



EUROPEAN CENTRE FOR RESEARCH AND ADVANCED TRAINING IN **SCIENTIFIC COMPUTING**

Numerics

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Theoretical insights

- Mathematical background
- Numerical methods

Description of AVBP schemes

- Lax Wendroff
- Two-step Taylor Galerkin schemes
- Properties of AVBP schemes

Practical elements

- Wiggles
- Issues at corners
- Artificial viscosity
- *Shock sensors*



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Mathematical background

- Conservation law (**conservative** formulation)

$$\frac{\partial \mathbf{u}}{\partial t} + \frac{\partial}{\partial x} f(\mathbf{u}) = 0$$

with

$$\mathbf{u} = \begin{pmatrix} \rho \\ \rho u \\ \rho v \\ \rho w \\ \rho E \end{pmatrix}$$

- Non-conservative formulation

$$\frac{\partial \mathbf{u}}{\partial t} + \mathbf{a}(\mathbf{u}) \frac{\partial \mathbf{u}}{\partial x} = 0$$

with

$$\mathbf{a}(\mathbf{u}) = \frac{\partial f(\mathbf{u})}{\partial \mathbf{u}} \text{ Jacobian}$$

- The Cauchy problem (**initial value** problem)

$$\begin{cases} \frac{\partial \mathbf{u}}{\partial t} + \frac{\partial}{\partial x} f(\mathbf{u}) = 0 \\ \mathbf{u}(\mathbf{x}, 0) = \mathbf{u}_0 \end{cases}$$

- A solution $\mathbf{u}(x, t), \forall (x, t)$ **satisfying the Cauchy** problem is called a “**Strong**” solution (or “**Classical**”).
- Such solution **may not exit** (if discontinuity in the initial solution).
➔ For this reason, “**Weak**” solutions are of interest.



Weak solutions for conservation equations

Will be of interest for finite-elements ...

Let's consider a function $\phi(\mathbf{x}, t)$ defined on a compact support.

- Let's multiply the conservation equation :

$$\left(\frac{\partial \mathbf{u}}{\partial t} + \frac{\partial}{\partial x} f(\mathbf{u}) \right) \cdot \phi(\mathbf{x}, t) = 0$$

- Integration over space-time: *(Scalar product)*

$$\int_{\mathbb{R}} \int_0^{\infty} \left(\frac{\partial \mathbf{u}}{\partial t} + \frac{\partial}{\partial x} f(\mathbf{u}) \right) \cdot \phi(\mathbf{x}, t) \, d\mathbf{x} \, dt = 0$$

- Provided $\phi(\mathbf{x}, t)$ nullifies on the support boundaries, after **integration by parts**:

$$\int_{\mathbb{R}} \int_0^{\infty} \left(\mathbf{u} \frac{\partial \phi}{\partial t} + f(\mathbf{u}) \frac{\partial \phi}{\partial x} \right) \, dx \, dt + \int_{\mathbb{R}} \mathbf{u}_0 \phi(\mathbf{x}, 0) \, dx = 0$$

Variational formulation for the conservation equation

➔ No longer derivatives of $\mathbf{u}(\mathbf{x}, t)$!

➔ Derivatives of $\phi(\mathbf{x}, t)$ instead (mass-matrix in the FE formalism...)



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Finite Volumes and Finite Elements

Finite-volume is based on the **strong formulation** of the conservation equation

- **Conservation** of transported quantities in a **control volume**
- The unknown is approximated by the **mean** over the control volume

Finite elements rely on the concept of **variational formulation**, and **weak solutions**

- **Shape functions** are defined within each cell.
- Unknowns are determined at the **node** (*instead average value in cell for FV*)

Both approaches rely on :

- Space and time **discretization of the conservation equation**
- Integration over a control volume, **fluxes balance** over the control volume

$$\frac{\partial U}{\partial t} = -\frac{1}{V_{\Omega}} \int_{\Omega} \underbrace{\vec{\nabla} \cdot \vec{F}}_{R_{\Omega} \text{ Residual}} dV$$

➔ Requires to the computation of a **residual**



Let's focus on space and time discretization...

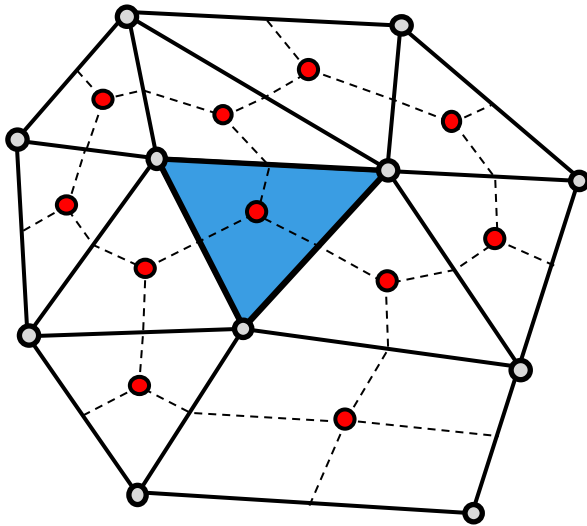
$$\frac{\partial \mathbf{u}}{\partial t} + \frac{\partial}{\partial x} f(\mathbf{u}) = 0$$

- The continuous conservation equation is discretized **on a mesh**
Allows to store variables at each iteration
Allows to define control volumes
- The continuous conservation equation is discretized **in time**

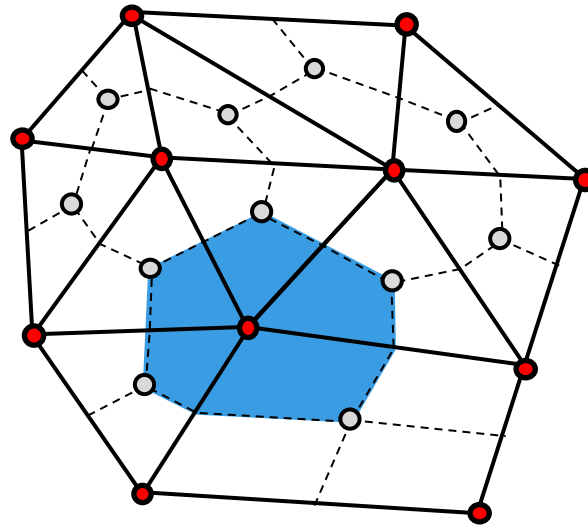


Spatial discretization

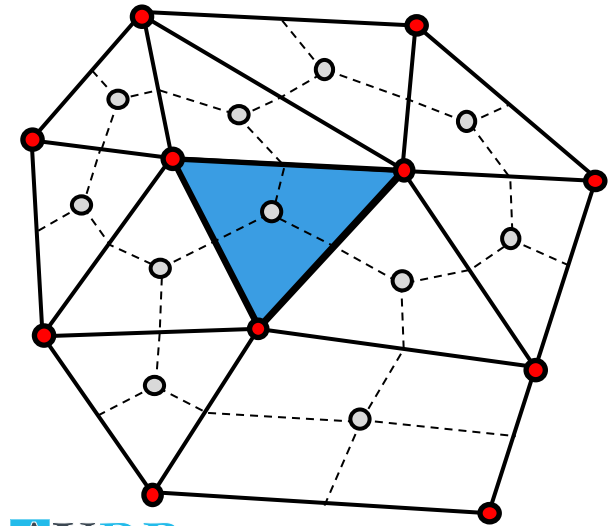
Cell-centred



Vertex-centred



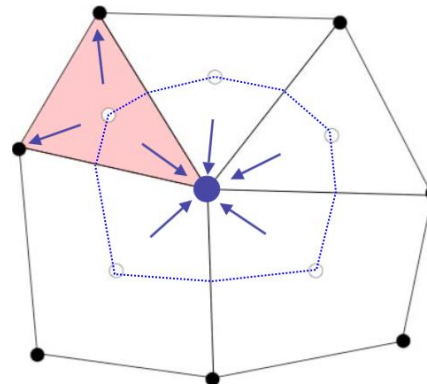
Cell-vertex



AVBP

Variables are stored at the nodes
Control volumes are primal cells

- Primary mesh
- - - (Median) dual mesh
- Variable storage
- Control volume





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Presentation of 2 main Numerical schemes of AVBP

- **Lax-Wendroff (LW)**
- **Two-Step Taylor Galerkin (TTG) schemes**




$$\frac{\partial \mathbf{u}}{\partial t} + \frac{\partial}{\partial x} f(\mathbf{u}) = 0$$

Summary

In summary, two families of schemes are available in AVBP:



- **LW**

Finite Volume

-  second order accurate in space and time
-  not expensive

- **TTG4A and TTGC**

Finite Element

-  third order accurate in space and time
- dissipation and dispersion properties meeting LES requirements
-  ~ 2.5 times more expensive than LW



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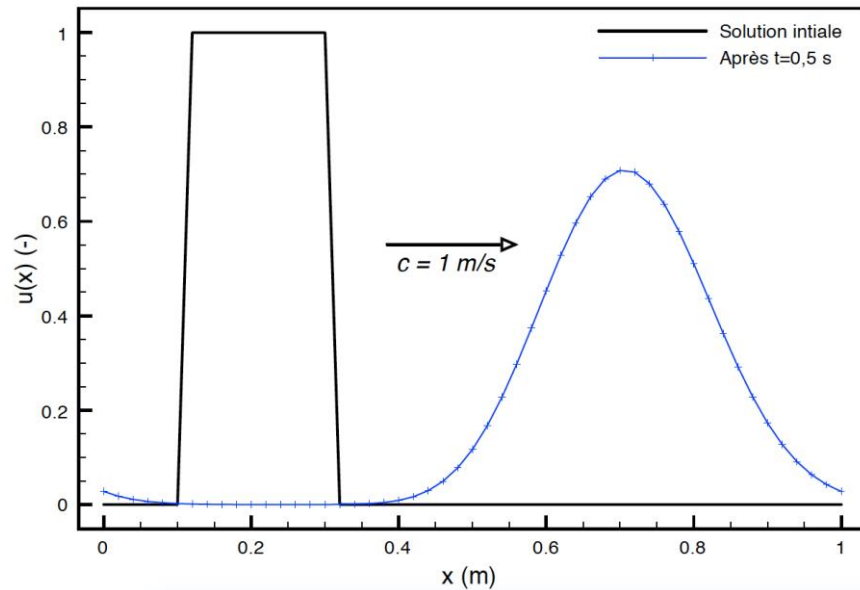
- Wiggles
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Properties of AVBP schemes

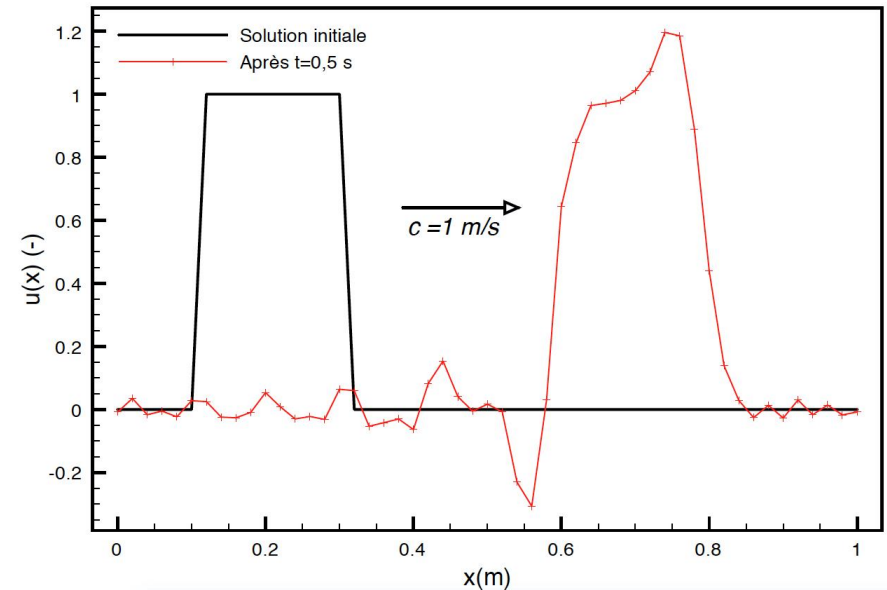
Mesh discretization introduces errors:

Dissipation error



- Decrease of the wave amplitude
- Smoothing of its gradients

Dispersion error



Dispersive medium:

A medium in which wave velocities depend on the wave frequency

➔ Harmonics of the initial solution are not transported at the same speed

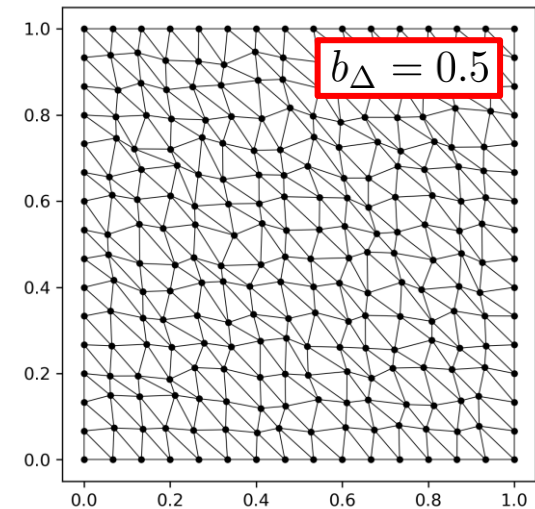
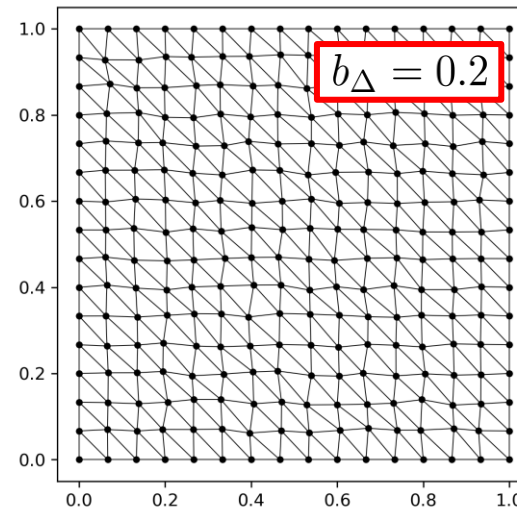
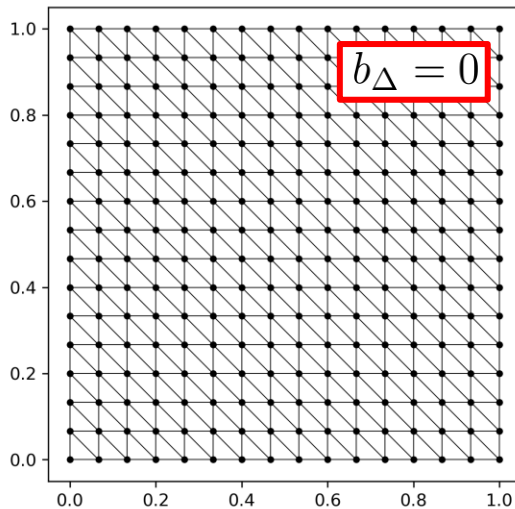
Application

Reference



Let's consider the advection of a 2D isotropic vortex...

