain of the robot with a single end-effector. For the case of krang, we will have two wrenches  $\mathbb{f}_{el}$  and  $\mathbb{f}_{er}$  applied at two end-effectors on the right and the left arms respectively. The points on which this wrench is being sensed on the two end-effectors is  $E_L$  and  $E_R$ . So the contribution becomes:

$$\mathbb{J}_L^T \mathbb{f}_{el} + \mathbb{J}_R^T \mathbb{f}_{er} \tag{20}$$

where

$$\bullet \ \, \mathbb{J}_L = \begin{bmatrix} e_1 \times \bar{r}_{E_L/O_1} & e_2 \times \bar{r}_{E_L/O_2} & e_3 \times \bar{r}_{E_L/O_3} & e_{4l} \times \bar{r}_{E_L/O_{4l}} & \dots & e_{10l} \times \bar{r}_{E_L/O_{10l}} & O_{3 \times 7} \\ e_1 & e_2 & e_3 & e_{4l} & \dots & e_{10l} & O_{3 \times 7} \end{bmatrix}$$

$$\bullet \ \, \mathbb{J}_{10r} = \begin{bmatrix} e_1 \times \bar{r}_{E_R/O_1} & e_2 \times \bar{r}_{E_R/O_2} & e_3 \times \bar{r}_{E_R/O_3} & O_{3\times7} & e_{4r} \times \bar{r}_{E_R/O_4r} & \dots & e_{10r} \times \bar{r}_{E_R/O_{10r}} \\ e_1 & e_2 & e_3 & O_{3\times7} & e_{4r} & \dots & e_{10r} \\ \end{bmatrix}$$

#### 5.3 Gravitational Forces

The contribution of the weight of a joint k to the right-hand side of the equation corresponding to generalized speed  $\dot{q}_j$  is

$$m_k \bar{g} \cdot \frac{\partial \bar{v}_{Gk}}{\partial \dot{q}_j} \tag{21}$$

Now

$$\bar{v}_{Gk} = \bar{v}_k + \bar{\omega}_k \times \bar{r}_{Gk/O_k} 
= \bar{v}_{k-1} + \bar{\omega}_{k-1} \times r_{O_k/O_{k-1}} + \bar{\omega}_k \times \bar{r}_{Gk/O_k} 
= \bar{v}_0 + \sum_{i=1}^k \left( \bar{\omega}_{i-1} \times r_{O_i/O_{i-1}} \right) + \bar{\omega}_k \times \bar{r}_{Gk/O_k} 
\frac{\partial \bar{v}_{Gk}}{\partial \dot{q}_j} = \frac{\partial \bar{v}_0}{\partial \dot{q}_j} + \sum_{i=1}^k \left( \frac{\partial \bar{\omega}_{i-1}}{\partial \dot{q}_j} \times r_{O_i/O_{i-1}} \right) + \frac{\partial \bar{\omega}_k}{\partial \dot{q}_j} \times \bar{r}_{Gk/O_k} 
= 0 + \sum_{i=1}^k \left( \frac{\partial \bar{\omega}_{i-1}}{\partial \dot{q}_j} \times r_{O_i/O_{i-1}} \right) + \frac{\partial \bar{\omega}_k}{\partial \dot{q}_j} \times \bar{r}_{Gk/O_k} 
= \begin{cases} 0 & \text{if } k < j \\ \sum_{i=j+1}^k \left( \bar{e}_j \times r_{O_i/O_{i-1}} \right) + \bar{e}_j \times \bar{r}_{Gk/O_k} & \text{if } k \ge j \end{cases} 
= \begin{cases} 0 & \text{if } k < j \\ \bar{e}_j \times \left( \sum_{i=j+1}^k \bar{r}_{O_i/O_{i-1}} + \bar{r}_{Gk/O_k} \right) & \text{if } k \ge j \end{cases} 
= \begin{cases} 0 & \text{if } k < j \\ \bar{e}_j \times \bar{r}_{Gk/O_i} & \text{if } k \ge j \end{cases}$$

$$(22)$$

The total contributions of all joints will therefore be:

