# AE 737 - Mechanics of Damage Tolerance

Lecture 6

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#### schedule

- 9 Feb Fracture Toughness, Homework 2 Due, Homework 3 Assigned
- 11 Feb Fracture Toughness
- 16 Feb Residual Strength, Homework 3 Due, Homework 4 Assigned
- · 18 Feb Residual Strength
- 23 Feb Multiple Site Damage, Homework 4 Due, Homework 5 Assigned
- · 25 Feb Mixed-mode Fracture

### outline

- 1. fracture toughness
- 2. plane strain, brittle
- 3. plane stress, ductile

- The critical load at which a cracked specimen fails produces a critical stress intensity factor
- The "critical stress intensity factor" is known as  $K_c$
- For Mode I, this is called  $K_{IC}$
- The critical stress intensity factor is also known as fracture toughness

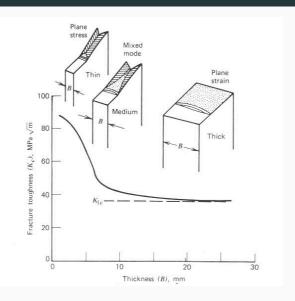
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• NOTE: "Fracture Toughness" can also refer to  $G_{lc}$ , which is analogous to  $K_{lc}$ , but not the same

- Fracture toughness is a material property, but it is only well-defined in certain conditions
- · Brittle materials
- Plane strain (smaller plastic zone)
- In these cases ASTM E399-12 is used.



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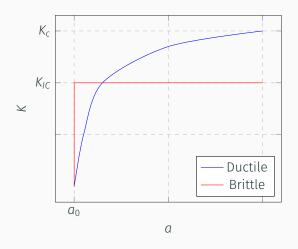
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- The load must be continually increased until a critical value causes unstable crack growth

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- We can calculate a unique stress intensity factor  $K_{li}$  at each of these points
- These are then used to create a "K-curve", plotting  $K_l$  vs.  $\alpha$



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- K<sub>IC</sub> for brittle/plane strain is very well defined
- $K_C$  for plane stress can refer to two things
- Either the maximum  $K_C$  during a test, or tangent point on  $K_R$ -curve (R-curve)

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 (6.2a)  

$$C = \frac{2a^3}{3EI}$$
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· For crack growth to be stable we need

$$\frac{dG}{da} \le 0 \tag{6.3}$$

· Under fixed-load conditions, we find

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 Which is always stable, so for DCB tests, displacement control is generally used

# plane strain, brittle

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- For relatively brittle materials, we don't need to worry about the R-curve
- Specimens are made according to these specifications

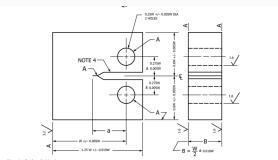
$$a \ge 2.5 \left(\frac{K_{IC}}{\sigma_{YS}}\right)^2$$
 (6.6a)

$$b \ge 2.5 \left(\frac{K_{IC}}{\sigma_{YS}}\right)^2 \tag{6.6b}$$

$$W \ge 5 \left(\frac{K_{IC}}{\sigma_{YS}}\right)^2 \tag{6.6c}$$

#### ASTM E399

- 1. Select specimen size (see (6.6))
- 2. Select specimen type (Compact Tension or Single Edge Notched Bend)



Noτe 1-Surface finishes in μm.

Note 2-A surfaces shall be perpendicular and parallel to within 0.002 W TIR.

Norn 3—The intersection of the crack starter notch tips with the two specimen surfaces shall be equally distant from the top and bottom edges of the specimen within 0.005 W.

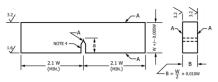
Note 4—Integral or attachable knife edges for clip gage attachment to the crack mouth may be used (see Figs. 3 and 4).

Note 5—For starter notch and fatigue crack configuration see Fig. 5.

Note 6-1.6 μm = 63 μin., 3.2 μm = 125 μin.

FIG. A4.1 Compact C(T) Specimen—Standard Proportions and Tolerances

#### ASTM E399



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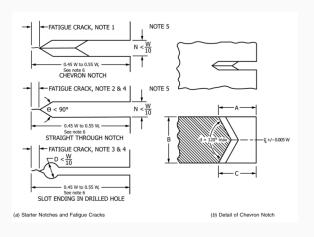
Note 2-A surfaces shall be perpendicular and parallel as applicable within 0.001 W TIR.

Note 3—Crack starter notch shall be perpendicular to specimen surfaces within 2°.

