

## 07. Steady-State Error

### §1. Introduction

- Control systems analysis and design focus on three specifications
  - transient response
  - stability
  - steady-state errors

taking into account the robustness of the design along with economic and social considerations

- Control system design entails trade-offs between desired
  - transient response,
  - steady-state error, and
  - the requirement that the system be stable

### §1. Introduction

- In order to explain how these test signals are used, let us assume a position control system, where the output position follows the input commanded position

Satellite in geostationary orbit  
Satellite orbiting at constant velocity

Accelerating missile



- **Step inputs** represent constant position and thus are useful in determining the ability of the control system to position itself with respect to a stationary target, such as a satellite in geostationary orbit

**Ex.:** An antenna position control is a system that can be tested for accuracy using step inputs

### Learning Outcome

After completing this chapter, the student will be able to

- Find the steady-state error for a unity feedback system
- Specify a system's steady-state error performance
- Design the gain of a closed-loop system to meet a steady-state error specification
- Find the steady-state error for disturbance inputs
- Find the steady-state error for nonunity feedback systems
- Find the steady-state error sensitivity to parameter changes
- Find the steady-state error for systems represented in state space

### §1. Introduction

#### Definition and Test Inputs

- Steady-state error is the difference between the input and the output for a prescribed test input as  $t \rightarrow \infty$ . Test inputs used for steady-state error analysis and design are summarized

Test waveforms for evaluating steady-state errors of position control systems

Waveform	Name	Physical interpretation	Time function	Laplace transform
	step	constant position	1	$\frac{1}{s}$
	ramp	constant velocity	t	$\frac{1}{s^2}$
	parabola	constant acceleration	$\frac{1}{2}t^2$	$\frac{1}{s^3}$

### §1. Introduction

- **Ramp inputs** represent constant-velocity inputs to a position control system, and can be used to test a system's ability to follow a linearly increasing input or, equivalently, to track a constant velocity target

Satellite in geostationary orbit  
Satellite orbiting at constant velocity

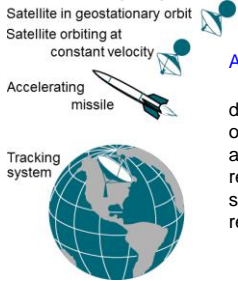
Accelerating missile



**Ex.:** A position control system that tracks a satellite that moves across the sky at a constant angular velocity, would be tested with a ramp input to evaluate the steady-state error between the satellite's angular position and that of the control system

## §1. Introduction

- Parabola inputs represent constant acceleration inputs to position control systems and can be used to represent accelerating targets, such as the missile, to determine the steady-state error performance

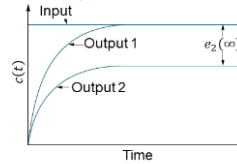


## Application to Stable Systems

Since we are concerned with the difference between the input and the output of a feedback control system after the steady state has been reached, our discussion is limited to stable systems, where the natural response approaches zero as  $t \rightarrow \infty$

## §1. Introduction

## Evaluating Steady-State Errors

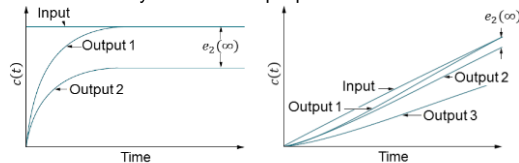


Steady-state error with step input

- Consider the system with step input
- output 1 has zero steady-state error
- output 2 has a finite steady-state error,  $e_2(\infty)$

## §1. Introduction

- Consider the system with ramp input



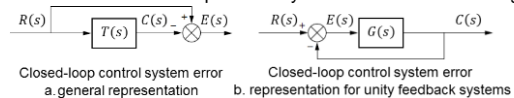
Steady-state error with step input

Steady-state error with step ramp input

- output 1 has zero steady-state error
- output 2 has a finite steady-state error,  $e_2(\infty)$ , as measured vertically after the transients have died down
- output 3 has infinite steady-state error, as measured vertically after the transients have died down, if the output's slope is different from that of the input

## §1. Introduction

- Consider the closed-loop control system error as the following



Closed-loop control system error

a. general representation

b. representation for unity feedback systems

$G(s)$  : system transfer function

$T(s)$  : closed-loop transfer function

$E(s)$  : error, the difference between the input and the output

⇒ study the steady-state, or final, value of  $e(t)$

- First, study and derive expressions for the steady-state error for unity feedback systems
- Then, expand to nonunity feedback systems

## §1. Introduction

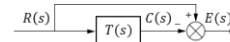
## Sources of Steady-State Error

Many steady-state errors in control systems arise from nonlinear sources, such as backlash in gears or a motor that will not move unless the input voltage exceeds a threshold

⇒ study the steady state errors that arise from the configuration of the system itself and the type of applied input

- steady-state error for unity feedback systems
- static error constants and system type

## §2. Steady-State Error for Unity Feedback Systems

Steady-State Error in Terms of  $T(s)$ 

Closed-loop control system error - general representation

The error between the input,  $R(s)$ , and the output,  $C(s)$

$$E(s) = R(s) - C(s) \quad (7.2)$$

$$C(s) = R(s)T(s) \quad (7.3)$$

$$\Rightarrow E(s) = R(s)[1 - T(s)] \quad (7.4)$$

Applying the final value theorem

$$e(\infty) = \lim_{t \rightarrow \infty} e(t) = \lim_{s \rightarrow 0} sE(s) \quad (7.5)$$

$$= \lim_{s \rightarrow 0} sR(s)[1 - T(s)] \quad (7.6)$$

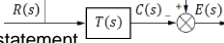
## §2. Steady-State Error for Unity Feedback Systems

- Ex.7.1

Steady-State Error in Terms of  $T(s)$ 

Find the steady-state error for the system  $T(s) = 5/(s^2 + 7s + 10)$  if the input is a unit step

Solution



From the problem statement

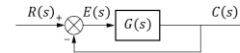
$$T(s) = \frac{5}{s^2 + 7s + 10}, \quad R(s) = \frac{1}{s}$$

$$\text{The error } E(s) = R(s)[1 - T(s)] = \frac{s^2 + 7s + 5}{s(s^2 + 7s + 10)}$$

Since  $T(s)$  is stable and, subsequently,  $E(s)$  does not have RHP poles or  $j\omega$  poles other than at the origin, apply the final value theorem

$$e(\infty) = \lim_{s \rightarrow 0} sE(s) = \lim_{s \rightarrow 0} \frac{s^2 + 7s + 5}{s^2 + 7s + 10} = \frac{5}{10} = 0.5$$

## §2. Steady-State Error for Unity Feedback Systems

Steady-State Error in Terms of  $G(s)$ 

Closed-loop control system error - representation for unity feedback systems

Consider the unity feedback control system

$$E(s) = R(s) - C(s) \quad (7.8)$$

$$C(s) = E(s)G(s) \quad (7.9)$$

$$\Rightarrow E(s) = \frac{R(s)}{1 + G(s)} \quad (7.10)$$

Apply the final value theorem

$$e(\infty) = \lim_{s \rightarrow 0} \frac{sR(s)}{1 + G(s)} \quad (7.11)$$

## §2. Steady-State Error for Unity Feedback Systems

Step Input

$$e(\infty) = e_{\text{step}}(\infty) = \lim_{s \rightarrow 0} \frac{s(1/s)}{1 + G(s)} = \frac{1}{1 + \lim_{s \rightarrow 0} G(s)} \quad (7.12)$$

