# AE 737 - MECHANICS OF DAMAGE TOLERANCE

#### LECTURE 15

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Last Updated: March 22, 2016 at 2:15pm

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#### **SCHEDULE**

- · 22 Mar Stress based fatigue, Homework 6 assigned
- · 24 Mar Stress based fatigue
- 29 Mar Influence of notches on fatigue, Homework 7 assigned, Homework 6 due
- · 31 Mar Strain based fatigue, project abstract due

#### OUTLINE

- 1. fatigue
- 2. nominal and local stress
- 3. fatigue tests
- 4. fatigue life analysis
- 5. fatigue limit

# FATIGUE

#### **FATIGUE**

- We refer to damage from repeated, or cyclic loads as fatigue damage
- · Some of the earliest work on fatigue began in the 1800's
- · Chains, railway axles, etc.
- · An estimated 80% of failure expenses are due to fatigue

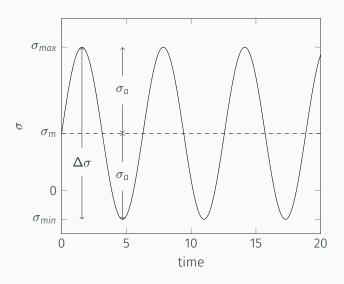
#### **FATIGUE**

- There are three main approaches to fatigue analysis
- Stress based fatigue analysis
- Strain based fatigue analysis
- · Fracture mechanics fatigue analysis

#### STRESS BASED FATIGUE

- One of the simplest assumptions we can make is that a load cycles between a constant maximum and minimum stress value
- This is a good approximation for many cases (axles, for example) and can also be easily replicated experimentally
- · This is referred to as constant amplitude stressing

## **CONSTANT AMPLITUDE STRESSING**



#### CONSTANT AMPLITUDE STRESSING

- $\Delta \sigma$  is known as the stress range, and is the difference between max and min stress
- $\sigma_m$  is the mean stress, and can sometimes be zero, but this is not always the case
- $\cdot$   $\sigma_a$  is the stress amplitude, and is the variation about the mean
- We can express all of these in terms of the maximum and minimum stress

$$\Delta \sigma = \sigma_{max} - \sigma_{min} \tag{15.1}$$

$$\sigma_m = \frac{\sigma_{max} + \sigma_{min}}{2} \tag{15.2}$$

$$\sigma_a = \frac{\sigma_{max} - \sigma_{min}}{2} \tag{15.3}$$

#### CONSTANT AMPLITUDE STRESSING

- · It is also common to describe some ratios
- The stress ratio, R is defined as

$$R = \frac{\sigma_{min}}{\sigma_{max}} \tag{15.4}$$

· And the amplitude ratio, A is defined as

$$A = \frac{\sigma_a}{\sigma_m} \tag{15.5}$$

#### **USEFUL RELATIONS**

 There are some useful relationships between the above equations

$$\Delta \sigma = 2\sigma_a = \sigma_{max}(1 - R) \tag{15.6a}$$

$$\sigma_m = \frac{\sigma_{max}}{2} (1 + R) \tag{15.6b}$$

$$R = \frac{1 - A}{1 + A} \tag{15.6c}$$

$$A = \frac{1 - R}{1 + R} \tag{15.6d}$$

# NOMINAL AND LOCAL STRESS

#### DEFINITION AND NOTATION

- It is important to distinguish between the nominal (global) stress and the local stress at some point of interest
- We use  $\sigma$  for the stress at a point (local stress)
- We use S as the nominal (global) stress
- In simple tension,  $\sigma = S$
- For many cases (bending, notches),  $\sigma \neq S$  in general
- We must also be careful to note  $\sigma_y$ , in some cases  $S<\sigma_y$  but at some locations  $\sigma>\sigma_y$

## SIMPLE TENSION

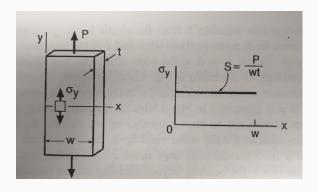
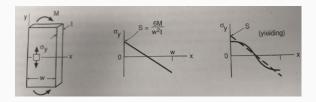
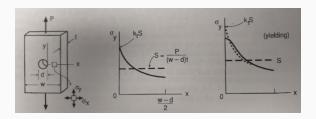


Figure 1: In this case  $S=\sigma$ 

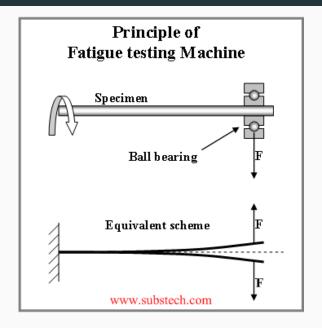


**Figure 2:** As long as  $\sigma < \sigma_y$ ,  $\sigma$  varies linearly. If  $\sigma > \sigma_y$  at any location, however, the relationship is non-linear

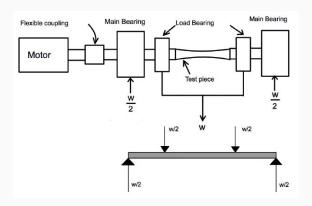


**Figure 3:** As long as  $\sigma < \sigma_y$ ,  $\sigma$  varies linearly. If  $\sigma > \sigma_y$  at any location, however, the relationship is non-linear

