

Validation of the blocks for flat collector with measured data for power tests (EN 12975)

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Version of Model, Carnot, Matlab and Operation system

coll12975.c (V 4.1.1), Carnot 4.7, Matlab R2010b, Windows XP

unicol.c (V 4.1), Carnot 4.7, Matlab R2010b, Windows XP

unicol_2xN.c (V 4.1), internal, Matlab R2010b, Windows XP

Complete path of the block in the Carnot Library

Carnot/heat_source/flat_plate_collector_EN12975

Carnot/heat_source/collector_flat_plate

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1 Data used for validation

I used data from ISFH (Institut für Solarenergieforschung GmbH, Hameln/Emmerthal). These data are been stored during the EN 12975 test of the Vitosol 200F SV2B collector from Viessmann. They come from a test bench.

- Time step : 30 seconds
- Duration : 8 hours

2 Description of the model

2.1 Block

2.1.1 Flat_plate_collector_EN12975

The block is built around the s-function coll12975.c which implements a model of a flat plate collector that includes thermal capacity of the collector and the incidence angle modifier.

The energy-balance for the collector is a differential equation:

$$\begin{aligned} \text{mdot cp (Tout-Tin) / A} = & F'(\text{TauAlfa}) \text{ Kdir Idir} \\ & + F'(\text{TauAlfa}) \text{ Kdfu Idfu} - c6 \text{ v_wind lglb} \\ & - c1 (Tm-Tamb) - c2 (Tm-Tamb)^2 \\ & - c3 \text{ v_wind (Tm-Ta)} \\ & + c4 (\text{ELongwave} - \text{sigmaSB}*(Tamb+273.15)^4) \\ & - c5 \text{ dTm/dt} \end{aligned}$$

with the incidence angle modifier Kdir: $Kdir = 1 - b0 * (1/\cos(\text{teta}) - 1)$

and.

- bo: constant for the calculation of the incident angle modifier
- c1: heat loss coefficient at $(Tm - Ta)=0$ (W/(m².K))
- c2: temperature dependence of the heat loss coefficient (W/(m².K²))
- c3: wind speed dependence of the heat loss coefficient (J/(m³.K))
- c4: sky temperature dependence of the heat loss coefficient (W/m².K))
- c5: effective thermal capacity (J/(m².K))
- c6: wind dependence in the zero loss efficiency (s/m)
- F': collector efficiency factor

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- TauAlfa: effective transmittance-absorptance product for direct solar radiation at normal incidence
- teta: incidence angle of the direct radiation on the collector (radian)
- Tm: temperature of the collector node (Celsius degrees)
- Tamb: ambient temperature (Celsius degrees)
- v_wind: wind velocity (m/s)
- ELongwave: longwave irradiance with wave length > 3000 nm (set at -100) (W/m²)
- Iglb: global solar radiation (W/m²)
- Idir: direct solar radiation (W/m²)
- Idfu: diffuse solar radiation (W/m²)
- sigmaSB: Stefan-Boltzmann constant = $5.67 \times 10^{-8} \text{ W/(m}^2 \cdot \text{K}^4)$

The blocks also contains a weather inclined block which transforms the weather data on a ground surface to data for the collector inclined surface. A other block performs the pressure drop calculus.

Warning : In this case, weather data do not need weather inclined blocks, so I removed it.

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Function Block Parameters: flat_plate_collector_EN12975

collector (mask)
Collector model according to EN 12975 standard.
Energy balance:

$$\dot{m} c_p (T_{out} - T_{in}) / A = F'(\tau_{\alpha}) K_{dir} I_{dir} + F'(\tau_{\alpha}) K_{dfu} I_{dfu} - c_1 (T_m - T_{amb}) - c_2 (T_m - T_{amb})^2 - c_3 v_{wind} (T_m - T_a) - c_6 v_{wind} I_{glb} + c_4 (E_{longwave} - \sigma_{SB} (T_{amb} + 273.15)^4) - c_5 dT_m / dt$$

