

# AUTOMATIC CONTROL THEORY

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# Chapter 3 Linear system in frequency domain

- 3.1. SISO system
- 3.2. Modeling a nonlinear system by linearization
- 3.3. Typical linear dynamic systems
- 3.4. System stability and stability criteria
- 3.5. Transient and steady- state responses
- 3.6. Overshoot, settling time, steady-state error.
- 3.7. Open-loop controller design
- 3.8. PID controller
- 3.9. PID controller design methods

## 3.1. SISO system

- $T: \mathbf{u}(t) \rightarrow \mathbf{y}(t)$

- $\mathbf{u}(t) = \begin{bmatrix} u_1(t) \\ u_2(t) \\ \vdots \\ u_r(t) \end{bmatrix}; \mathbf{y}(t) = \begin{bmatrix} y_1(t) \\ y_2(t) \\ \vdots \\ y_s(t) \end{bmatrix}$

- System is linear if the superposition holds:

$$T \left\{ \sum_i^n a_i \mathbf{u}_i(t) \right\} = \sum_i^n a_i T \{ \mathbf{u}_i(t) \}$$

## 3.1. SISO system

- SISO system:  $r = s = 1$
- MIMO system:  $r > 1; s > 1$
- MISO system :  $r > 1; s = 1$
- SISO system can be represented as:
  - Differential equations
  - Transfer function  $G(s)$
  - State-space model

## 3.1. SISO system

- *Differential equation*

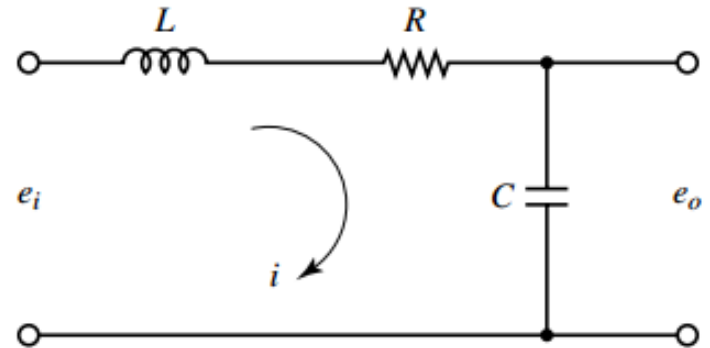
$$a_0 y + a_1 \frac{dy}{dt} + \cdots + a_n \frac{d^n y}{dt^n} = b_0 u + b_1 \frac{du}{dt} + \cdots + b_m \frac{d^m u}{dt^m}$$

- $a_i, b_j$ : constant
- $n > m$  integer number
- $u$  input
- $y$  output

## 3.1. SISO system

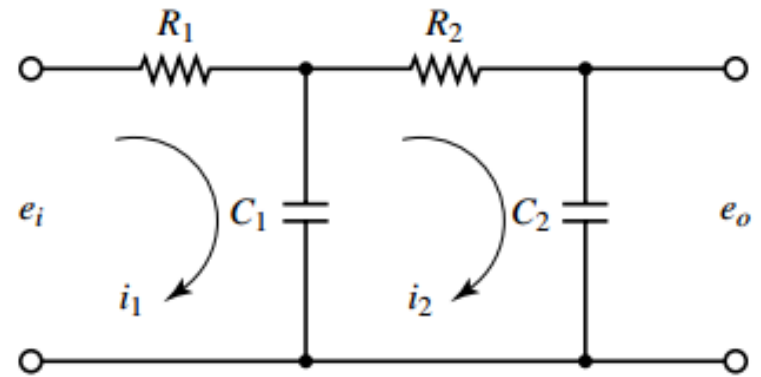
- ***Electrical system: Kirchhoff Law 1 & 2***
  - Sum of currents is zero for a node.
  - Sum of voltages is zero for a loop.
- Resistor:  $u = Ri$
- Capacitor:  $i_C(t) = C \frac{du_C(t)}{dt}$ ;
- Inductor:  $u_L(t) = L \frac{di_L(t)}{dt}$
- $u$  voltage,  $i$  current.

