

## 1

First we find  $K_I$  without any consideration for plasticity. Since we have an edge-crack in a finite-width panel, we use (2.4a) and substitute the provided values.

```
In [12]: import numpy as np
def beta(a,w):
    return 1.122 - 0.231*a/w + 10.55*(a/w)**2 - 21.71*(a/w)**3 + 30.82*(a/w)**4
def KI(a,w,s):
    return s*np.sqrt(np.pi*a)*beta(a,w)
a = 1.5
w = 6.
t = .25
s = 15. #ksi
sy = 65. #ksi
print KI(a,w,s)
```

48.99928079724123

We find  $K_I = 49.0 \text{ ksi}\sqrt{\text{in.}}$

For plane stress, we use (6.6) with  $I=2$ , while for plane strain we set  $I=6$ .

```
In [13]: #plane stress, I=2
I=2
rp = 1.0/(I*np.pi)*(KI(a,w,s)/sy)**2
print rp
```

0.09044255642790483

```
In [14]: #calculate aeff, KI(aeff) until solution converges
KI_old = KI(a,w,s)
aeff = a + rp
KI_new = KI(aeff,w,s)
#Loop through until the percent error is less than 1%
while ((KI_old-KI_new)/(KI_old))**2 > 0.00001:
    rp = 1.0/(I*np.pi)*(KI(aeff,w,s)/sy)**2
    aeff = a + rp
    KI_old = KI_new
    KI_new = KI(aeff,w,s)
```

```
In [15]: print rp
         print KI_new

0.10321764158945292
52.396000450790474
```

So for plane stress we have:  $K_I = 52.4 \text{ ksi}\sqrt{\text{in}}$

In plane strain we follow the same procedure, with  $I=6$

```
In [16]: #plane strain, I=6
         I=6
         rp = 1.0/(I*np.pi)*(KI(a,w,s)/sy)**2
         #calculate aeff, KI(aeff) until solution converges
         KI_old = KI(a,w,s)
         aeff = a + rp
         KI_new = KI(aeff,w,s)
         while ((KI_old-KI_new)/(KI_old))**2 > 0.00001:
             rp = 1.0/(I*np.pi)*(KI(aeff,w,s)/sy)**2
             aeff = a + rp
             KI_old = KI_new
             KI_new = KI(aeff,w,s)
         print KI_old
         print KI_new

49.97326249027704
50.01267113809021
```

And in plane strain we have  $K_I = 50.0 \text{ ksi}\sqrt{\text{in}}$

For  $t = 0.25$ , we can calculate  $I$  directly using (6.7)

```
In [17]: t=0.25
         I = 6.7 - 1.5/t*(KI(a,w,s)/sy)**2
         print I

3.290395949850567
```

We now proceed with the same solution method for  $I = 3.29$

```

In [18]: rp = 1.0/(I*np.pi)*(KI(a,w,s)/sy)**2
          #calculate aeff, KI(aeff) until solution converges
          KI_old = KI(a,w,s)
          aeff = a + rp
          KI_new = KI(aeff,w,s)
          while ((KI_old-KI_new)/(KI_old))**2 > 0.00001:
              I = 6.7 - 1.5/t*(KI_new/sy)**2
              rp = 1.0/(I*np.pi)*(KI(aeff,w,s)/sy)**2
              aeff = a + rp
              KI_old = KI_new
              KI_new = KI(aeff,w,s)
          print KI_old
          print KI_new

51.083846403989384
51.139566264483655

```

```
In [19]: rp
```

```
Out[19]: 0.06566316998590092
```

As expected, we find  $K_I$  somewhere between the plane strain and plane stress solutions,  $K_I = 51.1 \text{ ksi}\sqrt{\text{in}}$

