FYS2150

Lab Report: Elasticity

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Abstract

Determining the Youngs' modulus of a brass rod by measuring the deflection in three-point bending and determining its root frequency. Comparing the two methods and investigating if their results overlap.

1 Introduction

2 Theory

2.1 Three-point bending

From the Euler-Bernoulli beam theory [4], it follows that the deflection of a beam supported by two points of distance l and a load mg halfway between the two knives is given by

$$h(m) = \frac{mgl^3}{48EI} \tag{1}$$

Where E denotes the Young's modulus [2] of the beam and I the second moment area given by

$$I = \frac{\pi d^4}{4 \cdot 2^4} \tag{2}$$

Where d denotes the diameter of the beam.

Further, Eqn. 1 can be rewritten in the following way

$$E = \frac{4l^3g}{3\pi|A|d^4} \tag{3}$$

Where $A \equiv h(m)/m$, which can be obtained as the gradient from a linear fit on the data gathered when varying the load subjected and on the beam and recording the resulting deflection. Which gives the following relationship

$$h(m) = Am + B \tag{4}$$

2.2 Sound emitted from a brass rod

When struck on its axial side, a metallic rod of certain specifications emit an audible sound made up of signals of varying frequencies. The most audible of which is the root tone. The root tones' frequency is determined by Eqn. 5 where d denotes the diameter of the rod, M its mass, L its length and E its Young's modulus.

$$f = \frac{d}{4}\sqrt{\frac{\pi E}{ML}}\tag{5}$$

2.3 Beats

$$f_S = \frac{|\omega - \omega'|}{2} \frac{1}{2\pi} \tag{6}$$

When two signals of similar frequencies, ω, ω' , superimpose, there is an audible beat of a certain frequency, f_S . The frequency of this audible beat is given by Eqn. 6, and becomes shorter the lower the difference in frequencies between the two signals.

2.4 Errors

When performing arithmetic operations on recorded data, the uncertainty in the data must also carry over to the derived results. How these uncertainties are propagated in different operations can be found in Practical Physics [1].

3 Experimental Procedure

3.1 Three-point flexural test

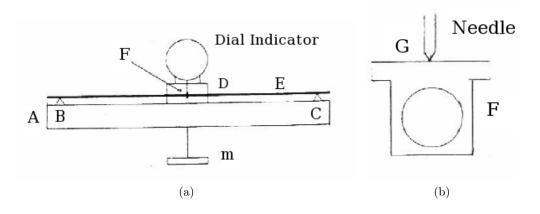


Figure 1: (a) shows the apparatus used for measuring the deflection of a rod and (b) a cross section of the apparatus at point F.

Using Fig. 1a as a reference; The brass rod, A, was laid on the "knives" B and C, in order to ensure that that the dial gauge, manufactured by Baker¹, was positioned halfway between B and C, we measured the distance from the dial gauge to B and C using a measuring tape of type Lufkin pee wee 2m Y612CM with uncertainty ±0.1cm, adjusting the rod such that the difference in measurements would be sufficiently small. At the middle of the rod, there was a ring attached, as shown in Fig. 1b. The flat surface of the ring was in contact with the needle of the dial gauge at G. In order to ensure that the flat surface of the ring was at right angle with the needle, we turned the rod such that the reading of the dial gauge would be at a minimum, as the skewer the surface, the greater the reading. This process was repeated at the start of every attempt of the experiment.

After having prepared the apparatus, three masses of roughly 0.5g, 1kg and 2kg which we denoted m_a, m_b and m_c respectively. They were placed carefully in the tray denoted m in Fig. 1a, in different combinations so that we would get readings for the deflection of the rod at $\approx \{0.5, 1, 1.5, 2, 2.5, 3, 3.5\}$ kg, recorded by reading the dial gauge.

Due to seemingly disturbing the system significantly when adding masses, we were worried that there might be a significant systematic error in the experiment. So we opted to repeat the readings in this experiment several times in order to investigate if the data in the later readings (when the system had been disturbed multiple times in succession) had an increase in its deviation.