### 3.1. SISO system



$$L \frac{di}{dt} + Ri + \frac{1}{C} \int i dt = e_i$$
$$\frac{1}{C} \int i dt = e_o$$

### 3.1. SISO system



$$\frac{1}{C_1} \int (i_1 - i_2) dt + R_1 i_1 = e_i$$

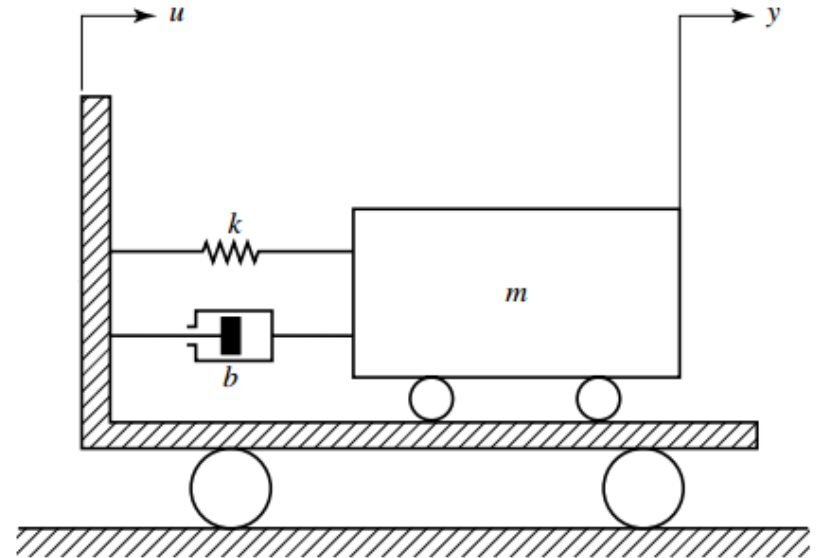
$$\frac{1}{C_1} \int (i_2 - i_1) dt + R_2 i_2 + \frac{1}{C_2} \int i_2 dt = 0$$

$$\frac{1}{C_2} \int i_2 dt = e_o$$



## 3.1. SISO system

- ***Mechanical system:***
- *Newton's second law:*
- $\sum F_i = ma$
- $F_i$  force
- $m$  mass
- $a$  acceleration



$$m \frac{d^2 y}{dt^2} = -b \left( \frac{dy}{dt} - \frac{du}{dt} \right) - k(y - u)$$

$$m \frac{d^2 y}{dt^2} + b \frac{dy}{dt} + ky = b \frac{du}{dt} + ku$$

## 3.2. Modeling a nonlinear system by linearization

$$\begin{aligned}\dot{x} &= f(x, u) \\ y &= g(x)\end{aligned}$$

Working point:

$$\begin{aligned}u &= u_0 \\ f(x_0, u_0) &= 0\end{aligned}$$

$$\dot{z} = Az + Bv$$

$$A = \left. \frac{df}{dx} \right|_{x=x_0, u=u_0} \quad B = \left. \frac{df}{du} \right|_{x=x_0, u=u_0}$$

$$v = u - u_0$$

$$z = x - x_0$$

## 3.2. Modeling a nonlinear system by linearization

$$\begin{aligned}\dot{x} &= -x^3 - x + u \\ y &= x\end{aligned}$$

Working point:

$$\begin{aligned}u &= 0 \\ f(x_0, u_0) &= 0 \\ x_0 &= 0 \\ A &= -1; B = 1\end{aligned}$$

$$\begin{aligned}\dot{z} &= -z + v \\ v &= u - u_0 \\ z &= x - x_0\end{aligned}$$

