

# **AE 737: Mechanics of Damage Tolerance**

Lecture 16 - Strain based fatigue

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# schedule

- 17 Mar - Strain-based fatigue
- 19 Mar - Crack growth, HW6 Due
- 23-27 Mar - Spring Break
- 31 Mar - Crack growth
- 2 Apr - Crack growth, HW7 Due

# outline

- strain based fatigue
- variable amplitude strains
- mean stress effects
- general trends
- notches
- multiaxial loading
- other factors affecting fatigue
- crack growth rate
- crack growth rate equations

# strain based fatigue

# strain based fatigue

- The strain based fatigue method uses local stresses and strains (instead of global, nominal values)
- The strain-based method gives greater detail, and validity at lower cycles
- It is still valid for high cycle fatigue (but gives same result as stress-based fatigue)
- Does not include crack growth analysis or fracture mechanics

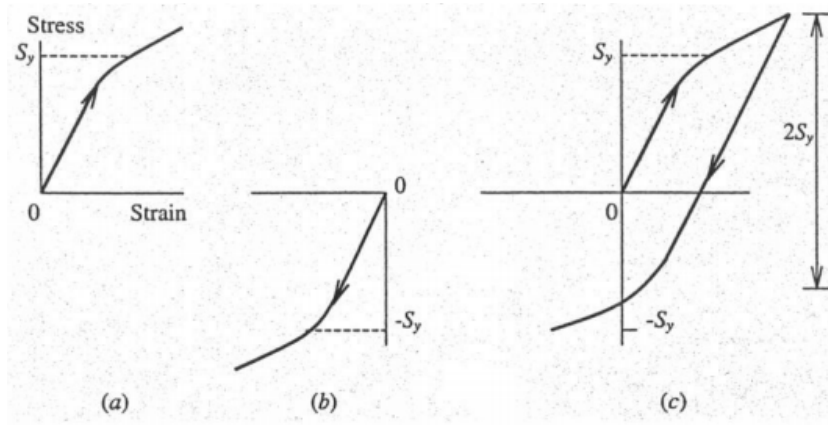
# strain life curve

- Similar to the S-N curves in stress-based fatigue analysis, we can plot the cyclic strain amplitude vs. number of cycles to failure
- This is most commonly done using axial test machines (instead of rotating bending tests)
- The test is run in strain control (not load control)
- Generally plotted on log-log scale

# plastic and elastic strain

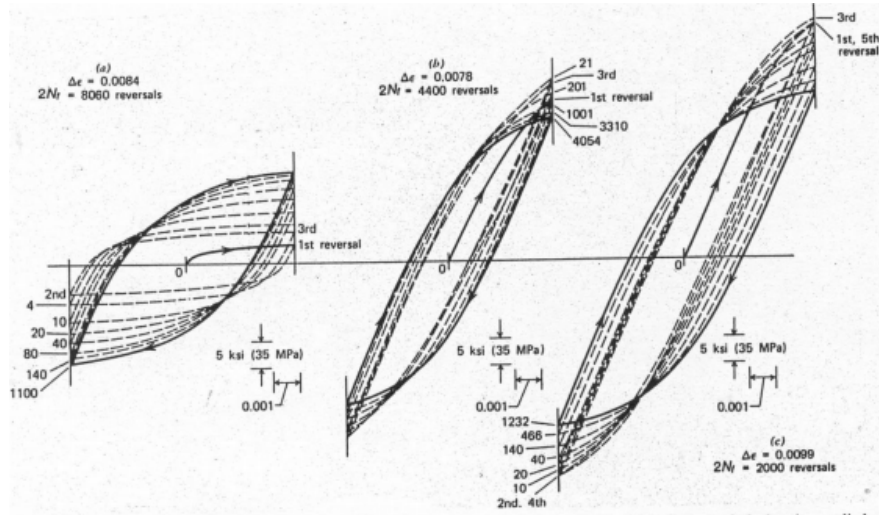
- We can separate the total strain into elastic and plastic components  $\epsilon_a = \epsilon_{ea} + \epsilon_{pa}$

