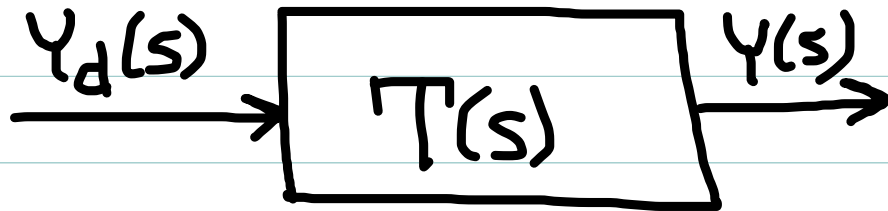
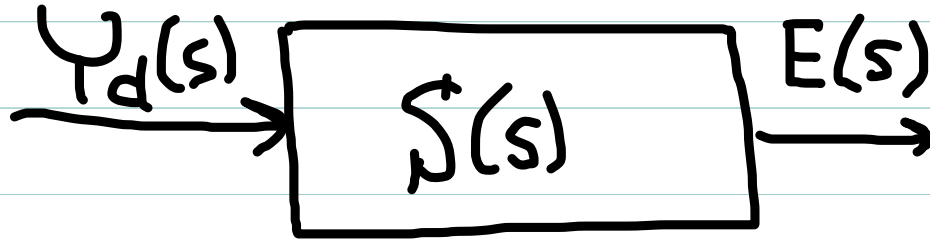


Three Derived TFs for Feedback Loops

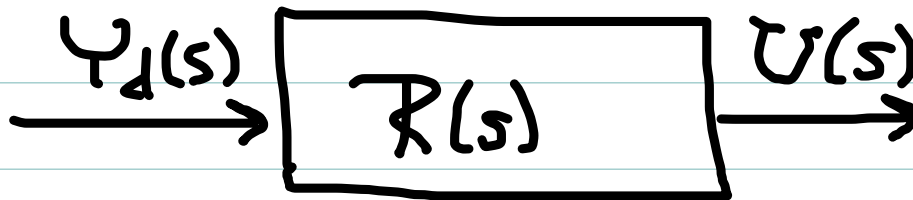
Given $G(s)$ and $H(s)$, we can derive $R(s)$, $S(s)$, $T(s)$ so that:



$$T(s) = \frac{L(s)}{1+L(s)}$$



$$S(s) = \frac{1}{1+L(s)}$$



$$R(s) = \frac{H(s)}{1+L(s)}$$

\Rightarrow Each of these derived TFs can be analyzed using the same tools developed for $G(s)$.

Uses of derived TF:

$\Rightarrow T(s)$ tells us about actual response of controlled system for specific $y_d(t)$

$$Y(s) = T(s)Y_d(s)$$

$\Rightarrow S(s)$ tells us about tracking accuracy for specific $y_d(t)$

$$E(s) = S(s)Y_d(s)$$

$\Rightarrow R(s)$ tells us about required input for specific $y_d(t)$:

$$U(s) = R(s)Y_d(s)$$

Note: all 3 of these TF have the same denominator, hence same poles!!!

Example use of loop TF:

Suppose $y_d(t) = A \cdot 1(t)$ (step of magnitude A)

Then:

$$y(t) = A \times \{\text{step response of } T(s)\}$$

$$u(t) = A \times \{\text{step response of } R(s)\}$$

$$e(t) = A \times \{\text{step response of } S(s)\}$$

Note in particular here that:

$$e_{ss}(t) =$$

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$$e(t) = A \times \{\text{step response of } S(s)\}$$

Note in particular here that:

$$e_{ss}(t) = A \dot{S}(\phi) \quad (\text{constant})$$

Thus generally we'd like to make sure $\dot{S}(\phi) = \phi$
(or at least very small).

Example Application: Tracking Ability

A good feedback loop needs to ensure $|e_{ss}(t)|$ small for a wide variety of $y_d(t)$.

Suppose $y_d(t) = A$ (constant)

Then (assuming all poles of $\dot{S}(s)$ at least stable)

$$e_{ss}(t) = A \dot{S}(0)$$

So good tracking requires $|\dot{S}(0)|$ small.

Ideally, $\dot{S}(0) = 0 \Rightarrow e_{ss}(t) = 0$ "perfect tracking"

and this is often a basic design requirement.

