

# Stability

A mode  $e^{pt}$  is stable if  
 $|e^{pt}| \rightarrow 0$  as  $t \rightarrow \infty$

A system is stable if  
 $|e^{p_k t}| \rightarrow 0$  for all  $k=1, \dots, n$

i.e. if every mode is stable

Note: if true then  $y_h(t) \rightarrow 0$  for any set  
of initial conditions.

# Stability Condition

As usual, let  $p = \sigma + j\omega$ . Then:

$$|e^{pt}| = |e^{(\sigma + j\omega)t}|$$

$$= |e^{\sigma t} e^{j\omega t}| = |e^{\sigma t}| |e^{j\omega t}|$$

$$= |e^{\sigma t}|$$

So  $|e^{pt}| \rightarrow 0$  only if  $\sigma < 0$ . Hence:

A mode is stable if  $\sigma = \operatorname{Re}\{p\} < 0$

# System Stability

The system is stable if:

$$\operatorname{Re}\{p_k\} < 0 \text{ for all } k=1, \dots, n$$

$\Rightarrow$  all roots of  $r(s)$  have negative real parts

$\Rightarrow$  all roots of  $r(s)$  lie to the left of imaginary axis in the complex plane.

$\Rightarrow$  all roots of  $r(s)$  lie in left half of complex plane (LHP)

# Instability

A mode  $e^{pt}$  is unstable if  $\operatorname{Re}\{p\} > 0$

$\Rightarrow$  root  $p$  lies to right of imag axis

$\Rightarrow p$  is in "right half plane" (RHP)

A system is unstable if:

$\operatorname{Re}\{p_k\} > 0$  for any  $k=1, \dots, n$

i.e. if any roots of  $r(s)$  are in RHP.

## What about repeated modes?

Repeated real modes have terms like:

$$t^i e^{pt} \quad (\text{powers of } t \text{ multiplying } e^{pt})$$

Fact: for any  $i > 0$ , if  $\operatorname{Re}\{p\} < 0$  then

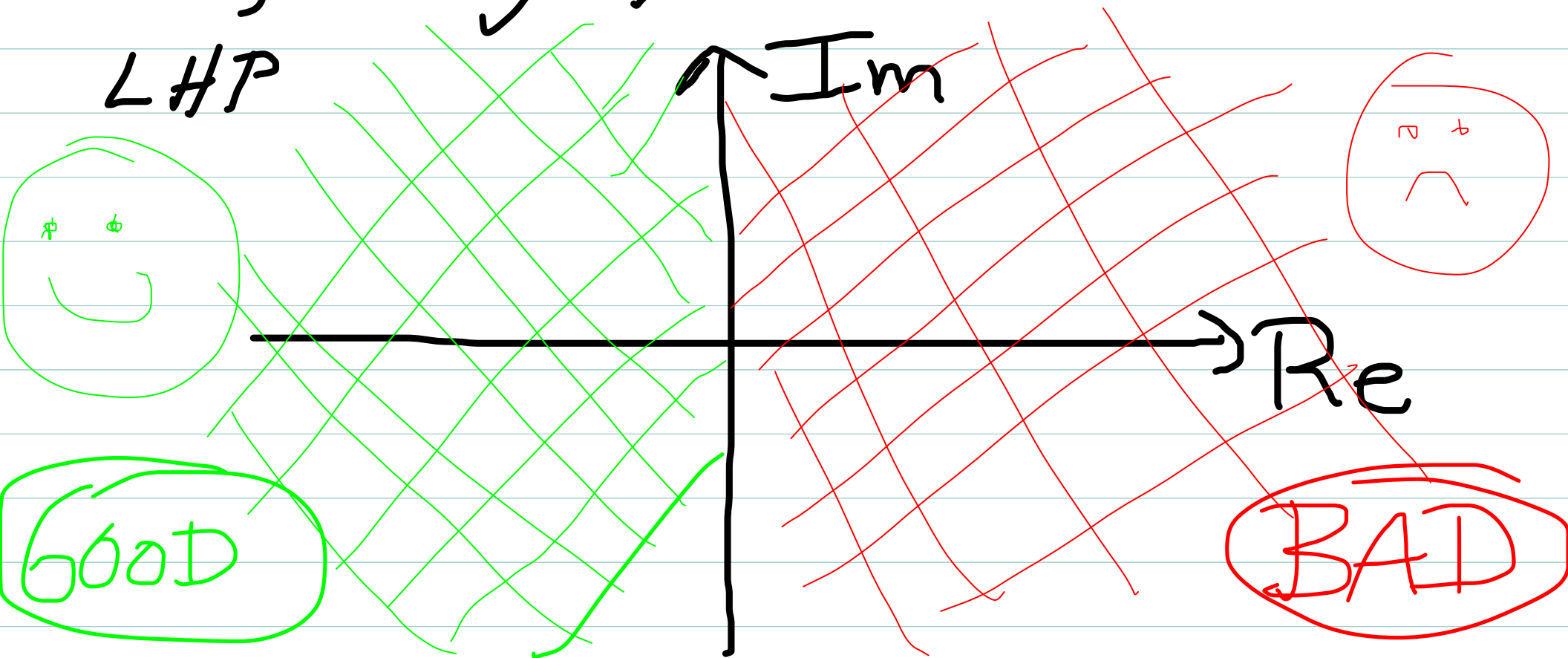
$$\lim_{t \rightarrow \infty} |t^i e^{pt}| \rightarrow 0$$

