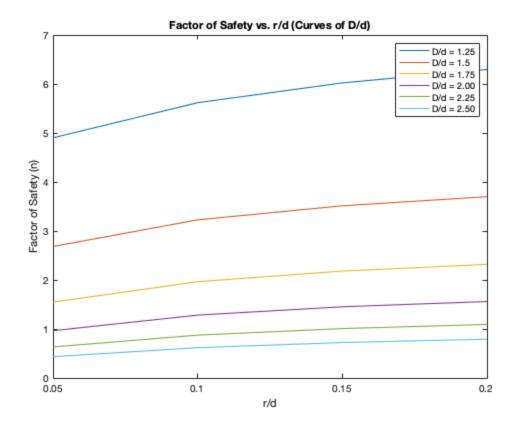
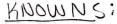
```
% NASH ELDER %
% ME 328 DESIGN I, DR. DAVOL %
% FATIGUE %
% Equations and recommended values from Shigley's 9th ed. %
% MATERIAL = STEEL %
clear;
Sy = 60.2; % ksi %
Sut = 95; % ksi %
Pa = 0.2; % kip %
Pm = 0.8; % kip %
P = 1;
Dbiq = 3.5; % in. %
L = 15; % in. %
a = 8; % in. %
aSurf = 2.70; % Finish Factor a (machined finish) %
bSurf = -0.265; % Finish Factor b (machined finish) %
SePrime = 0.5*Sut; % ksi -- For Sut <= 200 %
for i = 1:6
    Doverd = 1.25 + 0.25*(i-1);
    d = (Dbig/Doverd);
    for j = 1:4
        roverd = 0.05 + 0.05*(j-1);
        r = (roverd * d);
       Ka = aSurf*Sut^bSurf;
        if d <= 2.0
            Kb = 0.879*(0.370*d)^(-0.107); % Size Factor -- small
 radius %
        else
            Kb = 0.91*(0.370*d)^(-0.157); % Size Factor -- large
 radius %
        end
        Kc = 1; % Load Factor -- Bending Load %
        Kd = 1; % Temp Factor -- Temp = 20 deg. C %
        Za = 1.288; % Transformation Variate for 90% (TBL 6-5) %
        Ke = 1 - 0.08*Za; % Reliability Factor %
```

