

Chapter 1

Perturbative Effective Slip Lengths

We derived an expression for the effective slip length using the homogenization technique; this is presented in the previous chapter. However, prior to doing this, we derived an expression for the effective slip length using a different technique – the method of perturbation. The homogenized expression for b_{eff} holds for a non-flat surface, while the perturbative expression assumes a flat surface. The derivation of b_{eff} by perturbation methods is presented here.

We model the fluid system as incompressible, Stokes ‘creeping’ flow, with velocity vector $\vec{u} = (u, v, w)$:

$$\nabla^2 \vec{u} = \frac{1}{\mu} \nabla p \quad (1.1)$$

$$\nabla \cdot \vec{u} = 0 \quad (1.2)$$

The bottom solid surface is modeled as the $z = 0$ plane. The surface is **flat**, so simple Navier slip holds:

$$u(0) = b(x, y) \frac{\partial u}{\partial z} \Big|_{z=0} \quad (1.3)$$

The intrinsic slip length of the surface $b(x, y)$ is a rectangular-periodic function, with period L in the x direction.

The flow is Couette-like, shear-driven by a driving plate at height P above the surface, with a constant driving velocity of u_P .

$$\vec{u}(x, y, P) = (u_P, 0, 0) \quad (1.4)$$

Since flow is generally in the x direction, driven by shear only, there is no pressure gradient, and the pressure has the same x -periodicity as the surface:

$$p(x, y, z) = p(x + L, y, z) \quad (1.5)$$

Plug Flow

Our method is to perturb an exact case of fluid flow known as plug flow, which we shall describe forthwith. If fluid is shear-driven by a constant-velocity plate at the top boundary, and experiences **perfect slip** at the bottom boundary:

$$u(x, y, \text{top}) = u_P \text{ (constant)} \quad (1.6)$$

$$\frac{\partial u}{\partial z}|_{z=0} = 0 \quad (1.7)$$

then the fluid has no resistance at the bottom, so the entire bulk quickly accelerates up to velocity of the driving plate. So fluid flows as a **plug** of fluid all at the same velocity. This flow solution is known as plug flow, and is shown in Figure (1.1).

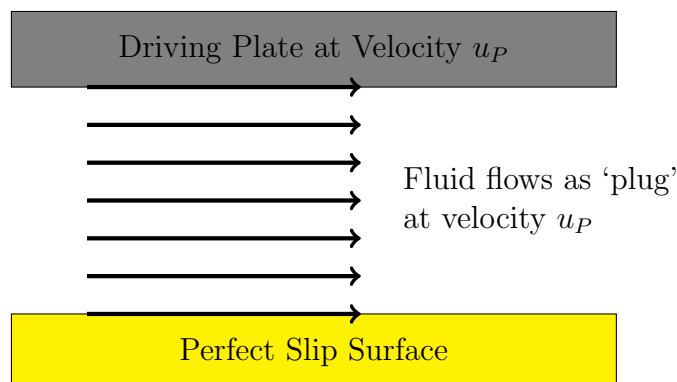


Figure 1.1: Plug flow.

Perturbed Plug Flow

We now consider shear-driven flow that is perturbed slightly away from true plug flow. What does it mean for a flow to be *close to* plug flow?

In true plug flow, the velocity at the bottom of the fluid is the *same as* the driving velocity at the top of the fluid u_P . So, fluid flow is *close to* plug flow if the bottom velocity is *close to* the driving velocity u_P . Such plug-like flow is shown in Figure (1.2).

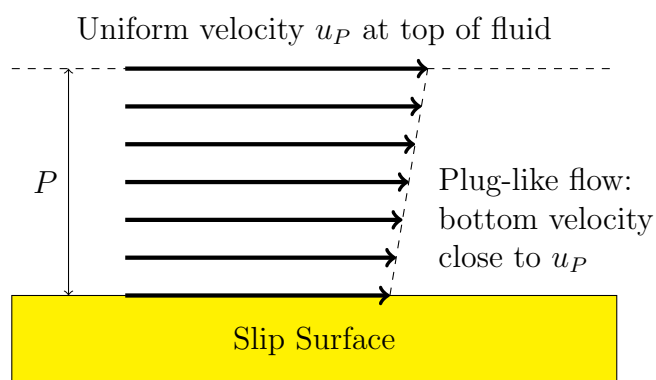


Figure 1.2: Plug-like flow.

If the slip surface is heterogeneous, then the slip velocity (at the bottom of the fluid) may vary across the surface. The flow may be described as

plug-like if the *lowest* slip velocity is still close to u_P . Since the lowest slip velocity is expected to occur over the region with the lowest intrinsic slip length, b_{\min} , flow is plug-like if b_{\min} is sufficiently large. By simple geometry, flow can be described as plug-like if b_{\min} is large compared to driving height P , as shown in Figure (1.3).