Thus a repeated mode is stable as long as the repeated roots are in LHP.

Conversely, a repeated mode is unstable if repeated roots in RHP.

Hence:

A system is stable if all roots of  $r(s)$ , including repeated roots, lie in LHP



## Recap

Stable mode:  $|e^{pt}| \rightarrow 0$  as  $t \rightarrow \infty$

$$\iff \operatorname{Re}\{p\} < 0$$

Unstable mode:  $|e^{pt}| \rightarrow \infty$  as  $t \rightarrow \infty$

$$\iff \operatorname{Re}\{p\} > 0$$

Stable system:  $\operatorname{Re}\{p_k\} < 0$  for all  $k=1, \dots, n$

Unstable system:  $\operatorname{Re}\{p_k\} > 0$  for any  $k=1, \dots, n$

What happens if  $\operatorname{Re}\{p\} = 0$ ?

## Marginally stable MODES

$$\operatorname{Re}\{p\} = 0 \Rightarrow |e^{pt}| = |e^{j\omega t}| = 1 \quad \forall t \geq 0$$

i.e. the magnitude is constant

$\Rightarrow$  neither increasing nor decreasing with time

$\Rightarrow$  neither stable nor unstable

"Marginally stable"

Repeated modes with  $\operatorname{Re}\{p\} = 0$  will increase in magnitude polynomially in  $t$

$\Rightarrow$  Not as "bad" as exponential growth



## An alternate decomposition of $y(t)$

$$y(t) = y_h(t) + y_f(t)$$

$$= y_{tr}(t) + y_{ss}(t) \quad \} \text{ (re group terms)}$$

$y_{tr}(t)$  is the "transient response", which satisfies:

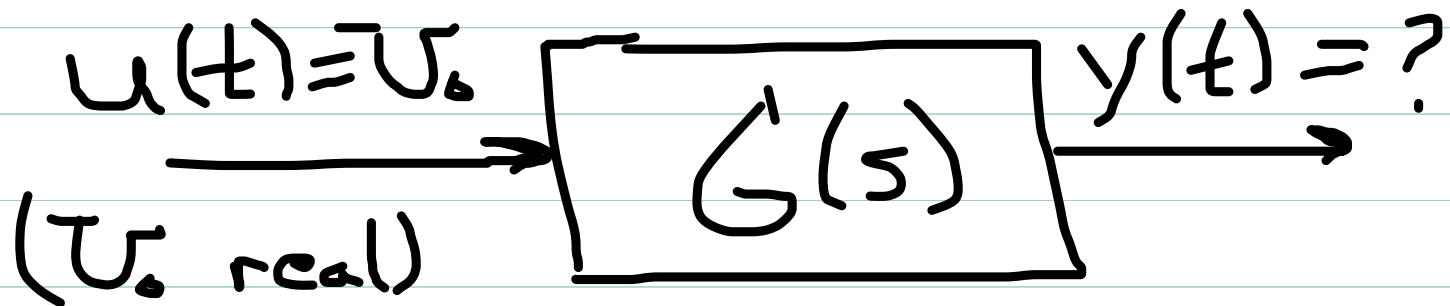
$$\lim_{t \rightarrow \infty} |y_{tr}(t)| \rightarrow 0$$

$y_{ss}(t)$  is the "steady-state" response, which is all remaining terms in  $y(t)$ .