# plastic strain





# hysteresis loops



# cyclic stress strain curve

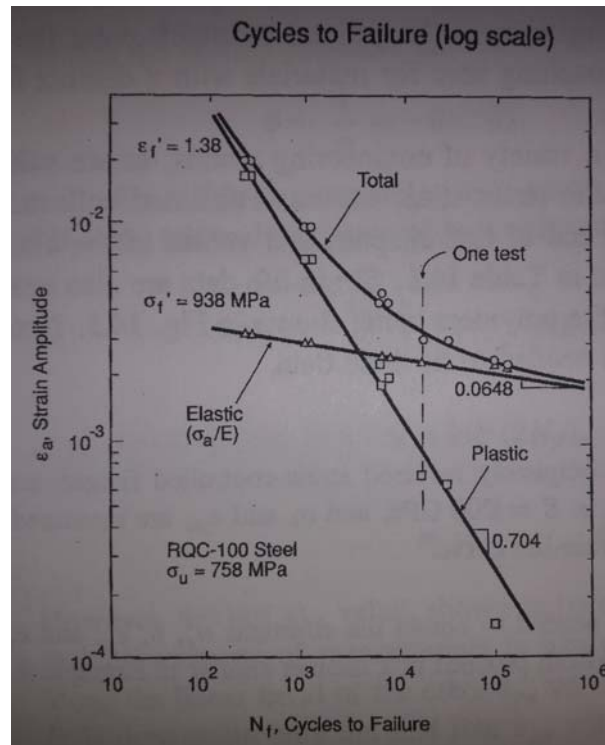
- While strain-life data will generally just report  $\epsilon_a$  and  $\epsilon_{pa}$ , some will also tabulate a form for the cyclic stress-strain curve

$$\epsilon_a = \frac{\sigma_a}{E} + \left( \frac{\sigma_a}{H'} \right)^{\frac{1}{n'}}$$

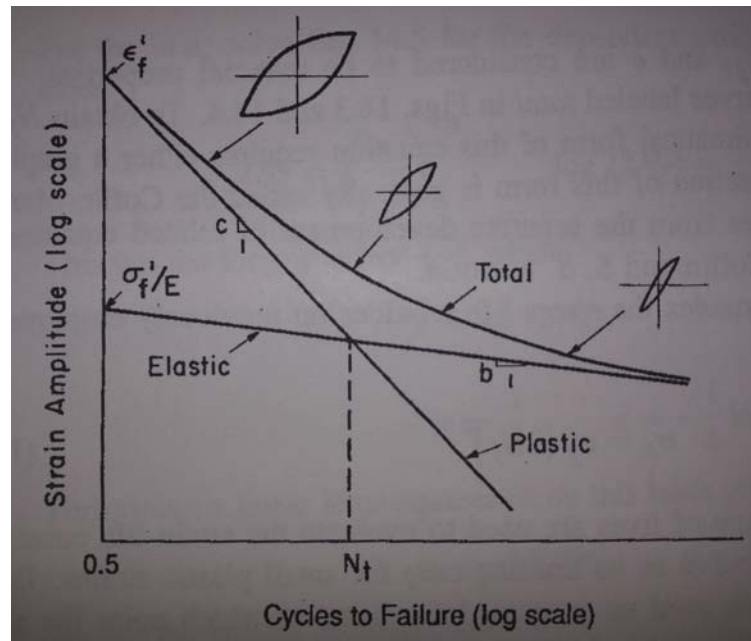
# plastic and elastic strain

- On strain life curves, the strain is often plotted three times per each experiment
- Once for total strain, once for plastic strain, and once for elastic strain
- Since plastic strain and elastic strain vary by the number of cycles, a hysteresis loop from half the fatigue life is generally used
- This is considered representative of stable behavior

# experimental data



# trends



# lines

- We notice that the data for elastic and plastic strains are represented by straight lines, in the log-log scale
- If we recall the form used for a straight line in log-log plots for S-N curves:  $\sigma_a = \sigma_f'(2N_f)^b$
- We can convert this to find the elastic component of strain

$$\epsilon_{ea} = \frac{\sigma_f'}{E}(2N_f)^b$$

# lines

- We can use the same form with new constants for the plastic component of strain  
 $\epsilon_{pa} = \epsilon_f'(2N_f)^c$
- We can combine the elastic and plastic portions to find the total strain-life curve

$$\epsilon_a = \frac{\sigma_f'}{E}(2N_f)^b + \epsilon_f'(2N_f)^c$$

# example

$\epsilon_a$	$\sigma_a$ (MPa)	$\epsilon_{pa}$	$N_f$
0.0202	631	0.01695	227
0.0100	574	0.00705	1030
0.0045	505	0.00193	6450
0.0030	472	0.00064	22250
0.0023	455	(0.00010)	110000



