Mechanics of Materials

Lecture 23 - Deflection of Beams

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schedule

- 15 Apr Beam Deflection
- 17 Apr Recitation, HW8 Due
- 20 Apr Beam Deflection (discontinuity functions)
- 20 Apr Exam 3a Due by midnight
- 22 Apr Beam Deflection (superposition)
- 24 Apr Recitation, HW9 Due

outline

- deflection of beams and shafts
- slope and displacement

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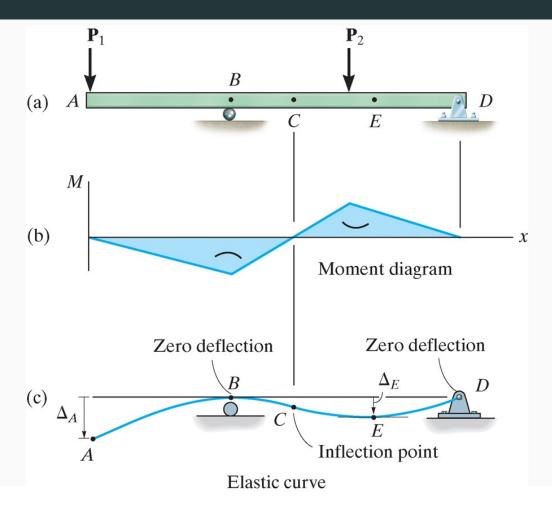
deflection of beams and shafts

elastic curve

- Before finding the exact displacement, it is useful to sketch the approximate deformed shape of a beam
- For difficult beams, it is useful to draw the moment curve first
- Positive internal moment tends to bend the beam concave up,
 while negative moment tends to bend the beam concave down

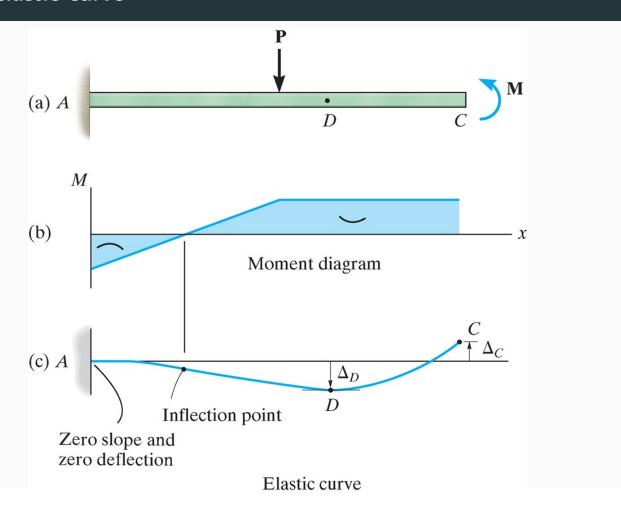
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elastic curve



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elastic curve



moment-curvature

 In Chapter 6 we compared the strain in a segement of a beam to the radius of curvature and found

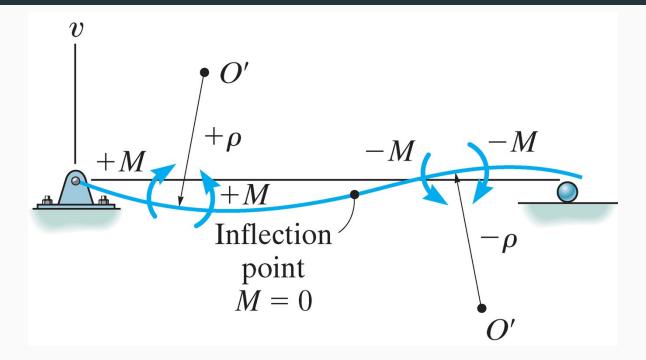
$$\frac{1}{\rho} = -\frac{\epsilon}{y}$$

• Since Hooke's Law applies, $\epsilon = \sigma/E = -My/EI$, substituting gives

$$\frac{1}{\rho} = \frac{M}{EI}$$

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sign convention



 ρ is positive when the center of the arc is above the beam, negative when it is below.

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slope and displacement

curvature

- When talking about displacement in beams, we use the coordinates v and x, where v is the vertical displacement and x is the horizontal position
- In this notation, curvature is formally related to displacement according to

$$\frac{1}{\rho} = \frac{d^2v/dx^2}{[1 + (dv/dx)^2]^{3/2}} = \frac{M}{EI}$$

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curvature

■ The previous equation is difficult to solve in general, but for cases of small displacement, (dv/dx)2 will be negligible compared to 1, which then simplifies to

$$\frac{d^2v}{dx^2} = \frac{M}{EI}$$

flexural rigidity

- In general, M, is a function of x, but EI (known as the flexural rigidity) is a constant along the length of the beam
- In this case, we can say

$$EI\frac{d^{2}v}{dx^{2}} = M(x)$$

$$EI\frac{d^{3}v}{dx^{3}} = V(x)$$

$$EI\frac{d^{4}v}{dx^{4}} = w(x)$$

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boundary conditions

- If a support restricts displacement, but not rotation, we will have a boundary condition of v=0 at that point
- Supports that restrict rotation give a boundary condition that $\theta=0$

continuity conditions

- If we have a piecewise function for M(x), not all integration constants can be found from the boundary conditions
- Instead, we must also use continuity conditions to ensure that the slope and displacement are continuous at every point
- In other words, for two sets of functions, $\theta_1(x)$ and $v_1(x)$, $\theta_2(x)$, and $v_2(x)$, $\theta_1(a)=\theta_2(a)$ and $v_1(a)=v_2(a)$

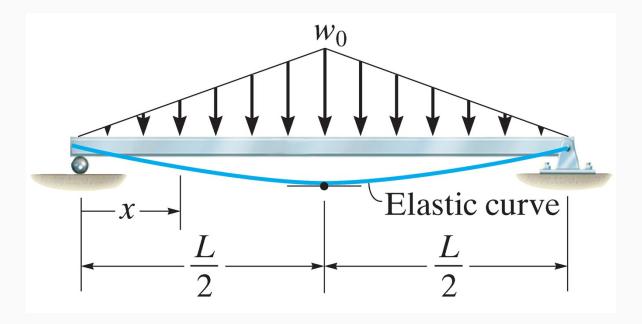
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slope

For small displacements, we have

$$\theta \approx \tan(\theta) = dv/dx$$

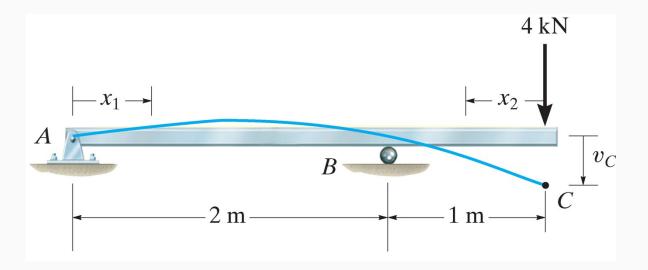
example 12.1



Determine the maximum deflection if EI is constant.

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example 12.4



Determine the displacement at C, EI is constant.