## AE 760AA: Micromechanics and multiscale modeling

Lecture 6 - Orientation Averaging

Dr. Nicholas Smith

Wichita State University, Department of Aerospace Engineering

February 11, 2019

#### schedule

- Feb 11 Orientation Averaging
- Feb 13 Physical measurements (HW2 Due)
- Feb 18 Variational Calculus
- Feb 20 Variational Calculus

#### outline

- orientation averaging
- closure approximations
- variational calculus

# orientation average

#### orientation tensor

- Within a given volume, a distribution of fibers can be defined by some orientation distribution function,  $\psi(\theta, \phi)$ .
- Advani and Tucker introduced tensor representations of fiber orientation distribution functions

$$a_{ij}=\oint p_i p_j \psi(p) dp$$

• And

$$a_{ijkl} = \oint p_i p_j p_k p_l \psi(p) dp$$

• Note: any order tensor may be defined in this manner, the orientation distribution function must be even, due to fiber symmetry, and thus any odd-ordered tensor will be zero.

- Consider T(p) to be some tensor property of a material, as a function of material orientation
- The orientation average of T(p) is denoted by angle brackets and is given by

$$\langle T 
angle = \oint T(p) \psi(p) dp$$

- For a uni-directional fiber, we would expect  $\langle T \rangle$  to be transversely isotropic, which for a second-order tensor requires  $\langle T_{ij} \rangle = A_1 \langle p_i p_j \rangle + A_2 \delta_{ij}$
- but  $\langle p_i p_j \rangle$  is the second-order orientation tensor
- The unknown constants,  $A_1$  and  $A_2$ , can be easily solved for in terms of the uni-directional properties

• Similarly, if *T* is a fourth-order tensor property then transverse isotropy requires that

$$egin{aligned} \langle T_{ijkl}
angle &= B_1 a_{ijkl} + B_2 (a_{ij}\delta_{kl} + a_{kl}\delta_{ij}) + \ B_3 (a_{ik}\delta_{jl} + a_{il}\delta_{jk} + a_{jl}\delta_{ik} + a_{jk}\delta_{il}) + \ B_4 (\delta_{ij}\delta_{kl}) + B_5 (\delta_{ik}\delta_{jl} + \delta_{il}\delta_{jk}) \end{aligned}$$

- We can solve for  $B_{\alpha}$  by considering fibers aligned in the three-direction, we have  $a_{3333} = 1$  and all other  $a_{ijkl} = 0$ .
- We can choose any values of i, j, k, l that would give 5 unique equations to solve equations for  $B_{\alpha}$

• Here we will consider  $T_{1111},\,T_{3333},\,T_{1122},\,T_{2233},\,T_{1313}.$ 

$$egin{aligned} T_{1111} &= B_4 + 2B_5 \ T_{3333} &= B_1 + 2B_2 + 4B_3 + B_4 + 2B_5 \ T_{1122} &= B_4 \ T_{2233} &= B_2 + B_4 \ T_{1313} &= B_3 + B_5 \end{aligned}$$

• After some manipulation, we find

$$egin{align} B_1 &= T_{1111} + T_{3333} - 2T_{2233} - 4T_{1313} \ B_2 &= T_{2233} - T_{1122} \ B_3 &= T_{1313} - rac{1}{2}(T_{1111} - T_{1122}) \ B_4 &= T_{1122} \ B_5 &= rac{1}{2}(T_{1111} - T_{1122}) \ \end{aligned}$$

## closure approximations

#### closure approximations

- While theoretically any-order orientation tensor is possible, in practice only the second-order tensor is used
- Microscopic measurements do not give enough information for higher-order tensors to be useful
- Software simulations have not implemented the fourth-order tensor
- To predict stiffness, we need the fourth-order tensor
- Closure Approximations are a way to approximate the fourth-order tensor from the second-order tensor

#### linear closure approximate

• For 3D orientations, the linear approximation is given by

$$a_4^l = -rac{1}{35}(\delta_{ij}\delta_{kl} + \delta_{ik}\delta_{jl} + \delta_{il}\delta_{jk}) +$$

$$rac{1}{7}(a_{ij}\delta_{kl}+a_{ik}\delta_{jl}+a_{il}\delta_{jk}+a_{kl}\delta_{ij}+a_{jl}\delta_{ik}+a_{jk}\delta_{il})$$

• For planar orientations we simply replace the two coefficients with  $-\frac{1}{24}$  and  $\frac{1}{6}$ 

#### quadratic closure

- We can also use a quadratic closure method  $a_4^q = a_{ij}a_{kl}$
- If the fibers are randomly aligned, the linear closure will give the exact result
- If the fibers are perfectly oriented, the quadratic closure will give the exact result

#### hybrid closure

- Advani proposed a hybrid closure to take advantage of both the linear and quadratic methods
- We will introduce a parameter *f* and use it to combine both linear and quadratic closures

$$a_4^h = (1 - f)a_4^l + fa_4^q$$

- Ideally, we would like *f* to be 1 for perfectly oriented fibers and 0 for random fibers
- Advani proposes  $f = Aa_{ij}a_{ji} B$
- Where A = 3/2 and B = 1/2 for 3D orientations and A = 2 and B = 1 for planar orientation