# transition life

- With the strain-based fatigue method we are better equipped to discuss the difference between high and low-cycle fatigue
- Low-cycle fatigue is dominated by plastic effects, while high-cycle fatigue has little plasticity
- We can find the intersection of the plastic strain and elastic strain lines
- This point is  $N_t$ , the transition fatigue life

$$N_t = \frac{1}{2} \left( \frac{\sigma_f'}{\epsilon_f'} \right)^{\frac{1}{c-b}}$$

# inconsistencies in constants

- If we consider the equation for the cyclic stress strain curve

$$\epsilon_a = \frac{\sigma_a}{E} + \left( \frac{\sigma_a}{H'} \right)^{\frac{1}{n'}}$$

- We can consider the plastic portion and solve for  $\sigma_a$   $\sigma_a = H' \epsilon_{pa}^{n'}$

# inconsistencies in constants

- We can eliminate  $2N_f$  from the plastic strain equation  $\epsilon_{pa} = \epsilon_f'(2N_f)^c$
- By solving the stress-life relationship for  $2N_f \sigma_a = \sigma_f'(2N_f)^b$  and substituting that into the plastic strain

# inconsistencies in constants

- We then compare with stress-life equations and find

$$H' = \frac{\sigma_f'}{(\epsilon_f')^{b/c}}$$

$$n' = \frac{b}{c}$$

# inconsistencies in constants

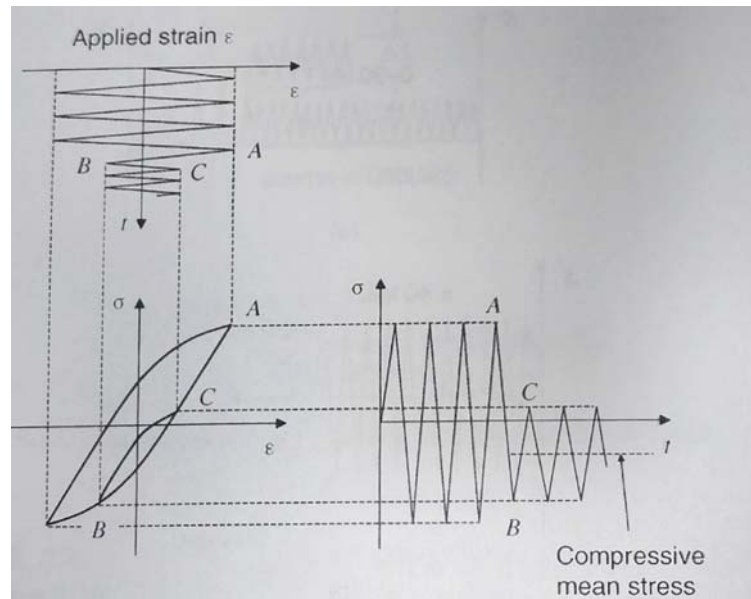
- However, in practice these constants are fit from different curves
- In some cases there can be large inconsistencies in these values
- One cause for this is data that do not lie on a straight line in the log-log domain
- For ductile materials at short lives, the true stresses and strains may differ significantly from engineering stress and strain

# **variable amplitude strains**

# variable amplitude strains

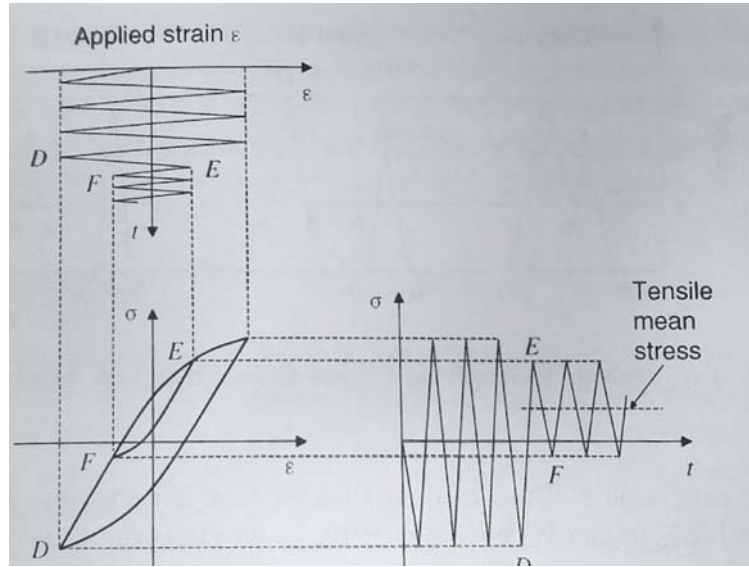
- As with stresses, we can apply variable amplitude strains
- However, when the change is made will affect whether there is a tensile or compressive mean stress

