

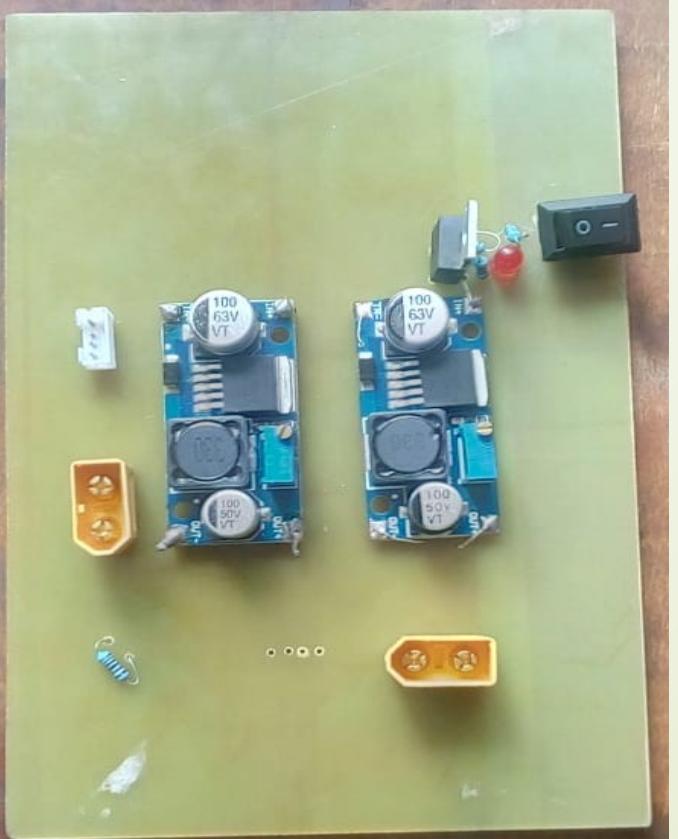


# **LIQUID PROPULSION PROGRESS REPORT**

# TASKS FOR THIS WEEK

- Remake the power board pcb
- Redesign the loadcell holder and design to attach to the engine
- Simulate the current test stand design
- pressure sensor calibration