Let's try several schemes, and several mesh qualities



Reference

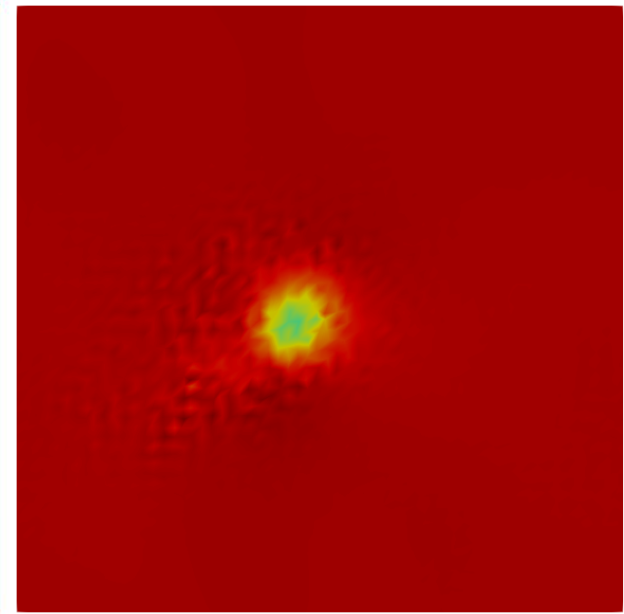
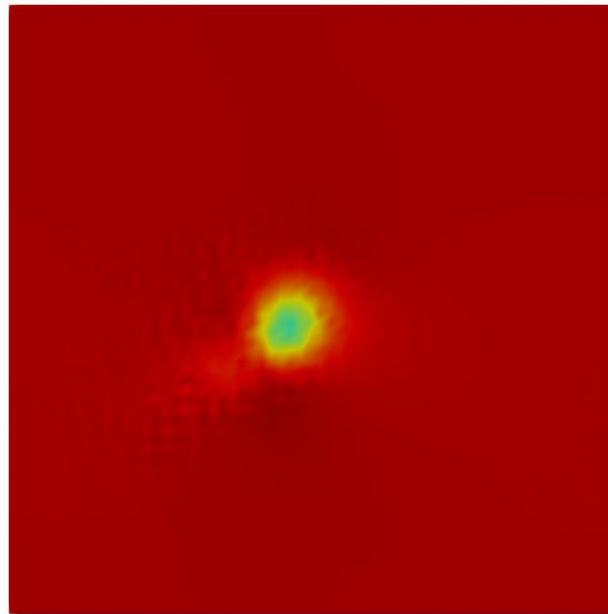
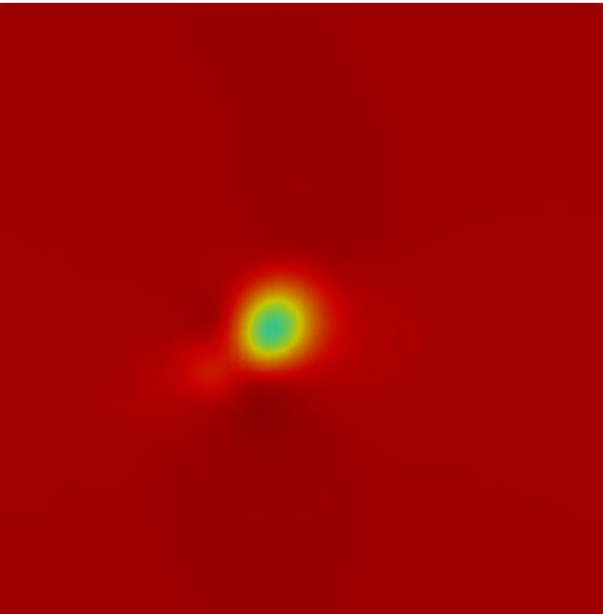
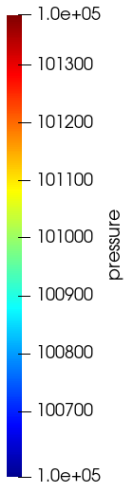
Lax-Wendroff

⇒ Dissipative
⇒ Some dispersion

$$b_{\Delta} = 0$$

$$b_{\Delta} = 0.2$$

$$b_{\Delta} = 0.5$$



Reference

TTG4A

⇒ Still dissipation
⇒ Smaller dispersion

$b_{\Delta} = 0$

$b_{\Delta} = 0.2$

$b_{\Delta} = 0.5$

1.0e+05
101300
101200
101100
101000
100900
100800
100700
1.0e+05
pressure

Reference

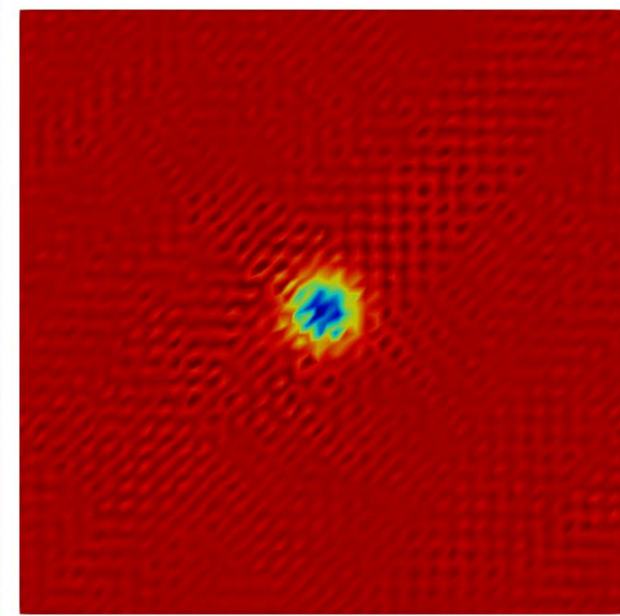
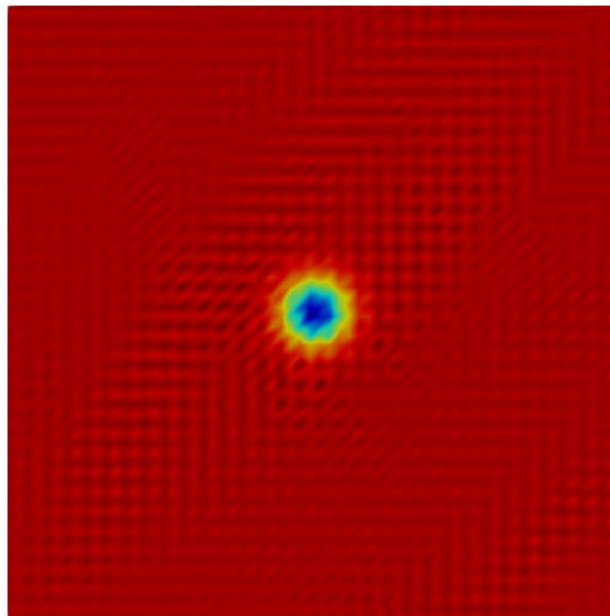
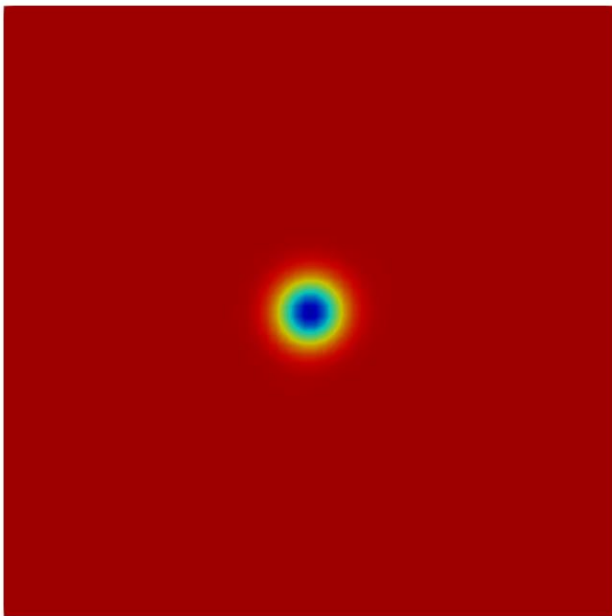
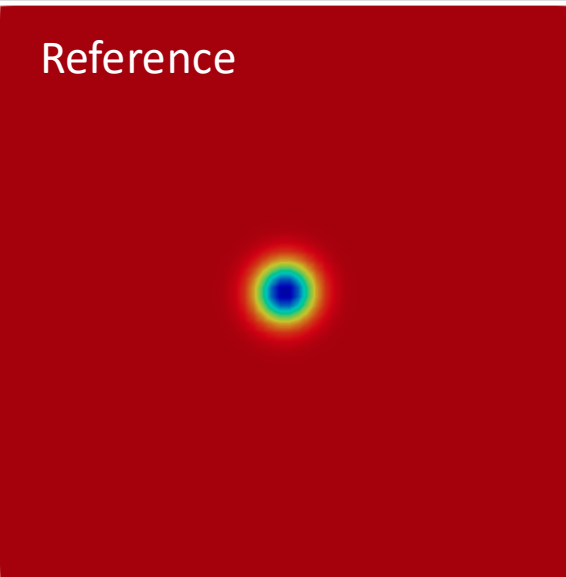
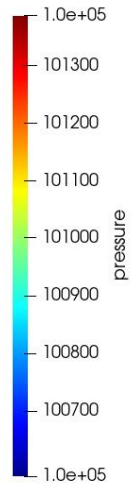
TTGC

⇒ Very few dissipation
⇒ Large dispersion

$$b_{\Delta} = 0$$

$$b_{\Delta} = 0.2$$

$$b_{\Delta} = 0.5$$





Partial conclusion

- **Lax Wendroff is dissipative**
- **TTG4A is also dissipative**, but induces **small dispersion**
- **TTGC very accurate on perfect grids**. **Dispersion** errors however occur for **distorted** grids
- **TTG scheme ~ 2.5 times more expensive**

⇒ **No scheme is perfect !**

LW is of interest for LES initialization.

TTG schemes better suited for statistics.

Errors decrease with increasing:

- **Mesh quality**
- **Mesh resolution**
- **Order of numerical scheme**
- **Lower CFL**





A simple tip...

HIP allows mesh adaptation thanks to the MMG3D library.

"Mmg3d isofactor -f 1"

Can significantly improve your initial Centaur / Ansys mesh !



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Wiggles

- AVBP schemes are **centred schemes**.
- In case of **discontinuity in the discrete approximation** of the hyperbolic equation (*e.g.* mesh **refinement interface**, mesh **boundary**):

➔ Creation of a **spurious wave**, with a wavelength $\lambda = 2\Delta x$:
“Wiggles”, “q-waves”

Wiggles move forward at a group velocity -c

Vichnevetsky/Boyles, 1982
Vichnevetsky Comp. Math. Appls 1985
Vichnevetsky Int. J. Num. methods Fluids 1987
Sengupta App. Maths. Comp. 2012



Centred schemes and *q*-waves

It can be showed that: when **using centred schemes** *i.e.* $\left[\frac{du_n}{dt} + \left(\frac{u_{n+1} - u_{n-1}}{2h} \right) = 0 \right]$

to discretize a continuous **wave equation** $\frac{\partial^2 u}{\partial t^2} - c^2 \frac{\partial^2 u}{\partial x^2} = 0$

leads to a **numerical** (discrete) solution consisting in **2 waves** :

$$\begin{cases} \text{"p-waves"} \\ p(\mathbf{x}, t) = \frac{u + v}{2} \\ q(\mathbf{x}, t) = \frac{u - v}{2} \\ \text{"q-waves"} \end{cases}$$

satisfying
$$\begin{cases} \frac{\partial p}{\partial t} + c \frac{\partial p}{\partial x} = 0 \\ \frac{\partial q}{\partial t} - c \frac{\partial q}{\partial x} = 0 \end{cases}$$

$\xrightarrow{+c}$
Moving **forward**

$\xleftarrow{-c}$
Moving **backward**

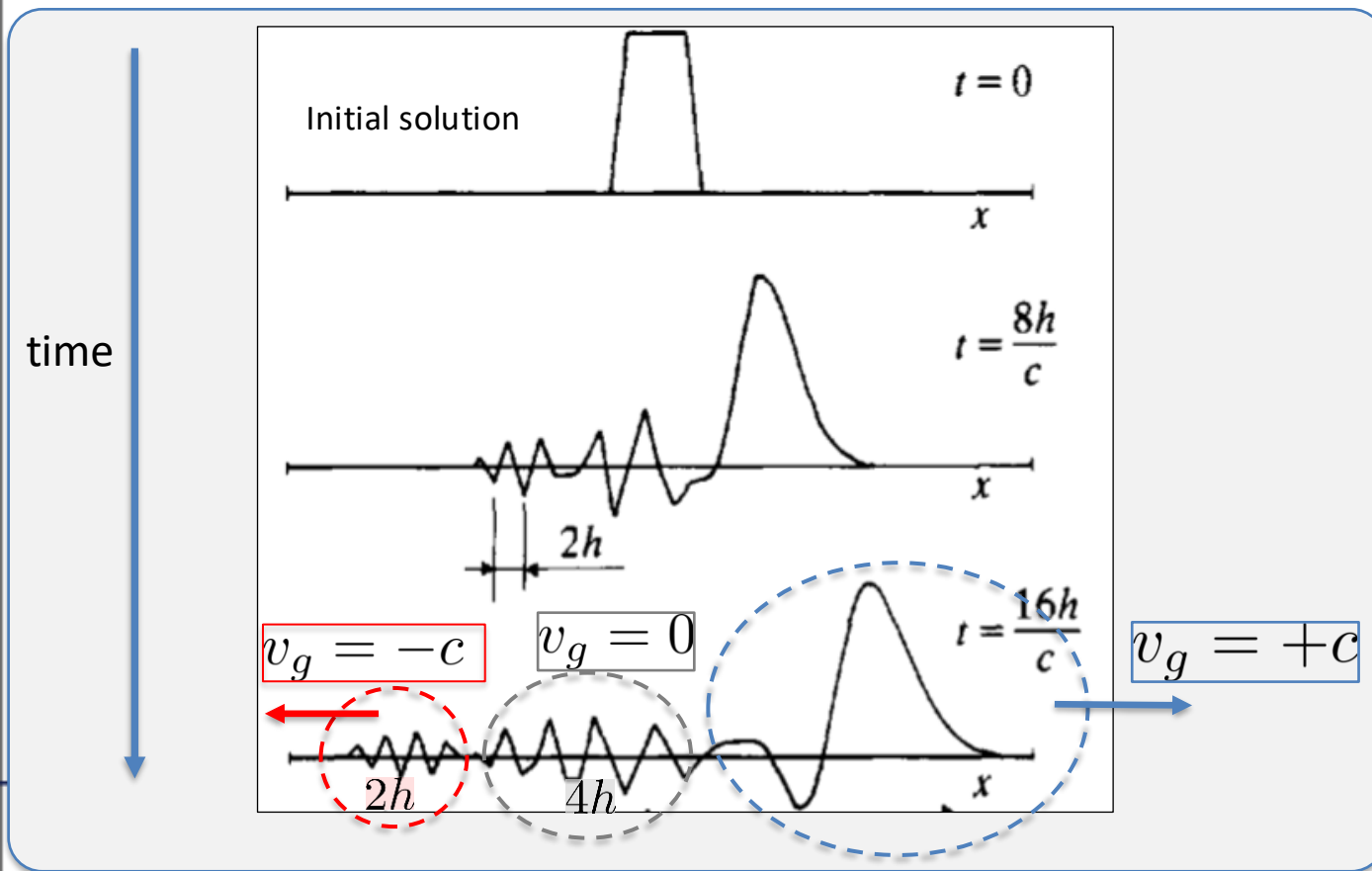


Centred schemes and q -waves

Discrete numerical solution

$$= \frac{\text{"Smooth" part}}{p(\mathbf{x}, t) = \frac{u + v}{2}} + \frac{\text{"Oscillatory" part}}{q(\mathbf{x}, t) = \frac{u - v}{2} = u - \left(\frac{u + v}{2} \right)}$$

$+c$ (blue arrow pointing right) $-c$ (red arrow pointing left)