In order to have zero steady-state error, the dc gain,  $\lim_{s \rightarrow 0} G(s)$ , of the forward transfer function

$$\lim_{s \rightarrow 0} G(s) = \infty \quad (7.13)$$

To satisfy Eq. (7.13),  $G(s)$  must take on the following form

$$G(s) = \frac{(s + z_1) \cdots}{s^n (s + p_1) \cdots} \Rightarrow s^n (s + p_1)(s + p_2) \cdots \rightarrow 0 \quad (7.14)$$

 $n \geq 1$ : at least one pure integration in the forward path $n = 0$ :  $\lim_{s \rightarrow 0} G(s) = \frac{z_1 z_2 \cdots}{p_1 p_2 \cdots} \neq \infty \Rightarrow$  finite steady-state error

## §2. Steady-State Error for Unity Feedback Systems

Ramp Input

$$e(\infty) = e_{\text{ramp}}(\infty) = \lim_{s \rightarrow 0} \frac{s(1/s^2)}{1 + G(s)} = \frac{1}{\lim_{s \rightarrow 0} sG(s)} \quad (7.16)$$

To have zero steady-state error for a ramp input

$$\lim_{s \rightarrow 0} sG(s) = \infty \quad (7.17)$$

To satisfy Eq. (7.17),  $G(s)$  must take on the following form

$$G(s) = \frac{(s + z_1) \cdots}{s^n (s + p_1) \cdots} \Rightarrow s^n (s + p_1)(s + p_2) \cdots \rightarrow 0 \quad (7.14)$$

 $n \geq 2$ : at least two integrations in the forward path $n = 1$ :  $\lim_{s \rightarrow 0} sG(s) = \frac{z_1 z_2 \cdots}{p_1 p_2 \cdots} \neq \infty \Rightarrow$  finite steady-state error $n = 0$ :  $\lim_{s \rightarrow 0} sG(s) = 0 \Rightarrow$  diverging ramps

## §2. Steady-State Error for Unity Feedback Systems

Parabolic Input

$$e(\infty) = e_{\text{parabola}}(\infty) = \lim_{s \rightarrow 0} \frac{s(1/s^3)}{1 + G(s)} = \frac{1}{\lim_{s \rightarrow 0} s^2 G(s)} \quad (7.20)$$

To have zero steady-state error for a parabolic input

$$\lim_{s \rightarrow 0} s^2 G(s) = \infty \quad (7.21)$$

To satisfy Eq. (7.21),  $G(s)$  must take on the following form

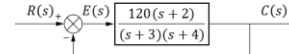
$$G(s) = \frac{(s + z_1) \cdots}{s^n (s + p_1) \cdots} \Rightarrow s^n (s + p_1)(s + p_2) \cdots \rightarrow 0 \quad (7.14)$$

 $n \geq 3$ : at least three integrations in the forward path $n = 2$ :  $\lim_{s \rightarrow 0} s^2 G(s) = \frac{z_1 z_2 \cdots}{p_1 p_2 \cdots} \neq \infty \Rightarrow$  finite steady-state error $n \leq 1$ :  $\lim_{s \rightarrow 0} s^2 G(s) = 0 \Rightarrow$  infinite steady-state error

## §2. Steady-State Error for Unity Feedback Systems

- Ex.7.2 Steady-State Errors for Systems with No Integrations

Find the steady-state errors for inputs of  $5u(t)$ ,  $5tu(t)$ ,  $5t^2(t)$  to the system

The function  $u(t)$  is the unit step

Solution

The closed-loop system is stable

$$5u(t) \text{ or } 5/s: \quad e_{\text{step}}(\infty) = 5 \times \frac{1}{1 + \lim_{s \rightarrow 0} G(s)} = \frac{5}{1 + 20} = \frac{5}{21}$$

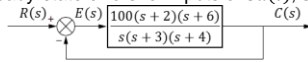
$$5tu(t) \text{ or } 5/s^2: \quad e_{\text{ramp}}(\infty) = 5 \times \frac{1}{\lim_{s \rightarrow 0} sG(s)} = \frac{5}{0} = \infty$$

$$5t^2u(t) \text{ or } 10/s^3: \quad e_{\text{parabola}}(\infty) = 10 \times \frac{1}{\lim_{s \rightarrow 0} s^2 G(s)} = \frac{10}{0} = \infty$$

## §2. Steady-State Error for Unity Feedback Systems

### - Ex.7.3 Steady-State Errors for Systems with One Integration

Find the steady-state errors for inputs of  $5u(t)$ ,  $5tu(t)$ ,  $5t^2(t)$  to the system



The function  $u(t)$  is the unit step

#### Solution

The closed-loop system is stable

$$5u(t) \text{ or } 5/s: e_{\text{step}}(\infty) = 5 \times \frac{1}{1 + \lim_{s \rightarrow 0} G(s)} = \frac{5}{\infty} = 0$$

$$5tu(t) \text{ or } 5/s^2: e_{\text{ramp}}(\infty) = 5 \times \frac{1}{\lim_{s \rightarrow 0} sG(s)} = \frac{5}{100} = \frac{1}{20}$$

$$5t^2u(t) \text{ or } 10/s^3: e_{\text{parabola}}(\infty) = 10 \times \frac{1}{\lim_{s \rightarrow 0} s^2 G(s)} = \frac{10}{0} = \infty$$

## §2. Steady-State Error for Unity Feedback Systems

### Skill-Assessment Ex.7.1

**Problem** A unity feedback system has the following forward TF

WileyPLUS  
WPCS  
Control Solutions

$$G(s) = \frac{10(s+20)(s+30)}{s(s+25)(s+35)}$$

- a. Find the steady-state error for the following inputs  $15u(t)$ ,  $15tu(t)$ , and  $15t^2(t)$

b. Repeat for

$$G(s) = \frac{10(s+20)(s+30)}{s^2(s+25)(s+35)(s+50)}$$

## §2. Steady-State Error for Unity Feedback Systems

#### Solution

- a. First check stability

$$T(s) = \frac{G(s)}{1 + G(s)} = \frac{10s^2 + 500s + 6000}{s^3 + 70s^2 + 1375s + 6000}$$

$$= \frac{10(s+30)(s+20)}{(s+26.03)(s+37.89)(s+6.085)}$$

$\Rightarrow$  the closed-loop system is stable

$$5u(t): e_{\text{step}}(\infty) = \frac{15}{1 + \lim_{s \rightarrow 0} G(s)} = \frac{5}{1 + \infty} = 0$$

$$5tu(t): e_{\text{ramp}}(\infty) = \frac{15}{\lim_{s \rightarrow 0} sG(s)} = \frac{15}{\frac{10 \times 20 \times 30}{25 \times 35}} = 2.1875$$

$$5t^2u(t): e_{\text{parabola}}(\infty) = \frac{30}{\lim_{s \rightarrow 0} s^2 G(s)} = \frac{30}{0} = \infty$$

## §2. Steady-State Error for Unity Feedback Systems

- b. First check stability

$$T(s) = \frac{G(s)}{1 + G(s)} = \frac{10s^2 + 500s + 6000}{s^5 + 110s^4 + 3875s^3 + 437 \times 10^4 s^2 + 500s + 6000}$$

$$= \frac{10(s+30)(s+20)}{(s+5001)(s+35)(s+25)(s^2 - 7.189 \times 10^{-4}s + 0.1372)}$$

