



Vehicle States, Dynamics, and Simulation

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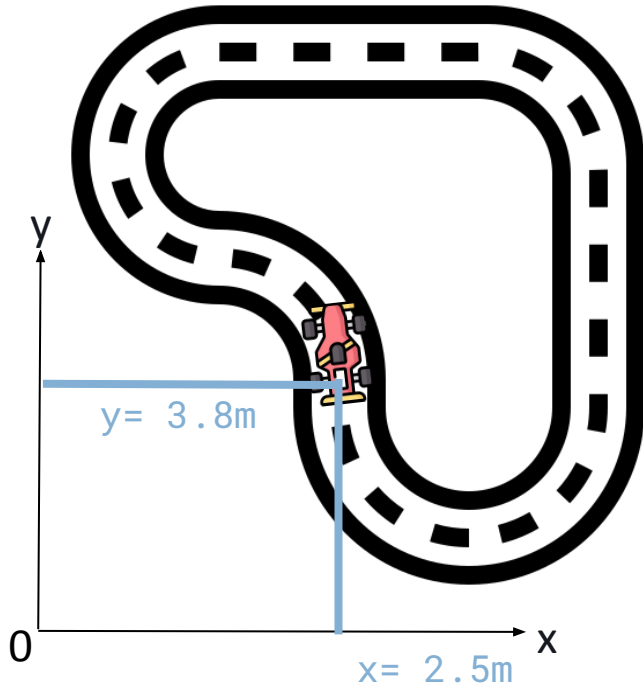
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Overview

- Vehicle States
- Vehicle Dynamics Modelling
- Vehicle Dynamics Simulation

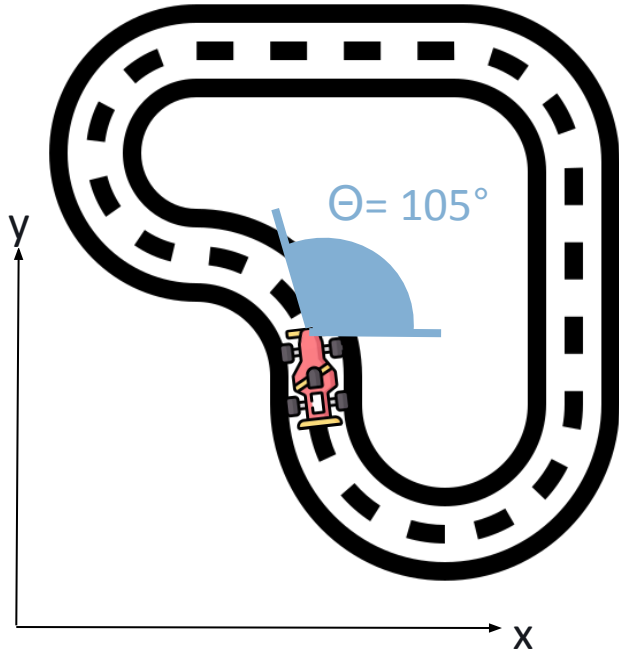
Vehicle States

Position



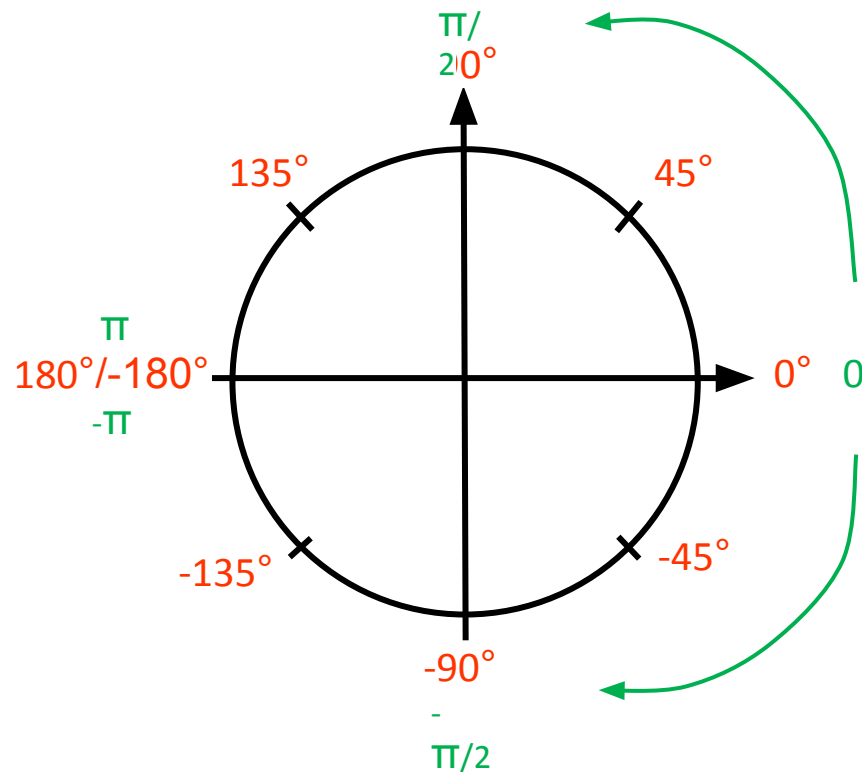
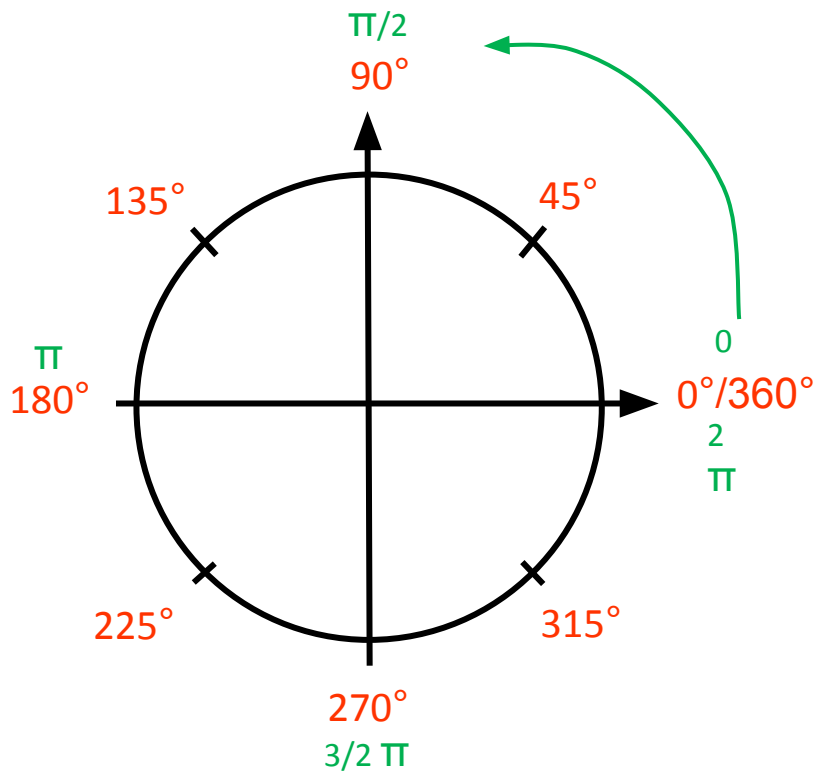
- **Position** defines the translation of the vehicle in some global or local frame
- Respective to the vehicle's center of gravity (CoG), or a pre-defined base frame
- **Normally:** X- and Y- position in **meters**

Heading

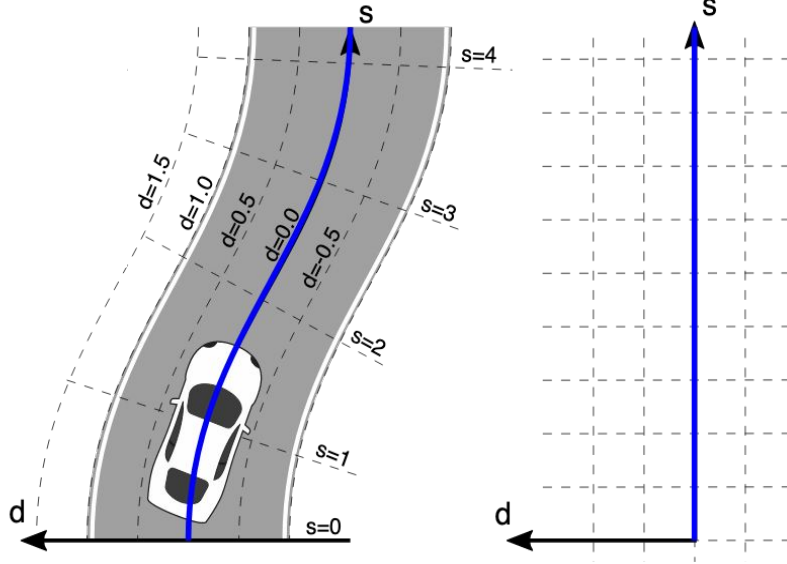


- **Heading** defines the rotation of the vehicle in some local and global frame
- Usually with respect to the x-axis of the coordinate system of current frame
- When represented as a single angle reading, heading can be displayed in ranges from:
 - $-\pi - \pi = -180^\circ$ to 180°
 - $0 - 2\pi = 0^\circ$ to 360°
- Could be represented as RPY, Rotation matrix, Quaternion, etc.
- Only **rotation**