Often in test reports the optical efficiency or conversion factor η_0 is given.
The correlation for $F'(\tau_{\alpha})$ is:

$$\eta_0 = F'(\tau_{\alpha}) * (K_{dir}(15^\circ) * 0.85 + K_{dfu} * 0.15)$$

or
$$F'(\tau_{\alpha}) = \eta_0 / (K_{dir}(15^\circ) * 0.85 + K_{dfu} * 0.15)$$

where $K_{dir} = 1 - b_0 * (1 / \cos(\theta) - 1)$
with $\theta = 60^\circ$

Parameters

A : collector surface [m²]
2.327

$F'(\tau_{\alpha})$: collector efficiency factor x effective transmittance-absorptance product for direct solar radiation at normal incidence
0.954*0.916*0.95

c1 : heat loss coefficient at $(T_m - T_a) = 0$ [W/(m²K)]
4.07

c2: temperature dependence of the heat loss coefficient [W/(m²K)²]
0.016

c3 : wind speed dependence of the heat loss coefficient [J/(m³K)]
0

c4 : sky temperature dependence of the heat loss coefficient [W/(m²K)]
0

c5 : effective thermal capacity [J/(m²K)]
4600

c6 : wind dependence in the zero loss efficiency [s/m]
0

b0 : constant for the calculation of the incident angle modifier (direct radiation)
0.23

Kd : Incidence angle modifier diffuse radiation
0.83

length between inlet and outlet [m]
21.881

initial temperature [degree centigrade]
20

OK Cancel Help Apply

Figure 1: Parameters for the validation of the flat_plate_collector_EN12975 block

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2.1.2 Collector_flat_plate

The block is built around the s-function unicol.c which implements a model of a flat plate collector that includes thermal capacity of the collector and the incidence angle modifier.

The collector is divided into "NODES" nodes. The energy-balance for every node is a differential equation:

$$\begin{aligned} c_{col} * dT/dt = & ULIN * (T_{amb} - T_{node}) + UQUA * (T_{amb} - T_{node})^2 \\ & + USKY * (T_{node} - T_{sky}) + U_{wind} * v_{wind} * (T_{amb} - T_{node}) \\ & + \dot{m} * c_p / A_{coll} * (T_{lastnode} - T_{node}) + \dot{q}_{solar} \end{aligned}$$

where:

- c_p : heat capacity of fluid (J/(kg.K))
- c_{col} : heat capacity of collector per surface (J/(m².K))
- \dot{m} : mass flow rate (kg/s)
- \dot{q}_{solar} : power input per surface from sun (W/m²)
- T : temperature (K)
- $ULIN$: linear heat loss coefficient (W/(m².K))
- $UQUA$: quadratic heat loss coefficient (W/(m.K)²)
- $USKY$: sky loss coefficient
- U_{wind} : wind speed dependant heat losses (W/((m/s).m².K))

The blocks also contains a weather inclined block which transforms the weather data on a ground surface to data for the collector inclined surface. An other block simulates the optics of a single glazing extra white glass and a last block performs the pressure drop calculus.

Warning : In this case, weather data do not need weather inclined blocks, so I removed it.

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Function Block Parameters: collector_flat_plate

collector (mask)
Model for tested collectors. Enter test results in this mask.
Look under mask to change the incidence angle modifier and the radiation calculation method.

Parameters

collector surface (aperture) in [m²]
2.327

optical efficiency [-]
0.782

linear heat loss coefficient [W/(m²*K)]
4.07

quadratic heat loss coefficient [W/(m²*K)²]
0.016

radiative loss coefficient [W/(m²*K)]
0

wind dependent heat loss [W/(m²*K*m/s)]
0

length between inlet and outlet [m]
21.881

effective heat capacity of collector [J/(m²*K)]
4600

initial temperature [degree centigrade]
20

number of nodes
5

OK Cancel Help Apply

Figure 2: Parameters for the validation of the collector_flat_plate block

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2.1.3 2xN_collector_flat_plate

The block is built around the s-function unicol_2xN.c which implements a model of a flat plate collector that includes thermal capacity of the collector and the incidence angle modifier.

The collector is divided in two parts :

- the absorber
- the fluid

Each part is divided into "NODES" nodes.

