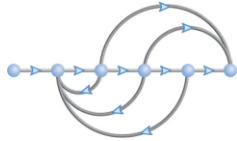


## Modeling in the Time Domain

3



### Chapter Objectives

- After completing this chapter, the student will be able to
- find a mathematical model, called a state-space representation, for a linear, time invariant system
  - model electrical and mechanical systems in state space
  - convert a transfer function to state space
  - convert a state-space representation to a transfer function
  - linearize a state-space representation

### §1. Introduction

Two approaches are available for the analysis and design of feedback control systems

- The classical, or frequency-domain, technique

**Major disadvantage:** can be applied only to linear, time-invariant systems or systems that can be approximated as such

**Major advantage:** rapidly provide stability and transient response information

### §1. Introduction

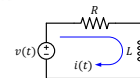
- The modern, or time domain, state-space technique

- A unified method for modeling, analyzing, and designing a wide range of systems
- Can be used to represent nonlinear systems that have backlash, saturation, and dead zone
- Can handle, conveniently, systems with nonzero initial conditions
- Can be used to represent time-varying systems, (for example, missiles with varying fuel levels or lift in an aircraft flying through a wide range of altitudes)
- Can be compactly represented in state space for multiple-input, multiple-output systems
- Can be used to represent systems with a digital computer in the loop or to model systems for digital simulation

### §1. Introduction

- With a simulated system, system response can be obtained for changes in system parameters - an important design tool
- The state space approach is also attractive because of the availability of numerous state-space software packages for the personal computer

### §2. Some Observations

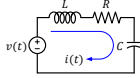


Select the current  $i(t)$  as a variable, write the loop equation

$$L \frac{di(t)}{dt} + Ri(t) = v(t)$$

$$\rightarrow \frac{di(t)}{dt} = -\frac{R}{L}i(t) + \frac{1}{L}v(t)$$

## §2. Some Observations



Select the current  $i(t)$  as a variable, write the loop equation

$$i(t) = \frac{dq(t)}{dt}$$

$$L \frac{di(t)}{dt} + Ri(t) + \frac{1}{C} \int i dt = v(t)$$

$$\rightarrow \frac{dq(t)}{dt} = i(t)$$

$$\frac{di(t)}{dt} = -\frac{1}{LC}q(t) - \frac{R}{L}i(t) + \frac{1}{L}v(t)$$

## §2. Some Observations

$$\frac{dq(t)}{dt} = i(t)$$

$$\frac{di(t)}{dt} = -\frac{1}{LC}q(t) - \frac{R}{L}i(t) + \frac{1}{L}v(t)$$

The state equation can be written in vector-matrix form

$$\dot{x} = Ax + Bu$$

where,  $x = \begin{bmatrix} q(t) \\ i(t) \end{bmatrix}$ ,  $A = \begin{bmatrix} 0 & 1 \\ -\frac{1}{LC} & -\frac{R}{L} \end{bmatrix}$ ,  $B = \begin{bmatrix} 0 \\ \frac{1}{L} \end{bmatrix}$ ,  $u = v(t)$

The output equation can be written in vector-matrix form

$$y = Cx + Du$$

where,  $y = v_L(t)$ ,  $C = \begin{bmatrix} -\frac{1}{C} & -R \end{bmatrix}$ ,  $x = \begin{bmatrix} q(t) \\ i(t) \end{bmatrix}$ ,  $D = 1$ ,  $u = v(t)$

## §3. The General State-Space Representation

Review

- **Linear combination:** A linear combination of  $n$  variables,  $x_i$ , for  $i = 1 \div n$ , is given by the following sum,  $S$

$$S = K_n x_n + K_{n-1} x_{n-1} + \dots + K_1 x_1 \quad K_i: \text{constant}$$

- **Linear independence:** A set of variables is said to be linearly independent if none of the variables can be written as a linear combination of the others

- **System variable:** Any variable that responds to an input or initial conditions in a system

- **State variables:** The smallest set of linearly independent system variables such that the values of the members of the set at time  $t_0$  along with known forcing functions completely determine the value of all system variables for all  $t \geq t_0$

- **State vector:** A vector whose elements are the state variables

## §3. The General State-Space Representation

- **State space:**  $n$ -dimensional space whose axes are the state variables

In the figure

- state variables:  $v_R$  and  $v_C$
- state trajectory can be thought of as being mapped out by the state vector,  $x(t)$ , for a range of  $t$

Graphic representation of state space and a state vector

- **State equations:** A set of  $n$  simultaneous, first-order differential equations with  $n$  variables, where the  $n$  variables to be solved are the state variables

- **Output equation:** The algebraic equation that expresses the output variables of a system as linear combinations of the state variables and the inputs

## §3. The General State-Space Representation

- A system is represented in state space by the following equations

$$\dot{x} = Ax + Bu$$

$$y = Cx + Du$$

for  $t \geq t_0$  and initial conditions,  $x(t_0)$ , where

$x$ : state vector

$\dot{x}$ : derivative of the state vector with respect to time

$y$ : output vector

$A$ : system matrix

$C$ : output matrix

$u$ : input or control vector

$B$ : input matrix

$D$ : feedforward matrix

## §4. Applying the State-Space Representation

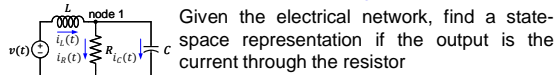
In this section, the state-space formulation is applied to the representation of more complicated physical systems. The first step in representing a system is to select the state vector, which must be chosen according to the following considerations