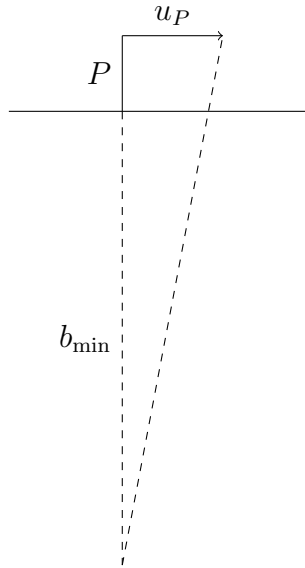


Figure 1.3: Geometry of plug-like flow.

Thus, a flow may be described as perturbed away from pure plug flow if the ratio P/b_{\min} is small compared to unity. The lengths in this dimensionless ratio are fixed, measurable lengths of the physical system. Hence, we can carry out a formal perturbation analysis with the small perturbation parameter:

$$\epsilon = \frac{P}{b_{\min}} \quad (1.8)$$

Surface Period L small compared to P

The surface is patterned with a rectangular-periodic variation in slip length with period L in the x direction. The surface patterning causes a perturbation to the otherwise-laminar velocity profile of plug flow. Due to the diffusion of momentum, the perturbation decays with height, so that the velocity solution converges to smooth laminar flow, as suggested in Figure (1.4).

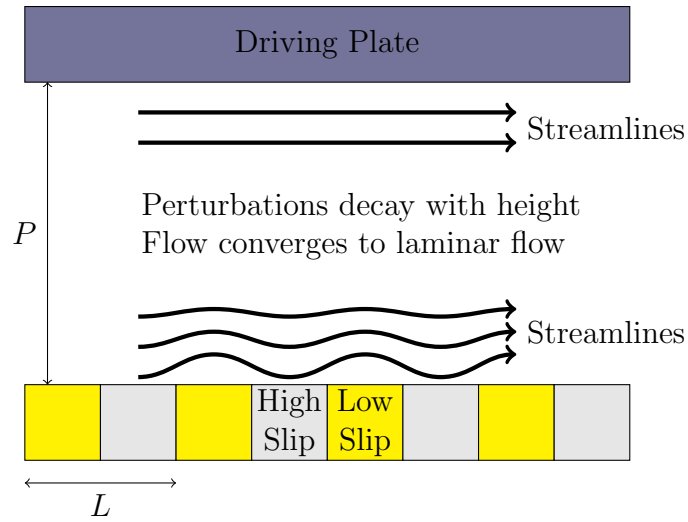


Figure 1.4: Plug-like flow over patterned slip with $L < P$.

Calculating the effective slip length involves finding the shear rate at the driving plate, which is assumed to be constant over the plate. For the shear rate to be constant, the flow near the plate must be unperturbed. In our analysis, we find that elements of the velocity perturbation are of order $\exp(-P/L)$ at $z = P$. These must be negligible, which requires that:

$$L \ll P \quad (1.9)$$

Perturbed Navier Slip

The Navier slip condition relates the shear rate to the slip velocity. With ϵ , we can express the slip condition as a perturbation away from shear-free plug flow. Multiplying both sides by ϵ gives:

$$\frac{P}{b_{\min}} b(x, y) \frac{\partial u}{\partial z} = \epsilon u(0) \quad (1.10)$$

Define the normalised slip length:

$$\hat{b} = \frac{b(x, y)}{b_{\min}}, \quad \hat{b} \geq 1 \quad (1.11)$$

So the perturbed slip condition is:

$$P\hat{b} \frac{\partial u}{\partial z} = \epsilon u(0) \quad (1.12)$$

Perturbation Expansion

The velocity solution to Stokes flow is assumed to be expressible as a power series in ϵ :

$$\vec{u} = \vec{u}_0 + \epsilon \vec{u}_1 + O(\epsilon^2) \quad (1.13)$$

where

$$\vec{u}_0 + \epsilon \vec{u}_1 = (u_0, v_0, w_0) + \epsilon(u_1, v_1, w_1) \quad (1.14)$$

The pressure is similarly expressed as a power series in ϵ :

$$p = p_0 + \epsilon p_1 + O(\epsilon^2) \quad (1.15)$$

Both are inserted into the equations of Stokes flow with perturbed slip, giving to first order:

$$\nabla^2 \vec{u}_0 + \epsilon \nabla^2 \vec{u}_1 = \frac{1}{\mu} \nabla p_0 + \epsilon \frac{1}{\mu} \nabla p_1 \quad (1.16)$$

$$\nabla \cdot \vec{u}_0 + \epsilon \nabla \cdot \vec{u}_1 = 0 \quad (1.17)$$

$$p_0(x, y, z) + \epsilon p_1(x, y, z) = p_0(x + L, y, z) + \epsilon p_1(x + L, y, z) \quad (1.18)$$

$$u_0(x, y, P) + \epsilon u_1(x, y, P) = u_P \quad (1.19)$$

$$P\hat{b} \frac{\partial u_0}{\partial z} + \epsilon P\hat{b} \frac{\partial u_1}{\partial z} = \epsilon u_0 \quad (1.20)$$