Heading

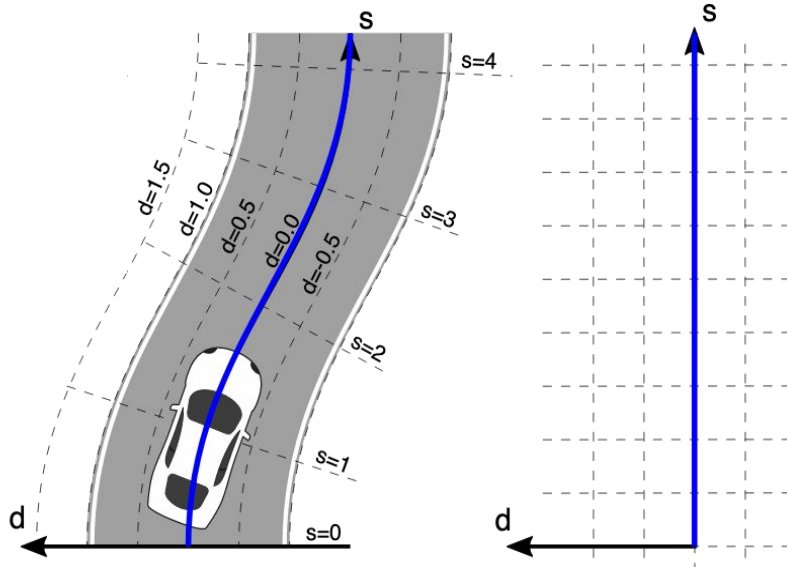


Position in Frenet Frame



- **The Frenet frame** along a curve is a moving coordinate system determined by the tangent line and curvature.
- The frame itself is defined as the coordinate system spanned by a **tangential vector T** and the **normal vector N** , and a **binormal vector B** at any point of the reference line (e.g. centerline of a track).
- (The binormal vector is the cross product of the tangential vector and the normal vector)

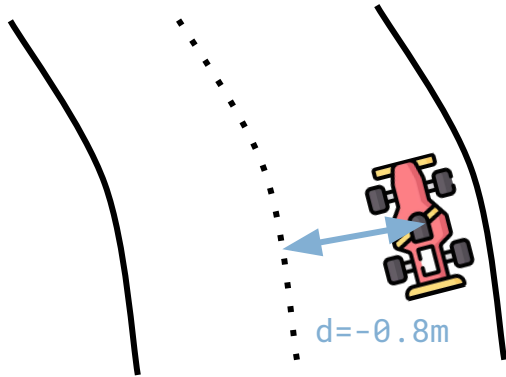
Position in Frenet Frame



- 2 coordinates in 2D.
- The s coordinate represents the run length. **Starts with $s = 0$** at the beginning of the reference path.
- The d coordinate represents lateral position relative to the reference path. **Starts with $d = 0$** for points on the reference path. Measured on the normal vector.
- **d is positive to the left** of the reference path and **negative on the right** of it. (Right hand rule)

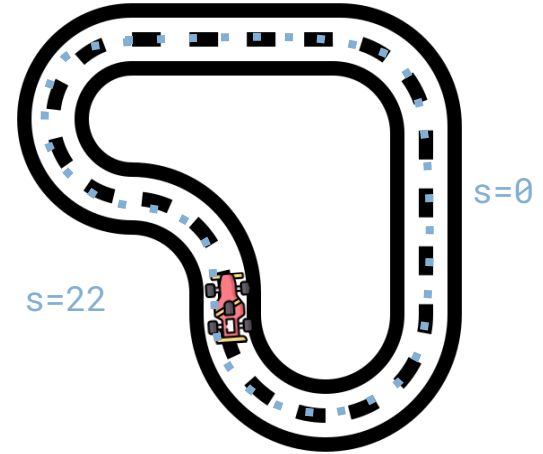
Frenet Frame in Practice

Distance to Centerline



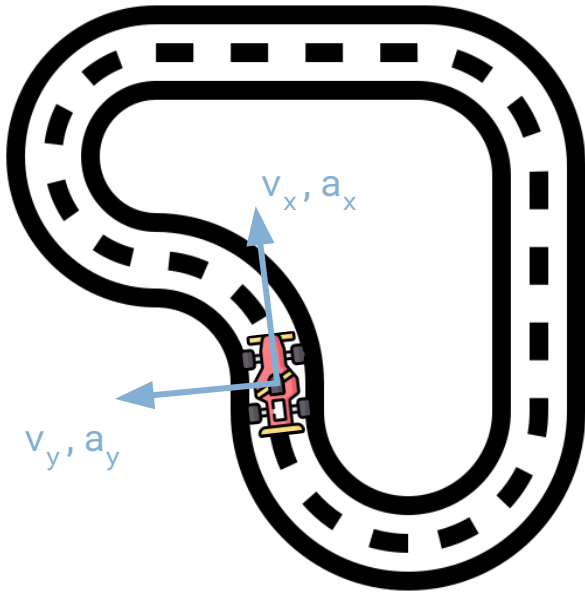
- Displacement (in meters) between the **agent center and the track center**
- The observable maximum displacement occurs when any of the agent's wheels are outside a track border and, depending on the width of the track border

Progress along the track



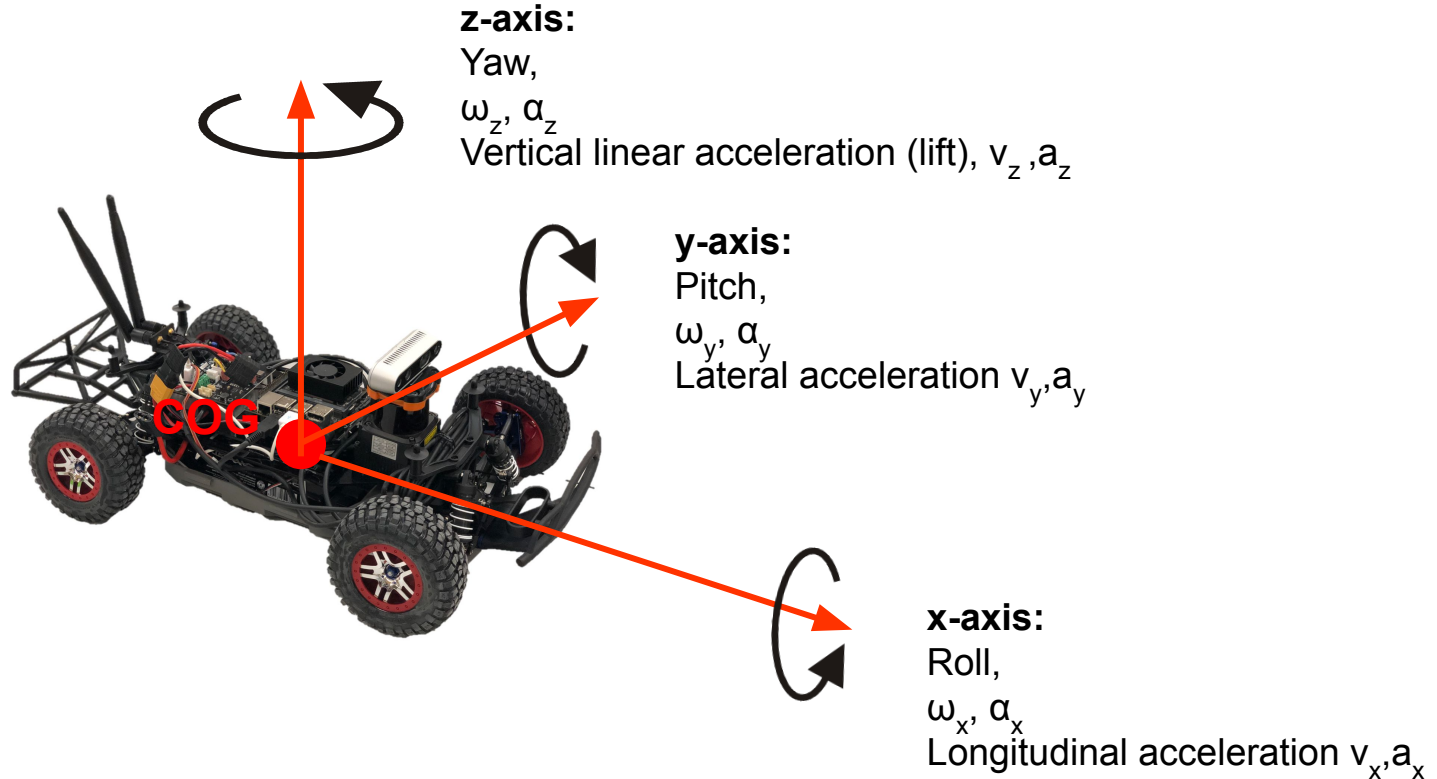
- An **ordered list** of track-dependent milestones along the track (waypoints).
- Each milestone is described by a coordinate of $(x_i ; y_i, \dots)$.
- For a looped track, **the first and last waypoints are the same.**

