## HIGH TEMPERATURE HELICAL TUBULAR RECEIVER FOR CONCENTRATING SOLAR POWER SYSTEMS



Presented by Nazmul Hossain

## Acknowledgement

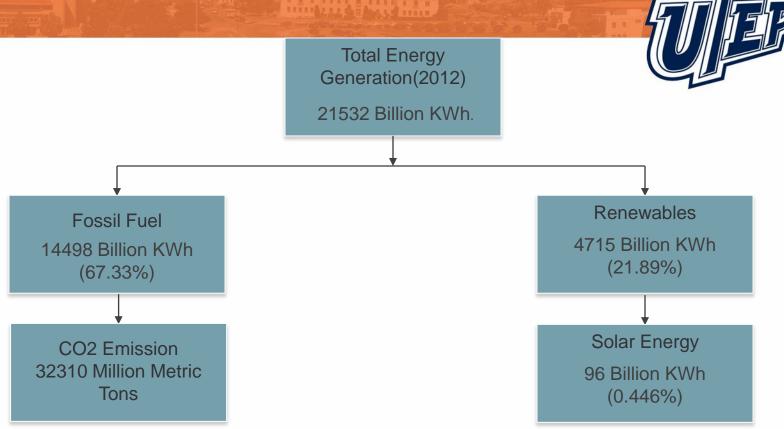


- Dr. Vinod Kumar
- Dr. Shirley Moore
- Dr. Pavana Prabhakar
- Dr. Norman Love

## Agenda

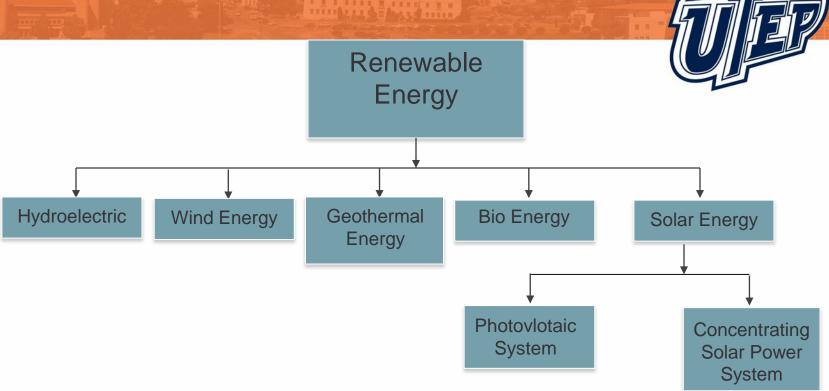


- Introduction
- Background
- Objective
- Methodology
- Results & Discussion
- Conclusion
- Future Study



CO2 emission increased **155 million metric tons** from previous year(2011)

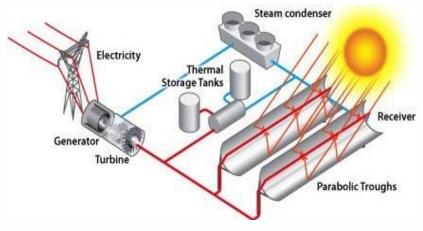
<sup>\*</sup>International Energy Statistics



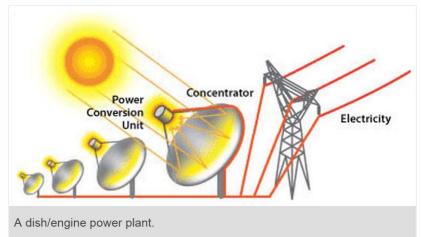
Amount of energy consumed in the world is 1/10000th energy from sun

\*Direct Solar Energy in IPCC special report on renewable energy sources and climate change mitigation

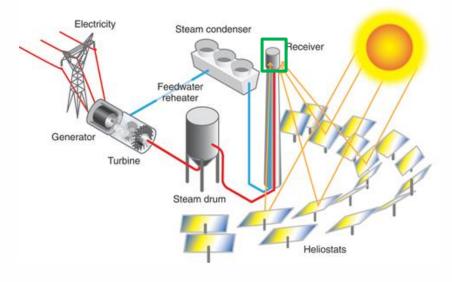




**Trough System** 



Dish/Engine System



Solar Tower System

#### Why Solar Tower?

## Background



#### Ivanpah (California, United States)

- Operating since 2014
- 392 MW Turbine capacity
- 550°C (Direct Steam)
- External tube receiver