## 3.3 Typical Linear Systems

- *Unit Impulse:*
- *Impulse Response:*

$$g(t) = L^{-1}\{G(s)\}$$

## 3.3 Typical Linear Systems

- ***Unit step function:***
- ***Unit Step Response:***

$$h(t) = L^{-1} \left\{ \frac{G(s)}{s} \right\}$$

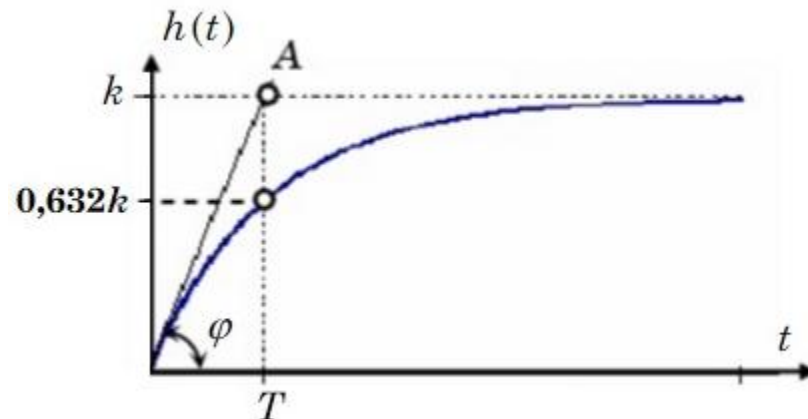
## 3.3 Typical Linear Systems

- ***First-order system:***

$$G(s) = \frac{k}{1 + Ts}$$

- Step response:

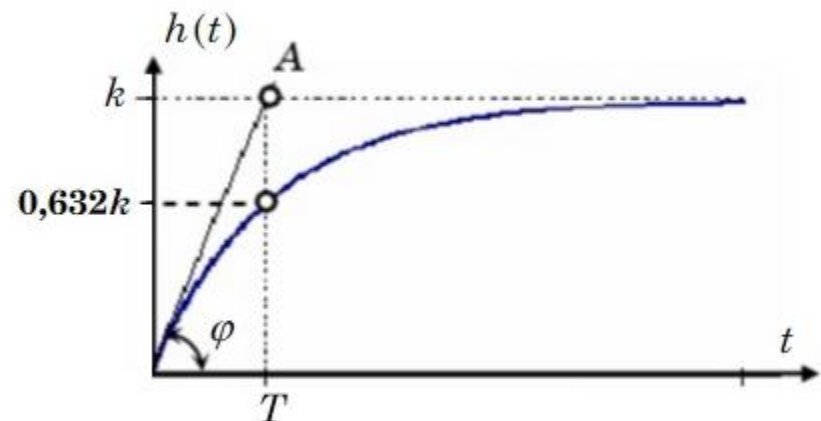
$$h(t) = L^{-1} \left\{ \frac{G(s)}{s} \right\} = k(1 - e^{-\frac{t}{T}})$$



## 3.3 Typical Linear Systems

$$\frac{dh(0)}{dt} = \lim_{s \rightarrow \infty} s[sH(s) - h(0)] = \frac{k}{T} = \tan(\varphi)$$

- Determine  $k$  và  $T$ :
  - Apply  $1(t)$  to the system.
  - $k$  steady state output
  - $T$  t-coordinate of A