# compressive mean





# tensile mean



# mean stress effects

# mean stress in strain-based fatigue

- In regions where plastic strain is significant, some applied mean stress is likely to be relaxed through cyclic plastic strain
- When the plastic strain is not significant, mean stress will exist
- Mean strain does not generally affect fatigue life

# morrow approach

- Recall the Morrow approach for mean stress effects from the stress-based method

$$\frac{\sigma_a}{\sigma_{ar}} + \frac{\sigma_m}{\sigma'_f} = 1$$

- We can rearrange the equation such that

$$\sigma_a = \sigma'_f \left[ \left( 1 - \frac{\sigma_m}{\sigma'_f} \right)^{\frac{1}{b}} (2N_f) \right]^b$$

# morrow approach

- If we compare to the stress-life equation ( $\sigma_a = \sigma_f'(2N_f)^b$ ), we see that we can replace  $N_f$  with

$$N^* = N_f \left( 1 - \frac{\sigma_m}{\sigma_f'} \right)^{\frac{1}{b}}$$

- We can now substitute  $N^*$  for  $N_f$  in the strain-life equation to find

$$\epsilon_a = \frac{\sigma_f'}{E} \left( 1 - \frac{\sigma_m}{\sigma_f'} \right) (2N_f)^b + \epsilon_f' \left( 1 - \frac{\sigma_m}{\sigma_f'} \right)^{\frac{c}{b}} (2N_f)^c$$

# morrow approach

- Graphically, we can use the Morrow approach very easily using only the zero-mean stress graph
- From the zero-mean stress graph, find the point corresponding to your applied strain
- For a non zero mean stress, this point represents  $(\epsilon_a, N^*)$ , we can now solve for  $N_f$  using the equation for  $N^*$

# modified morrow

- While the Morrow equation agrees very well with many data, some are better fit with a modification
- In the modified version, it is assumed that the mean stress has no effect on the plastic term

$$\epsilon_a = \frac{\sigma_f'}{E} \left( 1 - \frac{\sigma_f}{\sigma_f'} \right) (2N_f)^b + \epsilon_f' (2N_f)^c$$

- There is no convenient solution method for this form, and it generally must be solved numerically, or plotted with many families of  $\sigma_m$

# smith watson topper

- The Smith, Watson, and Topper approach assumes that the life for any given state is dependent on the product  $\sigma_{max} \epsilon_a$
- After some manipulation, this gives

$$\sigma_{max} \epsilon_a = \frac{(\sigma'_f)^2}{E} (2N_f)^{2b} + \sigma'_f \epsilon'_f (2N_f)^{b+c}$$

- This method can also be solved graphically if a plot of  $\sigma_{max} \epsilon_a$  is made using zero-mean data. All we need to do is find the new  $\sigma_{max} \epsilon_a$  point to find a new  $N_f$