## Notes:

- ① If system is stable,  $y_{tr}(t)$  contains  $y_h(t)$  but  $y_{tr}(t)$  would also contain decaying terms in  $y_f(t)$  (if any).
- ② Conversely, marginally stable terms in  $y_h(t)$  (if any) would be part of  $y_{ss}(t)$ .
- ③ "Steady-state" is not a useful concept if system is unstable.

Example: Stable system with constant input



constant!

$$y(t) = y_h(t) + y_f(t) = y_h(t) + \boxed{G(\emptyset)U_0}$$

Since system is stable,  $y_h(t) \rightarrow 0$  as  $t \rightarrow \infty$

So here:

$$y_{tr}(t) = y_h(t)$$

$$y_{ss}(t) = G(\emptyset)U_0 \quad (\text{constant})$$

Very common and important case!

## A Different Example

$$G(s) = \frac{s+2}{s(s+1)}, \quad u(t) = e^{-3t}$$

$$y_h(t) = C_1 + C_2 e^{-t}$$

$$y_f(t) = G(-3)e^{-3t} = -\frac{1}{6}e^{-3t}$$

$$y_{tr}(t) = C_2 e^{-t} - \frac{1}{6}e^{-3t}$$

$$y_{ss}(t) = C_1$$

Note: system is not stable here.

## Convergence metrics

Useful to quantify how quickly stable modes decay to  $\emptyset$ .

"2% criterion": Defines the settling time

$t_s$  for a mode to be such that

$$|e^{pt}| \leq .02 \quad \forall t \geq t_s$$

For a  $1^{\text{st}}$  order mode ( $p = \sigma$ , real)

$$t_s = \frac{\ln(.02)}{\sigma} \approx \frac{4}{|\sigma|} = \frac{4}{|\operatorname{Re}\{p\}|}$$

## 2<sup>nd</sup> order settling time

For a 2<sup>nd</sup> order mode  $e^{pt}$ ,  $e^{\bar{p}t}$  with

$p = \sigma + j\omega$ ,  $\omega \neq 0$ , the calculation is more complicated due to the oscillations.

However: 
$$\tau_s \approx \frac{4}{|\sigma|} = \frac{4}{|\operatorname{Re}\{p\}|}$$

is still a useful approximation in these cases also.