Linear Velocity and Acceleration

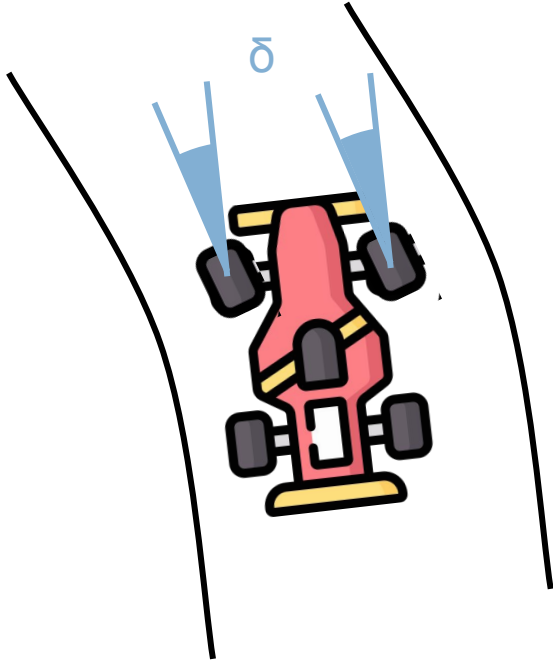


- **Linear velocity and acceleration** are measured in the x- and y-, (and z-) axis in the coordinate system of the vehicle.
- For cars: **longitudinal** (x-axis) and **lateral** (y-axis) velocities and accelerations.
- Can be measure with GPS, IMU, wheel speed sensors, pitot sensors, etc.
- Usually velocity in meters per second: m/s, and acceleration in meters per second squared: m/s^2
- (Right hand rule)

Angular Velocity and Acceleration



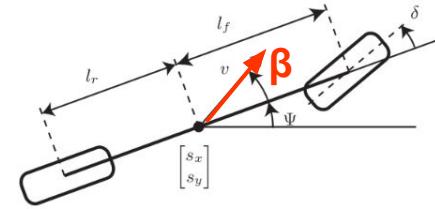
Vehicle State - Steering Angle



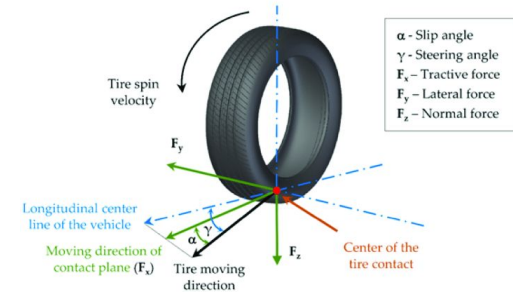
- **Steering angle δ** is the angle formed by the direction the front wheels are pointing at and the vehicle's x-axis.
- Steering angle is the same for both front wheels.
- Usually in radians or degrees.

Different Slip Angles

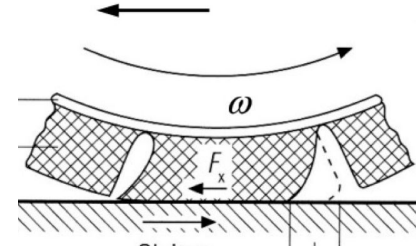
- Sideslip angle β



- Slip Angle α



- Wheelslip ratio s



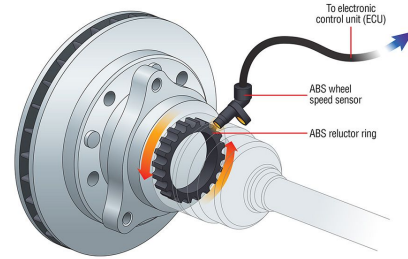
Measuring Vehicle States

- Vehicle states can be derived from different sensor

- GPS (Global Positioning System)



- Odometry: Gyrometer, Wheel Speed Sensor



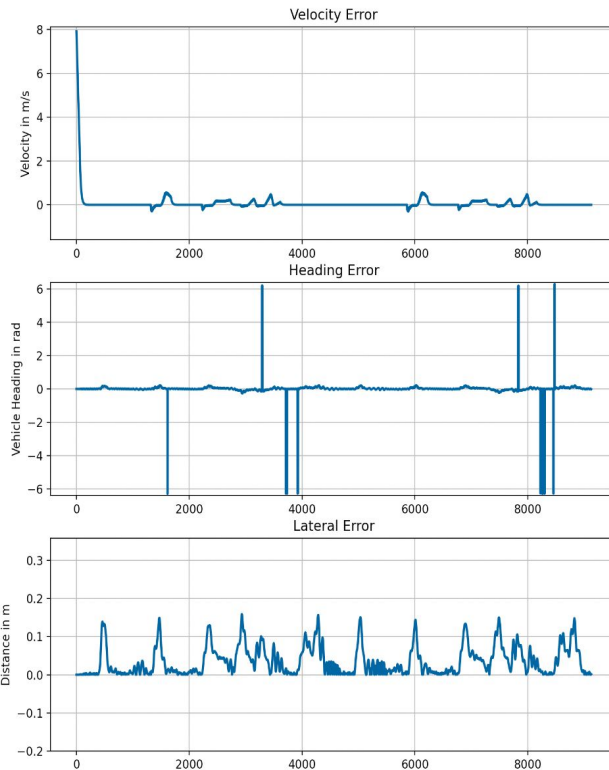
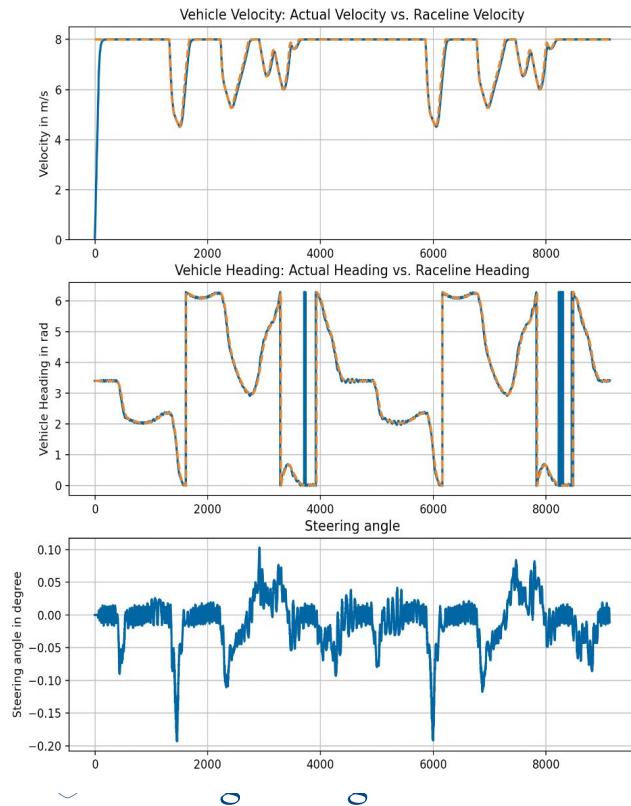
- Camera



- Lidar



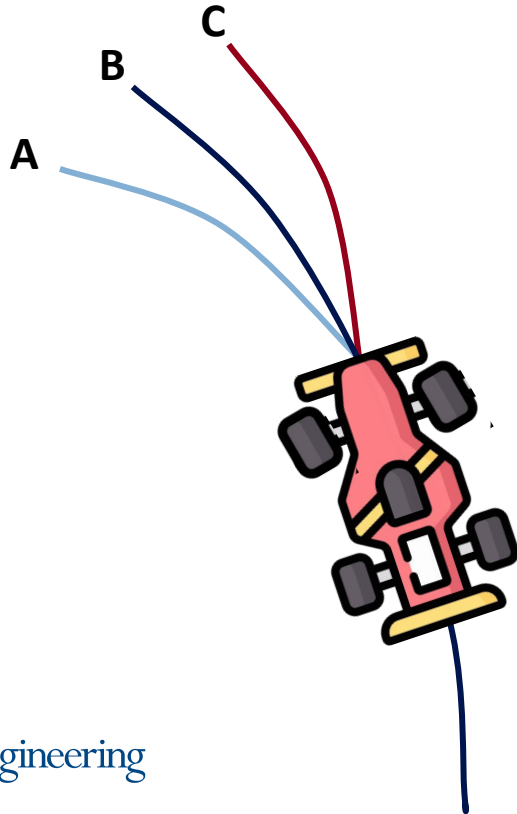
Important - Visualize your States



- Visualize
- Show trends
- Calculate errors
- Learn to Interpret the signals
- Plotjuggler, matplotlib, rqt_plot, etc.