# comparison

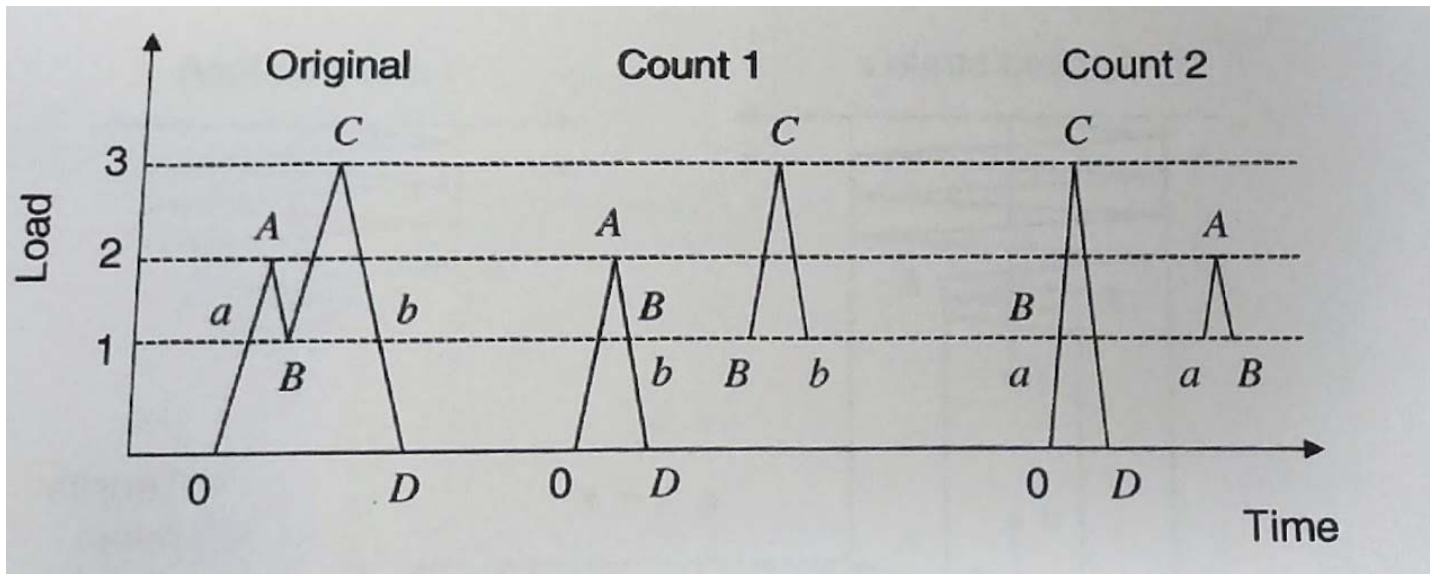
- All three methods discussed are in general use
- The Morrow method is very good for steel
- The modified Morrow method gives improved results in many materials
- The SWT approach is very good for general use, but is non-conservative with a compressive mean stress

**example p. 285**

# cycle counting

- In all fatigue methods (stress, strain, and crack propagation) the way we count load cycles can have an effect on our results
- To avoid being non-conservative, we need to always count the largest amplitudes first
- We will discuss some specific cycle-counting algorithms during crack propagation

# cycle counting



# general trends

# true fracture strength

- We can consider a tensile test as a fatigue test with  $N_f = 0.5$
- We would then expect the true fracture strength  $\tilde{\sigma}_f \approx \sigma_f'$
- And similarly for strain  $\tilde{\epsilon}_f \approx \epsilon_f'$

# ductile materials

- Since ductile materials experience large strains before failure, we expect relatively large  $\epsilon_f'$  and relatively small  $\sigma_f'$
- This will cause a less steep slope in the plastic strain line
- In turn this intersects with the elastic strain line much later, resulting a longer transition life for ductile materials

# brittle materials

- Brittle materials exhibit the opposite effect, with relatively low  $\epsilon_f'$  and relatively high  $\sigma_f'$
- This results in a steeper plastic strain line
- And shorter transition life



# tough materials

- Tough materials have intermediate values for both  $\epsilon_f'$  and  $\sigma_f'$
- This gives a transition life somewhere between brittle and ductile materials
- It is also noteworthy that strain-life for many metals pass through the point  $\epsilon_a = 0.01$  and  $N_f = 1000$  cycles
- Steels also follow a trend with Brinell Hardness, the higher they are on the HB scale, the lower their transition life

# typical property ranges

- Most common engineering materials have  $-0.8 < c < -0.5$ , with most values being very close to  $c = -0.6$
- The elastic strain slope generally has  $b = -0.085$
- A “steep” elastic slope is around  $b = -0.12$ , common in soft metals
- While “shallow” slopes are around  $b = -0.05$ , common for hardened metals

# notches

# fatigue notch factor

- We previously found expressions for stress-based fatigue analysis when notches are present
- Due to yielding, the notch sensitivity is not the same for stress and strain controlled fatigue analysis
- One simple approach to find the strain fatigue notch factor is to use

$$K_t = \sqrt{K_f^\sigma K_f^\epsilon}$$

# multiaxial loading

# multiaxial loading

- Multi-axial loading in strain-based fatigue analysis is still an active field of research
- We are currently only capable of handling proportional loads that are in-phase (i.e. have the same frequency)