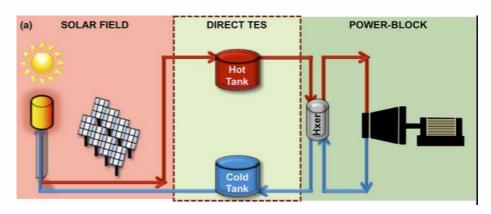




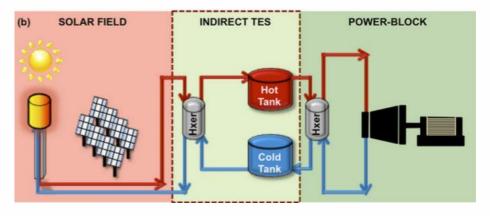
<sup>\*</sup>National Renewable Energy Laboratory



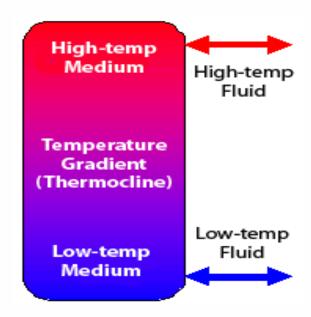
#### **Thermal Storage System**



Two tank direct system



Two tank indirect system



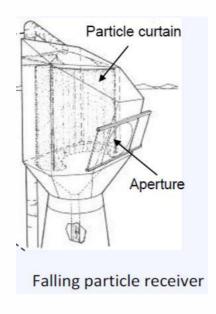
Single tank thermocline system



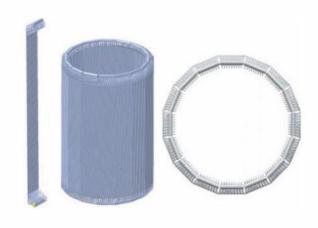
- Heat Transfer Fluid
  - Molten Salt
  - Nanoparticle suspension in molten salt
  - Solid particle
  - Supercritical CO2



#### **Type of Receiver**



Cavity type Receiver



**Tubular Receiver** 



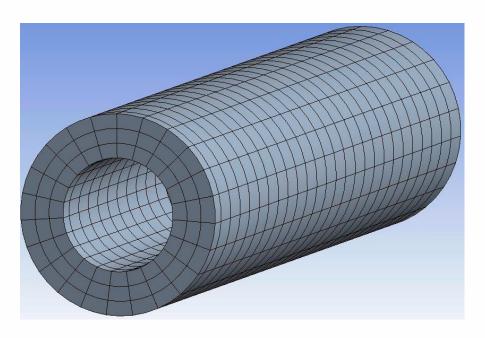
Volumetric Receiver

<sup>\*</sup>Department of Energy

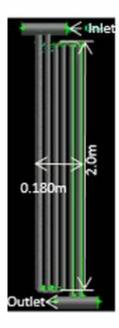
<sup>\*\*</sup> Sandia National Laboratory

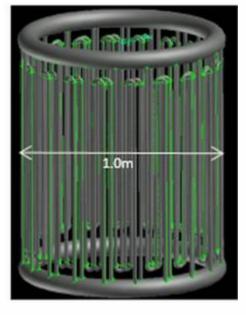


#### Work from our group



**Tubular Receiver** 





Serpentine Receiver

<sup>\*</sup> Work from Jusus Ortega & Samia Afrin

## Objectives

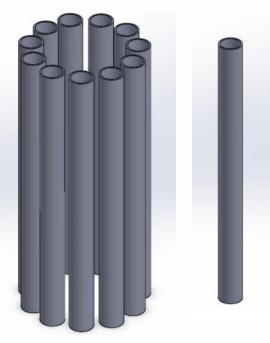


- Design high temperature tubular receiver by varying size and shape
- Investigate receiver at high temperature using single and multiphase computational modeling approach
- Analyze performance of the receivers

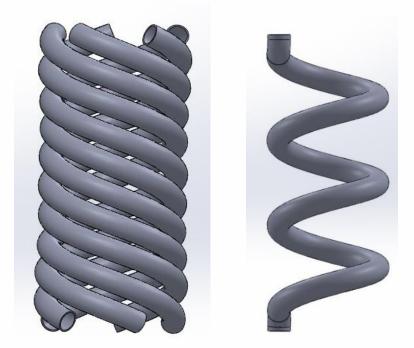
\* DOE's Sunshot Goal: 6¢/kWh

## Receiver





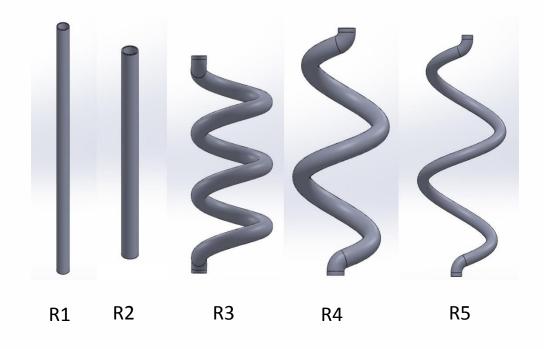
Complete and single part of vertical receiver

