

Lecture 11 - Boundary Conditions

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schedule

- 22 Feb - Boundary Conditions
- 24 Feb - Project Descriptions (HW 4 Due)
- 1 Mar - FEA In-class
- 3 Mar - SwiftComp (In-class) (HW 5 Due)

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- hashin-shtrikman
- boundary conditions

hashin-shtrikman

- We consider the Voigt and Reuss micromechanics models as bounding cases, properties should never exceed the limits of these two cases
- Hashin and Shtrikman used variational principles to define more rigorous bounds for composite properties
- They did this by comparing a heterogeneous composite RVE with an equivalent homogeneous RVE

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heterogeneous

$$\sigma_{ij,j} = 0$$

$$\sigma_{ij} = C_{ijkl}\epsilon_{kl}$$

$$U = \frac{1}{2} C_{ijkl} \epsilon_{ij} \epsilon_{kl}$$

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$$\begin{aligned}\sigma_{ij,j}^{(0)} &= 0 \\ \sigma_{ij}^{(0)} &= C_{ijkl}^{(0)} \epsilon_{kl}^{(0)} \\ U &= \frac{1}{2} C_{ijkl}^{(0)} \epsilon_{ij}^{(0)} \epsilon_{kl}^{(0)}\end{aligned}$$

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relation

- To relate the two boundary problems, we introduce the following

$$\begin{aligned}u_i &= u_i^{(0)} + u_i^d \\ \epsilon_{ij} &= \epsilon_{ij}^{(0)} + \epsilon_{ij}^d \\ \sigma_{ij} &= p_{ij} + C_{ijkl}^{(0)} \epsilon_{kl} = p_{ij} + C_{ijkl}^{(0)} (\epsilon_{ij}^{(0)} + \epsilon_{ij}^d)\end{aligned}$$

- u_i^d is the disturbance displacement field and p_{ij} is called the polarization stress

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boundary conditions

- One common RVE boundary condition is known as homogeneous displacement
- Under homogeneous displacement boundary conditions we have

$$u_i = \bar{u}_i = u_i^{(0)}$$

along the boundary

- Under this condition we have $u_d = 0$ along the boundary

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hashin-shtrikman

- Hashin-Shtrikman then considered the following functional

$$\Pi = \int_V (C_{ijkl}^{(0)} \epsilon_{ij}^{(0)} \epsilon_{kl}^{(0)} - \Delta C_{ijkl}^{-1} p_{ij} p_{kl} + p_{ij} \epsilon_{ij}^d + 2p_{ij} \epsilon_{ij}^{(0)}) dV$$

- Where

$$\begin{aligned}\Delta C_{ijkl} &= C_{ijkl} - C_{ijkl}^{(0)} \\ p_{ij} &= \Delta C_{ijkl} \epsilon_{kl} \\ \epsilon_{ij}^d &= \epsilon_{ij} - \epsilon_{ij}^{(0)}\end{aligned}$$

- This functional corresponds to the strain energy in a composite when the strain field and polarization field are exact solutions

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- We can choose the comparison solid such that $\delta\Pi$ will either be a local maximum or a local minimum
- When ΔC is negative definite then the stationary value of the functional is a minimum
- When ΔC is positive definite then the stationary value of the functional is a maximum
- The functional will be stationary when

$$\left(C_{ijkl}^{(0)}\epsilon_{kl}^d\right)_{,j} + p_{ij,j} = 0$$

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- In general, I don't know how often you will need to use the Hashin-Shtrikman bounds
- For a more complete derivation, see textbook pp. 170-186