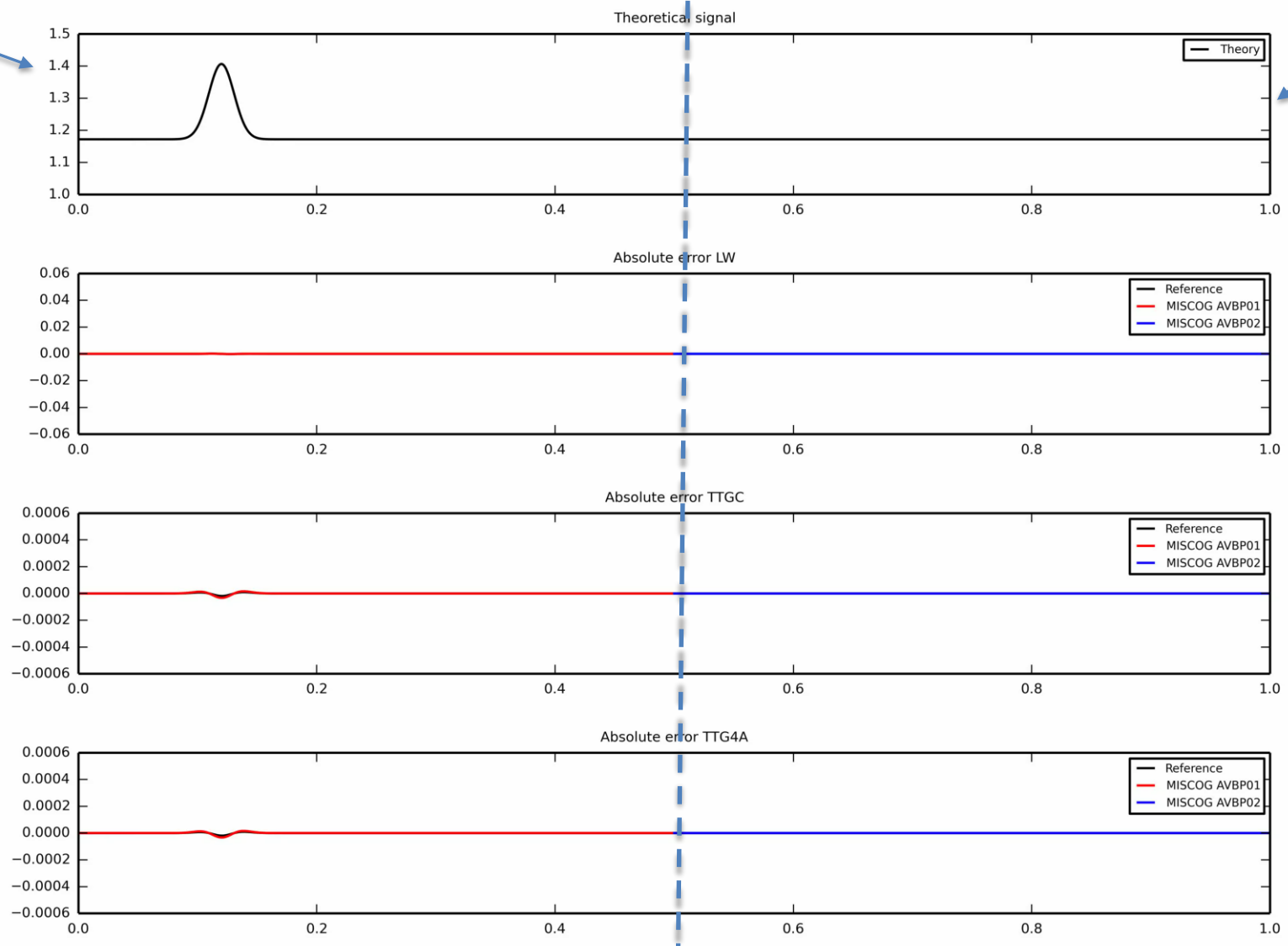
Mesh interface

Wiggles: illustration

Boundary condition

Boundary condition

q-waves
generate
p-wave !





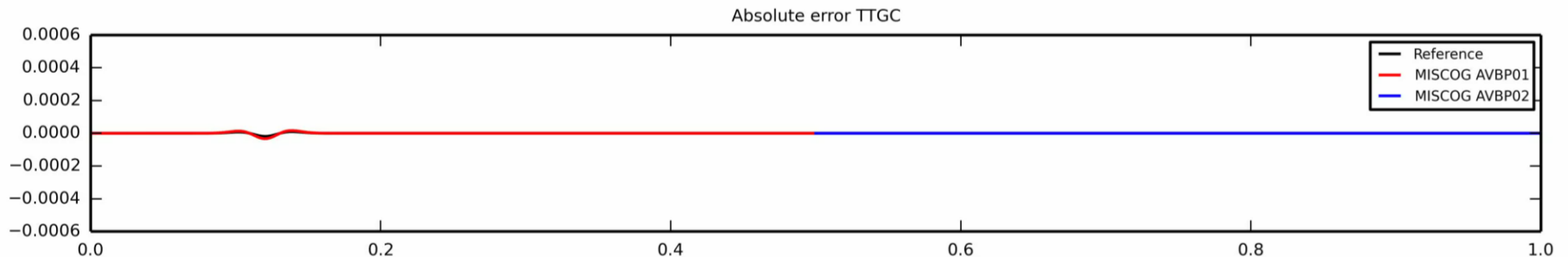
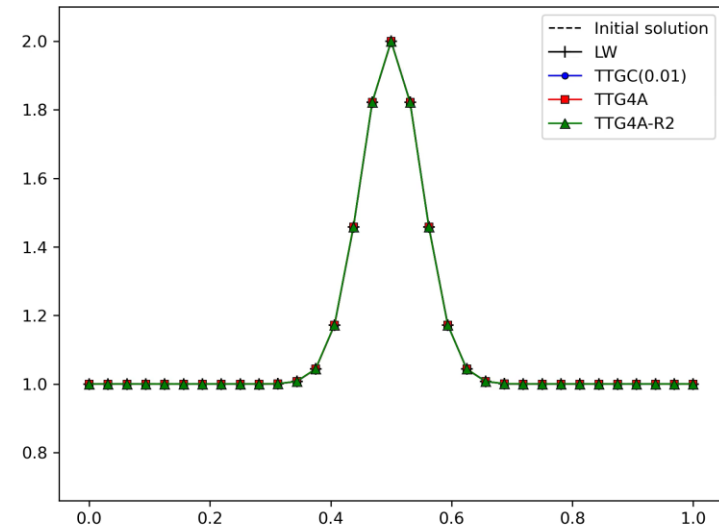
Dispersion errors and *q*-waves

Dispersion error

Mostly convected at the phase velocity $u+c$, along with the physical wave

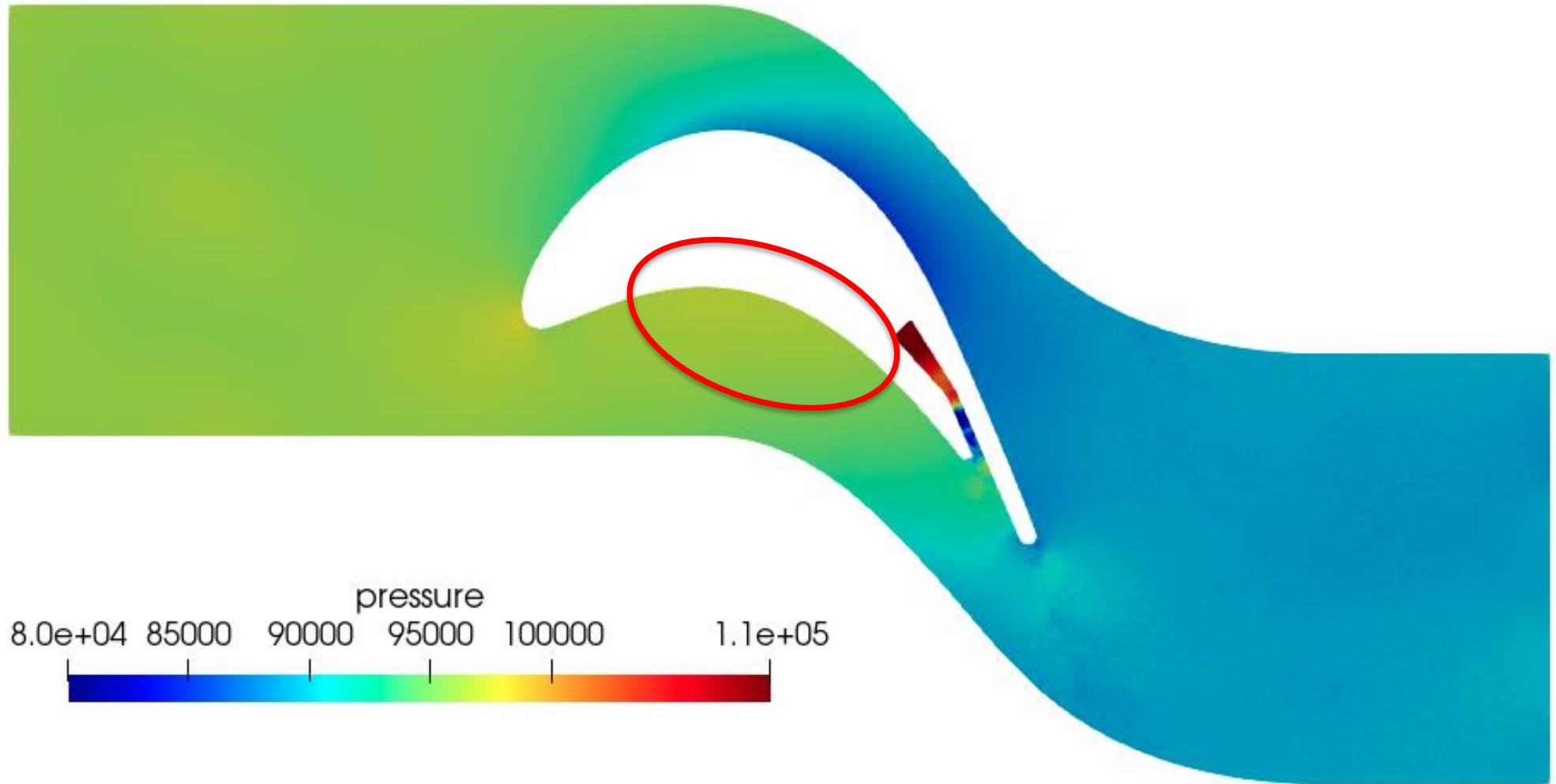
q-waves (wiggles)

Convected with a group velocity $-c$, as physical wave interacts with a boundary *q*-waves may generate *p*-waves (physical waves) when interacting back the other boundary