The energy-balance for every node in the absorber is a differential equation:

$$\begin{aligned} c_{col} * dT_a/dt = & ULIN * (T_{amb} - T_{a,node}) \\ & + UQUA * (T_{amb} - T_{a,node})^2 \\ & + USKY * (T_{a,node} - T_{sky}) \\ & + U_{wind} * v_{wind} * (T_{amb} - T_{a,node}) \\ & - h_i * (T_{a,node} - T_{f,node}) \\ & + q_{dot_solar} \end{aligned}$$

with:

- c_{col} : heat capacity of collector per surface (J/(m²*K))
- q_{dot_solar} : power input per surface from the sun (W/m²)
- T: temperature (K)
- t: time (s)
- ULIN: linear heat loss coefficient (W/(m²*K))
- UQUA: quadratic heat loss coefficient (W/(m²*K)²)
- USKY: sky loss coefficient
- U_{wind} : wind speed dependant heat losses (W/((m/s)*m²*K))
- h_i : heat transfer coefficient between absorber and fluid (W/(m²*K))

The energy-balance for every node in the fluid is a differential equation:

$$\begin{aligned} c_{fl} * dT_f/dt = & m_{dot} * c_p / A_{coll} * (T_{f,lastnode} - T_{f,node}) \\ & + h_i * (T_{a,node} - T_{f,node}) \end{aligned}$$

with:

- c_p : heat capacity of fluid (J/(kg*K))

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- c_{fl} : heat capacity of fluid per surface ($J/(m^2 \cdot K)$). It is calculated with :
 $c_{fl} = M \cdot c_p / A_{coll}$, with:

- M : mass of fluid in collector (kg)
- \dot{m} : mass flow rate (kg/s)
- T : temperature (K)
- t : time (s)
- h_i : heat transfer coefficient between absorber and fluid ($W/(m^2 \cdot K)$)

The blocks also contains a weather inclined block which transforms the weather data on a ground surface to data for the collector inclined surface. An other block simulates the optics of a single glazing extra white glass and a last block performs the pressure drop calculus.

Warning : In this case, weather data do not need weather inclined blocks, so I removed it.

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Function Block Parameters: 2xN_collector_flat_plate

collector (mask)
Model for tested collectors. Enter test results in this mask.
Look under mask to change the incidence angle modifier and the radiation calculation method.

Parameters:

collector surface (aperture) in [m²]
2.327

mass of fluid in the collector [kg]
1.827

optical efficiency [-]
0.782

linear heat loss coefficient [W/(m²*K)]
4.07

quadratic heat loss coefficient [W/(m²*K)²]
0.016

radiative loss coefficient [W/(m²*K)]
0

wind dependent heat loss [W/(m²*K*m/s)]
0

heat transfer between absorber and fluid [W/(m²*K)]
500

length between inlet and outlet [m]
21.881

effective heat capacity of collector [J/(m²*K)]
4600

thermal conductivity of absorber [W/(m*K)]
385

thickness of absorber [m]
0.0002

distance between 2 tubes [m]
0.1

external diameter of tubes [m]
0.002

collector efficiency F' [-]
0.954

initial temperature [degree centigrade]
20

number of nodes
1

OK Cancel Help Apply

Figure 3: Parameters for the validation of the 2xNcollector_flat_plate block

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2.2 Model File

I used the following data:

- as input of the weather vector:
 - Sun height (degrees)
 - Sun azimuth (degrees)
 - Diffuse irradiation (W/m^2)
 - Global irradiation (W/m^2)
 - Wind velocity (m/s)
 - Ambient temperature (Celsius degrees)
- as input of the collector:
 - Mass flow (kg/h)
 - Inlet temperature of the fluid in the collector (Celsius degrees)

I compared:

- the outlet temperature of the fluid (Celsius degrees)
- the power and the energy provided by the collector to the fluid (Watts and Joules)