$$\sum_{k=1}^{K} \left( m_k \bar{g} \cdot \frac{\partial \bar{v}_{Gk}}{\partial \dot{q}_j} \right)$$

$$= \bar{g} \cdot \left( \sum_{k=1}^{j-1} m_k \frac{\partial \bar{v}_{Gk}}{\partial \dot{q}_j} + \sum_{k=j}^{K} m_k \frac{\partial \bar{v}_{Gk}}{\partial \dot{q}_j} \right)$$

$$= \bar{g} \cdot \left( \sum_{k=1}^{j-1} 0 + \sum_{k=j}^{K} m_k \left( e_j \times \bar{r}_{Gk/O_j} \right) \right)$$

$$= \bar{g} \cdot \left( e_j \times \sum_{k=j}^{K} m_k \bar{r}_{Gk/O_j} \right)$$
(23)

Note that this derivation is assuming a single serial chain. In the case of Krang however, if j corresponds to a speed of joint before the branching takes place i.e. if  $j \in \{imu, w, torso\}$  the range of summation will include all joints in the tree above the current joint. If j corresponds to the speed of joint in one of the branches i.e. if  $j \in \{1l, ..., 7l\}$  or  $j \in \{1r, ..., 7r\}$  then the range of summation will only be consisting of joints following the current joint in the specific branch.

## 6 MATLAB code For Finding the Dynamic Model

The dynamic model is generated using a script <code>dynamicModel.m</code> found in the folder <code>stableForceInteraction/Implementation/1-ForceControlWhileBalancing/1-ControlProblem1/1-DynamicModeling/2-DynamicModelOfTreeStructuredRobot/2-matlab/Kanes</code>. The function populates the frame information in a map container using <code>getKrangFrames()</code>, then calculates the left-hand side of the Kane's terms using the function <code>kanesLHS()</code> and finally constructs the <code>A</code> and <code>C</code> matrices in the dynamic model using the function <code>getAC()</code>. These functions and the data structures are briefly discussed in the following subsections.

#### 6.1 Map Container for all the Frame Information

The function getKrangFrames() populates the information in a map container f. A map container is a data structure in MATLAB that stores a list of data that is retrievable using a key. We store a frame structure in each cell of the map and use strings  $s \in S = \{ \ '0', \ '1', \ '2', \ '3', \ '41', \ '51', \ '61', \ '71', \ '81', \ '91', \ '101', \ '4r', \ '5r', \ '6r', \ '7r', \ '8r', \ '9r', \ '10r' \}$  as a key to retrieve information. The frame structure stores the following elements:

- $\mathbf{x}$  the unit vector along x-axis represented in the antecedent frame
- y the unit vector along y-axis represented in the antecedent frame
- z the unit vector along z-axis represented in the antecedent frame
- P the position of the origin frame represented in the antecedent frame
- e the unit vector along direction of positive rotation of the frame represented in the local frame
- a the string  $\in \mathbf{S}$  (defined above) that is the key that maps to the antecedent frame
- ${\tt q}$  the symbolic variable used for representing the generalized position q associated with this frame
- dq the symbolic variable used for representing the generalized speed  $\dot{q}$  associated with this frame
- ddq the symbolic variable used for representing the generalized acceleration  $\ddot{q}$  associated with this frame
  - o the row number in the inertia matrix  ${\bf A}$  (i.e. in the final dynamic model  ${\bf A}\ddot{{\bf q}}+{\bf C}\dot{{\bf q}}+{\bf Q}={\bf F})$  that corresponds to the current joint

param array of ten symbolic variables used to represent the inertial parameters of the joint [m] MX MY MZ XX YY ZZ XY XZ YZ $]^T$ 

#### 6.2 Functions associated with the Map Container

There are a number of functions that take the map container **f** as the input argument and construct a useful information as an output. Here is a list of those functions:

- isBefore(f, key1, key2) returns true if frame 1 (identified by key1) is before frame 2 (identified by key2) in the tree structure of the robot
  - Rot(f, key1, key2) returns rotation transform  ${}^{j}A_{i}$  i.e. represention of frame i (identified by key1) in frame j (identified by key2)
  - Tf(f, key1, key2) returns rotation transform  ${}^{j}T_{i}$  i.e. represention of frame i (identified by key1) in frame j (identified by key2)
    - qVec(f) generates the vector q containing generalized positions of all frames
    - dqVec(f) generates the vector  $\dot{\mathbf{q}}$  containing generalized speeds of all frames
    - ddqVec(f) generates the vector  $\ddot{\mathbf{q}}$  containing generalized accelerations of all frames
    - mass(f, key) returns the mass of the frame identified by the key
    - mcom(f, key) returns the mass times COM ( $MS = [MX \ MY \ MZ]^T$ ) of the joint identified by the key represented in the local frame
  - $\begin{array}{ll} \text{inertiaMat(f, key)} & \text{returns the inertia matrix } \left( \mathbf{J} = \begin{bmatrix} \mathbf{X}\mathbf{X} & \mathbf{X}\mathbf{Y} & \mathbf{X}\mathbf{Z} \\ \mathbf{X}\mathbf{Y} & \mathbf{Y}\mathbf{Y} & \mathbf{Y}\mathbf{Z} \\ \mathbf{X}\mathbf{Z} & \mathbf{Y}\mathbf{Z} & \mathbf{Z}\mathbf{Z} \end{bmatrix} \right) \text{ of the joint indentified by key represented in the local frame} \end{array}$ 
    - angVel(f, key) returns the symbolic expression for the angular velocity  $(\bar{\omega})$  of the joint represented in local frame calculated recursively using eq. 3
    - angAcc(f, key) returns the symbolic expression for the angular acceleration  $(\bar{\alpha})$  of the joint represented in local frame calculated recursively using eq. 4
    - linVel(f, key) returns the symbolic expression for the linear velocity  $(\bar{v})$  of the joint represented in local frame calculated recursively using eq. 5
    - linAcc(f, key) returns the symbolic expression for the linear acceleration ( $\bar{a}$ ) of the joint represented in local frame calculated recursively using eq. 6

#### 6.3 Kane's Left-Hand Side

Kane's formulation (eq.1) gives a number of equations, one for each generalized speed  $\dot{q}_j$ . The left-hand side of each equation is a summation of a number of terms one for each joint k. Each term is calculated using eq.12 which gives a simplified version of what occured in the brackets on which the summation is operating on the left-hand side of the original Kane's equation.

The function kanesLHS() is written to calculate the symbolic expression for the left hand side of each equation. The function iterates through each joint k using the keys to the map container. For each joint it:

• evaluates the symbolic expressions for the respective joint velocities and accelerations  $(\bar{\omega}_k, \bar{\alpha}_k, \bar{v}_k \text{ and } \bar{a}_k)$  using the helper functions described in the previous subsection

- evaluates for this joint k the symbolic expression for the term in the brackets, on which the summation is operating on the left-hand side of the original Kane's equation, using eq.12. A term for each equation with respect to each generalized speed  $\dot{q}_j$  (i.e. a sub-loop iterating through all generalized speeds). This gives the contribution of the inertial forces of joint k to the left-hand side of each equation in the Kane's formulation.
- adds the final expression to an accumulator (to apply the summation)

A vector of size the same as the number of Kane's eqations is thus generated each element of which corresponds to a unique generalized speed  $q_j$  and represents the sum of all joint contributions with respect to the unique joint speed corresponding to this element. So each element represents the left-hand side of the each equation in the Kane's formulation.

Finally, once all the symbolic expressions are generated using kanesLHS(), we collect the coefficients of each  $\ddot{q}_j$  and  $\dot{q}_j$  in each equation to generate our **A** and **C** matrices in the final dynamic equation  $\mathbf{A}\ddot{\mathbf{q}} + \mathbf{C}\dot{\mathbf{q}} + \mathbf{Q} = \mathbf{F}$ . These matrices are saved in the symbolic variable matrices **AA** and **CC**. These variables can be loaded into the workspace by loading the file kanesLHS.mat.