Lastly, the distance between the knives, $l_{B,C}$, was measured using the measuring tape and a micrometer of type Moore & Wright 1965 MI with uncertainty ± 0.01 mm. The measuring tape was used to measure the distance between the outer edges of the "knives" at B and C, l_{Bouter} , l_{Couter} . The micrometer was then used to measure the knives thickness l_{knife} , which was needed as the contact points between the rod and the knives are (assumed) at the middle of the identical knives. Since there are two contact points, $l_{Bouter,Couter}$ - $l_{knife} = l_{B,C}$

3.2 Measuring the speed of sound in the rod

The brass rod, with a ring attached to it (same as before), was laid to rest on the flat side of the ring on a solid surface such that the rod is held up by the ring, and nothing else. We also made sure that the rod was not to be disturbed in any way while it was vibrating. When hit with a hammer, it will emit a sound consisting of different frequencies. Following are the two different methods we used for determining the root frequency of the rod. During both experiments, we ensured there were no significant noise pollution during our recording (By which i mean people performing the same experiment as us).

¹I did not take note of the model number of the particular dial gauge that was used during the lab. While working on this report, i have become aware that each Baker dial gauge is individually calibrated. Therefore, i have no values for the instrumental error in the deflection measurements.

3.2.1 By hearing for beats

A speaker was connected to a signal generator. We started the signal generator at 1200Hz and hit the brass rod with a plastic hammer on the flat surface on one end of the rod. By ear, there was an audible beat due to the superposition of the two signals. We adjusted the signal generator such that the frequency of the beat was minimized, and there was essentially no audible difference between the two signals. We did this by trying above and below where we thought the root frequency was, eventually zeroing in on a value.

3.2.2 By Fourier transform

A USB microphone was placed close to the rod, and faced towards it. The microphone was connected to a computer running matlab, with a script that collects audio data from it and Fourier transforms it using FFT. The recordings made were made with a sampling frequency of 8×1024 Hz and varying durations. As before, we hit the rod using a plastic hammer and recorded the data. A total of 7 recordings were made.

3.3 Other measurements

3.3.1 Mass

In order to accurately measure the mass of the rough loads and the rod, the balance scale (Ohaus triple beam balance) which we used had to be calibrated. We did this by weighing a set of three reference weights on the scale, and comparing their measured value to the measured value of the rough loads and the rod using a linear fit. When placing the masses on the scale, we made sure to position the masses in the center of the scale plate and not take a reading until the needle of the balance scale was not sufficiently stable.

3.3.2 Length and thickness of the rod

The length of the rod was measured using the measuring tape, and the thickness using the micrometer. In order to accurately determine the thickness, accounting for any irregularities in the rod due to deformation etc. The thickness was measured several times in different places on the rod, so that we could calculate the mean thickness.

4 Results

4.1 Length and mass measurements

Table 1: Mass of rough load and reference

Stated mass	Measured reference load	Measured rough load	Measured ring	
500g	500.0g	500.1g		
1000g	999.9g	1000.3g		
2000g	2000.1g	2000.5g		
n/a			34.4g	2482.5g

Table 2: Calibrated masses

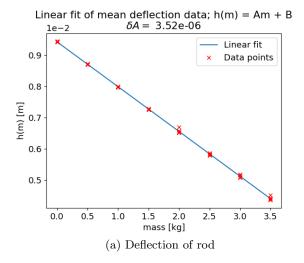
Stated mass	Calibrated rough load	Calibrated rod
500g	500.2 ± 0.1 g	
1000g	1000.3 ± 0.1 g	
2000g	2000.4 ± 0.1 g	
n/a		$2447.9 \pm 0.1g$

Table 1 contains all of the masses measured by the balance scale. The measured mass of the reference weight (which is assumed to be its true mass) is fitted against its measured value with the balance scale using a least square fit, the gradient and zero point in this fit is used to calibrate the weight, and the corrected masses are given in table 2.