## 2

For an infinitely wide, center-cracked panel we use (2.1)

$$K_I = \sigma\sqrt{\pi a}$$

In plane strain, the plastic stress intensity factor,  $K_{Ie}$  is given by

$$K_{Ie} = \sigma\sqrt{\pi(a + r_p)}$$

where (in plane strain)

$$r_p = \frac{1}{6\pi} \left( \frac{K_{Ie}}{\sigma_{YS}} \right)^2$$

Substituting  $r_p$  into  $K_{Ie}$  gives

$$K_{Ie} = \sigma\sqrt{\pi \left( a + \frac{1}{6\pi} \left( \frac{K_{Ie}}{\sigma_{YS}} \right)^2 \right)}$$

We square both sides to find

$$K_{Ie}^2 = \sigma^2 \pi \left( a + \frac{1}{6\pi} \left( \frac{K_{Ie}}{\sigma_{YS}} \right)^2 \right)$$

Multiplying out we get

$$K_{Ie}^2 = \sigma^2 \pi a + \frac{\sigma^2}{6} \left( \frac{K_{Ie}}{\sigma_{YS}} \right)^2$$

We can subtract the second term from both sides

$$K_{Ie}^2 - \frac{\sigma^2}{6} \left( \frac{K_{Ie}}{\sigma_{YS}} \right)^2 = \sigma^2 \pi a$$

And simplify

$$K_{Ie}^2 \left( 1 - \frac{\sigma^2}{6\sigma_{YS}^2} \right) = \sigma^2 \pi a$$

We can now divide both sides by  $\left( 1 - \frac{\sigma^2}{6\sigma_{YS}^2} \right)$  to find

$$K_{Ie}^2 = \frac{\sigma^2 \pi a}{1 - \frac{\sigma^2}{6\sigma_{YS}^2}}$$

We take the square root of both sides

$$K_{Ie} = \frac{\sigma \sqrt{\pi a}}{\sqrt{1 - \frac{\sigma^2}{6\sigma_{YS}^2}}}$$

We can now replace  $\sigma \sqrt{\pi a}$  with  $K_I$

$$K_{Ie} = \frac{K_I}{\sqrt{1 - \frac{\sigma^2}{6\sigma_{YS}^2}}}$$

And divide both sides by  $K_I$

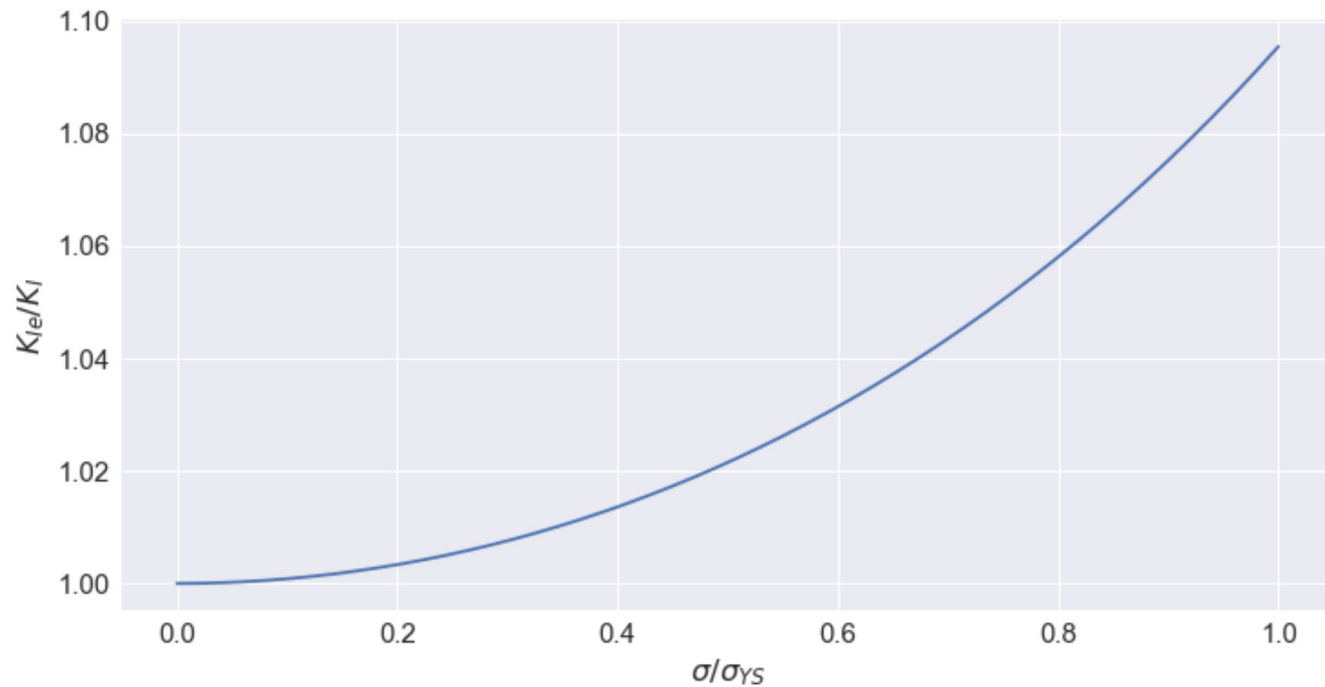
$$\frac{K_{Ie}}{K_I} = \frac{1}{\sqrt{1 - \frac{\sigma^2}{6\sigma_{YS}^2}}}$$