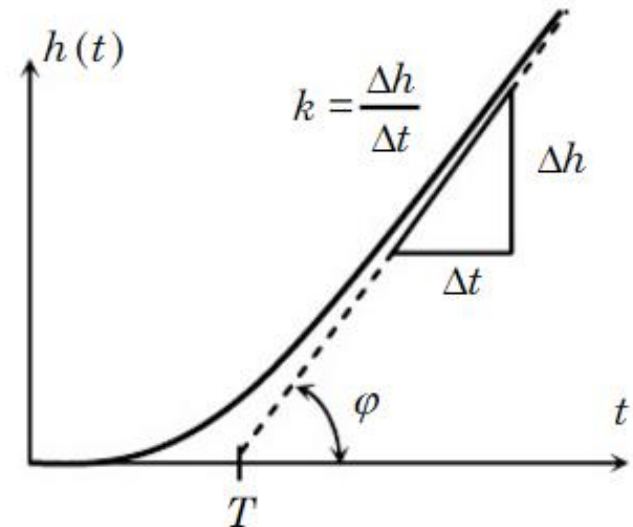
## 3.3 Typical Linear Systems

- ***First-order system with integrator***

$$G(s) = \frac{k}{s(1 + Ts)}$$

$$H(s) = \frac{G(s)}{s} = \frac{k}{s^2(1 + Ts)}$$

$$h(t) = k[t - T(1 - e^{-\frac{t}{T}})]$$

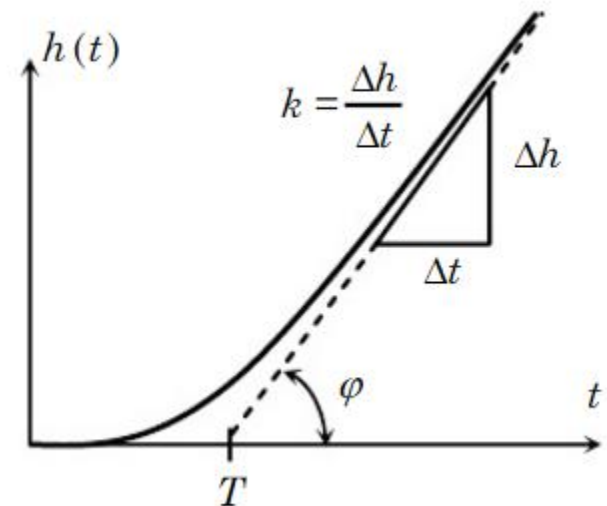




## 3.3 Typical Linear Systems

$$h(t) = k[t - T(1 - e^{-\frac{t}{T}})]$$

- Steady-state,  $h(t) = k(t - T) = h_{tc}(t)$
- $k = \tan(\varphi)$
- Asymptote at  $\infty$ .
- $T$ :  $t$ -coordinate of the intersection between the asymptote and the time axis.
- $k = \tan(\varphi)$ .



## 3.3 Typical Linear Systems

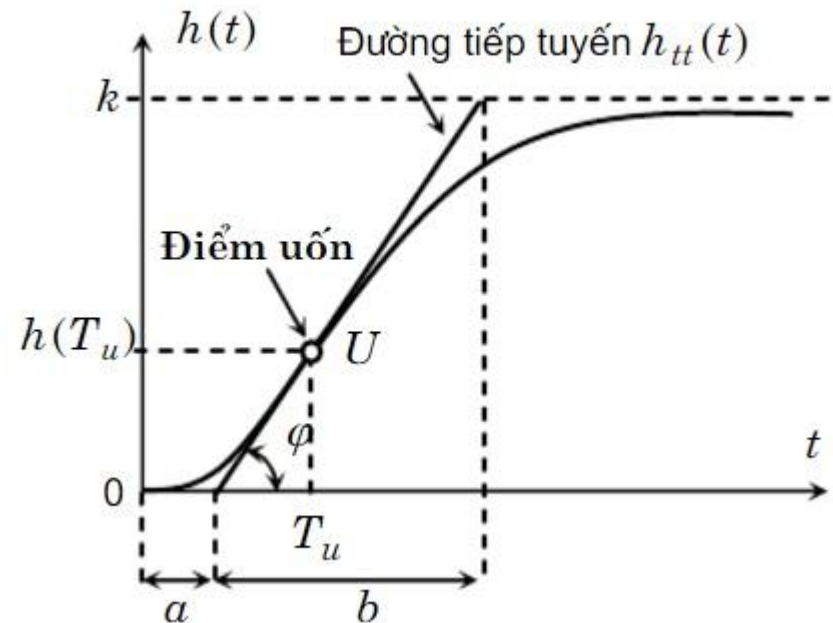
- ***Second-order system***

$$G(s) = \frac{k}{(1+sT_1)(1+sT_2)}, \quad T_2 < T_1, \quad x = \frac{T_2}{T_1}$$