Zeroth Order

By construction, setting ϵ to zero gives shear-free flow:

$$\nabla^2 \vec{u}_0 = \frac{1}{\mu} \nabla p_0 \quad (1.21)$$

$$u_0(x, y, P) = u_P \quad (1.22)$$

$$\frac{\partial u_0}{\partial z} \Big|_{z=0} = 0 \quad (1.23)$$

whose solution is plug flow. That is, $u_0(x, y, z) = u_P$, constant everywhere.

First Order

Cancelling the zeroth order terms and dividing by ϵ gives the first order problem:

$$\nabla^2 \vec{u}_1 = \frac{1}{\mu} \nabla p_1 \quad (1.24)$$

$$\nabla \cdot \vec{u}_1 = 0 \quad (1.25)$$

$$p_1(x, y, z) = p_1(x + L, y, z) \quad (1.26)$$

$$u_1(x, y, P) = 0 \quad (1.27)$$

$$P \hat{b} \frac{\partial u_1}{\partial z} \Big|_{z=0} = u_0 = u_P \quad (1.28)$$

Note that the zeroth order solution appears in the slip condition.

Eliminate Pressure with Vorticity

The standard way to eliminate the pressure is to use the vorticity $\nabla \times \vec{u}$. Taking the curl of both sides of the Stokes equation gives:

$$\nabla \times \nabla^2 \vec{u}_1 = \nabla \times \frac{1}{\mu} \nabla p_1 \quad (1.29)$$

The right hand side is identically zero, leaving $\nabla \times \nabla^2 \vec{u}_1 = 0$. Recall that the vector Laplacian is:

$$\nabla^2 \vec{u}_1 = (\nabla^2 u_1, \nabla^2 v_1, \nabla^2 w_1) \quad (1.30)$$

so that $\nabla \times \nabla^2 \vec{u}_1 = 0$ is

$$\left(\frac{\partial}{\partial y} \nabla^2 w_1 - \frac{\partial}{\partial z} \nabla^2 v_1, \frac{\partial}{\partial z} \nabla^2 u_1 - \frac{\partial}{\partial x} \nabla^2 w_1, \frac{\partial}{\partial x} \nabla^2 v_1 - \frac{\partial}{\partial y} \nabla^2 u_1 \right) = (0, 0, 0) \quad (1.31)$$

This gives three PDEs. It turns out that the successful strategy is to use the last two. Expanding out the Laplacian operator, these are:

$$\frac{\partial^3 u_1}{\partial z \partial x^2} + \frac{\partial^3 u_1}{\partial z \partial y^2} + \frac{\partial^3 u_1}{\partial z^3} = \frac{\partial^3 w_1}{\partial x^3} + \frac{\partial^3 w_1}{\partial x \partial y^2} + \frac{\partial^3 w_1}{\partial x \partial z^2} \quad (1.32)$$

$$\frac{\partial^3 u_1}{\partial y \partial x^2} + \frac{\partial^3 u_1}{\partial y^3} + \frac{\partial^3 u_1}{\partial y \partial z^2} = \frac{\partial^3 v_1}{\partial x^3} + \frac{\partial^3 v_1}{\partial x \partial y^2} + \frac{\partial^3 v_1}{\partial x \partial z^2} \quad (1.33)$$

It also happens that the successful strategy is to convert the last equation into an expression in u_1 and w_1 . We can do this because the incompressibility couples u , v and w . Specifically, the continuity equation $\nabla \cdot \vec{u}_1 = 0$ can be rearranged to:

$$\frac{\partial v_1}{\partial y} = -\frac{\partial u_1}{\partial x} - \frac{\partial w_1}{\partial z} \quad (1.34)$$

To use this substitution, we first differentiate the last equation with respect to y :

$$\frac{\partial^4 u_1}{\partial y^2 \partial x^2} + \frac{\partial^4 u_1}{\partial y^4} + \frac{\partial^4 u_1}{\partial y^2 \partial z^2} = \frac{\partial^4 v_1}{\partial y \partial x^3} + \frac{\partial^4 v_1}{\partial x \partial y^3} + \frac{\partial^4 v_1}{\partial x \partial y \partial z^2} \quad (1.35)$$

then make the substitution, giving:

$$\begin{aligned} & \frac{\partial^4 u_1}{\partial y^2 \partial x^2} + \frac{\partial^4 u_1}{\partial y^4} + \frac{\partial^4 u_1}{\partial y^2 \partial z^2} = \\ & -\frac{\partial^3}{\partial x^3} \left[\frac{\partial u_1}{\partial x} + \frac{\partial w_1}{\partial z} \right] - \frac{\partial^3}{\partial x \partial y^2} \left[\frac{\partial u_1}{\partial x} + \frac{\partial w_1}{\partial z} \right] - \frac{\partial^3}{\partial x \partial z^2} \left[\frac{\partial u_1}{\partial x} + \frac{\partial w_1}{\partial z} \right] \end{aligned} \quad (1.36)$$