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boundary conditions

macro and micro fields

- In micromechanics, one of our primary goals is to relate a heterogeneous material to some equivalent homogeneous material
- We call ϵ_{ij} and σ_{ij} the point-wise or microscopic strain and stress
- $\bar{\epsilon}_{ij}$ and $\bar{\sigma}_{ij}$ are the macroscopic strain and stress, and are related by some unknown homogenized stiffness

$$\bar{\sigma}_{ij} = C_{ijkl}^* \bar{\epsilon}_{kl}$$

- In a homogeneous body (or equivalent homogeneous body), $\bar{\sigma}_{ij}$ and $\bar{\epsilon}_{ij}$ will be constant throughout

average stress theorem

- In general the stress field σ_{ij} will not be constant in a heterogeneous body
- If a heterogeneous body is subjected to homogeneous tractions with no body forces such that

$$t_i^0 = \bar{\sigma}_{ij} n_j$$

- And we find that

$$\langle \sigma_{ij} \rangle = \bar{\sigma}_{ij}$$

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average strain theorem

- Similarly, in general the strain field, ϵ_{ij} will not be constant in a heterogeneous body
- If a heterogeneous body is subjected to a homogeneous displacement such that

$$u_i^0 = \bar{\epsilon}_{ij} x_j$$

- And we find that

$$\langle \epsilon_{ij} \rangle = \bar{\epsilon}_{ij}$$

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hill mandel macrohomogeneity condition

- Hill and Mandel posed the question: Under what conditions will the average strain energy density of a heterogeneous body be equivalent to a homogeneous body?
- In other words, they wanted show under what conditions

$$\langle \sigma_{ij} \epsilon_{ij} \rangle = \bar{\sigma}_{ij} \bar{\epsilon}_{ij}$$

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hill mandel macrohomogeneity

- First we note that

$$\bar{\sigma}_{ij} \bar{\epsilon}_{ij} = \frac{1}{V} \int_V \sigma_{ij} \bar{\epsilon}_{ij} dV = \frac{1}{V} \int_V \bar{\sigma}_{ij} \epsilon_{ij} dV = \frac{1}{V} \int_V \bar{\sigma}_{ij} u_{i,j} dV$$

- Thus we can say that when $\langle \sigma_{ij} \epsilon_{ij} \rangle = \bar{\sigma}_{ij} \bar{\epsilon}_{ij}$

$$\langle \sigma_{ij} \epsilon_{ij} \rangle - \bar{\sigma}_{ij} \bar{\epsilon}_{ij} = \frac{1}{V} \int_V (\sigma_{ij} u_{i,j} - \bar{\sigma}_{ij} u_{i,j} - \sigma_{ij} \bar{\epsilon}_{ij} + \bar{\sigma}_{ij} \bar{\epsilon}_{ij}) dV$$

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- After some algebra and applying the divergence theorem, we can write this as

$$\langle \sigma_{ij} \epsilon_{ij} \rangle - \bar{\sigma}_{ij} \bar{\epsilon}_{ij} = \frac{1}{V} \oint_V n_k (\sigma_{ik} - \bar{\sigma}_{ik}) (u_i - x_j \bar{\epsilon}_{ij}) dS$$

- The right-hand side can be made to vanish in various ways, but the most common are homogeneous traction, homogeneous displacement, and periodic boundary conditions

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finite elements

- There are a few things we need to do when using finite elements
- First, we should ensure the mesh we use is periodic
- Second, we should ensure that our boundary conditions satisfy Hill-Mandel and that our mesh is converged
- Periodic boundary conditions converge more quickly than homogeneous stress or displacement
- Third, we should repeat our periodic structure (2x2, 3x3) to check that the effective stiffness remains constant
- We find homogenized properties by taking the volume-averaged stress and strain

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periodic boundary conditions

periodic boundary conditions

- We have used the phrase multiple times in this course
- Periodic Boundary Conditions are used in many different fields, and can be implemented in slightly different ways
- Here we will first discuss the equations for periodic boundary conditions in structural mechanics, then discuss how these can be applied

periodic boundary conditions

- If we have a periodic structure, we denote corresponding faces with + and - superscripts
- From equilibrium, we know that the tractions on opposing faces must be equal and opposite

$$t_i^+ = -t_i^-$$

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periodic boundary conditions

- The displacement on opposing surfaces will also be equal with

$$\xi_i^+ = \xi_i^-$$

where

$$\xi_i = u_i - \bar{\epsilon}_{ij}x_j$$

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- In general, homogeneous displacement boundary conditions give an upper bound estimate to stiffness properties
- Homogeneous traction boundary conditions give a lower estimate
- They converge to periodic boundary conditions for increasing RVE size

general application

- In finite element software, solutions are derived from displacement
- Thus it is easiest to implement periodic displacement conditions
- Traction will be automatically satisfied