4.2 Results from Three-point flexural test

Table 3: Deflection of rod

Attempt	h(0kg)	h(0.5kg)	h(1kg)	h(1.5kg)	h(2.0kg)	h(2.5kg)	h(3.0kg)	h(3.5kg)
no.	[mm]	[mm]	[mm]	[mm]	[mm]	[mm]	[mm]	[mm]
1	9.44	8.72	8.00	7.28	6.58	5.84	5.15	4.43
2	9.42	8.70	7.98	7.26	6.53	5.80	5.09	4.39
3	9.42	8.71	7.98	7.26	6.53	5.80	5.09	4.37
4	9.41	8.69	7.97	7.25	6.52	5.79	5.08	4.36
5	9.42	8.70	7.98	7.26	6.70	5.87	5.19	4.51



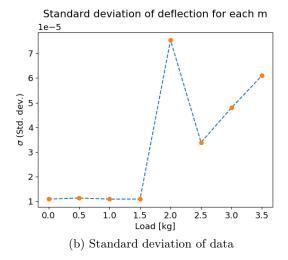


Figure 2: (a) Shows the deflection of the brass rod measured by the dial gauge. (b) Shows the standard deviation of the data points in (a) at their respective masses

Table 3 contains the deflection data recorded with the dial gauge where the loads listed are from the rough, uncalibrated masses. Their corrected value is listed in table 2.

Fig. 2a contains all the recorded data, as well as a linear fit on the mean deflection for each load using corrected values for the mass, m. The error of the linear fit, h(m) = Am + B, dA = 3.52e - 06. Fig. 2b contains the standard deviation of the deflection values for each load.

From the stated data and their given uncertainties, using Eqn. 3 as well as summing the error, the Young's modulus determined by deflection is as follows

$$E_{deflection} = 105.8 \pm 0.4\% \,\text{GPa} \tag{7}$$

4.3 Results from measuring the speed of sound in the rod

When hearing for beats, me and my lab-partner decided that the root frequency was ≈ 1240 Hz.

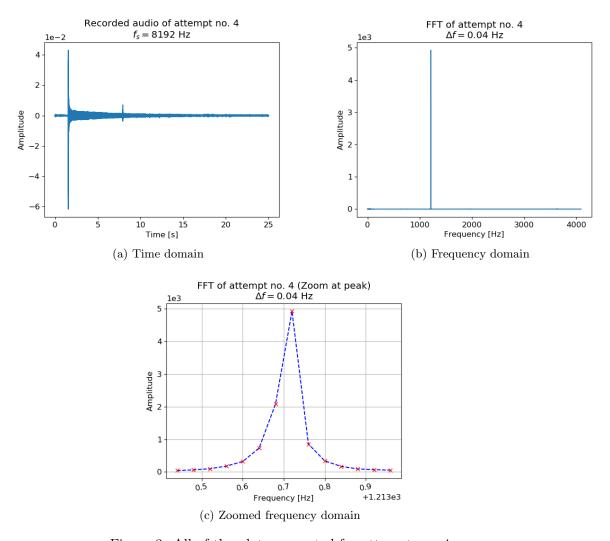


Figure 3: All of the plots generated for attempt no. 4

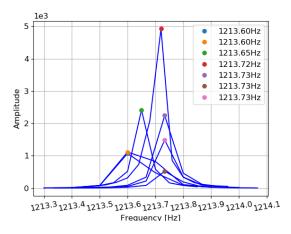


Figure 4: Zoomed frequency plot for all 7 attempts.

Fig. 3 contains the data and derived results from our fourth attempt of the experiment. We performed a total of 7 attempts, all of which yielded in similar results to attempt no. 4. The data yielded from all of the attempts is summarized in Fig. 4 which shows the peaks in the frequency domain in one plot. Table 4 contains all of the relevant numbers related to each attempt, where f denotes the root frequency, Δf the resolution of the frequency domain, t the time of the recording and f_s the sampling frequency.

