

Numerical Investigation of Synthetic, Buoyancy-Induced Columnar Vortices

Nicholas Malaya

Department of Mechanical Engineering
Institute for Computational Engineering and Sciences (ICES)
The University of Texas at Austin

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The dust devil phenomenon



Turbulent, unstable motions

- Incident solar energy absorbed into ground
- Ground heats air by conduction
- Heated air expands, lowering density, driven upward by force of buoyancy
- Arizona in Summer ($\Delta T = 30$ K)
- $Ra \approx 10^9 - 10^{11} \rightarrow$ unstably stratified fluid
- Velocities can exceed 33 m/s.

How much energy is present in one of these objects?

Energy Estimate [Sinclair]

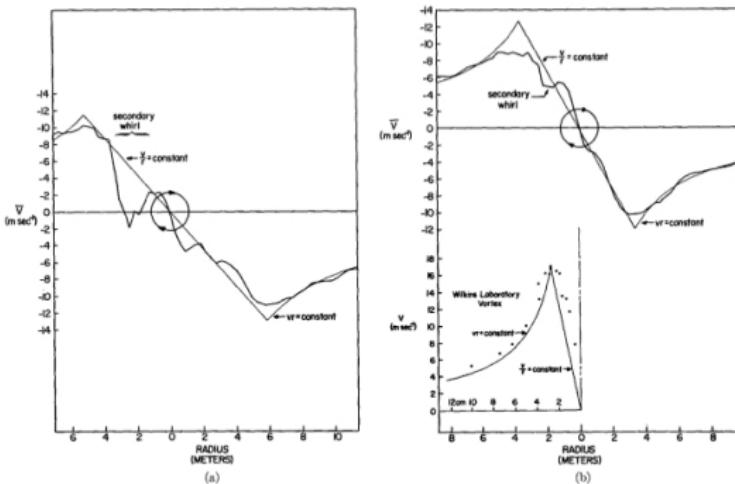
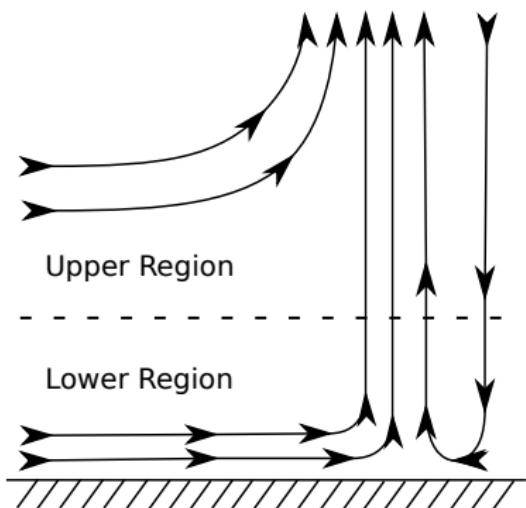


FIG. 11. Radial distribution of the mean tangential velocity at two levels, $z=7$ ft (a) and $z=31$ ft (b), for D-D #1 with superimposed Rankine combined profiles.

- Estimate energy by integrating Sinclair profile
- $R \approx 6$ meters, $V_\theta \approx 11$ m/s, $V_z \approx 10$ m/s
- Energy flux through horizontal plane = $\frac{1}{2}\rho \int V_z(V_z^2 + V_\theta^2) dA$
- $P \approx 24$ kW

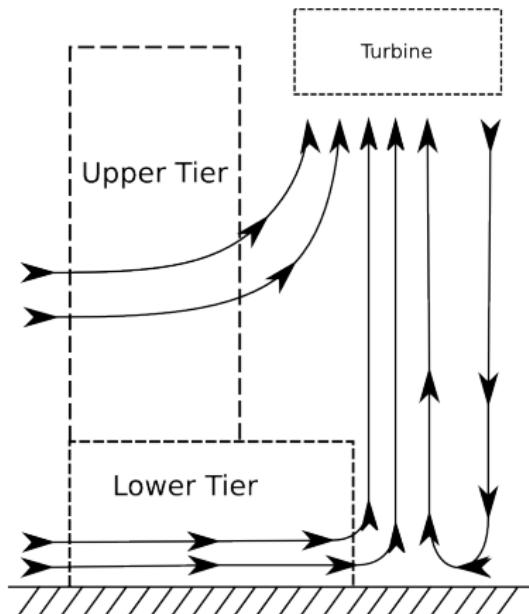
Dust Devil Structure [Sinclair,Kanak,Renno]



Dust devil structure

- Driven by buoyancy of hot ground
- Near surface: radial inflow spinning region (core)
- Downward flow in **hot**, low-pressure center (“eye”)
- Fluid from higher region entrained (inviscid potential flow region)
- Characterized by strong rotation
 - ▶ What generates vorticity?
 - vortex stretching
 - vortex tilting

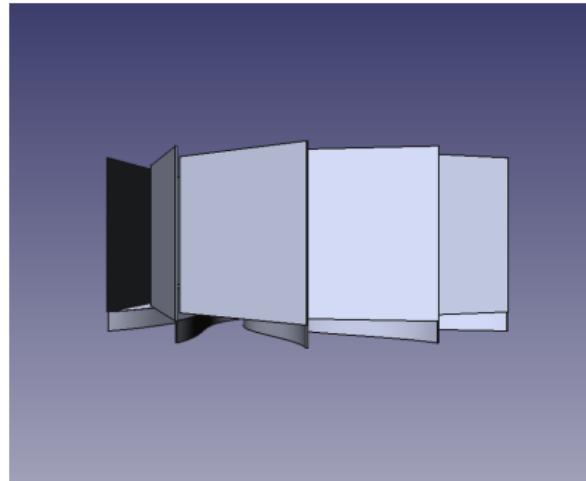
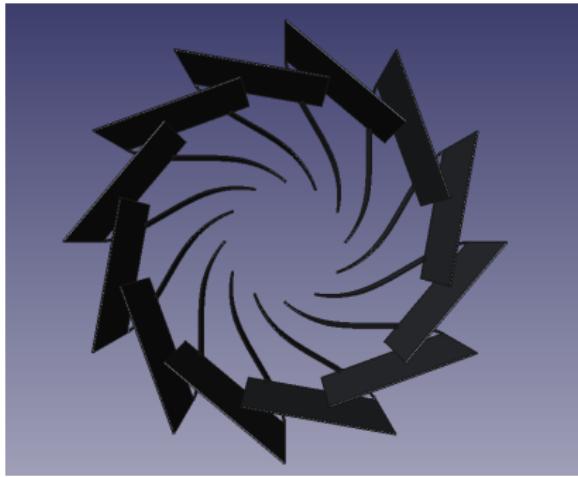
Synthetic Dust Devils



Motivation

- Engineer synthetic vortices for energy generation
 - ▶ Two tier configuration mimics natural phenomenon
 - ▶ Turbine at upper tier vanes to extract energy
- Difficult to explore space experimentally
 - ▶ Field tests in Arizona (GT)
 - ▶ expensive to alter configuration
 - ▶ difficult to measure everything

Example configuration



Turning Vanes

- Several (typically 12) control surfaces used to turn the flow
- Top tier significantly taller than the bottom
- A cone may be added to the top to contract the flow

Modeling Objectives

System Configuration

- Probe and optimize configuration with CFD
 - ▶ Vanes
 - ▶ Cone (and solid surfaces)
 - ▶ Turbine
- Inform design for 2016 field test
 - ▶ Predict energy output
 - ▶ Sensitivities to design
 - ▶ Sensitivities to conditions

Scenario Space – Not much prior work!

- Atmospheric model
- Impact of buoyancy
- Effect of wind

Dynamical Equations

The equations describing fluid flow with natural convection are,

$$\frac{\partial u}{\partial t} + u \cdot \nabla u = -\frac{1}{\rho} \nabla P + \nu \nabla^2 u - g \frac{T - T_0}{T_0} \quad (1)$$

$$\nabla \cdot u = 0 \quad (2)$$

$$\rho c_p \frac{\partial T}{\partial t} + u \cdot \nabla T = \nabla \cdot (K \nabla T) \quad (3)$$

- Incompressible N-S + Boussinesq buoyancy
- Temperature variation small compared to mean temperature
- Low mach number

However, $Ra \approx 10^9 - 10^{11}$ – Need turbulence model!