Vehicle Dynamics Modelling

Why Modelling



Which trajectory is the car following?

We can not tell

We need information about:

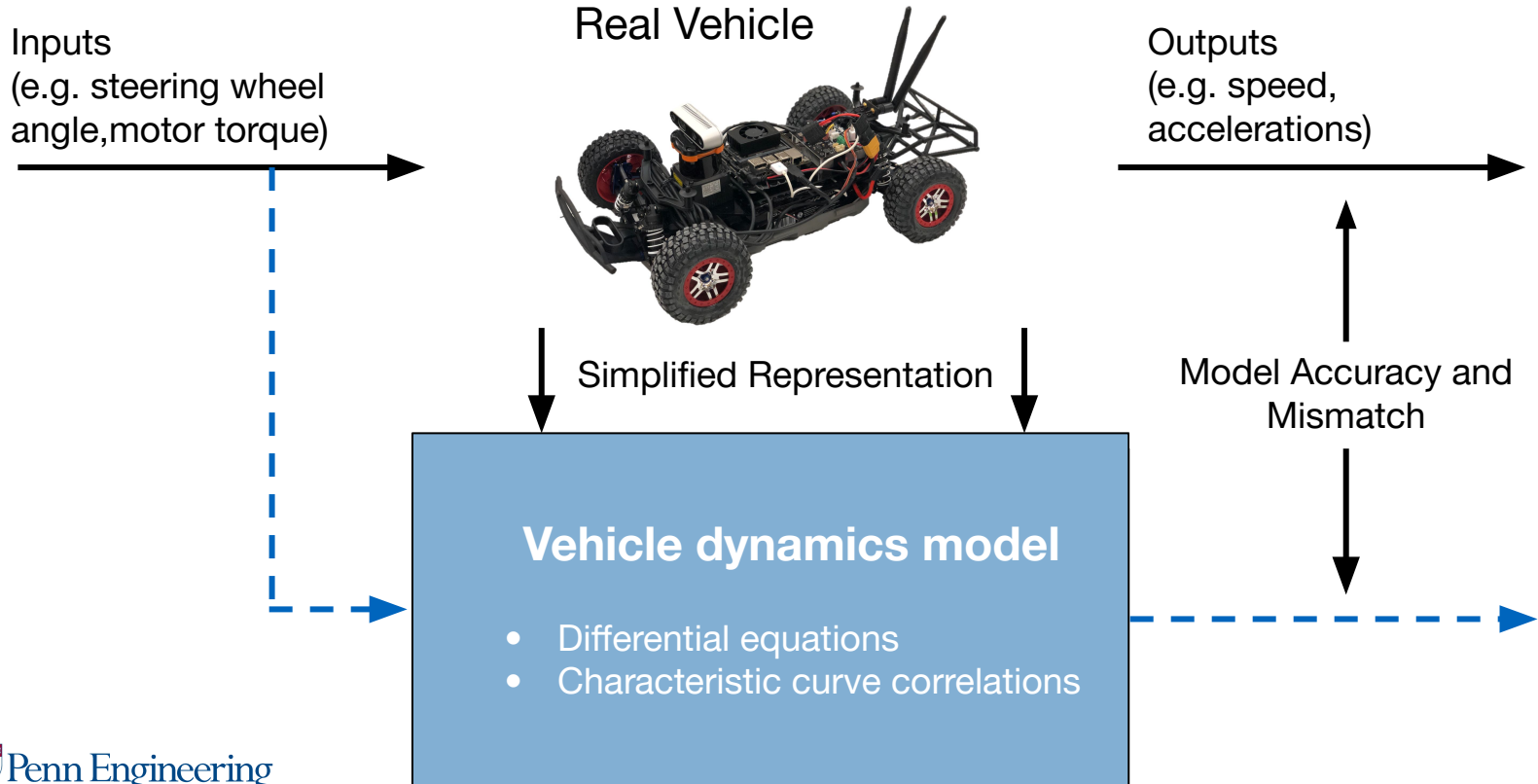
Velocity

Acceleration

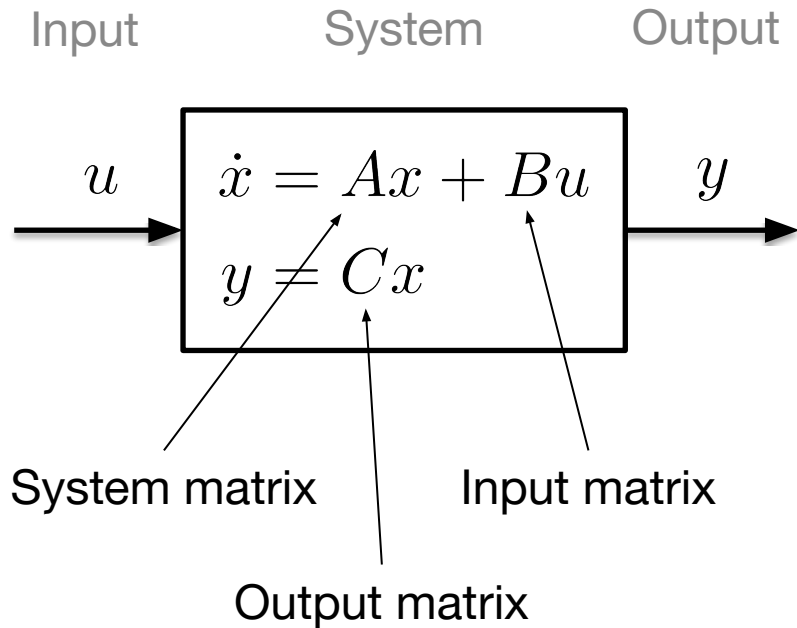
Steering angle

Vehicle: Mass, Length, Width,

Why Modelling?