#### 6.4 Kane's Right-Hand Side

The evaluation of right-hand is written in a separate script kanesRHS.m. It uses the original closed form expressions for the evaluation of the forces i.e. eqs. 13, 16 and 21. The derivation of the simplified expressions presented in eqs. 15, 19 and 23 could also be used but we did not use them. We can write a separate code later to verify the results of the former by comparing it with the result of derived expressions. We have not done that as yet.

The result of the right-hand side evaluation is stored in the symbolic variable vectors a, b1, b2, b3, b4 and c containing contributions of  $\bar{\tau}_j$ ,  $\bar{F}_{el}$ ,  $\bar{\tau}_{el}$ ,  $\bar{F}_{er}$ ,  $\bar{\tau}_{er}$  and  $\bar{F}_{qk}$  respectively. The code can be found in the appendix.

# 7 Appendix

We present the code listings of the various functions in this section.

#### 7.1 dynamicModel.m

```
f = getKrangFrames(17);
[KK, QQ, KHist] = kanesLHS(f);
[AA, CC] = getAC(f, KK);
```

#### 7.2 getKrangFrames()

```
function f = getKrangFrames(nFrames)

| % This function generates a map. The keys to the maps are string literals
| % in ('1', '2', '3', '4!', '5!', ...'10!', '5!', ...'10!', '5
| % and the values are structs that have members associated with the
| % respective joints of the robot.
| % 'x', 'y', 'z', 'P' define the unit vectors and position of origin of the
| % frame represented in the artecephs frame
| % 'e' is the local angular speed of the frame represented in the same frame
```

```
1%'d contains the key to the antecedent frame
                    * a what delegate the respective link

* 'param contains the irertial parameters of the respective link

* 'q', 'dd and 'ddf contain associated joint pos vel and accvariables

* 'd' defines the row of inertia matrix A (in the dynamic equation) that
                      % corresponds to the aurrent joint.
                  syms q_imu q_w q_torso q_il q_2l q_3l q_4l q_5l q_6l q_7l real
syms q_imu q_w q_torso q_il q_2l q_3l q_4l q_5l q_6l q_7l real
syms q_imu d_w q_torso q_il d_2l d_3l d_4l d_5l d_6l d_7l real
syms dq_imu dq_w dq_torso dq_il d_2l dq_3l dq_4l d_5l d_6l dq_fl dq_7l real
syms dq_imu dq_w dq_torso dq_il dq_2l dq_3l dq_4l d_5l dq_6l dq_fl dq_5l dq_6l
syms dq_imu ddq_w ddq_torso ddq_il dd_2l ddq_3l ddq_4l ddq_5l ddq_5l ddq_fl ddq_5l ddq_fl
syms ddq_imu ddq_w ddq_torso ddq_il ddq_2l ddq_3l ddq_4l ddq_5l ddq_5l ddq_fl d
15
30
                      syms m_10r MX_10r MY_10r MZ_10r XX_10r XY_10r XZ_10r YY_10r YZ_10r ZZ_10r real
                    nullSym = sym([0; 0; 0]); nullSymParam = sym(zeros(10,1));
frame.x = nullSym; frame.y = nullSym; frame.z = nullSym; frame.P = nullSym;
frame.e = nullSym; frame.a = ''; frame.param = nullSymParam;
frame.q = sym(0); frame.dq = sym(0); frame.o = 0;
frame.angVel = sym(0); frame.linVel = sym(0);
frame.angAcc = sym(0); frame.linVel = sym(0);
frame.angAcc = false; frame.gotLinVel = false;
frame.gotAngAcc = false; frame.gotLinAcc = false;
45
                    \begin{split} & \text{frame.x} = \text{sym}([1;\ 0;\ 0]);\ \text{frame.y} = [0;\ \cos(q\_w);\ -\sin(q\_w)];\\ & \text{frame.z} = [0;\ \sin(q\_w);\ \cos(q\_w)];\ \text{frame.P} = [0;\ L1;\ -L2];\\ & \text{frame.e} = [-1;\ 0;\ 0];\ \text{frame.a} = '1';\\ & \text{frame.q} = q\_w;\ \text{frame.dq} = dq\_w;\ \text{frame.ddq} = ddq\_w;\ \text{frame.o} = 2;\\ & \text{frame.param} = [m\_2\ MX\_2\ MY\_2\ MZ\_2\ XX\_2\ XY\_2\ XZ\_2\ YY\_2\ YZ\_2\ ZZ\_2];\\ & & \text{ff}(Z') = \underbrace{\text{frame}}_{F\{3\}} = \text{frame}; \end{split}
70
                    80
                    frame.x = [0; cos(q_11); -sin(q_11)]; frame.y = sym([1; 0; 0]); frame.z = [0; -sin(q_11); -cos(q_11)]; frame.P = sym([L6; L5; 0]); frame.e = [0; -1; 0]; frame.a = '3'; frame.q = q_11; frame.q = q_11; frame.dq = ddq_11; frame.o = 4; frame.param = [m_41 MX_41 MY_41 MZ_41 XX_41 XY_41 XZ_41 YY_41 YZ_41 ZZ_41]; % ff \( \frac{T}{2} \) = frame; F(5) = frame;
                    frame.x = sym([-1; 0; 0]); frame.y = [0; -cos(q_41); -sin(q_41)];
```

```
frame.z = [0; -sin(q_41); cos(q_41)]; frame.P = sym([0; 0; 0]); frame.e = [-1; 0; 0]; frame.a = '61'; frame.o = 7; frame.q = q_41; frame.dq = dq_41; frame.ddq = ddq_41; frame.prame = [m_71 MX_71 MY_71 MZ_71 XX_71 XY_71 XZ_71 YY_71 YZ_71 ZZ_71]; % f('71') = frame; F{8} = frame;
110
       115
120
       frame.x = [-cos(q_71); 0; sin(q_71)]; frame.y = [-sin(q_71); 0; -cos(q_71)]; frame.z = sym([0; -1; 0]); frame.P = [0; -1.9; 0]; frame.e = [0; 0; -1]; frame.a = '91'; frame.q = q_71; frame.dq = dq_71; frame.dq = dq_71; frame.dq = dq_71; frame.param = [m_101 MX_101 MY_101 MZ_101 XX_101 XY_101 XZ_101 YY_101 YZ_101 ZZ_101]; % ff(101') = frame;
130
135
       F{11} = frame;
       140
       145
       155
       F{14} = frame:
160
       170
175
       frame.x = [-cos(q_7r); 0; sin(q_7r)]; frame.y = [-sin(q_7r); 0; -cos(q_7r)]; frame.z = sym([0; -1; 0]); frame.P = [0; -19; 0]; frame.e = [0; 0; -1]; frame.a = '9r'; frame.q = q_7r; frame.dq = dq_7r; frame.ddq = ddq_7r; frame.o = 17; frame.param = [m_10r MX_10r MY_10r MZ_10r XX_10r XY_10r XZ_10r YY_10r YZ_10r ZZ_10r]; % f('10r') = frame; F(18) = frame;
185
190
       keys = {'0', '1', '2', '3', '41', '51', '61', '71', '81', ...
'91', '101', '4r', '5r', '6r', '7r', '8r', '9r', '10r'};
195
       for i=1:nFrames+1: kevSet{i} = kevs{i}: frames{i} = F{i}: end
       f = containers.Map( keySet, frames );
```