Atmospheric Viscosity Model

Introduction to Monin-Obukhov

- Under stationary, homogeneous conditions turbulent quantity \bar{f} ,

$$\bar{f} = f(z, \frac{g}{T_0}, \rho_0, U_\infty, q, \nu_l, K_l) \quad (4)$$

- 5 parameters - 4 dimensions = 1 dimension-less group:

$$\xi = -\frac{\kappa \frac{g}{T_0} \frac{q}{c_p \rho_0} z}{U_\infty^3} \quad (5)$$

- Interpret as a length scale,

$$\xi = \frac{z}{L_{M-O}}; \quad L_{M-O} = -\frac{U_\infty^3}{\kappa \frac{g}{T_0} \frac{q}{c_p \rho_0}} \quad (6)$$

- L_{M-O} is height where shear production \approx buoyancy

Atmospheric Viscosity Model

M-O, cont.

By similarity, mean turbulent quantities are functions of the non-dimensional group,

$$\frac{\bar{f}}{f_{MO}} = \phi\left(\frac{z}{L_{M-O}}\right) \quad (7)$$

e.g.

$$\bar{u}(z) = \frac{U_\infty}{\kappa} \phi_u\left(\frac{z}{L_{M-O}}\right) \quad \text{and} \quad \frac{\partial \bar{u}(z)}{\partial z} = \frac{U_\infty}{\kappa L_{M-O}} \varphi_u\left(\frac{z}{L_{M-O}}\right) \quad (8)$$

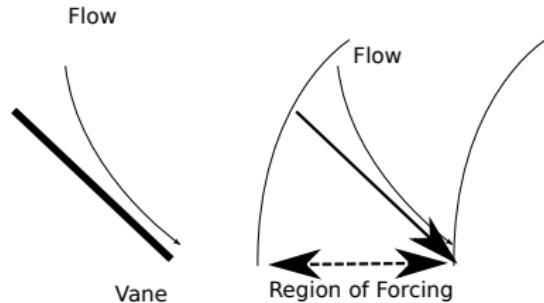
Our interest is in the eddy viscosity, $u'v' = \nu_T \frac{\partial \bar{u}}{\partial z}$ where $\xi \rightarrow -\infty$,

$$\nu_T = \frac{1}{C_{\nu_T}} \left(\frac{q}{c_p \rho_0} \frac{g}{T_0} \right)^{\frac{1}{3}} z^{\frac{4}{3}} \quad \text{for } z \gg L_{M-O}. \quad (9)$$

Turning Vane Representation

Virtual Vanes

- Requirements:
 - ▶ Very general formulation
 - ▶ No re-meshing
- Volumetric forcing:
 - ▶ Vane surface not represented
 - ▶ Similar to actuator-disk



Formulation

- Unit normal to vane surface,

$$\mathbf{n}(\mathbf{x}) = \sin(\phi(r)) \hat{\mathbf{r}} + \cos(\phi(r)) \hat{\theta} \quad (10)$$

- Apply \mathbf{f}_v to force flow parallel to vanes,

$$\mathbf{f}_v = -\frac{1}{\ell_v} |\mathbf{u}| (\mathbf{u} \cdot \mathbf{n}) \mathbf{n} \quad (11)$$

Numerics

Formulation

- Using Galerkin FEM discretization (N-S in weak form)
- Stabilized with SUPG stabilization *a la* Hughes/Becker+Braack
 - ▶ Adjoint stabilization scheme (τ)
 - ▶ Circumvents Babuska-Brezzi (we are using linear elements)
- Time discretization via backward Euler (if not steady)
- Newton method used to solve resulting non-linear implicit system

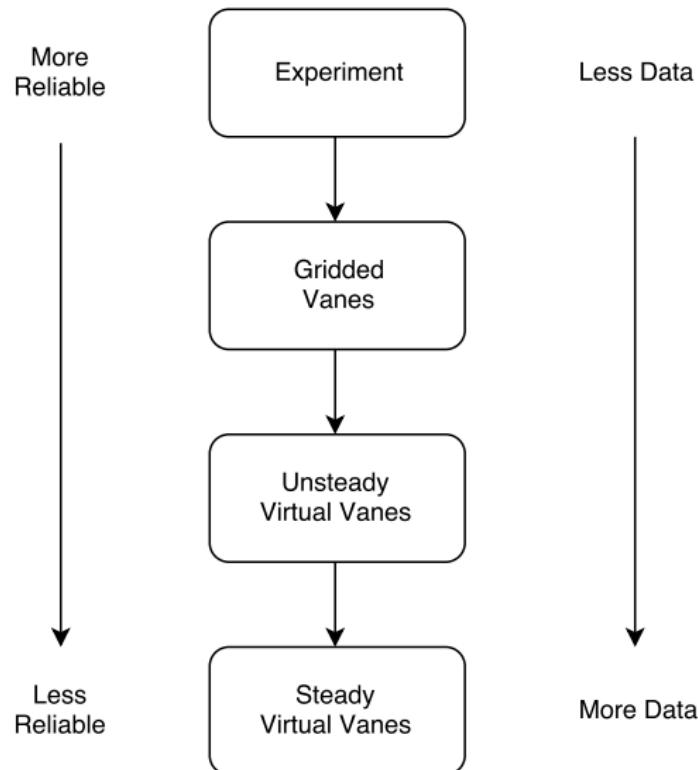
Software

- Using GRINS(+Libmesh) library developed by Bauman and Stogner

Hardware

- Most of the runs performed on LS4,LS5, Stampede at TACC
- 264-528 processing cores
- Several million degrees of freedom for each run, locally $O(10^4)$

Modeling Hierarchy



Steady vs. Transient Virtual Vanes

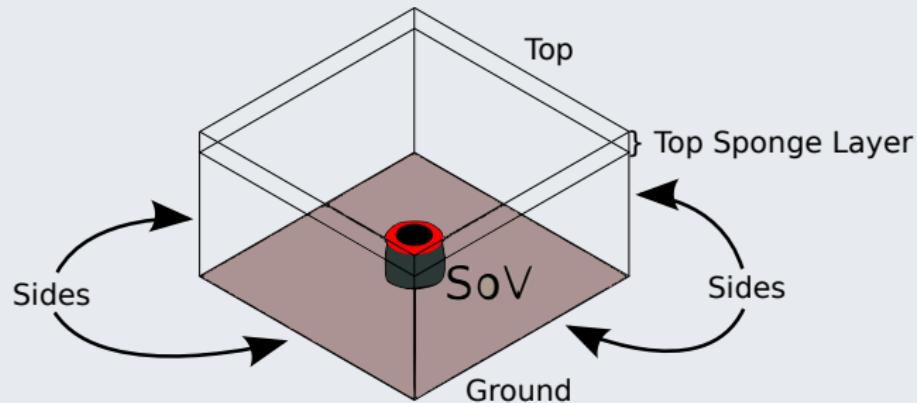
Unsteady VV

- Transient, like V-LES
 - ▶ captures dynamics of plume and wake
- Solutions temporally averaged
- Run-time ≈ 12 hours

Steady VV

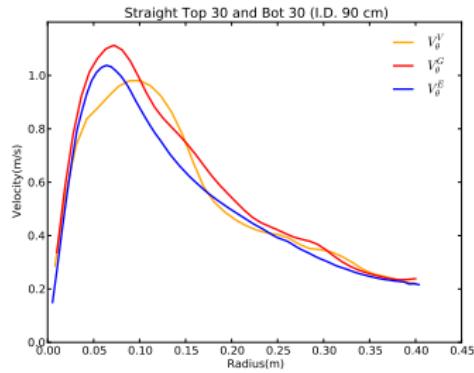
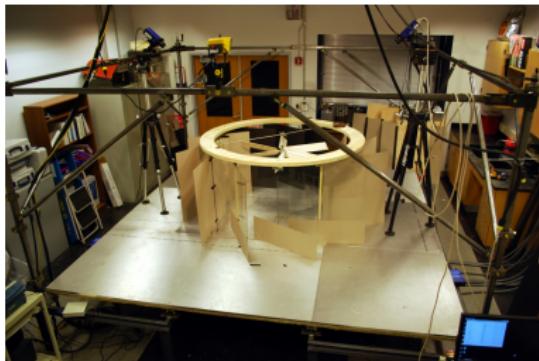
- Steady, like RANS (no $\frac{\partial(\cdot)}{\partial t}$)
 - ▶ no dynamics
- Run-time ≈ 2 minutes
- Critical design tool

No Wind (Thermal-Only)



- Domain extents scaled by system diameter
- Sides are periodic
- Sponge layers are finite thickness high viscosity boundaries
- Top has special mixed B.C.