Tracking (cont)

Suppose more generally $y_d(t) = A \cos \omega t$

then $e_{ss}(t) = A |S(j\omega)| \cos(\omega t + \angle S(j\omega))$

and in particular $|e_{ss}(t)| \leq A |S(j\omega)|$

So we want $|S(j\omega)| \ll 1$ for a wide range of frequencies ω (including $\omega = 0$)

\Rightarrow Want Bode magnitude diagram $|S(j\omega)| \ll 0 \text{ dB}$ for a large range of ω (including 0).

\Rightarrow We will show feedback loops with good tracking properties place constraints on design process, which often conflict with other requirements (stability + performance).

Bandwidth

Define ω_B to be largest ω for which

$$|S(j\omega)| \leq -3\text{dB} \quad \text{for all } \omega \in [0, \omega_B]$$

this is the (tracking) bandwidth of the system.

\Rightarrow We want designs with high bandwidth.

Note: -3dB is an arbitrary boundary between acceptable and poor tracking. Realistic performance constraints are typically much tighter:

$$|S(j\omega)| \leq -20\text{dB} \quad (\leq 10\% \text{ worst case error})$$

or

$$|S(j\omega)| \leq -40\text{dB} \quad (\leq 1\% \text{ worst case error})$$

Example Application: Utility of $R(s)$

$\Rightarrow R(s)$ lets us theoretically predict the $u(t)$ which will be generated under ideal circumstances given a specified $y_d(t)$.

$$u(t) = \mathcal{L}^{-1}\{R(s)Y_d(s)\}$$

\Rightarrow Primary quantity of interest is $\max_{t \geq 0} |u(t)|$

\Rightarrow Quantifies maximum control effort required.

\Rightarrow Real actuators have limits $|u(t)| \leq u_{\max}$

\Rightarrow Must ensure our control strategy does not "saturate" the actuators, i.e. $\max_t |u(t)| \leq u_{\max}$

Saturation

Saturation of actuators, i.e. $|u(t)| = u_{\max}$ for some $t \geq 0$, may produce performance degradation or even instability even when the poles of $R(s)$ are "good."

Unfortunately, no simple design guidelines for $H(s)$ which ensure saturation does not occur.

Some degree of design iteration typically required

Advanced (graduate level) techniques do exist to incorporate actuator limits into the design process.

Closed-loop poles

- \Rightarrow Performance of controlled system (settling time, steady-state, overshoot, etc) depends on poles of $T(s)$
- $\Rightarrow (R(s) \text{ and } S(s) \text{ have same poles!!})$
- \Rightarrow Where are these poles??
- \Rightarrow Determined by denominator of $T(s)$
- $\Rightarrow (R(s) \text{ and } S(s) \text{ have same denominator})$
- \Rightarrow Denom of all 3 derived TF is:
$$1 + L(s)$$

Characteristic Equation

Poles of $T(s)$, $R(s)$, $S(s)$ are at values of $s \in \mathbb{C}$ such that

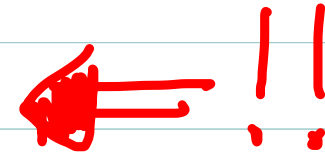
(CE) $1 + L(s) = 0$ "Characteristic equation" of feedback system

We need sol's of this equation to be in "good" locations of complex plane.

Will identify required properties for $L(s)$ so this is true, then work backwards to determine required properties of $H(s)$.

