Mechanics of Materials

Lecture 15 - Combined Loading

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schedule

- 5 April Combined Loading, HW 6 Self-Grade Due
- 7 April Stress Transformation
- 12 April Stress Transformation, HW 7 Due
- 14 April Strain Transformation

outline

- pressure vessels
- combined loading
- group problems

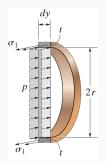
pressure vessels

thin-walled pressure vessels

- If the radius to wall thickness ratio is 10 or more, we can treat a pressure vessel as "thin-walled"
- Cylindrical pressure vessels will have two primary sources of stress, and serve as an introduction to more general states of combined loading

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cylindrical vessels



cylindrical vessels

 From equilibrium of a section of a cylindrical vessel, we see that

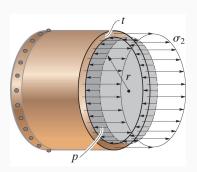
$$\sum F_x = 0$$

$$= 2(\sigma_1 t dy) - p(2r) dy$$

$$\sigma_1 = \frac{pr}{t}$$

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cylindrical vessels



cylindrical vessels

Considering another section we can find the longitudinal stress

$$\sum F_y = 0$$

$$= \sigma_2(2\pi rt) - p(\pi r^2)$$

$$\sigma_2 = \frac{pr}{2t}$$

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spherical vessels

 We can find the stress in spherical vessels using an identical section to the longitudinal section for a cylindrical vessel, and we find that

$$\sigma = \frac{pr}{2t}$$

• Which is valid everywhere in a cylindrical vessel

example 8.1

- A cylindrical pressure vessel has an inner diameter of 4 ft and a thickness of 1/2 in.
- Determine the maximum internal pressure it can sustain if the maximum stress it can support is 20 ksi.
- What is the maximum internal pressure a spherical pressure vessel could sustain under identical conditions?

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combined loading

combined loading

 We can use the principle of superposition to treat various loading conditions separately and then add them together to find the total stress

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procedure

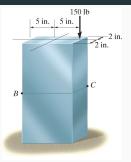
- Section the member at the point of interest, internal force components should be drawn acting through the centroid of the section
- Moment components should be calculated about the centroidal axis

stress components

- Normal stress: $\sigma = N/A$
- Torsion: $\tau = \frac{T\rho}{J}$
- Bending: $\sigma = \frac{-My}{I}$
- Transverse Shear: $\tau = \frac{VQ}{It}$
- Pressure Vessels: $\sigma_1 = \frac{pr}{t}$, $\sigma_2 = \frac{pr}{2t}$

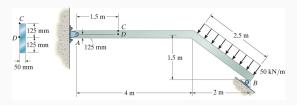
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example 8.2



Neglect the weight of the member and find the stress at $\ensuremath{\mathsf{B}}$ and $\ensuremath{\mathsf{C}}.$

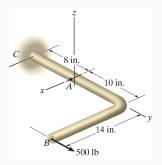
example 8.4



Determine the stress at C and D.

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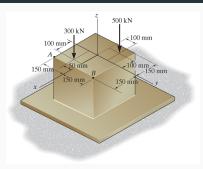
example 8.5



The rod shown has a radius of 0.75 in. Find the stress at A.

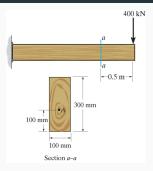
group problems

group one



Find the stress at the corners A and B for the column shown.

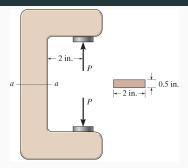
group two



Find the stress at point A for the cantilever beam shown.

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group three



Find the load P that will cause a maximum normal stress of $\sigma=30$ ksi along the section a-a.