Tabletop Experiment Validation



Laboratory Comparison of Azimuthal Velocity

- Slightly different B.C.s (inflow)
- Straight, single-tier, thirty degree vanes
- Red: Gridded, Blue: Experiment, Gold: Virtual
- Gridded consistent with experimental data
- Virtual vane solution more diffuse, incorrect peak predict

Thermal-Only Virtual Vanes

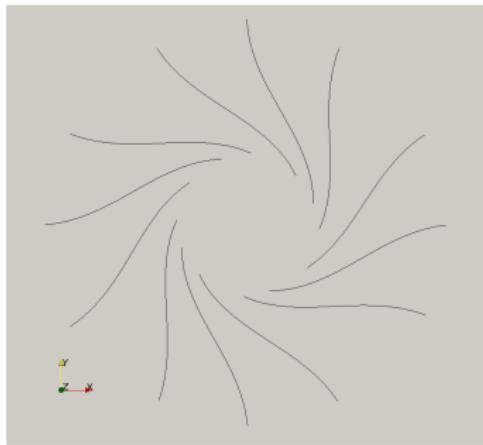


Figure: Bottom Vanes

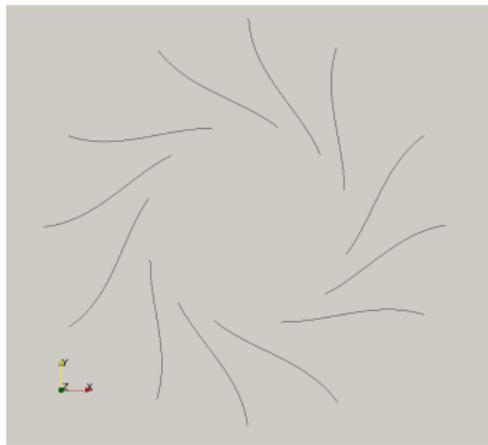


Figure: Top Vanes

Two Tier Configuration

- Curves drawn by advancing particle through forcing field
- No cone
- The top and bottom tier final angles are 70° and 85° , respectively

Probing the Solution: Vertical Slices

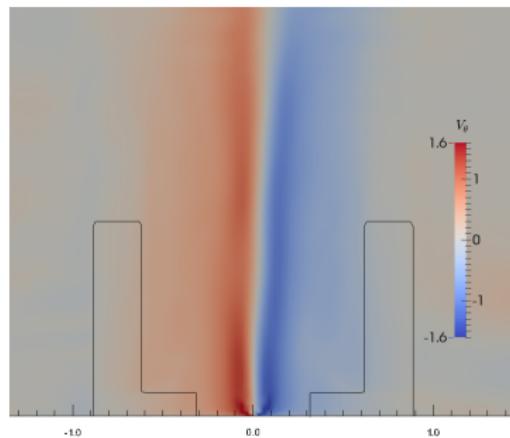


Figure: Azimuthal velocity

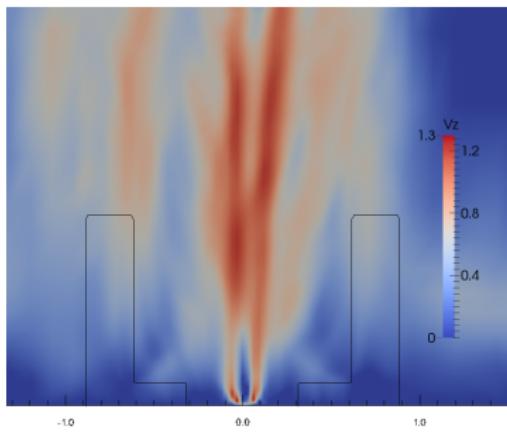
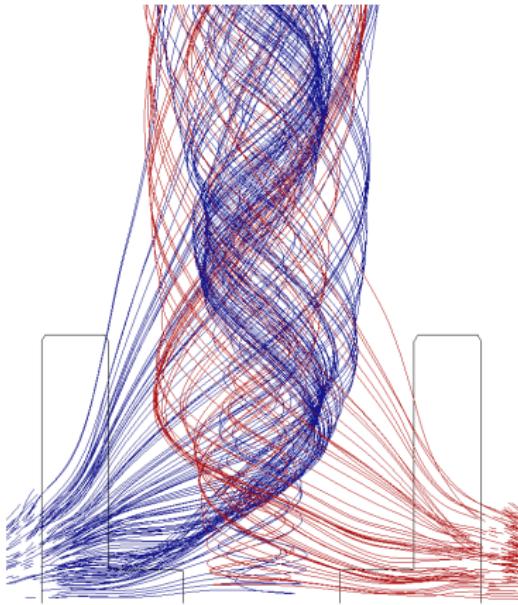


Figure: Vertical velocity

Velocity Observations

- Slices are temporally averaged virtual vane cases
- Virtual vane forcing region outlined in black
- Velocity highest in tight region near center

Solution structure



Particle Paths

- “Seed” particles outside of device
- RK4 to advance the particles through the velocity field

Probing the Solution: Vertical Slices

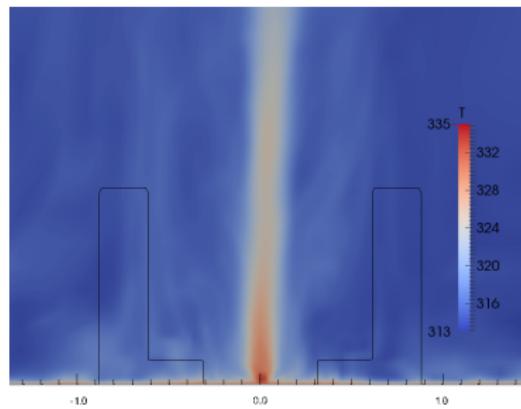


Figure: Temperature

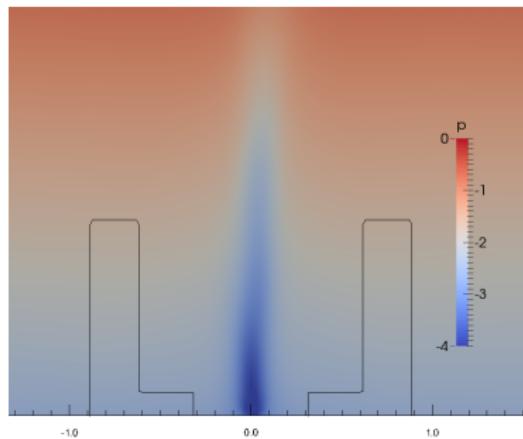


Figure: Pressure

Pressure and Temperature Observations

- Tight velocity core contains hot, low pressure “eye”

Probing the Solution: Horizontal Slices

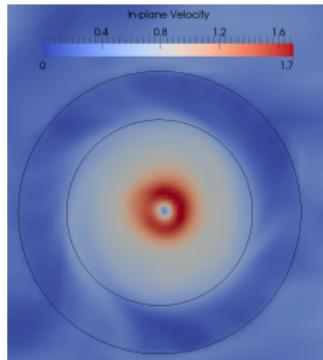


Figure: In-plane Velocity

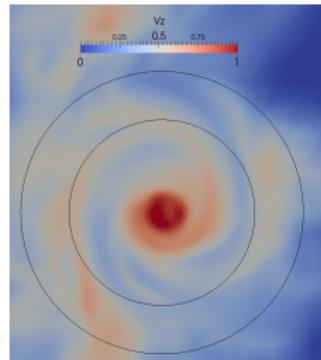


Figure: Vertical Velocity

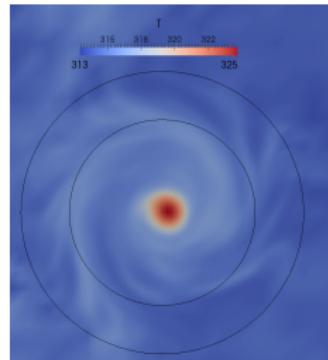
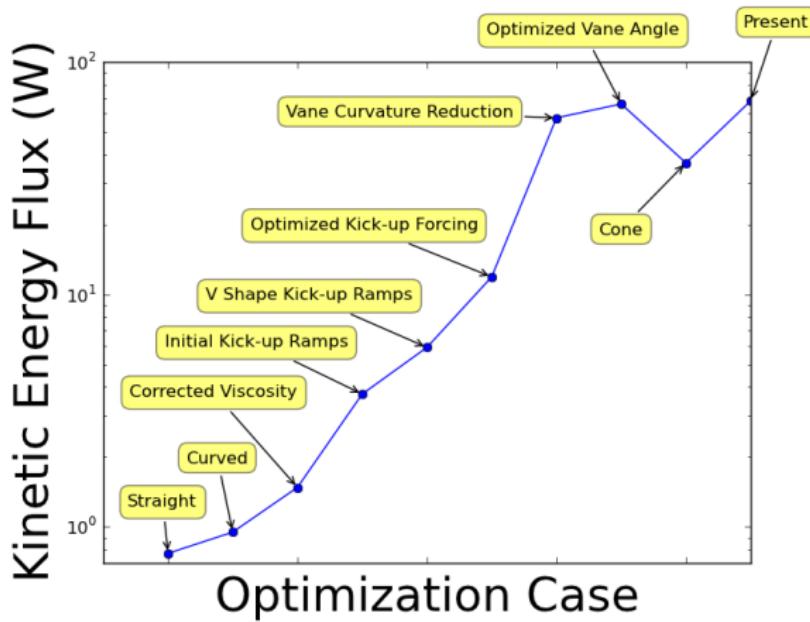