Now we are ready to generate our plot. Fracture mechanics is only valid when  $\sigma < \sigma_{YS}$ , so we consider  $0 < \sigma < \sigma_{YS}$  for our plot.

```
In [20]: s_sys = np.linspace(0,1)
K_Ie_KI = 1./(1.-s_sys**2/6.)**.5

import matplotlib.pyplot as plt
import seaborn as sb
sb.set(font_scale=1.5)
%matplotlib inline
plt.figure(figsize=(12,6))
plt.plot(s_sys,K_Ie_KI)
plt.xlabel(r'$\sigma / \sigma_{YS}$')
plt.ylabel(r'$K_{Ie} / K_I$')
```

Out[20]: Text(0,0.5,'\$K\_{Ie} / K\_I\$')



### 3

In this problem we are asked to find the ratio,  $K_{Ie}/K_I$  for some specific conditions on a finite-width, center-cracked panel.

In this case we use (2.2a) for  $K_I$  and we use (6.6) to find  $r_p$ , with  $I = 2$  for plane stress and  $I = 6$  for plane strain.

```

In [36]: #4.12
def K_I(a,w,s):
    return s*np.sqrt(np.pi*a)*np.sqrt(1/np.cos(np.pi*a/w))

a = 1.
w = 7.
s = 45.
sy = 75.

#plane strain
KIa = K_I(a,w,s)
I = 6.
rp = 1.0/(I*np.pi)*(K_I(a,w,s)/sy)**2
#calculate aeff, KI(aeff) until solution converges
KI_old = K_I(a,w,s)
aeff = a + rp
KI_new = K_I(aeff,w,s)
while ((KI_old-KI_new)/(KI_old))**2 > 0.00001:
    rp = 1.0/(I*np.pi)*(K_I(aeff,w,s)/sy)**2
    aeff = a + rp
    KI_old = KI_new
    KI_new = K_I(aeff,w,s)
print KI_old
print KI_new

87.71532300053244
87.7391424415274

```

```

In [37]: print KI_new/KIa

1.0441450344891823

```

For plane strain we have  $K_{Ie}/K_I = 1.04$

```

In [38]: #plane stress
w=7.
KIb = K_I(a,w,s)
I = 2.
rp = 1.0/(I*np.pi)*(K_I(a,w,s)/sy)**2
#calculate aeff, KI(aeff) until solution converges
KI_old = K_I(a,w,s)
aeff = a + rp
KI_new = K_I(aeff,w,s)
while ((KI_old-KI_new)/(KI_old))**2 > 0.00001:
    rp = 1.0/(I*np.pi)*(K_I(aeff,w,s)/sy)**2
    aeff = a + rp
    KI_old = KI_new
    KI_new = K_I(aeff,w,s)
print KI_old
print KI_new

97.76029395762107
97.9830521282994