Note 4—Integral or attachable knife edges for clip gage attachment may be used (see Figs. 3 and 4)

Note 5—For starter notch and fatigue crack configuration see Fig. 5. Note 6—1.6  $\mu$ m = 63  $\mu$ in., 3.2  $\mu$ m = 125  $\mu$ in.

FIG. A3.1 Bend SE(B) Specimen—Standard Proportions and Tolerances



- 3. Machine specimen
- 4. Fatigue crack specimen  $K_f < 0.6K_{IC}$ 
  - This is to ensure that the plastic zone size during fatigue is smaller than the plastic zone size during testing
  - If  $K_{IC}$  has not yet been determined, you may have to guess the first time

- 5. Mount specimen, attach gage
- 6. Load rate should ensure "static" load conditions. (30 150 ksi√in./min.)
- 7. Determine the "provisional" value of  $K_{IC}$  (known as  $K_Q$ )



7.1 If the load-displacement curve is like the first figure, with some non-linearity, we let  $P_Q$  be the point of intersection between the load-displacement curve and a line whose slope is 5% lower than the slope in the elastic region

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$$K_Q = \frac{P_Q}{BW^{1/2}} f\left(\frac{a}{W}\right)$$
 Compact Tension (6.7)

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 SENB (6.8)

8. Ensure that your specimen is still valid

$$a \ge 2.5 \left(\frac{K_Q}{\sigma_{YS}}\right)^2 \tag{6.9}$$

$$b \ge 2.5 \left(\frac{K_Q}{\sigma_{YS}}\right)^2 \tag{6.10}$$

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23

 Load-displacement should have an initial slope between 0.7 and 1.5

# plane stress, ductile

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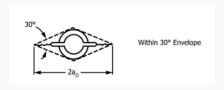
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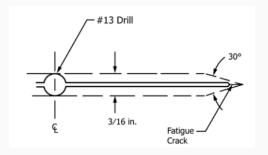
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- The other specimen which is permitted is a middle-cracked tension specimen (M(T))
- M(T) specimens are preferred in many cases due to a more uniform stress distribution (particularly important for anisotropic materials)





# minimum sample dimensions

Table of Minimum M(T) Specimen Geometry for Given Conditions											
K <sub>Rma</sub>	K <sub>Rmax</sub> /σ <sub>YS</sub>		Width		2a <sub>o</sub>		Length <sup>A</sup>				
√m	√in.	m	in.	m	in.	m	in.				
0.08	0.5	0.076	3.0	0.025	1.0	0.229	9				
0.16	1.0	0.152	6.0	0.051	2.0	0.457	18				
0.24	1.5	0.305	12.0	0.102	4.0	0.914	36				
0.32	2.0	0.508	20.0	0.170	6.7	0.762	30				
0.48	3.0	1.219	48.0	0.406	16.0	1.829	72				

Figure 1: M(T) minimum recommended dimensions

Table of Minimum C(T) Specimen Width W for Given Conditions, m (in.)										
K <sub>Rmax</sub> /σ <sub>YS</sub>		Maximum a <sub>p</sub> /W								
√m	√in.	0.4	0.5	0.6	0.7	0.8				
0.10	0.6	0.02	0.03	0.03	0.04	0.06				
		(0.8)	(1.0)	(1.3)	(1.7)	(2.5)				
0.20	1.3	0.08	0.10	0.13	0.17	0.25				
		(3.3)	(4.0)	(5.0)	(6.7)	(10.0)				
0.30	1.9	0.19	0.23	0.29	0.38	0.57				
		(7.5)	(9.0)	(11.3)	(15.0)	(22.6)				
0.40	2.5	0.34	0.40	0.51	0.67	1.01				
		(13.3)	(15.9)	(19.9)	(26.5)	(39.8)				
0.50	3.1	0.53	0.64	0.80	1.06	1.59				
		(20.9)	(25.1)	(31.3)	(41.8)	(62.7)				

Figure 2: C(T) minimum recommended dimensions

# effective crack length

- ASTM E561 describes three ways to obtain the effective crack length during testing
  - 1. Measure the crack length visually and calculate  $r_p$

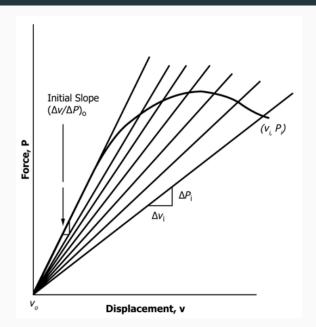
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  - 2. Measure crack length using "unloading compliance" and adding plastic zone size
  - 3. Measure the effective crack size directly using "secant compliance"