Figure: Temperature

Observations

- Slice taken at the height of the 2nd tier of vanes

Optimizing the Thermal-Only Configuration



Kinetic energy flux,

$$\dot{E} = -\frac{\rho}{2} \int V_z(V_\theta^2 + V_z^2)dA$$

- 78X increase in flux!
- Performed by exploration of design space
- flux < 100 Watts

Estimate of Energy Scaling

- Medium size dust devil: $3\text{m} = D$, $U = 5 \text{ m/s}$, $\Delta T = 30 \text{ K}$, 3 meters tall
- Assume a turbulent boundary layer ($\delta = 10 \text{ cm}$),

$$u(z) = U \min \left(\left(\frac{z}{\delta} \right)^7, 1 \right)$$

Boussinesq potential energy flux over the upstream flow [Renno]:

$$E_p = \int u(z)(\rho(z) - \rho_\infty)gz dA$$

$$E_p = g\pi R\beta\rho_0 U \Delta T \left[\frac{z_{\max}^2}{2} - \frac{7\delta^2}{18} \right]$$

$= 103 \text{ Watts}$

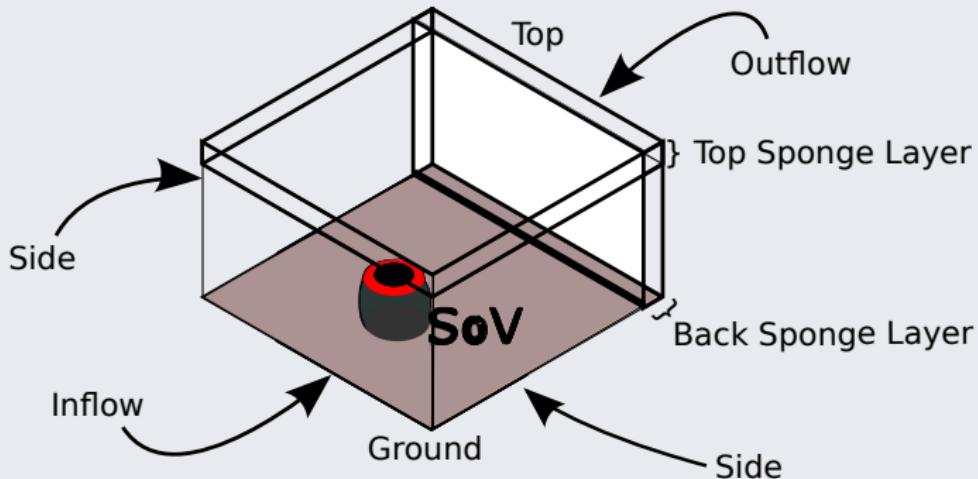
The KE flux: surface integral over upstream face of dust devil:

$$\text{KE} = \int \frac{\vec{V}^2}{2} \rho \vec{V} \cdot \hat{n} dA$$

$$\text{KE} = R\rho U^3 \left[z_{\max} - \frac{10}{11}\delta \right]$$

$= 1144 \text{ Watts}$

Wind Cases



- Specified Inflow (Dirichlet)
- Top and Outflow have special mixed B.C.
- Sponge layers on Top and Outflow
- Natural (do nothing) boundaries on the side (Neumann)

Simulation Geometry and Boundary Conditions, cont.

Specified Inflow

- 7th order turbulent boundary layer,

$$u_{\text{in}}(z) = U \min \left(\left(\frac{z}{\delta} \right)^7, 1 \right)$$

- The thermal inflow is then,

$$T_{\text{in}}(z) = \Delta T \left(1 - \min \left(\left(\frac{z}{\delta} \right)^7, 1 \right) \right) + T_0 - 2z/3.$$

Ground

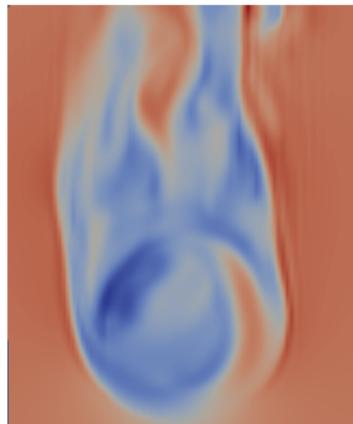
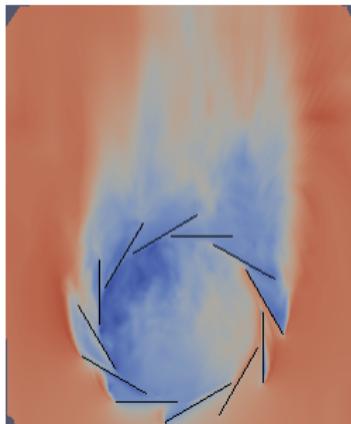
- modeled with a Dirichlet boundary condition such that,

$$\vec{u} = 0 \quad \text{on } \Gamma_G$$

$$T = T_g$$

Cold Wind Tunnel Validation

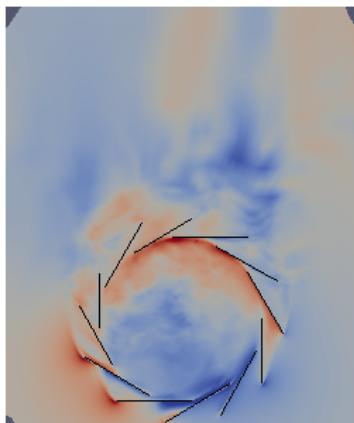
Streamwise Velocity



Horizontal slice (midway through the 60° vanes)

- Constant velocity wind, no temperature gradient
- Gridded vanes on left, Virtual vanes on right
- Flow penetrates vane region where aligned
- Results qualitatively agree with wind tunnel observations

Wind Tunnel: Spanwise Velocity



Horizontal slice (midway through the 60° vanes)

- Gridded Vanes on left, Virtual Vanes on right
- Not particularly accurate in the wake region
- Virtual vanes reproduce direction and magnitude of spanwise velocity
- Flow does not penetrate out back of device

Simulation of August Field Test



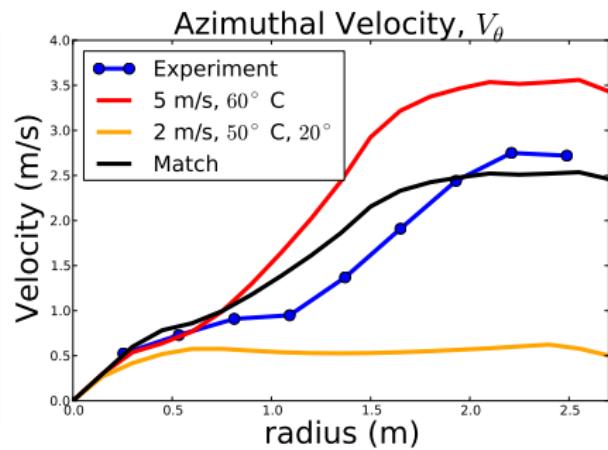
Model Problem

- Emulate field conditions in Arizona, August 2015
- Hot ground, ambient winds
- Lower tier, upper tier, and cone

Field Validation Study

Scenario parameter uncertainty

- Wind speed: 2-5 m/s
- Wind direction($\approx 20^\circ$)
- $T_s = 50 - 60^\circ \text{ C} (\approx 121 - 140^\circ \text{ F})$
- No boundary layer temperature data (DAQ equipment malfunctioned)



- CFD broadly consistent with field observations
 - ▶ Experimental data extremely limited
- Kinetic energy fluxes match to within 10%

Horizontal images at height of the vanes

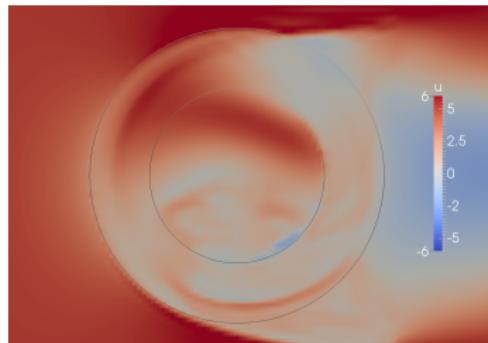


Figure: Streamwise Velocity

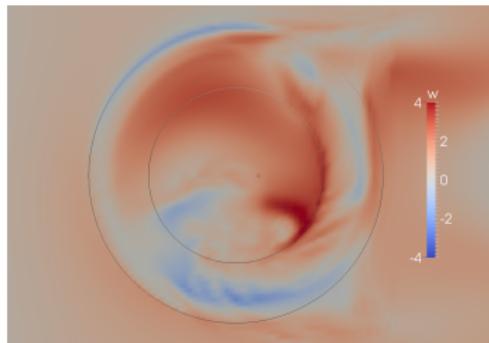
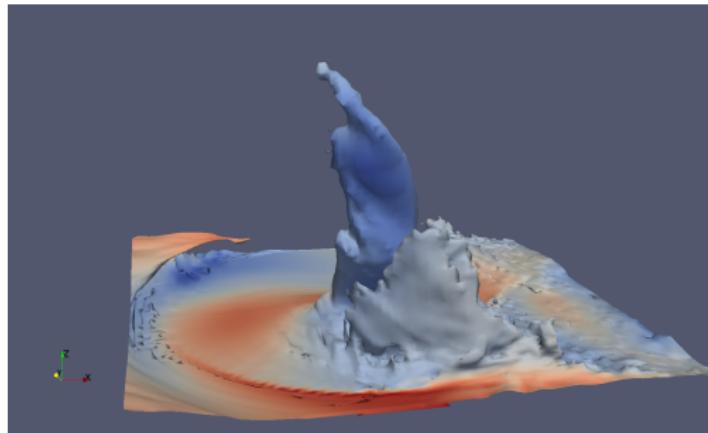


Figure: Vertical Velocity

Observations

- Freestream velocity is 3 m/s
- $\Delta T = 32$ K
- Some rotation visible in center

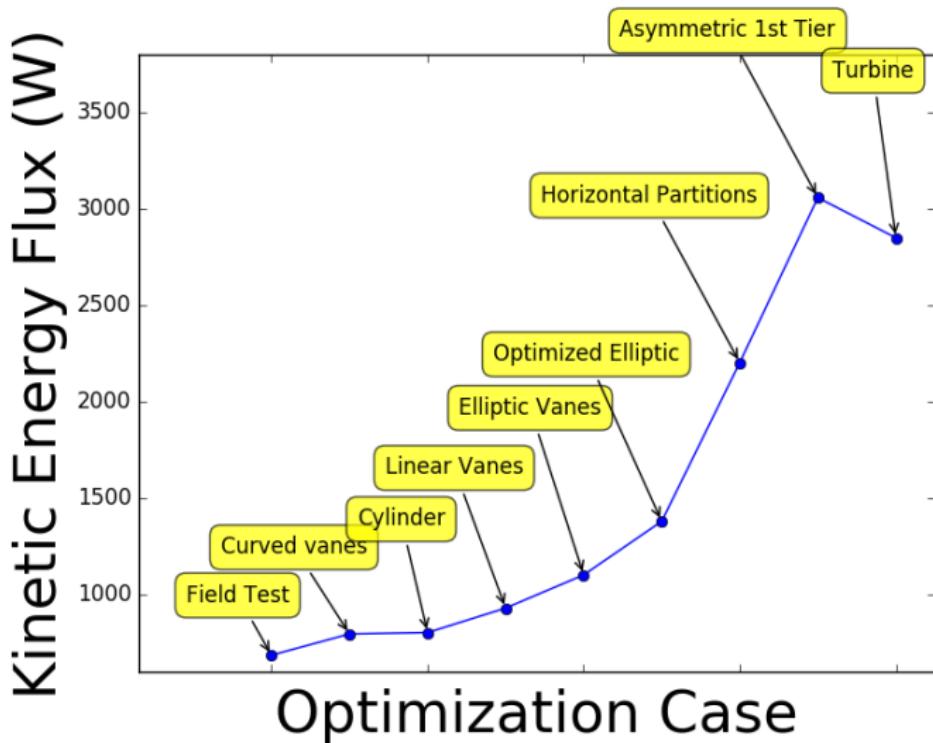
Temperature Plume



Temperature isocontour

- Potential temperature, $\tau(x, y, z) = T(x, y, z) - T_{\text{in}}(z)$
- Iso-contour corresponds to $\tau = 3K$ [Sinclair]
- Colored by streamwise velocity
- 600 Watts of Kinetic Energy Flux

Optimization of the apparatus for Field



Present Field Configuration



Figure: Bottom Vanes



Figure: Top Vanes

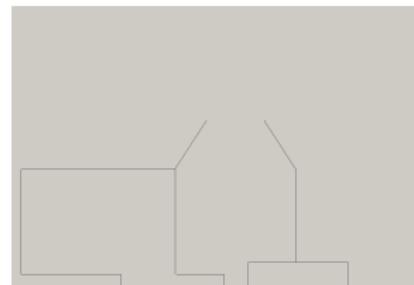


Figure: Vertical Slice

Two Tier Configuration

- Highly asymmetric second tier
- Vanes arranged to align with incoming wind
- Bottom tier asymmetric in height and angle

Present Field Results

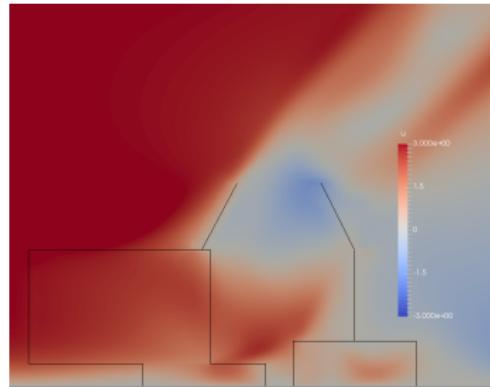


Figure: Streamwise velocity

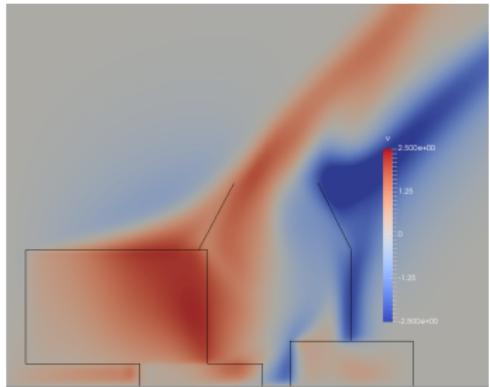


Figure: out of plane velocity

Observations

- Configuration space explored with steady Virtual Vanes
- $\approx 3 \text{ kW KE flux}$

Conclusions

Summary

- Developed modeling approach for domain of interest
- Simulation results consistent with available measurements
- Solution structure is cyclonic, dust devil-like

Future Work

- Final optimization of apparatus
- Turbine drag polar optimization
- Field test summer 2016
- Assessment of technological feasibility
- Connection to natural phenomena (Rankine Vortex?)

Shifting Gears– What is Scientific Software Verification?

Reality



↓ (Validation)

Mathematical Model

$$\frac{d^2x(t)}{dt^2} = \frac{F}{M}$$

↓ (Verification)

Numerical Representation

$$\frac{d^2x}{dt^2} = f''(t) = \frac{f(t+h) - 2f(t) + f(t-h)}{h^2}$$

Methods of Verification

Code Verification

- Code verification: Ensuring that the code used in the simulation correctly implements the intended numerical discretization of the model.

Method of Exact Solutions

- Numerically solve the governing equations for which the solution can be determined analytically.

