

Today:

1) 4-bar position analysis

2) Matlab ←

3) Velocity analysis → power flow

Midterm posted after class

→ due Tuesday before class

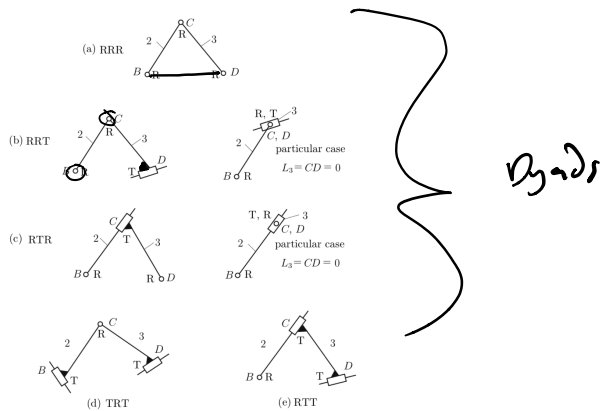
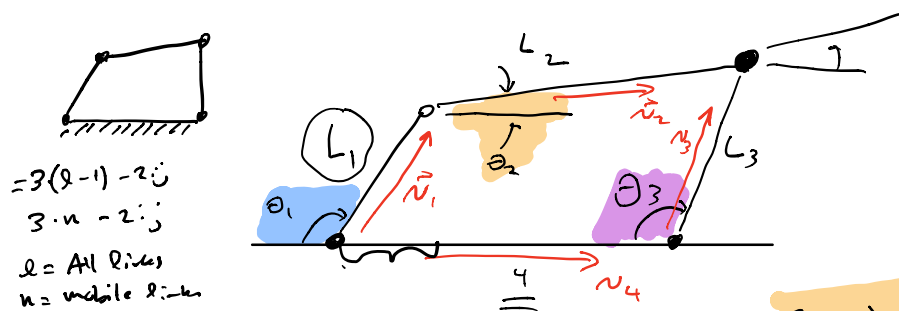


Fig. 1.9 Types of dyads: (a) RRR, (b) RRT, (c) RTR, (d) TRT, and (e) RTT

4-bar linkage



Any link that can
fully rotate \rightarrow crank

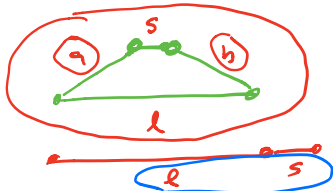
Any that oscillates \rightarrow rocker



Rocker

Shortest + Longest $> a + b$

Shortest + Longest $\leq a + b$



Closed chain $\rightarrow \underline{f(\vec{x}) = 0}$

$$\vec{v}_1 + \vec{v}_2 = \vec{v}_3 + \vec{v}_4$$

$$\rightarrow \vec{v}_1 + \vec{v}_2 - \vec{v}_3 - \vec{v}_4 = 0$$

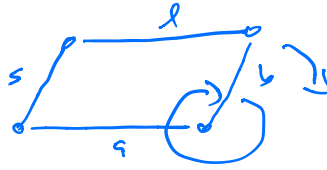
\rightarrow

$$x: L_1 \cos \theta_1 + L_2 \cos \theta_2 - L_4 - L_3 \cos \theta_3 = 0$$

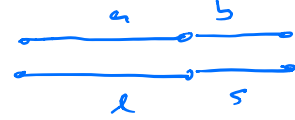
$$y: L_1 \sin \theta_1 + L_2 \sin \theta_2 - 0 - L_3 \sin \theta_3 = 0$$