# Power PCB RE-fabrication



Previous board



Current board



# Solenoid valve test





# Ignition coil PCB test



# Pressure sensor



Wika A-10 pressure transducer

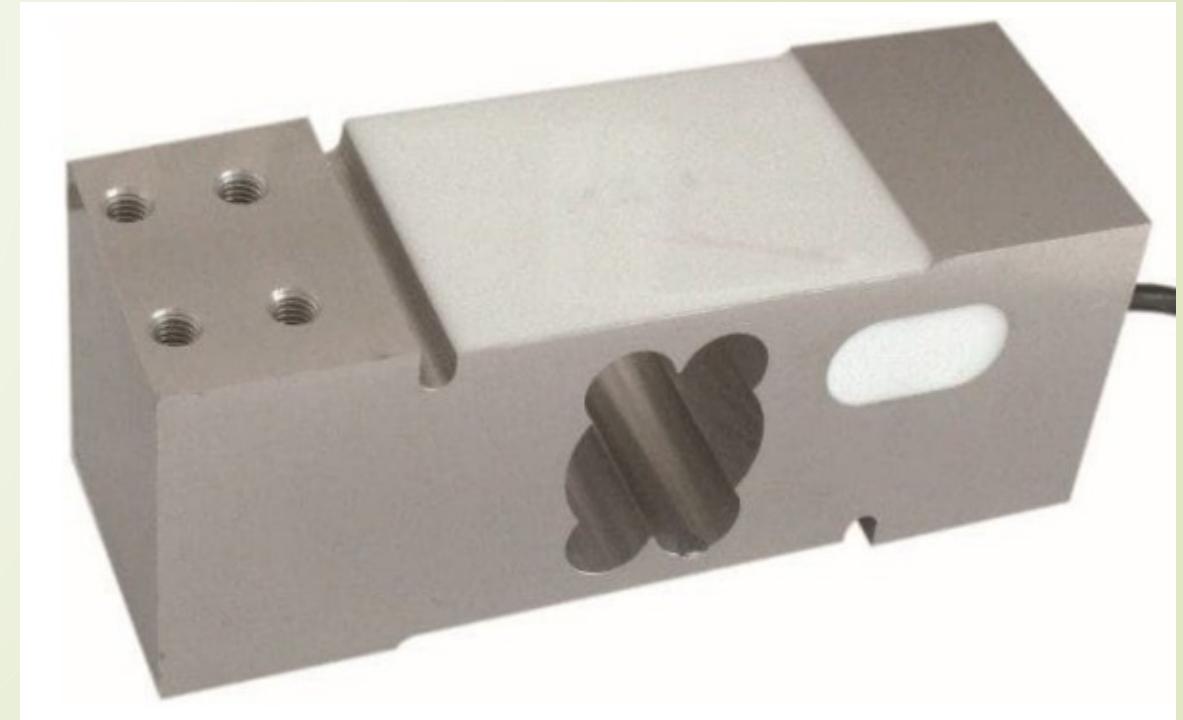


FR-01-752 Pressure transducer

# LOAD CELL TYPE

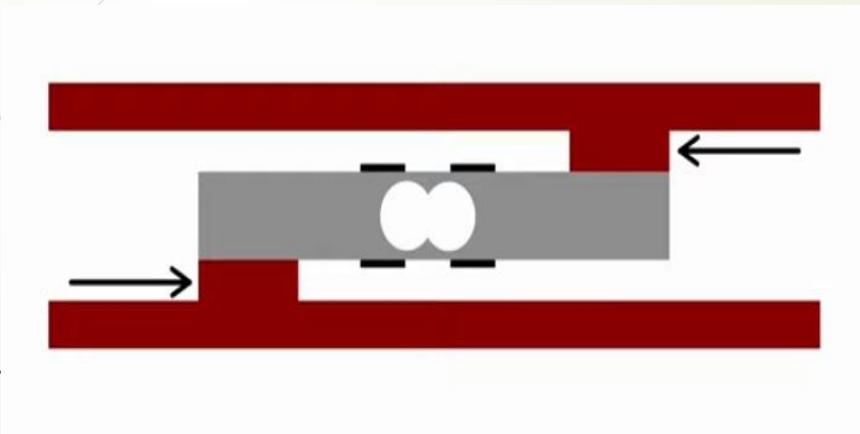


Compressive load cell

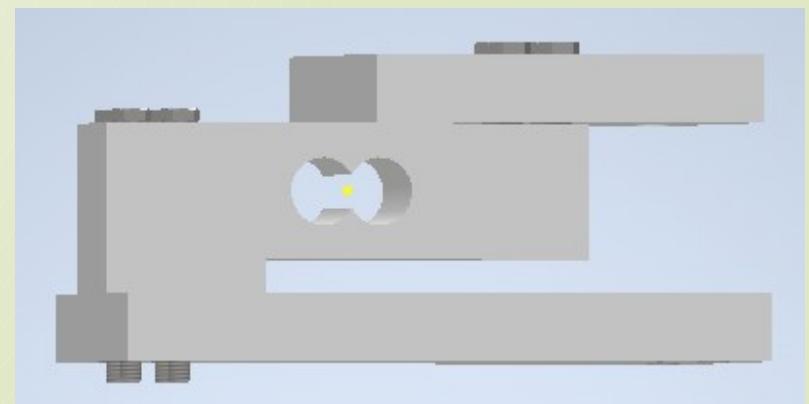
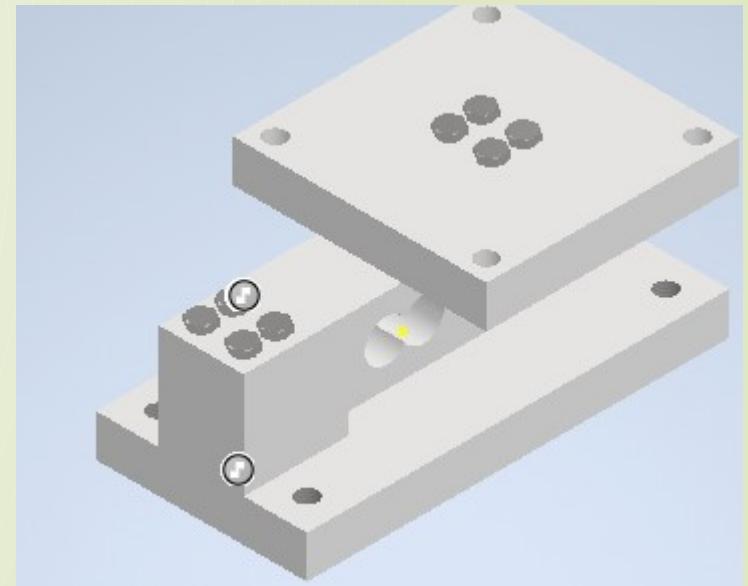