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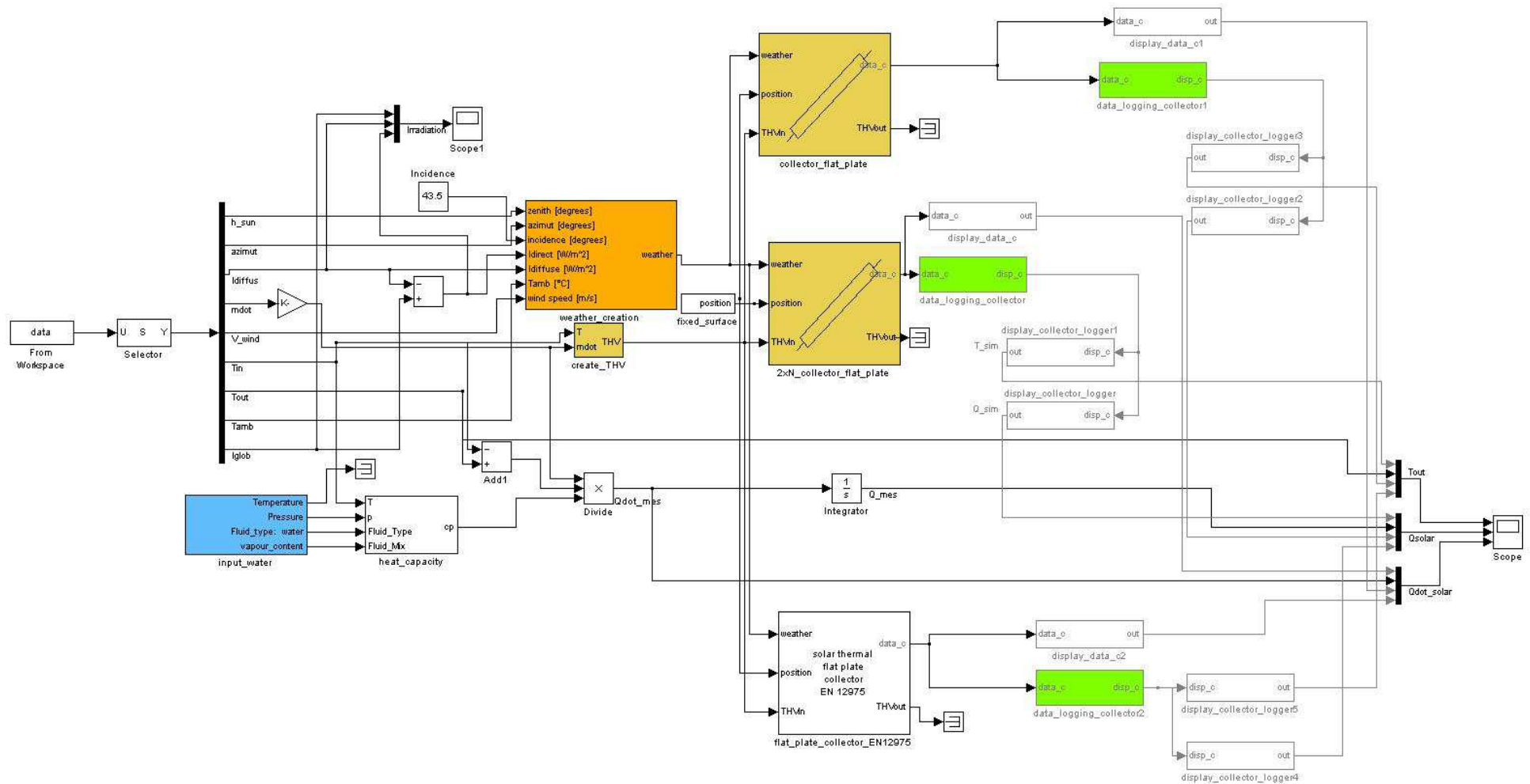