#### 7.3 kanesLHS()

```
function [K, Q, KHist] = kanesLHS(f, varargin)
              if nargin == 1
   w0 = sym([0 0 0]');
   v0 = sym([0 0 0]');
   alpha0 = sym([0 0 0]');
   a0 = sym([0 0 0]');
10
                       8
w0 = varargin{1}(:,1);
v0 = varargin{1}(:,2);
alpha0 = varargin{1}(:,3);
a0 = varargin{1}(:,4);
15
              dq=dqVec(f);
key = keys(f);
K=sym(zeros(length(dq),1));
KHist=sym(zeros(length(dq),length(dq)));
Q=sym(zeros(length(dq),1));
for i=1:length(key)
20
                         % Donothing for frame 0 if (isequal(key{i}, '0')); continue; end
25
                         %Kinematics and Inertials Params
                         %AUTHOLICS and IPETLIAIS variatis
w=angwel(f, key(i), w0);
alpha=angAcc(f, key(i), [w0 alpha0]);
v=linvel(f, key(i), [w0 v0]);
a=linAcc(f, key(i), [w0 alpha0 v0 a0]);
J=inertiaMat(f, key(i));
m=mass(f, key(i));
mS=mCOM(f, key(i));
                         S=mS/m;
35
                        % Inertial Forces and Torques
vG = v + cross(w, S);
maG = m*(a + cross(alpha, S) + cross(w, cross(w, S)));
dHnew = cross(mS, a) + J*alpha + cross(w, J**);
40
                         % Kanes LHS contributions
                         or j=1:length(K)
    disp(['kanesF: i=', num2str(i), ', j=', num2str(j), ', key=', key{i}]);
    KHist(j,f(key{i}).o) = maG'*diff(v, dq(j)) + dHnev'*diff(v, dq(j));
    K(j) = K(j) + KHist(j,f(key{i}).o);
                       % Cravity terms
T = Tf(f, '0', key{i});
ROT = T(1:3,1:3);
mg = m*ROT*(0 o -g]';
for j=1:length(0)
    disp(['kanesQ: i=', num2str(i), ', j=', num2str(j)]);
q(j)=q(j)-mg'*diff(vG, dq(j));
end
50
55
```