1. A **minimum number of state variables** must be selected as components of the state vector. This minimum number of state variables is sufficient to describe completely the state of the system
2. The components of the state vector (that is, this minimum number of state variables) **must be linearly independent**

## §4. Applying the State-Space Representation

## - Ex.3.1

## Representing an Electrical Network



## Solution

**Step 1** Label all of the branch currents in the network. These include  $i_L$ ,  $i_R$ , and  $i_C$

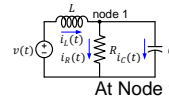
**Step 2** Select the state variables by writing the derivative equation for all energy storage elements,  $L$  and  $C$

$$C \frac{dv_C}{dt} = i_C \quad (3.22)$$

$$L \frac{di_L}{dt} = v_L \quad (3.23)$$

## §4. Applying the State-Space Representation

**Step 3** Apply network theory, such as Kirchhoff's voltage and current laws, to obtain  $i_C$  and  $v_L$  in terms of the state variables,  $v_C$  and  $i_L$ .



$$i_C = -i_R + i_L = -\frac{1}{R} v_C + i_L \quad (3.24)$$

which yields  $i_C$  in terms of the state variables,  $v_C$  and  $i_L$

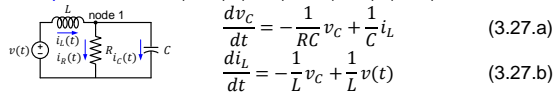
Around the outer loop,

$$v_L = -v_C + v(t) \quad (3.25)$$

which yields  $v_L$  in terms of the state variable,  $v_C$ , and the source,  $v(t)$

## §4. Applying the State-Space Representation

**Step 4** Substitute (3.24), (3.25) into (3.22), (3.23) to obtain



$$\frac{di_L}{dt} = -\frac{1}{L} v_C + \frac{1}{L} v(t) \quad (3.27.b)$$

**Step 5** Find the output equation. Since the output is  $i_R(t)$

$$i_R = v_C / R \quad (3.28)$$

The state-space representation is found by representing Eqs. (3.27) and (3.28) in vector-matrix form

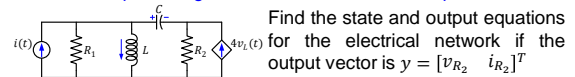
$$\begin{bmatrix} \dot{v}_C \\ \dot{i}_L \end{bmatrix} = \begin{bmatrix} -1/RC & 1/C \\ -1/L & 1/L \end{bmatrix} \begin{bmatrix} v_C \\ i_L \end{bmatrix} + \begin{bmatrix} 0 \\ 1/L \end{bmatrix} v(t)$$

$$i_R = \begin{bmatrix} 1/R & 0 \end{bmatrix} \begin{bmatrix} v_C \\ i_L \end{bmatrix}$$

$$C \frac{dv_C}{dt} = i_C \quad (3.22), \quad L \frac{di_L}{dt} = v_L \quad (3.23), \quad i_C = -\frac{1}{R} v_C + i_L \quad (3.24), \quad v_L = -v_C + v(t) \quad (3.25)$$

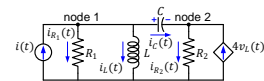
## §4. Applying the State-Space Representation

## - Ex.3.2 Representing an Electrical Network with a Dependent Source



## Solution

**Step 1** Label all of the branch currents in the network



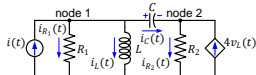
**Step 2** Select the state variables by listing the voltage-current relationships for all of the energy-storage elements

$$L \frac{di_L}{dt} = v_L \quad C \frac{dv_C}{dt} = i_C \quad (3.30)$$

Select the state variables:  $x_1 = i_L$  and  $x_2 = v_C$  (3.31)

## §4. Applying the State-Space Representation

**Step 3** Using Kirchhoff's voltage and current laws to find  $i_L$ ,  $v_C$  in terms of the state variables



Around the mesh containing  $L$  and  $C$

$$v_L = v_C + v_{R_2} = v_C + i_{R_2} R_2$$

At node 2,  $i_{R_2} = i_C + 4v_L$

$$v_L = v_C + (i_C + 4v_L) R_2 = \frac{1}{1 - 4R_2} (v_C + i_C R_2) \quad (3.35)$$

At node 1

$$i_C = i - i_{R_1} - i_L = i - \frac{v_{R_1}}{R_1} - i_L = i - \frac{v_L}{R_1} - i_L \quad (3.36)$$

