

Lecture 3 - Strain

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schedule

- 25 Aug - Strain, Homework 1 Due
- 27 Aug - Mechanical Properties
- 1 Sep - Exam Review, Homework 2 Due
- 3 Sep - Axial Load (not on Exam 1)
- 4 Sep Project 1 Due
- 8 Sep - Exam 1 (online)

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- allowable stress design
- limit state design
- strain

allowable stress

- Most of the time, we design structures so the stress is less than some limit
- By setting a conservative allowable stress, we account for some manufacturing tolerances, unintended loads, and variability in mechanical properties

factor of safety

- The factor of safety is the failure load divided by the allowable load

$$FS = \frac{F_{fail}}{F_{allow}}$$

- Since load and stress are linearly proportional, we could also define the factor of safety in terms of stress and it would be identical

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factor of safety

- Typical values for the factor of safety will vary based on application
- Aircraft and space vehicles might have a factor close to 1 to minimize weight
- Nuclear power plants might have a factor close to 3 since weight is not as important and failure would be catastrophic

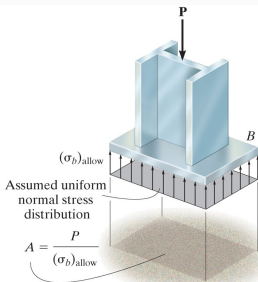
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- We can rearrange the equations $\sigma = N/A$ and $\tau = V/A$ to size components based on some allowable stress

$$A = \frac{N}{\sigma_{allow}} \quad A = \frac{V}{\tau_{allow}}$$

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bearing stress



The area of the column base plate B is determined from the allowable bearing stress for the concrete.

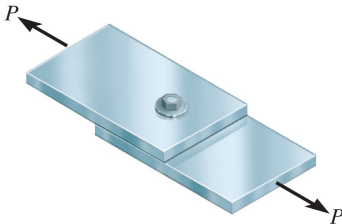
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embedded shear stress



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lap joint shear



The area of the bolt for this lap joint is determined from the shear stress, which is largest between the plates.

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limit state design

- Allowable stress design accounts for uncertainty in the applied loading and the material properties in one factor of safety
- Limit state design separates these two into load and resistance factors

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load factors

- The load factor combines uncertainty in various types of load
- For example, a building can have loading from a few different sources, such as its own weight, people in the building, and snow on top of the building
- Weight is considered a *dead load* and can usually be determined more precisely than moving things like people

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load factors

- In this simple example, we consider a load factor, $\gamma_D = 1.2$ for the dead load, $\gamma_L = 1.6$ and $\gamma_S = 0.5$

$$R = 1.2D + 1.6L + 0.5S$$

- These load factors combine the concept of a safety factor with the probability that loads will occur

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resistance factors

- Resistance factors, ϕ are used to express the probability a material will fail at its limit load
- If we are very confident in the failure stress of a material (i.e. steel has little variability), we might use $\phi = 0.9$
- If we are not as confident, (using a new material, or an organic material like wood with higher variability), we might use $\phi = 0.7$

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- If we call the nominal load P , then we can combine load and resistance factors using

$$\phi P \geq R$$

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example 1-12

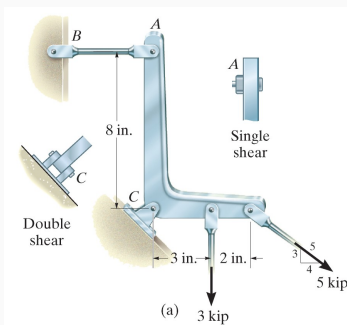


Figure 1: An L-shaped bracket has an 8 inch vertical leg and a 3 inch horizontal leg. A single shear pinned connector holds the point of the leg, A, in place while a double shear pinned connector holds the point of the leg, C, in place. A 3 kip vertical force is applied at the end of the horizontal leg, and a 5 kip force is applied at the end of the vertical leg. A 5 kip force is also applied at the end of the horizontal leg.