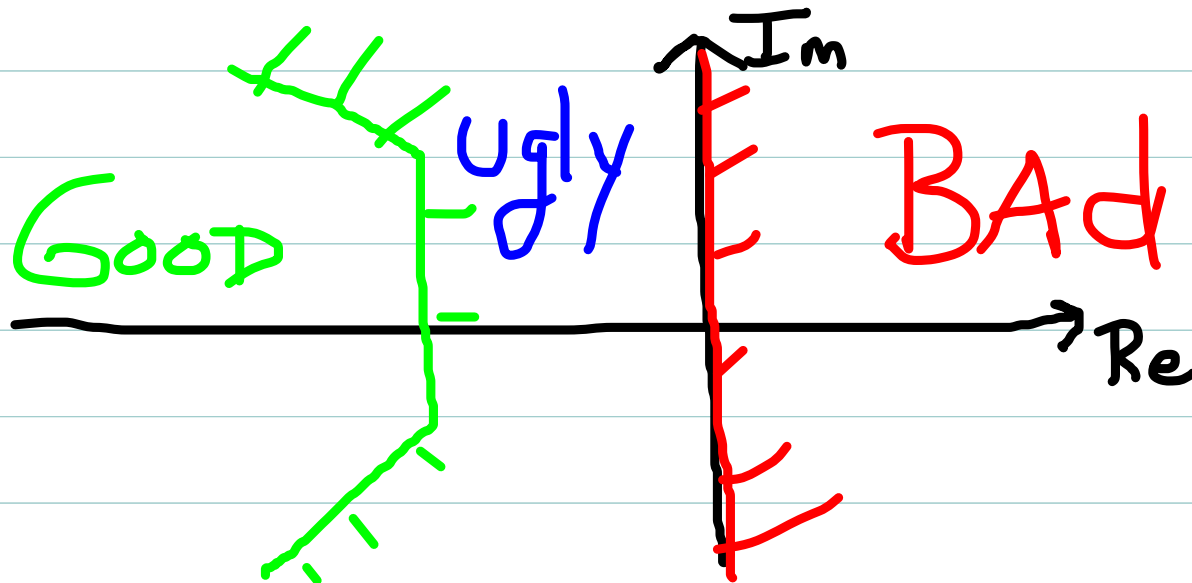
Fundamental Consideration: Closed-loop Stability

Most basic design consideration:



Closed-loop poles should be "good", and certainly must be stable.

Thus, sol'n's of $CE: 1+L(s)=0$ must be in left half of complex plane, preferably in "good region" (far from imag Axis, relatively close to or on the real Axis).



A crucial Observation:

If $L(j\omega) = -1$ for some ω , then

$1 + L(s) = 0$ has a sol'n $s = j\omega$ for some ω

\Rightarrow closed-loop dynamics has poles at $\pm j\omega$, on imag Axis

\Rightarrow Such poles are on the boundary between bad and ugly

\Rightarrow This situation must be avoided!!!

Now if $L(j\omega) = -1$ for some $\omega > 0$, then:

\Rightarrow polar plot of $L(j\omega)$ passes through -1

$\Rightarrow \omega_a = \omega_\gamma$ (both crossover freqs same)

$\Rightarrow a = 0 \text{ dB}, \gamma = 0^\circ$ (both margins 0)

Any such feedback loop is bad!

Now, suppose $\exists \omega \geq 0 \ni L(j\omega) \approx -1$ (i.e. close to, but not exactly -1)

By continuity of $L(s)$, $1 + L(s) = 0$ would have a sol'n very near (but not exactly on) the imag axis.

Some poles of $T(s)$ would be in bad or ugly region
 \Rightarrow Also undesirable!

Now, if $L(j\omega) \approx -1$ for some $\omega \geq 0$

\Rightarrow polar plot of $L(j\omega)$ comes very close to -1
but doesn't pass exactly through it

\Rightarrow (typically) $|a_{dB}|$ and $|\gamma|$ very small
(small margins)

\Rightarrow This should also be avoided.

Thus, for $T(s)$ to have only good poles, we need conditions:

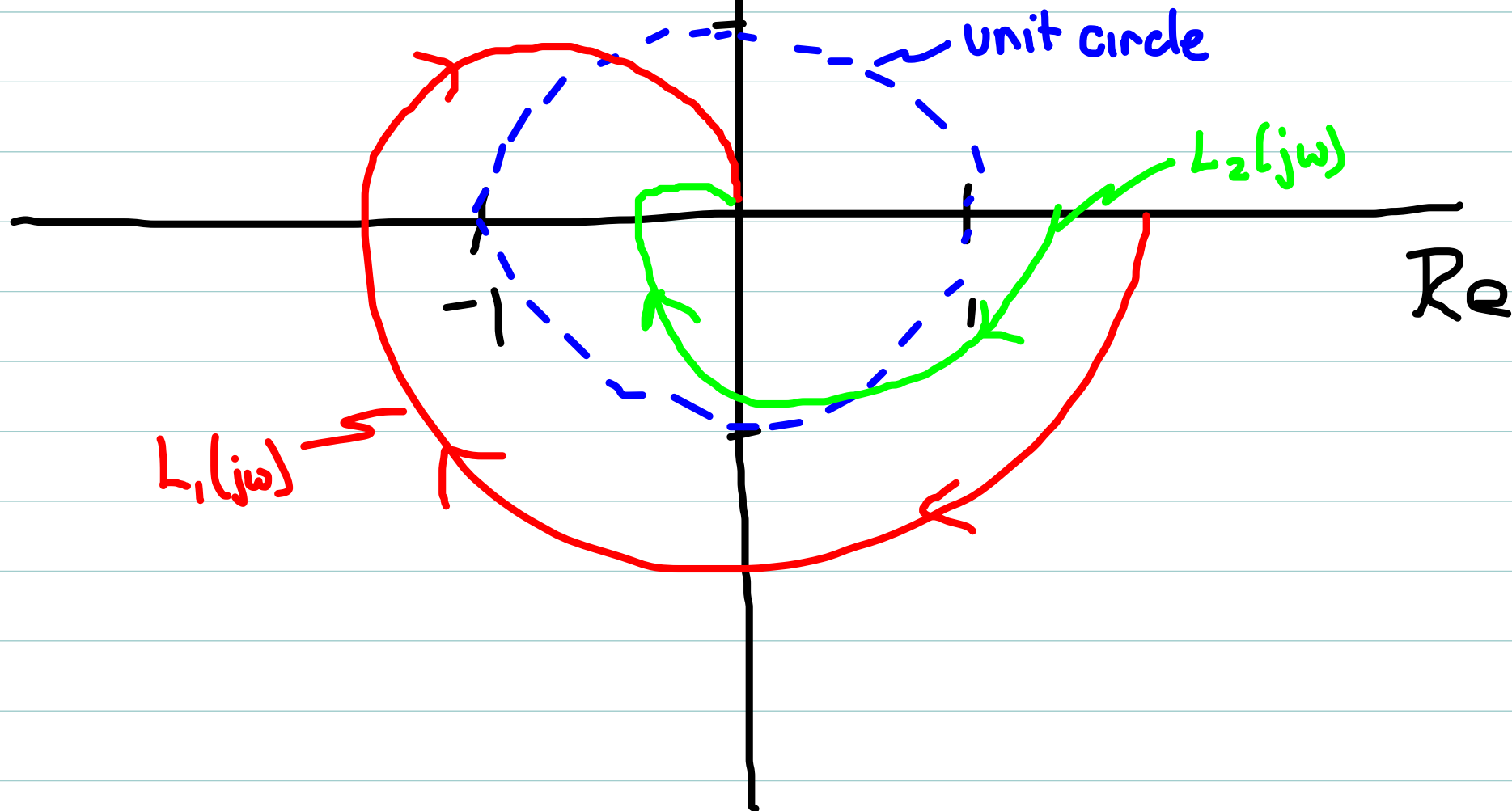
\Rightarrow Gain and phase margins of $L(s)$ \leftarrow !!!
to be large

\Rightarrow polar plot of $L(j\omega)$ avoids -1 by wide margins

Necessary, but not sufficient!

Both plots avoid -1 by
large margins

Is one better?
Yes! But criterion is
un-obvious!



Nyquist Stability Criterion

All roots of $1+L(s)=0$ are in LHP if.

the Nyquist diagram (a modified polar plot) of $L(j\omega)$

circles the -1 point the correct number of times).