$\Rightarrow$  From the second-order term in the denominator, the system is unstable. Instability could also be determined using the Routh-Hurwitz criteria on the denominator of  $T(s)$

Since the system is unstable, calculations about steady-state error cannot be made

## §3. Static Error Constants and System Type

### Static Error Constants

- The relationships for steady-state error

• For a step input,  $u(t)$  
$$e_{\text{step}}(\infty) = \frac{1}{1 + \lim_{s \rightarrow 0} G(s)}$$

• For a ramp input,  $tu(t)$  
$$e_{\text{ramp}}(\infty) = \frac{1}{\lim_{s \rightarrow 0} sG(s)}$$

• For a parabolic input,  $\frac{1}{2}t^2u(t)$  
$$e_{\text{parabola}}(\infty) = \frac{1}{\lim_{s \rightarrow 0} s^2 G(s)}$$

- The limits static error constants

• Position constant,  $K_p$  
$$K_p = \lim_{s \rightarrow 0} G(s)$$

• Velocity constant,  $K_v$  
$$K_v = \lim_{s \rightarrow 0} sG(s)$$

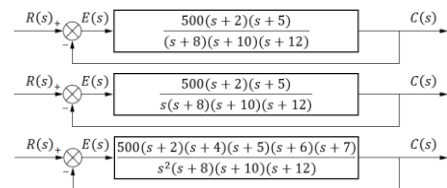
• Acceleration constant,  $K_a$  
$$K_a = \lim_{s \rightarrow 0} s^2 G(s)$$

## §3. Static Error Constants and System Type

### - Ex.7.4

### Steady-State Error via Static Error Constants

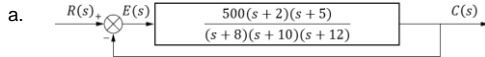
Evaluate the static error constants and find the expected error for the standard step, ramp, and parabolic inputs



#### Solution

All closed-loop systems are indeed stable

## §3. Static Error Constants and System Type



- The limits static error constants

$$K_p = \lim_{s \rightarrow 0} G(s) = \frac{500 \times 2 \times 5}{8 \times 10 \times 12} = 5.208$$

$$K_v = \lim_{s \rightarrow 0} sG(s) = 0$$

$$K_a = \lim_{s \rightarrow 0} s^2 G(s) = 0$$

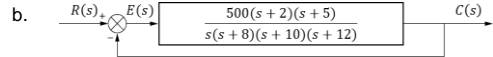
- The steady-state error

$$e_{\text{step}}(\infty) = \frac{1}{1 + K_p} = \frac{1}{1 + 5.208} = 0.161$$

$$e_{\text{ramp}}(\infty) = \frac{1}{K_v} = \frac{1}{0} = \infty$$

$$e_{\text{parabola}}(\infty) = \frac{1}{K_a} = \frac{1}{0} = \infty$$

## §3. Static Error Constants and System Type



- The limits static error constants

$$K_p = \lim_{s \rightarrow 0} G(s) = \infty$$

$$K_v = \lim_{s \rightarrow 0} sG(s) = \frac{500 \times 2 \times 5 \times 6}{8 \times 10 \times 12} = 31.25$$

$$K_a = \lim_{s \rightarrow 0} s^2 G(s) = 0$$

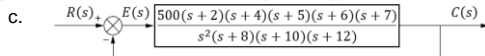
- The steady-state error

$$e_{\text{step}}(\infty) = \frac{1}{1 + K_p} = \frac{1}{1 + \infty} = 0$$

$$e_{\text{ramp}}(\infty) = \frac{1}{K_v} = \frac{1}{31.25} = 0.032$$

$$e_{\text{parabola}}(\infty) = \frac{1}{K_a} = \frac{1}{0} = \infty$$

## §3. Static Error Constants and System Type



- The limits static error constants

$$K_p = \lim_{s \rightarrow 0} G(s) = \infty$$

$$K_v = \lim_{s \rightarrow 0} sG(s) = \infty$$

$$K_a = \lim_{s \rightarrow 0} s^2 G(s) = \frac{500 \times 2 \times 4 \times 5 \times 6 \times 7}{8 \times 10 \times 12} = 875$$

- The steady-state error

$$e_{\text{step}}(\infty) = \frac{1}{1 + K_p} = \frac{1}{1 + \infty} = 0$$

$$e_{\text{ramp}}(\infty) = \frac{1}{K_v} = \frac{1}{\infty} = 0$$

$$e_{\text{parabola}}(\infty) = \frac{1}{K_a} = \frac{1}{875} = 1.14 \times 10^{-3}$$

## §3. Static Error Constants and System Type



Run ch7p1 in Appendix B

Learn how to use MATLAB to

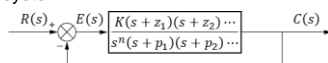
- test the system for stability
- evaluate static error constants
- calculate steady-state error
- solve Ex.7.4 with System (b)

## §3. Static Error Constants and System Type

## System Type

- The values of the static error constants, again, depend upon the form of  $G(s)$ , especially the number of pure integrations in the forward path

- Given the system



Feedback control system for defining system type

define **system type** to be the value of  $n$  in the denominator or, equivalently, the number of pure integrations in the forward path

- $n = 0$  : type 0 system
- $n = 1$  : type 1 system
- $n = 2$  : type 2 system

## §3. Static Error Constants and System Type

- Relationships between input, system type, static error constants, and steady-state errors

Input	Steady-state error formula	Type 0		Type 1		Type 2	
		Static error constant	Error	Static error constant	Error	Static error constant	Error
Step, $u(t)$	$\frac{1}{1 + K_p}$	$K_p = \text{Constant}$	$\frac{1}{1 + K_p}$	$K_p = \infty$	0	$K_p = \infty$	0
Ramp, $tu(t)$	$\frac{1}{K_v}$	$K_v = 0$	$\infty$	$K_v = \text{Constant}$	$\frac{1}{K_v}$	$K_v = \infty$	0
Parabola, $\frac{1}{2}t^2 u(t)$	$\frac{1}{K_a}$	$K_a = 0$	$\infty$	$K_a = 0$	$\infty$	$K_a = \text{Constant}$	$\frac{1}{K_a}$

### §3. Static Error Constants and System Type

#### Skill-Assessment Ex.7.2

**Problem** A unity feedback system has the following forward TF

WileyPLUS  
WPCS  
Control Solutions

$$G(s) = \frac{1000(s+8)}{(s+7)(s+9)}$$

- Evaluate system type,  $K_p$ ,  $K_v$ , and  $K_a$
- Use your answers to (a.) to find the steady-state errors for the standard step, ramp, and parabolic inputs

**Solution** The system is stable

### §3. Static Error Constants and System Type

a.

The closed-loop transfer function

$$\begin{aligned} T(s) &= \frac{G(s)}{1+G(s)} \\ &= \frac{1000(s+8)}{(s+9)(s+7)+1000(s+8)} \\ &= \frac{1000(s+8)}{s^2+1016s+8063} \end{aligned}$$

The system is Type 0. Therefore

$$\begin{aligned} K_p &= \lim_{s \rightarrow 0} G(s) = \frac{1000 \times 8}{7 \times 9} = 127 \\ K_v &= \lim_{s \rightarrow 0} sG(s) = 0 \\ K_a &= \lim_{s \rightarrow 0} s^2G(s) = 0 \end{aligned}$$

### §3. Static Error Constants and System Type

b.