Method of Manufactured Solutions

- Often, analytical solutions either:
 - ▶ Do not exist (Navier-Stokes)
 - ▶ Do not fully exercise equations (e.g. a symmetric solution, nonlinearities)
- Alleviate this using Method of Manufactured Solutions (MMS)
 - ▶ Simply put, we “create” our own solutions

Manufactured solution to Laplace's Equation

Laplace's Equation:

$$\nabla^2 \phi = 0$$

In two dimensions:

$$\frac{\partial^2 \phi}{\partial x^2} + \frac{\partial^2 \phi}{\partial y^2} = 0$$

“Manufacture” a solution, with two constants:

$$\phi(x, y) = (\textcolor{red}{Ly} - y)^2(\textcolor{red}{Ly} + y)^2 + (\textcolor{red}{Lx} - x)^2(\textcolor{red}{Lx} + x)^2$$

Calculating the Source Term

We insert our manufactured solution back into the governing equations:

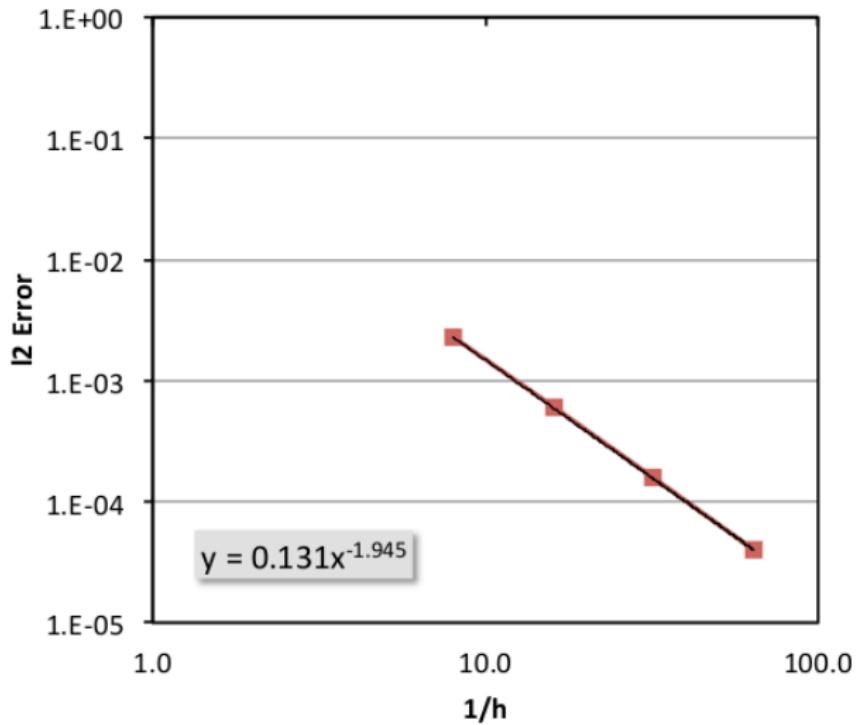
$$\frac{\partial^2((Lx - x)^2(Lx + x)^2)}{\partial x^2} + \frac{\partial^2((Ly - y)^2(Ly + y)^2)}{\partial y^2} = 0$$

$$\begin{aligned} &= 2(Lx - x)^2 - 8(Lx - x)(Lx + x) + 2(Lx + x)^2 \\ &+ 2(Ly - y)^2 - 8(Ly - y)(Ly + y) + 2(Ly + y)^2 \\ &\neq 0 \end{aligned}$$

This does not satisfy Laplace's Equation!

To balance the equation, add the residual to the RHS as a source term.

Example Results: What we're hoping for 2nd Order Central Finite-difference Scheme



Useful for Detecting Subtle Bugs

Verification of FIN-S

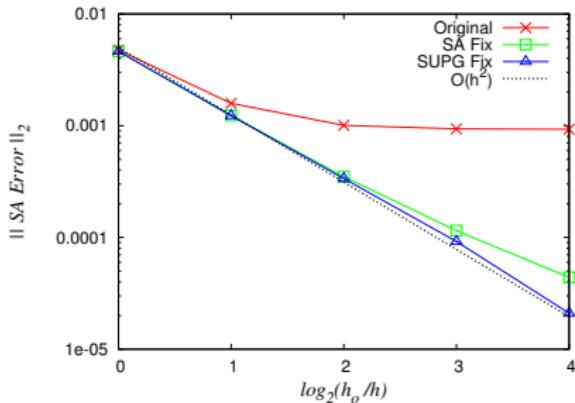
- FANS, Spalart-Allmaras

- Derivative:

$$\frac{d(sa)}{dx} = \frac{1}{\rho} * \left(\frac{d(\rho * sa)}{dx} - sa \frac{d\rho}{dx} \right)$$

- In code:

$$\frac{d(sa)}{dx} = \frac{1}{\rho} * \frac{d(\rho * sa)}{dx} - sa \frac{d\rho}{dx}$$



A Real Example

MMS Creation Process

- Start by “manufacturing” a suitable closed-form exact solution
- For example, the 10 parameter trigonometric solution of the form:
(Roy, 2002)

$$\hat{u}(x, y, z, t) = \hat{u}_0 + \hat{u}_x f_s\left(\frac{a_{\hat{u}x}\pi x}{L}\right) + \hat{u}_y f_s\left(\frac{a_{\hat{u}y}\pi y}{L}\right) + \\ + \hat{u}_z f_s\left(\frac{a_{\hat{u}z}\pi z}{L}\right) + \hat{u}_t f_s\left(\frac{a_{\hat{u}t}\pi t}{L}\right)$$

- Apply this solution to equations of interest, solve for source terms (residual)

Accomplished using symbolic manipulation SymPy, Maple, Mathematica, Macsyma, etc.

