

Lecture 9 - Torsion

Dr. Nicholas Smith

Wichita State University, Department of Aerospace
Engineering

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schedule

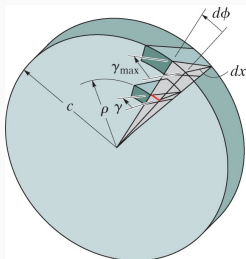
- 15 Sep - Torsion, Homework 3 Due
- 17 Sep - Torsion
- 22 Sep - Bending, Homework 4 Due
- 24 Sep - Bending

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- torsion
- power transmission
- group problems

torsion

- Torque is a moment that tends to twist a member about its axis
- For small deformation problems, we assume that the length and radius do not change significantly under torsion
- The primary deformation we are concerned with in torsion is the angle of twist, denoted with ϕ



The shear strain at points on the cross section increases linearly with ρ , i.e., $\gamma = (\rho/c)\gamma_{\max}$.

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torsion formula

- For a linearly elastic material, Hooke's Law in shear will hold ($\tau = G\gamma$)
- This means that, like shear strain, shear stress will vary linearly along the radius

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torsion formula

- We can find the total force on an element, dA by multiplying the shear stress by the area

$$dF = \tau dA$$

- The torque ($dT = \rho dF$) produced by this force is then

$$dT = \rho(\tau dA)$$

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torsion formula

- Integrating over the whole cross-section gives

$$T = \int_A \rho(\tau dA) = \frac{\tau_{max}}{c} \int_A \rho^2 dA$$

- The integral $\int_A \rho^2 dA$ is also called the Polar Moment of Inertia, J , so we can write

$$\tau_{max} = \frac{Tc}{J}$$

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polar moment of inertia

- We know that $J = \int_A \rho^2 dA$, so we can compute this for some common shapes
- For a solid circular cross-section, we have

$$J = \int_0^c \rho^2 (2\pi \rho d\rho) = \frac{\pi}{2} c^4$$

- For a circular tube we have

$$J = \int_{c_1}^{c_2} \rho^2 (2\pi \rho d\rho) = \frac{\pi}{2} (c_2^4 - c_1^4)$$

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example 5.1

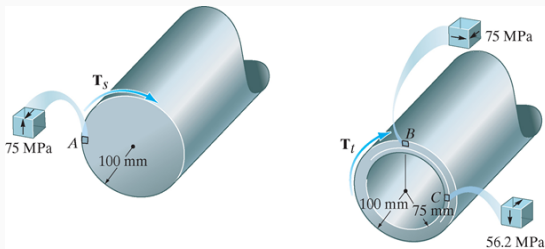


Figure 1: On left is a solid 100 mm radius tube, while on the right is a hollow tube with outer radius of 100 mm and inner radius of 75 mm. Element A is on the surface of the solid tube on the left, element B is on the outer surface of the hollow

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power transmission

- Shafts and tubes are often connected to belts and drives, and the torque, speed, and power are all related
- Power is the rate of work done, for rotation problems, $P = T\omega$
- We are often given the frequency f instead of the angular velocity, ω , in this case $P = 2\pi fT$

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power units

- In SI Units, power is expressed in Watts $1 \text{ W} = 1 \text{ N m} / \text{sec}$
- In Freedom Units, power is expressed in Horsepower
 $1 \text{ hp} = 555 \text{ ft lb} / \text{sec}$

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- We often know the power and frequency of a drive, and need to design a shaft such that the shear stress is acceptable
- We can easily find the torque as $T = P/2\pi f$, we can use this combined with the torsion equation

$$\tau_{max} = \frac{Tc}{J}$$

to find the appropriate shaft diameter. - For solid shafts we can solve for c uniquely, but not for hollow shafts

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example 5.4

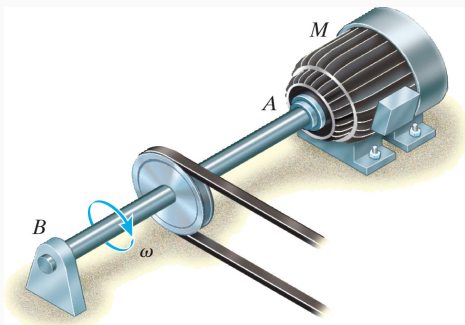


Figure 2: A rotating shaft connected to a motor

The steel shaft shown is connected to a 5-hp motor that

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group one

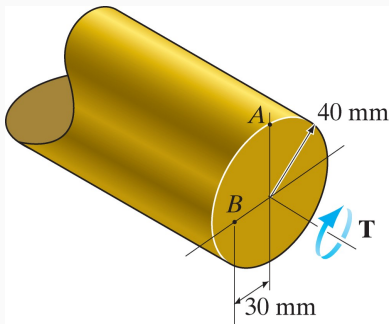


Figure 3: A 40 mm radius solid shaft. Point A is on the outer surface, Point B is 30 mm away from the center.

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group two

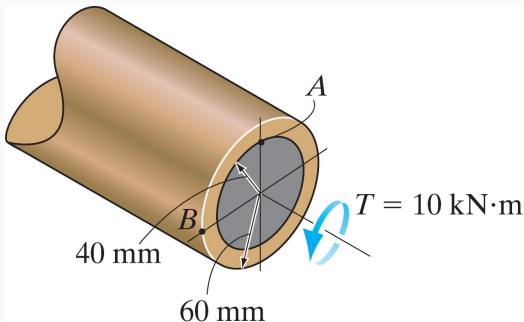


Figure 4: A hollow circular shaft with outer radius of 60 mm and inner radius of 40 mm. Point A is on the inner surface, Point B is on the outer surface.

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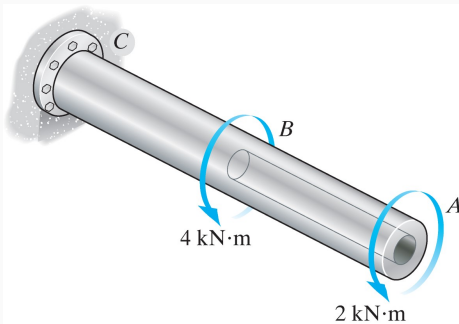


Figure 5: There is a fixed support at C, an applied torque of $4 \text{ kN}\cdot\text{m}$ at B (in the middle) and an applied torque of $2 \text{ kN}\cdot\text{m}$ at A (at the free end)

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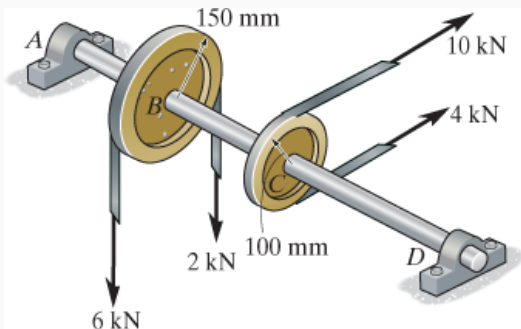


Figure 6: A shaft supports two pulleys, one with a 150 mm radius and tension of 6 kN at one end and 2 kN at the other other. The other pulley has a 100 mm radius and tensions of 10 kN

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