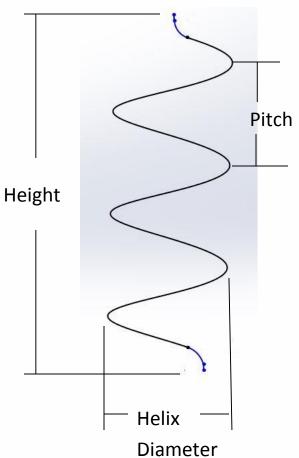


Complete and single part of helical receiver

## Receiver Design







**Designed Receivers** 

### Receiver Design



#### Helix arc length is calculated using following formula

$$Helix\ arc\ length = \sqrt{(\pi DT)^2 + H^2}$$

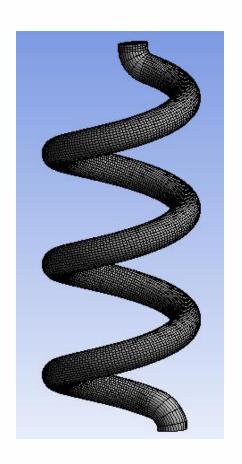
#### Table: Height and arc length of receivers

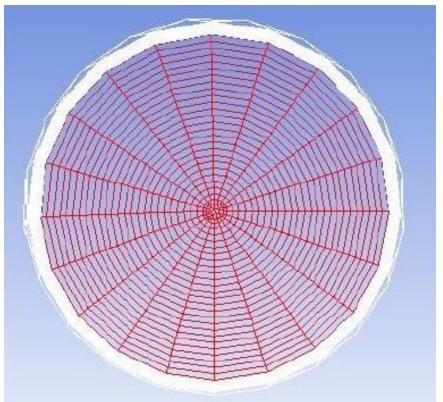
	tube	helix dia	Height		Arc
Receiver	outer dia			Rotation	Length
	(mm)	(mm)	(mm)		(mm)
<b>R</b> 1	20	0	1000	0	1000
R2	46	0	600	0	600
R3	46	246	600	3	2393.74
R4	46	246	600	2	1657.30
R5	30	230	600	2	1564.06

<sup>\*</sup> Formula for the Arc Length of a Helix

## MESH







Mesh at receiver surface

Mesh at inlet

### Modeling



#### Single Phase

- Heat transfer fluid: Molten salt
- Receiver material: Alloy625
- Discrete Ordinate Radiation Model

#### MultiPhase

- Heat transfer fluid: Solid particle
- Primary phase: Air, Secondary phase: Alumina
- Receiver material: Alloy625
- Eulerian Granular Multiphase & DO Radiation Model

Following equation of radiative heat transfer is solved

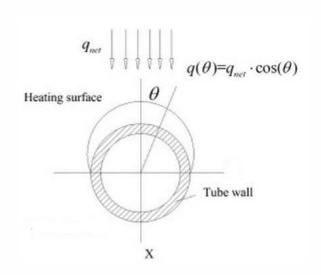
$$\nabla \cdot (I(\vec{r}, \vec{s})\vec{s}) + (a + \sigma_s)I(\vec{r}, \vec{s}) = an^2 \frac{\sigma T^4}{\pi} + \frac{\sigma_s}{4\pi} \int_0^{4\pi} I(\vec{r}, \vec{s}') \Phi(\vec{s} \vec{s}') d\Omega' \nabla \cdot (I(\vec{r}, \vec{s})\vec{s}) + (a + \sigma_s)I(\vec{r}, \vec{s})$$