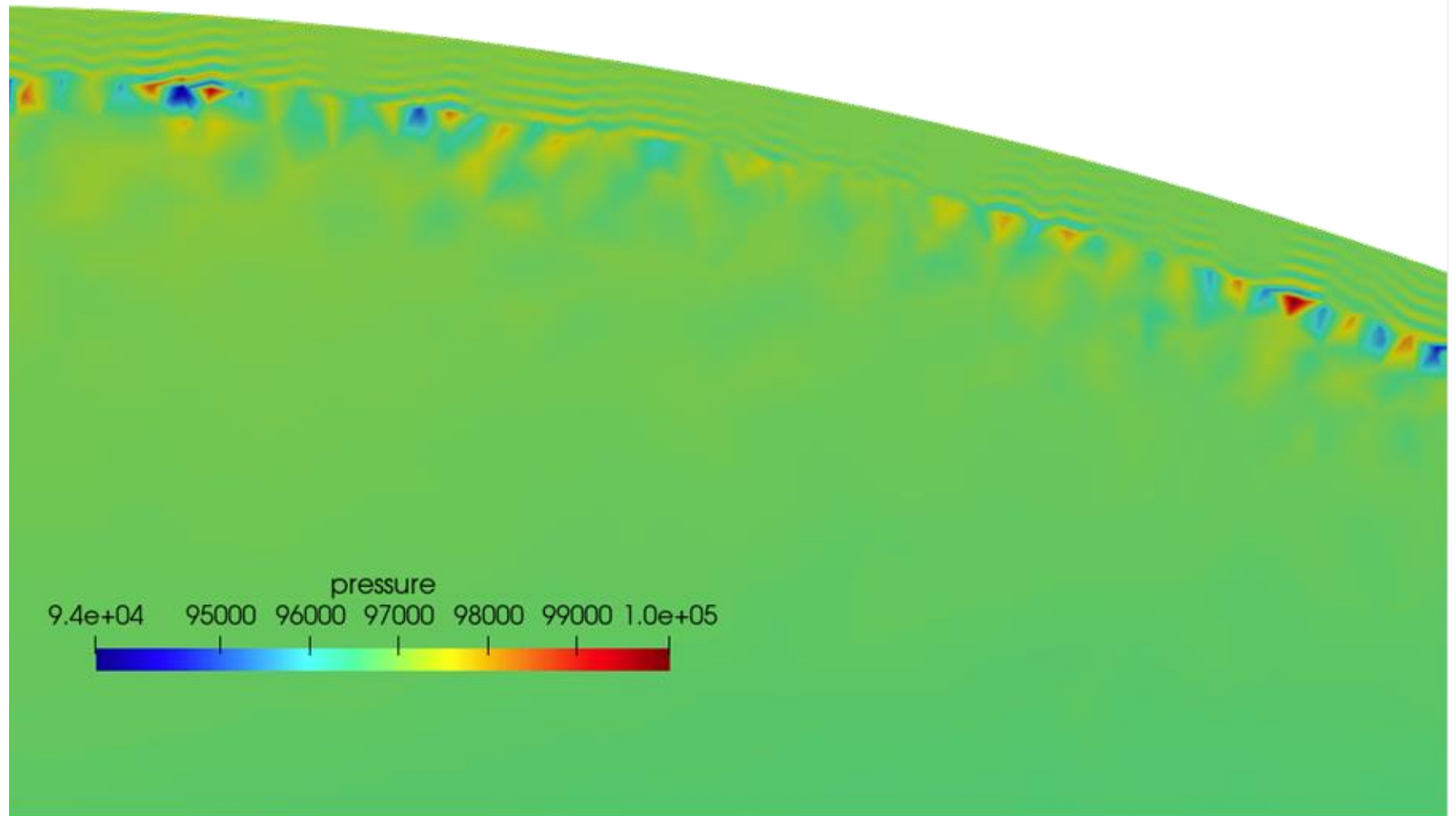


Issue at mesh interface



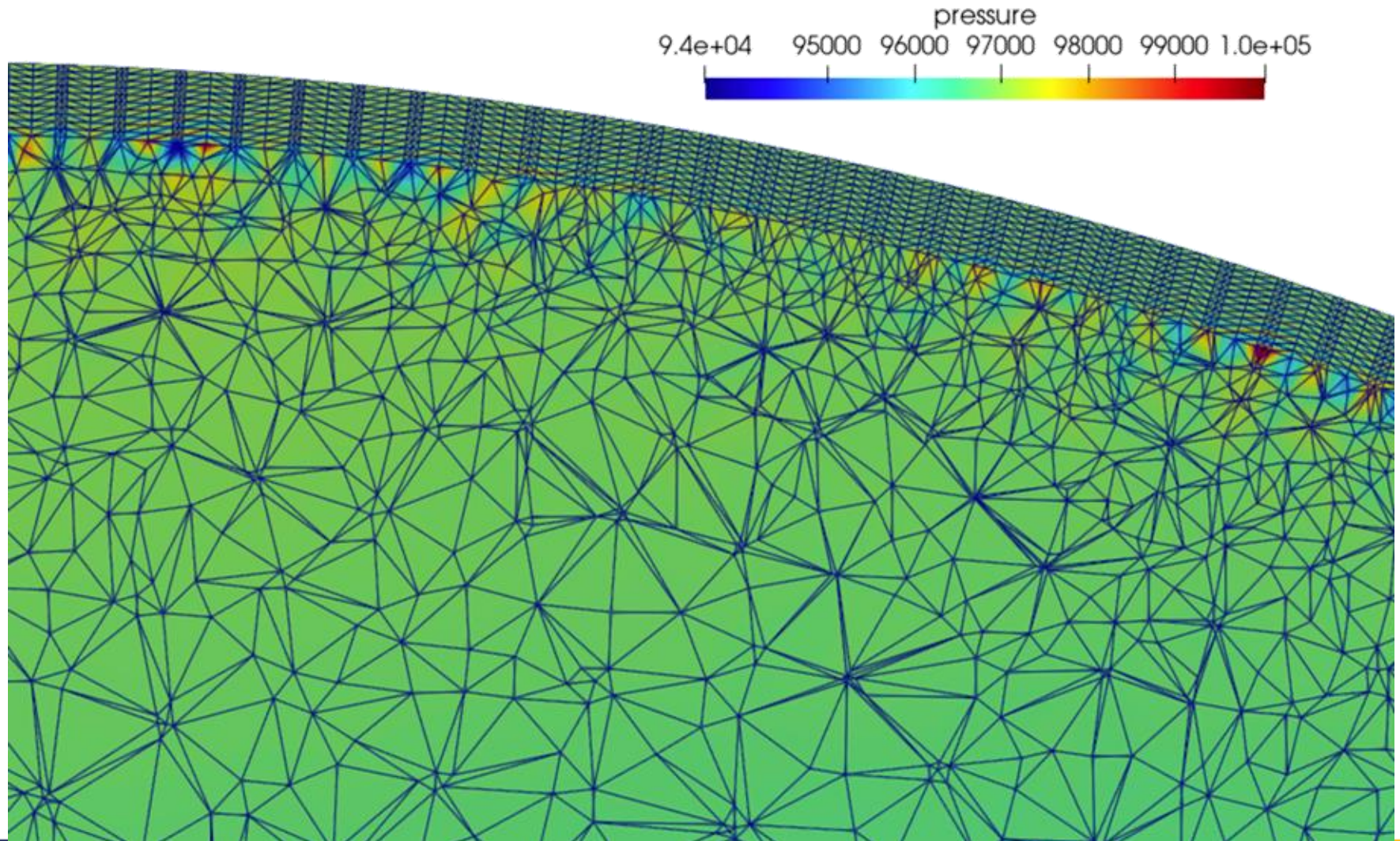


Issue at mesh interface



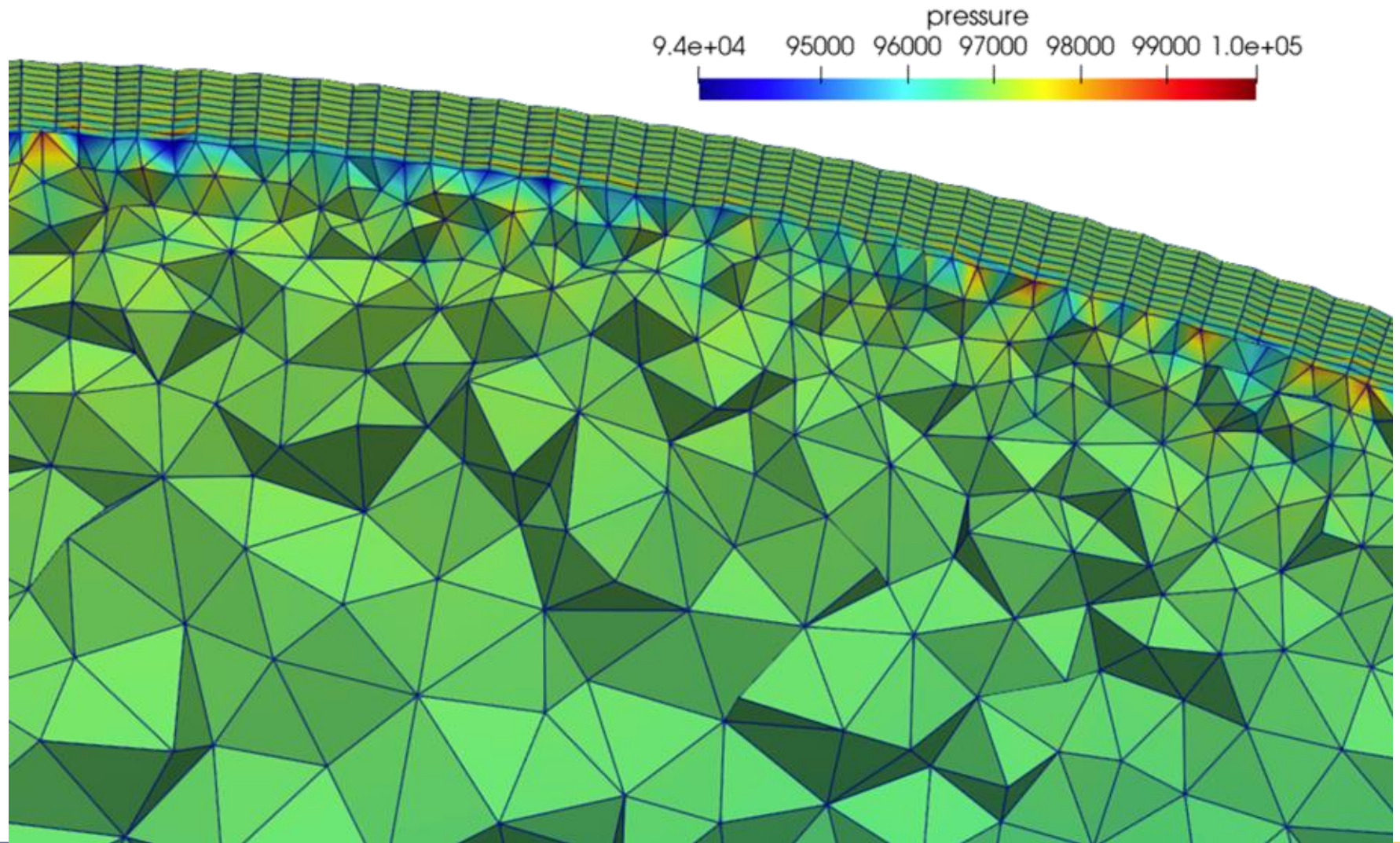


Issue at mesh interface



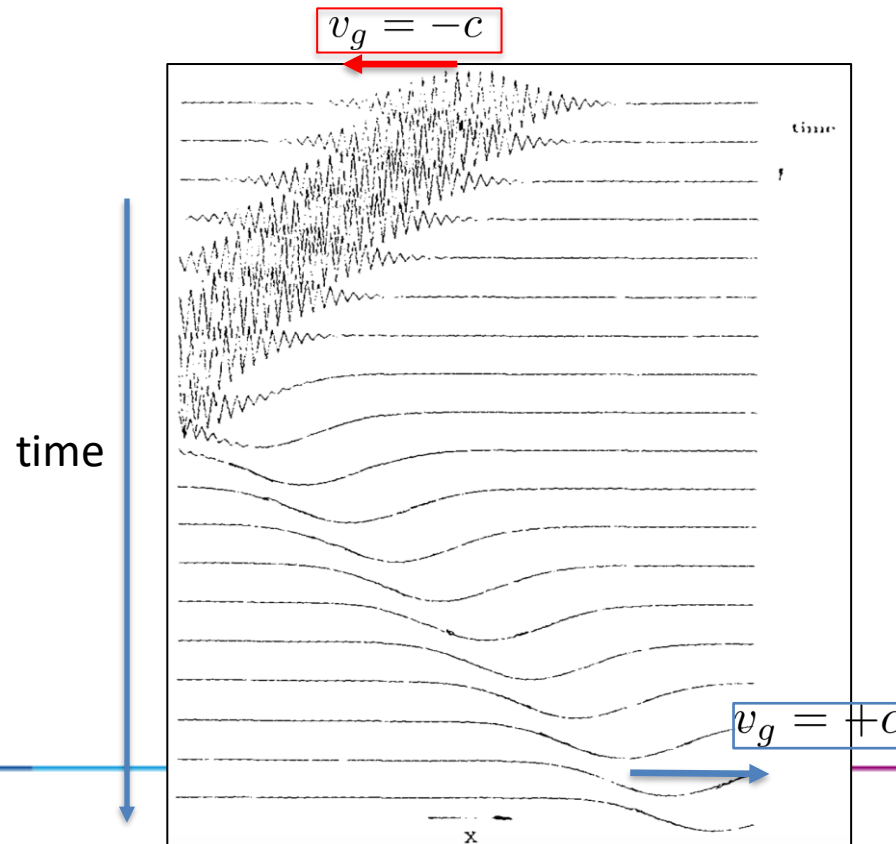


Issue at mesh interface



Partial conclusion on Wiggles

- Wiggles are not just dispersion error downstream of physical waves
- Wiggles are actual waves, propagating into the domain, at a group velocity $-c$
- Wiggles can generate physical waves when interacting with a BC !



See

Vichnevesky Bowles, SIAM 1982

Poinsot Veynante Ch.9

Sengupta et al. App. Math. Comp. 2012



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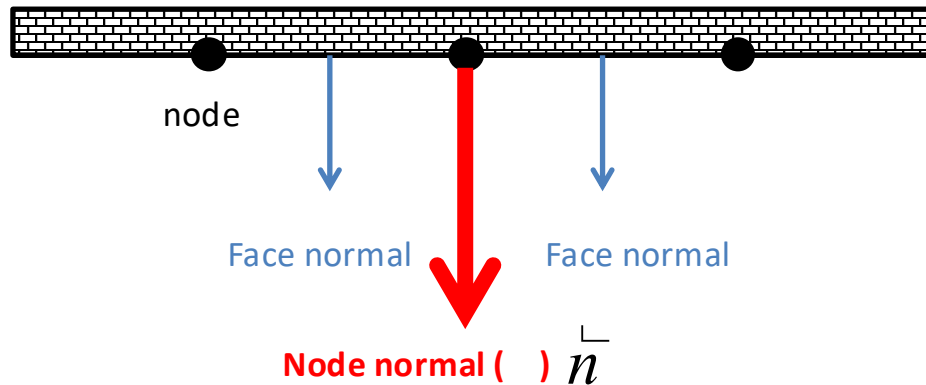
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WALL/ WALL interaction : critical issue on corner

To ensure mass conservation, the normal (to the wall) velocity is cancelled:

$$\vec{U} \cdot \vec{n} = 0$$

Node normals are an average of normals on neighboring surfaces:

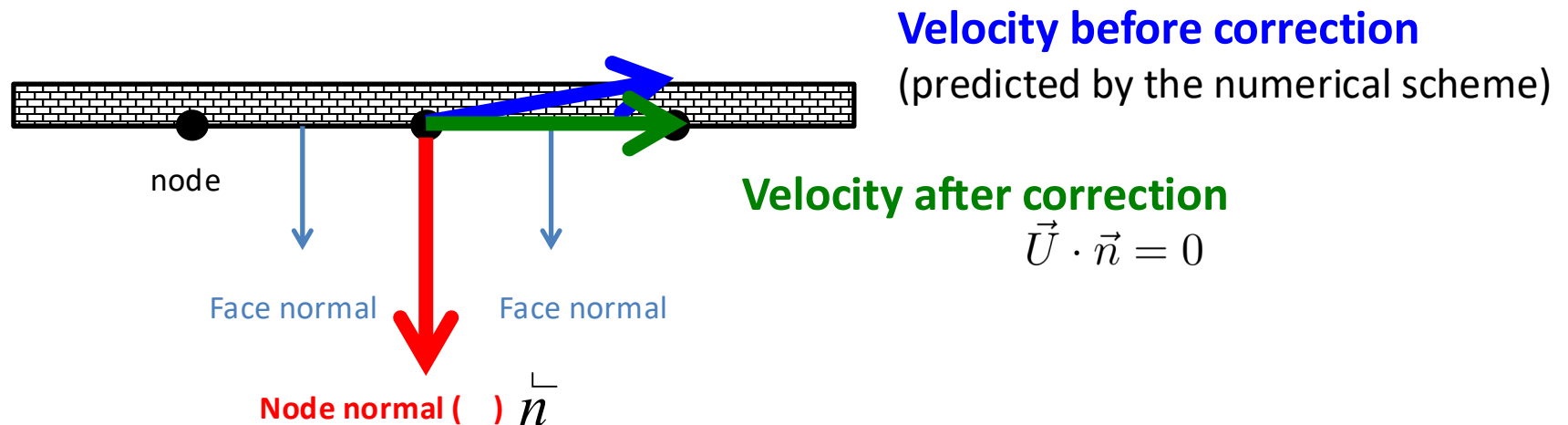


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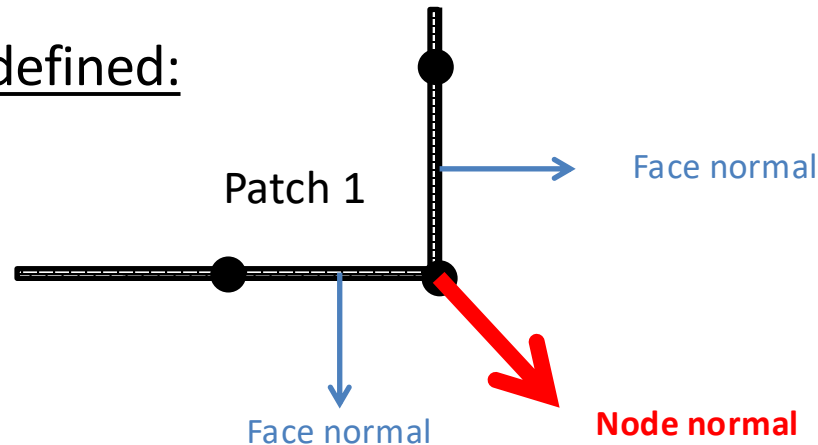


WALL/ WALL interaction : critical issue on corner

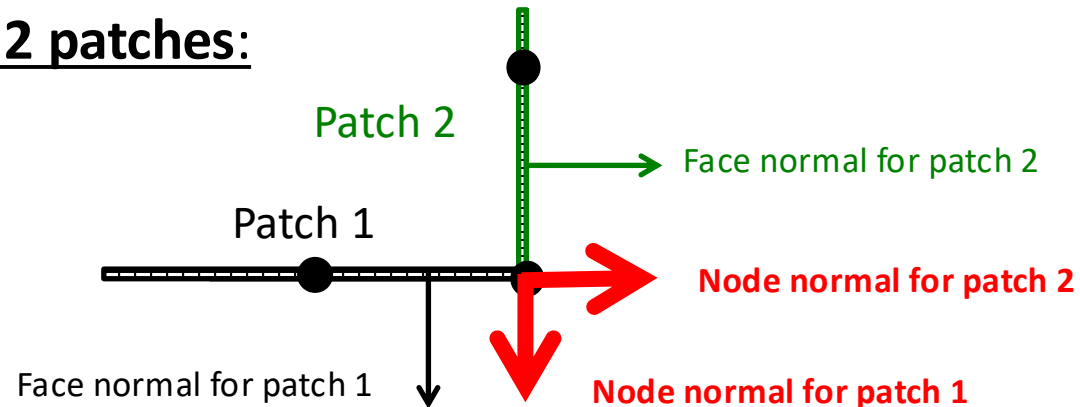
Wall normal direction at corners is an **ambiguous quantity**!