\Rightarrow Major Theoretical result! Used extensively in
Control theory

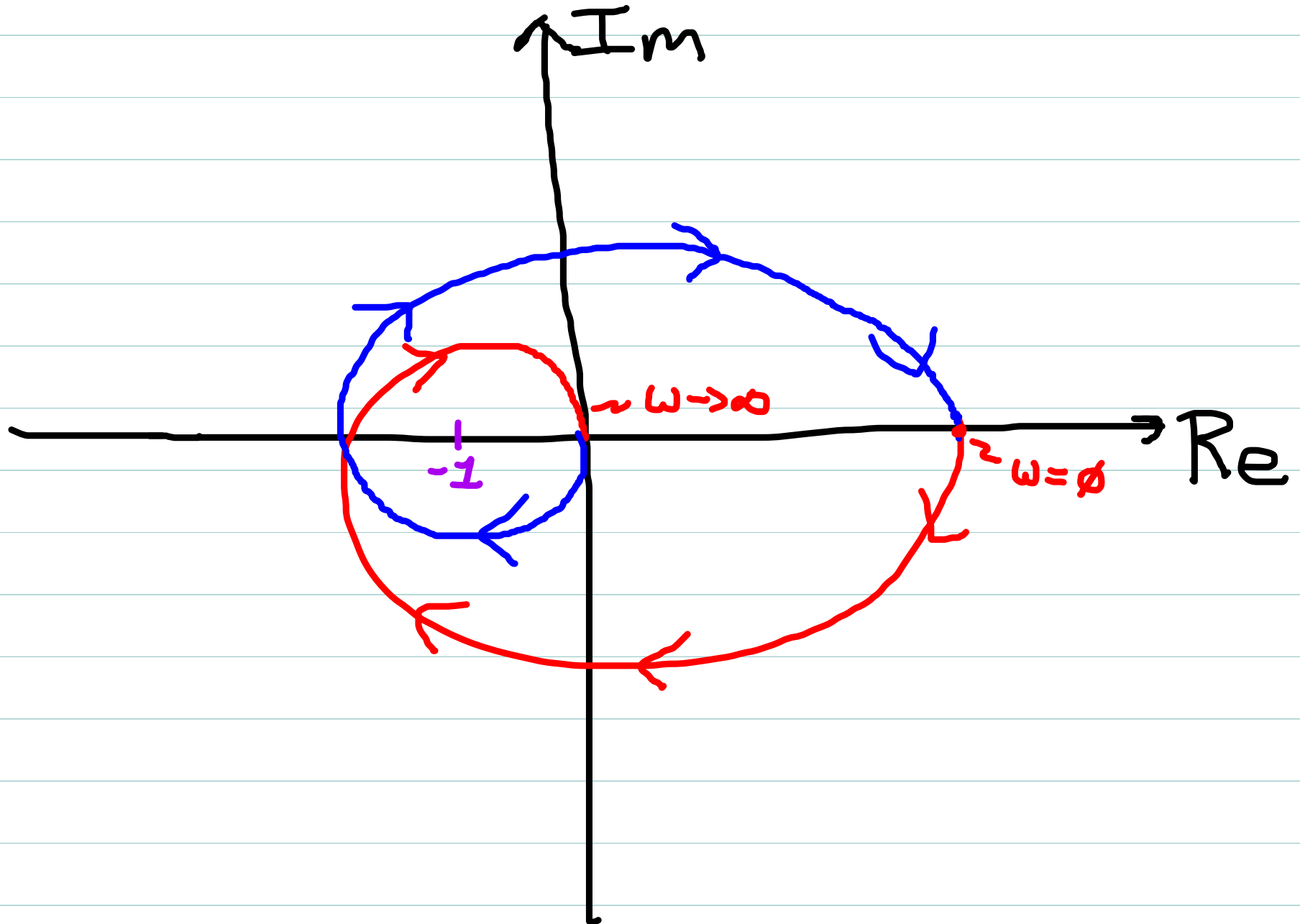
\Rightarrow Questions to answer

\Rightarrow How to create diagram from polar?

\Rightarrow How to count encirclements of -1?

\Rightarrow How many encirclements needed?

Example: $L(s) = \frac{K_D}{(\tau s + 1)^3}$ $K_D, \tau > 0$



Nyquist Diagram

When $L(s)$ is type $N \leq 0$ (no poles at origin)

\Rightarrow Draw polar of $L(j\omega)$

\Rightarrow "Flip" polar of L about real axis
(this is the polar of $L(-j\omega)$, i.e. for negative frequencies)