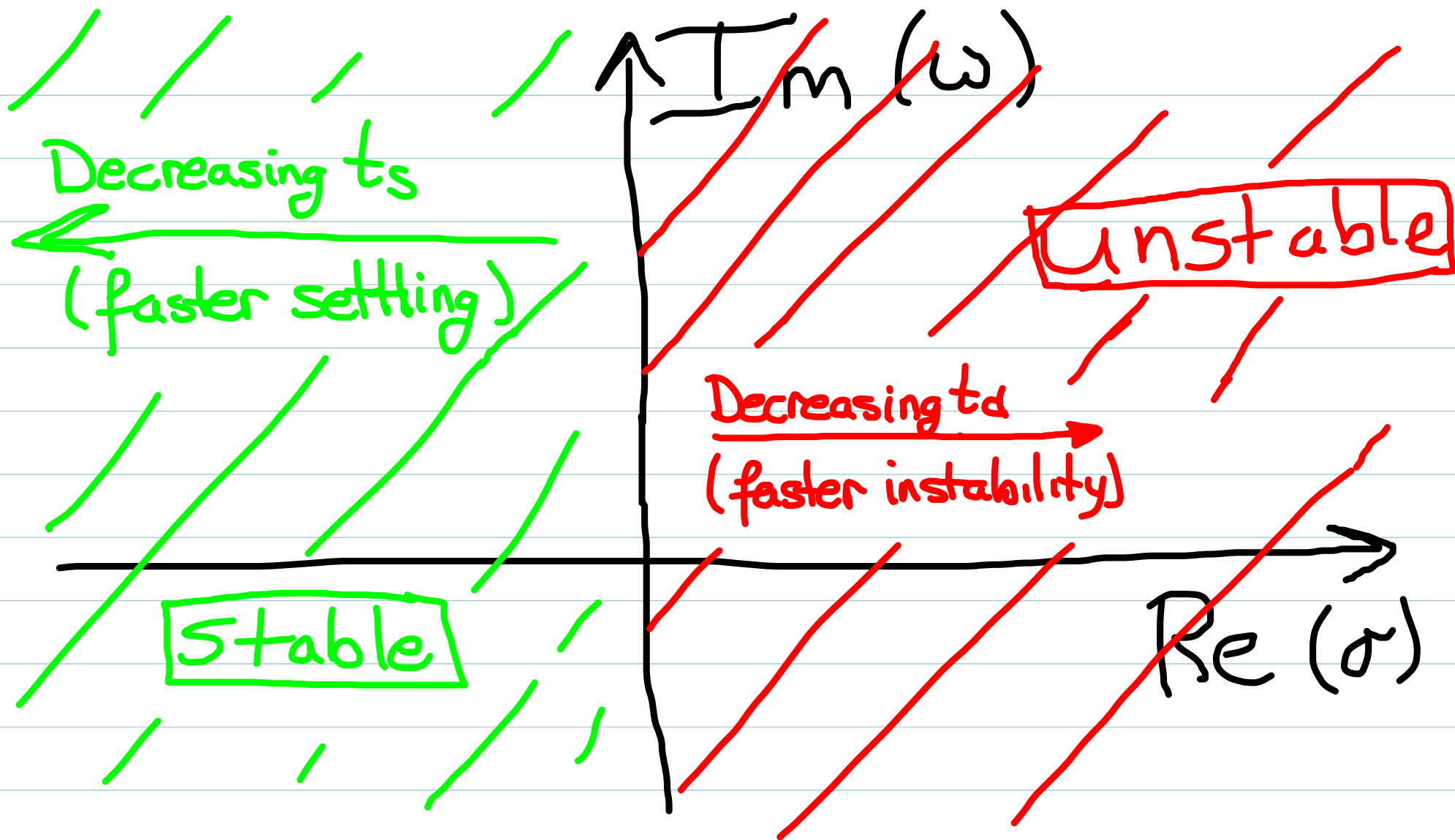
## "Doubling time" of unstable modes

When  $\sigma > 0$ , the doubling time  $t_d$  is such that

$$|e^{\sigma t_d}| = 2 \Rightarrow t_d \approx \frac{0.7}{\sigma}$$

Smaller  $t_d \Leftrightarrow$  "more unstable" system

$\Rightarrow$  Faster rate of increase for amplitude



Settling times decrease the further left of the imag axis the root  $p$  is.



To a first approximation, the settling time of a system is the settling time of its slowest mode

=> Mode closest to imag Axis determines settling time

=> Called the "dominant mode"

