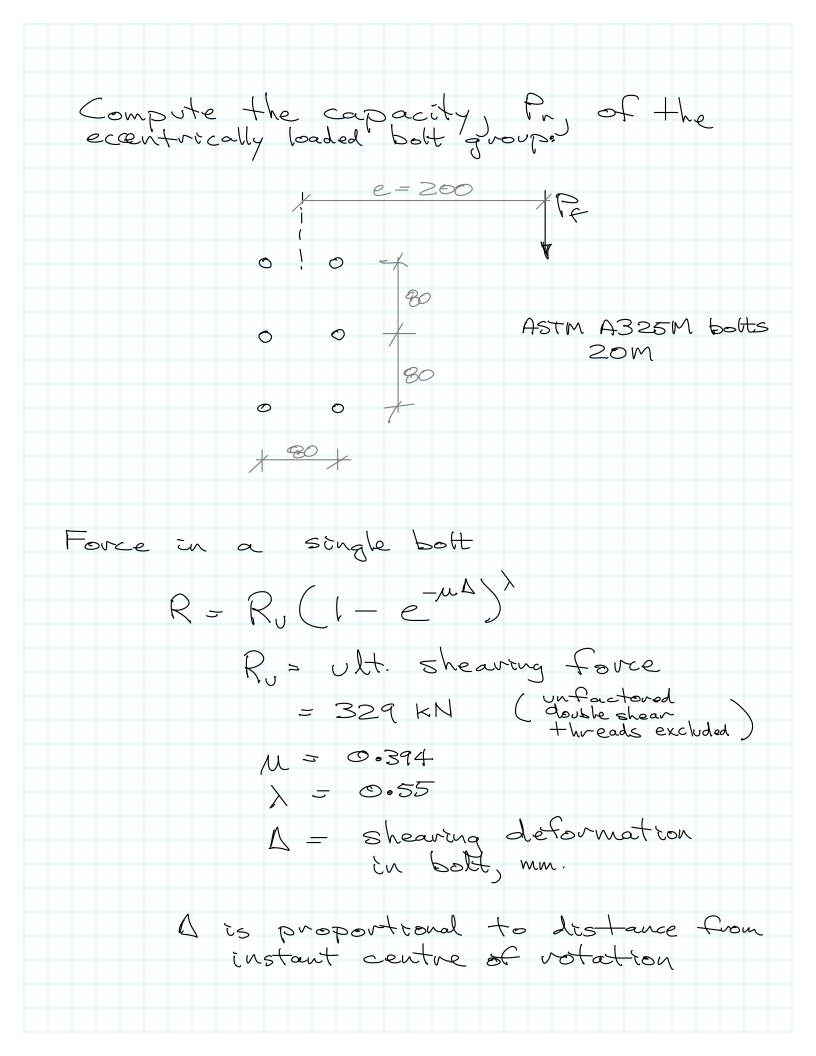
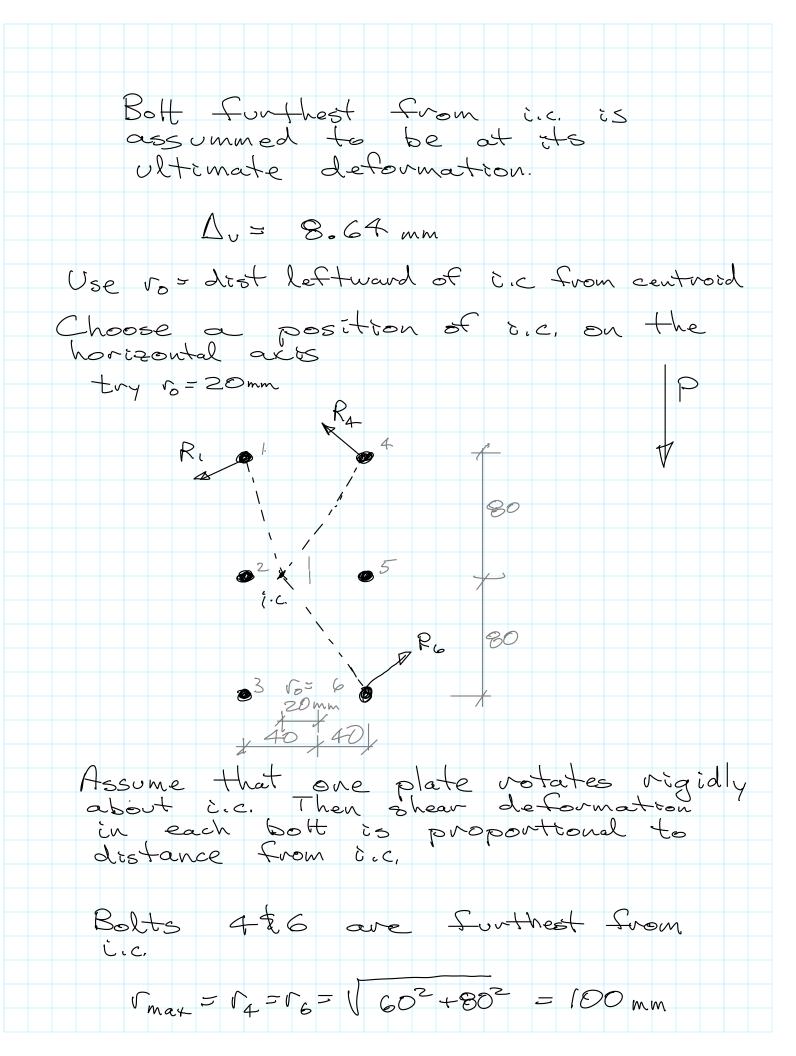
CIVE 3205 Example Bott-4 Eccentrically Loaded March 11, 2013