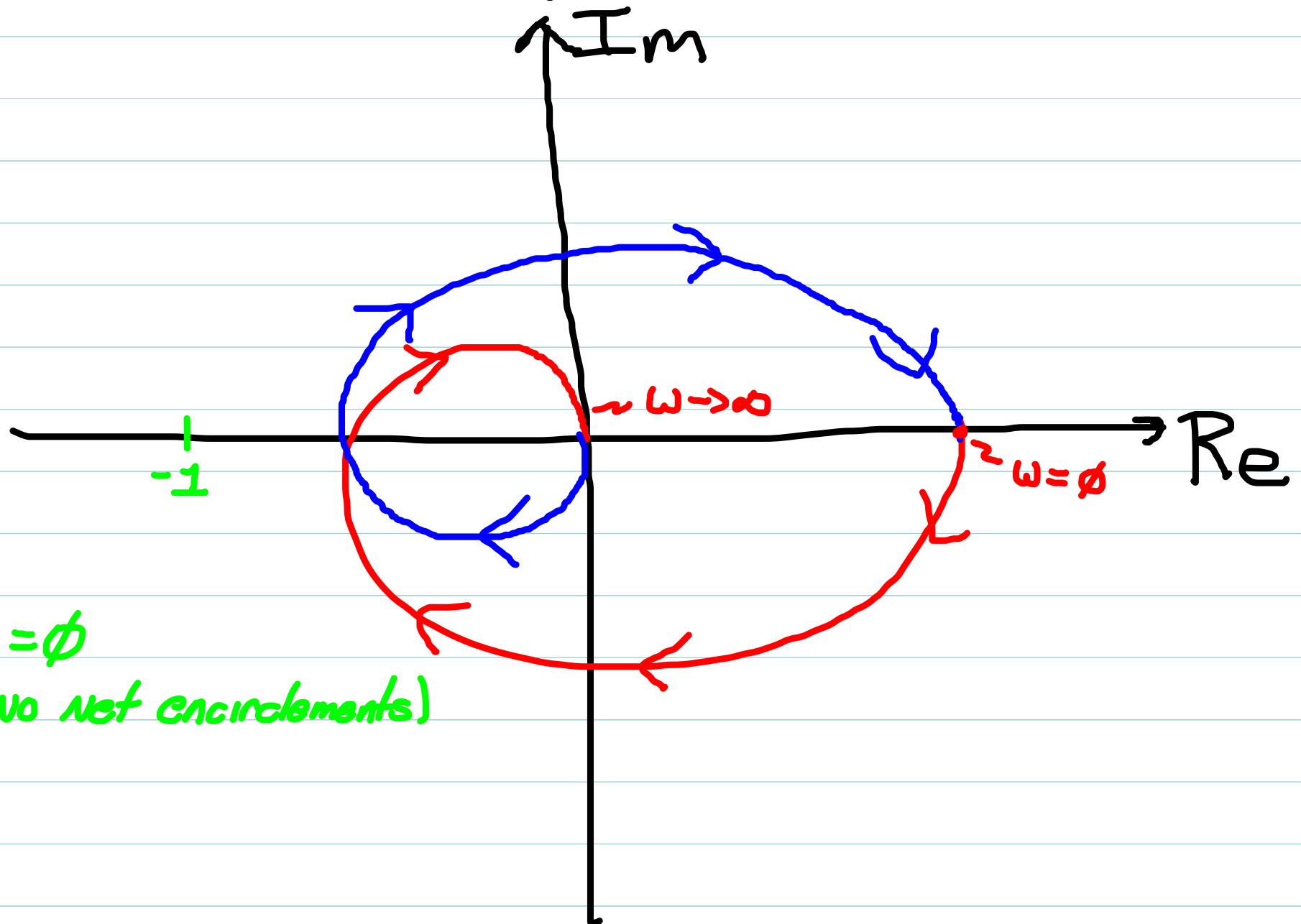
\Rightarrow Put arrows on flipped plot whose direction is consistent with direction of arrows on original polar plot
(i.e. arrows show direction of increasing frequency, from $\omega = -\infty$, through $\omega = 0$, to $\omega = \infty$).

(We will modify for $N > 0$ after we examine complete stability condition.)