=> Only a "first cut." Will refine later

## 2<sup>nd</sup> Order "Damping ratio"

For 2<sup>nd</sup> order modes we also define the damping ratio

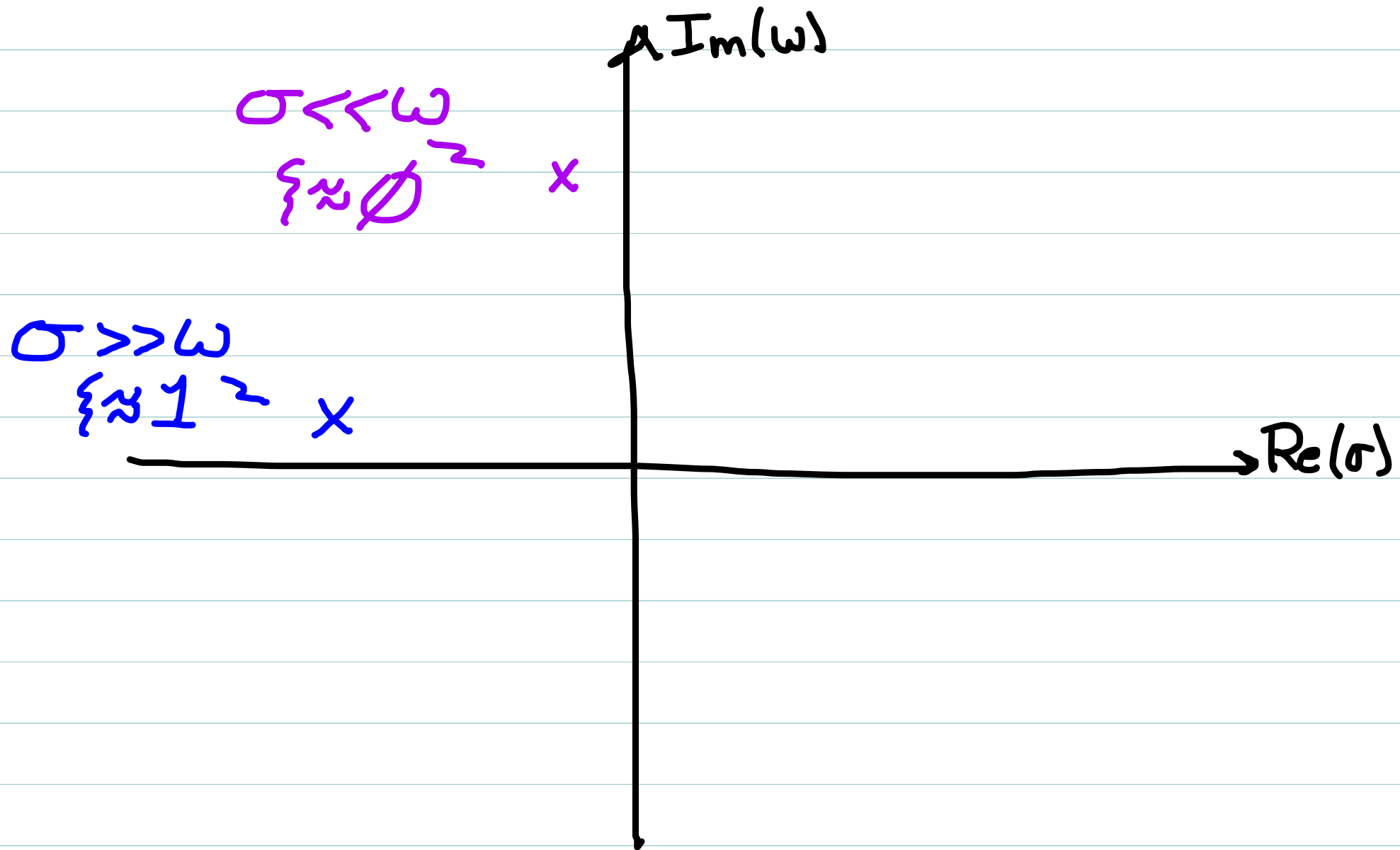
$$\xi = \left| \frac{\sigma}{p} \right| = \frac{|\sigma|}{\sqrt{\sigma^2 + \omega^2}}$$

