

Composite Beam Bending

ASEN 2001 Lab 3
Fall 2016

Boeing 787 – Bending test



Airbus A380 wing test

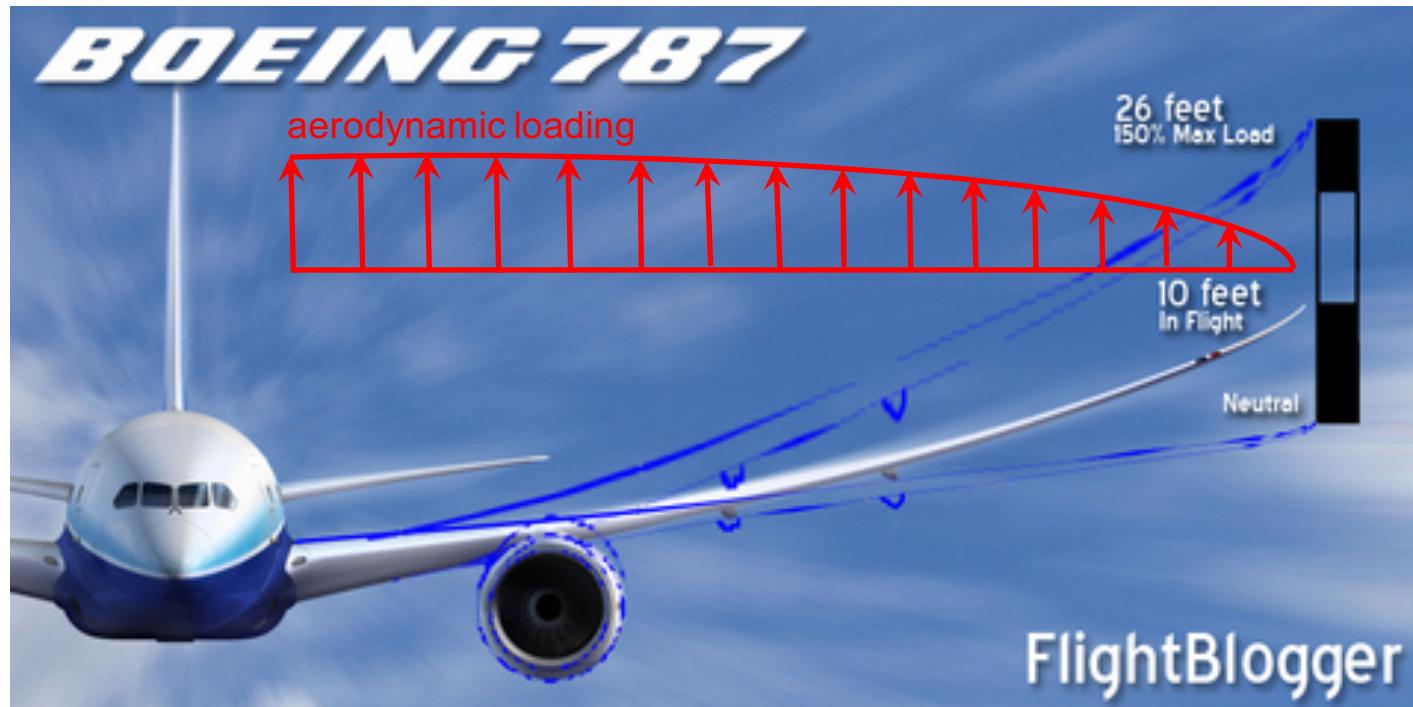


Glider wing test

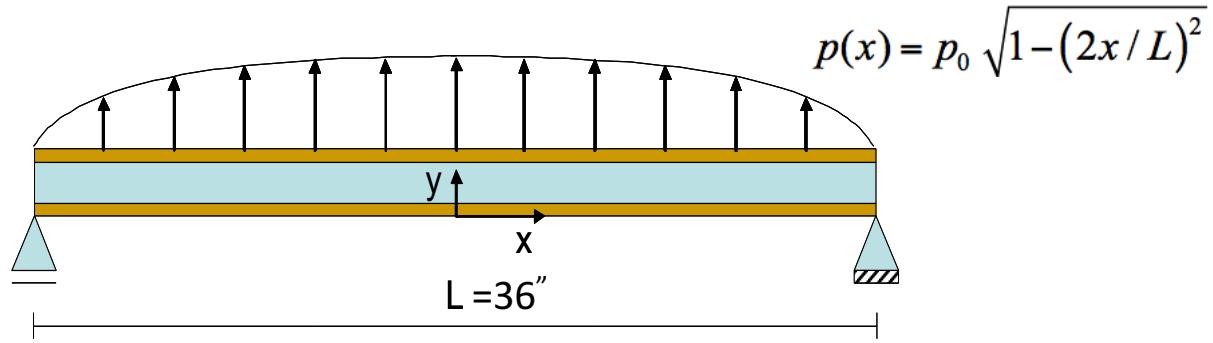


Lab 3 Goals

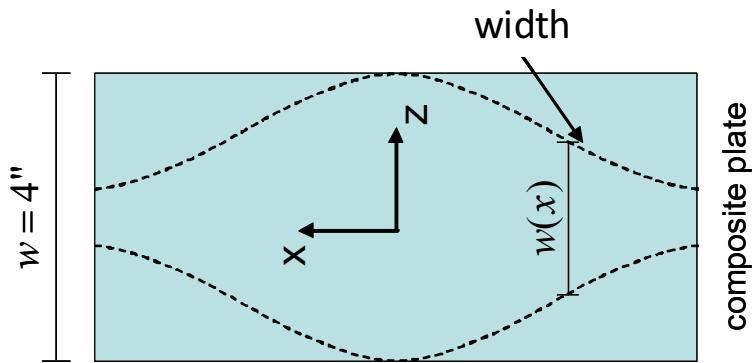
Analyze, design, manufacture, and test a composite wing with **maximum strength** while having **minimum weight** subject to aerodynamic loading.



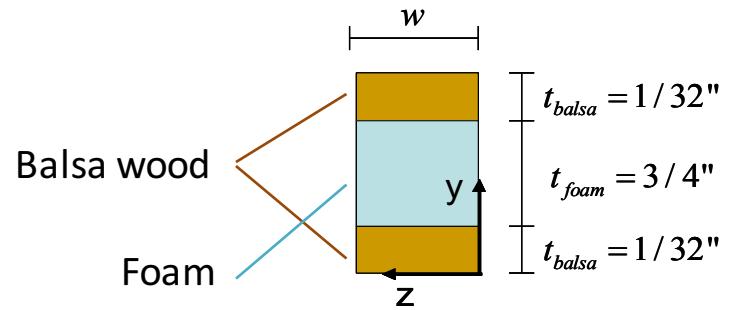
Lab 3 – A simpler set up



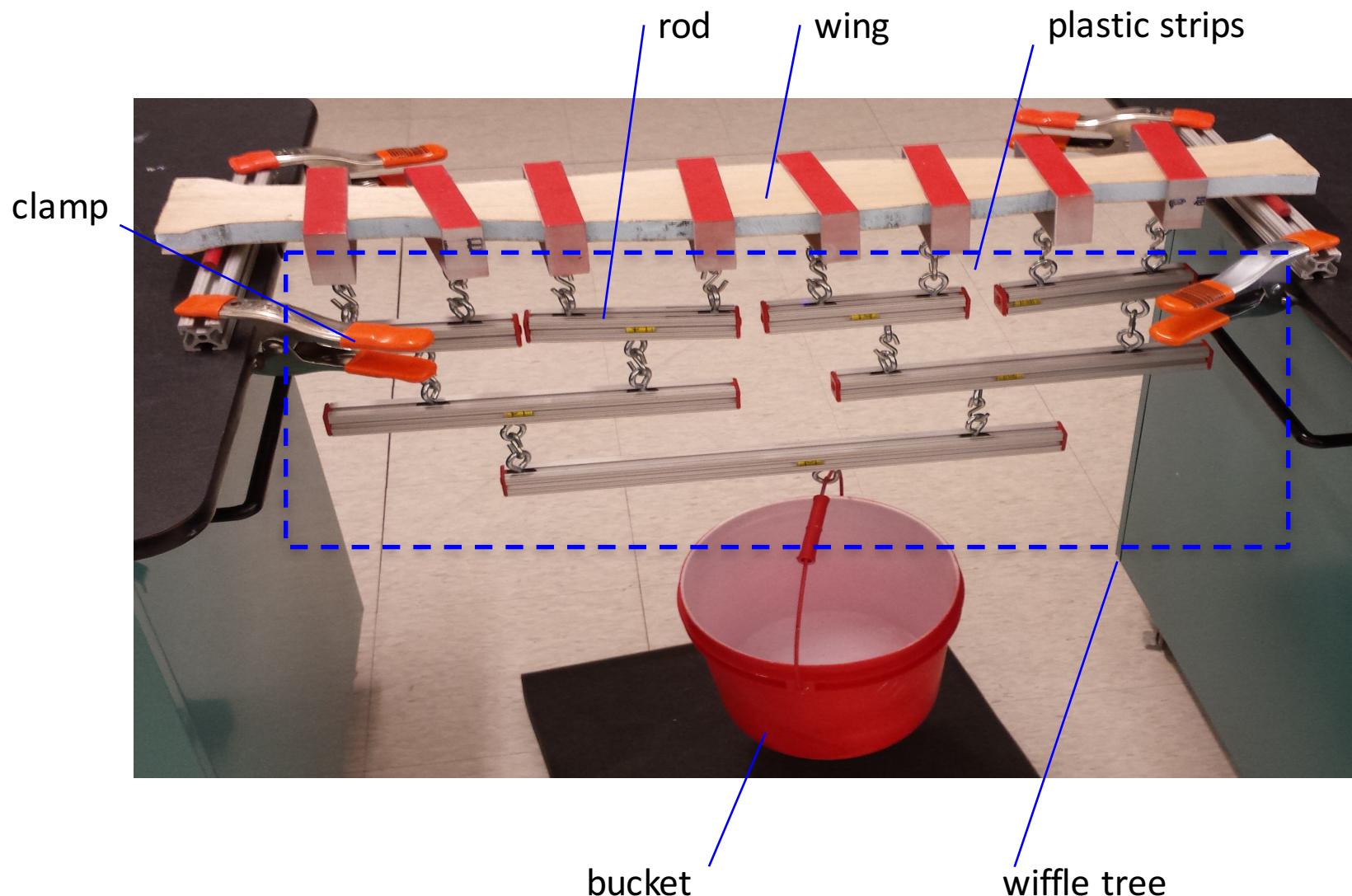
Plan view:



Cross section:



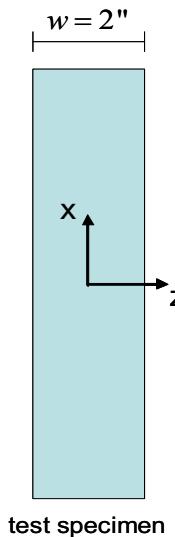
Lab 3 – How will it look like?



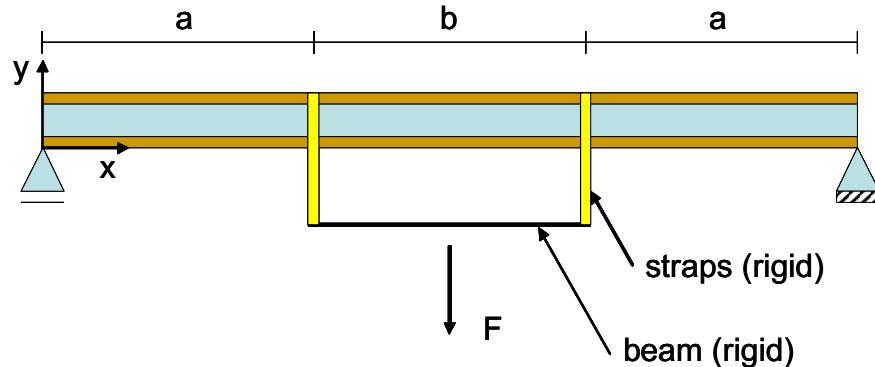
Lab 3 – First identify beam strength

- What is σ_{fail} ?
- What is τ_{fail} ?

Data (available on D2L) on failure of several wings under **four point bending test**:



Four point bending test:



EXAMPLE 11.1

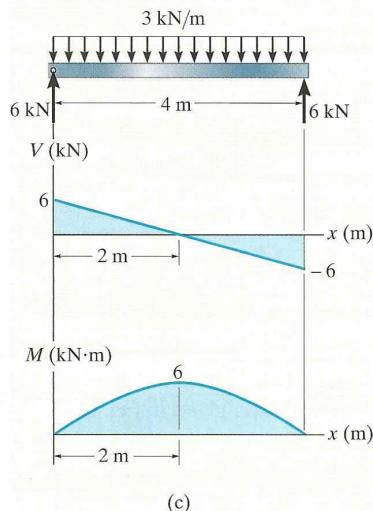
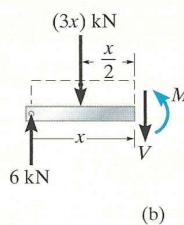
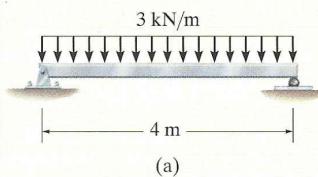


Fig. 11-4

Draw the shear and moment diagrams for the beam shown in Fig. 11-4a.

SOLUTION

Support Reactions. The support reactions are shown in Fig. 11-4c.

Shear and Moment Functions. A free-body diagram of the left segment of the beam is shown in Fig. 11-4b. The distributed loading on this segment is represented by its resultant force $(3x)$ kN, which is found only *after* the segment is isolated as a free-body diagram. This force acts through the centroid of the area under the distributed loading, a distance of $x/2$ from the right end. Applying the two equations of equilibrium yields

$$\begin{aligned} +\uparrow \sum F_y &= 0; & 6 \text{ kN} - (3x) \text{ kN} - V &= 0 \\ V &= (6 - 3x) \text{ kN} \end{aligned} \quad (1)$$

$$\begin{aligned} \zeta + \sum M &= 0; & -6 \text{ kN}(x) + (3x) \text{ kN} \left(\frac{1}{2}x \right) + M &= 0 \\ M &= (6x - 1.5x^2) \text{ kN} \cdot \text{m} \end{aligned} \quad (2)$$

Shear and Moment Diagrams. The shear and moment diagrams shown in Fig. 11-4c are obtained by plotting Eqs. 1 and 2. The point of *zero shear* can be found from Eq. 1:

$$V = (6 - 3x) \text{ kN} = 0$$

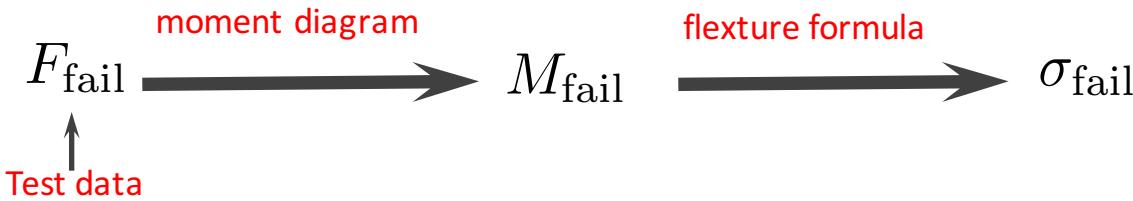
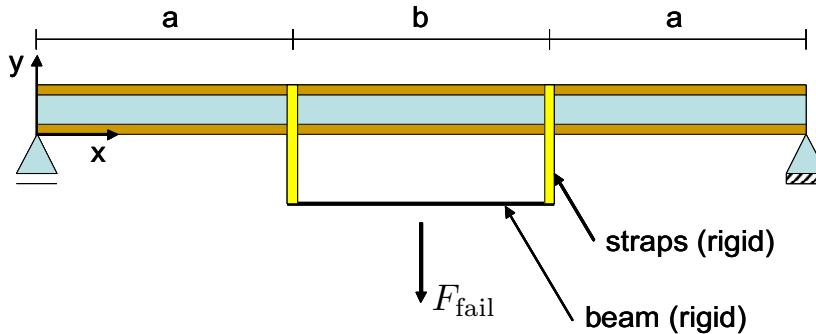
$$x = 2 \text{ m}$$

NOTE: From the moment diagram, this value of x represents the point on the beam where the *maximum moment* occurs, since by Eq. 11-2 (see Sec. 11.2) the *slope* $V = dM/dx = 0$. From Eq. 2, we have

$$\begin{aligned} M_{\max} &= [6(2) - 1.5(2)^2] \text{ kN} \cdot \text{m} \\ &= 6 \text{ kN} \cdot \text{m} \end{aligned}$$

Lab 3 – First identify beam strength

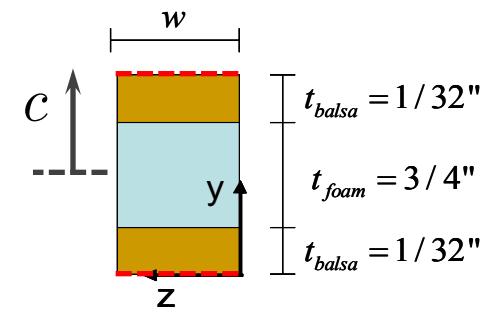
1. Bending failure:



Flexure formula:
$$\sigma_{fail} = -\frac{M_{fail}c}{(I_b + (E_f/E_b)I_f)}$$

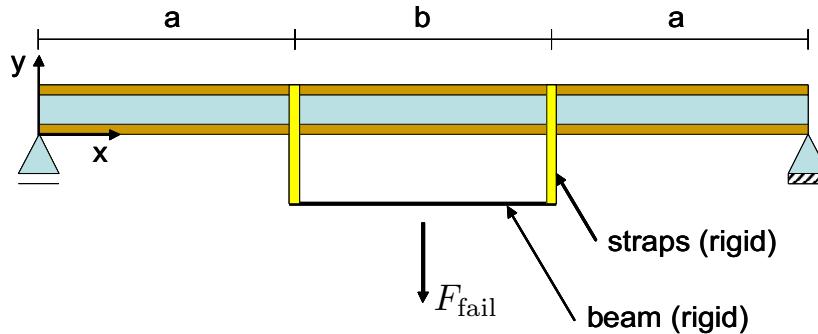
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balsa foam



Lab 3 – First identify beam strength

2. Shear failure:



$$F_{fail} \xrightarrow{\text{shear diagram}} V_{fail} \xrightarrow{\text{shear formula}} \tau_{fail}$$

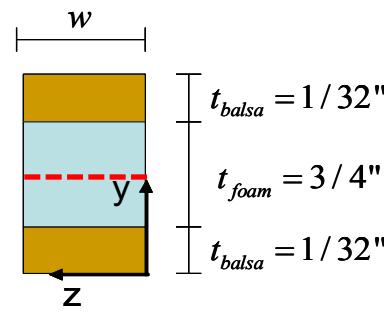
↑
Test data

Shear formula:

$$\tau_{fail} = \frac{3}{2} \frac{V_{fail}}{A_f}$$

foam

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Parameter/variable definitions

F_{fail} Total force on the beam at failure (see data file)

M_{fail} Bending moment at the failure section (under bending)

V_{fail} Shear force at the failure section (under shear)

σ_{fail} Max normal stress at the failure section (under bending)

τ_{fail} Max shear stress at the failure section (under shear)

I_b, I_f Moment of inertia of Balsa wood and foam around the axis passing through centroid of section

E_b Young's modulus of Balsa wood

E_f Young's modulus of foam

A_f Area of foam cross section

c Half of cross section's total height

Data analysis – Tasks for Oct 31st

- Identify E_b and E_f (search online)
- Generate the shear and moment diagrams of the four point test
- For each data entry identify shear or bending failure, and **exclude outliers**
- For each data entry compute σ_{fail} and τ_{fail} depending on the failure mode
- Pick σ_{fail} and τ_{fail} and a corresponding **FOS** for your actual design

Writing a **MATLAB function** to do these is suggested!