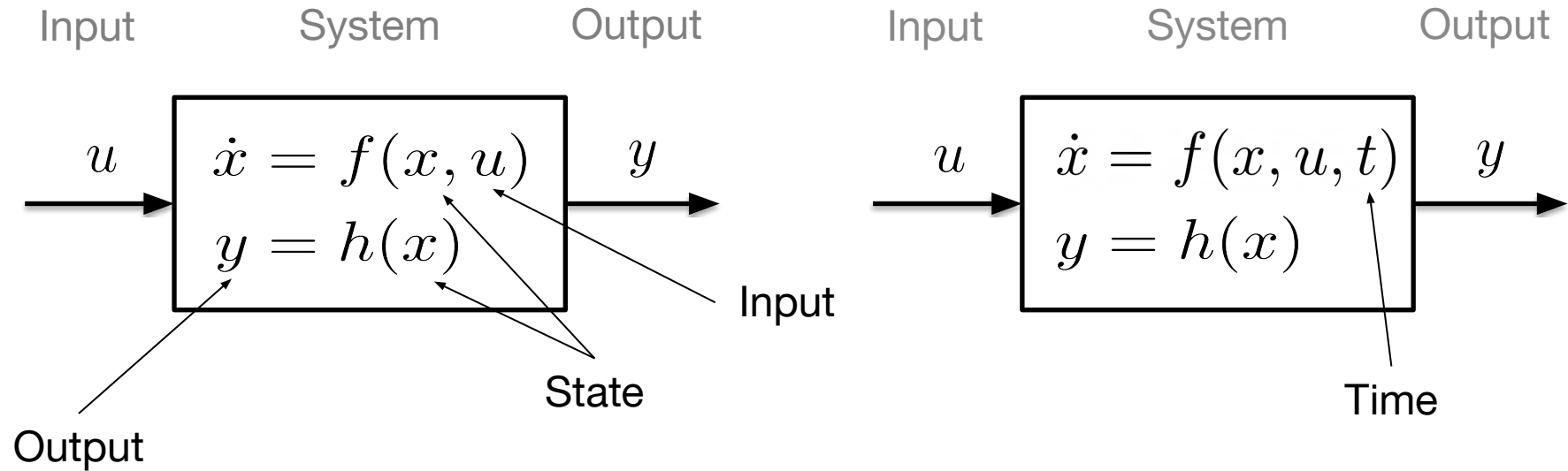


System Dynamics

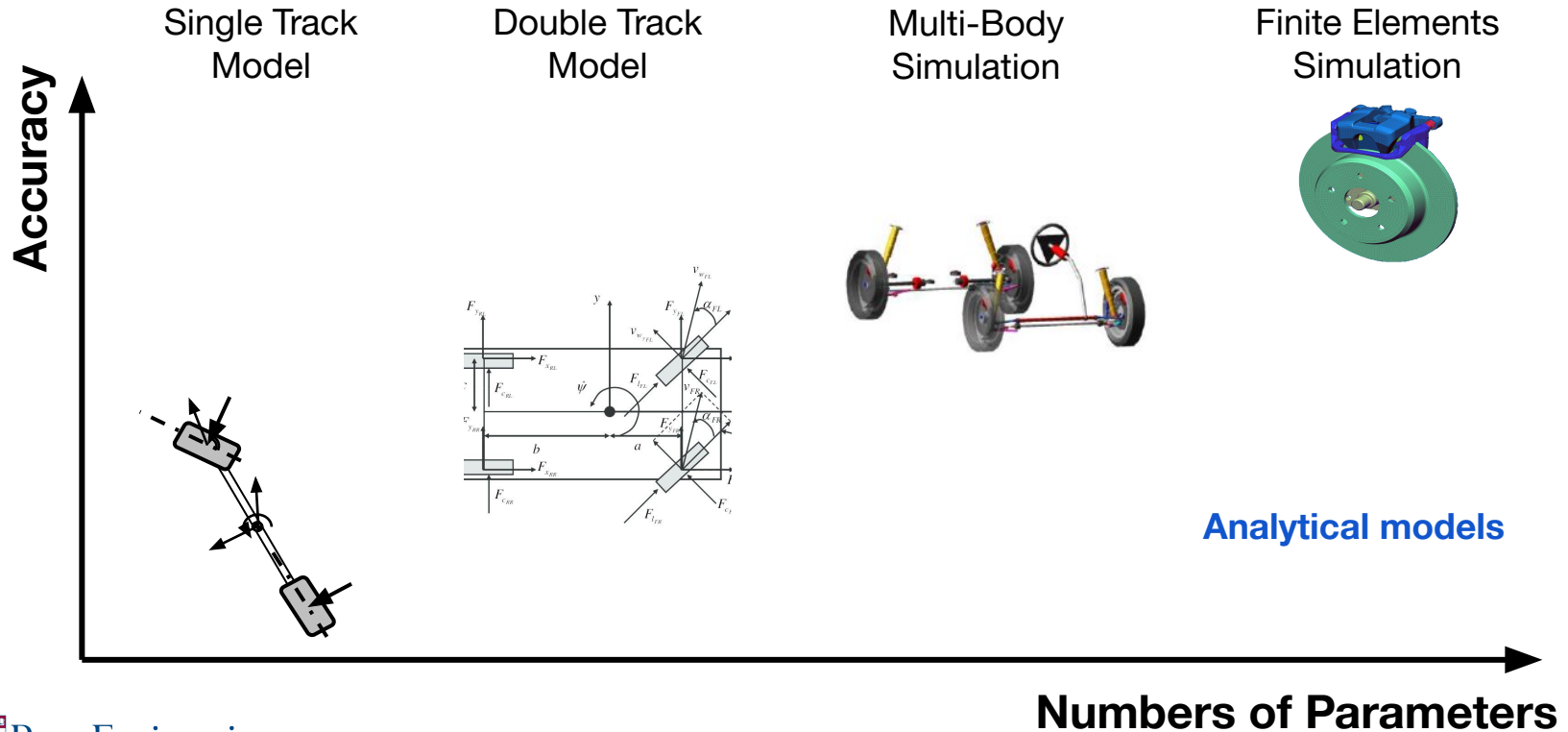


- A dynamic system is a mathematical description of the relation between an *input* u and an *output* y signal. This description is usually given in form of an ODE (Ordinary Differential Equation).
- The system has **states** x . These are variables which allow us to **formulate the system behavior** in form of a set of first-order ODE for each of this variables.
- The standard system description consists of the **system dynamics** and the **output equation**. The former describes the timely behavior of the states as a reaction to the inputs and the initial state. The latter describes the relation between the state and the output.

System Dynamics

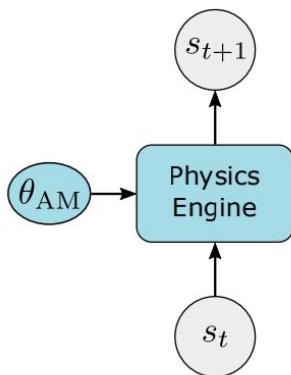


Vehicle Dynamics Model Types

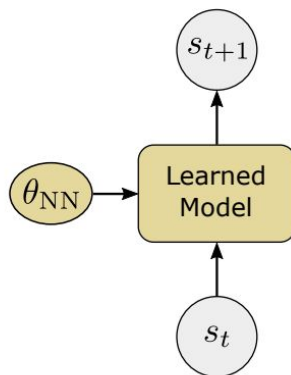


Vehicle Dynamics Model Types

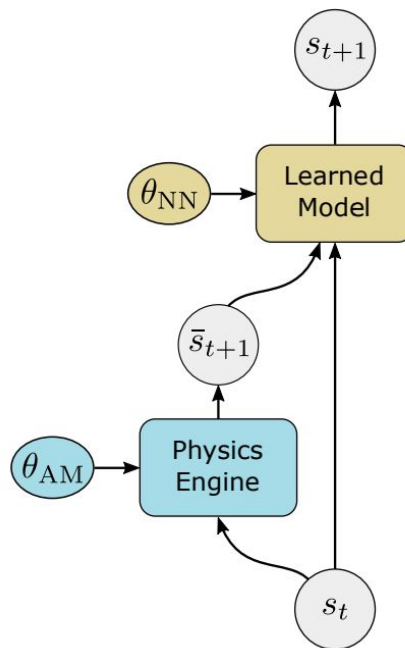
Analytical Model



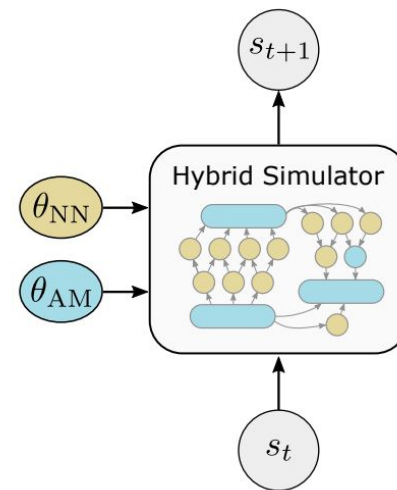
Data-driven Model



Residual Physics



Hybrid

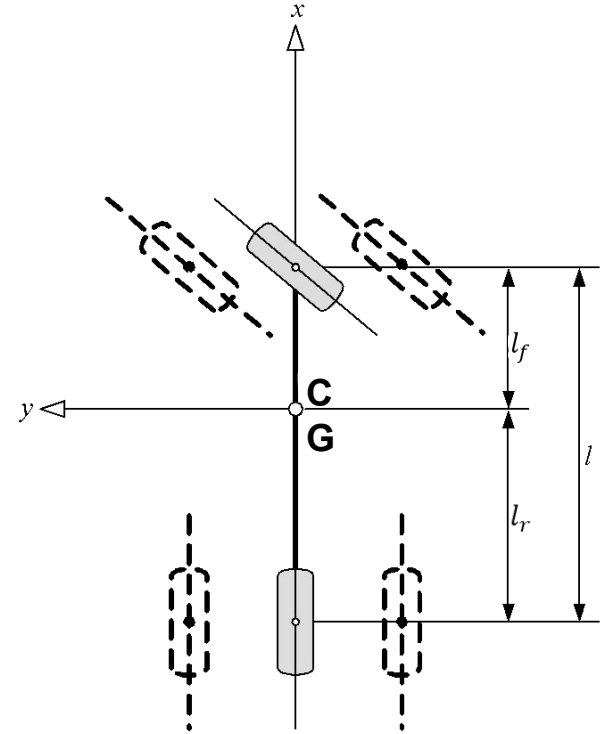


Single Track Models

Single Track Model

Simplifications:

- Wheels of one axle are combined
- Center of gravity is at road level
- No rolling
- No pitching
- No wheel load differences left/right
- No vertical dynamics



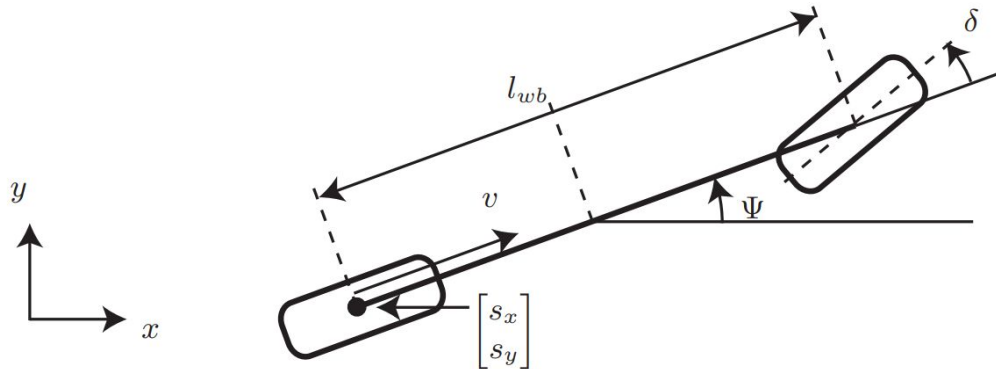
Kinematic Single Track Model

Simplifications:

- No tire dynamics, hence **no lateral forces/accelerations and slip angles**

At slow velocities, especially when cornering slowly, kinematic model is usually accurate enough for simulation.