Counting Encirclements

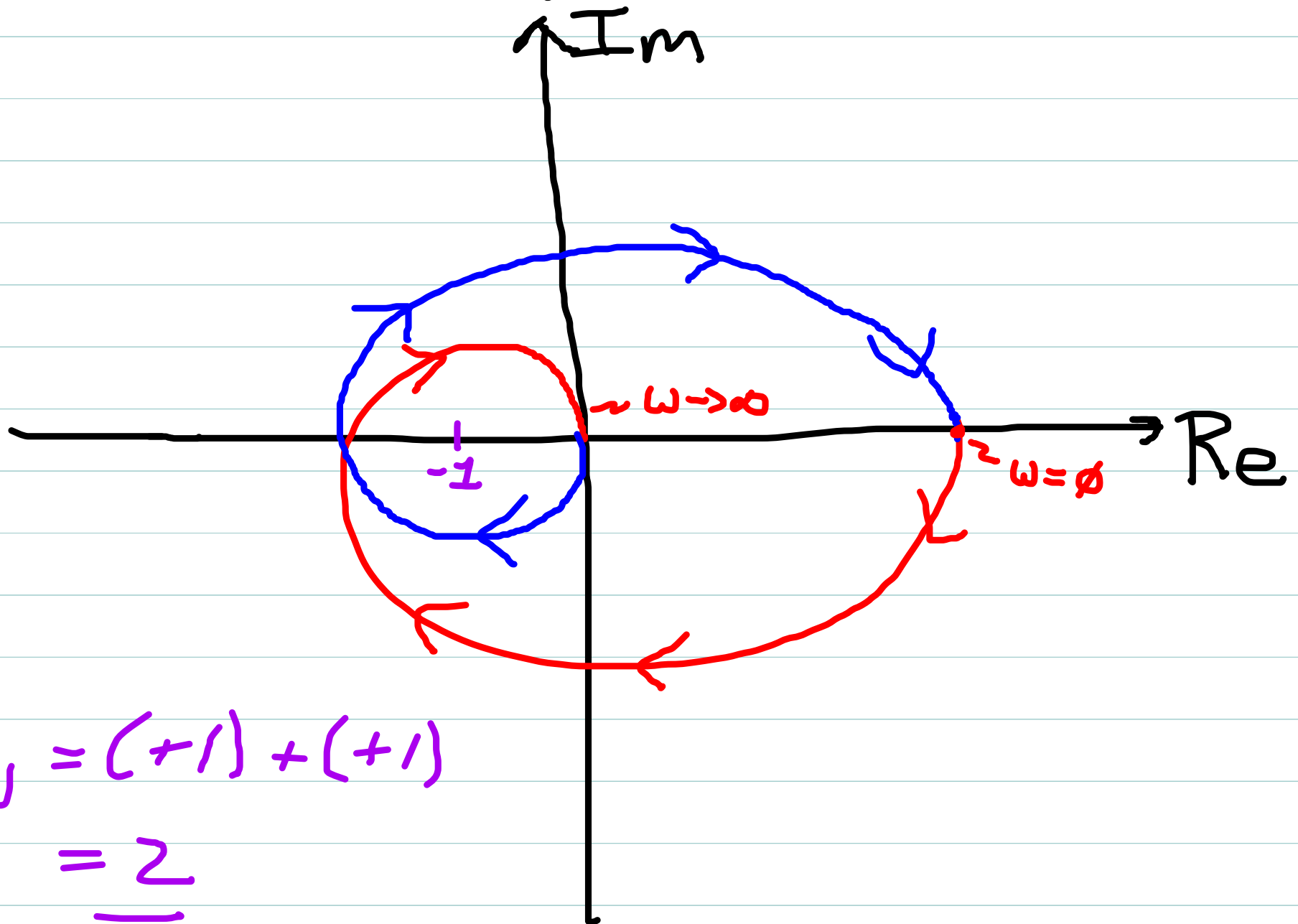
- \Rightarrow Count the number of complete loops the diagram makes around -1.
- \Rightarrow A clockwise loop counts as +1 encirclement
A counter-clockwise loop counts as -1 encirclement
- \Rightarrow Diagrams may have both CW or CCW loops around -1
- \Rightarrow Let $N_{cw}(L)$ be the net number of CW encirclements for Nyquist diagram of L
(i.e. result of adding contribution of each loop using the ± 1 convention above).

Example: $L(s) = \frac{K_B}{(\tau s + 1)^3}$ $K_B, \tau > 0$



$N_{CW} = 0$
(no net encirclements)

Example: $L(s) = \frac{K_B}{(\tau s + 1)^3} \quad K_B, \tau > 0$



$$N_{CW} = (+1) + (+1) \\ = \underline{2}$$