Table 4: FFT data

Attempt no.	f [Hz]	$\Delta f [Hz]$	t [s]	f_s [Hz]
1	1213.60	0.10	10	8192
2	1213.60	0.10	10	8192
3	1213.65	0.05	20	8192
4	1213.72	0.04	25	8192
5	1213.72	0.04	25	8192
6	1213.72	0.07	15	8192
7	1213.73	0.07	15	8192

Using the root frequency gathered from attempt 4 and 5 (which are identical), the Young's modulus, using Eqn. ?? is

$$E_{sound} = 103.7 \pm 0.2\% \,\text{GPa}$$
 (8)

5 Discussion

Note STD.DEV of deflection increases with m (system is disturbed). Assume the disturbance is normally distributed, therefore error -> given by STD. DEV.

6 Conclusion

References

- [1] G. L. Squires. Practical Physics 4th Edition. Cambridge University Press, 2001.
- [2] https://en.wikipedia.org/wiki/Young's_modulus
- [3] https://en.wikipedia.org/wiki/Beat_(acoustics)#Mathematics_and_physics_of_beat_tones.
- [4] https://en.wikipedia.org/wiki/Euler%E2%80%93Bernoulli_beam_theory# Three-point_bending.
- [5] http://www.uio.no/studier/emner/matnat/fys/FYS2150/v18/kursmateriell/elastisitet/elastisitet.pdf

*

A Code

All of the code used to produce this report. Anything noteworthy should already be mentioned in the main body of the report. Note that when this code was written, readability was not a huge concern, so some of it may not be very easy to interpret.