# multiaxial loading

- If we consider the principal directions where  $\sigma_{2a} = \lambda\sigma_{1a}$ , we find an expression for the strain-life as

$$\epsilon_{1a} = \frac{\frac{\sigma'_f}{E}(1 - \nu\lambda)(2N_f)^b + \epsilon'_f(1 - 0.5\lambda)(2N_f)^c}{\sqrt{1 - \lambda + \lambda^2}}$$

# stress triaxiality factor

- Another approach is to consider the stress triaxiality factor

$$T = \frac{1 + \lambda}{\sqrt{1 - \lambda + \lambda^2}}$$

- Three notable cases of this are
  1. Pure planar shear:  $\lambda = -1, T = 0$
  2. Uniaxial stress:  $\lambda = 0, T = 1$
  3. Equal biaxial stress:  $\lambda = 1, T = 2$



# stress triaxiality factor

- Marloff suggests the following inclusion of stress triaxiality

$$\epsilon_a = \frac{\sigma_f'}{E}(2N_f)^b + 2^{1-T}\epsilon_f'(2N_f)^c$$

# **other factors affecting fatigue**

# factors affecting fatigue life

- At temperatures above one-half the melting temperature (absolute scale), creep-relaxation is significant
- This will cause the strain/stress-life curves to become rate dependent
- Occurs at room temperature for many materials (lead, tin, many polymers)
- At a sufficiently elevated temperature for any material

# surface finish

- High cycle fatigue is sensitive to surface finish, samples are generally polished
- Low cycle fatigue is not sensitive to surface finish or residual stress
- The plastic deformation tends to remove residual stresses
- In high-cycle fatigue, crack initiation is important (poor surface finish allows cracks to form earlier)
- When plastic deformation is present (low-cycle fatigue), cracks form relatively quickly regardless of surface finish

# surface finish

- Since low-cycle fatigue has little effect from surface finish, we could modify the strain life curve by altering only the elastic portion
- If we define the surface effect factor,  $m_s$ , we can find a new  $b_s$  to replace  $b$  in the strain-life equation

$$b_s = \frac{\log(m_s(2N_e)^b)}{\log(2N_e)}$$

# surface treatments

- Treatments which decrease fatigue life:
  - Electro-plating (chrome, +corrosion resistance, -fatigue life)
  - Grinding improves surface finish, but introduces surface tension, and heat generated can temper quench
  - Stamping introduces discontinuities and irregularities
  - Forging can refine grain structure and improve physical properties, but can cause decarburization in steels, which hurts fatigue life
  - Hot rolling can also cause decarburization

# surface treatments

- Some treatments improve fatigue life:
  - Cold rolling improves surface finish, introduces residual compressive stress on surface (slows crack initiation on surface)
  - Shot peening introduces many small divots on surface, which can be detrimental in corrosion, but it does cause a residual compressive stress on the surface

# size

- Size can also have effects on fatigue life
- Larger parts are more susceptible to damage/imperfections at the same stress level
- This is why composites are often made from very small fibers (glass fiber, carbon fiber, ceramic-matrix composites)



# size

- The exact effect of size will depend on material, one study for low carbon steels found

$$m_d = \left( \frac{d}{25.4\text{mm}} \right)^{-0.093}$$

- Which is then used to re-calculate material constants  $\sigma_{fd}' = m_d \sigma_f'$ ,  $\epsilon_{fd}' = m_d \epsilon_f'$

# thermal fatigue

- Thermal loading can be introduced when two dissimilar parts are attached together, the coefficient of thermal expansion causes them to expand differently, introducing extra stresses due to the temperature change
- If the temperature is significantly different between two sides of a part thermal stresses can also be introduced

# thermal fatigue

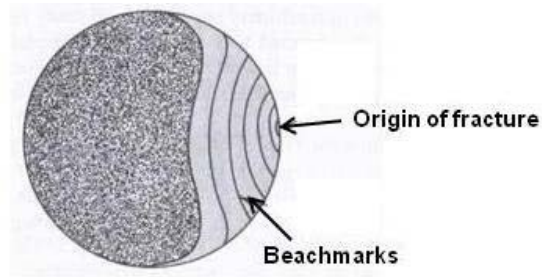
- Low temperatures generally cause a material to behave in a more brittle fashion, which alters the fatigue life
- High temperatures cause problems with creep-relaxation and can also affect the crystalline structure