## §4. Applying the State-Space Representation

Rewriting Eqs (3.35) and (3.36)

$$(1 - 4R_2)v_L - R_2 i_C = v_C$$

$$-(1/R_1)v_L - i_C = i_L - i$$

Writing the result in vector-matrix form

$$v_L = \frac{1}{\Delta} [R_2 i_L - v_C - R_2 i] \quad (3.38)$$

$$i_C = \frac{1}{\Delta} \left[ (1 - 4R_2) i_L + \frac{1}{R_1} v_C - (1 - 4R_2) i \right] \quad (3.39)$$

where,

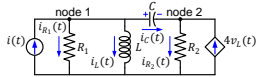
$$\Delta = - \left[ (1 - 4R_2) + \frac{R_2}{R_1} \right]$$

$$\begin{bmatrix} \dot{i}_L \\ \dot{v}_C \end{bmatrix} = \begin{bmatrix} R_2/(L\Delta) & -1/(L\Delta) \\ (1 - 4R_2)/(C\Delta) & 1/(R_1 C\Delta) \end{bmatrix} \begin{bmatrix} i_L \\ v_C \end{bmatrix} + \begin{bmatrix} -R_2/(L\Delta) \\ (1 - 4R_2)/(C\Delta) \end{bmatrix} i$$

$$v_L = \frac{1}{1 - 4R_2} (v_C + i_C R_2) \quad (3.35), \quad i_C = i - \frac{v_L}{R_1} - i_L \quad (3.36)$$

## §4. Applying the State-Space Representation

Step 4 Derive the output equation



Since the specified output variables are  $v_{R_2}$  and  $i_{R_2}$ , note that around the mesh containing  $C$ ,  $L$ , and  $R_2$

$$v_{R_2} = -v_C + v_L \quad (3.42.a)$$

$$i_{R_2} = i_C + 4v_L \quad (3.42.b)$$

Substituting Eqs. (3.38) and (3.39) into Eq.(3.42)

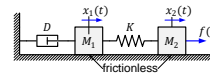
$$\begin{bmatrix} v_{R_2} \\ i_{R_2} \end{bmatrix} = \begin{bmatrix} R_2/\Delta & -(1+1/\Delta) \\ 1/\Delta & (1-4R_1)/(R_1\Delta) \end{bmatrix} \begin{bmatrix} i_L \\ v_C \end{bmatrix} + \begin{bmatrix} -R_2/\Delta \\ -1/\Delta \end{bmatrix} i$$

$$v_L = \frac{1}{\Delta} [R_2 i_L - v_C - R_2 i] \quad (3.38), \quad i_C = \frac{1}{\Delta} [(1-4R_2) i_L + \frac{1}{R_1} v_C - (1-4R_2) i] \quad (3.39)$$

## §4. Applying the State-Space Representation

- Ex.3.3

Representing a Translational Mechanical System



Find the state equations for the translational mechanical system

## Solution

Find the Laplace-transformed equations of motion

$$\begin{aligned} & \left[ \begin{array}{c} \text{sum of impedances} \\ \text{connected to the} \\ \text{motion at } x_1 \end{array} \right] \times X_1(s) - \left[ \begin{array}{c} \text{sum of impedances} \\ \text{between } x_1 \text{ and } x_2 \end{array} \right] \times X_2(s) = \left[ \begin{array}{c} \text{sum of applied} \\ \text{forces at } x_1 \end{array} \right] \\ & + [M_1 s^2 + Ds + K] X_1 - K X_2 = 0 \\ & - \left[ \begin{array}{c} \text{sum of impedances} \\ \text{between } x_1 \text{ and } x_2 \end{array} \right] \times X_1(s) + \left[ \begin{array}{c} \text{sum of impedances} \\ \text{connected to the} \\ \text{motion at } x_2 \end{array} \right] \times X_2(s) = \left[ \begin{array}{c} \text{sum of applied} \\ \text{forces at } x_2 \end{array} \right] \\ & - K X_1 + [M_2 s^2 + K] X_2 = F \end{aligned}$$

## §4. Applying the State-Space Representation

Take the inverse Laplace transform assuming zero initial conditions

$$[M_1 s^2 + Ds + K] X_1 - K X_2 = 0 \rightarrow M_1 \frac{d^2 x_1}{dt^2} + D \frac{dx_1}{dt} + K(x_1 - x_2) = 0 \quad (3.44)$$

$$-K X_1 + [M_2 s^2 + K] X_2 = F \rightarrow -K x_1 + M_2 \frac{d^2 x_2}{dt^2} + K x_2 = f(t) \quad (3.45)$$

$$\text{Let } \frac{dx_1}{dt} \equiv v_1, \frac{dx_2}{dt} \equiv v_2 \rightarrow \frac{d^2 x_1}{dt^2} = \frac{dv_1}{dt}, \frac{d^2 x_2}{dt^2} = \frac{dv_2}{dt}$$

$$\begin{aligned} \text{State equations} \quad \frac{dx_1}{dt} &= +v_1 \\ \frac{dv_1}{dt} &= -\frac{K}{M_1} x_1 - \frac{D}{M_1} v_1 + \frac{K}{M_1} x_2 \\ \frac{dx_2}{dt} &= +v_2 \\ \frac{dv_2}{dt} &= +\frac{K}{M_2} x_1 - \frac{K}{M_2} x_2 + \frac{1}{M_2} f(t) \end{aligned}$$

## §4. Applying the State-Space Representation

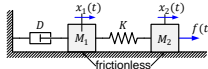
$$\begin{aligned} \frac{dx_1}{dt} &= +v_1 \\ \frac{dv_1}{dt} &= -\frac{K}{M_1} x_1 - \frac{D}{M_1} v_1 + \frac{K}{M_1} x_2 \\ \frac{dx_2}{dt} &= +v_2 \\ \frac{dv_2}{dt} &= +\frac{K}{M_2} x_1 - \frac{K}{M_2} x_2 + \frac{1}{M_2} f(t) \end{aligned}$$