It depends on how patches are defined :

- If only 1 wall patch is defined:



- If the wall is split in 2 patches:

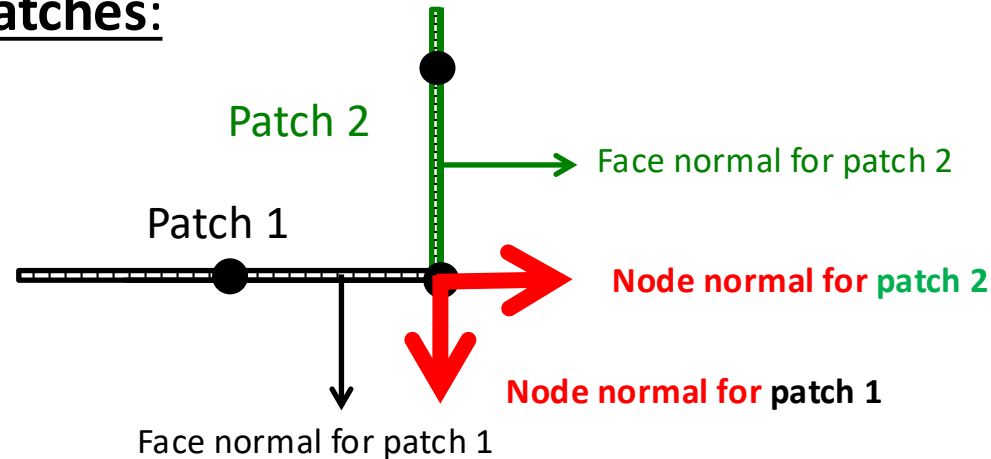




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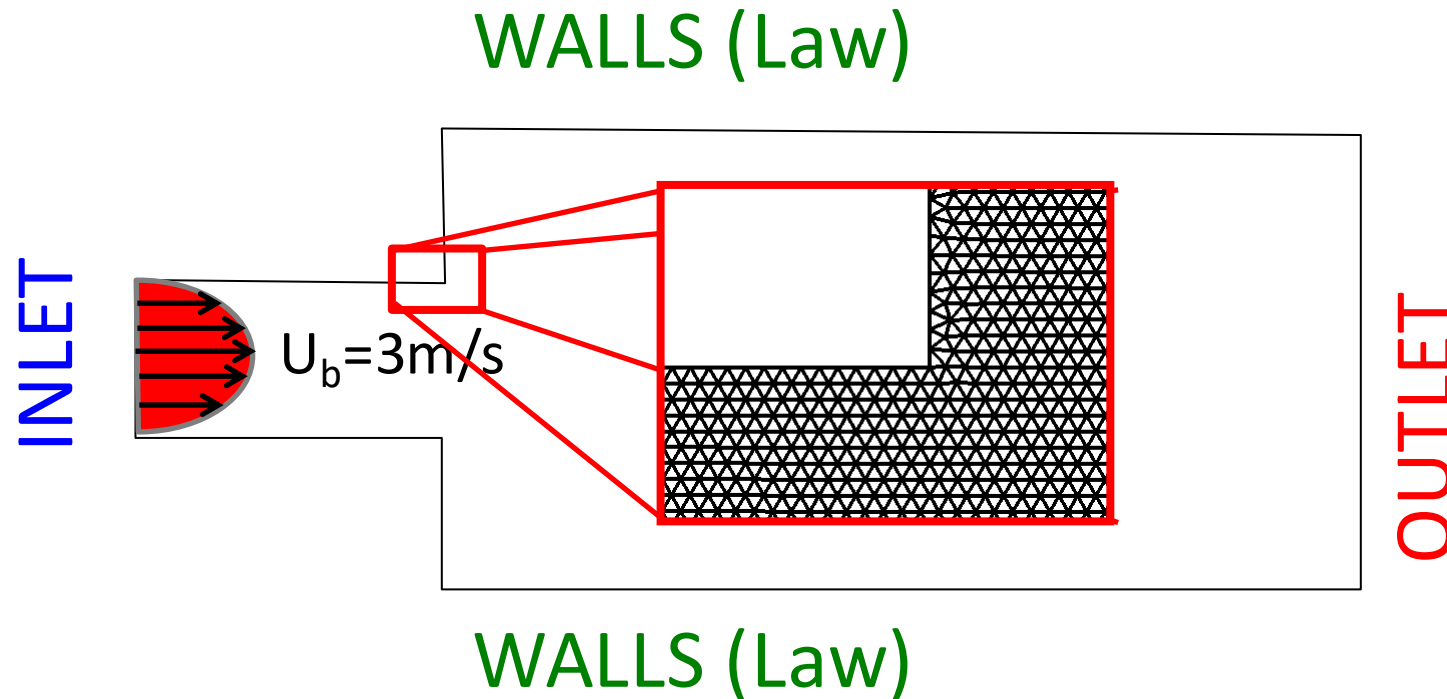
➔ **Two normals** are defined for the same node !

➔ **Two treatments** are successively applied



Wall corners

Example : sudden expansion

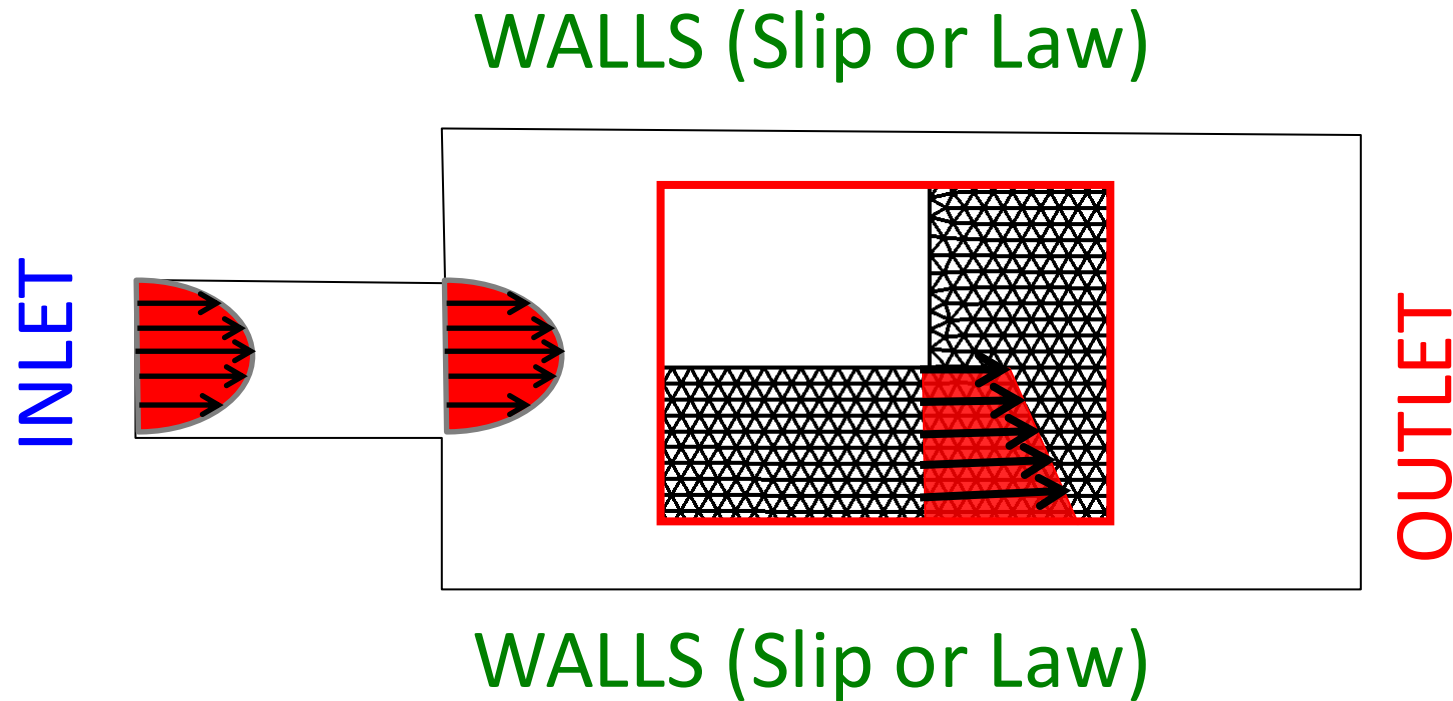


WHAT HAPPENS AT THE CORNER ?