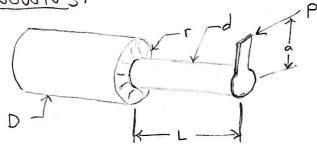
```
Se = SePrime * Ka * Kb * Kc * Kd * Ke;
        % Kt due to bending and torsion (ref: AMESWEB.INFO) %
        h = (Dbig-d)/2;
        if 0.1 <= h/r <= 2.0
            c1b = 0.947 + 1.206*(sqrt(h/r)) - 0.131*h/r;
            c2b = 0.022-3.405*(sqrt(h/r))+0.915*h/r;
            c3b = 0.869+1.777*(sqrt(h/r))-0.555*h/r;
            c4b = -0.810 + 0.422*(sqrt(h/r)) - 0.260*h/r;
        elseif 2.0 < h/r <= 20.0
            c1b = 1.232 + 0.832*(sqrt(h/r)) - 0.008*h/r;
            c2b = -3.813 + 0.968*(sqrt(h/r)) - 0.260*h/r;
            c3b = 7.423-4.868*(sqrt(h/r))+0.869*h/r;
            c4b = -3.839+3.070*(sqrt(h/r))-0.600*h/r;
        else
            c1b = NaN; c2b = NaN; c3b = NaN; c4b = NaN;
        end
        Kt = c1b + c2b*(2*h/Dbig) + c3b*(2*h/Dbig)^2 + c4b*(2*h/Dbig)
Dbiq)^3;
        if 0.25 <= h/r <= 4.0
            c1t = 0.905+0.783*(sqrt(h/r))-0.075*h/r;
            c2t = -0.437-1.969*(sqrt(h/r))+0.553*h/r;
            c3t = 1.557+1.073*(sqrt(h/r))-0.578*h/r;
            c4t = -1.061+0.171*(sqrt(h/r))+0.086*h/r;
        else
            c1t = NaN; c2t = NaN; c3t = NaN; c4t = NaN;
        end
        Kts = c1t + c2t*(2*h/Dbig) + c3t*(2*h/Dbig)^2 + c4t*(2*h/Dbig)
Dbiq)^3;
        % q calculation to find Kf %
        sqrtA = 0.246-3.08*10^{(-3)}*Sut
+1.51*10^(-5)*Sut^2-2.67*10^(-8)*Sut^3;
        sqrtA_tor = 0.190-2.51*10^{(-3)}*Sut
+1.35*10^(-5)*Sut^2-2.678*10^(-8)*Sut^3;
        q = 1/(1+sqrtA/sqrt(r));
        qs = 1/(1+sqrtA_tor/sqrt(r));
        Kf = q * (Kt-1) + 1;
        Kfs = qs * (Kts-1) + 1;
        % Sigma calculation %
        Ma = Pa * L;
        Mm = Pm * L;
        Ta = Pa * a;
```

```
Tm = Pm * a;
        I = pi/4*(d/2)^4;
        J = pi/2*(d/2)^4;
        sig_a_bend = Ma*(d/2)/I;
        sig_m_bend = Mm*(d/2)/I;
        tau_a = Ta*(d/2)/J;
        tau m = Tm*(d/2)/J;
        sig_prime_a = ((Kf*sig_a_bend)^2 + 3*(Kfs*tau_a)^2)^(0.5);
        sig_prime_m = ((Kf*sig_m_bend)^2 + 3*(Kfs*tau_m)^2)^(0.5);
        sig prime max = sig m bend + sig a bend;
        tau_prime_max = tau_a + tau_m;
        % Safety factors %
        n_fatigue = 1/((sig_prime_a/Se) + (sig_prime_m/Sut));% Safety
 factor for fatigue %
       n_SF = Sy / sig_prime_max; % Safety factor Yield / Allowable %
       n = min(n_fatigue, n_SF);
        SF(j,i) = n; % Allocation of safety factors on each iteration
 (for plot) %
    end
end
disp('4x6 Vector from 6 different D/d values and 4 different r/d
 values:');
disp(SF);
plot(SF);
xlabel('r/d');
ylabel('Factor of Safety (n)');
legend('D/d = 1.25', 'D/d = 1.5', 'D/d = 1.75', 'D/d = 2.00', 'D/d =
 2.25', 'D/d = 2.50');
set(gca, 'XTick', 0:1:4);
set(gca, 'XTickLabel', [0.20, 0.05, 0.10, 0.15]);
title('Factor of Safety vs. r/d (Curves of D/d)');
4x6 Vector from 6 different D/d values and 4 different r/d values:
    4.9016
             2.6835 1.5490
                                  0.9650
                                            0.6333 0.4350
    5.6184
             3.2275
                       1.9651
                                  1.2834
                                            0.8750
                                                      0.6191
    6.0222
             3.5146
                        2.1822
                                  1.4534
                                            1.0085
                                                      0.7242
    6.2969
             3.7004
                       2.3191
                                 1.5602
                                           1.0931
                                                     0.7918
```



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CRITICAL FAILURE AT THE NOTCH RADIUS r

## OBJECTIVE:

AID SHAFT DESIGN BY GENERATING CURVES SHOWING EFFECT ON SAFETY FACTOR WITH VARYING NOTCH RADIUS, r. AND SMALLER DIAMETER, d.

#### INPUTS:

· Pa - ALTERNATING LOAD

· Pm - MEAN LOAD

MATERIAL: STEEL

Sy = 180 Ksi

Su7 = 210 Ksi

VARYING DIAMETERS:

DId = 1.25, 1.5, 1.75, 2.0, 2.25, 2.5

VARYING RADII:

 $\Gamma/d = 0.05, 0.10, 0.15, 0.20$ 

### MODIFYING FACTORS:

Se = ENDURANCE LIMIT AT CRITICAL LOCATION.

Se = Se' Kakbkckd Keke

Ka = SURFACE CONDITION M.F.