## **Boundary Condition**

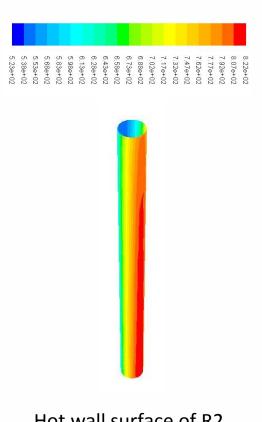
■ Inlet velocity: 0.5 m/s

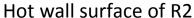
■ Inlet temperature: 523 K

■ Turbulent flow for single phase and Plug flow for Multiphase

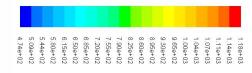


Heat flux schematic





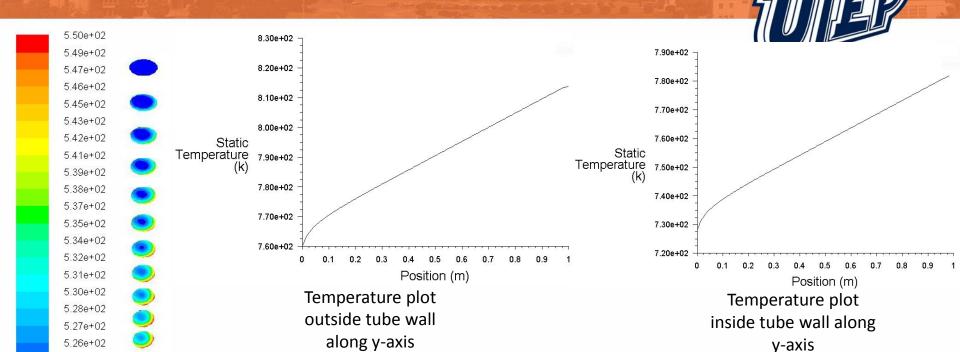






Hot wall surface of R3

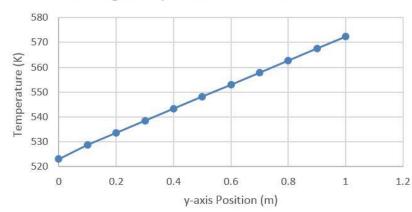
#### Receiver 1



Temperature profile at different section

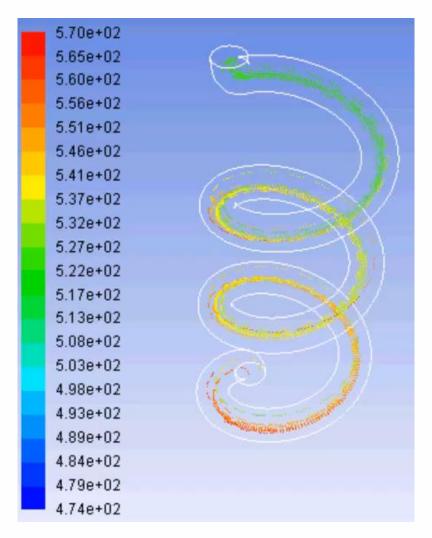
5.24e+02 5.23e+02





#### Flow through Receiver 3(Singlephase)







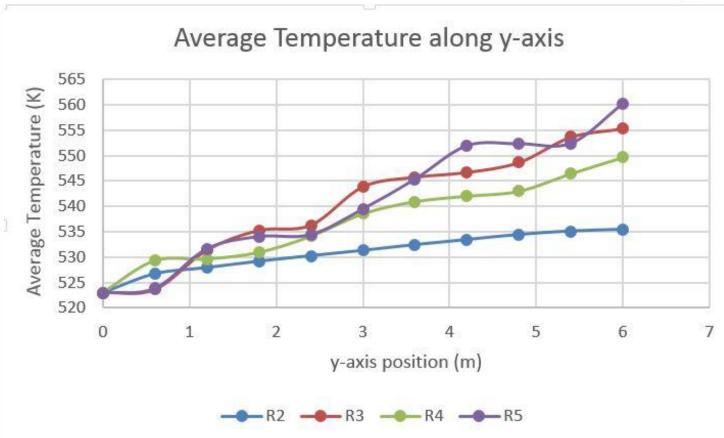


Figure: Average Temperature of receivers Single Phase



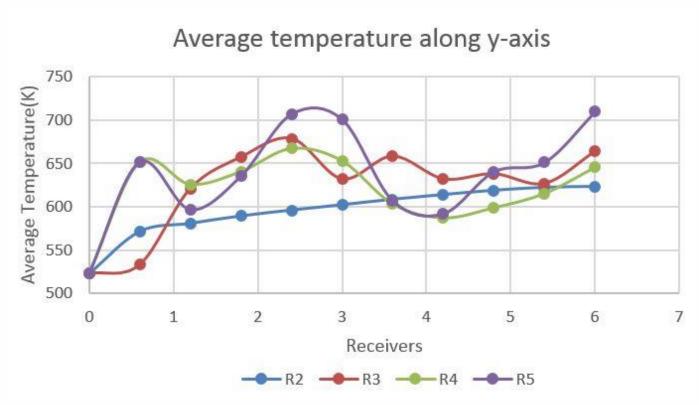


Figure : Average Temperature of receivers Multiphase

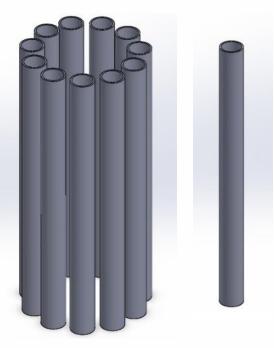


Table 4.1: Receiver performance single phase

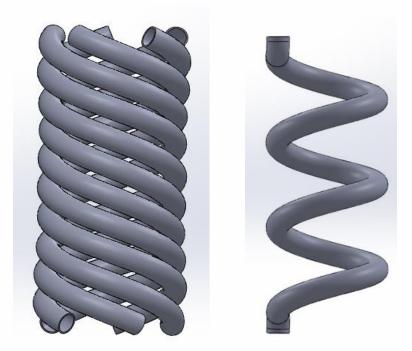
Receiver	Outlet temp (K)	Heat loss (W)	Efficiency(%)	Avg Temp rise
R2	535.43	1172.8	93.23	12.43
R3	555.29	8462.6	87.76	32.29
R4	549.65	7298.3	84.75	26.65
R5	560.25	3804.5	87.09	37.25

## Receiver





Complete and single part of vertical receiver



Complete and single part of helical receiver