Wall corners

Example : sudden expansion

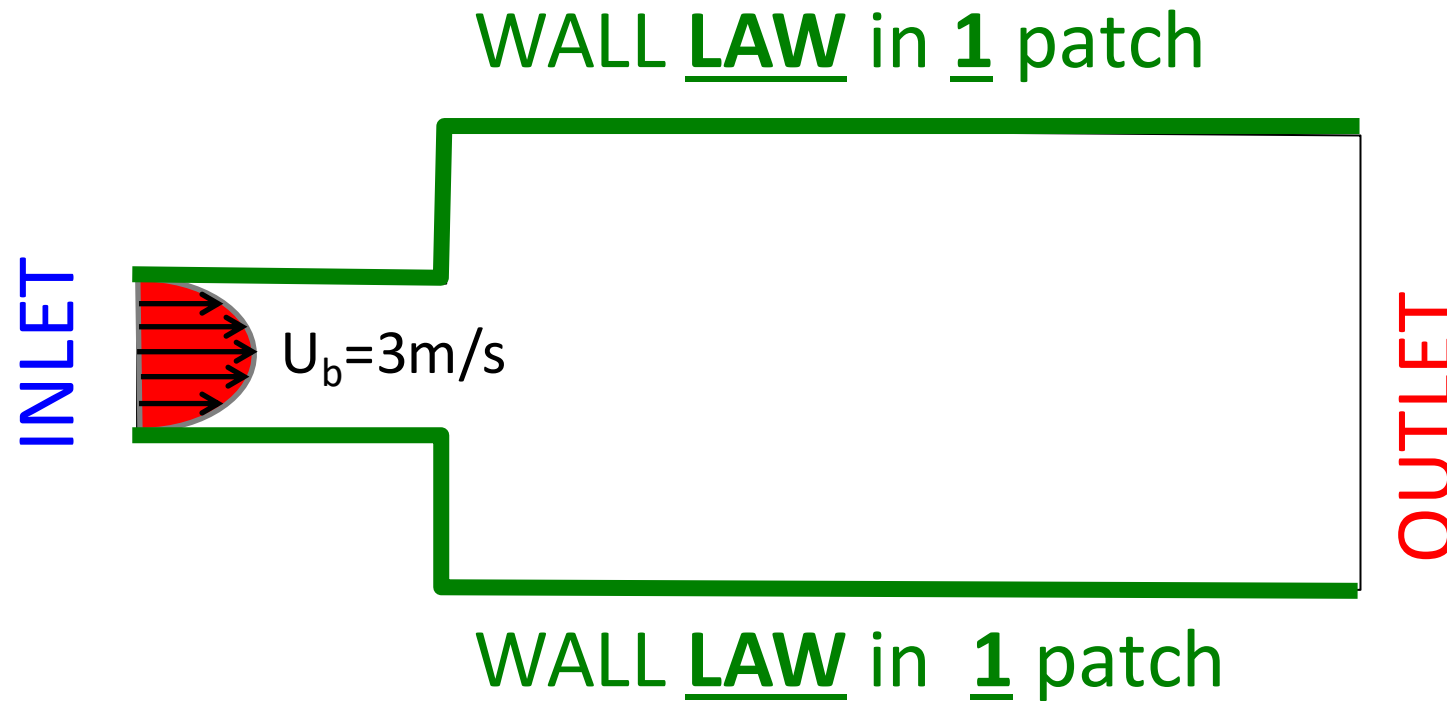


Expected solution: the **flow** should go **straight**



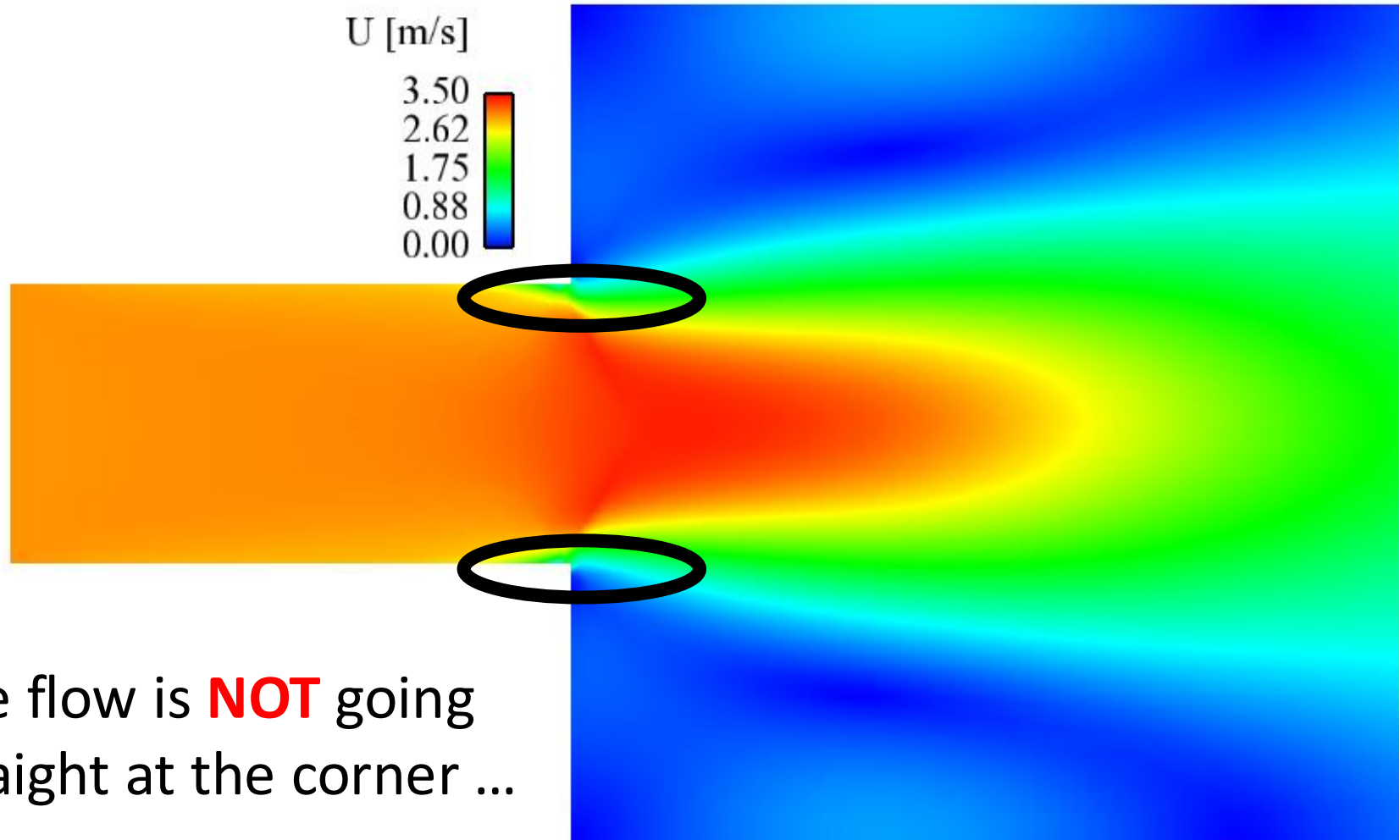
Joint patches test case

→ First, let's consider **wall Law** using **one single** patch





Joint patches test case

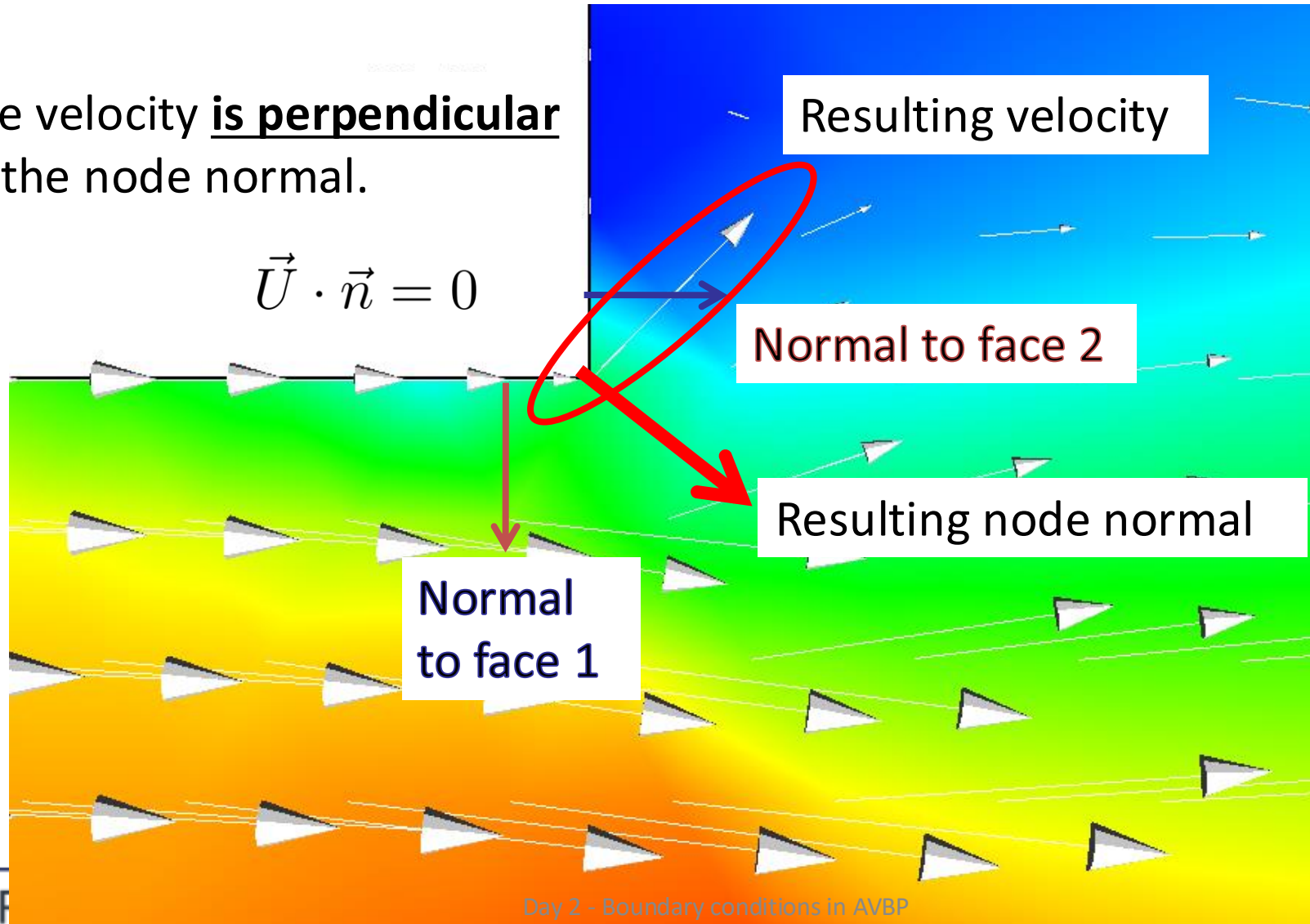




Joint patches test case

The velocity **is perpendicular** to the node normal.

$$\vec{U} \cdot \vec{n} = 0$$

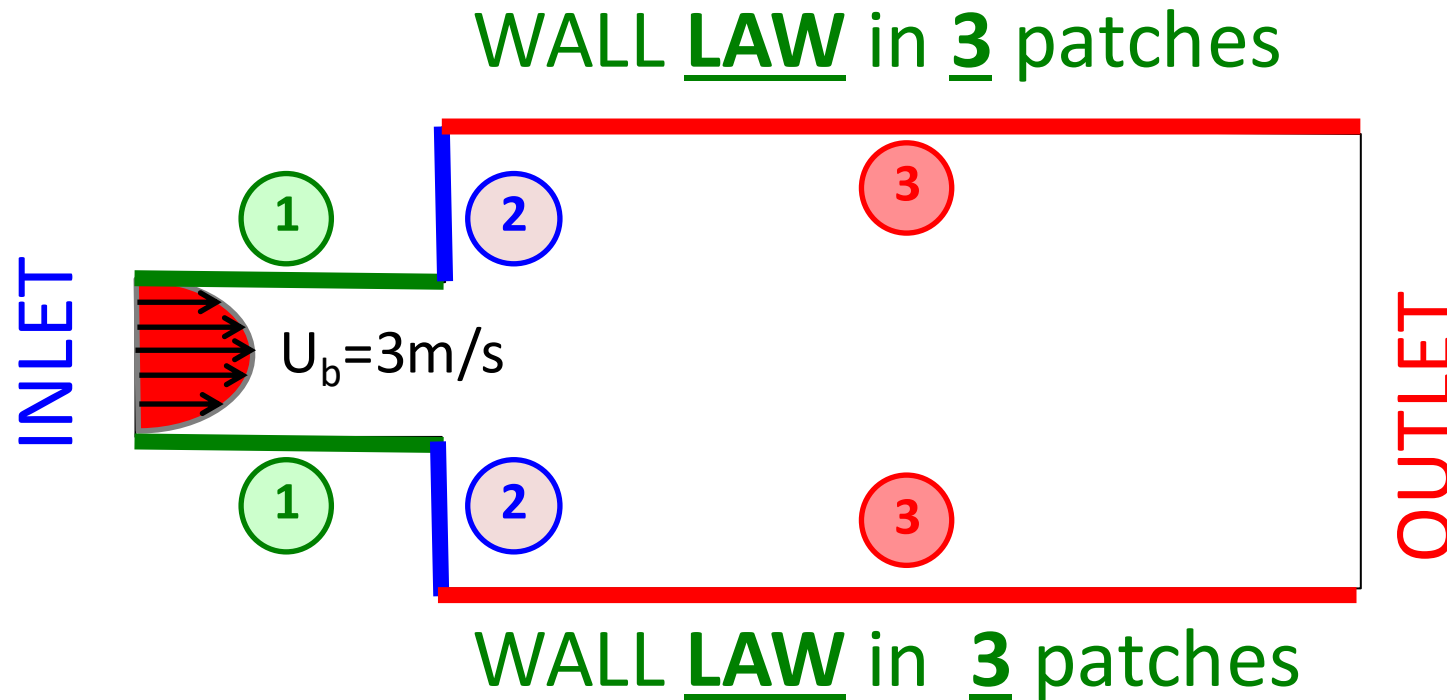




Wall corners

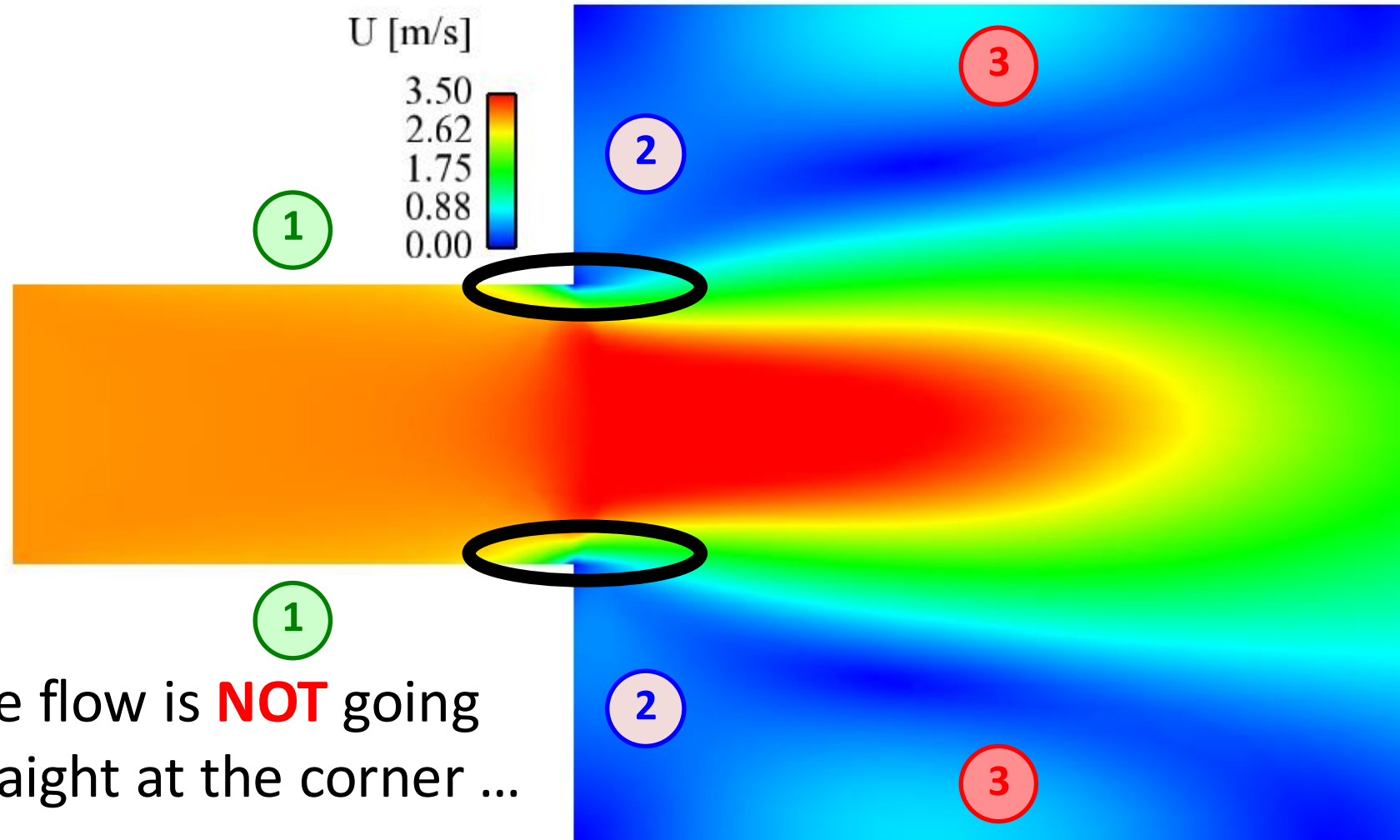
Example : sudden expansion

→ Let's consider **wall Law** using **2 separated** at the corner





Separated patches test case



The flow is **NOT** going straight at the corner ...



Separated patches test case

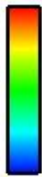
BC treatment for
patch 1

$$\vec{U} \cdot \vec{n} = 0$$

1

U [m/s]

3.50
2.62
1.75
0.88
0.00



Patch 1 allows a
velocity in this
direction only

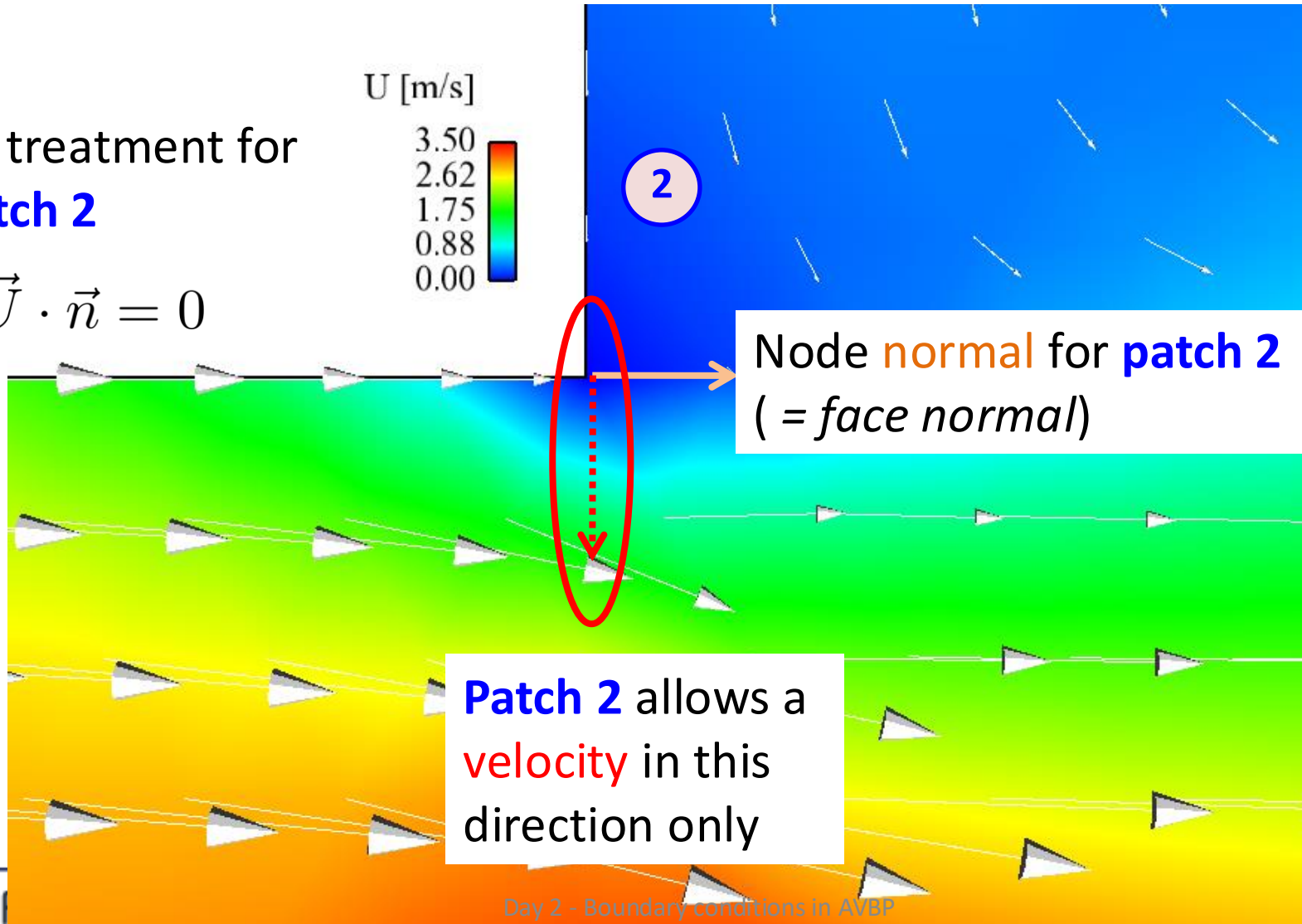
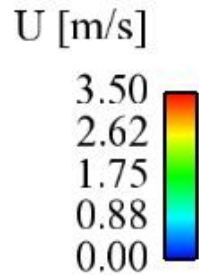
Node normal for **patch 1** (
= *face normal*)



Separated patches test case

BC treatment for
patch 2

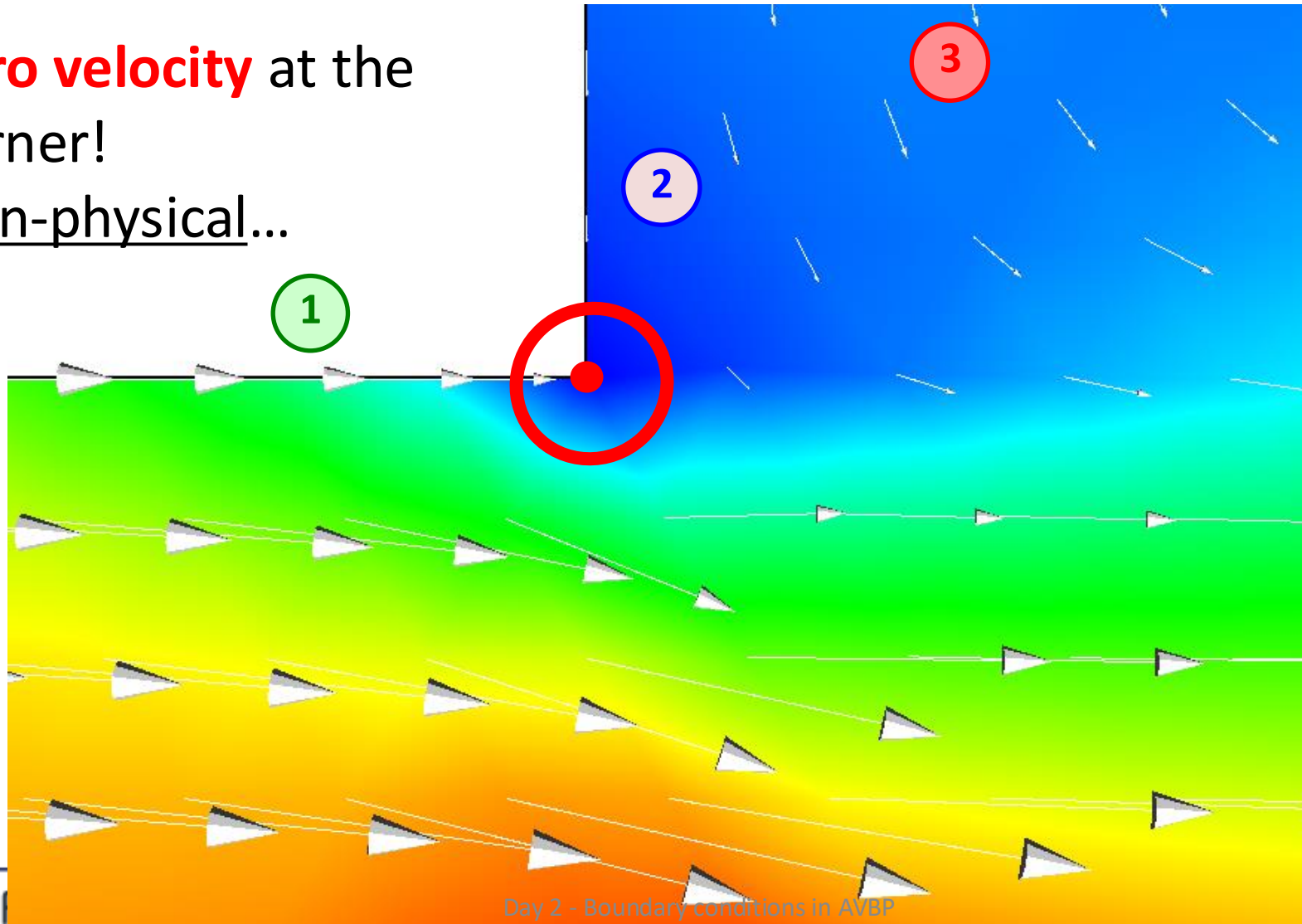
$$\vec{U} \cdot \vec{n} = 0$$





Separated patches test case

Zero velocity at the
corner!
Non-physical...