Ka = a Sut WHERE 9 AND 6 FROM TBL 6-2

FOR TEST CASE, ASSUME GROUND SURFACE, a = 1.34, b = -0.085

 $K_{\alpha} = (1.34)(210 \text{ ksi})^{-0.085}$ 

Ka = 0.8506

Kb = SIZE M.F.

 $K_{b} = \begin{cases} 0.879 d_{e}^{-0.107}, & 0.11 \le d \le 2 \text{ in} \\ 0.91 d_{e}^{-0.157}, & 2 \le d \le 10 \text{ in} \end{cases}$ 

de = 0.370 d (FROM TBL 6-3)

Ko [SMALLRADIUS] = 0.879 (0.3704) 0.107

Kb [LAGGE RADIUS] = 0.91(0,370d) -0.157

FOR TEST CASE, d = 2in

Kbs = 0.879 (0.370 (2)) -0.107

Kbs = 0.9078

#### MODIFYING FACTORS

KC = LOADING M.F.

FOR THE BENDING | Kc=1

OF ON THE DEMAND SIDE.

Kd = TEMPERATURE M.F.

Ke = RELIABILITY M.F.

Zg = 3.091 FOR Re = 99.91. (TBL 6-5)

FOR TEST CASE :

\$ 0.753 (AGREES WITH TBL 6-5)

NOTCH SENSITIVITY, FACTOR KE

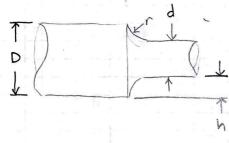
KE EQUATION FROM AMESWEB. INFO

BENDING :

$$K_{t} = C_{1} + C_{2}(2h|D) + C_{3}(2h|D)^{2} + C_{4}(2h|D)^{3}$$

TORSION:

$$H_{t_s} = C_1 + C_2(2h|0) + C_3(2h|0)^2 + C_4(2h|0)^3$$
  
CONSTANTS DIFFER (SEE REFERENCE SHEET)



$$h = \frac{D-d}{2}$$

FOR 
$$D = 2$$
 in ,  $\frac{D}{1} = 1.25$ :

$$h = (2 - 1.6) in$$

$$h = 0.2 in$$

$$\frac{r}{d} = 0.05$$

$$Y = 0.05 (1.6 in)$$

$$r = 0.08 in$$

SAMPLE CALCS FOR NOTCH SENSITIVITY

D= 2in, D/d = 1.25, d=1.6in, h= 0.2in, r=0.05

r= 0.08in BENDING CASE

 $h/r = \frac{0.2in}{0.09in}$ 

h/r = 2.5  $\Rightarrow$  2.0 \le 2.5 \le 20.0

C, = 1.232 + 0.832 N/ -0.008 h/r

= 1.232+0.832 -0.008(2.5)

= 2.528

C2 = -3.813 +0.968 1/7 -0.260 h/r

= -3.813+0.968/2.5-0,260(2.5)

= -2.932

C3 = 7.423-4.8681/1/r + 0.8691/r

= 7.423 - 4.868 \( 2.5 + 0.869 (2.5)

= 1.899

C4 = -3.839 + 9.070 V/ -0.600 h/r

= -3,839 + 3,070 (2,5 - 0,600 (2,5)

= -0.485

 $K_{E} = C_{1} + C_{2}(2h|D) + C_{3}(2h|D)^{2} + C_{4}(2h|D)^{3}$ 

Kt = 2.014

WHERE 2h/D = 2 (0.2 in/2 in) = 0.2

# SAMPLE CALCS FOR NOTCH SENSITIVITY

# TORSION CASE

$$= 2.042$$

$$= -0.437 - 1.769\sqrt{2.5} + 0.553(2.5)$$

VALUES FOR Kt AND Kts ARE CONSISTENT WITH MATLAB CODE

7/11

SAMPLE CALCS FOR NOTCH SENSITIVITY

$$\sqrt{a}$$
 BENDING = 0.246 - 3.08(10<sup>-3</sup>) Sur + 1.51(10<sup>-5</sup>) Sur   
- 2,47(10<sup>-8</sup>) Sur