In vector matrix form

$$\begin{bmatrix} \dot{x}_1 \\ \dot{v}_1 \\ \dot{x}_2 \\ \dot{v}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ -\frac{K}{M_1} & -\frac{D}{M_1} & \frac{K}{M_1} & 0 \\ 0 & 0 & 0 & 1 \\ \frac{K}{M_2} & 0 & -\frac{K}{M_2} & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ v_1 \\ x_2 \\ v_2 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 0 \\ \frac{1}{M_2} \end{bmatrix} f(t)$$

## §4. Applying the State-Space Representation

Note: The equations of motion (3.44) and (3.45) can be derived directly from the figure using Newton's laws of motion



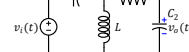
$$\begin{aligned} M_1 \frac{d^2 x_1}{dt^2} + D \frac{dx_1}{dt} + K(x_1 - x_2) &= 0 \\ M_2 \frac{d^2 x_2}{dt^2} + K(x_2 - x_1) &= f(t) \\ \rightarrow M_1 \frac{d^2 x_1}{dt^2} + D \frac{dx_1}{dt} + K(x_1 - x_2) &= 0 \quad (3.44) \end{aligned}$$

$$-K x_1 + M_2 \frac{d^2 x_2}{dt^2} + K x_2 = f(t) \quad (3.45)$$

## §4. Applying the State-Space Representation

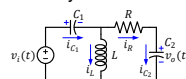
Skill-Assessment Ex.3.1

Problem Find the state-space representation of the electrical network with the output is  $v_o(t)$



## Solution

Identifying appropriate variables on the circuit yields

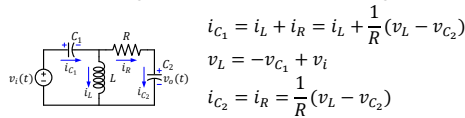


Writing the derivative relations

$$C_1 \frac{dv_{C_1}}{dt} = i_{C_1} \quad L \frac{di_L}{dt} = v_L \quad C_2 \frac{dv_{C_2}}{dt} = i_{C_2}$$

## §4. Applying the State-Space Representation

Using Kirchhoff's current and voltage laws



$$i_{C1} = i_L + i_R = i_L + \frac{1}{R}(v_L - v_{C2})$$

$$v_L = -v_{C1} + v_i$$

$$i_{C2} = i_R = \frac{1}{R}(v_L - v_{C2})$$

Substituting and rearranging

$$\frac{dv_{C1}}{dt} = -\frac{1}{RC_1}v_{C1} + \frac{1}{C_1}i_L - \frac{1}{RC_1}v_{C2} + \frac{1}{RC_1}v_i$$

$$\frac{di_L}{dt} = -\frac{1}{L}v_{C1} + \frac{1}{L}v_i$$

$$\frac{dv_{C2}}{dt} = -\frac{1}{RC_2}v_{C1} - \frac{1}{RC_2}v_{C2} + \frac{1}{RC_2}v_i$$

The output  $v_o = v_{C2}$ 

## §4. Applying the State-Space Representation

$$\frac{dv_{C1}}{dt} = -\frac{1}{RC_1}v_{C1} + \frac{1}{C_1}i_L - \frac{1}{RC_1}v_{C2} + \frac{1}{RC_1}v_i$$

$$\frac{di_L}{dt} = -\frac{1}{L}v_{C1} + \frac{1}{L}v_i$$

$$\frac{dv_{C2}}{dt} = -\frac{1}{RC_2}v_{C1} - \frac{1}{RC_2}v_{C2} + \frac{1}{RC_2}v_i$$

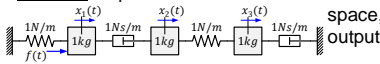
The output  $v_o = v_{C2}$ , the equations in vector-matrix form

$$\dot{\mathbf{x}} = \begin{bmatrix} \dot{v}_{C1} \\ \dot{i}_L \\ \dot{v}_{C2} \end{bmatrix} = \begin{bmatrix} -\frac{1}{RC_1} & \frac{1}{C_1} & -\frac{1}{RC_1} \\ -\frac{1}{L} & 0 & 0 \\ -\frac{1}{RC_2} & 0 & -\frac{1}{RC_2} \end{bmatrix} \begin{bmatrix} v_{C1} \\ i_L \\ v_{C2} \end{bmatrix} + \begin{bmatrix} \frac{1}{RC_1} \\ \frac{1}{L} \\ \frac{1}{RC_2} \end{bmatrix} v_i$$

$$\mathbf{y} = [0 \quad 0 \quad 1]\mathbf{x}$$

## §4. Applying the State-Space Representation

## Skill-Assessment Ex.3.2

**Problem** Represent the translational mechanical system in state-space, where  $x_3(t)$  is the output**Solution** Writing the equations of motion

$$(s^2 + s + 1)X_1 - sX_2 = F$$

$$-sX_1 + (s^2 + s + 1)X_2 - X_3 = 0$$

$$-X_2 + (s^2 + s + 1)X_3 = 0$$

Taking the inverse Laplace transform and simplifying

$$\ddot{x}_1 = -\dot{x}_1 - x_1 + \dot{x}_2 + f$$

$$\ddot{x}_2 = +\dot{x}_1 - \dot{x}_2 - x_2 + x_3$$

$$\ddot{x}_3 = -\dot{x}_3 - x_3 + x_2$$

## §4. Applying the State-Space Representation

Defining state variables

$$z_1 = x_1, z_2 = \dot{x}_1, z_3 = x_2, z_4 = \dot{x}_2, z_5 = x_3, z_6 = \dot{x}_3$$