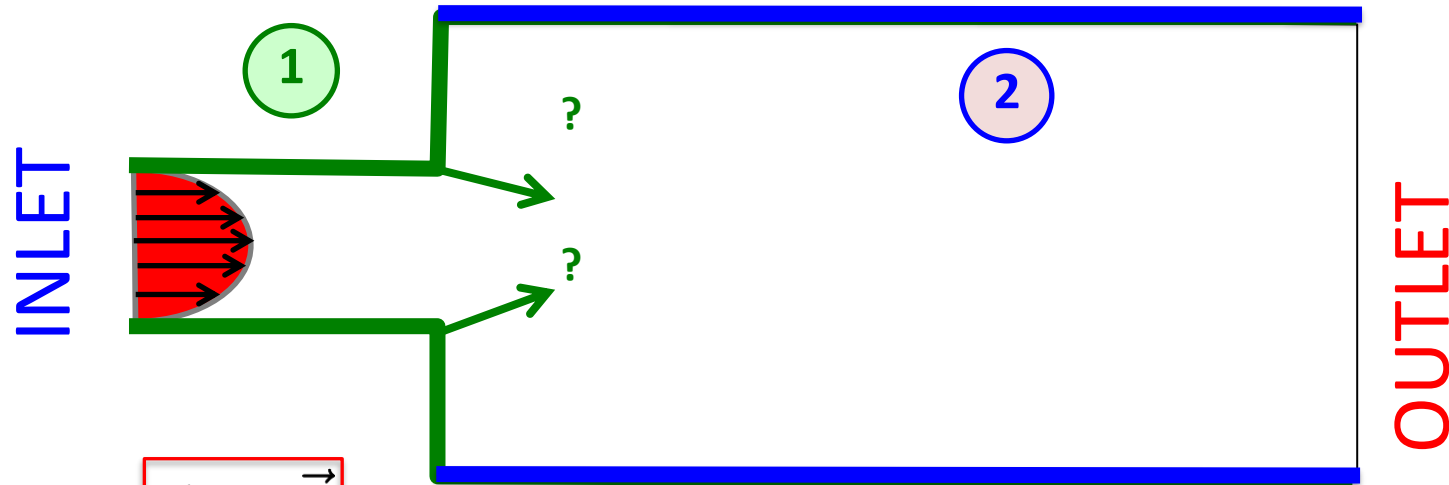


Wall corners

The free corner method

What is the solution ?

→ The free-corner procedure !



Let's impose $\vec{n} = \vec{0}$:

- $\vec{U} \cdot \vec{n} = 0$ is always satisfied.
- The flow **direction** is let **free**
- Additional treatments are imposed to enforce mass conservation
(PhD P. Schmitt)



Asciibound file

- You must enforce a single patch

```
!-----  
patch_name = wall_carter  
boundary_condition = WALL_LAW_ADIAB  
free_corner_min_angle = 60.0D00  
!-----
```



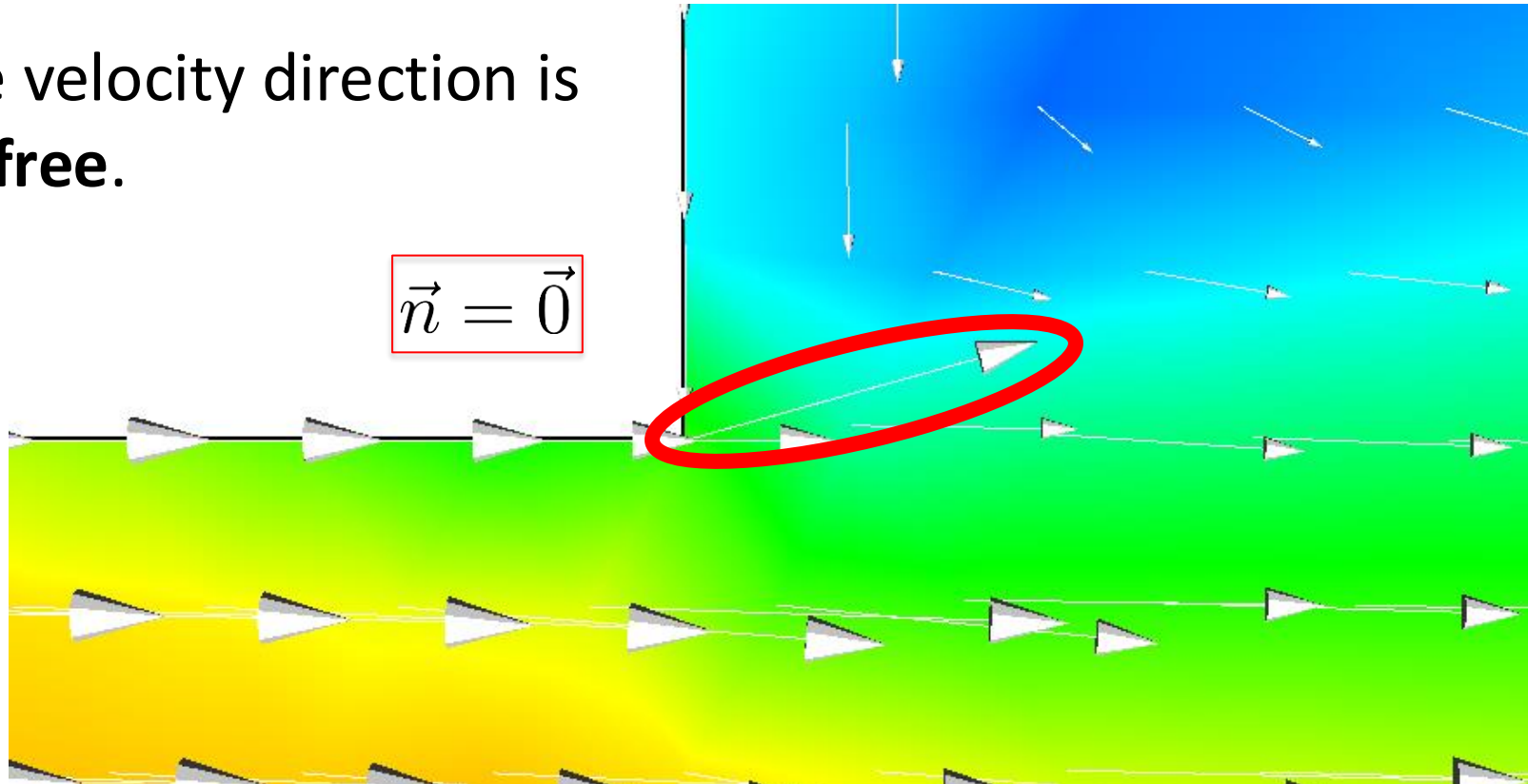
Activates free-corner procedure for angle between face normals $> 60^\circ$



Example of corner treatments : 1 **single patch** + free corner

The velocity direction is
let **free**.

$$\vec{n} = \vec{0}$$

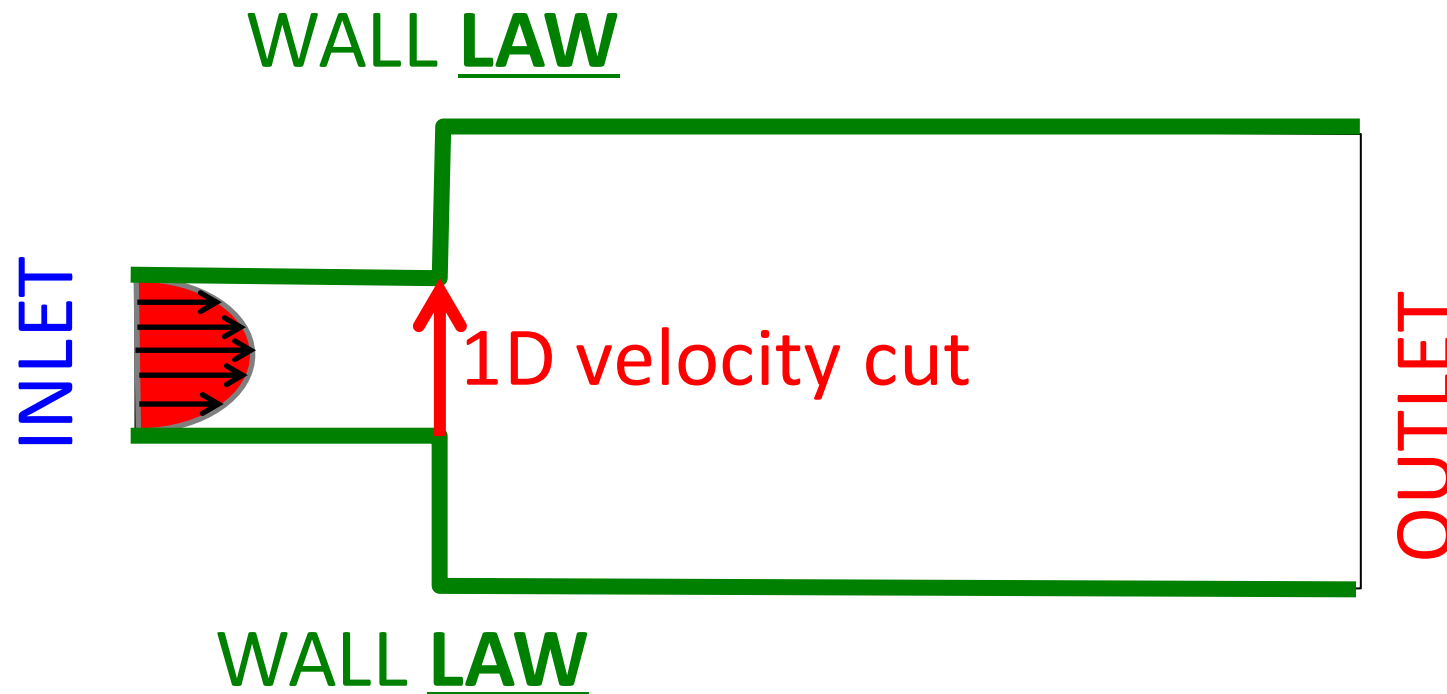


In that case, an **additional correction** ensures
the **mass conservation**



Wall corners

Example : sudden expansion

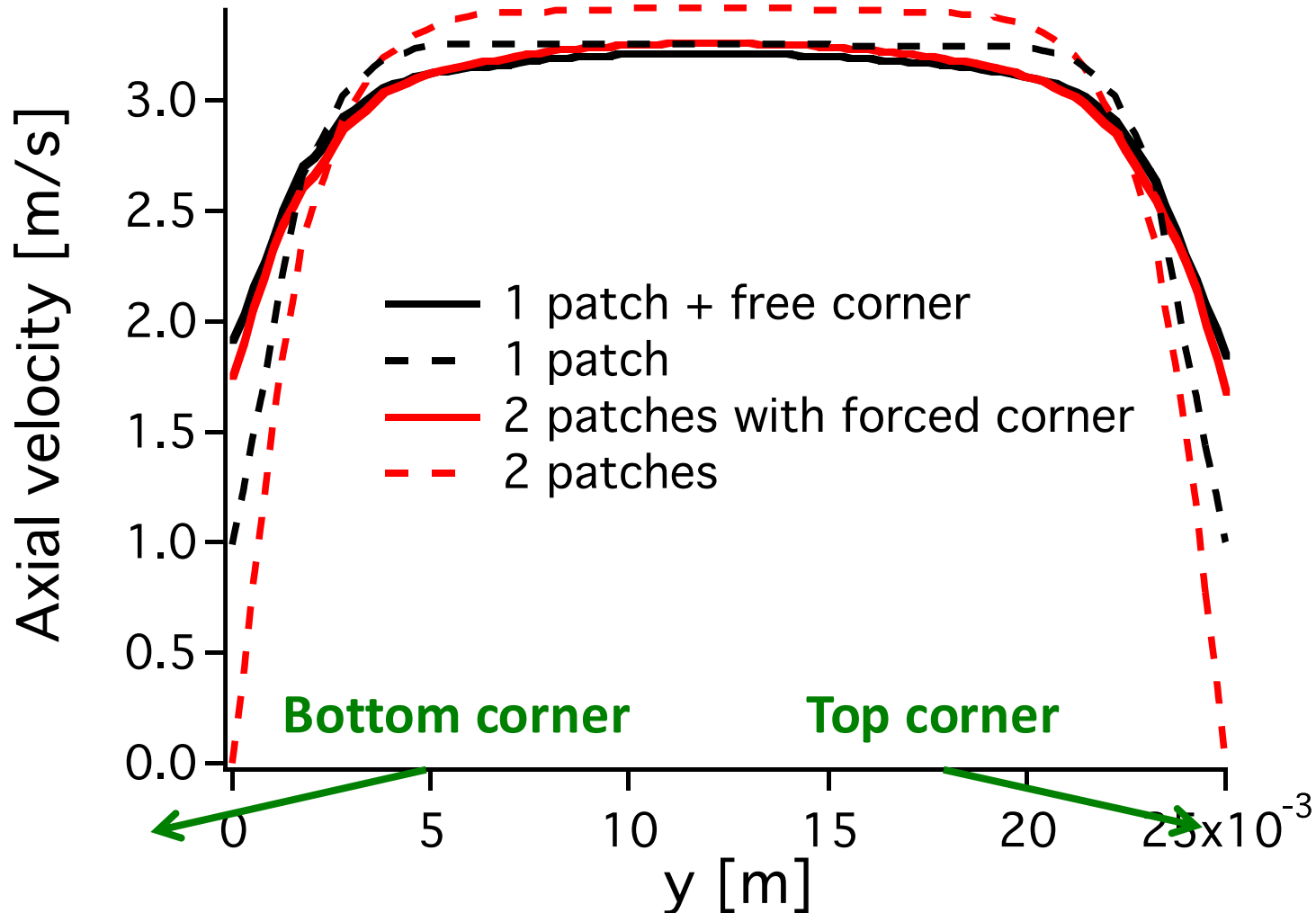


Velocity profiles at the cut position ?