Figure 4: Model for the validation

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3 Results

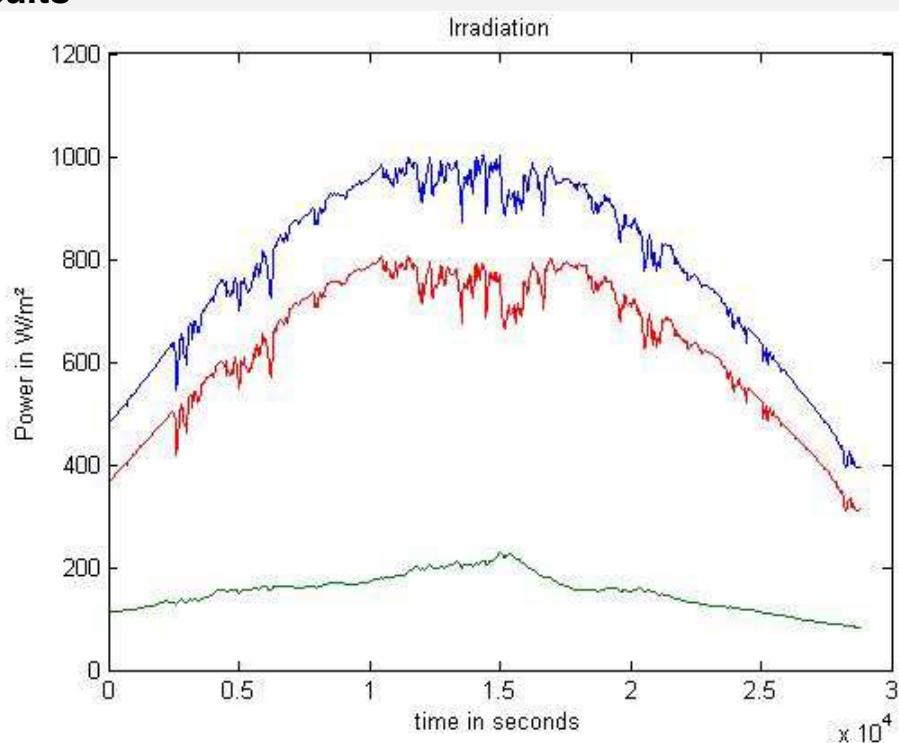


Figure 5: Irradiation on the collector

The EN12975 model is valid for a global irradiation above 700 W/m². That is why I made the comparison on a shorter period than the data total duration.