#### orthotropic fitted closure

- A more recent method that is commonly used is known as the orthotropic fitted closure
- The fourth-order tensor is approximated in the principal direction, then rotated back out if necessary
- In the principal direction, the fourth-order tensor will be orthotropic (represented in 6x6 as)

$$A_4 = egin{bmatrix} A_{11} & A_{12} & A_{13} & 0 & 0 & 0 \ A_{12} & A_{22} & A_{23} & 0 & 0 & 0 \ A_{13} & A_{23} & A_{33} & 0 & 0 & 0 \ 0 & 0 & 0 & A_{44} & 0 & 0 \ 0 & 0 & 0 & 0 & A_{55} & 0 \ 0 & 0 & 0 & 0 & 0 & A_{66} \end{bmatrix}$$

#### orthotropic fitted closure

- The symmetry of the orientation tensor requires that  $A_{66}$  (which is  $a_{1212}$ ) be equal to  $A_{12}$  (which is  $a_{1122}$ ).
- By the same symmetry, we have  $A_{55} = A_{13}$  and  $A_{44} = A_{23}$ .
- We also know that  $a_{ijkk} = a_{ij}$ , which imposes the following equations:

$$A_{11} + A_{66} + A_{55} = a_{11}$$

$$A_{66} + A_{22} + A_{44} = a_{22}$$

$$A_{55} + A_{44} + A_{33} = a_{33}$$

#### orthotropic fitted closure

- This leaves only three independent variables in the fourth-order tensor that need to be found.
- Different authors have proposed different functions to fit these three independent variables
- They are fit to give the best mold simulation predictions, but do not necessarily have any physical application

#### discrete calculations

• To compare with our laminate analogy we can calculate the orientation tensor for discrete orientation states

$$a_{ij} = rac{1}{N} \sum p_i p_j$$

for second-order tensors and

$$a_{ijkl} = rac{1}{N} \sum p_i p_j p_k p_l$$

#### example

- Compare Mori-Tanaka stiffness predictions for direct calculation and orientation averaging
- Compare  $[0/90]_S$ ,  $[\pm 45]_S$ , and  $[0/\pm 45/90]_S$
- <u>link</u>
- Also compare the results with a closure approximation of the fourthorder tensor

## variational calculus

#### differential and variational statements

- A differential statement includes a set of governing differential equations established inside a domain and a set of boundary conditions to be satisfied along the boundaries
- A variational statement is to find stationary conditions for an integral with unknown functions in the integrand
- Variational statements are advantageous in the following aspects
  - Clear physical meaning, invariant to coordinate system
  - Can provide more realistic descriptions than differential statements (concentrated loads)
  - More easily suited to solving problems numerically or approximately
  - Can be more systematic and consistent than building a set of differential equations

#### stationary problems

- If the function  $F(u_1)$  is defined on a domain, then at  $\frac{dF}{du_1} = 1$  it is considered to be stationary
- This stationary point could be a minimum, maximum, or saddle point
- We use the second derivative to determine which of these it is: >0 for a minimum, <0 for a maximum and =0 for a saddle point
- For a function of *n* variables,  $F(u_n)$  the stationary points are

$$\frac{\partial F}{\partial u_i} = 0$$

for all values of i

and to determine the type of stationary point we use

$$\sum_{i,j=1,n} rac{\partial^2 F}{\partial u_i \partial u_j}$$

## lagrange multipliers

• Let us now consider a function of several variables, but the variables are subject to a constraint

$$f(u_1, u_2, ...) = 0$$

- Algebraically, we could use each provided constraint equation to reduce the number of variables
- For large problems, it can be cumbersome or impossible to eliminate some variables
- The Lagrange Multiplier method is an alternative, systematic approach

#### lagrange multiplier

For a constrained problem at a stationary point we will have

$$dF = rac{\partial F}{\partial u_1} du_1 + \ldots + rac{\partial F}{\partial u_n} du_n = 0$$

• The relationship between  $du_i$  can be found by differentiating the constraint

$$df=rac{\partial f}{\partial u_1}du_1+\ldots+rac{\partial f}{\partial u_n}du_n\,=0$$

• We can combine these two equations using a Lagrange Multiplier

$$\left[rac{\partial F}{\partial u_1}du_1\!+\!\ldots\!+\!rac{\partial F}{\partial u_n}du_n\,+\lambda\left[rac{\partial f}{\partial u_1}du_1\!+\!\ldots\!+\!rac{\partial f}{\partial u_n}du_n
ight]$$

• We can re-group terms as

$$\sum_{i=1}^n \left[rac{\partial F}{\partial u_i} + \lambda rac{\partial f}{\partial u_i}
ight] du_i = 0$$

#### lagrange multiplier

- The Lagrange Multiplier,  $\lambda$  is an arbitrary function of  $u_i$
- We can choose the Lagrange Multiplier such that

$$\frac{\partial F}{\partial u_n} + \lambda \frac{\partial f}{\partial u_n} = 0$$

• Which now leaves

$$rac{\partial F}{\partial u_i} + \lambda rac{\partial f}{\partial u_i} = 0 \qquad i = 1, 2, \dots, n-1$$

• We now define a new function  $F^* = F + \lambda f$ 

#### lagrange multiplier

- This converts a constrained problem in n variables to an unconstrained problem in n + 1 variables
- Notice that while the stationary values of  $F^*$  will be the same as the stationary values to F, they will not necessarily correspond
- For example, a minimum in  $F^*$  might be a maximum in F
- This provides a systematic method for solving problems with any number of variables and constraints, and is also well-posed for numeric solutions

## example

- Design a box with given surface area such that the volume is maximized
- The box has no cover along one of the surfaces (open-face box)
- This gives the surface area as A = xy + 2yz + 2xz = C
- worked example