Write the state equations

$$\dot{z}_1 = z_2$$

$$\dot{z}_2 = \ddot{x}_1 = -\dot{x}_1 - x_1 + \dot{x}_2 + f = -z_2 - z_1 + z_4 + f$$

$$\dot{z}_3 = \dot{x}_2 = z_4$$

$$\dot{z}_4 = \ddot{x}_2 = \dot{x}_1 - \dot{x}_2 - x_2 + x_3 = z_2 - z_4 - z_3 + z_5$$

$$\dot{z}_5 = \dot{x}_3$$

$$= z_6$$

$$\dot{z}_6 = \ddot{x}_3 = -\dot{x}_3 - x_3 + x_2 = -z_6 - z_5 + z_3$$

$$\dot{\mathbf{z}} = \begin{bmatrix} \dot{z}_1 \\ \dot{z}_2 \\ \dot{z}_3 \\ \dot{z}_4 \\ \dot{z}_5 \\ \dot{z}_6 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 & 0 & 0 \\ -1 & -1 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 1 & -1 & -1 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \\ 0 & 0 & 1 & 0 & -1 & -1 \end{bmatrix} \mathbf{z} + \begin{bmatrix} 0 \\ f(t) \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

## §4. Applying the State-Space Representation

$$\dot{z}_1 = +z_2$$

$$\dot{z}_2 = -z_1 - z_2 + z_4 + f$$

$$\dot{z}_3 = +z_4$$

$$\dot{z}_4 = +z_2 - z_4 - z_3 + z_5$$

$$\dot{z}_5 = +z_6$$

$$\dot{z}_6 = +z_3 - z_5 - z_6$$

The output  $y = z_5$ , the equations in vector-matrix form

$$\dot{\mathbf{z}} = \begin{bmatrix} 0 & 1 & 0 & 0 & 0 & 0 \\ -1 & -1 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 1 & -1 & -1 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \\ 0 & 0 & 1 & 0 & -1 & -1 \end{bmatrix} \mathbf{z} + \begin{bmatrix} 0 \\ f(t) \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

$$\mathbf{y} = [0 \quad 0 \quad 0 \quad 0 \quad 1 \quad 0]\mathbf{z}$$

## §5. Converting a Transfer Function to State-Space

Consider the differential equation

$$\frac{d^n y}{dt^n} + a_{n-1} \frac{d^{n-1} y}{dt^{n-1}} + \dots + a_1 \frac{dy}{dt} + a_0 y = b_0 u$$

 $y$ : output  $u$ : input  $a_i$ 's,  $b_0$ : constantChoose the output,  $y$ , and its derivatives as the state variables

$$x_1 = y, x_2 = \frac{dy}{dt}, x_3 = \frac{d^2 y}{dt^2}, \dots, x_n = \frac{d^{n-1} y}{dt^{n-1}}$$

Differentiating both sides yields

$$\dot{x}_1 = \frac{dy}{dt}, \dot{x}_2 = \frac{d^2 y}{dt^2}, \dot{x}_3 = \frac{d^3 y}{dt^3}, \dots, \dot{x}_n = \frac{d^n y}{dt^n}$$

Define the state variables

$$\dot{x}_1 \stackrel{\text{def}}{=} x_2, \dot{x}_2 \stackrel{\text{def}}{=} x_3, \dot{x}_3 \stackrel{\text{def}}{=} x_4, \dots, \dot{x}_{n-1} \stackrel{\text{def}}{=} x_n$$

$$\dot{x}_n \stackrel{\text{def}}{=} -a_0 x_1 - a_1 x_2 - \dots - a_{n-1} x_n + b_0 u$$

### §5. Converting a Transfer Function to State-Space

The phase-variable form of the state equations

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \vdots \\ \dot{x}_{n-1} \\ \dot{x}_n \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & \cdots & 0 & 0 \\ 0 & 0 & 1 & \cdots & 0 & 0 \\ 0 & 0 & 0 & \cdots & 0 & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots & \vdots \\ 0 & 0 & 0 & \cdots & 0 & 1 \\ -a_1 & -a_2 & -a_3 & \cdots & -a_{n-2} & -a_{n-1} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_{n-1} \\ x_n \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 0 \\ \vdots \\ 0 \\ b_0 \end{bmatrix} u$$

$$y = [1 \quad 0 \quad 0 \quad \cdots \quad 0 \quad 0] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_{n-1} \\ x_n \end{bmatrix}$$