scripts/FFTlyd.py

```
1 | \#! / usr/bin/env python
  \# -*- coding: utf-8 -*-
2
3
4
  Generates the same figures as FFTlyd.m
5
  author: Nicholas Karlsen
6 """
  import scipy.io as sio
  import matplotlib.pyplot as plt
9
  import numpy as np
10
11
|12| \# Sets font size of matplot
13 plt.rcParams.update({'font.size': 12})
14
15
16 def import matlab (filename):
      # Opens . mat file
```

```
mfile = sio.loadmat(filename)
19
       # Fetches data
       data = mfile.get("data")
20
21
       energi = mfile.get("energi")
22
       fut = mfile.get("fut")
23
       L = mfile.get("L")
24
       t = mfile.get("t")
25
26
       return data, energi, fut, L, t
27
28
29 rel path = "data/"
30 | n = 1
31 mat file = "forsok%i.mat" % n
32
33
34 def raw fig (filename):
       data\,,\ energi\,\,,\ fut\,\,,\,\,L\,,\,\,t\,\,=\,\,import\ matlab\,(\,filename\,)
35
36
       plt.plot(t, data)
37
       plt.xlabel("Time [s]")
       plt.ylabel("Amplitude")
38
39
       plt.ticklabel format(style='sci', axis='y', scilimits=(0,0))
40
41
42 raw_fig(rel_path + "forsok1.mat")
43 plt.title ("Recorded audio of attempt no. 1 \times s = 8192 Hz")
44 plt.savefig("raw_exp2_1.png")
45 plt.close()
46
47 raw_fig(rel_path + "forsok4.mat")
48 plt.title ("Recorded audio of attempt no. 4 \times 1 s = 8192 Hz")
49 plt.savefig("raw exp2 4.png")
50 plt.close()
51
52
53 def figure1 (filename):
       data, energi, fut, L, t = import matlab(filename)
54
55
       fut = np.transpose(fut)
56
       fh = int(len(energi) / 2.0) # half lenght of data
       # Only plot first half of data, as FF mirrors in half-way point.
57
       plt.plot(fut[:fh], energi[:fh])
58
59
       plt.xlabel("Frequency [Hz]")
60
       plt.ylabel("Amplitude")
61
       plt.ticklabel format(style='sci', axis='y', scilimits=(0,0))
62
63
64 | figure1 (rel_path + "forsok1.mat")
65 plt.title("FFT of attempt no. 1\n\$\Delta\ f=0.10\ Hz")
66 plt.savefig("energy_exp2_1.png")
67 plt.close()
68
69 figure1 (rel path + "forsok4.mat")
70 plt.title("FFT of attempt no. 4\n\$\Delta\ f=0.04\ Hz")
```

```
71 plt.savefig ("energy exp2 4.png")
 72 plt. close()
73
 74 eigenfreqs = []
75
 76
   \mathbf{def}\ \mathsf{figure2}\,(\,\mathsf{filename}\;,\ \mathsf{style}{=}"-"\;,\ \mathsf{cross}{=}0)\colon
 77
 78
        data, energi, fut, L, t = import_matlab(filename)
 79
        fut = np.transpose(fut)
 80
 81
        fh = int(len(energi) / 2.0) # half lenght of data
 82
        ipeak = np.argmax(energi[:fh])
 83
 84
        eigenfreqs.append(fut[ipeak])
 85
 86
        i = ipeak
 87
        while energi [i] > np.amax(energi [: fh]) * 0.01:
 88
            i -= 1
 89
 90
        j = ipeak
 91
        while energi[j] > np.amax(energi[:fh]) * 0.01:
 92
            j += 1
 93