Ackermann steering modeled around an instantaneous pole.



Kinematic Single Track Model

$$x_1 = s_x, \quad x_2 = s_y, \quad x_3 = \delta, \quad x_4 = v, \quad x_5 = \Psi$$

$$u_1 = v_\delta, \quad u_2 = a_{\text{long}}$$

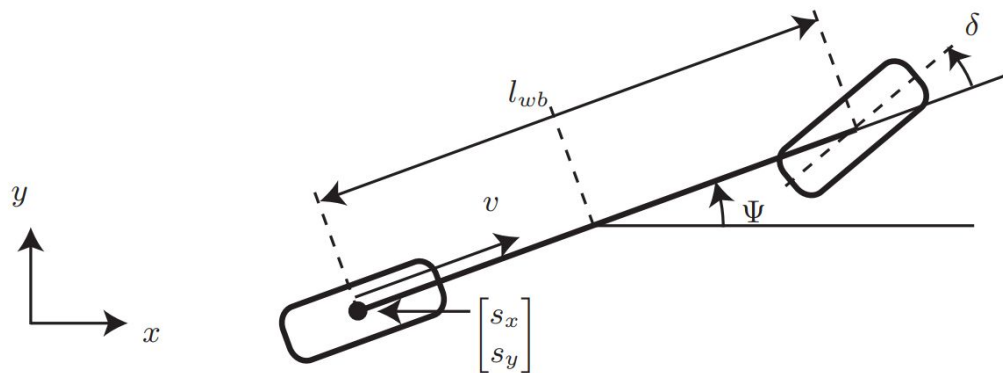
$$\dot{x}_1 = x_4 \cos(x_5)$$

$$\dot{x}_2 = x_4 \sin(x_5)$$

$$\dot{x}_3 = f_{\text{steer}}(x_3, u_1)$$

$$\dot{x}_4 = f_{\text{acc}}(x_4, u_2)$$

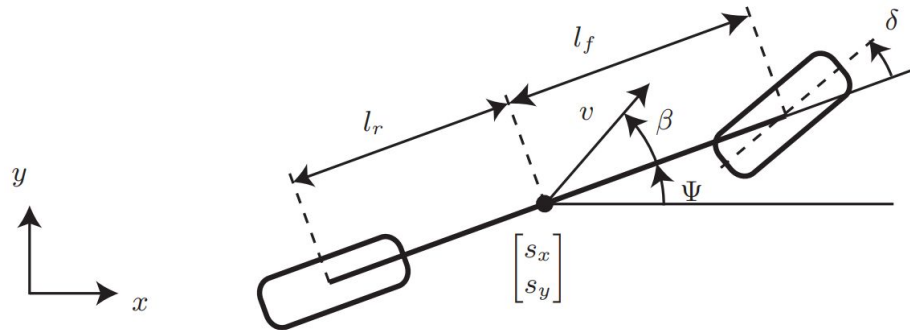
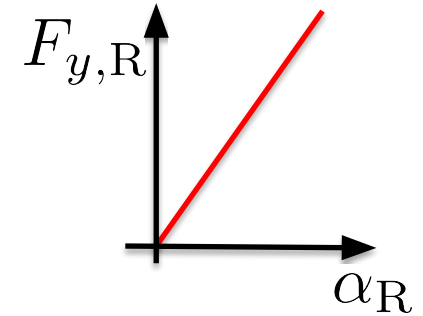
$$\dot{x}_5 = \frac{x_4}{l_{wb}} \tan(x_3)$$



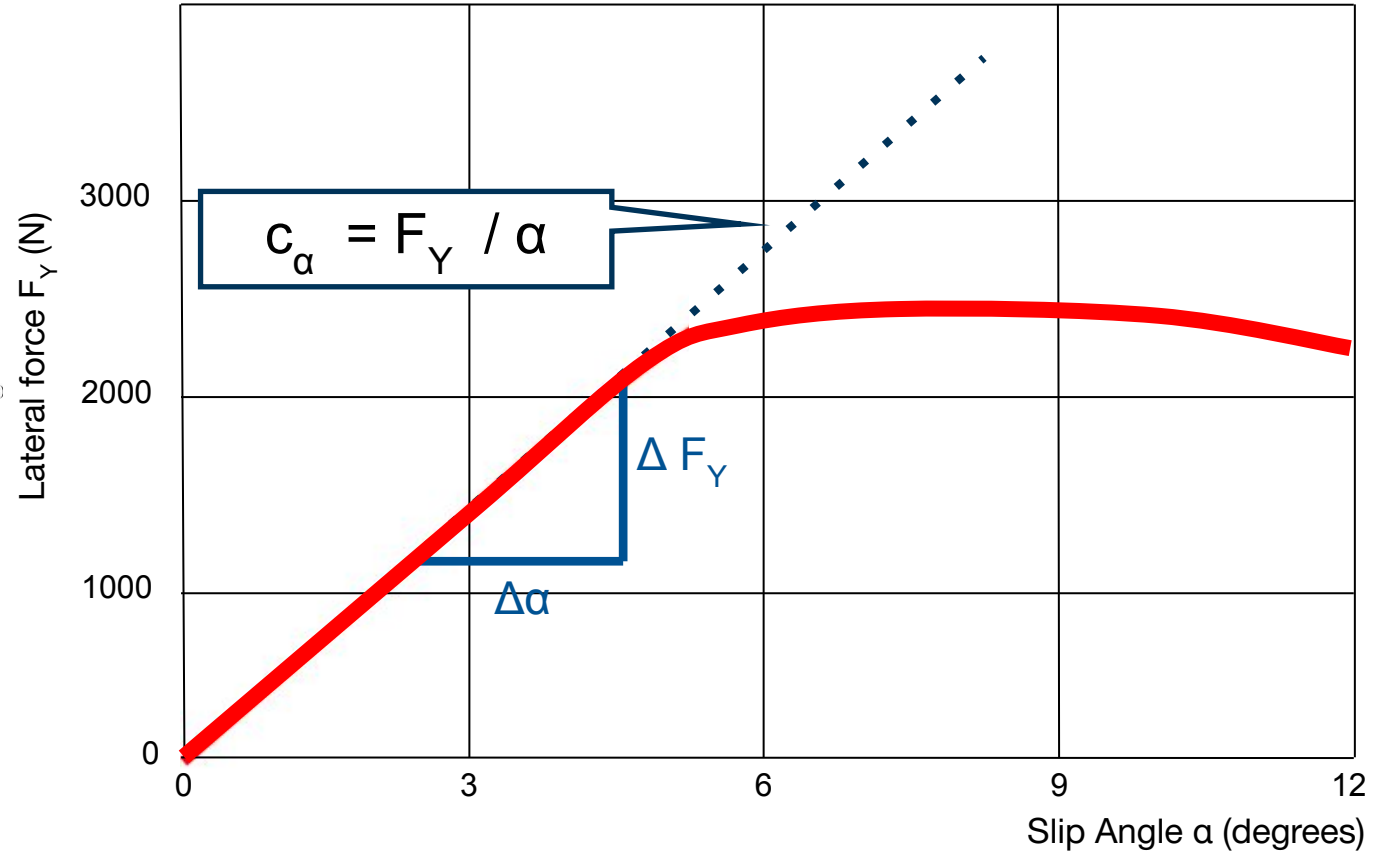
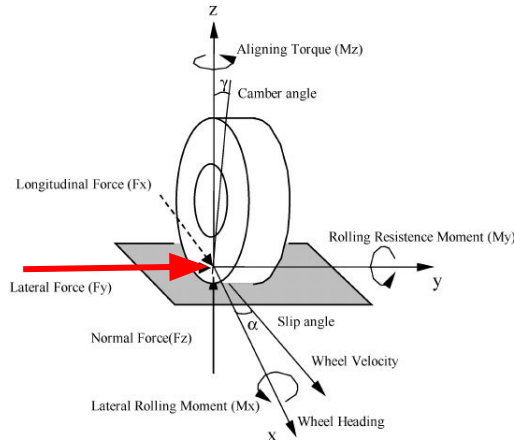
Dynamic Single Track Model - Linear Tire Model

Single Track Model with Linear Tire Model:

- Consider important effects such as understeer or oversteer.
- Introduction of tire forces
 - A tire can apply lateral and longitudinal tire forces
 - A tire can apply more forces if there is a higher friction coefficient μ
 - **Linear** relation between tire force and side slip angle
 - Model the tire dynamics with the **cornering stiffness C** , or the **cornering stiffness coefficient C_s**