Maple MMS: 3D Navier-Stokes Energy Term

$$\begin{aligned}
Qe = & - \frac{a_{px}\pi p_x}{L} \frac{\gamma}{\gamma - 1} \sin\left(\frac{a_{px}\pi x}{L}\right) \left[u_0 + u_x \sin\left(\frac{a_{ux}\pi x}{L}\right) + u_y \cos\left(\frac{a_{uy}\pi y}{L}\right) + u_z \cos\left(\frac{a_{uz}\pi z}{L}\right) \right] + \\
& + \frac{a_{py}\pi p_y}{L} \frac{\gamma}{\gamma - 1} \cos\left(\frac{a_{py}\pi y}{L}\right) \left[v_0 + v_x \cos\left(\frac{a_{vx}\pi x}{L}\right) + v_y \sin\left(\frac{a_{vy}\pi y}{L}\right) + v_z \sin\left(\frac{a_{vz}\pi z}{L}\right) \right] + \\
& - \frac{a_{pz}\pi p_z}{L} \frac{\gamma}{\gamma - 1} \sin\left(\frac{a_{pz}\pi z}{L}\right) \left[w_0 + w_x \sin\left(\frac{a_{wx}\pi x}{L}\right) + w_y \sin\left(\frac{a_{wy}\pi y}{L}\right) + w_z \cos\left(\frac{a_{wz}\pi z}{L}\right) \right] + \\
& + \frac{a_{px}\pi p_x}{2L} \cos\left(\frac{a_{px}\pi x}{L}\right) \left[u_0 + u_x \sin\left(\frac{a_{ux}\pi x}{L}\right) + u_y \cos\left(\frac{a_{uy}\pi y}{L}\right) + u_z \cos\left(\frac{a_{uz}\pi z}{L}\right) \right] \left[\left(u_0 + u_x \sin\left(\frac{a_{ux}\pi x}{L}\right) + u_y \cos\left(\frac{a_{uy}\pi y}{L}\right) + u_z \cos\left(\frac{a_{uz}\pi z}{L}\right) \right)^2 + \right. \\
& \quad \left. + \left[w_0 + w_x \sin\left(\frac{a_{wx}\pi x}{L}\right) + w_y \sin\left(\frac{a_{wy}\pi y}{L}\right) + w_z \cos\left(\frac{a_{wz}\pi z}{L}\right) \right]^2 + \left[v_0 + v_x \cos\left(\frac{a_{vx}\pi x}{L}\right) + v_y \sin\left(\frac{a_{vy}\pi y}{L}\right) + v_z \sin\left(\frac{a_{vz}\pi z}{L}\right) \right]^2 \right] + \\
& - \frac{a_{py}\pi p_y}{2L} \sin\left(\frac{a_{py}\pi y}{L}\right) \left[v_0 + v_x \cos\left(\frac{a_{vx}\pi x}{L}\right) + v_y \sin\left(\frac{a_{vy}\pi y}{L}\right) + v_z \sin\left(\frac{a_{vz}\pi z}{L}\right) \right] \left[\left(u_0 + u_x \sin\left(\frac{a_{ux}\pi x}{L}\right) + u_y \cos\left(\frac{a_{uy}\pi y}{L}\right) + u_z \cos\left(\frac{a_{uz}\pi z}{L}\right) \right)^2 + \right. \\
& \quad \left. + \left[w_0 + w_x \sin\left(\frac{a_{wx}\pi x}{L}\right) + w_y \sin\left(\frac{a_{wy}\pi y}{L}\right) + w_z \cos\left(\frac{a_{wz}\pi z}{L}\right) \right]^2 + \left[v_0 + v_x \cos\left(\frac{a_{vx}\pi x}{L}\right) + v_y \sin\left(\frac{a_{vy}\pi y}{L}\right) + v_z \sin\left(\frac{a_{vz}\pi z}{L}\right) \right]^2 \right] + \\
& + \frac{a_{pz}\pi p_z}{2L} \cos\left(\frac{a_{pz}\pi z}{L}\right) \left[w_0 + w_x \sin\left(\frac{a_{wx}\pi x}{L}\right) + w_y \sin\left(\frac{a_{wy}\pi y}{L}\right) + w_z \cos\left(\frac{a_{wz}\pi z}{L}\right) \right] \left[\left(u_0 + u_x \sin\left(\frac{a_{ux}\pi x}{L}\right) + u_y \cos\left(\frac{a_{uy}\pi y}{L}\right) + u_z \cos\left(\frac{a_{uz}\pi z}{L}\right) \right)^2 + \right. \\
& \quad \left. + \left[w_0 + w_x \sin\left(\frac{a_{wx}\pi x}{L}\right) + w_y \sin\left(\frac{a_{wy}\pi y}{L}\right) + w_z \cos\left(\frac{a_{wz}\pi z}{L}\right) \right]^2 + \left[v_0 + v_x \cos\left(\frac{a_{vx}\pi x}{L}\right) + v_y \sin\left(\frac{a_{vy}\pi y}{L}\right) + v_z \sin\left(\frac{a_{vz}\pi z}{L}\right) \right]^2 \right] + \\
& + \frac{a_{ux}\pi u_x}{2L} \cos\left(\frac{a_{ux}\pi x}{L}\right) \left(\left[\left(u_0 + w_x \sin\left(\frac{a_{wx}\pi x}{L}\right) + w_y \sin\left(\frac{a_{wy}\pi y}{L}\right) + w_z \cos\left(\frac{a_{wz}\pi z}{L}\right) \right)^2 + \left[v_0 + v_x \cos\left(\frac{a_{vx}\pi x}{L}\right) + v_y \sin\left(\frac{a_{vy}\pi y}{L}\right) + v_z \sin\left(\frac{a_{vz}\pi z}{L}\right) \right]^2 \right. \right. + \\
& \quad \left. \left. + 3 \left[u_0 + u_x \sin\left(\frac{a_{ux}\pi x}{L}\right) + u_y \cos\left(\frac{a_{uy}\pi y}{L}\right) + u_z \cos\left(\frac{a_{uz}\pi z}{L}\right) \right]^2 \right] \left[\rho_0 + \rho_x \sin\left(\frac{a_{px}\pi x}{L}\right) + \rho_y \cos\left(\frac{a_{py}\pi y}{L}\right) + \rho_z \sin\left(\frac{a_{pz}\pi z}{L}\right) \right] + \right. \\
& \quad \left. + \left[p_0 + p_x \cos\left(\frac{a_{px}\pi x}{L}\right) + p_y \sin\left(\frac{a_{py}\pi y}{L}\right) + p_z \cos\left(\frac{a_{pz}\pi z}{L}\right) \right] \frac{2\gamma}{(\gamma - 1)} \right) + \\
& - \frac{a_{uy}\pi u_y}{L} \sin\left(\frac{a_{uy}\pi y}{L}\right) \left[v_0 + v_x \cos\left(\frac{a_{vx}\pi x}{L}\right) + v_y \sin\left(\frac{a_{vy}\pi y}{L}\right) + v_z \sin\left(\frac{a_{vz}\pi z}{L}\right) \right] \left[\rho_0 + \rho_x \sin\left(\frac{a_{px}\pi x}{L}\right) + \rho_y \cos\left(\frac{a_{py}\pi y}{L}\right) + \rho_z \sin\left(\frac{a_{pz}\pi z}{L}\right) \right] \cdot \\
& \quad \cdot \left[u_0 + u_x \sin\left(\frac{a_{ux}\pi x}{L}\right) + u_y \cos\left(\frac{a_{uy}\pi y}{L}\right) + u_z \cos\left(\frac{a_{uz}\pi z}{L}\right) \right] + \\
& - \frac{a_{uz}\pi u_z}{L} \sin\left(\frac{a_{uz}\pi z}{L}\right) \left[w_0 + w_x \sin\left(\frac{a_{wx}\pi x}{L}\right) + w_y \sin\left(\frac{a_{wy}\pi y}{L}\right) + w_z \cos\left(\frac{a_{wz}\pi z}{L}\right) \right] \left[\rho_0 + \rho_x \sin\left(\frac{a_{px}\pi x}{L}\right) + \rho_y \cos\left(\frac{a_{py}\pi y}{L}\right) + \rho_z \sin\left(\frac{a_{pz}\pi z}{L}\right) \right] \cdot \\
& \quad \cdot \left[u_0 + u_x \sin\left(\frac{a_{ux}\pi x}{L}\right) + u_y \cos\left(\frac{a_{uy}\pi y}{L}\right) + u_z \cos\left(\frac{a_{uz}\pi z}{L}\right) \right] +
\end{aligned}$$

But wait, there's more!

C-code output

Manufactured Analytical Solutions Abstractions Library

Goal: Provide a repository and standardized interface for MMS usage

High Priority:

- Extreme fidelity to generated MMS
- Portability
- Traceability
- Extensible

Low Priority:

- Speed/Performance

Verifying the “Verifier”

Precision is not negotiable.

MASA Testing

- Error target < 1e-15
 - ▶ Absolute error on local machines
 - ▶ Relative error (other)
 - ▶ On all supported compiler sets
- -O0 not sufficient
 - ▶ -fp-model precise (Intel)
 - ▶ -fno-unsafe-math-optimizations (GNU)
 - ▶ -Kieee -Mnofpapprox (PGI)
- “make check”
 - ▶ Run by Buildbot every two hours

```
[nick@magus trunk]$ make check
```

```
-----
```

```
Initializing MASA Tests
```

```
-----
```

```
PASS: init.sh
PASS: misc
PASS: fail_cond
PASS: catch_exception
PASS: register
PASS: poly
PASS: uninit
PASS: vec
PASS: purge
PASS: heat_const_steady
PASS: euler1d
```

```
: : :
```

```
-----
```

```
Finalizing MASA Tests
```

```
-----
```

```
=====
```

```
All 65 tests passed
```

```
=====
```

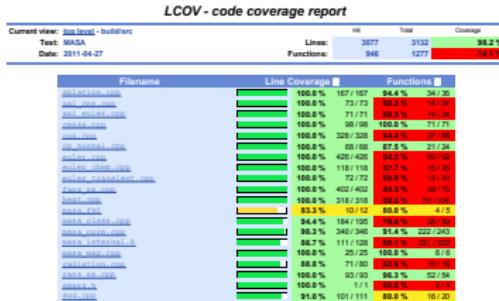
Software Library Snapshot

Software Environment

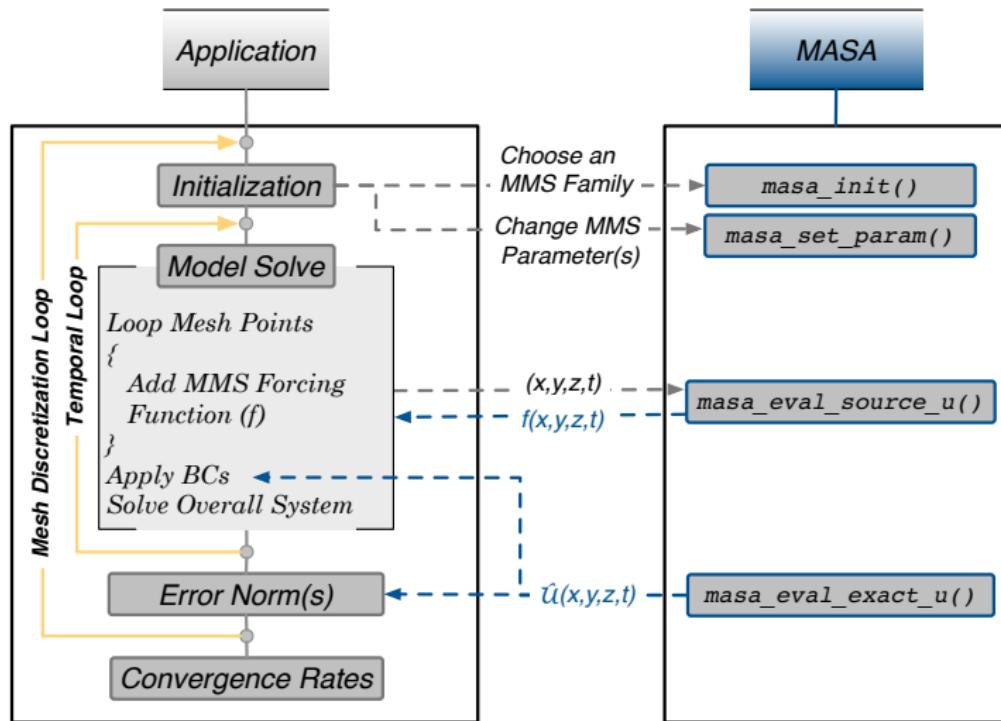
- Built with: Autotools, C++
- Supports Intel, GNU, Portland Group compilers
- C/C++/Fortran/Python interfaces
- Python interfaces generated with **SWIG**

