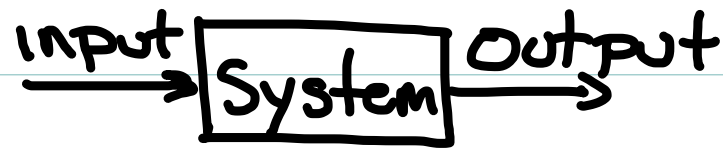


ENAE 301:



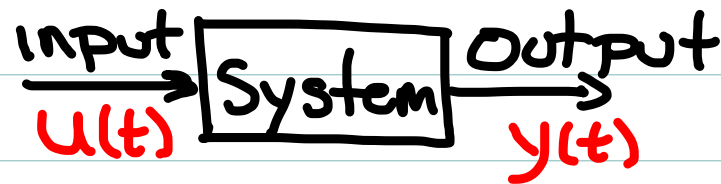
More generally:



ENAE 301:



More generally:



A (dynamic) system "transforms" inputs $u(t)$ into outputs $y(t)$.

We must first understand as completely as possible this "transformation".

Simple Hovercraft Example

$$\frac{d}{dt}(mv) = f \quad (\text{dynamics})$$

$$\frac{d}{dt}(y) = v \quad (\text{Kinematics})$$

Where:

m = mass (assume constant)

v = velocity

y = position

f = applied force

Thus:

$$\left. \begin{aligned} m \frac{dv}{dt} &= f \\ \frac{dy}{dt} &= v \end{aligned} \right\} \text{Governing DE}$$

Equivalently: $m \frac{d^2y}{dt^2} = f$

Driving Force

Force driving system is due to fan:

$$F \approx K_f \omega$$

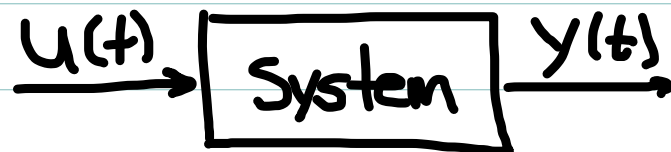
where K_f constant, ω is rotation rate of fan

Similarly: $\omega \approx K_m V_m$

where K_m constant, V_m is voltage applied to motor

Then $\ddot{y}(t) = K u(t)$, $K = \left[\frac{K_f K_m}{m} \right]$

treating $V_m(t) = u(t)$ as the input to the system



Analysis:

Given:

$$\dot{v}(t) = K u(t)$$
$$\dot{y}(t) = v(t)$$

($v(t)$ velocity)

Then:

$$v(t) = v_0 + K \int_0^t u(\tau) d\tau$$

$$y(t) = y_0 + \int_0^t v(\sigma) d\sigma$$

Take $v_0 = y_0 = 0$ for simplicity now, then

$$y(t) = K \int_0^t \left[\int_0^\sigma u(\tau) d\tau \right] d\sigma$$

So: $y(t) = K \int_0^t \int_0^\sigma u(\tau) d\tau d\sigma$ (Double integral!)

Or: $y(t) = K \int_0^t (t-\tau) u(\tau) d\tau$ (How...?)

Example Control Problem

Find $u(t)$ so that, for a specified t_f , y_f

$$\begin{aligned} v(t_f) = 0 &\Rightarrow \quad \dot{y} = K \int_0^{t_f} u(\tau) d\tau \\ y(t_f) = y_f &\Rightarrow \quad y_f = K \int_0^{t_f} (t_f - \tau) u(\tau) d\tau \end{aligned}$$

← Solve for $u(\tau)$

Here, we are assuming vehicle starts at rest ($v(0) = 0$)
On the "start line" ($y(0) = 0$).

Want the vehicle to move to position y_f in t_f seconds
and stop there.

Many sol's $u(t)$ possible! Typically would also constrain:

1.) $|u(t)| \leq u_{\max}$

2.) Behavior of $y(t)$, $t \in [0, t_f]$

Issues

1.) m, K_f, K_m not known precisely:

Hovercraft will not stop exactly where we want.

2.) Requires an accurate clock:

Must use correct $u(t)$ at exactly right times t .

3.) Cannot handle an external ("disturbance") force:

Headwind or cross-breeze will drive hovercraft off the track.

Mathematically sound, but not practical!

Do you drive like that? I hope not!

Mathematically sound, but not practical!

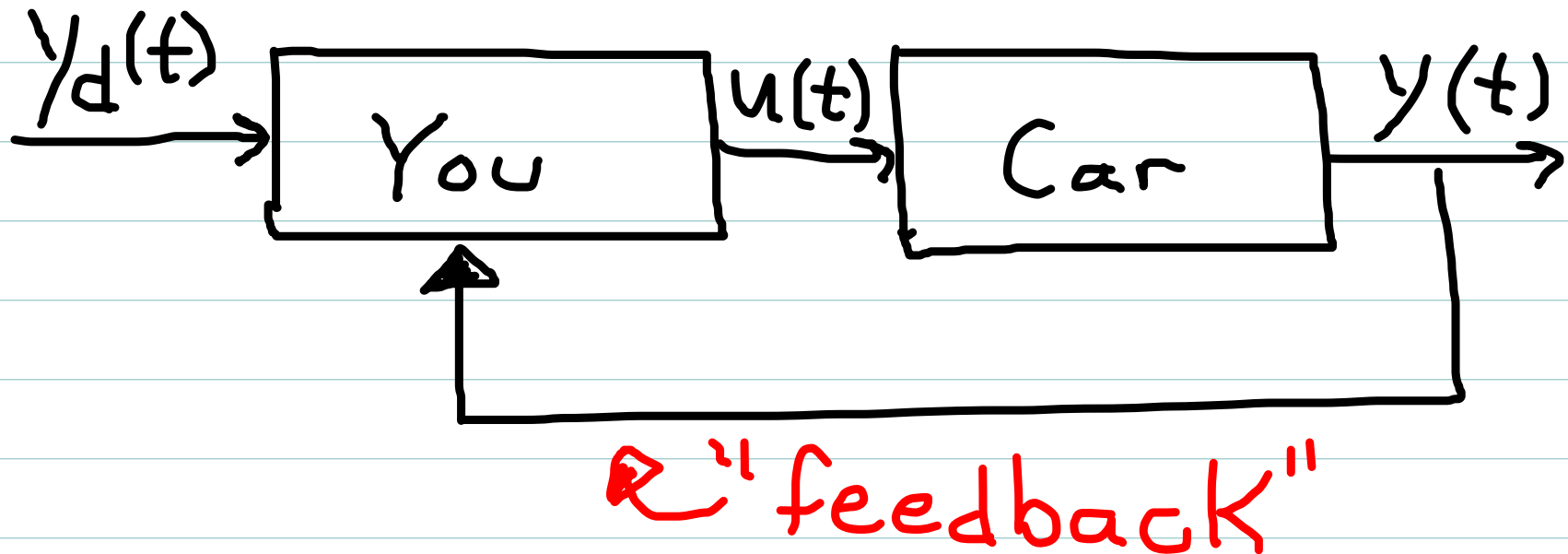
Do you drive like that? I hope not!

Instead you continually compare where you are ($y(t)$) with where you want to be ($y_d(t)$) and continually adjust actions ($u(t)$) based on difference.

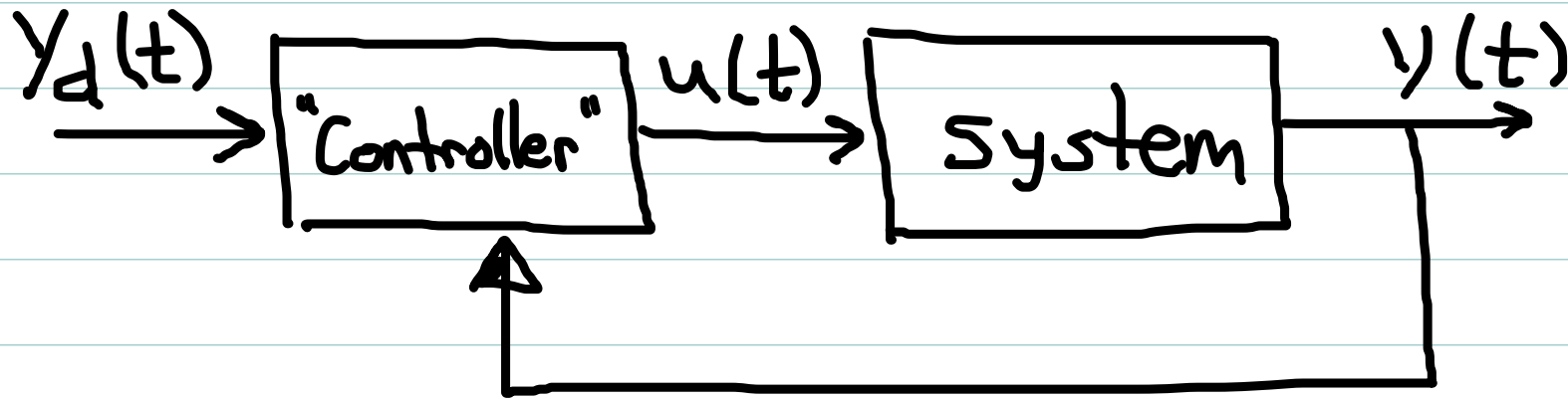
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Feedback Control



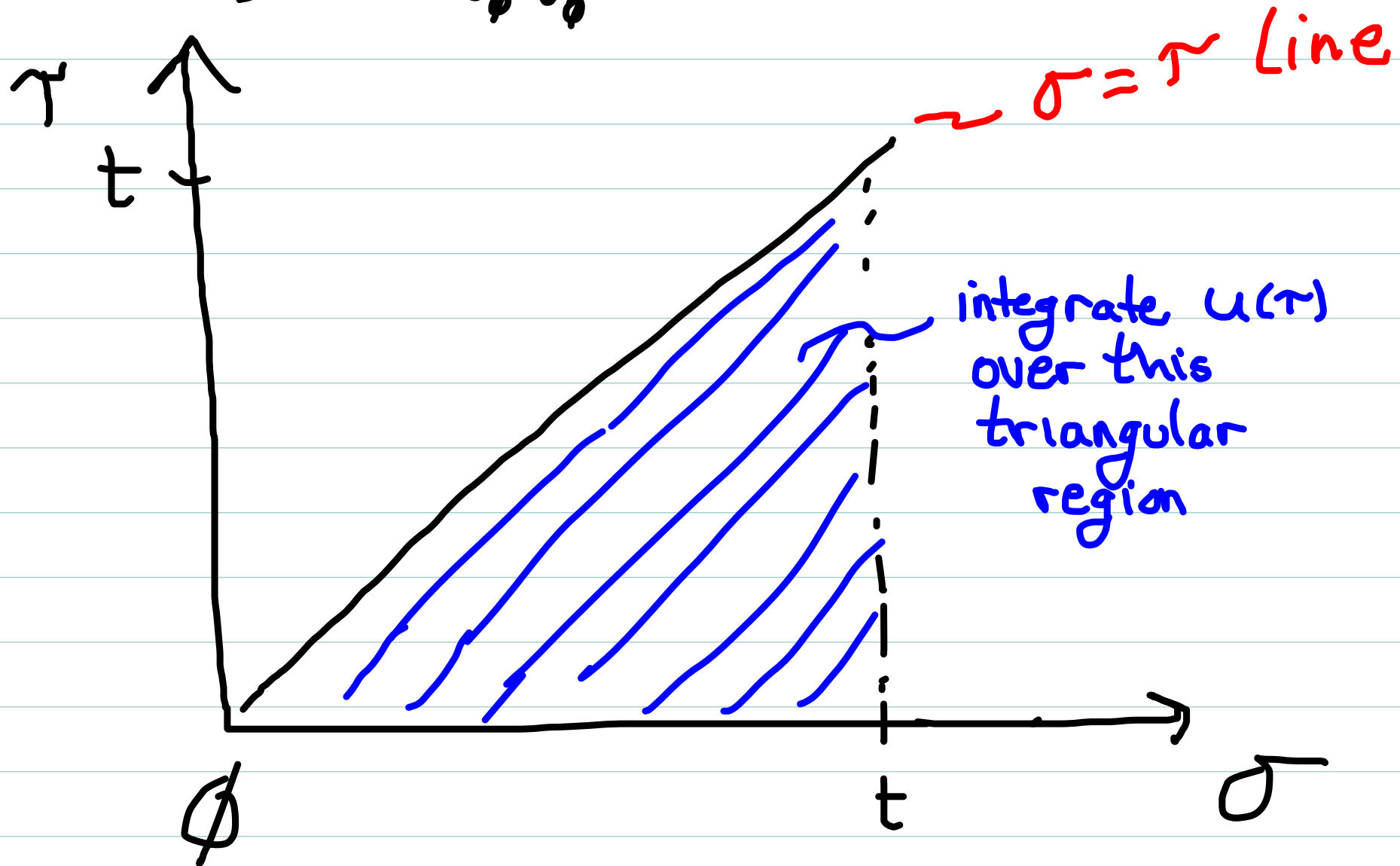
The controller is a mathematical algorithm (implemented as a computer program) which calculates required $u(t)$ from $y(t)$ and $y_d(t)$.

Addresses all 3 issues: uncertainty, disturbance, clocking

This course is about the derivation + implementation of suitable feedback control algorithms based on governing dynamics of system.

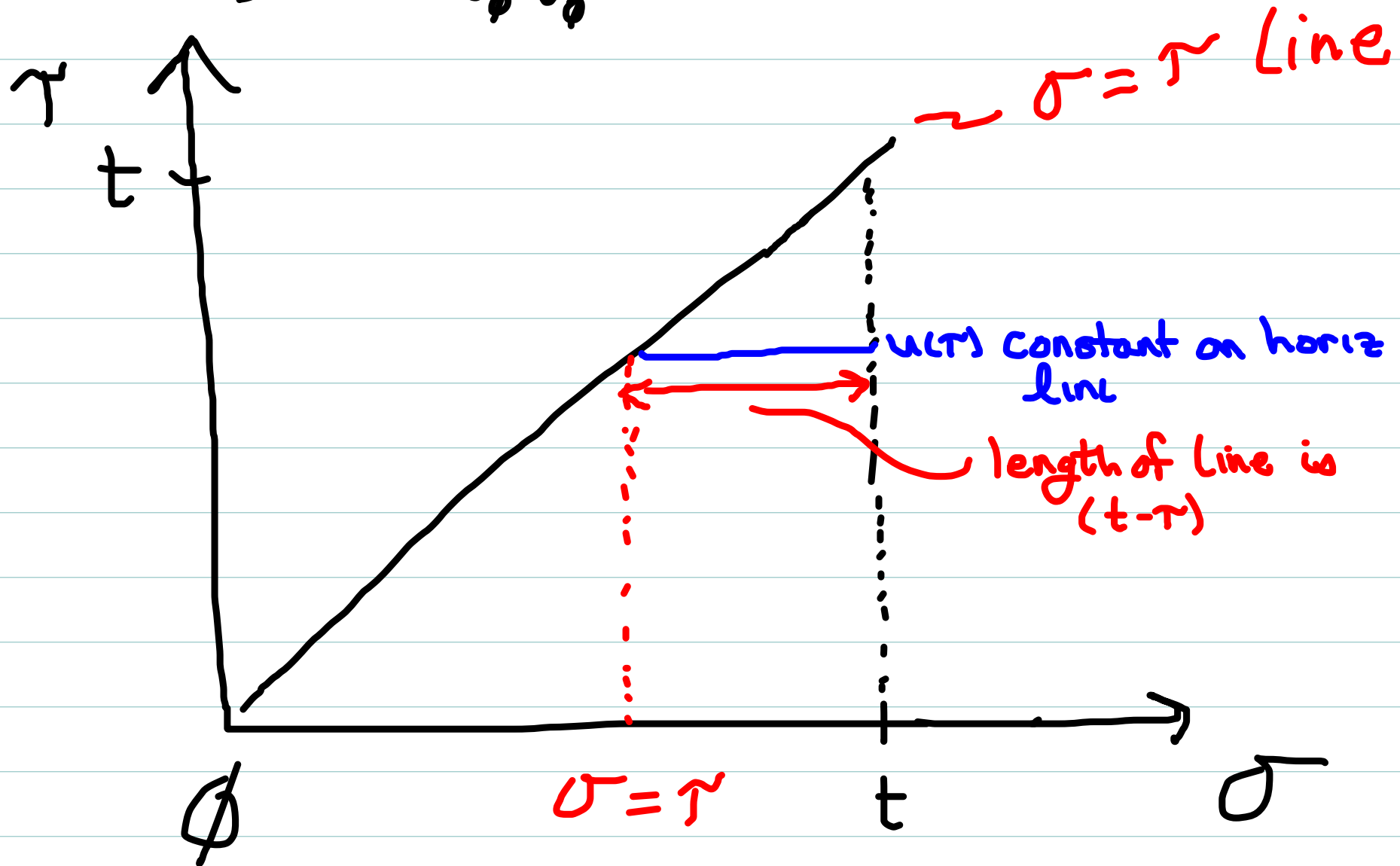
Reduction of the double integral

$$y(t) = K \int_0^t \int_0^\sigma u(\tau) d\tau d\sigma$$



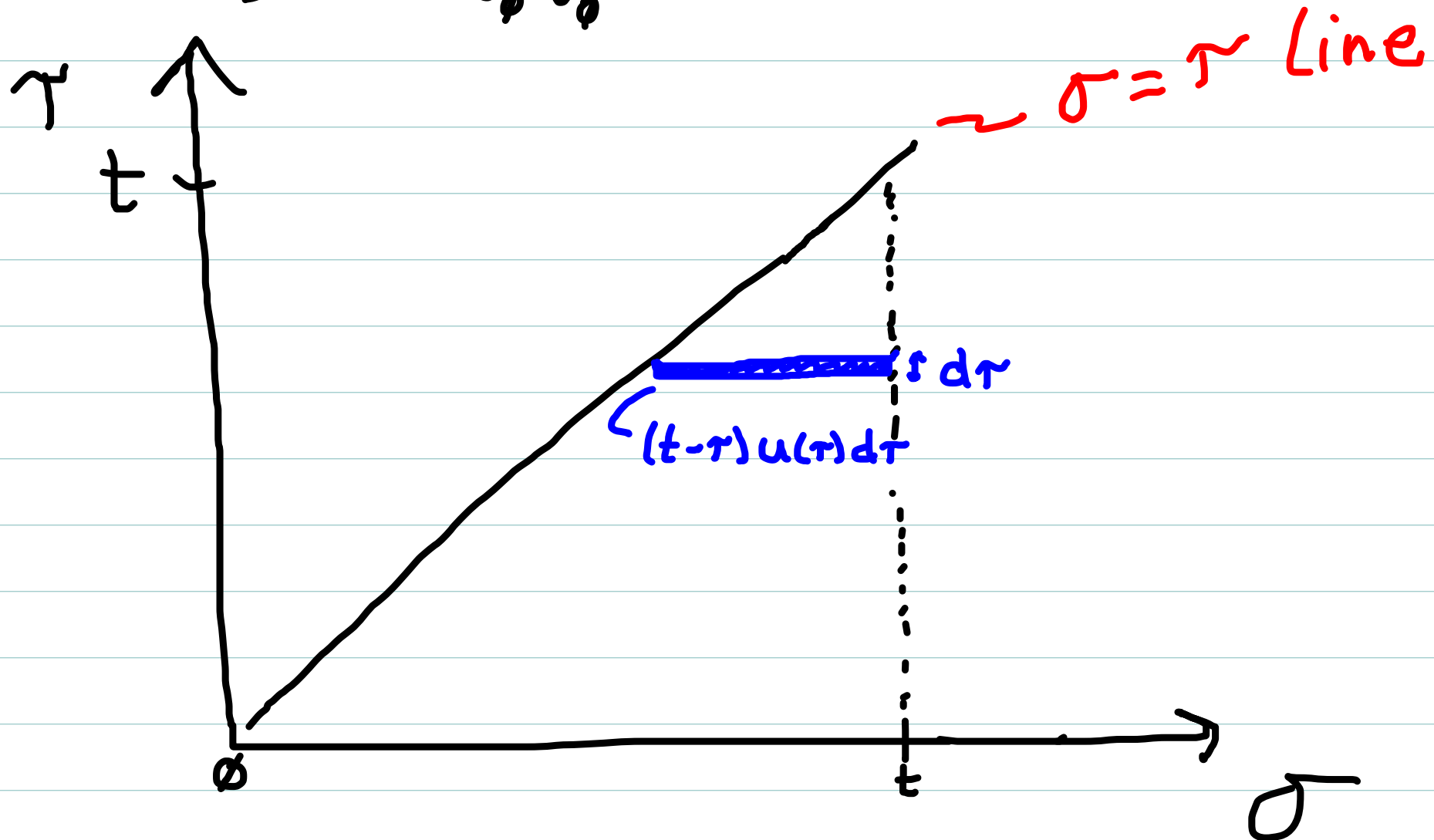
Reduction of this double integral

$$y(t) = K \int_0^t \int_0^\sigma u(\tau) d\tau d\sigma$$



Reduction of this double integral

$$y(t) = K \int_0^t \int_0^\sigma u(\tau) d\tau d\sigma$$



Integrate over all strips $\Rightarrow y(t) = K \int_0^t (t-\tau) u(\tau) d\tau$
 + multiply by K :

An alternate form

Our sol'n has the general form:

$$y(t) = \int_0^t g(t-\tau) u(\tau) d\tau$$

where here $g(t) = Kt$ [so $g(t-\tau) = K(t-\tau)$]

We will (indirectly) show that for any system, no matter how complex the dynamics, this relationship between $u(t)$ and $y(t)$ holds.

Different systems are characterized by different functions $g(t)$.