Simplified:

$$\begin{aligned} & \frac{\partial^4 u_1}{\partial x^4} + 2\frac{\partial^4 u_1}{\partial x^2 \partial y^2} + \frac{\partial^4 u_1}{\partial y^4} + \frac{\partial^4 u_1}{\partial x^2 \partial z^2} + \frac{\partial^4 u_1}{\partial y^2 \partial z^2} = \\ & -\frac{\partial^4 w_1}{\partial x^3 \partial z} - \frac{\partial^4 w_1}{\partial x \partial y^2 \partial z} - \frac{\partial^4 w_1}{\partial x \partial z^3} \end{aligned} \quad (1.37)$$

Thus we have two PDEs in two variables, u_1 and w_1 .

$$\frac{\partial^3 u_1}{\partial z \partial x^2} + \frac{\partial^3 u_1}{\partial z \partial y^2} + \frac{\partial^3 u_1}{\partial z^3} = \frac{\partial^3 w_1}{\partial x^3} + \frac{\partial^3 w_1}{\partial x \partial y^2} + \frac{\partial^3 w_1}{\partial x \partial z^2} \quad (1.38)$$

$$\frac{\partial^4 u_1}{\partial x^4} + 2 \frac{\partial^4 u_1}{\partial x^2 \partial y^2} + \frac{\partial^4 u_1}{\partial y^4} + \frac{\partial^4 u_1}{\partial x^2 \partial z^2} + \frac{\partial^4 u_1}{\partial y^2 \partial z^2} = - \frac{\partial^4 w_1}{\partial x^3 \partial z} - \frac{\partial^4 w_1}{\partial x \partial y^2 \partial z} - \frac{\partial^4 w_1}{\partial x \partial z^3} \quad (1.39)$$

Fourier Series

Because the flow is periodic, it is natural to write u_1 as a Fourier series:

$$u_1(x, y, z) = \sum_{\vec{k}}^{\infty} U_{\vec{k}}(z) \exp(i\vec{k} \cdot \vec{r}) \quad (1.40)$$

where $\vec{r} = (x, y)$ and the wave vector \vec{k} is a reciprocal lattice vector defined by integers a and b :

$$\vec{k} = (m, n) = \left(\frac{2\pi a}{L}, \frac{2\pi b}{L} \right), \quad k = \frac{2\pi q}{L} \quad (1.41)$$

$$\text{where } k^2 = m^2 + n^2, \quad q^2 = a^2 + b^2 \quad (1.42)$$

The Fourier coefficient is:

$$U_{\vec{k}}(z) = \frac{1}{L^2} \int_0^L \int_0^L u(x, y, z) \exp(i\vec{k} \cdot \vec{r}) \, dx dy \quad (1.43)$$

Similarly for w_1 :

$$w_1(x, y, z) = \sum_{\vec{k}}^{\infty} W_{\vec{k}}(z) \exp(i\vec{k} \cdot \vec{r}) \quad (1.44)$$

The two Fourier expansions for velocity are inserted into the two PDEs. At this point, resulting algebra was tackled with the computer package Maple.

At length, one has two expressions that are true for arbitrary $\vec{r} = (x, y)$. As a consequence, the following two ODEs in U and W are true for all \vec{k} :

$$\frac{d^3 U}{dz^3} - k^2 \frac{dU}{dz} = i \left(\frac{d^2 W}{dz^2} - k^2 W \right) m \quad (1.45)$$

$$k^2 \frac{d^2 U}{dz^2} - k^4 U = i \left(\frac{d^3 W}{dz^3} - k^2 \frac{dW}{dz} \right) m \quad (1.46)$$

(The parameters k and m are of course not independent.)

Solving the DEs

It turns out that a successful strategy is to solve for $W(z)$ first, then substitute the solution back into Equation (1.45), allowing us to solve for $U(z)$.