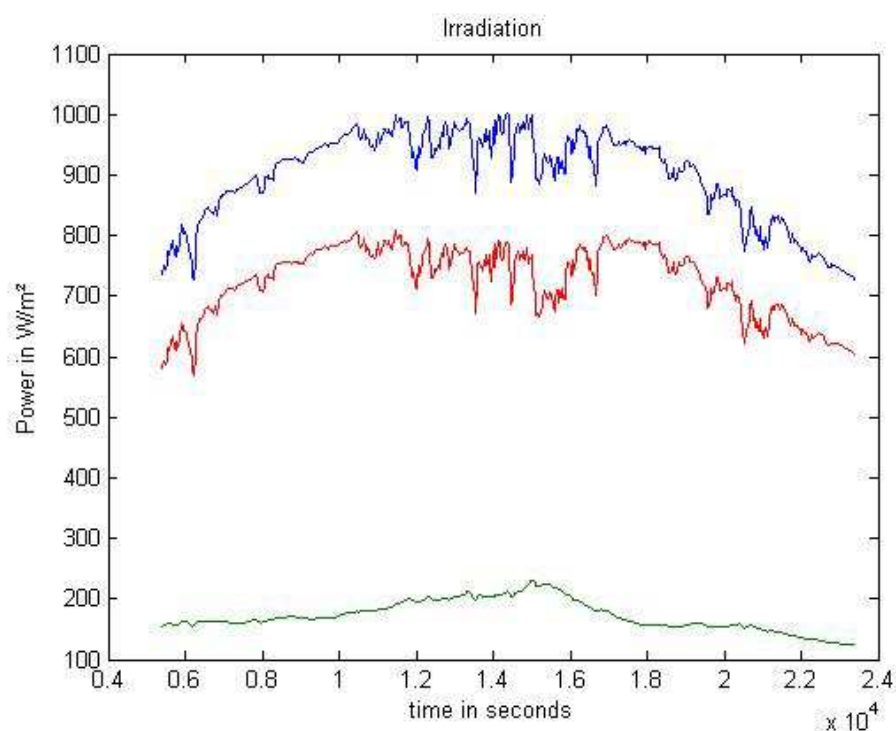
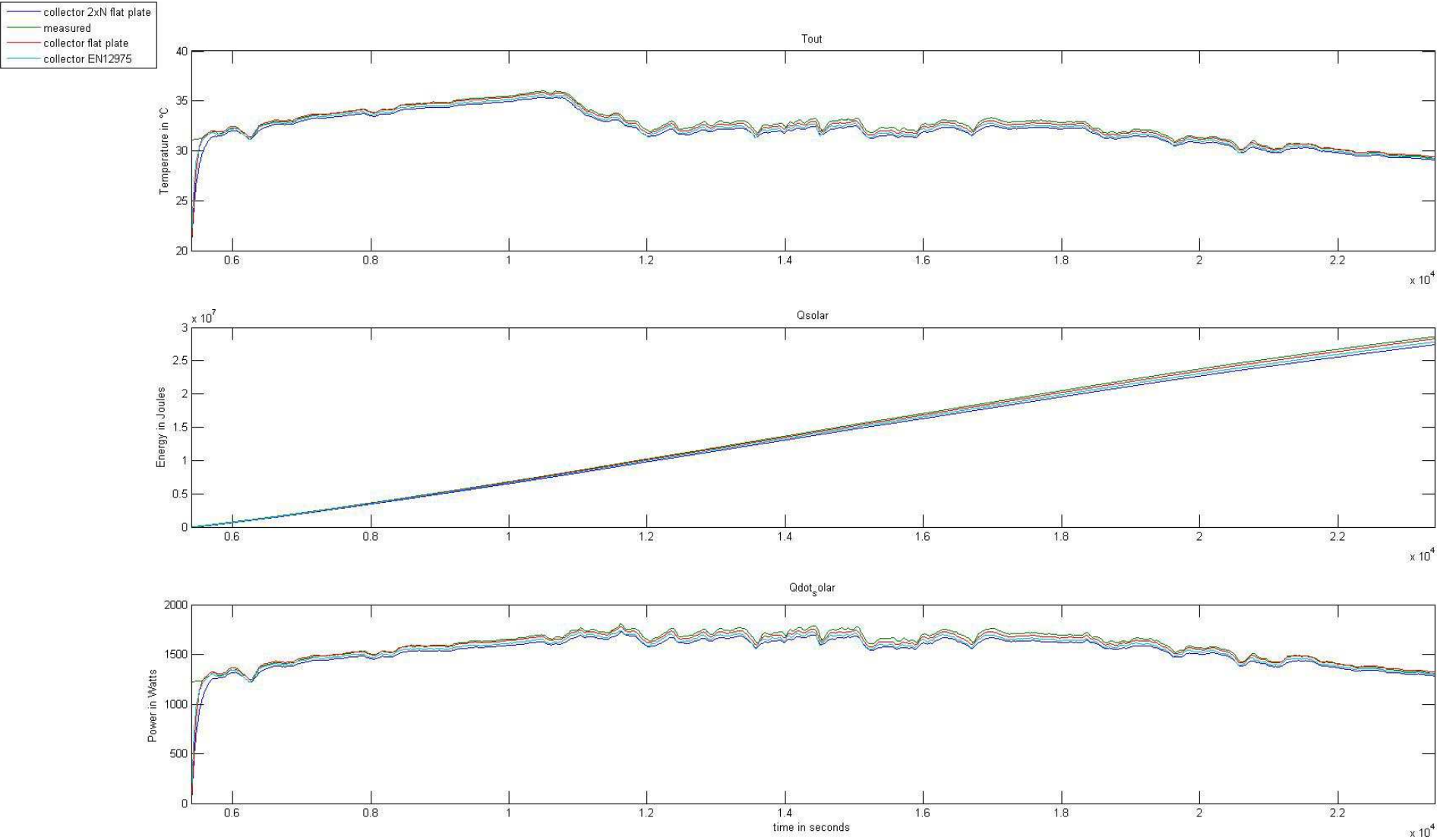


Figure 6: Irradiation above 700 W/m²

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The relative error on total produced energy is :

- For the collector_flat_plate : 1.16 %
- For the flat_plate_collector_EN12975 : 2.79 %
- For the 2xN collector_flat_plate : 4.25 %

The validation is succeed.

4 Literature

Any.

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