- $H(s) = \frac{k}{s(1+sT_1)(1+sT_2)}$

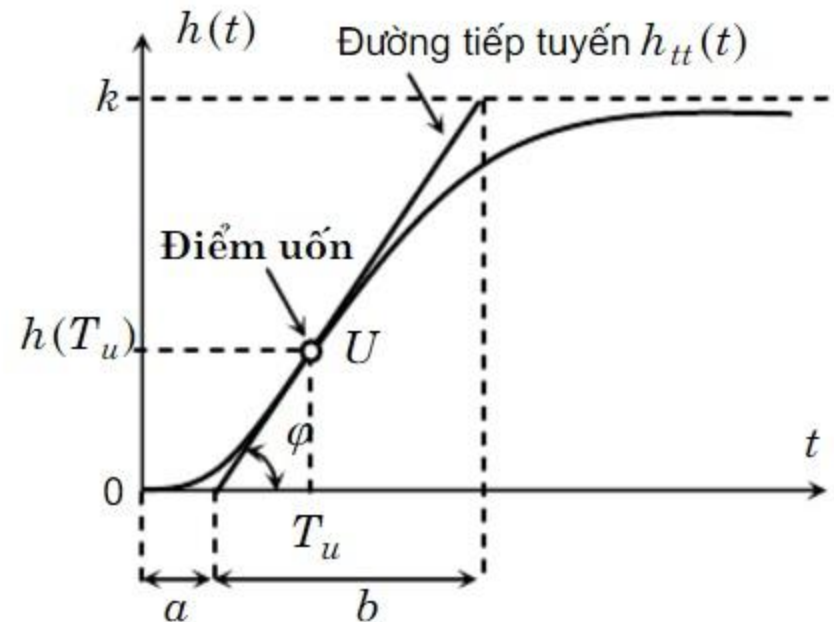
- $h(t) = k \left[ 1 - \frac{T_1 e^{-\frac{t}{T_1}} - T_2 e^{-\frac{t}{T_2}}}{T_1 - T_2} \right]$

- Điểm uốn (inflection point)
- Đường tiếp tuyến (tangent line)



## 3.3 Typical Linear Systems

- $\frac{dh(t)}{dt} = k \frac{e^{-\frac{t}{T_1}} - e^{-\frac{t}{T_2}}}{T_1 - T_2}$
- $\frac{d^2h(t)}{dt^2} = k \left[ \frac{e^{-\frac{t}{T_1}}}{T_1(T_2 - T_1)} + \frac{e^{-\frac{t}{T_2}}}{T_2(T_1 - T_2)} \right]$
- $T_u = \frac{T_1 T_2}{T_2 - T_1} \ln \frac{T_2}{T_1}$
- $v^* = \frac{dh(T_u)}{dt} = \frac{k}{T_1} \left( \frac{T_2}{T_1} \right)^{\frac{T_2}{T_1 - T_2}}$



## 3.3 Typical Linear Systems

- Let  $x = \frac{T_2}{T_1}$ ,  $v^* = \frac{k}{T_1} x^{\frac{x}{1-x}}$
- But  $v^* = \frac{k}{b}$ ,
- So  $\frac{b}{T_1} = x^{\frac{x}{x-1}}$
- $a = T_u - \frac{h(T_u)}{v^*}$

$$\frac{a}{b} = x^{\frac{x}{1-x}} \frac{x \ln x + x^2 - 1}{x - 1} - 1 = g(x)$$

$$x = g^{-1}\left(\frac{a}{b}\right)$$

$$0 < \frac{a}{b} < 0,103$$

## 3.3 Typical Linear Systems

- *Second-order system with oscillation*

$$G(s) = \frac{k}{1 + 2D Ts + T^2 s^2}, \quad 0 < D < 1$$

$$H(s) = \frac{G(s)}{s}$$

$$h(t) = k \left( 1 - \frac{e^{-\frac{D}{T}t}}{\sqrt{1 - D^2}} \sin \left( \frac{\sqrt{1 - D^2}}{T} t + \arccos(D) \right) \right)$$