Solve for $W(z)$

After multiplying Equation (1.45) by k^2 , and differentiating Equation (1.46) with respect to z , the two equations may be combined to:

$$\frac{d^4 W}{dz^4} - 2k^2 \frac{d^2 W}{dz^2} + k^4 W = 0 \quad (1.47)$$

The general solution of which is:

$$W(z) = (P_{\vec{k}} + Q_{\vec{k}} z) e^{-kz} + (R_{\vec{k}} + S_{\vec{k}} z) e^{kz} \quad (1.48)$$

Now, at the top of the fluid, flow is in the x direction only. Therefore $w_1(x, y, P) = 0$, which requires that

$$W(P) = 0 \quad (1.49)$$

At $z = P$ the e^{kz} term is $\exp(2\pi q P/L)$ where $q \geq 1$. Since we have assumed $P \gg L$, this term is large. Therefore we must have $R_{\vec{k}} = S_{\vec{k}} = 0$.

Furthermore, the bottom surface is impermeable, so $w_1(x, y, 0) = 0$, which requires that $W(0) = 0$. An immediate corollary is that $W(0) = P_{\vec{k}} = 0$. We are left with:

$$W(z) = Q_{\vec{k}} z e^{-kz} \quad (1.50)$$

Solving for $U(z)$

We insert the solution for $W(z)$ into Equation (1.46), yielding an ODE in $U(z)$:

$$\frac{d^3 U}{dz^3} - k^2 \frac{dU}{dz} = iQ_{\vec{k}} m k^2 e^{-kz} \quad (1.51)$$

For non-zero k , the general solution is:

$$U_{\vec{k}}(z) = \left(P_{\vec{k}} + iQ_{\vec{k}} \frac{m}{k^2} \right) e^{-kz} + B_{\vec{k}} e^{kz} \quad (1.52)$$

For $k = 0$, the ODE reduces to:

$$\frac{d^3 U}{dz^3} = 0 \quad (1.53)$$

whose solution is:

$$U_0 = A_0 + B_0 z + C_0 z^2 \quad (1.54)$$

Assemble $u_1(x, y, z)$ solution

We have found the Fourier coefficients in their most general form. We now insert them into the Fourier series expression $u_1(x, y, z) = \sum_{\vec{k}}^{\infty} U_{\vec{k}}(z) \exp(i\vec{k} \cdot \vec{r})$:

$$u_1(x, y, z) = A_0 + B_0 z + C_0 z^2 + \sum_{k \neq 0} (A_{\vec{k}} e^{-kz} + B_{\vec{k}} e^{kz}) \exp(i\vec{k} \cdot \vec{r}) \quad (1.55)$$

$$\text{where } A_{\vec{k}} = \left(P_{\vec{k}} + i Q_{\vec{k}} \frac{m}{k^2} \right) \quad (1.56)$$

Use periodicity to eliminate C_0

Inserting our expression for $u_1(x, y, z)$ into the x component of the Stokes equation:

$$\frac{\partial^2 u_1}{\partial x^2} + \frac{\partial^2 u_1}{\partial y^2} + \frac{\partial^2 u_1}{\partial z^2} = \frac{1}{\mu} \frac{\partial p_1}{\partial x} \quad (1.57)$$

gives:

$$\begin{aligned} & -m^2 \sum_{k \neq 0} (A_{\vec{k}} e^{-kz} + B_{\vec{k}} e^{kz}) \exp(i\vec{k} \cdot \vec{r}) - n^2 \sum_{k \neq 0} (A_{\vec{k}} e^{-kz} + B_{\vec{k}} e^{kz}) \exp(i\vec{k} \cdot \vec{r}) \\ & + 2C_0 + k^2 \sum_{k \neq 0} (A_{\vec{k}} e^{-kz} + B_{\vec{k}} e^{kz}) \exp(i\vec{k} \cdot \vec{r}) = \frac{1}{\mu} \frac{\partial p_1}{\partial x} \end{aligned} \quad (1.58)$$

Since $k^2 = m^2 + n^2$, this reduces to:

$$2C_0 = \frac{1}{\mu} \frac{\partial p_1}{\partial x} \quad (1.59)$$

Integrate this over one period:

$$\int_0^L 2C_0 dx = \int_0^L \frac{1}{\mu} \frac{\partial p_1}{\partial x} dx \quad (1.60)$$

$$2C_0 L = \frac{1}{\mu} [p_1(L, y, z) - p_1(0, y, z)] = 0 \quad (1.61)$$

The flow is shear-driven only, so the pressure is periodic: $p_1(x, y, z) = p_1(x + L, y, z)$. Therefore the right-hand side of the integral vanishes, and we are left with $C_0 = 0$.