The characteristic function $g(t)$ is called the Impulse response

Implication

Suppose:

$$y_1(t) = \int_0^t g(t-\tau) u_1(\tau) d\tau$$

$$y_2(t) = \int_0^t g(t-\tau) u_2(\tau) d\tau$$

are two known input-output pairs.

Suppose that $u(t) = \alpha_1 u_1(t) + \alpha_2 u_2(t)$; α_1, α_2 constant

Then:

$$y(t) = \int_0^t g(t-\tau) [\alpha_1 u_1(\tau) + \alpha_2 u_2(\tau)] d\tau$$

$$= \alpha_1 \int_0^t g(t-\tau) u_1(\tau) d\tau + \alpha_2 \int_0^t g(t-\tau) u_2(\tau) d\tau$$

hence

$y(t) = \alpha_1 y_1(t) + \alpha_2 y_2(t)$ **Principle of Linearity.**

This suggests an approach:

- ① Identify a "family" of functions $u_k(t)$ for which it is easy to calculate response $y_k(t)$:

$$u_k(t) \mapsto y_k(t) \quad (\text{easy})$$

- ② "Break down" an arbitrarily complicated $u(t)$ into a linear combination of the $u_k(t)$:

$$u(t) = \sum \alpha_k u_k(t) \quad (\text{easy?})$$

- ③ Use Linearity:

$$y(t) = \sum \alpha_k y_k(t) \quad (\text{easy})$$