```

```

In [39]: print KI_new/KIb

1.1660533086705365

```

For plane stress with  $W = 7$  we have  $K_{Ie}/K_I = 1.17$

If the thickness of the panel was undecided, we can also plot the plasticity effect for varying thickness

```
In [40]: t = np.linspace(1./4,1.)
a = 1.
W = 7.
def calcI(t,K,sy):
    I = 6.7 - 1.5/t*(K/sy)**2
    for i in range(len(I)):
        if I[i] < 2.:
            I[i] = 2.
        elif I[i] > 6.:
            I[i] = 6.
    return I
I = calcI(t,KI(a,W,s),sy)
rp = 1.0/(I*np.pi)*(KI(a,w,s)/sy)**2
#calculate aeff, KI(aeff) until solution converges
KI_old = [K_I(a,w,s),0]
aeff = a + rp
KI_new = K_I(aeff,w,s)
while ((max(KI_old)-max(KI_new))/(max(KI_old)))**2 > 0.00000000001:
    I = calcI(t,KI_new,sy)
    rp = 1.0/(I*np.pi)*(K_I(aeff,w,s)/sy)**2
    aeff = a + rp
    KI_old = KI_new
    KI_new = K_I(aeff,w,s)
print max(KI_old)
print max(KI_new)
```