#### Average temperature rise on receiver 4

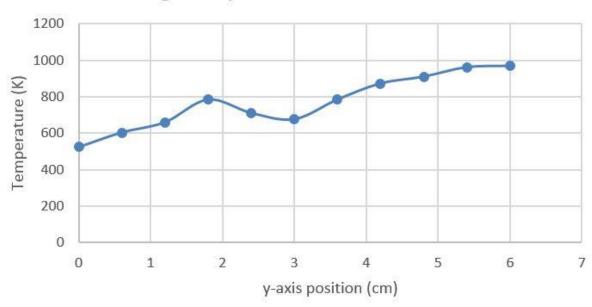


Figure : Average Temperature rise with supercritical CO2



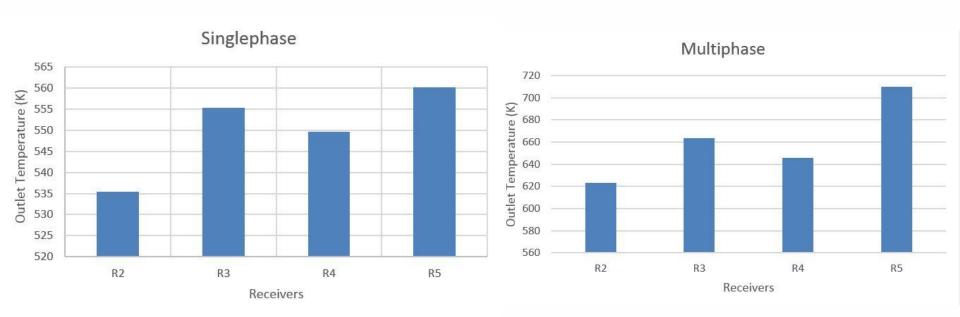


Figure : Outlet temperature bar chart of receivers

#### Conclusion



- Twisting path of vertical fall increases residence time, which leads to higher temperature gain in one cycle
- Receiver performance decreases with reduction of number of rotations and increases with reduction of tube diameter
- Mixing of HTF is better in helical receiver so that less possibility of getting very high temperature in one region and very low temperature in other region of HTF

## Future Study



- Experiment with a single helical receiver on small scale
- Considering shadow effect for more realistic results
- Considering different potential high temperature HTF and
  Tube material for better performance.
- ullet Analysis for supercritical  $CO_2$  as HTF with stress analysis
- Comparison of performance with different solid particle size

## Future Study



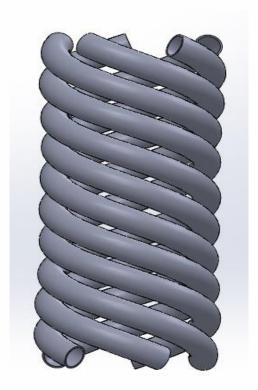


Figure: Complete helical receiver arrangement



# **Thank You**



# Questions