        plt.plot(fut[i:j], energi[i:j], color="blue", linestyle=style)
 94
 95
        if cross == 1:
 96
             plt.plot(fut[i:j], energi[i:j], "rx")
 97
        else:
 98
            plt.plot(fut[ipeak], energi[ipeak], "o", label="%.2fHz" % fut[ipeak
        1)
99
100
        plt.grid("on")
101
102 figure 2 (rel path + "forsok1.mat", style="--", cross=1)
103 plt. xlabel ("Frequency [Hz]")
104 plt.ylabel ("Amplitude")
105 plt.ticklabel format(style='sci', axis='y', scilimits=(0,0))
106 plt. xticks (rotation=10)
107 plt.title ("FFT of attempt no. 1 (Zoom at peak) \n\Delta f = 0.10\Begin{cases} Hz")
108 plt.savefig("freq_exp2_1.png")
109 plt. close()
110
111
112 figure 2 (rel path + "forsok4.mat", style="--", cross=1)
113 plt.xlabel("Frequency [Hz]")
114 plt.ylabel("Amplitude")
115 plt.ticklabel_format(style='sci', axis='y', scilimits=(0,0))
116 plt. xticks (rotation=10)
117 plt.title("FFT of attempt no. 4 (Zoom at peak)\n\Delta f=0.04\Hz")
118 plt.savefig("freq_exp2_4.png")
119 plt.close()
120
121
122 for i in range (1, 8):
```

```
figure2 (rel_path + "forsok%i.mat" % i)

plt.xlabel("Frequency [Hz]")
plt.ylabel("Amplitude")

plt.legend()
plt.ticklabel_format(style='sci', axis='y', scilimits=(0,0))

plt.xticks(rotation=10)

plt.savefig("freq_exp2_all.png")
plt.close()
```

scripts/FYS2150lib.py

```
1 | \#!/usr/bin/env python
 2 | \# -*- coding: utf-8 -*-
 3 """
 4 A collection of commonly used functions in FYS2150.
 5
   author: Nicholas Karlsen
   11 11 11
6
7
  import numpy as np
 8
9
10 | \mathbf{def} \operatorname{stddev}(x) :
11
        Finds the standard deviation, and standard deviation of
12
13
        a 1D array of data x.
14
        See. Eqn D. Page 24 squires
15
16
        n = len(x)
17
        sigma = np. sqrt((np.sum(x**2) - 1.0 / n * np.sum(x)**2) / (n - 1))
        sigma_m = np. sqrt((np.sum(x**2) - 1.0 / n * np.sum(x)**2) / (n * (n - 1.0 / n * np.sum(x)))
18
        1)))
19
20
        return sigma, sigma m
21
22
23 | \mathbf{def} | \mathbf{linfit}(\mathbf{x}, \mathbf{y}) :
24
25
        Finds the line of best-fit in the form y=mx+c given two
26
        1D \ arrays \ x \ and \ y.
27
        11 11 11
28
        n = np. size(y)
29
        D = np.sum(x**2) - (1.0 / n) * np.sum(x)**2
30
        E = \operatorname{np.sum}(x * y) - (1.0 / n) * \operatorname{np.sum}(x) * \operatorname{np.sum}(y)
31
        F = np.sum(y**2) - (1.0 / n) * np.sum(y)**2
32
        dm \,=\, np.\,sqrt\,(\,1.0\ /\ (\,n\,-\,2\,)\ *\ (\,D\ *\ F\,-\,E**2\,)\ /\ D**2\,)
33
        dc = np. sqrt (1.0 / (n - 2) * (float (D) / n + np. mean(x)) *
34
                         ((D * F - E**2) / (D**2))
35
36
       m = float(E) / D
        c \ = \ np.mean(y) \ - \ m \ * \ np.mean(x)
37
38
39
        return m, c, dm, dc
```

scripts/lab_data.py

```
1/\#!/usr/bin/env python
 2 \mid \# -*- coding: utf-8 -*-
 3 """
 Elacticity lab, module 2 of FYS2150
 6
   author: Nicholas Karlsen
 7
 8
 9 from pylab import *
10 import scipy.constants as const
11 import FYS2150lib as fys
12
13
14 rcParams.update({'font.size': 13}) # Sets font size of plots
15
16 | \mathbf{def} | \mathbf{weight} | \mathbf{data} (\mathbf{set} = 1):
17
        "set decides which data set the function returns."
18
        set = set.lower() # Forces lowercase
19
        sets = ["masses", "rod"]
20
        # Mass of weights measured with balance
        m\_a\_balance \,=\, 500.1\,e{-3}
21
        m\_b\_balance \,=\, 1000.3\,e{-3}
22
23
        m\_c\_balance~=~2000.5\,e{-3}
24
25
        \# Mass of reference weights
26
        m_{reference} = array([0.5, 1.0, 2.0])
27
        m reference balance = array([500.0e-3, 999.9e-3, 2000.1e-3]) # Weighed
28
29
        \# Using linear fit to correct for error in balance
30
        a\,,\;\;b\,,\;\;da\,,\;\;db\,=\,fys\,.\,linfit\,(\,m\_reference\,,\;\;m\_reference\_balance\,)
31
        \#\ Corrected\ masses
32
       m_a = (m_a_balance - b) / a
                                               \# approx 500g
                                           \# approx 1000g
33
        m_b = (m_b_balance - b) / a
34
        m_c = (m_c_balance - b) / a
                                              \# approx 2000g
35
36
        \# print "\nref weight \n", (m_reference_balance - b) / a
37
        \textbf{print} \ \ "\ \ \text{ncalibrated rough"} \ , \quad \  \  m\_a, \ \ m\_b, \ \ m\_c
        print "error rough", da
38
39
        print
40
        m_{rod_{ring}} = np.array([2482.7, 2482.5, 2482.1]) * 1e-3
41
42
        m_{ring} = 34.4 * 1e-3
                                   \# kg
43
        m\_rod\_ring\_c = (mean(m\_rod\_ring) - b) / a # kg
44
        \mathbf{m}_{\mathtt{ring\_c}} = (\mathbf{m}_{\mathtt{ring}} - \mathbf{b}) \ / \ \mathbf{a} \ \# \ kg
        m \text{ rod } c = m \text{ rod ring } c - m \text{ ring } c
45
46
47
48
        print mean(m rod ring)
        print "\ncalibrated rod", m_rod_c
49
        \mathbf{print} \ "\text{mass rod error"} \ , \ \text{np.sqrt} \ (2 \ * \ \text{da} **2)
50
51
        print
52
```

```
\mathbf{if} \ \mathbf{set} = \mathbf{sets} \, [\, 0 \, ] \colon \ \# \ \textit{Return corrected masses}
 54
                 return m a, m b, m c
 55
 56
           if set = sets[1]:
 57
                 {\bf return} \ \ {\rm m\_rod\_c}
 58
           if set not in sets:
 59
                 print "Invalid set"
 60
 61
                 print "List of valid sets:", sets
 62
                 print "exiting ... "
 63
                 exit()
 64
 65
     \mathbf{def}\ E_{\overline{\phantom{a}},\overline{\phantom{a}},} sound\,(\,f\,\,,\,\,\,L\,,\,\,\,d\,,\,\,M):
 66
 67
 68
           Returns youngs modulus given
 69
           f = root frequency
 70
           L \, = \, lenght \ between \ knives
 71
           d = diameter \ of \ rod
 72
           M = mass \ of \ rod
 73
 74
           return (16.0 * M * L * f**2) / (np.pi * d**2)
 75
 76
     \mathbf{def} \ \mathbf{E} \_ \mathbf{sound} \_ \mathbf{error} (\mathbf{E}, \ \mathbf{sd}, \ \mathbf{sf}, \ \mathbf{sL}, \ \mathbf{sM}, \ \mathbf{d}, \ \mathbf{f}, \ \mathbf{L}, \ \mathbf{M}) :
 77
 78
           return E * np. sqrt ((2 * sd / d)**2 + (2 * sf / f)**2 +
 79
                                        (2 * sL / L)**2 + (2 * sM / M)**2)
 80
 81
 82\,\big|\,d\,=\,\mathrm{np.array}\,(\,[\,1\,5.\,9\,8\,\,,\,\,\,1\,5.\,9\,9\,\,,\,\,\,1\,5.\,9\,9\,\,,\,\,\,1\,6.\,0\,0\,\,,\,\,
 83
                          15.99\,,\  \  15.99\,,\  \  15.98\,,\  \  15.99\,,
                          15.99, 15.99) * 1e-3
 84
 85 d mean = np.mean(d)
 86 d err = np. sqrt(fys.stddev(d)[1]**2 + (0.01e-3)**2) \# Std dev of mean +
          instrumentation \ error
 87
 88 | f_{root} = 1213.72
 89 \mid f\_err = 0.04 \# resolution of FFT
 90 \, \mathrm{M\_err} = 9.8974331835 \, \mathrm{e} - 05 \, \# from \, linfit \, above \, (da)
 91
 92 \mid 1 \mod = 144.4e-2
 93 \mid l\_rod\_err = 0.1e-2
 94
 95 E sound = E sound (f=f root,
 96
                                L=l rod,
 97
                                d=d mean,
 98
                                M=weight data("rod"))
 99
100 print "E from root f = %e" % E sound
101
102 E_sound_err = E_sound_error (E=E_sound,
103
                                               sd=d_err,
104
                                               sf=f_err ,
```

```
105
                                    sL=l rod err,
106
                                    sM=M err,
107
                                    d=d mean,
108
                                    f=f_root,
109
                                    L=l_rod,
110
                                   M=weight data("rod"))
111 print "E err root = %e" % E sound err
112
113 print "error percentage = %.3f percent" % ((E_sound_err / E_sound) * 100)
114
115 \mid \# Experiment 1
116
117 m a, m b, m c = weight data("masses")
118 \mid \text{mass dat} = \text{array}(
        [0, m_a, m_b, m_a + m_b, m_c, m_a + m_c,
119
                                                          # [Kg]
120
         m b + m c, m a + m b + m c
121
122 \mid \# Round 1:
123 | h | 1 = array([9.44, 8.72, 8.00, 7.28, 6.58, 5.84, 5.15, 4.43]) * 1e-3 # [m]
124 # Round 2:
125 \mid h \mid 2 = array([9.42, 8.70, 7.98, 7.26, 6.53, 5.80, 5.09, 4.39]) * 1e-3 # [m]
126 \neq Round 3:
127 \, \text{h} \, 3 = \text{array} ([9.42, 8.71, 7.98, 7.26, 6.53, 5.80, 5.09, 4.37]) * 1e-3
|128| \# Round 4:
129 \mid h \mid 4 = array([9.41, 8.69, 7.97, 7.25, 6.52, 5.79, 5.08, 4.36]) * 1e-3 # [m]
130 \mid \# Round 5:
131 \mid h = array([9.42, 8.70, 7.98, 7.26, 6.70, 5.87, 5.19, 4.51]) * 1e-3 # [m]
132
133 | h \text{ mean} = (h 1 + h 2 + h 3 + h 4 + h 5) / 5.0
134
135 A, B, dA, dB = fys.linfit (mass dat, h mean)
136
137 | \text{mass} = \text{linspace}(0, 3.5, 8)
138 h mass = A * mass + B \# h(m)
139
140
141 def plotdata():
        h_{sets} = [h_1, h_2, h_3, h_4, h_5]
142
        plot(mass, h_mass, label="Linear fit")
143
        \# \; errorbar(mass, \; m * \; mass + \; c, \; yerr=dm, \; color='blue', \; fmt='o', \; label='
144
        Error Range')
145
        for dat in h sets:
146
             plot(mass\_dat, dat, "x", color="r")
147
        plot(NaN, NaN, "xr", label="Data points")
148
        xlabel("mass [kg]")
149
        ylabel("h(m) [m]")
150
        ticklabel format(style='sci', axis='y', scilimits=(0,0))
151
152
        legend()
153
        title ("Linear fit of mean deflection data; h(m) = Am + B \setminus m \setminus delta A = 
        \%.2e''\% dA)
154
        savefig("figs/h_m_fig.png")
155
        close()
```

```
156 plotdata ()
157
158 def plot stddev():
                                  """\overline{Plots} the standard deviation of h(m)
159
                                as\ m\ is\ increased"""
160
161
                                deviation = np.zeros(len(h 1))
162
                                for i in xrange(len(h_1)):
                                                  deviation[i] = fys.stddev(array([h_1[i],
163
164
                                                                                                                                                                                             h_2[i],
165
                                                                                                                                                                                             h_3[i],
                                                                                                                                                                                            h_4[i],
166
167
                                                                                                                                                                                             h_5[i]]))[0]
168
                                plot (mass dat, deviation, linestyle="--")
                                plot \, (\, mass\_dat \, , \ deviation \, , \ "o" \, )
169
                               \begin{array}{l} \label = 1 \\ \label = 1 
170
171
172
                                xlabel("Load [kg]")
                                ylabel("$\sigma$ (Std. dev.)")
173
                                savefig("figs/h_m_deviation.png")
174
175
                                close()
176 plot stddev()
177
178
179 | 1 \text{ BC outer} = 133.9 * 1e-2
180 \mid l_k = diameter = 4.09 * 1e-3
181 | l_BC = l_BC_outer - l_knife_diameter
182 | s | 1 | BC = np. sqrt((0.1e-2)**2 + (0.01e-3)**2)
183
184
185
186
187
             E deflect = (4.0 * 1 BC**3 * const.g / (3 * pi * abs(A) * d mean**4))
              print "\nE from deflection = %e"%E deflect
189 | S E = E | deflect * np.sqrt((dA / A)**2 + (4.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**2 + (3.0 * d | err / d | mean)**3 + (3.0 * d | err / d | mean)**3 + (3.0 * d | err / d | err
                           s l BC / l BC)**2)
              190
191
              print "percentage error in deflection = \%.3f percent\n" \%(100 * S_E)
192
                              E deflect)
193
194
195 print "indestigating if they override"
196 | D = E \text{ sound } - E \text{ deflect}
197 | s D = np. sqrt(S E**2 + E sound err**2)
198 if abs(D) > s_D:
                               print "D > s D"
199
200 | if abs(D) < s_D:
201
                               \mathbf{print} "D < s D"
202 print abs(D) - s_D
203
204 | print " | D | = \%e " \%abs(D)
205 print "s D = %e" %s D
206 | \mathbf{print} \| 2s \| D = \%e^{\parallel} \% (2*s \| D)
```