### §5. Converting a Transfer Function to State-Space

- Ex.3.4

Converting a TF with Constant Term in Numerator

Find the state-space representation in phase-variable form for the TF

Solution

**Step 1** Find the associated differential equation

$$\frac{C(s)}{R(s)} = \frac{24}{s^3 + 9s^2 + 26s + 24}$$

$$\rightarrow (s^3 + 9s^2 + 26s + 24)C(s) = 24R(s)$$

Take the inverse Laplace transform, assuming zero initial conditions

$$\ddot{c} + 9\dot{c} + 26c + 24c = 24r$$

### §5. Converting a Transfer Function to State-Space

**Step 2** Select the state variables

Choosing the state variables  $x_1 = c$ ,  $x_2 = \dot{c}$ ,  $x_3 = \ddot{c}$

$$\begin{aligned} \dot{x}_1 &= x_2 \\ \dot{x}_2 &= x_3 \\ \dot{x}_3 &= -24x_1 - 26x_2 - 9x_3 + 24r \\ y &= c = x_1 \end{aligned}$$

In vector-matrix form

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -24 & -26 & -9 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 24 \end{bmatrix} r$$

$$y = [1 \quad 0 \quad 0] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

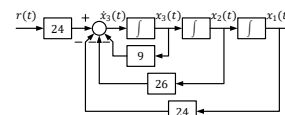
$$\ddot{c} + 9\dot{c} + 26c + 24c = 24r$$

### §5. Converting a Transfer Function to State-Space

An equivalent block diagram of the system

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -24 & -26 & -9 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 24 \end{bmatrix} r$$

$$y = [1 \quad 0 \quad 0] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$



### §5. Converting a Transfer Function to State-Space

MATLAB

ML

Run ch3p1 through ch3p4 in Appendix B

Learn how to use MATLAB to

- represent the system matrix A, the input matrix B, and the output matrix C
- convert a transfer function to the state-space representation in phase-variable form
- solve Ex.3.4

### §5. Converting a Transfer Function to State-Space

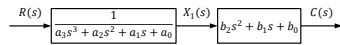
- If a TF has a polynomial in  $s$  in the numerator that is of order less than the polynomial in the denominator

$$\frac{R(s)}{C(s)} = \frac{b_2s^2 + b_1s + b_0}{a_3s^3 + a_2s^2 + a_1s + a_0}$$

the numerator and denominator can be handled separately: separate the transfer function into two cascaded TFs

$$\frac{R(s)}{C(s)} = \frac{1}{a_3s^3 + a_2s^2 + a_1s + a_0} X_1(s) \rightarrow \frac{b_2s^2 + b_1s + b_0}{C(s)}$$

### §5. Converting a Transfer Function to State-Space



- The first TF with just the denominator is converted to the phase-variable representation in state space, phase variable  $x_1$  is the output, and the rest of the phase variables are the internal variables of the first block
- The second TF with just the numerator yields

$$Y(s) = C(s) = (b_2s^2 + b_1s + b_0)X_1(s)$$

Taking the inverse Laplace transform with zero initial conditions

$$\begin{aligned} y(t) &= b_2 \frac{d^2x_1}{dt^2} + b_1 \frac{dx_1}{dt} + b_0x_1 \\ &= b_0x_1 + b_1x_2 + b_2x_3 \end{aligned}$$

Hence, the second block simply forms a specified linear combination of the state variables developed in the first block

### §5. Converting a Transfer Function to State-Space

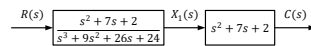
#### - Ex.3.5

#### Converting a TF with Polynomial in Numerator

Find the state-space representation of the transfer function

#### Solution

**Step 1** Separate the system into two cascaded blocks



**Step 2** Find the state equations for the block containing the denominator

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -24 & -26 & -9 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} r$$

### §5. Converting a Transfer Function to State-Space

**Step 3** Introduce the effect of the block with the numerator

The second block states that

$$C(s) = (b_2s^2 + b_1s + b_0)X_1(s) = (s^2 + 7s + 2)X_1(s)$$

Taking inverse Laplace transform with zero initial conditions

$$c = \ddot{x}_1 + 7\dot{x}_1 + 2x_1 = x_3 + 7x_2 + 2x_1$$

The output equation

$$y = \begin{bmatrix} b_0 & b_1 & b_2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} = \begin{bmatrix} 2 & 7 & 1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

An equivalent block diagram of the system

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -24 & -26 & -9 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} r$$

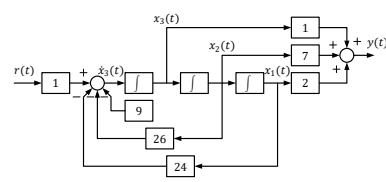
$$y = \begin{bmatrix} 2 & 7 & 1 \end{bmatrix} \begin{bmatrix} x_1 & x_2 & x_3 \end{bmatrix}^T$$

### §5. Converting a Transfer Function to State-Space

An equivalent block diagram of the system

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -24 & -26 & -9 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} r,$$

$$y = \begin{bmatrix} 2 & 7 & 1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$



### §5. Converting a Transfer Function to State-Space

#### Skill-Assessment Ex.3.3

**Problem** Find the state equations and output equation for the phase-variable representation of the TF

WileyPLUS

WPCS

Control Solutions

$$G(s) = \frac{2s + 1}{s^2 + 7s + 9}$$

#### Solution

#### First TF

$$\frac{X(s)}{R(s)} = \frac{1}{s^2 + 7s + 9} \rightarrow (s^2 + 7s + 9)X(s) = R(s)$$

