

NOTATION

SYMBOLS which only arise once, and are defined on the spot, are not listed.

$2b$	width of Hertzian contact zone
$\frac{b}{R} = 4 \left(\frac{W}{2\pi} \right)^{\frac{1}{2}} = 4P_0$	
C	capacitance
C_D	inter-specimen capacitance
C_F, C_S	pad-disc capacitances
c	specific heat
E	internal energy
E	e.m.f.
E_1, E_2	elastic moduli of solids in contact
$\frac{1}{E'} =$	$\frac{1}{2} \left[\frac{1 - \sigma_1^2}{E_1} + \frac{1 - \sigma_2^2}{E_2} \right]$
e_x, e_y, e_z, e_{xy}	strain components in solids
F	surface shear force
\bar{F}	$F/2\eta u$
F'	$F/E'R$
F_R	surface shear force in pure rolling
F_s	surface shear force due to sliding
G	$\alpha E'$
G	shear modulus
H	h/R
h	lubricant film thickness
h_m	film thickness at point of maximum pressure
h_0	film thickness on line of centres
K	$2/\pi E'$
k	thermal conductivity

l	gear centre distance
L	width of leading edge of electrode
M	W'/P'_0
N_1, N_2	speeds in r.p.m. of gear wheel and pinion
P	p/E'
P'	line load/unit length on solid surface
P_0	p_0/E'
$P'_0, 1, 2, \dots$	load per unit length carried by individual rollers in a roller bearing assembly
P'_0	maximum load per unit length carried by most heavily loaded roller
P_x, P_y	hydrodynamic force components per unit length of cylinder
$\frac{P'_x}{P_x}, \text{etc.}$	$\frac{P'_x/E'R}{P_x/2U}$
p	pressure
p_0	Hertzian maximum pressure
$p_r, p_\theta, q_{r\theta}$ p_x, p_y, q_{xy}	stresses in solids
Q	volume rate of flow of lubricant
\underline{Q}	q/E'
\bar{Q}	Q/uh_0 , dimensionless flow rate
q	reduced pressure, defined by $q = (1 - e^{-\alpha p})/\alpha$
q	heat flow
R	effective radius of roller pair = $\frac{R_a R_b}{R_a \pm R_b}$
R_a, R_b	radii of cylinders or rollers in contact
R_1	radius of roller bearing inner race
R_1, R_2	pitch circle radii of gear wheels
R_g	gear ratio
r	radius of roller
r, θ, z	coordinates
S	R/h_0
s	additional coordinate in x -direction
s	r/R_1

s	distance between point of gear contact and pitch point
$s_{\text{frac.}}$	fractional distance along line of action of gears measured from base circle of wheel
U	$\eta_0 u / E' R$
u	$\frac{1}{2}(u_1 + u_2)$
u_1, u_2	surface velocities of solids in x -direction
u, v, w	fluid velocities in x -, y -, z -directions
u, v, w	solid displacements in x -, y -, z -directions
V	$\eta_0(u_1 - u_2) / E' R$
W	$w / E' R$
W'	total load per unit length on roller bearing $\left(= \frac{\text{total load on bearing}}{\text{length of rollers}} \right)$
w	load per unit length of cylinder
X	x/b
x, y, z	coordinates
Z	total number of rollers in bearing
α	pressure exponent of viscosity, $\eta = \eta_0 \exp(\alpha p)$
α	$\frac{\omega_c}{\Omega} 2(1 + s)$
β	$\frac{\omega}{\Omega} \frac{2s(1 + s)}{(1 + 2s)}$
γ	temperature exponent of viscosity, $\eta = \eta_x \exp(-\gamma\theta)$
Δ	radial clearance in roller bearing
δ	deflection
ϵ	dielectric constant
ϵ	radial interference in roller bearing
η	viscosity
η_0	"controlling viscosity", viscosity at conditions of entry to contact
η_L	viscosity of lubricant at supply temperature
η_n	viscosity at ordinate where $du/dy = (u_2 - u_1)/h$
η_s	viscosity at solid surface temperature
θ	temperature

μ	coefficient of friction
ϱ	fluid density
ϱ_m	fluid density at point of max. pressure
ϱ_0	fluid density at conditions of entry to contact
σ_1, σ_2	Poisson's ratio
τ	tangential surface stress
φ	energy dissipation function
φ	stress function
ψ	pressure angle
Ω	angular velocity of bearing inner race
ω	angular velocity of a bearing roller about its centre relative to rotating axes
ω_c	angular velocity of bearing roller centre about shaft axis (cage speed)