Use top boundary condition to find A_0

At the top of the fluid, the flow is uniform laminar flow with velocity u_P in the x direction only. At this point, the zeroth order solution is exact, so the first order term vanishes:

$$u_1(x, y, P) = 0 \quad (1.62)$$

Inserting our expression gives:

$$A_0 + B_0P + \sum_{k \neq 0} (A_{\vec{k}}e^{-kP} + B_{\vec{k}}e^{kP}) \exp(i\vec{k} \cdot \vec{r}) = 0 \quad (1.63)$$

The e^{kP} term is $\exp(2\pi qP/L)$ with $q \geq 1$. Since we have assumed $P \gg L$, this term is large. Therefore we require that $B_{\vec{k}} = 0$ for all $k \neq 0$. Conversely, the e^{-kP} term is very small. Then $A_{\vec{k}}e^{-kP}$ may be considered negligible; otherwise, we simply stipulate that $A_{\vec{k}} = 0$ for all $k \neq 0$.

Then the sum term is negligible or zero, and we are left with:

$$A_0 + B_0P = 0 \quad (1.64)$$

from which it follows that $A_0 = -B_0P$. Our first-order velocity term is now:

$$u_1(x, y, z) = B_0(z - P) + \sum_{k \neq 0} (A_{\vec{k}}e^{-kz}) \exp(i\vec{k} \cdot \vec{r}) \quad (1.65)$$

Use Slip Boundary Condition to find B_0

We have found that the Fourier coefficient for $\vec{k} = (0, 0)$ is $U_0 = B_0(z - P)$. We may equate this with the formal definition:

$$U_0 = B_0(z - P) = \frac{1}{L^2} \int_0^L \int_0^L u_1(x, y, z) dx dy \quad (1.66)$$

and differentiate with respect to z :

$$\frac{d}{dz}(B_0z - B_0P) = \frac{1}{L^2} \int_0^L \int_0^L \frac{d}{dz} u_1(x, y, z) dx dy \quad (1.67)$$

then evaluate at $z = 0$:

$$B_0 = \frac{1}{L^2} \int_0^L \int_0^L \frac{du_1}{dz} \Big|_{z=0} dx dy \quad (1.68)$$

At this point we can substitute the slip boundary condition:

$$\frac{du_1}{dz} \Big|_{z=0} = \frac{1}{P\hat{b}} u_P \quad (1.69)$$

to get:

$$B_0 = \frac{1}{L^2} \int_0^L \int_0^L \frac{1}{P\hat{b}} u_P dx dy \quad (1.70)$$

The double integral is the area-weighted average:

$$B_0 = \frac{u_P}{P} \frac{1}{L^2} \int_0^L \int_0^L \frac{1}{\hat{b}} dx dy = \frac{u_P}{P} \left\langle \frac{1}{\hat{b}} \right\rangle \quad (1.71)$$

So the **first order velocity term** is:

$$u_1(x, y, z) = \frac{u_P}{P} \left\langle \frac{1}{\hat{b}} \right\rangle (z - P) + \sum_{k \neq 0} (A_{\vec{k}} e^{-kz}) \exp(i\vec{k} \cdot \vec{r}) \quad (1.72)$$

Bolt Together Velocity Solution

We now have all the parts of the x velocity perturbation expansion $u(x, y, z) = u_0(x, y, z) + \epsilon u_1(x, y, z)$. Bolting it together gives:

$$u(x, y, z) = u_P + \epsilon \frac{u_P}{P} \left\langle \frac{1}{\hat{b}} \right\rangle (z - P) + \epsilon \sum_{k \neq 0} (A_{\vec{k}} e^{-kz}) \exp(i\vec{k} \cdot \vec{r}) \quad (1.73)$$

Recall that:

$$\epsilon = \frac{P}{b_{\min}} \quad \text{and} \quad \hat{b} = \frac{b}{b_{\min}} \quad (1.74)$$

therefore:

$$\epsilon \frac{u_P}{P} \left\langle \frac{1}{\hat{b}} \right\rangle = \frac{P}{b_{\min}} \frac{u_P}{P} \left\langle \frac{b_{\min}}{b} \right\rangle = u_P \left\langle \frac{1}{b} \right\rangle \quad (1.75)$$

Thus, the final velocity solution is:

$$u(x, y, z) = u_P + u_P \left\langle \frac{1}{b} \right\rangle (z - P) + \epsilon \sum_{k \neq 0} (A_{\vec{k}} e^{-kz}) \exp(i\vec{k} \cdot \vec{r}) \quad (1.76)$$

Effective Slip Length

Since we know the height P and the velocity u_P of the driving plate, if we know the shear rate at the driving plate, we can calculate the effective slip length. The flow is uniform and laminar at the driving plate, so simple shear holds, and the shear rate is simply the velocity gradient $\frac{d}{dz}u$:

$$\frac{d}{dz}u(x, y, z) = u_P \left\langle \frac{1}{b} \right\rangle - \epsilon \sum_{k \neq 0} (k A_{\vec{k}} e^{-kz}) \exp(i\vec{k} \cdot \vec{r}) \quad (1.77)$$