Taking inverse Laplace transform with zero initial conditions

$$\ddot{x} + 7\dot{x} + 9x = r$$

Defining the state variables as  $x_1 = x$ ,  $x_2 = \dot{x}$

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = \ddot{x} = -7\dot{x} - x + r = -9x_1 - 7x_2 + r$$

### §5. Converting a Transfer Function to State-Space

#### Second TF

The output equation

$$c = 2\dot{x} + x = x_1 + 2x_2$$

Putting all equation in vector-matrix form

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -9 & -7 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} r$$

$$c = \begin{bmatrix} 1 & 2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$$

$$\dot{x}_2 = -9x_1 - 7x_2 + r, \quad c = x_1 + 2x_2$$





## §6. Converting from State Space to a Transfer Function

Symbolic Math

SM

Run ch3sp1 in Appendix F

Learn how to use the Symbolic Math Toolbox to

- write matrices and vectors
- solve Ex.3.6

## §6. Converting from State Space to a Transfer Function

## Skill-Assessment Ex.3.4

**Problem** Convert the state and output equations to a TF

$$\dot{x} = \begin{bmatrix} -4 & -1.5 \\ 4 & 0 \end{bmatrix} x + \begin{bmatrix} 2 \\ 0 \end{bmatrix} u(t), y = [1.5 \quad 0.625]x \quad (3.78)$$

**Solution**

$$sI - A = s \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} - \begin{bmatrix} -4 & -1.5 \\ 4 & 0 \end{bmatrix} = \begin{bmatrix} s+4 & 1.5 \\ -4 & s \end{bmatrix}$$

$$(sI - A)^{-1} = \frac{\text{adj}(sI - A)}{\det(sI - A)} = \frac{\begin{bmatrix} s & -1.5 \\ 4 & s+4 \end{bmatrix}}{s^2 + 4s + 6}$$

$$G(s) = C(sI - A)^{-1}B = [1.5 \quad 0.625] \frac{\begin{bmatrix} s & -1.5 \\ 4 & s+4 \end{bmatrix}}{s^2 + 4s + 6} \begin{bmatrix} 2 \\ 0 \end{bmatrix}$$

$$= \frac{3s + 5}{s^2 + 4s + 6}$$

## §6. Converting from State Space to a Transfer Function

## TryIt 3.2

Use the following MATLAB and the Control System Toolbox statements to obtain the transfer function shown in Skill-Assessment Exercise 3.4 from the state-space representation of Eq. (3.78).

```
A=[-4 -1.5; 4 0];
B=[2 0]';
C=[1.5 0.625];
D=0;
T=ss(A,B,C,D);
T=tf(T)
```

Matlab A=[-4 -1.5; 4 0]; B=[2 0]';  
C=[1.5 0.625]; D=0;  
T=ss(A,B,C,D); T=tf(T)

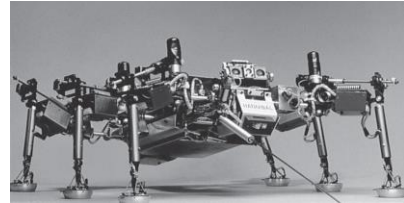
Result T =

$$3s + 5$$

$$s^2 + 4s + 6$$

Continuous-time transfer function

## §7. Linearization



Walking robots, such as Hannibal shown here, can be used to explore hostile environments and rough terrain, such as that found on other planets or inside volcanoes

## §7. Linearization

## - Ex.3.7

## Representing a Nonlinear System

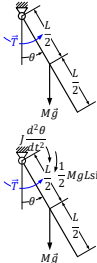
First represent the simple pendulum in state space ( $Mg$ : weight,  $T$ : applied torque in the  $\theta$  direction, and  $L$ : length). Assume the mass is evenly distributed, with the center of mass at  $L/2$ . Then linearize the state equations about the pendulum's equilibrium point - the vertical position with zero angular velocity

**Solution**

Drawing the free body diagram

Summing the torques

$$J \frac{d^2\theta}{dt^2} + \frac{MgL}{2} \sin\theta = T$$



## §7. Linearization

$$J \frac{d^2\theta}{dt^2} + \frac{MgL}{2} \sin\theta = T$$

Letting  $x_1 = \theta$ ,  $x_2 = d\theta/dt$ , the state equation

$$\dot{x}_1 = x_2 \quad (3.80.a)$$

$$\dot{x}_2 = -\frac{MgL}{2J} \sin x_1 + \frac{T}{J} \quad (3.80.b)$$

The nonlinear Eq. (3.80) represent a valid and complete model of the pendulum in state space even under nonzero initial conditions and even if parameters are time varying

To apply classical techniques and convert these state equations to a transfer function  $\rightarrow$  The nonlinear must be linearized