Strain gauge based load cell

# Load cell holder

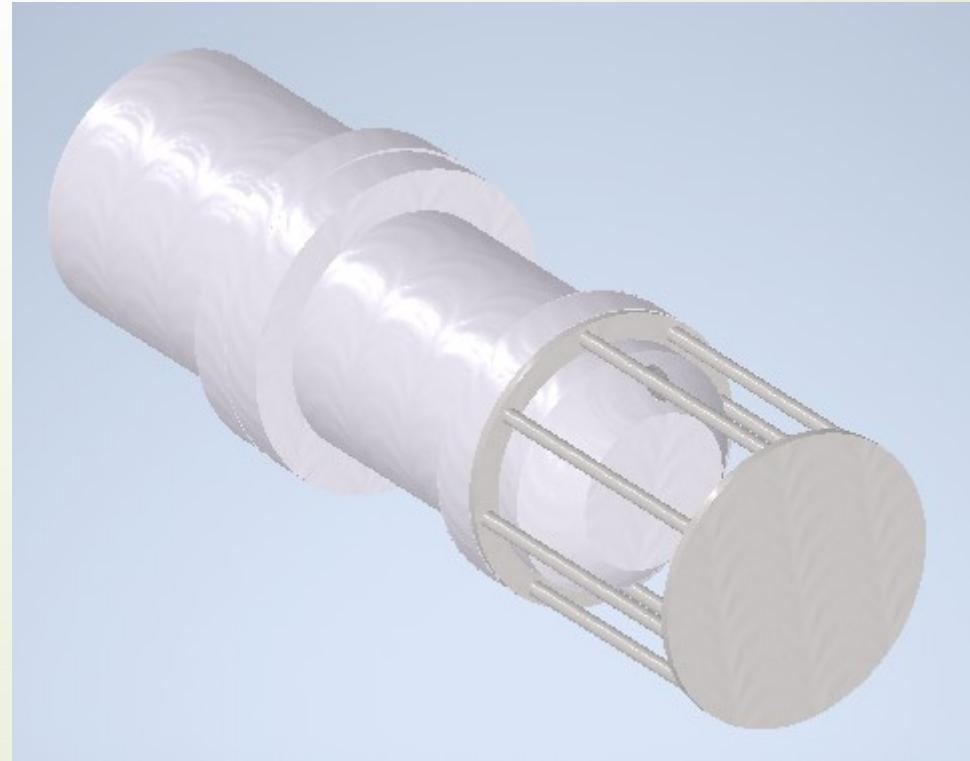


As shown in the image above that's just a sketch on what we are working on.

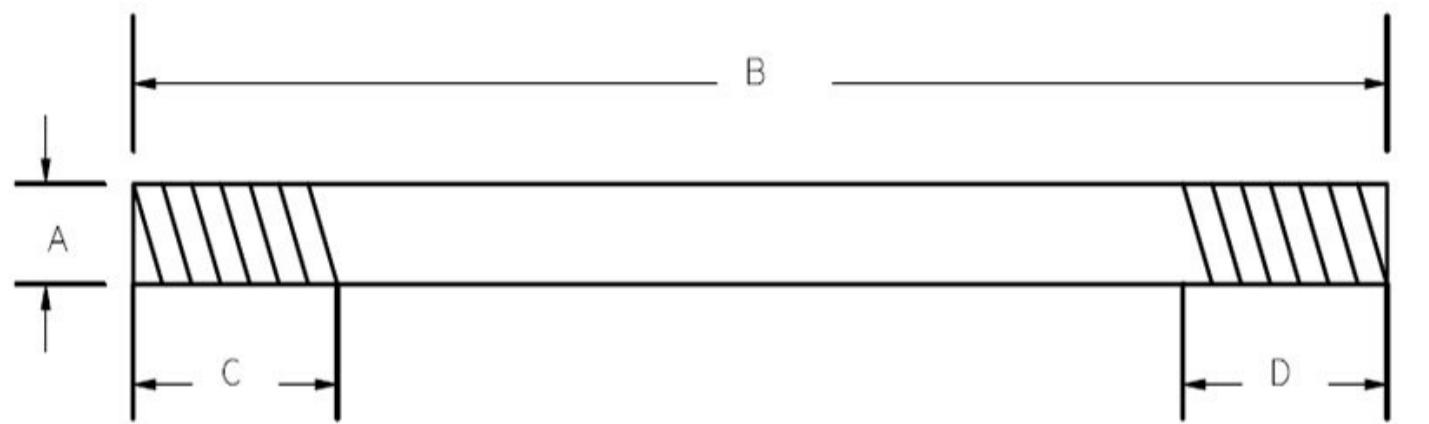


The design we came up with

# Engine load cell connector



# DOUBLE-END THREADED ROD



# Rod strength calculation

## Rod strength calculation

We are M8 double-end threaded rod.

D = 8 mm , F = 2kN, N = 4 rods, Force per rod = 500 N  
yield strength = 250 Mpa  
safety factor = 1.5

stress on each bolt

$$\sigma = \text{Force} / \text{Area}$$

$$A = \pi \times (4\text{mm})^2$$

$$A = \pi \times 16\text{mm}^2 \approx 50.27\text{mm}^2$$

stress per rod

$$\sigma = 500 \text{ N} / 50.27 = 9.94 \text{Mpa}$$

$$\text{Allowable Stress} = 250 \text{MPa} / 1.5$$

$$\text{Allowable Stress} \approx 166.67 \text{MPa}$$

stress on each rod is way below allowable stress.

## Buckling load

To determine if the four steel rods can withstand buckling under a compression load of 2 kN

$$L = 10 \text{ cm} \quad K = 0.5 \quad E = 200 \text{GPa} \quad I = \pi \times (4\text{mm})^4 / 4$$

$$P_{\text{critical}} = \pi^2 \times E \times I / (K \times L)^2$$

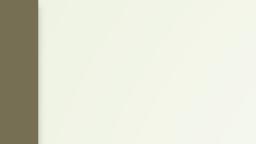
$$P_{\text{critical}} = 157 \text{kN}$$

Applied compression load <  $P_{\text{critical}}$



# Next weeks task

- Storing of pressure sensor data
- Programming and fabricating the control circuit
- Simulate the final design
- If possible buy the frames to fabricate the test stand



# **THANK YOU**