Linear Tire Model



Dynamic Single Track Model - Linear Tire Model

$$x_1 = s_x, \quad x_2 = s_y, \quad x_3 = \delta, \quad x_4 = v, \quad x_5 = \Psi, \quad x_6 = \dot{\Psi}, \quad x_7 = \beta$$

$$u_1 = v\delta, \quad u_2 = a_{\text{long}}$$

$$\dot{x}_1 = x_4 \cos(x_5 + x_7)$$

$$\dot{x}_2 = x_4 \sin(x_5 + x_7)$$

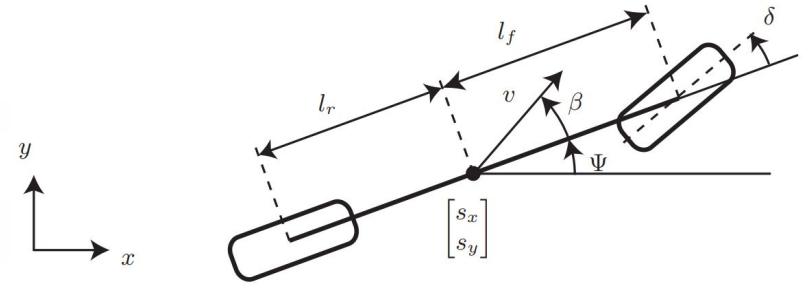
$$\dot{x}_3 = f_{\text{steer}}(x_3, u_1)$$

$$\dot{x}_4 = f_{\text{acc}}(x_4, u_2)$$

$$\dot{x}_5 = x_6,$$

$$\dot{x}_6 = \frac{\mu m}{I_z (l_r + l_f)} (l_f C_{S,f} (gl_r - u_2 h_{cg}) x_3 + (l_r C_{S,r} (gl_f + u_2 h_{cg}) - l_f C_{S,f} (gl_r - u_2 h_{cg})) x_7 \\ - (l_f^2 C_{S,f} (gl_r - u_2 h_{cg}) + l_r^2 C_{S,r} (gl_f + u_2 h_{cg})) \frac{x_6}{x_4})$$

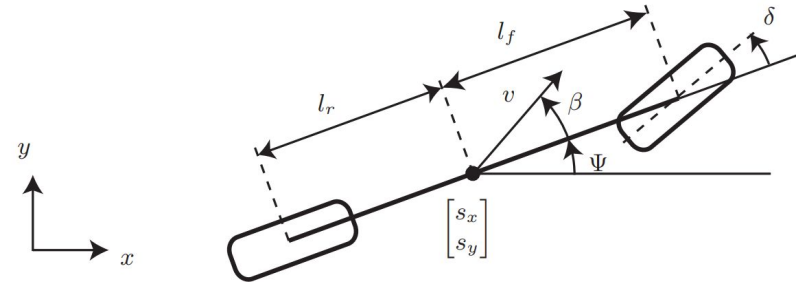
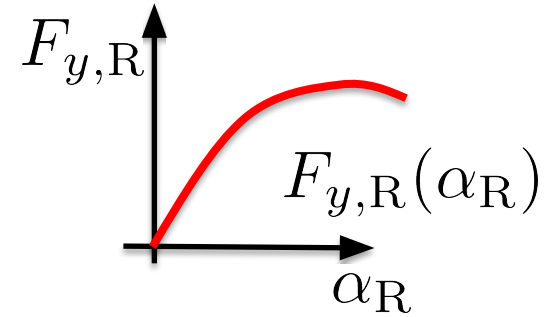
$$\dot{x}_7 = \frac{\mu}{x_4 (l_r + l_f)} (C_{S,f} (gl_r - u_2 h_{cg}) x_3 - (C_{S,r} (gl_f + u_2 h_{cg}) + C_{S,f} (gl_r - u_2 h_{cg})) x_7 \\ + (C_{S,r} (gl_f + u_2 h_{cg}) l_r - C_{S,f} (gl_r - u_2 h_{cg}) l_f) \frac{x_6}{x_4}) - x_6$$



Dynamic Single Track Model - Nonlinear Tire Model

Single Track Model with Nonlinear Tire Model:

- Based on the ST with linear tire model
- Extension of tire forces:
 - No small angle approximations
 - Considers longitudinal tire forces and longitudinal slip on the front and rear wheels
 - Nonlinear relation between tire force and side slip angle
 - Model the tire dynamics with the special models:
 - Pacejka Magic Formula
 - Fiala
 - ...



Dynamic Single Track Model - Nonlinear Tire Model

$$x_1 = s_x, \quad x_2 = s_y, \quad x_3 = \delta, \quad x_4 = v, \quad x_5 = \Psi, \quad x_6 = \dot{\Psi}, \quad x_7 = \beta, \quad x_8 = \omega_f, \quad x_9 = \omega_r$$

$$u_1 = v\delta, \quad u_2 = a_{\text{long}}$$

$$\dot{x}_1 = x_4 \cos(x_7 + x_5)$$

$$\dot{x}_2 = x_4 \sin(x_7 + x_5)$$

$$\dot{x}_3 = f_{\text{steer}}(x_3, u_1)$$

$$\dot{x}_4 = \frac{1}{m} (-F_{s,f} \sin(x_3 - x_7) + F_{s,r} \sin x_7 + F_{l,r} \cos(x_7) + F_{l,f} \cos(x_3 - x_7))$$

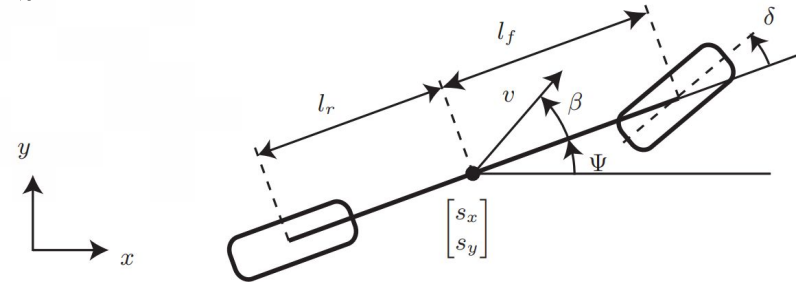
$$\dot{x}_5 = x_6$$

$$\ddot{x}_6 = \frac{1}{I_z} (F_{s,f} \cos(x_3) l_f - F_{s,r} l_r + F_{l,f} \sin(x_3) l_f)$$

$$\dot{x}_7 = -\dot{x}_5 + \frac{1}{m x_4} (F_{s,f} \cos(x_3 - x_7) + F_{s,r} \cos(x_7) - F_{l,r} \sin(x_7) + F_{l,f} \sin(x_3 - x_7))$$

$$\dot{x}_8 = \frac{1}{I_{y,w}} (-R_w F_{l,f} + T_{s,b} T_B + T_{s,e} T_E)$$

$$\dot{x}_9 = \frac{1}{I_{y,w}} (-R_w F_{l,r} + (1 - T_{s,b}) T_B + (1 - T_{s,e}) T_E)$$



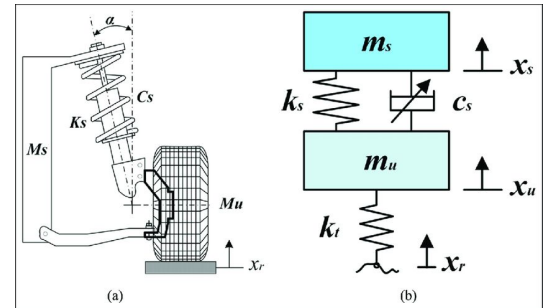
Multi-body Model

- Each wheel is modelled individually with sprung and unsprung mass
- Each wheel has individual wheel load
- Each wheel has individual lat./long forces
- Center of gravity is at certain height
 - Rolling
 - Pitching
 - Wheel load differences left/right
 - Axle load differences front/rear
- Vertical dynamics
- Tyre modelling with different nonlinear models, similar to nonlinear ST

Example:

https://gitlab.lrz.de/tum-cps/commonroad-vehicle-models/-/blob/master/vehicleModels_commonRoad.pdf

Section 9



Vehicle Dynamics - Use Cases

- **Vehicle Dynamics Simulation** - Provide the correct behavior for your vehicle in Simulation
- **State Estimation:** Calculate how your vehicle has moved
- **Behavior Prediction:** Calculate how other vehicles will move/behave
- **Model-Based Algorithms:**
 - Model Predictive Control (MPC)
 - Model based Reinforcement Learning