#### 7.4 getAC()

## 7.5 angVel()

## 7.6 angAcc()

```
function alpha = angAcc(f, key, varargin)
        % Calculate angular acceleration of the current frame represented in the same
        * frame.

% 'f' is the map container of the information for all the frames

% 'key' identifies the frame whose angular acceleration is demanded

% Chemore argument can define nonzero angular velocity of the base frame
         disp([' Computing angAcc of ', key]);
        if(f(key).gotAngAcc)
    alpha=f(key).angAcc;
else
15
              if(nargin == 2)
                    w0 = [0; 0; 0];
alpha0 = [0; 0; 0];
                    0
w0 = varargin{1}(:,1);
alpha0 = varargin{1}(:,2);
              % Angular acceleration of antecembert frame
              if (isequal (f(key).a, '0')) % if it's the first link in the chain alpha.a = Rot(f, f(key).a, key) * alpha0; else % rearsive call if the frame is not the first link in the chain
25
                     alpha_a = Rot(f, f(key).a, key) * angAcc(f, f(key).a, [w0 alpha0]);
              % Angular acceleration of the current frame
30
              alpha = alpha_a + f(key).e * f(key).ddq ...
+ f(key).dq * cross(angVel(f, key, w0), f(key).e);
              frame = f(key);
frame.angAcc = alpha;
frame.gotAngAcc = true;
f(key) = frame;
35
        disp([' Computed angAcc of ', key]);
```

#### 7.7 linVel()

```
function V = linVel(f, key, varargin)
% Calculate linear velocity of the current framemeasured in the world
```

```
| % frame represented in the ourrent frame
       %'f' is the map container containing the information of the frames of
         %therabot
        % 'key' ichrtifiestheoment frame
% Optional arguments:
% 'wU' isthe angular velocity of the base frame
        %'V0 is the linear velocity of the base frame
         disp([' Computing linVel of ', key]);
         if(f(key).gotLinVel)
         V=f(key).linVel;
else
15
               if(nargin == 2)
w0 = [0;0;0];
V0= [0;0;0];
                     e
  w0 = varargin{1}(:,1);
  V0 = varargin{1}(:,2);
25
              if(isequal(f(key).a, '0')) % if it's the first link in the chain
    V = Rot(f, f(key).a, key) * (V0 + cross(w0, f(key).P));
else % if it's same other link
    V = Rot(f, f(key).a, key) * (linVel(f, f(key).a) + ...
    cross(angVel(f, f(key).a), f(key).P));
end
30
               frame = f(key);
frame.linVel = V;
frame.gotLinVel = true;
f(key) = frame;
35
         end
40
         disp([' Computed linVel of ', key]);
```