At $z = P$, the term $e^{-kz} < \exp(-2\pi P/L)$ and is negligible since $P \gg L$. So:

$$\left. \frac{du}{dz} \right|_{z=P} = u_P \left\langle \frac{1}{b} \right\rangle \quad (1.78)$$

Rearranging to the familiar form of Navier slip:

$$u_P = \left\langle \frac{1}{b} \right\rangle^{-1} \left. \frac{du}{dz} \right|_{z=P} \quad (1.79)$$

This defines an effective slip length:

$$b_{\text{eff}} = \left\langle \frac{1}{b} \right\rangle^{-1} \quad (1.80)$$

The true slip length of the **solid surface** is less by the distance P from the driving plate to the surface:

$$b_{\text{eff}} = \left\langle \frac{1}{b} \right\rangle^{-1} - P \quad (1.81)$$

This result is derived by perturbing plug flow; the smallness of the perturbation is expressed by the smallness of the ratio P/b_{min} . Since $b_{\text{min}} < \left\langle \frac{1}{b} \right\rangle^{-1}$, the assumption $P \ll b_{\text{min}}$ implies that

$$P \ll \left\langle \frac{1}{b} \right\rangle^{-1} \quad (1.82)$$

Therefore, the P may be neglected, leaving Equation (1.80).

Perturbed Couette Flow

A similar analysis can be done for perturbed Couette flow. We still assume that the height of the driving plate is much larger than the period of surface patterning :

$$P \gg L \quad (1.83)$$

In this case, flow is *close to* Couette flow if the maximum slip length b_{\max} of the surface is *small* compared to the height P of the driving plate. So a suitable choice of perturbation parameter is:

$$\epsilon = \frac{b_{\max}}{P} \quad (1.84)$$

And the normalised slip length can be defined as:

$$\hat{b} = \frac{b(x, y)}{b_{\max}}, \quad 0 \leq \hat{b} \leq 1 \quad (1.85)$$

Then both sides of the Navier slip condition can be divided by ϵ :

$$\frac{1}{\epsilon} u(x, y, 0) = \frac{P}{b_{\max}} b(x, y) \frac{\partial u}{\partial z} \Big|_{z=0} \quad (1.86)$$

So the perturbed slip condition is:

$$u(x, y, 0) = \epsilon P \hat{b} \frac{\partial u}{\partial z} \Big|_{z=0} \quad (1.87)$$

As before, the velocity solution is written as a perturbation expansion:

$$\vec{u} = \vec{u}_0 + \epsilon \vec{u}_1 = (u_0, v_0, w_0) + \epsilon(u_1, v_1, w_1) \quad (1.88)$$

which is inserted into the Stokes, continuity, and various boundary equations.

The only difference is of course the slip condition. To first order in ϵ :

$$u_0 + \epsilon u_1 = \epsilon P \hat{b} \frac{\partial u_0}{\partial z} \Big|_{z=0} \quad (1.89)$$

Zeroth Order

By construction, setting $\epsilon = 0$ gives classic Couette flow:

$$u_0(x, y, P) = u_P \quad (1.90)$$

$$u_0(x, y, 0) = 0 \quad (1.91)$$

whose solution is a linear interpolation between the top and no-slip boundary conditions:

$$u_0(x, y, z) = \frac{u_P}{P} z \quad (1.92)$$

First Order

Cancelling the zeroth order terms and dividing by ϵ gives the first order slip condition:

$$u_1 = P \hat{b} \frac{\partial u_0}{\partial z} \Big|_{z=0} = P \hat{b} \frac{u_P}{P} \quad (1.93)$$

$$u_1 = \hat{b} u_P \quad (1.94)$$

Again, the zeroth order solution enters the slip condition.

The analysis proceeds exactly as before, until the point where we solve for the coefficient B_0 .

Use Slip Boundary Condition to find B_0

We have found that the Fourier coefficient for $\vec{k} = (0, 0)$ is $U_0 = B_0(z - P)$. We may equate this with the formal definition:

$$U_0 = B_0(z - P) = \frac{1}{L^2} \int_0^L \int_0^L u_1(x, y, z) dx dy \quad (1.95)$$

If we evaluate this at $z = 0$, we may substitute the first order slip condition:

$$-B_0 P = \frac{1}{L^2} \int_0^L \int_0^L \hat{b} u_P dx dy \quad (1.96)$$

The double integral is the area-weighted average:

$$B_0 = -\frac{u_P}{P} \frac{1}{L^2} \int_0^L \int_0^L \hat{b} dx dy = -\frac{u_P}{P} \langle \hat{b} \rangle \quad (1.97)$$

So the **first order velocity term** is:

$$u_1(x, y, z) = \frac{u_P}{P} \langle \hat{b} \rangle (P - z) + \sum_{k \neq 0} (A_{\vec{k}} e^{-kz}) \exp(i\vec{k} \cdot \vec{r}) \quad (1.98)$$