$$\theta_1 + \theta_2 + \theta_3 + \theta_4 = 360$$

$$l+s \geq a+b$$

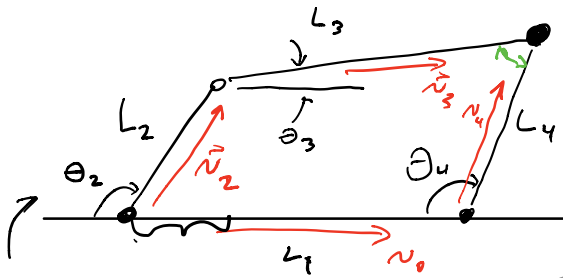


$$l+s = a+b$$



$$\theta_1 + \theta_2 + \theta_3 + \theta_4 = 2\pi$$

$$f(\vec{x}) = 0$$



$$\vec{n}_1 + \vec{n}_2 = \vec{n}_3 + \vec{n}_4$$

$$\Rightarrow \vec{n}_1 + \vec{n}_2 - \vec{n}_3 - \vec{n}_4 = 0$$

$$\begin{aligned} x: L_2 \cos \theta_2 + L_3 \cos \theta_3 - L_1 - L_4 \cos \theta_4 &= 0 \\ y: L_2 \sin \theta_2 + L_3 \sin \theta_3 - 0 - L_4 \sin \theta_4 &= 0 \end{aligned}$$

Loop closure equation

$$f(\vec{x}) = 0$$

$$\vec{n}_i = \begin{bmatrix} x_i \\ y_i \end{bmatrix}$$

$$\vec{x} = \begin{bmatrix} \theta_2 \\ \theta_3 \\ \theta_4 \end{bmatrix}$$

full configuration

$\theta_2 \rightarrow$ Driver joint

$\theta_4 \rightarrow$ output joint

want to eliminate



$$x: L_2 \cos \theta_2 + L_3 \cos \theta_3 - L_1 - L_4 \cos \theta_4 = 0$$

$$y: L_2 \sin \theta_2 + L_3 \sin \theta_3 - 0 - L_4 \sin \theta_4 = 0$$

$$\theta_2 \rightarrow \theta_4$$

$$\begin{aligned} 1) \quad L_3 \cos \theta_3 &= L_1 + L_4 \cos \theta_4 - L_2 \cos \theta_2 \\ 2) \quad L_3 \sin \theta_3 &= L_4 \sin \theta_4 - L_2 \sin \theta_2 \end{aligned}$$

square and add
1 + 2

$$(L_1 + L_4 \cos \theta_4 - L_2 \cos \theta_2)(L_1 + L_4 \cos \theta_4 - L_2 \cos \theta_2)$$

$$\begin{aligned} \underline{\underline{L_3 (\cos^2 \theta_1 + \sin^2 \theta_3)}} &= \cancel{L_4^2 \sin^2 \theta_4} + \cancel{L_2^2 \sin^2 \theta_2} - 2\cancel{L_4 L_2 \sin \theta_4 \sin \theta_2} + \\ &+ \cancel{L_1^2} + \cancel{L_1 L_4 \cos \theta_4} - \cancel{L_1 L_2 \cos \theta_2} + \cancel{L_1 L_4 \cos \theta_4} \\ &+ \cancel{L_4^2 \cos^2 \theta_4} - \cancel{L_2 \cos \theta_2 L_4 \cos \theta_4} - \cancel{L_1 L_2 \cos \theta_2} \\ &- \cancel{L_2 L_4 \cos \theta_2 \cos \theta_4} + \cancel{L_2^2 \cos^2 \theta_2} \end{aligned}$$

$$\begin{aligned} &= L_2^2 + L_4^2 + L_1^2 - 2L_1 L_2 \cos \theta_2 + 2L_1 L_4 \cos \theta_4 \\ &\quad - 2L_2 L_4 \left(\frac{\sin \theta_4 \sin \theta_2 + \cos \theta_2 \cos \theta_4}{\cos(\theta_2 - \theta_4)} \right) \end{aligned}$$

$$\underline{\underline{L_3}} = L_2^2 + L_4^2 + L_1^2 - 2L_1 (L_4 \cos \theta_4 - L_2 \cos \theta_2) - 2L_2 L_4 \cos(\theta_2 - \theta_4)$$

$$\rightarrow \underline{\underline{D_3 = \frac{L_1^2 + L_2^2 - L_3^2}{L_2 L_4}}} \quad \underline{\underline{D_2 = \frac{L_1}{L_2}}} \quad \underline{\underline{D_1 = \frac{L_1}{L_4}}}$$