#### 7.8 linAcc()

```
function a = linAcc(f, key, varargin)
            % Calculate linear velocity of the current framemeasured in the world
           * frameropresented in the current frame

* ff is the map constainer containing the information of the frames of
           % The Strength and the standing of the root % the root % they identifies the current frame % optional arguments % fw0 is the angular velocity of the base frame % 'V0' is the linear velocity of the base frame
            disp([' Computing linAcc of ', key]);
           if(f(key).gotLinAcc)
    a=f(key).linAcc;
else
15
                   if(nargin == 2)
   w0 = [0;0;0];
   a1pha0 = [0;0;0];
   v0 = [0;0;0];
   a0 = [0;0;0];
                             w0 = varargin{1}(:,1);
alpha0 = varargin{1}(:,2);
V0 = varargin{1}(:,3);
a0 = varargin{1}(:,4);
25
                   % Vel/Acc of artecedrt frame represented in artecedrt frame
if (isequal (f(key).a, '0')) % if it's the first link in the chain
    w_a = w0;
    alpha_a = alpha0;
    a_a = a0;
else % if it's same other link
30
35
                              w_a = angVel(f, f(key).a, w0);
alpha_a = angAcc(f, f(key).a, [w0 alpha0]);
a_a = linAcc(f, f(key).a, [w0 alpha0 v0 a0]);
40
                    % Acceleration of the current frame
a = Rot(f, f(key).a, key) * (a_a + cross(alpha_a, f(key).P) ...
+ cross(w_a, cross(w_a, f(key).P)));
                    frame = f(key);
frame.linAcc = a;
frame.gotLinAcc = true;
f(key) = frame;
45
```

```
60 end
disp([' Computed linAcc of ', key]);
```

#### 7.9 Rot()

```
function A = Rot(f, key1, key2)

% Findtherotation transform of a frame
% fountees the map container that contains information of the frames.

% key1 identifies the frame1 that is being represented
% key2 identifies frame2 in which frame1 is being represented
% This fination assumes frame1 immediately follows frame2 or frame2
% immerdiately follows frame1

if (isequal(f(key1).a, key2)) % if frame2 is the artecedant of frame1
    A = [f(key1).x f(key1).y f(key1).z];
elseif(isequal(f(key2).a, key1)) % if frame1 is the artecedant of frame2
    A = [f(key2).x'; f(key2).y'; f(key2).z'];
end
```

#### 7.10 Tf()

```
function T = Tf(f, key1, key2)

% Findthetransformof a frame
% f chrotes the map container that contains information of the frames,
% key1 ichartifies the frame1 that is being represented
% key2 ichartifies frame2 in which frame1 is being represented

if(isBefore(f, key2, key1)) % if frame2 is before frame1 in the chain
    T = [f(key1).x f(key1).y f(key1).z f(key1).P; 0 0 0 0 1];
    key = f(key1).a;
    while('isequal(key, key2))
    T = [f(key1).x f(key2).y f(key).z f(key).P; 0 0 0 0 1] * T;
    key = f(key1).a;
    end

15 elseif(isBefore(f, key1, key2)) % if frame1 is before frame2 in the chain
    T = [f(key2).x' - f(key2).x'**f(key2).P; ...
        f(key2).z' - f(key2).z'**f(key2).P; ...
        0 0 0 1];
    key = f(key1).a;
    while('isequal(key, key1))
    T = T * [f(key).x' - f(key).x'**f(key).P; ...
        f(key).z' - f(key).z'**f(key).P; ...
        f(key).z' - f(key).z'**f(key).P; ...
        f(key).z' - f(key).z'**f(key).P; ...
        f(key).z' - f(key).z'**f(key).P; ...
        f(key).a;
    end
end
```

#### $7.11 \quad mass()$

```
function m = mass(f, key)

param = f(key).param;
m = param(1);
```

#### 7.12 mCOM()

```
function MS = mCOM(f, key)
param = f(key).param;
MS = param(2:4);
```

#### 7.13 inertiaMat()

```
function J = inertiaMat(f, key)
param = f(key).param;
J = param([5 6 7; 6 8 9; 7 9 10]);
```

## 7.14 qVec()

## 7.15 dqVec()

## 7.16 ddqVec()

```
function ddq = ddqVec(f)

% Cenerate cbtyvector in the order defined by member 'd' of the frames
% contained in 'f'

key = keys(f);
n = length(key)-1;
ddq = sym(zeros(n, 1));
for i=1:length(key)

10
    if (isequal(key{i}, '0')); continue; end
    ddq(f(key{i}).o) = f(key{i}).ddq;
end
```