Bolt Together Velocity Solution

We now have all the parts of the x velocity perturbation expansion $u(x, y, z) = u_0(x, y, z) + \epsilon u_1(x, y, z)$. Boltting it together gives:

$$u(x, y, z) = \frac{u_P}{P} z + \epsilon \frac{u_P}{P} \langle \hat{b} \rangle (P - z) + \epsilon \sum_{k \neq 0} (A_{\vec{k}} e^{-kz}) \exp(i\vec{k} \cdot \vec{r}) \quad (1.99)$$

Recall that:

$$\epsilon = \frac{b_{\max}}{P} \quad \text{and} \quad \hat{b} = \frac{b}{b_{\max}} \quad (1.100)$$

therefore:

$$\epsilon \frac{u_P}{P} \langle \hat{b} \rangle = \frac{b_{\max}}{P} \frac{u_P}{P} \left\langle \frac{b}{b_{\max}} \right\rangle = \frac{u_P}{P^2} \langle b \rangle \quad (1.101)$$

so that:

$$\epsilon \frac{u_P}{P} \langle \hat{b} \rangle (P - z) = \frac{u_P}{P} \langle b \rangle - \frac{u_P}{P^2} \langle b \rangle z \quad (1.102)$$

Thus, the final velocity solution is:

$$u(x, y, z) = \frac{u_P}{P} \left[z \left(1 - \frac{\langle b \rangle}{P} \right) + \langle b \rangle \right] + \epsilon \sum_{k \neq 0} (A_{\vec{k}} e^{-kz}) \exp(i\vec{k} \cdot \vec{r}) \quad (1.103)$$

Effective Slip Length

As before, we calculate the effective slip length with the velocity gradient:

$$\frac{\partial}{\partial z} u(x, y, z) = \frac{u_P}{P} \left(1 - \frac{\langle b \rangle}{P} \right) - \epsilon \sum_{k \neq 0} (k A_{\vec{k}} e^{-kz}) \exp(i\vec{k} \cdot \vec{r}) \quad (1.104)$$

At $z = P$, the term $e^{-kz} < \exp(-2\pi P/L)$ and is negligible since $P \gg L$. So:

$$\frac{\partial}{\partial z} u(x, y, z) = \frac{u_P}{P} \left(1 - \frac{\langle b \rangle}{P} \right) \quad (1.105)$$

Rearranging to the form of Navier slip:

$$u_P = P \left(1 - \frac{\langle b \rangle}{P} \right)^{-1} \frac{\partial u}{\partial z} \Big|_{z=0} \quad (1.106)$$

Implying an effective slip length at the top boundary of:

$$b_{\text{eff}}(P) = P \left(1 - \frac{\langle b \rangle}{P} \right)^{-1} = \frac{P}{1 - \frac{\langle b \rangle}{P}} \quad (1.107)$$

The slip length of the surface is found by subtracting P from this:

$$b_{\text{eff}} = \frac{P}{1 - \frac{\langle b \rangle}{P}} - P \quad (1.108)$$

$$= \frac{P - P \left[1 - \frac{\langle b \rangle}{P} \right]}{1 - \frac{\langle b \rangle}{P}} = \frac{P - P + \langle b \rangle}{1 - \frac{\langle b \rangle}{P}} = \frac{\langle b \rangle}{1 - \frac{\langle b \rangle}{P}} \quad (1.109)$$

We have assumed that for Couette-like flow, $b_{\text{max}} \ll P$, and $\langle b \rangle < b_{\text{max}}$, therefore $\langle b \rangle \ll P$, and so $\langle b \rangle / P$ is negligible. We are left with:

$$b_{\text{eff}} = \langle b \rangle \quad (1.110)$$

(Note that for this Couette-like flow, the effective slip length ‘seen’ at the driving plate, $b_{\text{eff}}(P)$, is negligibly different from P , as expected.)

Conclusion

The effective slip length found by perturbing plug flow reconciles with the effective slip length found by homogenization. Our perturbation analysis assumed a strict definition of plug-like flow: $P \ll b_{\text{min}}$, and further assumed that near the driving plate, the perturbations had diffused into laminar flow, expressed by $L \ll P$. In summary, the regime satisfied:

$$L \ll P \ll b_{\text{min}} \quad (1.111)$$

Note that homogenization did not require any constraint on P ; the homogenized slip length is expected to be accurate whenever $L \ll b_{\text{min}}$.

In addition, the perturbative method can yield an effective slip length for the opposite case – where the slip lengths are *small* compared to length scales of the flow: $b_{\max} \ll P$. It was still assumed that perturbations diffused into laminar flow near the driving plate: $L \ll P$. Note that we cannot say anything about the relative sizes of L and b in this regime.