WHERE SUT = 210 KSC

$$\sqrt{a_{702510N}} = 0.190 - 2.51(10^{-3})S_{u7} + 1.35(10^{-5})S_{u7}^{2}$$
  
-2.67(10^8)Su7

$$= \frac{1}{1 + \frac{0.0178}{\sqrt{0.08}}}$$
$$= 0.9408$$

$$= 0.9408$$
 $K_{f} = 2(K_{f} - 1) + 1$ 

$$\left( K_{f}=1.954\right)$$

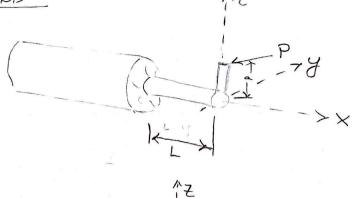
$$g_s = \frac{1}{1 + \sqrt{a_{7025100}}}$$

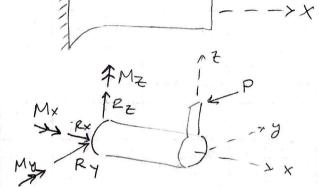
$$= \frac{1}{1 + 0.0110}$$

$$\sqrt{0.08}$$

 $Se' = \begin{cases} 0.5 \, S_{0.7} & S_{0.7} \leq 200 \, \text{ksi} \\ 100 \, \text{ksi} & S_{0.7} > 200 \, \text{ksi} \end{cases}$ IN THIS CASE,  $S_{0.7} = 210 \, \text{ksi}$   $Se' = 100 \, \text{ksi}$ 

FBD





$$D \leq M_{X} = M_{X} + P(a) = 0$$

$$M_{X} = -P_{c} \qquad \longleftarrow Torcsion!$$

$$D \leq M_{Z} = M_{Z} - P(L) = 0$$

$$\Re \sum M_z = M_z - P(L) = 0$$
 $\Re \sum P(L) \leftarrow BENDING$ 

$$\frac{\partial}{\partial x} = \frac{MC}{T}$$

$$= \frac{P(L)(\frac{d}{2})}{T}$$

$$= T(\frac{d}{2})$$

$$\tau = \frac{T(\frac{d}{2})}{J}$$

$$J_{\text{SMALL}} = \frac{\pi}{2} \left( \frac{d}{2} \right)^4$$

$$T_a = (20016)(8in)$$

FOR d=Zin

122	
ending	
CHUININ	



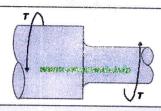
$0.1 \le h/r \le 2.0$	$2.0 \le h/r \le 20.0$
$C1 = 0.947 + 1.206\sqrt{h/r} - 0.131h/r$	$C1 = 1.232 + 0.832\sqrt{h/r} - 0.008h/r$
$C2 = 0.022 - 3.405 \sqrt{h/r} + 0.915 h/r$	$C2 = -3.813 + 0.968\sqrt{h/r} - 0.260h/r$
$C3 = 0.869 + 1.777 \sqrt{\overline{h/r}} - 0.555 h/r$	$C3 = 7.423 - 4.868\sqrt{h/r} + 0.869h/r$
$C4 = -0.810 + 0.422\sqrt{h/r} - 0.260h/r$	$C4 = -3.839 + 3.070\sqrt{h/r} - 0.600h/r$

$$K_t = C_1 + C_2(2h/D) + C_3(2h/D)^2 + C_4(2h/D)^3$$

$$\sigma_{nom} = 32M/\pi d^3$$

$$\sigma_{max} = K_t \sigma_{nom}$$

#### Torsion



$$0.25 \leq h/r \leq 4.0$$

$$C_1 = 0.905 + 0.783 \sqrt{h/r} - 0.075 h/r$$

$$C_2 = -0.437 - 1.969 \sqrt{h/r} + 0.553 h/r$$

$$C_3 = 1.557 + 1.073\sqrt{h/r} - 0.578h/r$$

$$C_4 = -1.061 + 0.171 \sqrt{h/r} + 0.086 h/r$$

$$K_t = C_1 + C_2(2h/D) + C_3(2h/D)^2 + C_4(2h/D)^3$$

$$\tau_{nom}=16T/\pi d^3$$

$$\tau_{max} = K_t \tau_{nom}$$