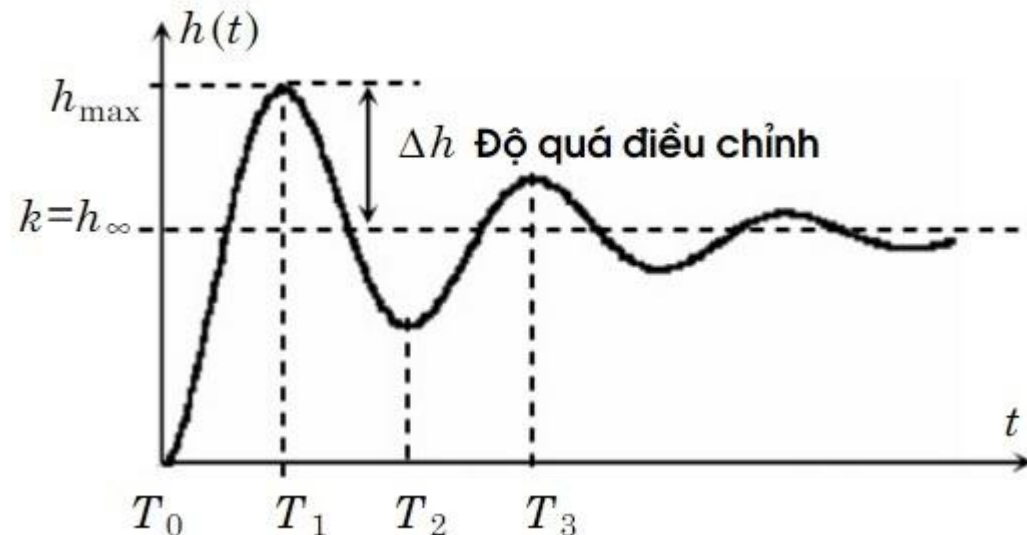
### 3.3 Typical Linear Systems

$$\frac{dh(t)}{dt} = k \frac{e^{-\frac{D}{T}t}}{T\sqrt{1-D^2}} \sin\left(\frac{\sqrt{1-D^2}}{T}t\right); T_i = \frac{i\pi T}{\sqrt{1-D^2}}$$

$$h_{max} = h(T_1) = k\left[1 + e^{-\frac{\pi D}{\sqrt{1-D^2}}}\right]$$

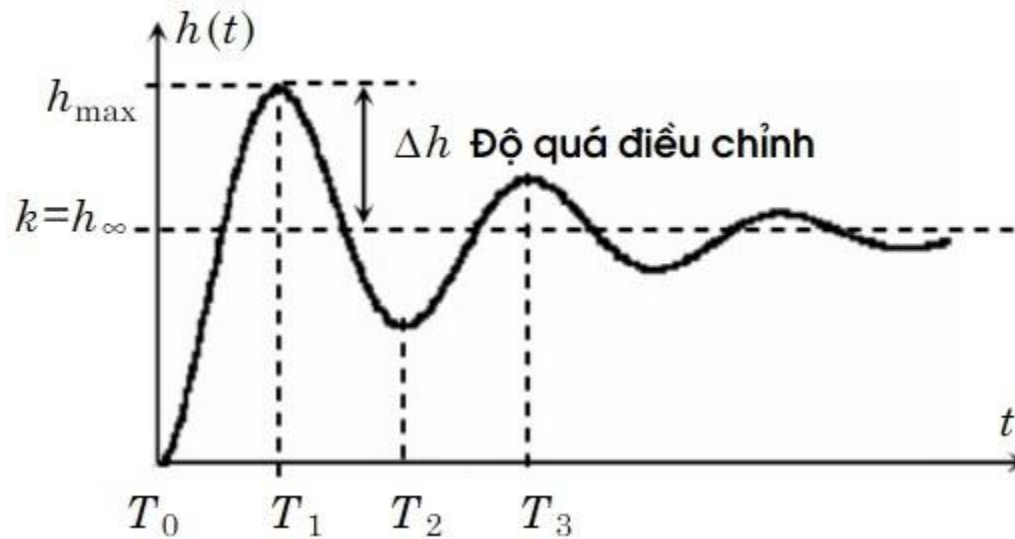
$$\bullet \Delta h = h_{max} - k = ke^{-\frac{\pi D}{\sqrt{1-D^2}}}$$