- We find the stiffness by applying some arbitrary strain in all directions (i.e. $\bar{\epsilon}_{11} = 1$, $\bar{\epsilon}_{22} = 1$, etc.)
- The volume average of stress then corresponds to the appropriate column of the stiffness matrix
- Some finite element software programs have a built-in method for periodic boundary conditions
- When such a method is not built-in, there are various strategies to enforce the boundary condition

ABAQUS implementation

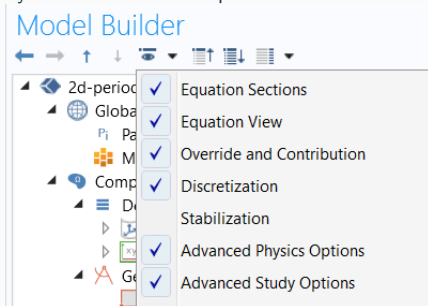
- In ABAQUS PBC's are implemented using equations for each boundary node
- Boundary nodes on a given face are tied to some “dummy” node
- Equations for each node ensure that $\xi_i^+ = \xi_i^-$

- While COMSOL has periodic boundary conditions built-in, there are some quirks to how it is implemented
- The default periodic boundary condition is $u_i^+ = u_i^-$
- This forces displacement to be exactly the same on opposing faces, but we would like for them to be the same with some arbitrary offset
- To implement this requires viewing and modifying the boundary condition equations

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COMSOL implementation

- From the Model Builder on the left-hand side, click the “eye” icon to show the equations

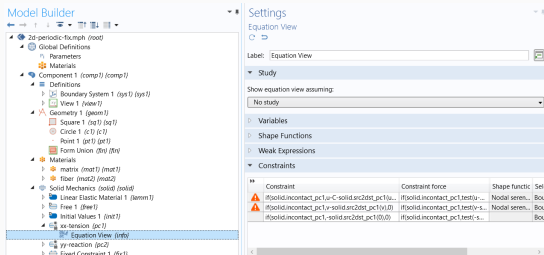


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- Next, add a global parameter for the arbitrary strain (in my equations I used 'C')
- Now we need to edit the periodic boundary equations in comsol to include an offset of 'C'
- For example we need to change u-solid.src2dst to u-C-solid.src2dst for the x-faces periodic boundary in COMSOL

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COMSOL implementation



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- In shear the COMSOL implementation becomes a little bit tricky

$$\xi = u_i - \bar{\epsilon}_{ij}x_j$$

- We need to define displacement on both surfaces under consideration (1 and 2 surface for 12 shear)

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rigid constraints

- In general the problem is not constrained against rigid body translation
- Since the displacements are periodic on the surfaces, it is most appropriate to create a point at the center and fix it
- Points can be generated from the geometry menu
- The constraint can be difficult to apply graphically (the point is not visible)
- Instead choose your point from the drop-down

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stiffness results

- For any software package, we take the results using the volume-average stiffness and strain
- Recall that in engineering notation we have

$$\sigma_i = C_{ij}\epsilon_j$$

- Before calculating stiffness values, you may want to check that you have a mostly uniform strain
- Stiffness values can be calculated as

$$C_{ij} = \sigma_i / \epsilon_j$$

- Where j is fixed for each load configuration

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other comments

- With boundaries under displacement control, we do not need to worry about constraining any other nodes to restrict rigid body motion
- The default (tetrahedral) mesh in COMSOL behaves adequately under these conditions (some small dimpling on the non-restricted surfaces)
- In COMSOL, volume-average properties can be calculated by right-clicking derived values (under results) > average > volume average
- Stress, strain, and stiffness can be found by typing in the “expressions”
- `solid.sl11` (Stress Local 11 direction), `solid.el11` (Strain Local 11 direction)

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- A sample COMSOL file to show the implementation of periodic boundary conditions can be viewed here¹

¹<http://ndaman.github.io/multiscale/handout/2d-periodic.mph>