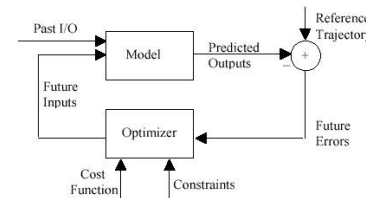
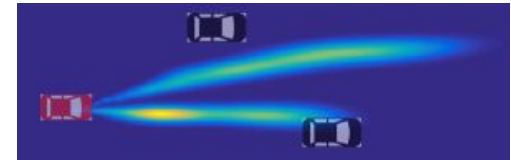
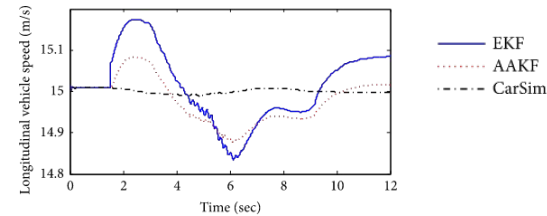
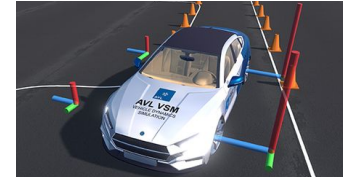
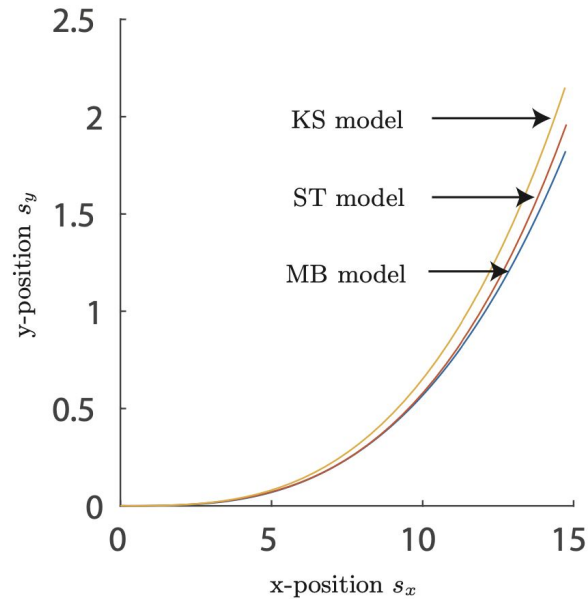


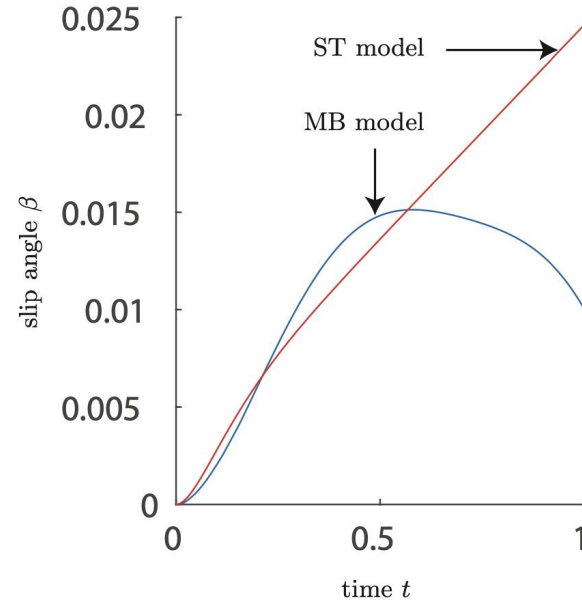
Figure 1: Basic Structure of MPC



Example: Cornering

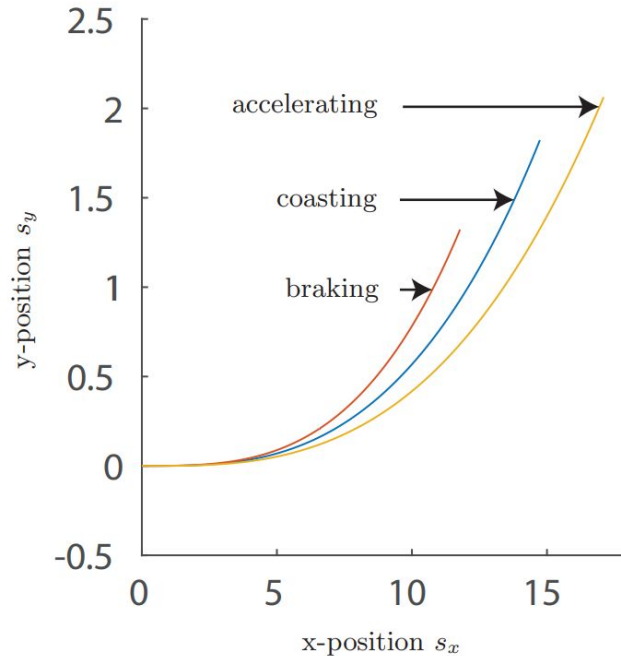


(a) Path of center of gravity.

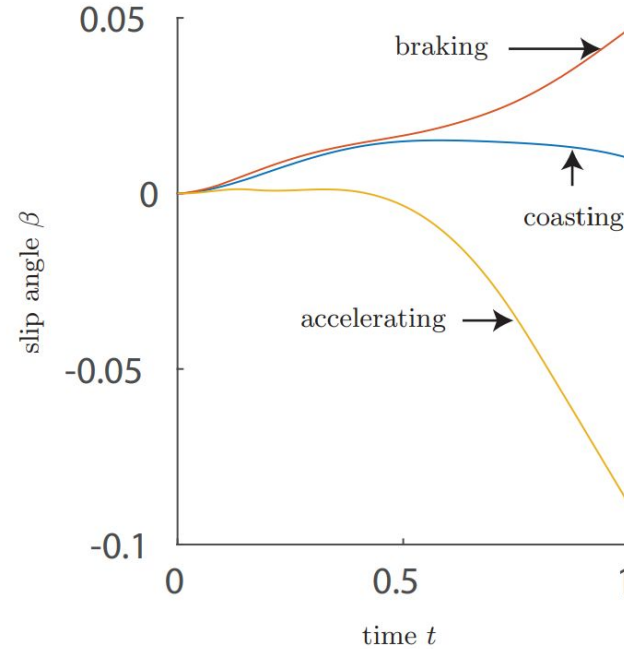


(b) Slip angle.

Example: Over and Under Steering

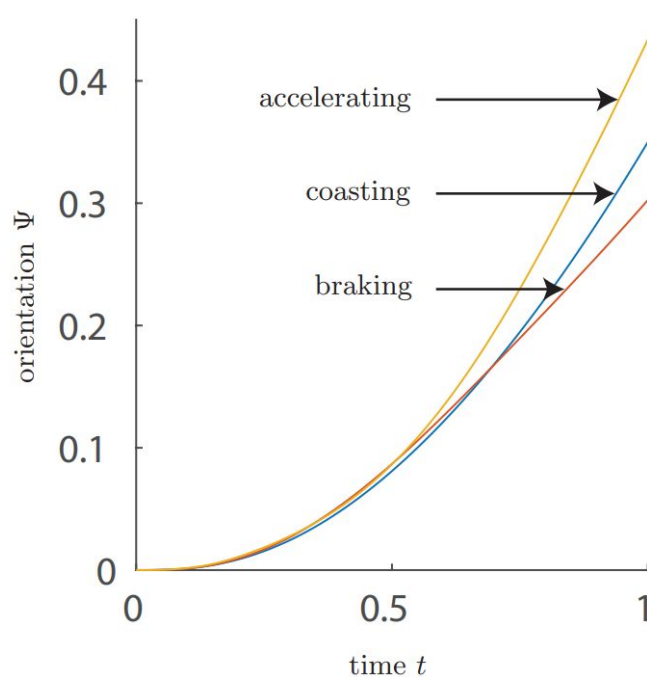


(a) Path of center of gravity.

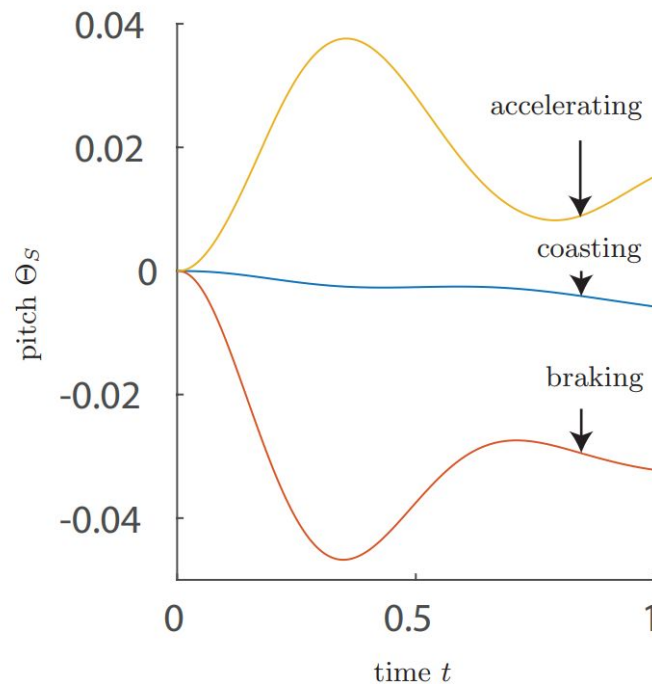


(b) Slip angle.

Example: Over and Under Steering



(c) Orientation.



(d) Pitch.

Questions?