Frenet-Serret equation

$$\boxed{D_3 - \cos(\theta_2 - \theta_4) + D_2 \cos(\theta_4) - D_1 \cos(\theta_2) = 0}$$

* Simple 1-equation constraint relationship
that solves the $\theta_2 \rightarrow \theta_4$

Two methods to
solve equation:

1) Symbolic tools to
explicitly solve

```
%% Start anew
clear
clc
close all

%% Define geometry
theta_2 = pi/2;
l1 = 2;
l2 = 1;
l3 = 2;
l4 = 1;

%% Set up the problem symbolically
% define variables
syms theta_3_sym theta_4_sym
assume(theta_3_sym, 'real');
assume(theta_4_sym, 'real');
assume(theta_3_sym >= 0 & theta_3_sym < 2*pi)
assume(theta_4_sym >= 0 & theta_4_sym < 2*pi)

% define constraint equations
A = l2*cos(theta_2) + l3*cos(theta_3_sym) - l4*cos(theta_4_sym) - l1;
B = l2*sin(theta_2) + l3*sin(theta_3_sym) - l4*sin(theta_4_sym);

%% Solve symbolically
sol1 = solve([A==0, B==0], [theta_3_sym, theta_4_sym]);
theta_3 = eval(sol1.theta_3_sym)
theta_4 = eval(sol1.theta_4_sym)

% two solutions
```

```
%% Let's look at an animation!!

% Original geometry
l1 = 2;
l2 = 1;
l3 = 2;
l4 = 1;

% test something else
l1 = 2;
l2 = 1;
l3 = 2;
l4 = 1.5;

for theta_2 = 0:0.1:100000
    a = mod(theta_2, 2*pi);

    % define constraint equations
    A = l2*cos(theta_2) + l3*cos(theta_3_sym) - l4*cos(theta_4_sym) - l1;
    B = l2*sin(theta_2) + l3*sin(theta_3_sym) - l4*sin(theta_4_sym);

    %% Solve symbolically
    sol1 = solve([A==0, B==0], [theta_3_sym, theta_4_sym]);
    theta_3 = eval(sol1.theta_3_sym);
    theta_4 = eval(sol1.theta_4_sym);

    %% plot
    ii=1;
    l1_vec = [l1, 0];
    l2_vec = [l2*cos(theta_2), l2*sin(theta_2)];
    l3_vec = l2_vec + [l3*cos(theta_3(ii)), l3*sin(theta_3(ii))];
    l4_vec = l3_vec + [l4*cos(theta_4(ii)), l4*sin(theta_4(ii))]; % -x because epsilon wrt interior angle

    clf;

    plot([0, l1_vec(1)], [0, l1_vec(2)], 'o-', 'linewidth', 3); % link1
    hold on;
    plot([0, l2_vec(1)], [0, l2_vec(2)], 'o-', 'linewidth', 3); % link2
    plot([l2_vec(1), l3_vec(1)], [l2_vec(2), l3_vec(2)], 'o-', 'linewidth', 3); % link3
    plot([l3_vec(1), l4_vec(1)], [l3_vec(2), l4_vec(2)], 'o-', 'linewidth', 3); % link4

    axis equal;
    axis(3.5*[-1, 1, -1, 1]);

    drawnow;
end
```

fsolve

Solve system of nonlinear equations

Syntax

```
x = fsolve(fun,x0)
x = fsolve(fun,x0,options)
x = fsolve(problem)
[x,fval] = fsolve(___)
[x,fval,exitflag,output] = fsolve(___)
[x,fval,exitflag,output,jacobian] = fsolve(___)
```

Description

Nonlinear system solver

Solves a problem specified by

$$F(x) = 0$$

for x , where $F(x)$ is a function that returns a vector value.

x is a vector or a matrix; see [Matrix Arguments](#).