# crack growth rate

# fracture surface



# fracture surface



Fatigue Fracture with Beachmarks

# crack growth rate

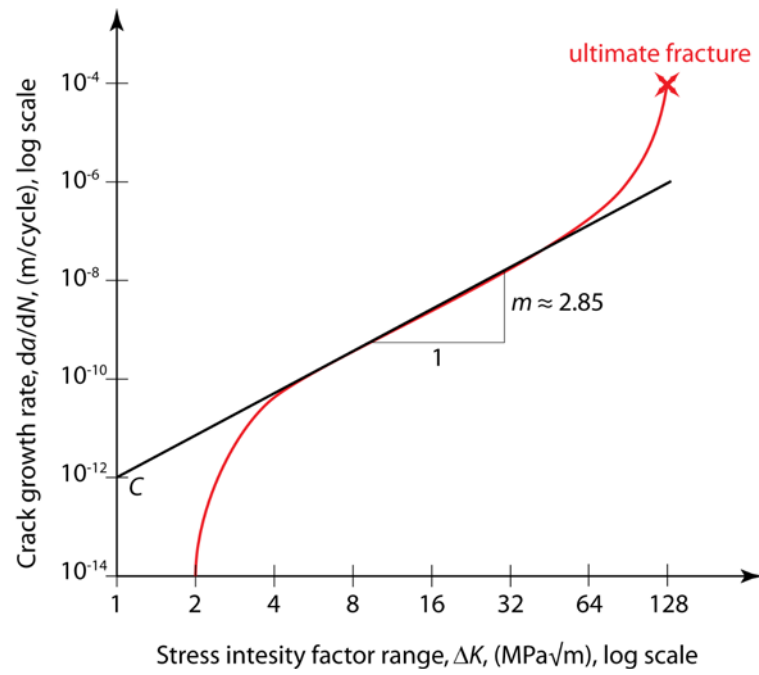
- We can observe that fatigue damage occurs through crack propagation
- “cracks” and fracture mechanics have been omitted from all our fatigue discussion thus far
- It would be beneficial to predict at what rate a crack will extend

# crack growth rate

- Crack growth rate can be measured experimentally
- Using a center-crack specimen, a fatigue load is applied
- The crack length is measured and plotted vs. the number of cycles
- The slope of this curve ( $\frac{da}{dN}$ ) is then plotted vs. either  $K_{I\max}$  or  $\Delta K_I$  on a log-log scale
- This chart is then commonly divided into three regions



# da-dN vs K



# region I

- In Region I crack growth is very slow and/or difficult to measure
- In many cases,  $da/dN$  corresponds to the spacing between atoms!
- The point at which the  $da/dN$  curve intersects the x-axis (usually with a relatively vertical slope) is called the fatigue threshold
- Typically 3-15  $\text{ksi}\sqrt{\text{in}}$  for steel
- 3-6  $\text{ksi}\sqrt{\text{in}}$  for aluminum

# region II

- Most important region for general engineering analysis
- Once a crack is present, most of the growth and life occurs in Region II
- Generally linear in the log-log scale

# region III

- Unstable crack growth
- Usually neglected (we expect failure before Region III fully develops in actual parts)
- Can be significant for parts where we expect high stress and relatively short life

# crack growth rate curve

- The crack growth rate curve is considered a material property
- The same considerations for thickness apply as with fracture toughness ( $K_c$  vs.  $K_{Ic}$ )
- Is also a function of the load ratio,  $R = \sigma_{min}/\sigma_{max}$

# R effects

- While the x-axis can be either  $\Delta K$  or  $K_{max}$ , the shape of the data is the same
- When we look at the effects of load ratio,  $R$ , the axis causes some differences on the plot
- With  $\Delta K$  on the x-axis, increasing  $R$  will shift the curve up and to the left, shifting the fatigue threshold and fracture toughness on the graph as well

# R effects

- With  $K_{max}$  on the x-axis, increasing  $R$  shifts the curve down and to the right, but fatigue threshold and fracture toughness keep same values
- In general,  $R$  dependence vanishes for  $R > 0.8$  or  $R < -0.3$ . This effect is known as the band width