$$\bullet D = \frac{1}{\sqrt{1 + \frac{\pi^2}{\ln^2 \frac{\Delta h}{k}}}}$$



## 3.3 Typical Linear Systems

$$T_1 = \frac{\pi T}{\sqrt{1 - D^2}}$$
$$T = \frac{T_1 \sqrt{1 - D^2}}{\pi}$$



## 3.3 Typical Linear Systems

*Frequency characteristics*

$$G(s) = \frac{Y(s)}{X(s)} = \frac{b_0 + b_1s + \cdots + b_ms^m}{a_0 + a_1s + \cdots + s^n} \quad (m \leq n)$$

$$G(j\omega) = G(s) \Big|_{s=j\omega}$$

$$G(s) = \frac{1}{1+s}$$

$$G(j\omega) = \frac{1}{1+j\omega} = \frac{1}{1+\omega^2} - \frac{j\omega}{1+\omega^2}$$



### 3.3 Typical Linear Systems

$$\begin{aligned} \operatorname{Re}\{G(j\omega)\} &= \frac{1}{1 + \omega^2} \rightarrow \operatorname{Re} \geq 0 \\ \operatorname{Im}\{G(j\omega)\} &= \frac{-\omega}{1 + \omega^2} \rightarrow \operatorname{Im} \leq 0 \\ \left(\operatorname{Re} - \frac{1}{2}\right)^2 + \operatorname{Im}^2 &= \frac{1}{4} \end{aligned}$$

## 3.3 Typical Linear Systems

- *Bode plot*

$$L(\omega) = 20\lg |G(j\omega)| - \text{biên tần}$$

$$\varphi(\omega) = \arctan \left( \frac{\text{Im}\{G_f(j\omega)\}}{\text{Re}\{G_f(j\omega)\}} \right)$$

- Gain:  $G(s) = k, k > 0$

- First-order system:  $G(s) = \frac{k}{1+Ts}$

## 3.3 Typical Linear Systems

- PD:  $G(s) = 1 + Ts$

- I:  $G(s) = \frac{1}{T_I s}$

- D:  $G(s) = T_D s$

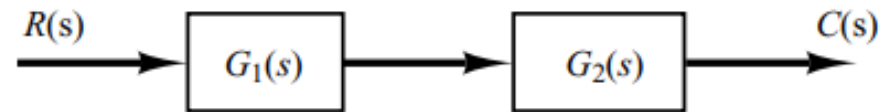
- Second-order system:

$$G(s) = \frac{k}{1 + 2DTs + T^2s^2}, \quad 0 < D < 1$$

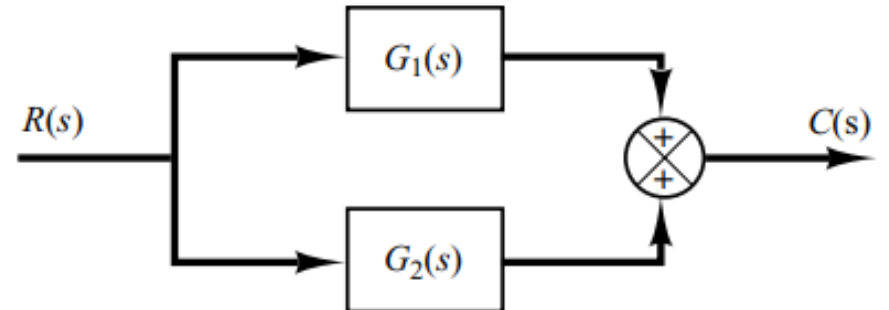
## 3.3 Typical Linear Systems

- *Conversion of diagram:*

- ☐ Series



- ☐ Parallel



- ☐ Feedback