Time Varying Complex numbers

$$\begin{aligned} Z(t) &= a(t) + b(t)j \\ &= r(t) e^{j\theta(t)} \end{aligned}$$

Important example:

$$Z(t) = e^{st} \text{ with } s \in \mathbb{C}$$

"Complex exponential functions"

Let $s = \sigma + j\omega$ $\sigma, \omega \in \mathbb{R}$

So $\operatorname{Re}\{s\} = \sigma$, $\operatorname{Im}\{s\} = \omega$

① If $\omega = 0$, then

$$e^{st} = e^{\sigma t} \text{ (real exponential)}$$

② If $\sigma = 0$ then

$$e^{st} = e^{j\omega t} = \cos \omega t + j \sin \omega t$$

Note: $\operatorname{Im}\{s\}$ gives frequency of the oscillations

③ Most general case

$$e^{st} = e^{(\sigma + j\omega)t} = e^{\sigma t} e^{j\omega t}$$

$$= e^{\sigma t} [\cos \omega t + j \sin \omega t]$$

$$\operatorname{Re}\{e^{st}\} = e^{\sigma t} \cos(\omega t)$$

$$\operatorname{Im}\{e^{st}\} = e^{\sigma t} \sin(\omega t)$$

$\sigma \rightarrow$ amplitude envelope

$\omega \rightarrow$ oscillation frequency

$s = \sigma + j\omega$ is the

"Complex frequency"