Testing

- GIT: version control (github)
- TravisCI: automated testing
- GCOV: line coverage
 - ▶ 15,826 lines of code
 - ▶ 13,195 lines of testing
 - ▶ 98%+ line coverage



General Verification Approach Using MMS and MASA



C: What you need from MASA

```
#include <masa.h>

int main()
{
    err += masa_init("laplace example","laplace_2d");

    // grab / set parameter values
    Lx = masa_get_param("Lx");
    masa_set_param("Ly",42.0);

    for(int i=0;i<nx;i++)
        for(int j=0;j<nx;j++)
    {
        x=i*dx;
        y=j*dy;

        // source term
        ffield = masa_eval_2d_source_f (x,y);

        // manufactured solution
        phi_field = masa_eval_2d_exact_phi(x,y);

    } // finished iterating over space
} //end program
```

Available Solutions in MASA 0.44.0

Equations	Dimensions	Time
Euler	1,2,3, axi	Transient, Steady
Non-linear heat conduction	1,2,3	Transient, Steady
Navier-Stokes	1,2,3, axi	Transient, Steady
N-S + Sutherland	3	Transient, Steady
N-S + ablation	1	Transient, Steady
Burgers	2	Transient, Steady
Sod Shock Tube	1	Transient
Euler + chemistry	1	Steady
RANS: Spalart-Allmaras	1	Steady
FANS: SA	2	Steady
FANS: SA + wall	2	Steady
Radiation	1	Steady
SMASA: Gaussian	1	Steady

Or users can import own solutions.

Snapshot

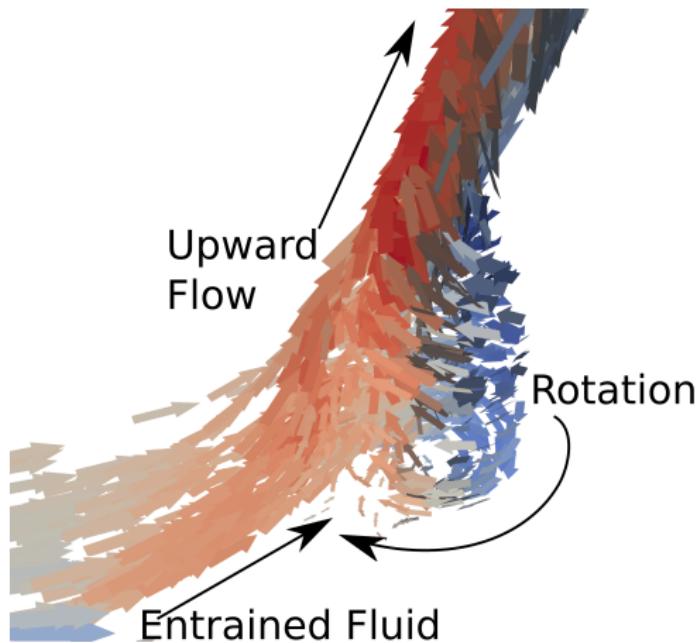
Release

- MASA 0.44.0 current release
- <https://github.com/manufactured-solutions/MASA>
- “Computational Physics Student Summer Workshop” at LANL (2011, 2012, 2013, 2014, 2016).

Publications

- “MASA: a library for verification using manufactured analytical solutions”
- A transient manufactured solution for the compressible Navier-Stokes equations with a power law viscosity
- Manufactured Solutions for the Favre-Averaged Navier-Stokes Equations with Eddy-Viscosity Turbulence Models
- ”A linear regression model for verification of linear problems using Bayesian calibration“ (in prep)

The End



Thank you!

nick@ices.utexas.edu

MASA PDE Examples

Source Terms: Euler

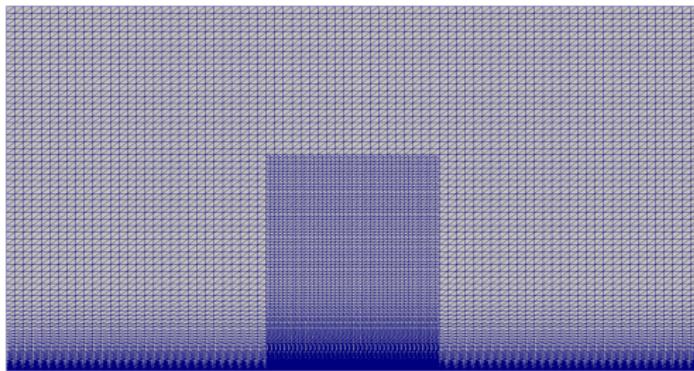
```
// Arbitrary manufactured solutions
U.template get<0>() = u_0 + u_x * sin(a_ux * PI * x / L)
+ u_y * cos(a_uy * PI * y / L);
```

$$\nabla \cdot (\rho u) = 0$$

$$\nabla \cdot (eu) + p \nabla \cdot u = 0$$

```
// Mass, momentum and energy
Scalar Q_rho    = raw_value(divergence(RHO*U));
RawArray Q_rho_u = raw_value(divergence(RHO*U.outerproduct(U)) +
                           P.derivatives());
Scalar Q_rho_e = raw_value(divergence((RHO*ET+P)*U));
```

Mesh



Discretization

- Single refinement in vane region
-

$$\text{Re}_{\text{cell}} = \frac{\max(\Delta x, \Delta y) u}{\nu_T}$$

- Boundary layer mesh visible

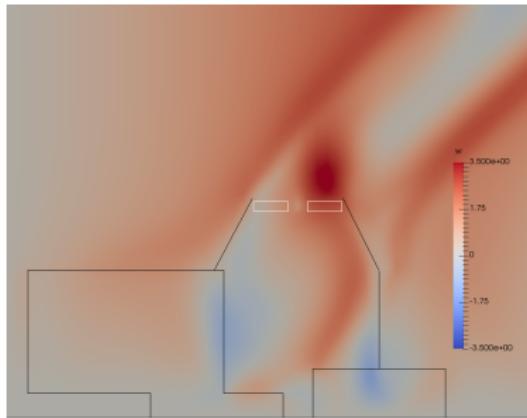
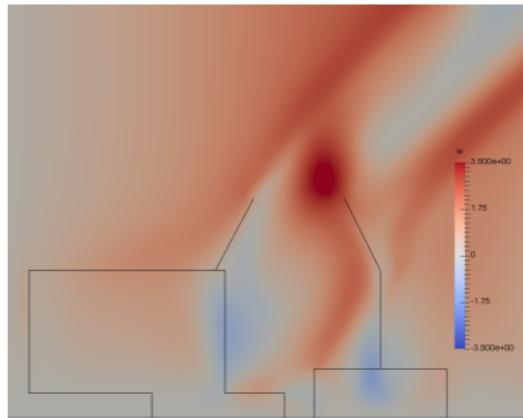
Turbine

Model

- Actuator-disk
- Implementation similar to virtual vanes
- 90° circular-arc lift and drag polars (C_D , C_L)
- 7 parameters (shown below)

Name	Symbol	Comments
Initial Angle of Attack	α_0	At R_G
Final Angle of Attack	α_f	At L_B
Chord Length	C	Blade thickness
# of blades	A_B	
Center Gap	R_G	Gap in turbine
Base Velocity	ω	fixed tip velocity
Blade Radius	L_B	Length of blade
Height of turbine	H_B	Height of center of turbine

Vertical velocity field with and without turbine



Impact of the turbine (drawn in white)

- Modestly impacting flow, vortex structure largely intact
- Only extracting $\approx 30\%$ of energy
- Optimized drag polars would provide “best case” (see timeline)

Stabilization Outline

- Cast Navier Stokes + Boussinesq equations into weak form
- Prepare as operator $Lc = f$
- Calculate Fréchet derivative
- Separate into differential (P) and constant (Z) components,
 $L'[c] = P + Z$
- Choose stabilization operator such that $S = -P^*$
- Then stabilization has form, $a_h(c, \phi) = a(c, \phi) + \langle Lc, S\phi \rangle_\tau$