# **crack growth rate equations**



# crack growth rate equations

- There are many crack growth rate equations of varying complexity
- The “best” form to use will depend on design needs

# growth equations

- The important features in curve-fit equations are
  1. Region II curve fit (linear on log-log scale)
  2. Region I curve fit (fatigue threshold)
  3. Region III curve fit (critical stress intensity)
  4. Stress ratio effects
  5. Band width of R-curves

# paris law

- The original
- Fits the linear portion (Region II)
- Does not fit Region I, Region III, or have R-dependence

$$\frac{da}{dN} = C(\Delta K)^n$$

- Note: this assumes the x-axis is  $\Delta K$ , but  $\Delta K = (1 - R)K_{max}$ , so we can easily convert

# walker

- Region II is usually all that is needed for engineering, but R-dependence is often an important effect to capture
- Walker modified the Paris law to account for R-dependence

$$\frac{da}{dN} = C \left[ (1 - R)^m K_{max} \right]^n$$

- Gives a good fit for Region II with R-dependence and band width

# forman

- The Forman equation was developed to capture the effects of Region II and Region III
- Also includes the effects of  $R$ , but does not control the band width of  $R$  effects

$$\frac{da}{dN} = \frac{C \left[ (1 - R)K_{max} \right]^n}{(1 - R)K_c - (1 - R)K_{max}}$$

# modified forman

- The Forman equation can be modified to include the effect of band width

$$\frac{da}{dN} = \frac{C \left[ (1 - R)^m K_{max} \right]^n}{\left[ (1 - R)^m K_c - (1 - R)^m K_{max} \right]^L}$$

# collipriest

- The Collipriest equation fits Regions I, II and III, but has no R-dependence

$$\frac{da}{dN} = C_1 + C_2 \tanh^{-1} \left[ \frac{\log \left( \frac{K_{max}^2}{K_o K_c} \right)}{\log(K_c / K_o)} \right]$$

# modified collipriest

- Following the same methods as before, we can modify the Collipriest equation for R-dependence and band width control

$$\frac{da}{dN} = C_1 + C_2 \tanh^{-1} \left[ \frac{\log \left( \frac{(1-R)^m K_{max}^2}{K_o K_c} \right)}{\log(K_c / K_o)} \right]$$

- For a cleaner graph, experimental data at different R-values is sometimes plotted vs.  $K_{eff}$
- $K_{eff}^* = (1 - R)^m K_{max}$



# nasgrow growth rate equation

- A very complicated curve fit is provided in the NASGROW growth rate equation

$$\frac{da}{dN} = C \left[ \frac{1-f}{1-R} \Delta K \right]^n \frac{\left[ 1 - \frac{\Delta K_{th}}{\Delta K} \right]}{\left[ 1 - \frac{K_{max}}{K_{crit}} \right]}$$

- The curve fit parameters can be found in p. 307 of your text (or the NASGROLW/AFGROW documentation)

# boeing-walker growth rate equation

- The Boeing-Walker growth equation is given as (for  $R \geq 0$  )

$$\frac{da}{dN} = 10^{-4} \left( \frac{1}{mT} \right)^p \left[ K_{max} (1 - R)^q \right]^p$$

# conversion of constants

- Much of the data available to us is from Boeing, and given in terms of the Boeing-Walker equation
- We can re-write some other equations to more easily convert parameters between the various equations
- Walker-Boeing:

$$\frac{da}{dN} = 10^{-4} \left( \frac{1}{mT} \right)^p \left[ \Delta K (1 - R)^{q-1} \right]^p$$

- Walker-AFGROW:

$$\frac{da}{dN} = C_w \left[ \Delta K (1 - R)^{m-1} \right]^{n_w}$$

- Forman:

$$\frac{da}{dN} = \frac{C_F}{(1 - R)K_c - \Delta K} (\Delta K)^{n_f}$$

# conversion of constants

Walker-Boeing	Walker-AFGROW	Forman
$10^{-4}\left(\frac{1}{mT}\right)^p$	$C_w = 10^{-4}\left(\frac{1}{mT}\right)^p$	$C_F = (K_c - 1)10^{-4}\left(\frac{1}{mT}\right)^p$
q	$m = q$	
p	$n_w = p$	$n_f = p$