The deformation of 446 is D4=10=8.64mm The force in 46 is R₄= R₆ = 329 (1 - e) 55 = 322.9 KN Bolts 1 k3 - distance from i.c. r, = r3 = 1202+802 = 82.46 mm shear deformations are: Di - Ti Amax r, = r3 = 82.46 x 8.64 = 7.125 mm the Forces in botts 1 \$ 3 R=R3=329(1-e-394x7.125).55 = 317.9 KN

Bolt 2:

$$C_2 = 20 \text{ mm}$$

 $A_2 = \frac{20}{100} \times 8.64 \text{ mm} = 1.728 \text{ mm}$
 $R_2 = 329 \left(1 - \frac{394 \times 1.728}{600}\right).55$
 $= 223.2 \text{ kN}$
Bolt 5:
 $C_3 = 60 \text{ mm}$

$$\Delta_{5} = \frac{60}{100} \times 8.64 \text{ mm} = 5.184 \text{ mm}.$$
 $R_{5} = 329 (1 - e^{-394 \times 5.184}) \cdot 55$
 $= 304.8 \text{ kN}.$

Now, each of these forces has a vertical component, Vi

† produces a moment about the i.c., Mi

V4 = V6 = 60 x 322.9 = 193.7 KN M4 = M6 = 100 x 322.9 = 32290 KN-mm Bolts 1 & 3 $\frac{1}{43}$ = $\frac{-20}{82.46}$ x 317.9 = $\frac{-77.10}{82.46}$ M, $\frac{-1}{3}$ = $\frac{82.46}{82.46}$ x 317.9 = $\frac{26214}{140}$ km $\frac{2}{V_{2}} = \frac{-20}{20} \times 223.2 = -223.2 \text{ KN}$ $M_{2} = 20 \times 223.2 = 4464 \text{ KN-mm}$ Bolt 5 $V_2 - \frac{60}{60} \times 304.8 = 304.8 KN$ Mz=60x3048= 18288KN-mm If the i.c. is at the correct places these forces will be in equilibrium. First, we can calculate the value of P causing this failure from Pu (e+10) - 2M; P. (200+20) = 32290×2 + 26214×2 +4464 + 18288 P = 635.3 KN this is the ottimate load that is in equilibrium with the moment caused by the both forces.

Now check SF, +A 2F, - P, + 5Vi = -635.3 + 193.7 x2 + -77.10 x -77.10×2 + -223.2 + 304.8 = -320.5 #0 N.G. The oft load, Po should also be in equilibrium with the vertical components of the bolt forces. In this case it isn't. ¿ The position chosen for 10=20mm is not correct. This procedure must be repeated with different values of vo litnu P = \(\frac{\xemin \chi_{\chi}}{\xemin_{\chi\chi_{\chi}}} = \frac{\xemin_{\chi}}{\xemin_{\chi\chi_{\chi}}} = \(\xemin_{\chi} \) For this problem, that happens (closely enough) at: SM; 5 145.3 x103 KN-mm P= 617KN = 211= 616 KM

In	tabular	form.	the.	calcu	lations	ave:
,						
(D=	30°/m	1				

Hod	Χi	Yi	v:	Di= Ti Dmax	R_{i}	\ <u>'</u> ;	M;
#	(mm)	(mm)	(mm)	[max	(KN)	(KN)	CKN-mm)
		62		(70%	313	1-7	
1	-4,3	පිට	80.12	6.286	313	-17	25114
2	-4.3	0	4.32	0.339	105	-105	453
3	-4.3	-80	80.12	6.286	313	— 17	25114
4	75.7	80	110.12	8.640	323	222	3556]
5	75.7	0	75.68	5-937	3//	31/	23545
6	75.7	80	110.12	8.640	323	222	35561
					5	616	145,3×103
			1150 0	3			

 $P_{\nu} = \frac{145.3 \times 10}{35.7 + 200} = 616 \text{ KN}$ Equilibrium satisfied

P' = 2 1'

E. Vo is correct

2. Po = 6/6 KN

The force Ru for a single bolt is unfactored, double shear A325M M20 bolts, threads excluded

Pf < 0, Pv < 0.8 × 616 × 2

Pf < 246 KN single shear

The above is not practical for hand calculation.

Tables 3-14 thru 3-20 are design aids for this From Table 3-15, 2 rows of botts

pitch = 80 mm 3 botts/row

eccentricity = 80mm C = 1.91 Vr = 125 KN (factored, single shear, threads excluded) Pf < C/v < 1.91 x 125 kN (threads excluded) Pc < 239 KN (cf 246 kN, above) (within 3%) Must ensure threads excluded, otherwise reduce strength to 0.70 Must ensure that bearing resistance is greater than shear resistance of 125 kM/bolt.