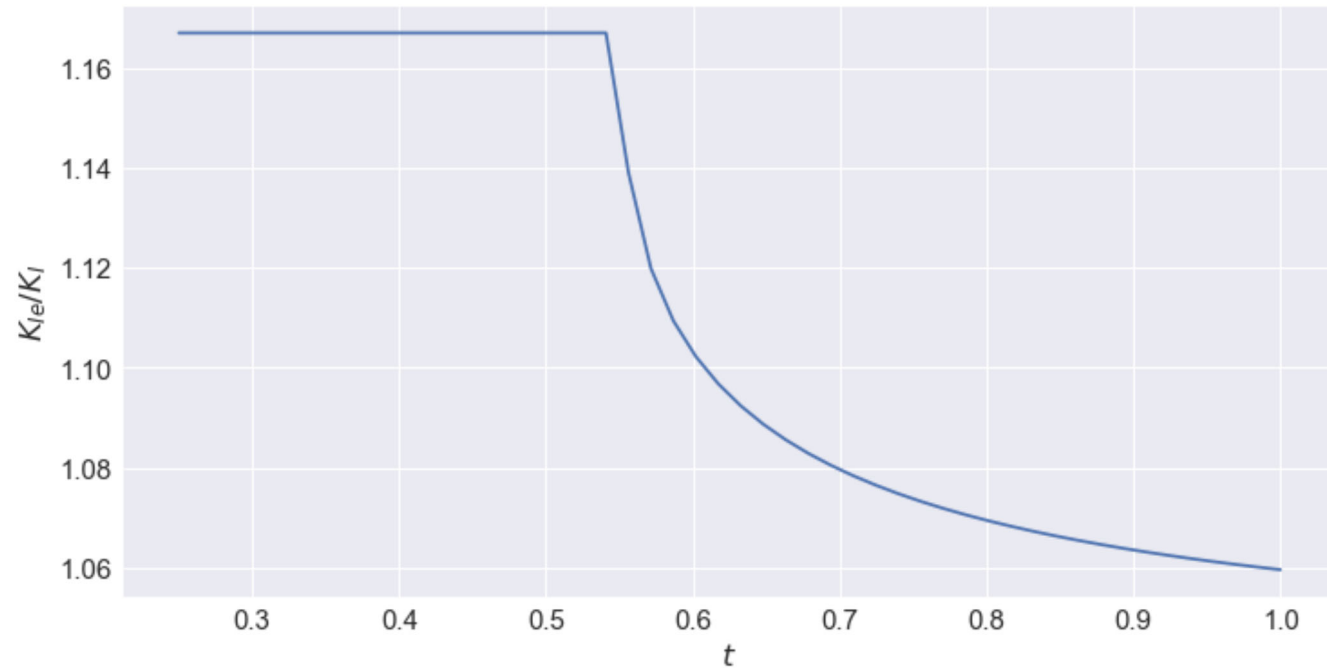
98.0752187044578

98.07496495686982



```
In [41]: plt.figure(figsize=(12,6))
plt.plot(t,KI_new/K_I(a,w,s))
plt.xlabel(r'$t$')
plt.ylabel(r'$K_{Ie} / K_{I}$')
```

```
Out[41]: Text(0,0.5, '$K_{Ie} / K_{I}$')
```



Here we see that the thicker the panel is, the lower the effect of plasticity. Panels less than 0.55" thick in this configuration are essentially in a state of plane stress.

## 4

First we calculate  $K_I$  for the given plate using (2.4a)

```
In [27]: def beta(a,w):
          return 1.122 - 0.231*a/w + 10.55*(a/w)**2 - 21.71*(a/w)**3 + 30.82*(a/w)**4
          def KI(a,w,s):
              return s*np.sqrt(np.pi*a)*beta(a,w)
          a = 2.
          w = 6.
          s = 10. #ksi
          sy = 75. #ksi
          v = 0.3
          print KI(a,w,s)

44.95993689873641
```

For Von Mises yield theory in plane stress we have

$$r_p = \frac{K_I^2}{2\pi\sigma_{YS}^2} \cos^2 \frac{\theta}{2} \left( 1 + 3 \sin^2 \frac{\theta}{2} \right)$$

```
In [28]: th = np.linspace(0,2*np.pi,200)
          rp_vm_stress = KI(a,w,s)**2/(2*np.pi*sy**2)*np.cos(th/2)**2*(1+3*np.sin(th/2)**2)
```

For Von Mises yield theory in plane strain we have

$$r_p = \frac{K_I^2}{2\pi\sigma_{YS}^2} \cos^2 \frac{\theta}{2} \left( 1 - 4\nu + 4\nu^2 + 3 \sin^2 \frac{\theta}{2} \right)$$

```
In [29]: rp_vm_strain = KI(a,w,s)**2/(2*np.pi*sy**2)*np.cos(th/2)**2*(1-4*v+4*v**2+3*np.sin(th/2)**2)
```

For Tresca yield in plane stress we have

```
In [30]: rp_tr_stress = KI(a,w,s)**2/(2*np.pi*sy**2)*np.cos(th/2)**2*(1+np.sin(th/2))**2
```

For Tresca yield in plane strain we must first find  $\theta_t$

```
In [31]: th1 = 2*np.arcsin(1-2*v)
```