# FATIGUE TESTS



#### ROTATING FOUR-POINT BEND



**Figure 5:** Four-point bend gives uniform stress (along top and bottom surfaces)

#### **FATIGUE TESTS**

- The above rotating methods are very common, but in their current configurations can only be used for zero mean stress
- a reciprocating bend test can be used for non-zero mean stress

#### RECIPROCATING BEND TEST

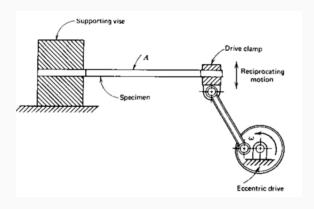


Figure 6: A reciprocating cantilever test allows for non-zero mean stress

#### **AXIAL FATIGUE TEST**



SHANGHAI HUALONG TEST INSRUMENTS CORP

Enter into a high precision testing world

# Universal Testing Machine



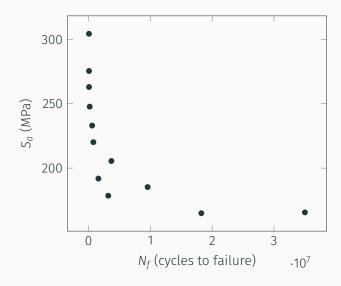
#### **FATIGUE TESTS**

- · The length of a fatigue test is determined by two factors
  - 1. How many cycles it takes for the specified load to cause failure
  - 2. The speed of the motor controlling the test
- · Servohydraulic machines generally have a speed of 10 100 Hz.
- At a speed of 100 Hz, it would take 28 hours for 10<sup>7</sup> cycles, 12 days for 10<sup>8</sup> cycles, and nearly 4 months for 10<sup>9</sup> cycles
- While some machines can test at very high speeds, the inertia of the sample can interfere with results

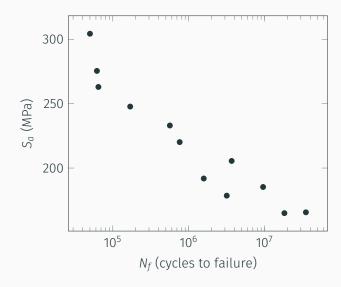


- Stress-life curves, or S-N curves, are generated from test data to predict the number of cycles to failure
- In general, one set (or family) of S-N curves is generated using the same  $\sigma_m$
- Usually  $S_a$  (the nominal stress equivalent of  $\sigma_a$ ) is plotted versus N (the number of cycles)

- Each individual point on an S-N curve represents one fatigue experiment
- To find enough data to form statistical significance, as well as to fit a curve across all levels of fatigue is very time-consuming
- In the following plot, if only one test was performed for each point, the total number of cycles tested would be about 7.3x10<sup>7</sup>
- For a 100 Hz machine, this represents over 200 hours of consecutive testing
- Each repetition would further increase the test time required



- On a linear scale, the data appear not to agree well with any standard fit
- It is also very difficult to differentiate between low-cycle fatigue failure stresses
- Instead S-N curves are often plotted on a semi-log or log-log scale, so pay attention to the axes



#### **CURVE FITS**

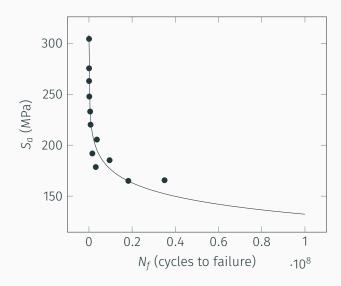
 If the curve is nearly linear on a log-linear plot, we use the following form to fit the data

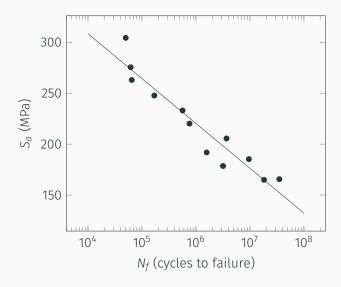
$$\sigma_a = C + D \log N_f \tag{15.7}$$

 When the data are instead linear on a log-log scale, the following form is generally used

$$\sigma_a = \sigma_f' \left( 2N_f \right)^b \tag{15.8}$$

•  $\sigma'_f$  and b are often considered material properties and can often be looked up on a table (p. 235)



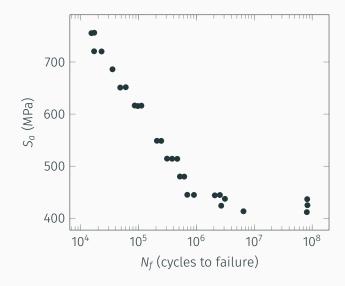




#### **FATIGUE LIMIT**

- The fatigue limit, or endurance limit, is a feature of some materials where below a certain stress, no fatigue failure is observed
- Below the fatigue limit, this material is considered to have infinite life
- This most notably occurs in plain-carbon and low-alloy steels
- $\cdot$  In these materials,  $\sigma_e$  is considered to be a material property
- This phenomenon is not typical of aluminum or copper alloys, but is sometimes arbitrarily assigned using whatever the failure stress is at some large number of cycles (10<sup>7</sup> or 10<sup>8</sup>)

#### **FATIGUE LIMIT**



#### HIGH AND LOW CYCLE FATIGUE

- Some other important terms are high cycle fatigue and low cycle fatigue
- "High cycle fatigue" generally is considered anything above 10<sup>3</sup> cycles, but varies somewhat by material
- High cycle fatigue occurs when the stress is sufficiently low that yielding effects do not dominate behavior
- When yielding effects do dominate behavior, the strain-based approach is more appropriate