## 7.17 getcoeff()

#### 7.18 kanesRHS.m

```
clear all
load kanesAndLagrange_03_27_15.mat
             % creating a new f with symbolic variables for torques in it
            % tall iml = tall | tall r (sumof reaction of torque applied to wheels) syms tau_imu tau_w tau_torso real syms tau_inu tau_w tau_torso real syms tau_inu tau_2 tau_3 tau_4 tau_5 tau_6 tau_7 real syms tau_1 tau_2 tau_3 tau_4 tau_5 tau_6 tau_7 real
           tauVec = [tau_imu tau_w tau_torso ...
    tau_il tau_2l tau_3l tau_4l tau_5l tau_6l tau_7l ...
    tau_ir tau_2r tau_3r tau_4r tau_5r tau_6r tau_7r]';
key = keys(f);
for i=i:length(key)
    if(isequal(key{i}, '0')); continue; end
    frame = f(key{i});
    frame.tau = tauVec(frame.o);
    f(key{i}) = frame;
end
10
15
20
            % Finding the contribution of body motor torques to kanés RHS
            dq = dqVec(f);
a = sym(zeros(length(dq),1));
aa = sym(zeros(length(dq),length(dq)));
for i=1:length(key)
                     % if key is'0' leave it if(isequal(key{i}, '0')); continue; end
                     % Collect relevant info of aurent frame and previous frame
30
                     frame = f(key{i});

e = frame.e;

tau = frame.tau;

w = angVel(f, key{i}, sym([0 0 0]'));

R = Rot(f, key{i}, frame.a);

wlast = angVel(f, frame.a, sym([0 0 0]'));
35
                      %Calculate the contribution
                     for j=1:length(dq)
    aa(j,frame.o) = tau*e'*diff(w,dq(j)) + (-R*tau*e)'*diff(wlast,dq(j));
    a(j) = a(j) + aa(j,frame.o);
40
            % Cantribution of forces/torques on the endeffector
            syms Flx Fly Flz Frx Fry Frz Tlx Tly Tlz Trx Try Trz real
Fl = [Flx Fly Flz]'; Tl = [Tlx Tly Tlz]'; %expressed in frame 101
Fr = [Frx Fry Frz]'; Tr = [Trx Try Trz]'; %expressed in frame 10r
50
            b1 = sym(zeros(length(dq),1));

b2 = sym(zeros(length(dq),1));

v = linVel(f, '101', sym([0 0 0]'));

v = angwel(f, '101', sym([0 0 0]'));

for i=1:length(dq)

b1(i) = b1(i) + F1'*diff(v, dq(i));

b2(i) = b2(i) + T1'*diff(w, dq(i));
55
             end
            b3 = sym(zeros(length(dq),1));
b4 = sym(zeros(length(dq),1));
v = linVel(f, 'lor', sym([0 0 0]'));
w = angWel(f, 'lor', sym([0 0 0]'));
for i=1:length(dq)
b3(i) = b3(i) + Fr'*diff(v, dq(i));
b4(i) = b4(i) + Tr'*diff(w, dq(i));
65
            % Contributions from the forces of gravity
70
            syms g real
c = sym(zeros(length(dq),1));
            for i=1:length(key)
75
                     % if key is '0' leave it if (isequal(key{i}, '0')); continue; end
                     % Get necessary info
                     v contended that
w = angVel(f, key{i}, sym([0 0 0]'));
v = linVel(f, key{i}, [sym([0 0 0]') sym([0 0 0]')]);
m = mass(f, key{i});
S = mCOM(f, key{i})/m;
T = Tf(f, '0', key{i});
80
85
                     %Calculatecontribution
                     % Cattleter Introductive vG = v + cross(w, S);

ROT = T(1:3,1:3);

mg = m*ROT*(0 0 -g]';

for j=1:length(dq)

c(j)= c(j) + mg'*diff(vG, dq(j));
```

end