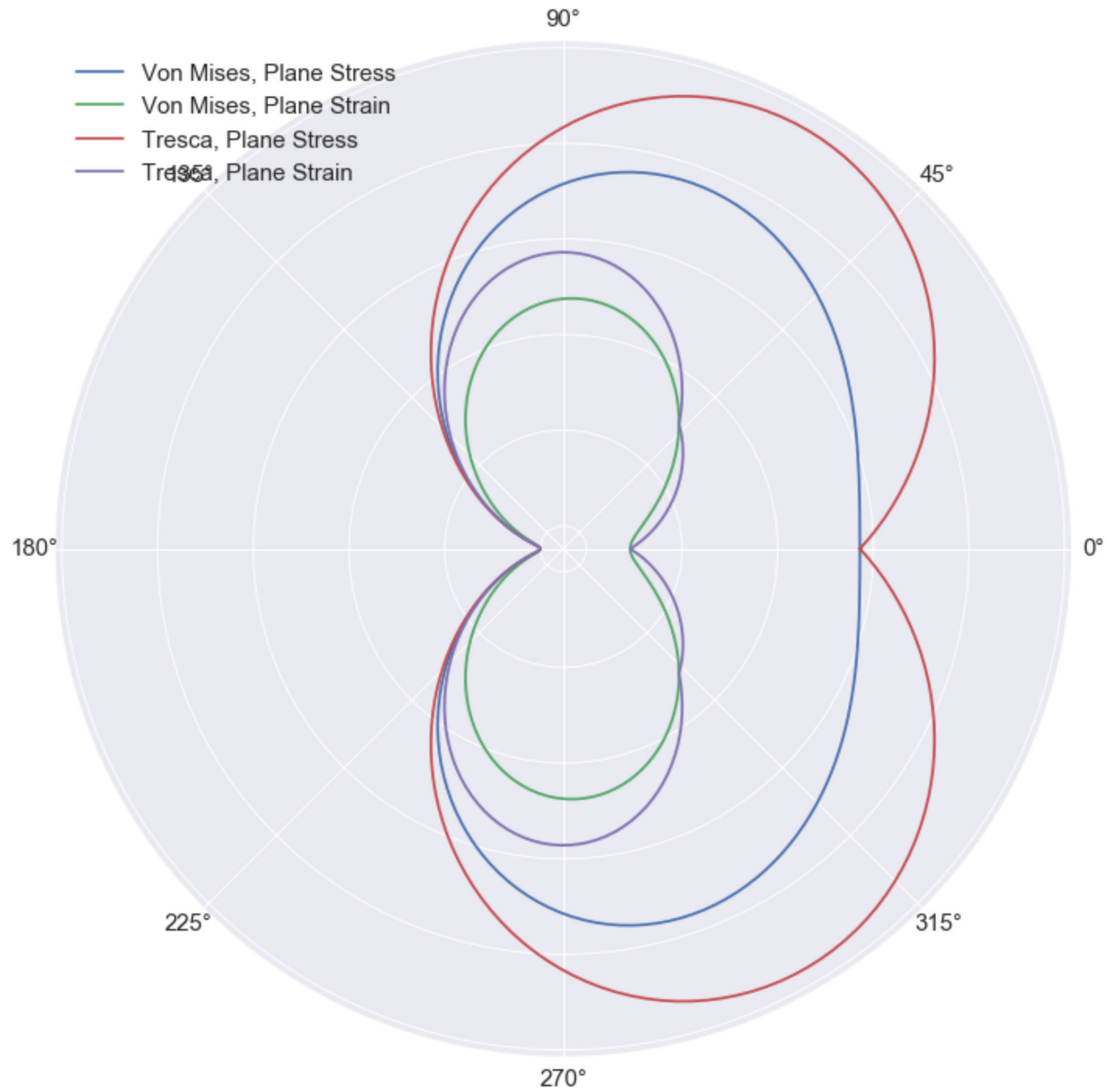
We then use the appropriate formulas, depending on whether  $\theta_t < \theta < 2\pi - \theta_t$

```
In [32]: rp_tr_strain = np.zeros(len(th)) #initiate array of zeros
for i in range(len(th)):
    if th[i] > th1 and th[i] < 2*np.pi - th1:
        rp_tr_strain[i] = 2*KI(a,w,s)**2/(np.pi*sy**2)*np.cos(th[i]/2)**2*np.sin(th[i]/2)**2
    else:
        rp_tr_strain[i] = KI(a,w,s)**2/(2*np.pi*sy**2)*np.cos(th[i]/2)**2*(1-2*v+np.sin(th[i]/2))**2
```

Now we make a polar plot to compare this plastic zone shapes

```
In [33]: fig = plt.figure(figsize=(12,12))
ax = plt.subplot(111,projection='polar')
ax.set_yticklabels([])
ax.plot(th,rp_vm_stress,label='Von Mises, Plane Stress')
ax.plot(th,rp_vm_strain,label='Von Mises, Plane Strain')
ax.plot(th,rp_tr_stress,label='Tresca, Plane Stress')
ax.plot(th,rp_tr_strain,label='Tresca, Plane Strain')
ax.legend(loc='best')
```

Out[33]: <matplotlib.legend.Legend at 0xb47d4e0>



In [ ]:

In [ ]: