

A.N.T. Team 07 MAE 3

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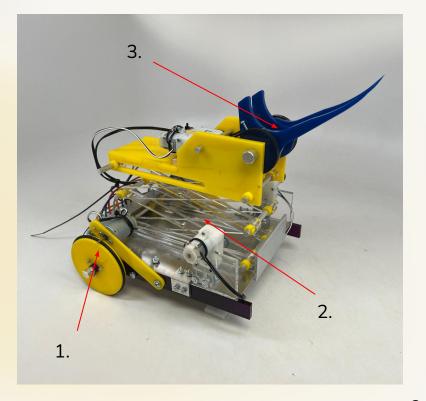
Determining Our Goals

Goals

- Be able to pick up both cue ball and camera
- Move around the arena
- Fit within the 10x10x10 limit starting off and reach the highest object

Key Components

- 1. Independently Controlled Friction Drivetrain
- 2. Scissor lift powered by 2 geared motors
- Dual gear motor arm



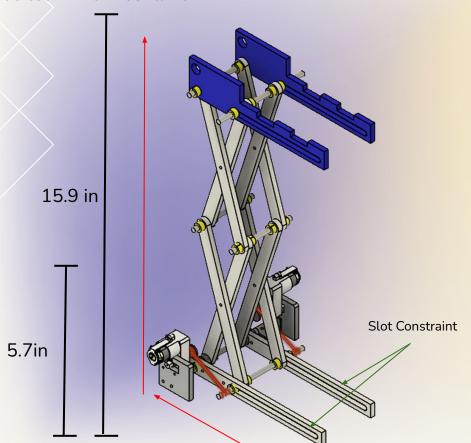


Moving with Precision

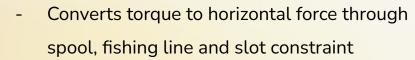
Arm/Scooper Scissor Lift Drivetrain

Reaching the Object

Scissor Lift Mechanism

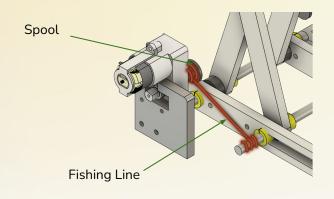


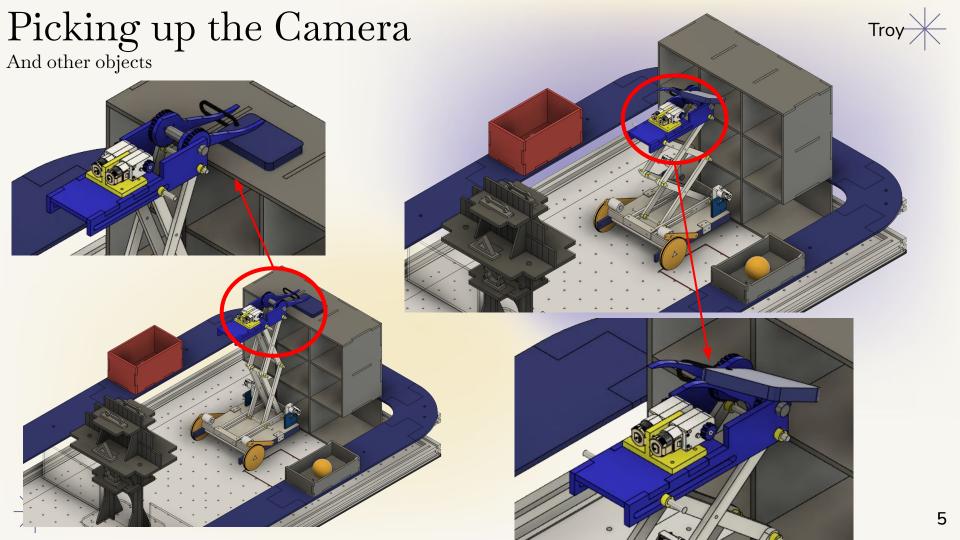
Mechanism

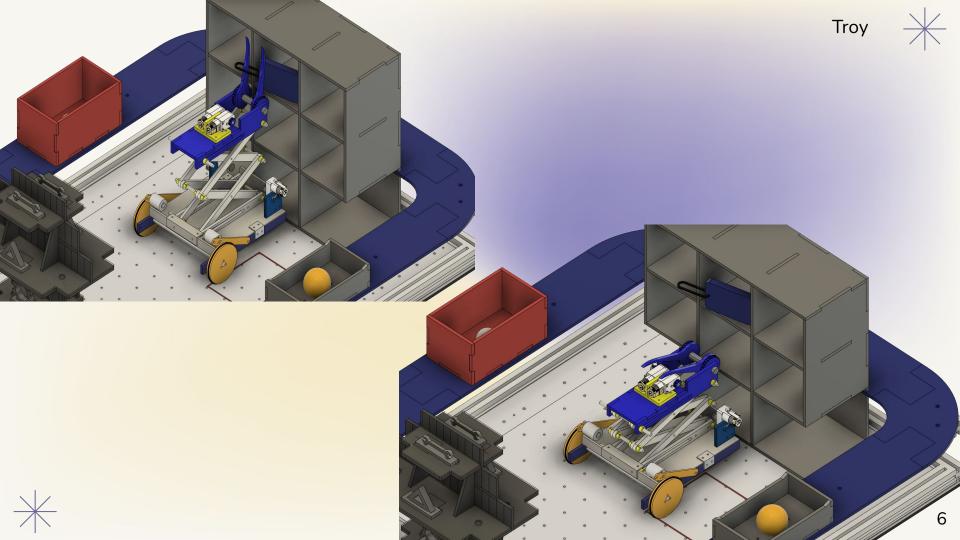


Specifications

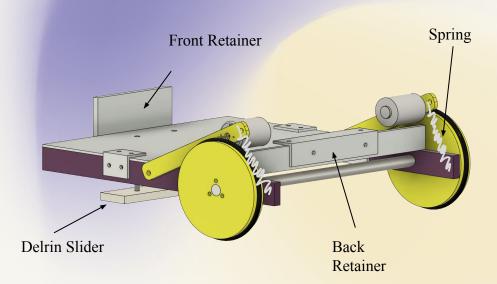
- 1 Degree of Freedom
- Initial height of 5.7in
- Max height of 15.9in







Only left with non-geared high speed motors Solution - Friction Drive!



Friction is Friend

Friction Drive Drivetrain

Objective/Goal:

- Rotate 360° ~7s
- Move from Geisel to lockers ~3s
- Bear load of robot while retaining rigidity

Design:

- Individually controlled friction drive
- Springs to increase normal force between shaft and o-ring
- Metal base to lower center of gravity
- Delrin Slider

Integration:

Scissor lift constrained by retainers



Scooping up the competition

Timing Belt Arm/Scooper

Objective/Goal:

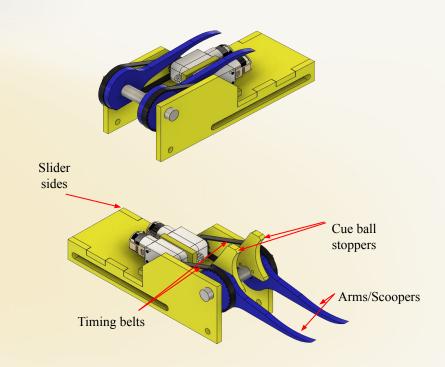
- Ability to grab most items
- Quick, Reliable and Accurate
- Compact

Design:

- Dual motor
- Gear reduction with timing belt 28:10
- 2 arms
- Able to extend ~5"

Robot Integration:

- Directly apart of the scissor lift





Scissor Lift Rises

Objective/Goal:

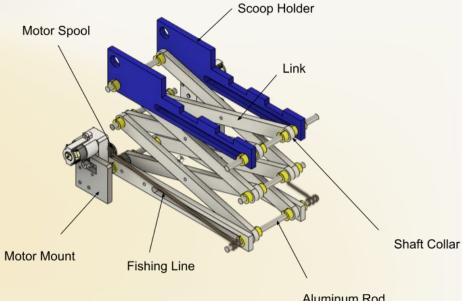
- Reach highest object (~15")
- Stable vertical movement
- Compact
- Lift desired weight of about 600 grams (.6 kg)

Initial Design:

- Pair of 3 linkages on each side, 12 links in total
- Little constraint
- Spools were shorter

Final Design:

- Remove 2 pairs of linkages
- Longer spools
- Used L brackets and bolts to constrain
- Shaft collar, nuts + bolts to hold linkages in place



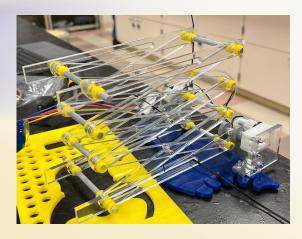
Aluminum Rod

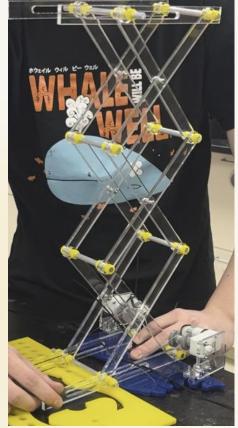


Risk Reduction -

Will the Scissor Lift Work?

- Created a 1:1 model of scissor lift design
- Had the lift be on a temporary base
 and attached the motors + motor mount
- Scissor lift was able to go up and return to initial position when string was slack
- Kept alignment throughout testing
- Motor could easily support weight of scissor jack

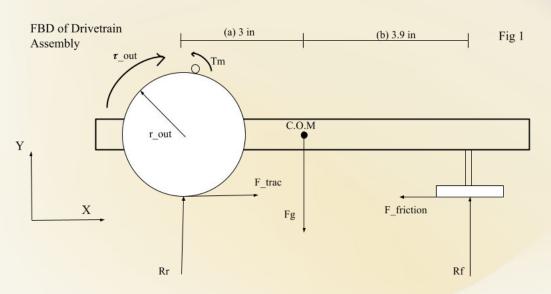


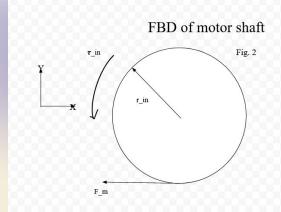


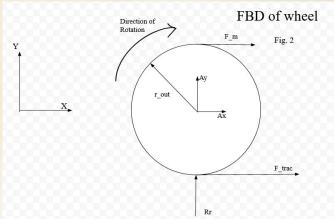




FREE BODY DIAGRAMS









Drivetrain Analysis (Theoretical)

$$(1) \Sigma F_{x} = F_{trac} - F_{friction}$$

(2)
$$\Sigma F_v = R_f + R_r - mg = 0$$

$$(3) \Sigma M_a = R_f(a+b) - mg(a)$$

$$F_{trac} = \mu_{ra}(mg - \frac{mg(a)}{a+b}) - \mu_{delrin}(mg)$$

$$F_{trac} = 2.73N$$

$$\Sigma F_{x} = A_{x} + F_{trac} = 0$$

$$\Sigma F_{y} = A_{y} + R_{r} = 0$$

$$\Sigma M_{a} = F_{trac}(r_{out}) - \tau_{out} = 0$$

Necessary torque output for movement

$$\tau_{out} = 0.11Nm$$

 $F_{trac}(r_{out}) = \tau_{out}$

Assumptions:

- Disregard internal friction and any friction that isn't caused by delrin slider/wheels
- No slippage between metal shaft and rubber o-ring or o-ring and contest table
- Max power output from non-geared motor
- All components of drivetrain operate as a rigid body
- Point mass approximates center of mass

$$\frac{\tau_{out}}{\tau_{in}} = \frac{r_{out}}{r_{in}}$$

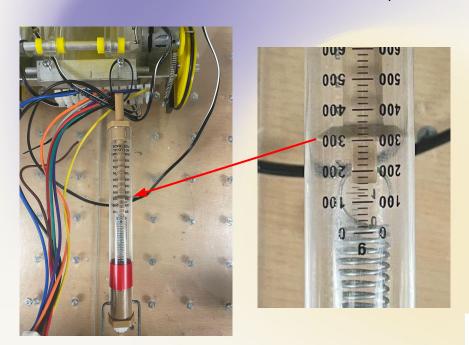
$$\tau_{out} = 3.3Nm$$

$$FOS = \frac{\tau_{avail}}{\tau_{need}} = \frac{3.3Nm}{0.11Nm} = 30$$

Theoretic max torque output given stall torque of DC motor

Theory vs Reality

How did we experiment? What were the results?



Net force found using spring scale

$$\begin{split} \Sigma F_x &= F_{traction} - F_{friction} - F_{spring} = 0 \\ Here \, F_{spring} &= 3.0N, \, F_{friction} = 4.56N \end{split}$$

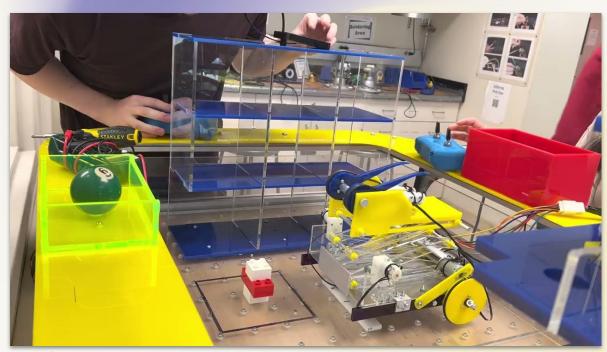
$$Thus \, F_{traction} = 7.56N \, > \, 3.53N \, necessary \, to \, initiate \, movement \end{split}$$

Measured Torque Output

$$\tau_{out} = F_{traction}(r_{out}) = 7.56N(0.04191m) = 0.317Nm$$
 $0.317Nm > 0.148Nm$



Forklift Certified in Action



Theoretical Max - 411 Points

60s run - 301 Points