The steady-state error

$$\begin{aligned} e_{\text{step}}(\infty) &= \frac{1}{1+K_p} = \frac{1}{1+127} = 7.8 \times 10^{-3} \\ e_{\text{ramp}}(\infty) &= \frac{1}{K_v} = \frac{1}{0} = \infty \\ e_{\text{parabola}}(\infty) &= \frac{1}{K_a} = \frac{1}{0} = \infty \end{aligned}$$

### §3. Static Error Constants and System Type

TryIt 7.1

Use MATLAB, the Control System Toolbox, and the following statements to find  $K_p$ ,  $e_{\text{step}}(\infty)$ , and the closed-loop poles to check for stability for the system of Skill-Assessment Ex.7.2

$$G(s) = \frac{1000(s+8)}{(s+7)(s+9)}$$

```
numg=1000*[1 8];
deng=poly([-7 -9]);
G=tf(numg,deng);
Kp=dcgain(G);
estep=1/(1+Kp);
T=feedback(G,1);
poles=pole(T)
```

### §4. Steady-State Error Specifications

A robot used in the manufacturing of semiconductor random-access memories (RAMs) similar to those in personal computers. Steady-state error is an important design consideration for assembly-line robots



### §4. Steady-State Error Specifications

- The specifications for a control system's transient response

- damping ratio,  $\zeta$
- settling time,  $T_s$
- peak time,  $T_p$
- percent overshoot, %OS

- The specification of a static error constant

- the position constant,  $K_p$
- velocity constant,  $K_v$
- acceleration constant,  $K_a$

#### §4. Steady-State Error Specifications

- For example, if a control system has the specification  $K_v = 1000$ , we can draw several conclusions

- The system is stable
- The system is of Type 1
- A ramp input is the test signal
- The steady-state error between the input ramp and the output ramp is  $1/K_v$  per unit of input slope

Input	Type 0			Type 1			Type 2		
	Steady-state error formula	Static error constant	Error	Static error constant	Error	Static error constant	Static error constant	Error	Error
Step, $u(t)$	$\frac{1}{1+K_p}$	$K_p = \text{Constant}$	$\frac{1}{1+K_p}$	$K_p = \infty$	0	$K_p = \infty$	0	0	0
Ramp, $tu(t)$	$\frac{1}{K_v}$	$K_v = 0$	$\infty$	$K_v = \text{Constant}$	$\frac{1}{K_v}$	$K_v = \infty$	0	0	0
Parabola, $\frac{1}{2}t^2u(t)$	$\frac{1}{K_a}$	$K_a = 0$	$\infty$	$K_a = 0$	$\infty$	$K_a = \text{Constant}$	$\frac{1}{K_a}$	0	0

#### §4. Steady-State Error Specifications

##### - Ex.7.5

##### Interpreting the Steady-State Error Specification

What information is contained in the specification  $K_p = 1000$ ?

##### Solution

The system is stable

The system is Type 0

The input test signal is a step

$$\text{The error per unit step is } e_{\text{step}}(\infty) = \frac{1}{1+K_p} = \frac{1}{1+1000} = \frac{1}{1001}$$

Input	Type 0			Type 1			Type 2		
	Steady-state error formula	Static error constant	Error	Static error constant	Error	Static error constant	Static error constant	Error	Error
Step, $u(t)$	$\frac{1}{1+K_p}$	$K_p = \text{Constant}$	$\frac{1}{1+K_p}$	$K_p = \infty$	0	$K_p = \infty$	0	0	0
Ramp, $tu(t)$	$\frac{1}{K_v}$	$K_v = 0$	$\infty$	$K_v = \text{Constant}$	$\frac{1}{K_v}$	$K_v = \infty$	0	0	0
Parabola, $\frac{1}{2}t^2u(t)$	$\frac{1}{K_a}$	$K_a = 0$	$\infty$	$K_a = 0$	$\infty$	$K_a = \text{Constant}$	$\frac{1}{K_a}$	0	0

#### §4. Steady-State Error Specifications

##### - Ex.7.6 Gain Design to Meet a Steady-State Error Specification



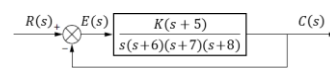
Find the value of  $K$  so that there is 10% error in the steady state

##### Solution

Since the system is Type 1, the error stated in the problem must apply to a ramp input; only a ramp yields a finite error in a Type 1 system

Input	Type 0			Type 1			Type 2		
	Steady-state error formula	Static error constant	Error	Static error constant	Error	Static error constant	Static error constant	Error	Error
Step, $u(t)$	$\frac{1}{1+K_p}$	$K_p = \text{Constant}$	$\frac{1}{1+K_p}$	$K_p = \infty$	0	$K_p = \infty$	0	0	0
Ramp, $tu(t)$	$\frac{1}{K_v}$	$K_v = 0$	$\infty$	$K_v = \text{Constant}$	$\frac{1}{K_v}$	$K_v = \infty$	0	0	0
Parabola, $\frac{1}{2}t^2u(t)$	$\frac{1}{K_a}$	$K_a = 0$	$\infty$	$K_a = 0$	$\infty$	$K_a = \text{Constant}$	$\frac{1}{K_a}$	0	0

#### §4. Steady-State Error Specifications



$$e(\infty) = \frac{1}{K_v} = 0.1$$

$$\Rightarrow K_v = 10 = \lim_{s \rightarrow 0} sG(s) = \frac{K \times 5}{6 \times 7 \times 8}$$

$$\Rightarrow K = 672$$

Applying the Routh-Hurwitz criterion, we see that the system is stable at this gain

Although this gain meets the criteria for steady-state error and stability, it may not yield a desirable transient response

#### §4. Steady-State Error Specifications



Run ch7p2 in Appendix B

Learn how to use MATLAB to

- find the gain to meet a steady-state error specification
- solves Ex.7.6

#### §4. Steady-State Error Specifications

##### Skill-Assessment Ex.7.3

##### Problem

WileyPLUS

Control Solutions

A unity feedback system has the following forward TF

$$G(s) = \frac{K(s+2)}{(s+14)(s+18)}$$

Find the value of  $K$  to yield a 10% error in the steady state

##### Solution

The system is stable for positive  $K$

For a step input

$$e_{\text{step}}(\infty) = \frac{1}{1+K_p} = 0.1$$

$$\Rightarrow K_p = 9 = \lim_{s \rightarrow 0} G(s) = \frac{12 \times K}{14 \times 18}$$

$$\Rightarrow K = 189$$

#### §4. Steady-State Error Specifications

##### TryIt 7.2

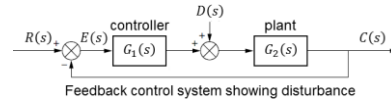
Use MATLAB, the Control System Toolbox, and the following statements to solve Skill-Assessment Ex.7.3 and check the resulting system for stability

$$G(s) = \frac{K(s+2)}{(s+14)(s+18)}$$

```
numg=[1 2];
deng=poly([-14 -18]);
G=tf(numg,deng);
Kpdk=dcgain(G);
estep=0.1;
K=(1/estep-1)/Kpdk;
T=feedback(G,1);
poles=pole(T)
```

#### §5. Steady-State Error for Disturbances

- Feedback control systems are used to compensate for disturbances or unwanted inputs that enter a system, result that regardless of these disturbances, the system can be designed to follow the input with small or zero error



Feedback control system showing disturbance

- Consider a feedback control system with a disturbance,  $D(s)$ , injected between the controller and the plant

$$C(s) = E(s)G_1(s)G_2(s) + D(s)G_2(s) \quad (7.58)$$

$$C(s) = R(s) - E(s) \quad (7.59)$$

$$E(s) = \frac{1}{1 + G_1(s)G_2(s)}R(s) - \frac{G_2(s)}{1 + G_1(s)G_2(s)}D(s) \quad (7.60)$$

#### §5. Steady-State Error for Disturbances

- To find the steady-state value of the error, apply the final value theorem to Eq. (7.60) and obtain

$$\begin{aligned} e(\infty) &= \lim_{s \rightarrow 0} sE(s) \\ &= \lim_{s \rightarrow 0} \frac{s}{1 + G_1(s)G_2(s)}R(s) - \lim_{s \rightarrow 0} \frac{sG_2(s)}{1 + G_1(s)G_2(s)}D(s) \\ &= e_R(\infty) + e_D(\infty) \end{aligned}$$

where,

- $e_R(\infty)$ : the steady-state error due to  $R(s)$

$$e_R(\infty) = \lim_{s \rightarrow 0} \frac{s}{1 + G_1(s)G_2(s)}R(s)$$

- $e_D(\infty)$ : the steady-state error due to the disturbance  $D(s)$

$$e_D(\infty) = -\lim_{s \rightarrow 0} \frac{sG_2(s)}{1 + G_1(s)G_2(s)}D(s)$$

$$E(s) = \frac{1}{1 + G_1(s)G_2(s)}R(s) - \frac{G_2(s)}{1 + G_1(s)G_2(s)}D(s) \quad (7.60)$$

#### §5. Steady-State Error for Disturbances

- Assume a step disturbance,  $D(s) = 1/s$

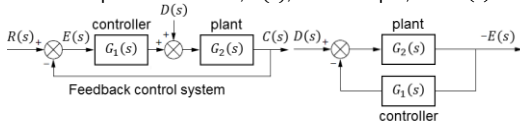
Substituting this value into the second term of Eq. (7.61),  $e_D(\infty)$ , the steady-state error component due to a step disturbance is found to be

$$e_D(\infty) = -\lim_{s \rightarrow 0} \frac{sG_2(s)}{1 + G_1(s)G_2(s)} \frac{1}{s} = -\frac{1}{\lim_{s \rightarrow 0} \frac{1}{G_2(s)} + \lim_{s \rightarrow 0} G_1(s)} \quad (7.62)$$

This equation shows that the steady-state error produced by a step disturbance can be reduced by increasing the dc gain of  $G_1(s)$  or decreasing the dc gain of  $G_2(s)$

#### §5. Steady-State Error for Disturbances

- Rearrange the system so that the disturbance,  $D(s)$ , is depicted as the input and the error,  $E(s)$ , as the output, with  $R(s) = 0$



Rearrange feedback system to show disturbance as input and error as output, with  $R(s) = 0$

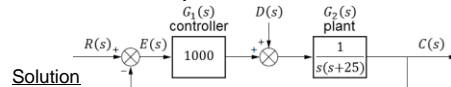
To minimize the steady-state value of  $E(s)$ , we must either

- increase the dc gain of  $G_1(s)$  so that a lower value of  $E(s)$  will be fed back to match the steady-state value of  $D(s)$ , or
- decrease the dc value of  $G_2(s)$ , which then yields a smaller value of  $e(\infty)$  as predicted by the feedback formula

#### §5. Steady-State Error for Disturbances

##### - Ex.7.7 Steady-State Error Due to Step Disturbance

Find the steady-state error component due to a step disturbance for the system



##### Solution

The system is stable

$$e_D(\infty) = -\frac{1}{\lim_{s \rightarrow 0} \frac{1}{G_2(s)} + \lim_{s \rightarrow 0} G_1(s)} = -\frac{1}{0 + 1000} = -\frac{1}{1000}$$

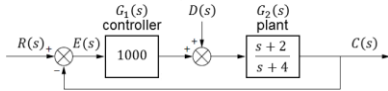
The result shows that the steady-state error produced by the step disturbance is inversely proportional to the dc gain of  $G_1(s)$ . The dc gain of  $G_2(s)$  is infinite in this example



### §5. Steady-State Error for Disturbances

#### Skill-Assessment Ex.7.4

**Problem** Evaluate the steady-state error component due to a step disturbance for the system



**Solution** The system is stable

For a step input

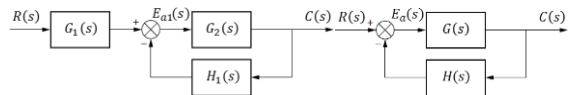
$$e_D(\infty) = -\frac{1}{\lim_{s \rightarrow 0} \frac{1}{G_2(s)} + \lim_{s \rightarrow 0} G_1(s)}$$

$$= -\frac{1}{\frac{1}{2+1000}}$$

$$= -9.98 \times 10^{-4}$$

### §6. Steady-State Error for Nonunity Feedback Systems

- A general feedback system, showing the input transducer,  $G_1(s)$ , controller and plant,  $G_2(s)$ , and feedback,  $H_1(s)$



- Pushing the input transducer to the right past the summing junction yields the general nonunity feedback system, where

$$G(s) = G_1(s)G_2(s)$$

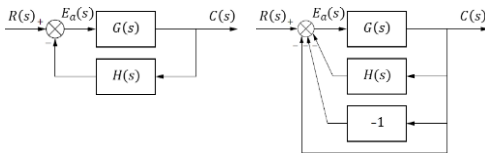
$$H(s) = H_1(s)/G_1(s)$$

$$E_a(s) : \text{actuating signal, } E_a(s) \neq E(s) = C(s) - R(s)$$

If  $r(t)$  and  $c(t)$  have the same units, the steady-state error can be found,  $e(\infty) = r(\infty) - c(\infty)$

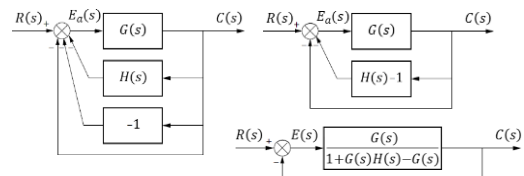
### §6. Steady-State Error for Nonunity Feedback Systems

- Form a unity feedback system by adding and subtracting unity feedback paths. This step requires that input and output units be the same



### §6. Steady-State Error for Nonunity Feedback Systems

- Combine  $H(s)$  with the negative unity feedback



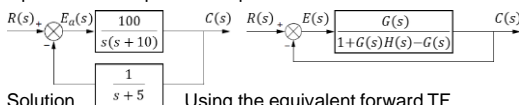
- Combine the feedback system consisting of  $G(s)$  and  $[H(s) - 1]$ , leaving an equivalent forward path and a unity feedback

Notice that the final figure shows  $E(s) = R(s) - C(s)$  explicitly

### §6. Steady-State Error for Nonunity Feedback Systems

- Ex.7.8 Steady-State Error for Nonunity Feedback Systems

Find the system type, the appropriate error constant associated with the system type, and the steady-state error for a unit step input. Assume input and output units are the same



**Solution** Using the equivalent forward TF

$$G_e(s) = \frac{G(s)}{1 + G(s)H(s) - G(s)}$$

$$= \frac{\frac{100}{s(s+10)}}{1 + \frac{100}{s(s+10)} \frac{1}{s+5} - \frac{100}{s(s+10)}}$$

$$\Rightarrow G_e(s) = \frac{100(s+5)}{s^3 + 15s^2 - 50s - 400}$$

### §6. Steady-State Error for Nonunity Feedback Systems

The equivalent forward TF

$$G_e(s) = \frac{100(s+5)}{s^3 + 15s^2 - 50s - 400}$$

Thus, the system is Type 0, since there are no pure integrations

The appropriate static error constant is then  $K_p$

$$K_p = \lim_{s \rightarrow 0} G_e(s) = \frac{100 \times 5}{-400} = -\frac{5}{4}$$

The steady-state error

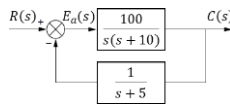
$$e(\infty) = \frac{1}{1 + K_p} = \frac{1}{1 - 5/4} = -4$$

The negative value for steady-state error implies that the output step is larger than the input step

### §6. Steady-State Error for Nonunity Feedback Systems

#### TryIt 7.3

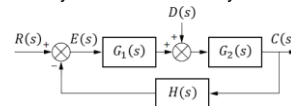
Use MATLAB, the Control System Toolbox, and the following statements to find  $G_e(s)$  in Ex.7.8



```
G=zpk([], [0 -10], 100);
H=zpk([], -5, 1);
Ge=feedback(G, (H-1));
'Ge(s)'
Ge=tf(Ge)
T=feedback(Ge, 1);
'Poles of T(s)'
pole(T)
```

### §6. Steady-State Error for Nonunity Feedback Systems

- Consider a nonunity feedback control system with disturbance



- The steady-state error for this system,  $e(\infty) = c(\infty) - r(\infty)$

$$e(\infty) = \lim_{s \rightarrow 0} s E(s) = \lim_{s \rightarrow 0} s \left\{ \left[ 1 - \frac{G_1 G_2}{1 + G_1 G_2 H} \right] R - \frac{G_2}{1 + G_1 G_2 H} D \right\} \quad (7.69)$$

with step inputs and step disturbances,  $R(s) = D(s) = 1/s$

$$e(\infty) = \left[ 1 - \frac{\lim_{s \rightarrow 0} (G_1 G_2)}{\lim_{s \rightarrow 0} (1 + G_1 G_2 H)} \right] - \frac{\lim_{s \rightarrow 0} G_2}{\lim_{s \rightarrow 0} (1 + G_1 G_2 H)} \quad (7.70)$$

### §6. Steady-State Error for Nonunity Feedback Systems

$$e(\infty) = \left[ 1 - \frac{\lim_{s \rightarrow 0} (G_1 G_2)}{\lim_{s \rightarrow 0} (1 + G_1 G_2 H)} \right] - \frac{\lim_{s \rightarrow 0} G_2}{\lim_{s \rightarrow 0} (1 + G_1 G_2 H)} \quad (7.70)$$

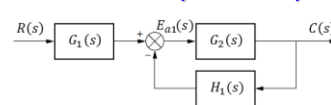
For zero error

$$\frac{\lim_{s \rightarrow 0} (G_1 G_2)}{\lim_{s \rightarrow 0} (1 + G_1 G_2 H)} = 1, \quad \frac{\lim_{s \rightarrow 0} G_2}{\lim_{s \rightarrow 0} (1 + G_1 G_2 H)} = 0 \quad (7.71)$$

The above two equations can always be satisfied if

- the system is stable
- $G_1(s)$  is a Type 1 system
- $G_2(s)$  is a Type 0 system
- $H(s)$  is a Type 0 system with a dc gain of unity

### §6. Steady-State Error for Nonunity Feedback Systems



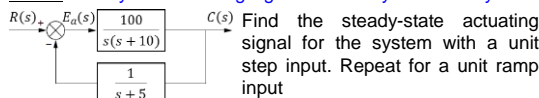
- The steady-state value of the actuating signal,  $E_{a1}(s)$ , for a general feedback system

$$e_{a1}(\infty) = \lim_{s \rightarrow 0} \frac{s R(s) G_1(s)}{1 + G_2(s) H_1(s)} \quad (7.72)$$

Note: There is no restriction that the input and output units be the same, since we are finding the steady-state difference between signals at the summing junction, which do have the same units

### §6. Steady-State Error for Nonunity Feedback Systems

#### - Ex.7.9 Steady-State Actuating Signal for Nonunity Feedback Systems



Find the steady-state actuating signal for the system with a unit step input. Repeat for a unit ramp input

Solution

$$e_a(\infty) = \lim_{s \rightarrow 0} \frac{s R(s) G_1(s)}{1 + G_2(s) H_1(s)} = \lim_{s \rightarrow 0} \frac{s R(s) \times 1}{1 + \frac{100}{s(s+10)} \times \frac{1}{s+5}}$$

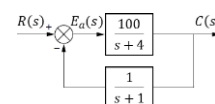
$$\text{For step input } e_a(\infty) = \lim_{s \rightarrow 0} \frac{s(1/s) \times 1}{1 + \frac{100}{s(s+10)} \times \frac{1}{s+5}} = 0$$

$$\text{For ramp input } e_a(\infty) = \lim_{s \rightarrow 0} \frac{s(1/s^2) \times 1}{1 + \frac{100}{s(s+10)} \times \frac{1}{s+5}} = \frac{1}{2}$$

### §6. Steady-State Error for Nonunity Feedback Systems

#### Skill-Assessment Ex.7.5

**Problem**  
WileyPLUS  
WPCS  
Control Solutions

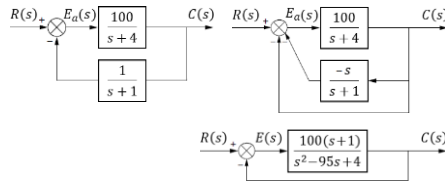


- Find the steady-state error,  $e(\infty) = c(\infty) - r(\infty)$ , for a unit step input given the nonunity feedback system. Repeat for a unit ramp input. Assume input and output units are the same
- Find the steady-state actuating signal,  $e_a(\infty)$ , for a unit step input given the nonunity feedback system. Repeat for a unit ramp input

### §6. Steady-State Error for Nonunity Feedback Systems

**Solution** The system is stable

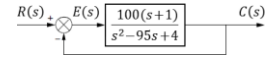
Create a unity-feedback system



$$H_e(s) = \frac{1}{s+1} - 1 = \frac{-s}{s+1}$$

$$G_e(s) = \frac{1}{1 + G(s)H_e(s)} = \frac{100(s+1)}{s^2 - 95s + 4}$$

### §6. Steady-State Error for Nonunity Feedback Systems



a. The steady-state error,  $e(\infty) = c(\infty) - r(\infty)$

The system is Type 0,  $K_p = \lim_{s \rightarrow 0} G(s) = 100/4 = 25$

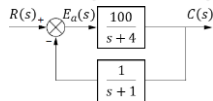
The steady-state error

$$e_{\text{step}}(\infty) = 1/(1 + K_p) = 0.0385$$

$$e_{\text{ramp}}(\infty) = \infty$$

Input	Steady-state error formula	Type 0		Type 1		Type 2	
		Static error constant	Error	Static error constant	Error	Static error constant	Error
Step, $u(t)$	$\frac{1}{1 + K_p}$	$K_p = \text{Constant}$	$\frac{1}{1 + K_p}$	$K_p = \infty$	0	$K_p = \infty$	0
Ramp, $tu(t)$	$\frac{1}{K_v}$	$K_v = 0$	$\infty$	$K_v = \text{Constant}$	$\frac{1}{K_v}$	$K_v = \infty$	0
Parabola, $\frac{1}{2}t^2u(t)$	$\frac{1}{K_a}$	$K_a = 0$	$\infty$	$K_a = 0$	$\infty$	$K_a = \text{Constant}$	$\frac{1}{K_a}$

### §6. Steady-State Error for Nonunity Feedback Systems



b. The steady-state actuating signal,  $e_a(\infty)$

$$e_a(\infty) = \lim_{s \rightarrow 0} \frac{sR(s)G_1(s)}{1 + G_2(s)H_1(s)} = \lim_{s \rightarrow 0} \frac{sR(s) \times 1}{1 + \frac{100}{s+4} \times \frac{1}{s+1}}$$

$$\text{step input } e_a(\infty) = \lim_{s \rightarrow 0} \frac{s(1/s) \times 1}{1 + \frac{100}{s+4} \times \frac{1}{s+1}} = \frac{1}{104} = 0.0385$$

$$\text{ramp input } e_a(\infty) = \lim_{s \rightarrow 0} \frac{s(1/s^2) \times 1}{1 + \frac{100}{s+4} \times \frac{1}{s+1}} = \frac{1}{0} = \infty$$

### §7. Sensitivity

**Sensitivity:** the ratio of the fractional change in the function to the fractional change in the parameter as the fractional change of the parameter approaches zero

$$S_{F:P} = \lim_{\Delta P \rightarrow 0} \frac{\text{Fractional change in the function, } F}{\text{Fractional change in the parameter, } P}$$

$$= \lim_{\Delta P \rightarrow 0} \frac{\Delta F / F}{\Delta P / P}$$

$$= \lim_{\Delta P \rightarrow 0} \frac{P \Delta F}{F \Delta P}$$

$$\Rightarrow S_{F:P} = \frac{P \delta F}{F \delta P} \quad (7.75)$$

### §7. Sensitivity

- Ex.7.10

**Sensitivity of a Closed-Loop Transfer Function**

Calculate the sensitivity of the closed-loop transfer function to changes in the parameter  $a$ . How would you reduce the sensitivity?

**Solution**

The closed-loop transfer function

$$T(s) = \frac{K}{s^2 + as + K}$$

The sensitivity

$$S_{T:a} = \frac{a}{T} \frac{\delta T}{\delta a} = \frac{a}{\frac{K}{s^2 + as + K}} \frac{-Ks}{\left(\frac{K}{s^2 + as + K}\right)^2} = \frac{-as}{s^2 + as + K}$$

An increase in  $K$  reduces the sensitivity of the closed-loop transfer function to changes in the parameter  $a$

### §7. Sensitivity

- Ex.7.11

**Sensitivity of Steady-State Error with Ramp Input**

Find the sensitivity of the steady-state error to changes in parameter  $K$  and parameter  $a$  with ramp inputs

**Solution**

The steady-state error for the system

$$e(\infty) = 1/K_v = a/K$$

The sensitivity of  $e(\infty)$  to changes in parameter  $a$

$$S_{e:a} = \frac{a}{e} \frac{\delta e}{\delta a} = \frac{a}{a/K} \frac{1}{K} = 1$$

The sensitivity of  $e(\infty)$  to changes in parameter  $K$

$$S_{e:K} = \frac{K}{e} \frac{\delta e}{\delta K} = \frac{K}{a/K} \frac{-a}{K^2} = -1$$

There is no reduction or increase in sensitivity

## §7. Sensitivity

## - Ex.7.12

## Sensitivity of Steady-State Error with Step Input

Find the sensitivity of the steady-state error to changes in parameter  $K$  and parameter  $a$  for the system with a step input

## Solution

The steady-state error for this Type 0 system

$$e(\infty) = \frac{1}{1 + K_p} = \frac{1}{1 + \frac{K}{ab}} = \frac{ab}{ab + K}$$

The sensitivity of  $e(\infty)$  to changes in parameter  $a$

$$S_{e:a} = \frac{a \delta e}{e \delta a} = \frac{a}{\frac{ab}{ab + K}} \frac{(ab + K)b - ab^2}{(ab + K)^2} = \frac{K}{ab + K}$$

## §7. Sensitivity

The steady-state error for this Type 0 system

$$e(\infty) = \frac{1}{1 + K_p} = \frac{1}{1 + \frac{K}{ab}} = \frac{ab}{ab + K}$$

The sensitivity of  $e(\infty)$  to changes in parameter  $a$

$$S_{e:a} = \frac{a \delta e}{e \delta a} = \frac{a}{\frac{ab}{ab + K}} \frac{(ab + K)b - ab^2}{(ab + K)^2} = \frac{K}{ab + K}$$

The sensitivity of  $e(\infty)$  to changes in parameter  $K$

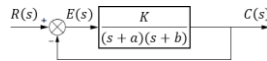
$$S_{e:K} = \frac{K \delta e}{e \delta K} = \frac{K}{\frac{ab}{ab + K}} \frac{-ab}{(ab + K)^2} = \frac{-K}{ab + K}$$

The sensitivity to changes in parameter  $K$  and parameter  $a$  is less than unity for positive  $a$  and  $b$ . Thus, feedback in this case yields reduced sensitivity to variations in both parameters

## §7. Sensitivity

## TryIt 7.4

Use MATLAB, the Symbolic Math Toolbox, and the following statements to find  $S_{e:a}$  in Ex.7.12

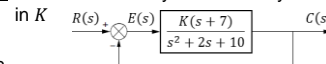


```
syms K a b s
G=K/((s+a)*(s+b));
Kp=subs(G,s,0);
e=1/(1+Kp);
Sea=(a/e)*diff(e,a);
Sea=simple(Sea);
pretty(Sea)
```

## §7. Sensitivity

## Skill-Assessment Ex.7.6

**Problem** Find the sensitivity of the steady-state error to changes



## Solution

The system is Type 0

$$K_p = \lim_{s \rightarrow 0} G(s) = 7K/10$$

The steady-state error for this Type 0 system

$$e(\infty) = \frac{1}{1 + K_p} = \frac{1}{1 + 7K/10} = \frac{10}{10 + 7K}$$

The sensitivity of  $e(\infty)$  to changes in parameter  $K$

$$S_{e:K} = \frac{K \delta e}{e \delta K} = \frac{K}{\frac{10}{10 + 7K}} \frac{(-10) \times 7}{(10 + 7K)^2} = -\frac{7K}{10 + 7K}$$

## §8. Steady-State Error for Systems in State Space

## Analysis via Final Value Theorem

- Consider the closed-loop system represented in state space

$$\dot{x} = Ax + Br, \quad y = Cx \quad (7.84)$$

- The Laplace transform of the error

$$E(s) = R(s) - Y(s) \quad (7.85)$$

$$Y(s) = R(s)T(s) \quad (7.86)$$

$T(s)$  : the closed-loop transfer function

$$\Rightarrow E(s) = R(s)[1 - T(s)] \quad (7.87)$$

- Using Eq.(3.73) for  $T(s)$

$$E(s) = R(s)[1 - C(sI - A)^{-1}B] \quad (7.88)$$

- Applying the final value theorem

$$\lim_{s \rightarrow 0} sE(s) = \lim_{s \rightarrow 0} \{sR(s)[1 - C(sI - A)^{-1}B]\} \quad (7.89)$$

$$T(s) = \frac{Y(s)}{R(s)} = C(sI - A)^{-1}B + D \quad (3.73)$$

## §8. Steady-State Error for Systems in State Space

## - Ex.7.13 Steady-State Error Using the Final Value Theorem

Evaluate the steady-state error for the system with unit step and unit ramp inputs. Use the final value theorem

$$A = \begin{bmatrix} -5 & 1 & 0 \\ 0 & -2 & 1 \\ 20 & -10 & 1 \end{bmatrix}, \quad B = \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}, \quad C = \begin{bmatrix} -1 & 1 & 0 \end{bmatrix}$$

## Solution

The steady-state error

$$e(\infty) = \lim_{s \rightarrow 0} \left[ sR(s) \left( 1 - \frac{s+4}{s^3 + 6s^2 + 13s + 20} \right) \right] \\ = \lim_{s \rightarrow 0} \left[ sR(s) \frac{s^3 + 6s^2 + 12s + 16}{s^3 + 6s^2 + 13s + 20} \right]$$

For a unit step,  $R(s) = 1/s$ , and  $e(\infty) = 4/5$ . For a unit ramp,  $R(s) = 1/s^2$ , and  $e(\infty) = \infty$ . Notice that the system behaves like a Type 0 system

## §8. Steady-State Error for Systems in State Space

## TryIt 7.5

Use MATLAB, the Symbolic Math Toolbox, and the following statements to find the steady-state error for a step input to the system of Ex.7.13

$$A = \begin{bmatrix} -5 & 1 & 0 \\ 0 & -2 & 1 \\ 20 & -10 & 1 \end{bmatrix}, B = \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}, C = \begin{bmatrix} -1 & 1 & 0 \end{bmatrix}$$

syms s

A=[-5 1 0; 0 -2 1; 20 -10 1];

B=[0;0;1];

C=[-1 1 0];

I=[1 0 0; 0 1 0; 0 0 1];

E=(1/s)\*[1-C\*(s\*I-A)^-1]\*B;

error=subs(E,s,0)

## §8. Steady-State Error for Systems in State Space

## Analysis via Input Substitution

Consider the closed-loop system represented in state space

$$\dot{x} = Ax + Br, y = Cx \quad (7.84)$$

## Step Inputs

If the input is a unit step,  $r = 1$ , a steady-state solution,  $x_{ss}$

$$x_{ss} = [V_1 \ V_2 \ \cdots \ V_n]^T = V, V_i \text{ is constant} \quad (7.92)$$

$$\dot{x}_{ss} = 0 \quad (7.93)$$

$$\text{Eq. (7.84)} \Rightarrow 0 = AV + B, y_{ss} = CV \quad (7.94)$$

$$V = -A^{-1}B \quad (7.95)$$

The steady-state error

$$e(\infty) = 1 - y_{ss} = 1 - CV = 1 + CA^{-1}B \quad (7.96)$$

## §8. Steady-State Error for Systems in State Space

## Ramp Inputs

If the input is a unit ramp,  $r = t$ , a steady-state solution,  $x_{ss}$

$$x_{ss} = [V_1 t + W_1 \ V_2 t + W_2 \ \cdots \ V_n t + W_n]^T \\ = Vt + W, W_i, V_i \text{ are constants} \quad (7.97)$$

$$\dot{x}_{ss} = [V_1 \ V_2 \ \cdots \ V_n]^T = V \quad (7.98)$$

$$\Rightarrow V = A(Vt + W) + Bt, y_{ss} = C(Vt + W) \quad (7.99)$$

In order to balance Eq. (7.99)

$$\text{equate the matrix coefficients of } t \quad AV = -B \text{ or } V = -A^{-1}B$$

$$\text{equating constant terms} \quad AW = V \text{ or } W = A^{-1}V$$

Substituting into (7.99) yields

$$y_{ss} = C[-A^{-1}Bt + A^{-1}(-A^{-1}B)] = -C[A^{-1}Bt + (A^{-1})^2 B]$$

The steady-state error

$$e(\infty) = \lim_{t \rightarrow \infty} (t - y_{ss}) = \lim_{t \rightarrow \infty} [(1 + CA^{-1}B)t + C(A^{-1})^2 B] \quad (7.103)$$

## §8. Steady-State Error for Systems in State Space

## - Ex.7.14 Steady-State Error Using Input Substitution

Evaluate the steady-state error for the system with unit step and unit ramp inputs. Use input substitution

$$A = \begin{bmatrix} -5 & 1 & 0 \\ 0 & -2 & 1 \\ 20 & -10 & 1 \end{bmatrix}, B = \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}, C = \begin{bmatrix} -1 & 1 & 0 \end{bmatrix}$$

Solution

For a unit step input, the steady-state error

$$e(\infty) = 1 + CA^{-1}B$$

$$= 1 - 0.2$$

$$= 0.8$$

For a ramp input, the steady-state error

$$e(\infty) = \lim_{t \rightarrow \infty} [(1 + CA^{-1}B)t + C(A^{-1})^2 B]$$

$$= \lim_{t \rightarrow \infty} (0.8t + 0.08)$$

$$= \infty$$

## §8. Steady-State Error for Systems in State Space

## Skill-Assessment Ex.7.7

**Problem** Find the steady-state error for a step input using both the final value theorem and input substitution methods

**WPCS**

Control Solutions

$$A = \begin{bmatrix} 0 & 1 \\ -3 & -6 \end{bmatrix}, B = \begin{bmatrix} 0 \\ 1 \end{bmatrix}, C = \begin{bmatrix} 1 & 1 \end{bmatrix}$$

Solution Using the final value theorem

$$e_{\text{step}}(\infty) = \lim_{s \rightarrow 0} sR(s) [1 - C(sI - A)^{-1}B] \\ = \lim_{s \rightarrow 0} \left( 1 - \begin{bmatrix} 1 & 1 \end{bmatrix} \begin{bmatrix} s & -1 \\ 3 & s+6 \end{bmatrix}^{-1} \begin{bmatrix} 0 \\ 1 \end{bmatrix} \right) \\ = \lim_{s \rightarrow 0} \left( 1 - \begin{bmatrix} 1 & 1 \end{bmatrix} \frac{\begin{bmatrix} s+6 & 1 \\ -3s & s \end{bmatrix}}{s^2 + 6s + 3} \begin{bmatrix} 0 \\ 1 \end{bmatrix} \right) \\ = \lim_{s \rightarrow 0} \frac{s^2 + 5s + 2}{s^2 + 6s + 3} \\ = \frac{2}{3}$$

## §8. Steady-State Error for Systems in State Space

Using input substitution

$$e_{\text{step}}(\infty) = 1 + CA^{-1}B$$

$$= 1 + \begin{bmatrix} 1 & 1 \end{bmatrix} \begin{bmatrix} 0 & 1 \\ -3 & -6 \end{bmatrix}^{-1} \begin{bmatrix} 0 \\ 1 \end{bmatrix} \\ = 1 + \begin{bmatrix} 1 & 1 \end{bmatrix} \frac{\begin{bmatrix} 3 & 0 \\ 1 & 1 \end{bmatrix}}{3} \begin{bmatrix} 0 \\ 1 \end{bmatrix} \\ = 1 + \begin{bmatrix} 1 & 1 \end{bmatrix} \begin{bmatrix} \frac{1}{3} \\ -\frac{1}{3} \end{bmatrix} \\ = \frac{2}{3}$$