$x = \text{fsolve}(\text{fun},x0)$ starts at $x0$ and tries to solve the equations $\text{fun}(x) = 0$, an array of zeros.

2) Numerical solution

fsolve \rightarrow constraint equation

Need

1) $f(\tilde{x}) = 0$

2) guess x_0

check solution

$$fval = f(\tilde{x}_{\text{solution}})$$

\downarrow
0 or small

```
%% Let's look at an animation!!

% Original
l1 = 2;
l2 = 1;
l3 = 2;
l4 = 1.2;

% l1 = 1;
% l2 = 2;
% l3 = .5;
% l4 = 1;

x_guess = [0, 0];

input_angle = 0;
output_angle = 0;
cnt = 1;

omega = 1;

for t = 0:0.1:100000

    theta_2 = mod(omega*t, 2*pi);

    %% Solve numerically

    [x,fval] = fsolve(@(x) four_bar_constraint(x, theta_2, l1, l2, l3, l4), [0, output_angle(end)]);

    theta_3 = x(1);
    theta_4 = x(2);

    input_angle(cnt) = theta_2;
    output_angle(cnt) = theta_4;
    cnt = cnt + 1;

    % Only plot solutions that satisfy a certain final tolerance
    if norm(fval) < 0.000001
        %% plot

        l1_vec = [l1, 0];
        l2_vec = [l2*cos(theta_2), l2*sin(theta_2)];
        l3_vec = l2_vec + [l3*cos(theta_3), l3*sin(theta_3)];
        l4_vec = l1_vec + [l4*cos(theta_4), l4*sin(theta_4)]; % -x because epsilon wrt interior angle

        clf;

        plot([0, l1_vec(1)], [0, l1_vec(2)], 'o-', 'linewidth', 3); % link1
        hold on;
        plot([0, l2_vec(1)], [0, l2_vec(2)], 'o-', 'linewidth', 3); % link2
        plot([l2_vec(1), l3_vec(1)], [l2_vec(2), l3_vec(2)], 'o-', 'linewidth', 3); % link3
        plot([l1_vec(1), l4_vec(1)], [l1_vec(2), l4_vec(2)], 'o-', 'linewidth', 3); % link4

        axis equal;
        axis(3.5*[-1, 1, -1, 1]);

        drawnow;

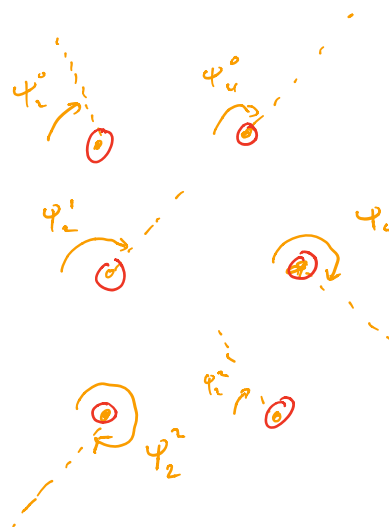
    end

end

% pause;
end
```

$$D_3 - \cos(\theta_2 - \theta_4) + D_2 \cos(\theta_4) - D_1 \cos(\theta_2) = 0$$

↗ ↖
Motion synthesis with F equation



$$\rightarrow \begin{cases} D_3 - \cos(\psi_2^0 - \psi_4^0) + D_2 \cos(\psi_4^0) - D_1 \cos \psi_2^0 = 0 \\ D_3 - \cos(\psi_2^1 - \psi_4^1) + D_2 \cos(\psi_4^1) - D_1 \cos \psi_2^1 = 0 \\ D_3 - \cos(\psi_2^2 - \psi_4^2) + D_2 \cos(\psi_4^2) - D_1 \cos \psi_2^2 = 0 \end{cases}$$

↗ ↖
 D_3, D_2, D_1

$$\underline{\underline{f(\vec{x}) = 0}}$$

$$\vec{x} = \begin{bmatrix} D_3 \\ D_2 \\ D_1 \end{bmatrix}$$