A non-dimensional comparison of convergence rate to oscillation frequency

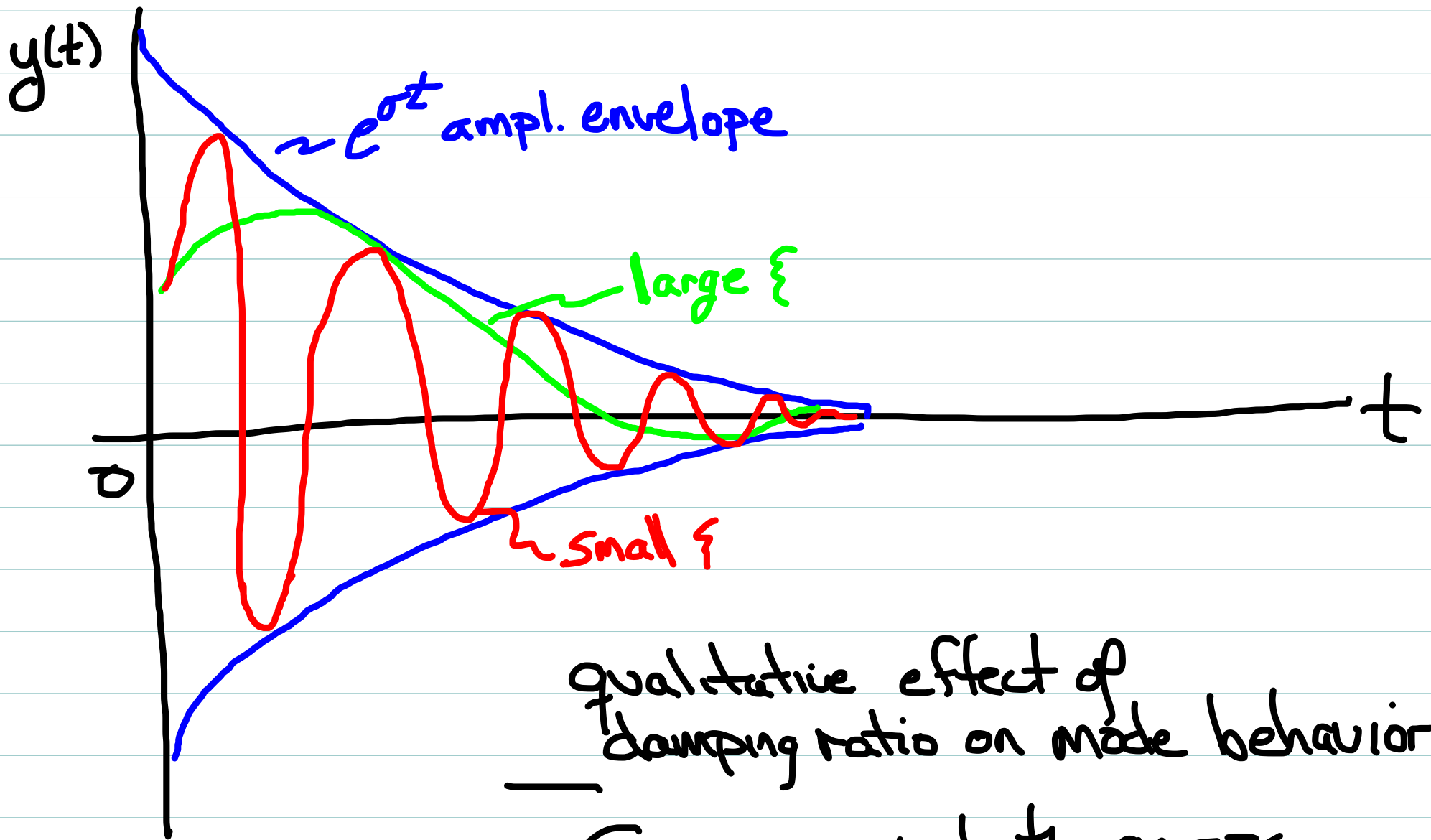
$0 \leq \xi \leq 1$  for a stable mode

$\xi \approx 0 \Leftrightarrow$  many oscillations before 2% criterion reached

$\xi \approx 1 \Leftrightarrow$  less than one complete oscillation before 2% criterion reached.



Will explore in greater detail later



qualitative effect of  
damping ratio on mode behavior

— Same  $\sigma$  in both CASES,  
different  $\omega$

# Transfer functions

$$G(s) = \frac{q(s)}{r(s)}$$

Compactly gives us all information we need to predict major features of system response

-  $y_h(t)$ , modes, stability: all from  $r(s)$   
the denominator polynomial of  $G(s)$

$$r(s) = \alpha_n \prod_{k=1}^n (s - p_k)$$

- forced response: Evaluate  $G(s)$   
at specific complex values of  $s$ .

# Numerator Terms

Can also factor  $q(s)$ :

$$q(s) = \beta_m (s - z_1)(s - z_2) \cdots (s - z_m)$$

where  $q(z_i) = 0$  for  $i = 1, \dots, m$

The values  $z_i$  are called the zeros of  $G(s)$

$$\text{Since } G(z_i) = \frac{\cancel{q(z_i)}^0}{r(z_i)} = 0$$

The values  $p_k$  are called the poles of  $G(s)$

$$\text{Since } G(p_k) = \frac{q(p_k)}{\cancel{r(p_k)}^0} = \infty$$

## Zero/Pole/Gain (ZPK) form

$$G(s) = K \left[ \frac{\prod_{i=1}^m (s - z_i)}{\prod_{k=1}^n (s - p_k)} \right]$$

Poles  $p_k$  satisfy  $r(p_k) = \emptyset$

Zeros  $z_i$  satisfy  $q(z_i) = \emptyset$

Gain:  $K = \frac{\beta_m}{\alpha_n}$  (always real)

## Alternate ZPK form:

When  $G(s)$  has complex poles and/or zeros, we commonly combine the conjugate roots of  $r(s)$  or  $q(s)$  into 2<sup>nd</sup> order polynomials.

For example, if  $p = \sigma + j\omega$  and  $\bar{p} = \sigma - j\omega$  are complex roots of  $r(s)$ :

$$(s-p)(s-\bar{p}) = s^2 - 2\sigma s + (\sigma^2 + \omega^2)$$

→ replace with  $\uparrow$  in  $G(s)$