# secant compliance



 Using the slope data from our load-displacement curve, we can calculate the effective crack length using

$$EB\left(\frac{\Delta v}{\Delta P}\right) = \frac{2Y}{W} \sqrt{\frac{\pi a/W}{\sin(\pi a/W)}}$$

$$\left[\frac{2W}{\pi Y} \cosh^{-1}\left(\frac{\cosh(\pi Y/W)}{\cos(\pi a/W)}\right) - \frac{1+\nu}{\sqrt{1+\left(\frac{\sin(\pi a/W)}{\sinh(\pi Y/W)}\right)^2}} + \nu\right]$$
(6.14)

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- The following equations are used to give a good initial guess to use in iterations

$$X = 1 - \exp\left[\frac{-\sqrt{[EB(\Delta v/\Delta P)]^2 - (2Y/W)^2}}{2.141}\right]$$
(6.15)

$$\frac{2a}{W} = 1.2235X - 0.699032X^2 + 3.25584X^3 - 6.65042X^4 + 5.54X^5 - 1.66989X^6$$
 (6.16)

 In the above equations, the following are the definitions of parameters used

*E* = Young's Modulus

 $\Delta v/\Delta P =$  specimen compliance

B = specimen thickness

W = specimen width

Y = half span of the displacement measurement points

a = effective crack length

 $\nu = Poisson's ratio$ 

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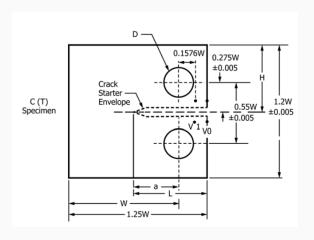
$$EB\frac{\Delta v}{\Delta P} = A_0 + A_1 \left(\frac{a}{W}\right) + A_2 \left(\frac{a}{W}\right)^2 + A_3 \left(\frac{a}{W}\right)^3 + A_4 \left(\frac{a}{W}\right)^4 \quad (6.17)$$

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 The coefficients will differ based on where the displacement is measured from

## secant compliance C(T)



# secant compliance C(T)

location	$A_0$	$A_1$	$A_2$	$A_3$	$A_4$	
$V_0$	120.7	-1065.3	4098.0	-6688.0	4450.5	
$V_1$	103.8	-930.4	3610.0	-5930.5	3979.0	
location	_	_	_	_	_	_
location	$C_0$	$C_1$	$C_2$	$\mathcal{L}_3$	$C_4$	$C_5$
	- 0	•			1214.90	

### secant compliance C(T)

· Where the initial guess for a is provided by

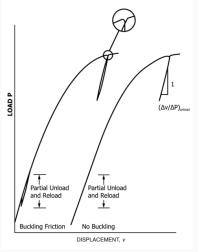
$$\frac{a}{W} = C_0 + C_1 U + C_2 U^2 + C_3 U^3 + C_4 U^4 + C_5 U^5$$
 (6.18)

· and U is given by

$$U = \frac{1}{1 + \sqrt{EB\frac{\Delta V}{\Delta P}}} \tag{6.19}$$

### buckling

 If the test is stopped and re-started frequently (to measure crack length by hand or to use the compliance method of crack measurement) buckling can interfere with results



### buckling

- If buckling is shown to be present in the test, supports can be used to prevent buckling
- These supports can introduce friction
- They should be well-lubricated for accurate test results

#### net section stress

- One final consideration when dealing with plane stress fracture mechanics is the net section stress
- For the test to be valid, failure must occur due to fracture, not general static failure
- Static failure will occur when  $\sigma_N = \sigma_{YS}$

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- For this reason, we often plot  $K_{le}$  vs.  $\Delta a$ , to subtract the initial crack length
- We can superpose constant-stress K-curves on this graph, the curve which intersects at a tangent point creates the most "standard" definition for  $K_C$

### example

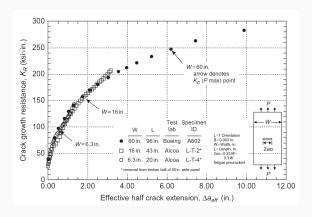
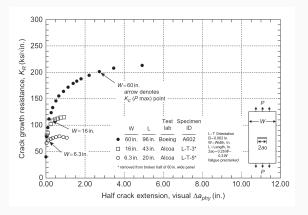


Figure 3:  $K_R$  Curve for C188-T3 aluminum for varying sample thickness (Boeing and Alcoa)

### example



**Figure 4:**  $K_R$  curve for the same specimens, but without adjusting for the plastic zone size.