## §7. Linearization

Linearize the equation about the equilibrium point,  $x_1 = \theta = 0$ ,  $x_2 = d\theta/dt = 0$ . Let  $x_1$  and  $x_2$  be perturbed about the equilibrium point, or

$$x_1 = 0 + \delta x_1$$

$$x_2 = 0 + \delta x_2$$

Using Eqs. (2.182)

$$\sin x_1 - \sin 0 = \frac{d(\sin x_1)}{dx_1} \bigg|_{x_1=0} \delta x_1 = \delta x_1 \rightarrow \sin x_1 = \delta x_1$$

The state equations now become

$$\dot{\delta x}_1 = \delta x_2$$

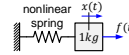
$$\dot{\delta x}_2 = -\frac{MgL}{2J} \delta x_1 + \frac{T}{J}$$

$$f(x) - f(x_0) \approx \frac{df}{dx} \bigg|_{x=x_0} (x - x_0) \quad (2.182), \quad \dot{x}_1 = x_2 \quad (3.80.a), \quad \dot{x}_2 = -\frac{MgL}{2J} \sin x_1 + \frac{T}{J} \quad (3.80.b)$$

## §7. Linearization

## Skill-Assessment Ex.3.5

**Problem** Represent the translational mechanical system in state space about the equilibrium displacement. The spring is nonlinear  $f_s(t) = 2x_s^2(t)$ . The applied force is  $f(t) = 10 + \delta f(t)$ , where  $\delta f(t)$  is a small force about the 10N constant value. Assume the output to be the displacement of the mass,  $x(t)$

**Solution**

The equation of motion

$$\frac{d^2 x}{dt^2} + 2x^2 = 10 + \delta f(t) \quad (1)$$

Letting  $x = x_0 + \delta x$

$$\frac{d^2(x_0 + \delta x)}{dt^2} + 2(x_0 + \delta x)^2 = 10 + \delta f(t) \quad (2)$$

## §7. Linearization

Linearize  $x^2$  at  $x_0$

$$(x_0 + \delta x)^2 - x_0^2 = \frac{d(x^2)}{dx} \bigg|_{x_0} \delta x = 2x_0 \delta x$$

$$\rightarrow (x_0 + \delta x)^2 = x_0^2 + 2x_0 \delta x \quad (3)$$

Substituting Eq.(3) into Eq.(1)

$$\frac{d^2 \delta x}{dt^2} + 4x_0 \delta x = -2x_0^2 + 10 + \delta f(t) \quad (4)$$

The force of the spring at equilibrium  $F = 10 = 2x_0^2 \rightarrow x_0 = \sqrt{5}$

Substituting this value of  $x_0$  into Eq.(4)

$$\frac{d^2 \delta x}{dt^2} + 4\sqrt{5} \delta x = \delta f(t)$$

$$f(x) - f(x_0) \approx \frac{df}{dx} \bigg|_{x=x_0} (x - x_0) \quad (2.182), \quad \frac{d^2 x}{dt^2} + 2x^2 = 10 + \delta f(t) \quad (1)$$

## §7. Linearization

$$\frac{d^2 \delta x}{dt^2} + 4\sqrt{5} \delta x = \delta f(t)$$

Selecting the state variables  $x_1 = \delta x$ ,  $x_2 = \dot{\delta x}$

The state and output equations

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = \ddot{x} = -4\sqrt{5}x_1 + \delta f(t)$$

$$y = x_1$$

Converting to vector-matrix form

$$\dot{\mathbf{x}} = \begin{bmatrix} 0 & 1 \\ -4\sqrt{5} & 0 \end{bmatrix} \mathbf{x} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} \delta f(t)$$

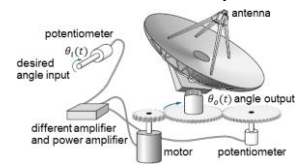
$$y = \begin{bmatrix} 1 & 0 \end{bmatrix} \mathbf{x}$$

## §8. Case Studies

## §8. Case Studies

## 1. Antenna Control: State-Space Representation

**Problem** Find the state-space representation in phase-variable form for each dynamic subsystem in the antenna azimuth position control. By dynamic, we mean that the system does not reach the steady state instantaneously. A pure gain, on the other hand, is an example of a non dynamic system, since the steady state is reached instantaneously



## §8. Case Studies

The transfer function of the power amplifier is given on the front endpapers as  $G(s) = 100/(s + 100)$ . We will convert this transfer function to its state-space representation. Letting  $v_p(t)$  represent the power amplifier input and  $e_a(t)$  represent the power amplifier output

$$G(s) = \frac{E_a(s)}{V_p(s)} = \frac{100}{s + 100} \quad (3.85)$$

Cross-multiplying,  $(s + 100)E_a(s) = 100V_p(s)$ , from which the differential equation can be written as

$$\frac{de_a(t)}{dt} + 100e_a(t) = 100v_p(t) \quad (3.86)$$

## §8. Case Studies

## 2. Pharmaceutical Drug Absorption

**Problem** In the pharmaceutical industry we want to describe the distribution of a drug in the body. A simple model divides the process into compartments: the dosage, the absorption site, the blood, the peripheral compartment, and the urine. The rate of change of the amount of a drug in a compartment is equal to the input flow rate diminished by the output flow rate. Figure 3.16 summarizes the system. Here each  $x_i$  is the amount of drug in that particular compartment (Lordi, 1972). Represent the system in state space, where the outputs are the amounts of drug in each compartment