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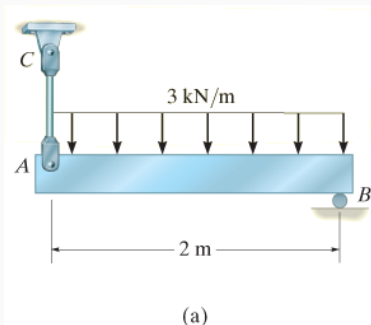


Figure 2: A 2 meter long beam is supported at the left end with a steel rod connecting vertically. It is subjected to a uniform load of 3 kilonewtons per meter and a roller support at the

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deformation

- When forces are applied to a body, it will change its shape and size
- We call these changes *deformation*
- Sometimes they are barely noticeable (steel), other times they are very significant (rubber)

strain

- Strain is a more precise measurement of the deformation of a body
- Normal strain is given as the change in length divided by the original length

$$\epsilon = \frac{L - L_0}{L_0}$$

- We can consider the average normal strain (over an entire body) or the local strain (take an infinitely small portion and calculate the strain there)

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units

- Since we divide length by length, strain is unitless
- However it is customary to use *in/in* or for stiff specimens to use the phrase *microstrain* as a unit
- Strain can also sometimes be represented as a percent

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- Normal strain causes a line segment to expand or contract
- Deformation can also cause two lines to change their relative angle
- The change in angle between two originally perpendicular line segments is called shear strain

$$\gamma = \frac{\pi}{2} - \theta$$

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shear strain

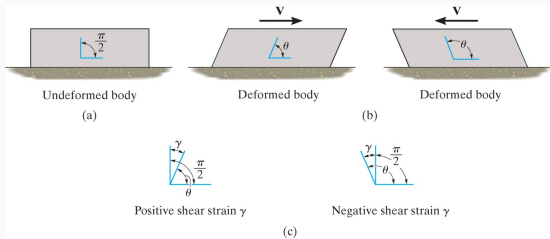


Figure 3: Three stages are shown, the first is a rectangular block at rest, with a fixed support on the ground. The second shows the block after it deforms with a horizontal force acting along the top to the right. The third shows the block after it deforms with a force acting along the top to the left. The first

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- If we consider a very small cube/prism with sides of Δx , Δy , and Δz , normal strains will change the side lengths to

$$(1 + \epsilon_x)\Delta x(1 + \epsilon_y)\Delta y(1 + \epsilon_z)\Delta z$$

- And the shear strains will change the shape

$$\frac{\pi}{2} - \gamma_{xy} \quad \frac{\pi}{2} - \gamma_{yz} \quad \frac{\pi}{2} - \gamma_{xz}$$

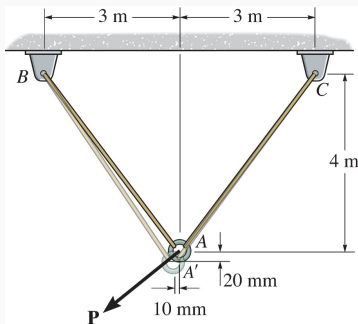
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small strain

- Most engineering analysis is based on the assumption of small strains
- This is valid for many materials (wood, metal), but not for rubbers and some other polymers
- When strains are small, we assume that the change in angle, $\Delta\theta$ is very small
- $\sin \Delta\theta \approx \Delta\theta$, $\cos \Delta\theta \approx 1$, $\tan \Delta\theta \approx \Delta\theta$

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example 2.1



Find the normal strains in the two wires if A moves to A'

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example 2.3

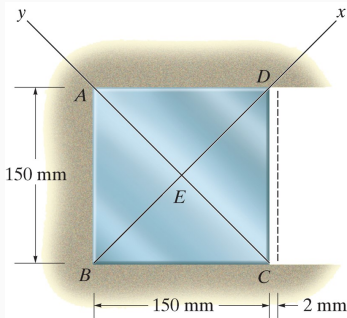


Figure 4: A 150 mm square block is constrained along the top, left, and bottom faces, but pushed 2 mm to the left on its right face. AC is the diagonal line going from the top left to the

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