Example of corner treatments : summary





Wall corners

Example : sudden expansion

Sometimes, we may want to **enforce** the flow **direction**



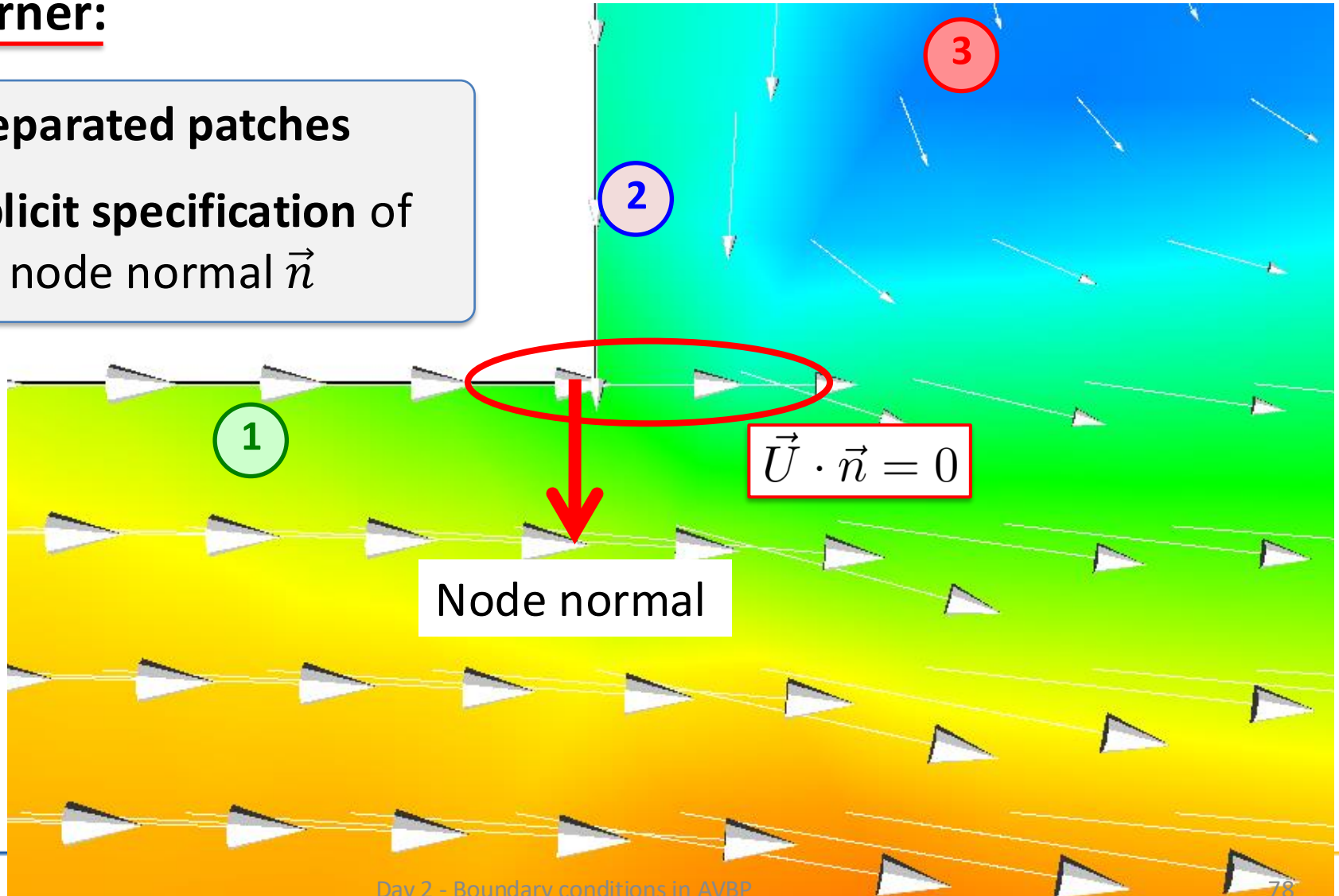
→ The forced-corner procedure !



Example of corner treatments : 2 patches with forced corner

Forced-corner:

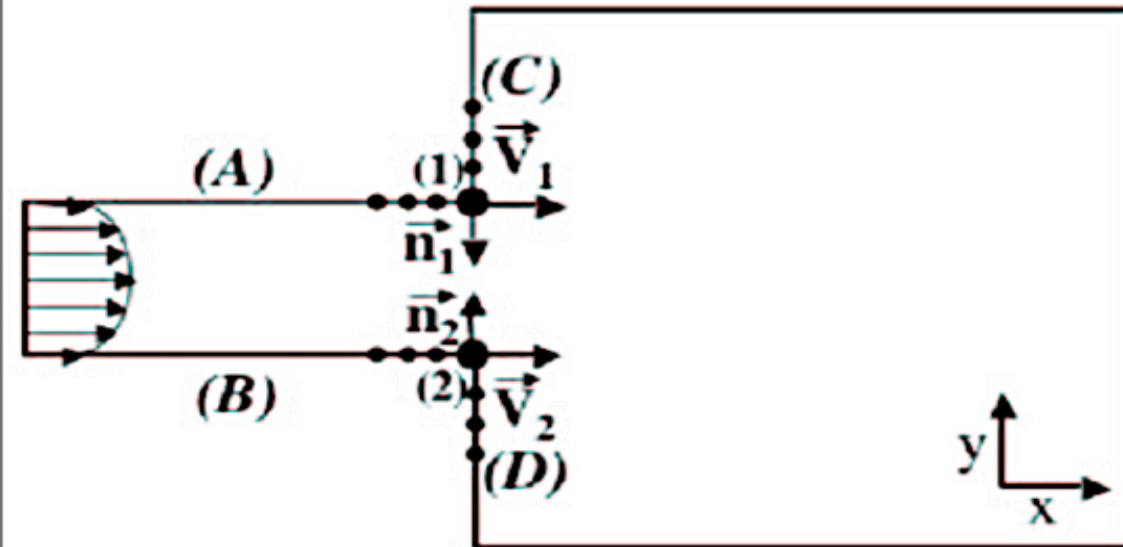
- 2 separated patches
- Explicit specification of the node normal \vec{n}





Issues on corners

Forced corner method



Separated patches A and B

```
$BOUNDARY-PATCHES
patch_name = Patch_A
boundary_condition = WALL_LAW_ADIAB
free_corner_min_angle = 0.60D+02
forced_corner_associated_patch = Patch_C
```

Patch_A imposes its normal to Patch_C

=> Enforce the velocity direction

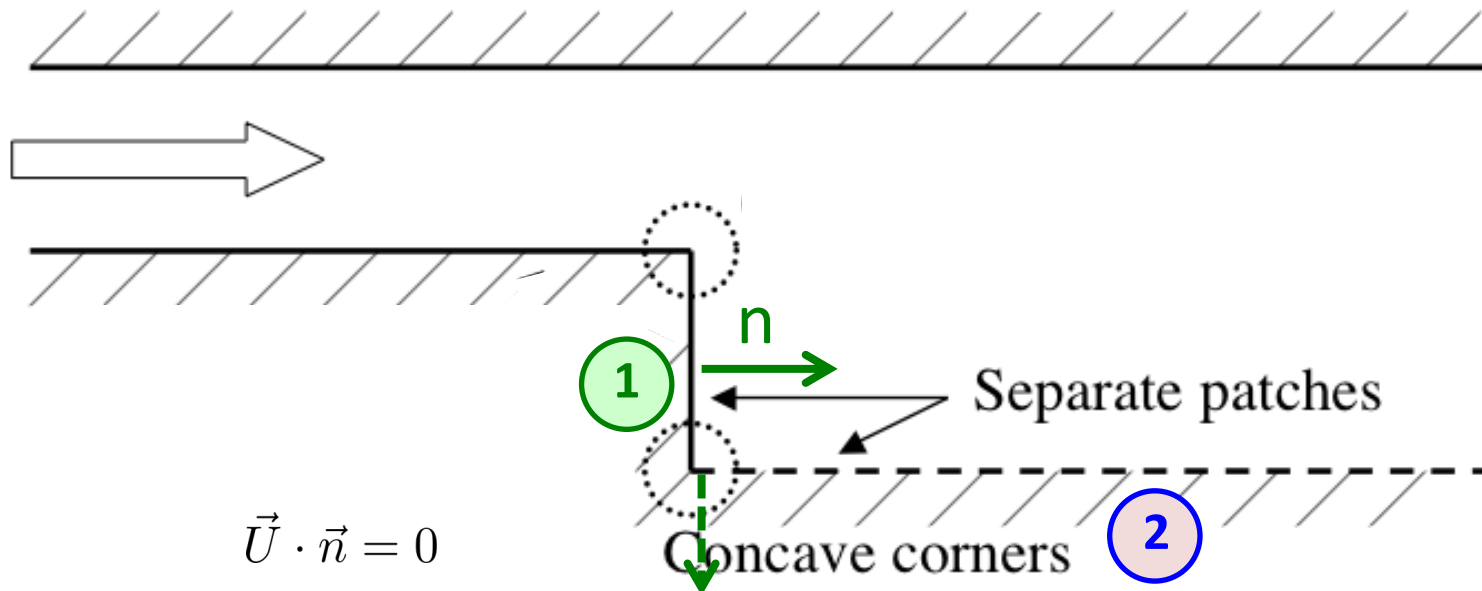


Issues on corners

Concave corners

For concave corners, the situation is simpler:

➔ The two patches (1 and 2) **must be separated**.



- The first treatment (**patch 1**) implies that the **velocity normal** to **1** is **zero**.

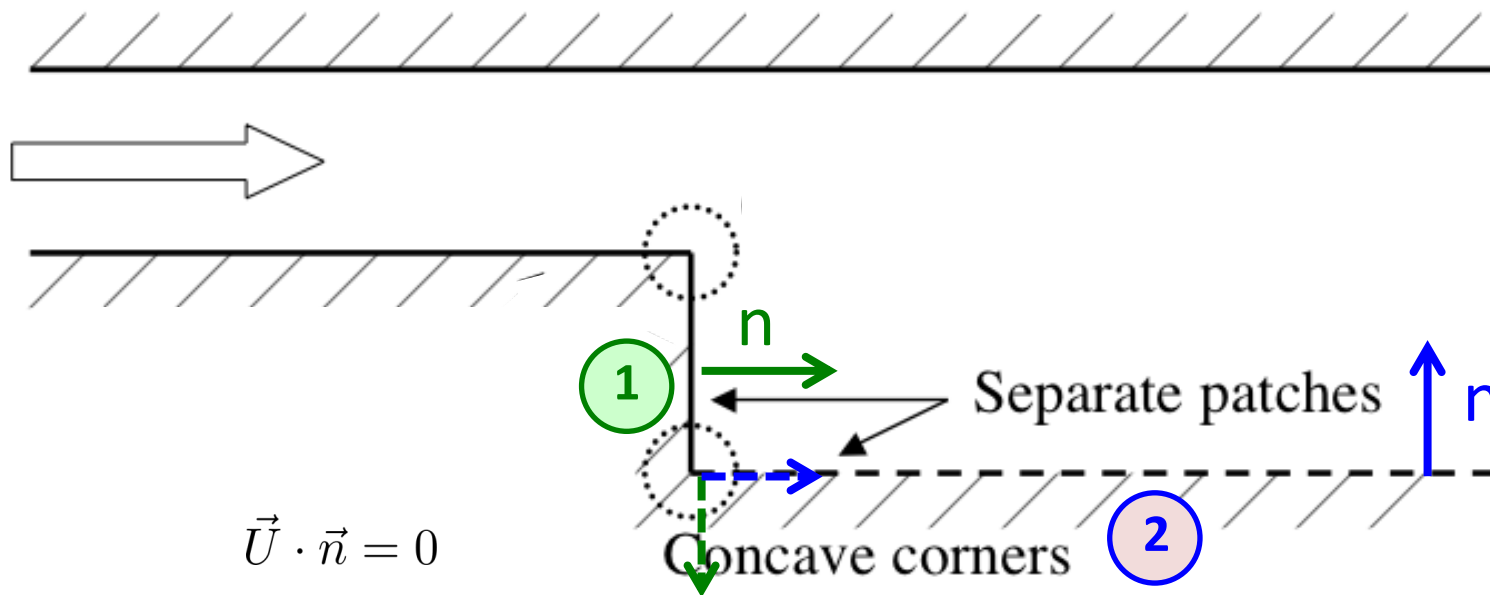


Issues on corners

Concave corners

For concave corners, the situation is simpler:

➔ The two patches (1 and 2) **must be separated**.



- ▶ The first treatment (**patch 1**) implies that the **velocity normal** to **1** is **zero**.
- ▶ The second treatment (**patch 2**) implies that the **velocity normal** to **2** is **zero**.

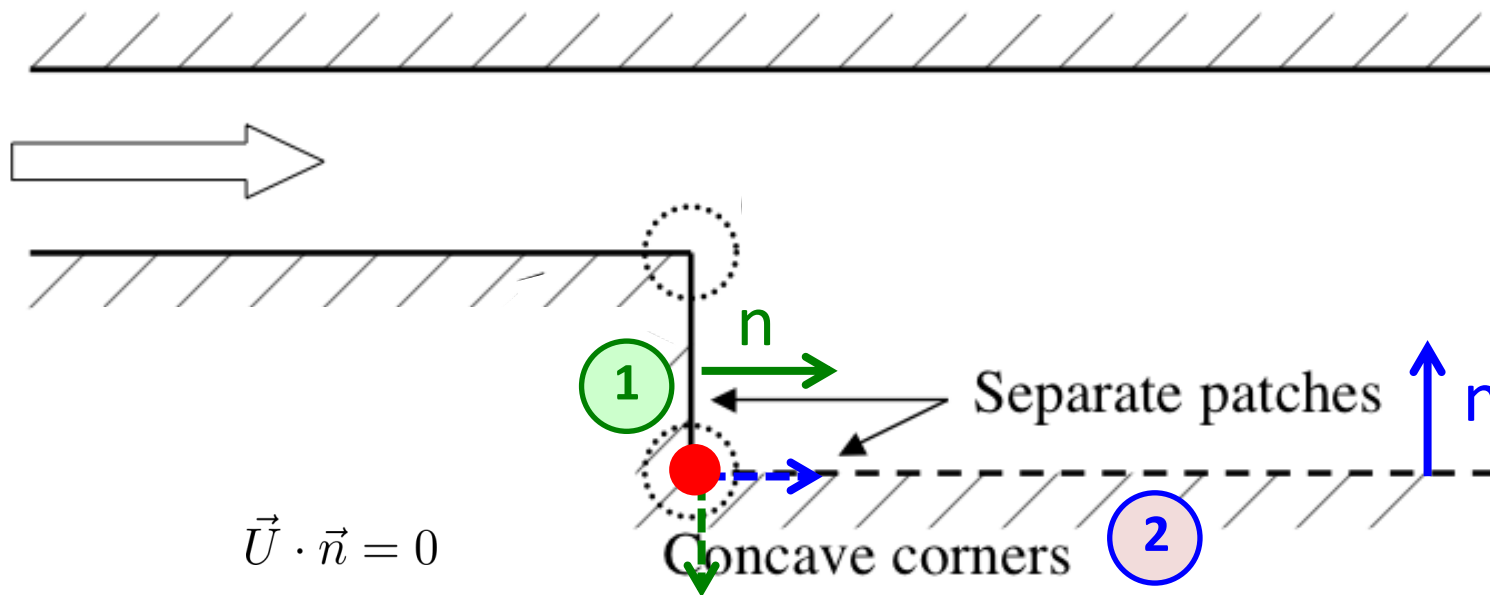


Issues on corners

Concave corners

For concave corners, the situation is simpler:

➔ The two patches (1 and 2) **must be separated**.



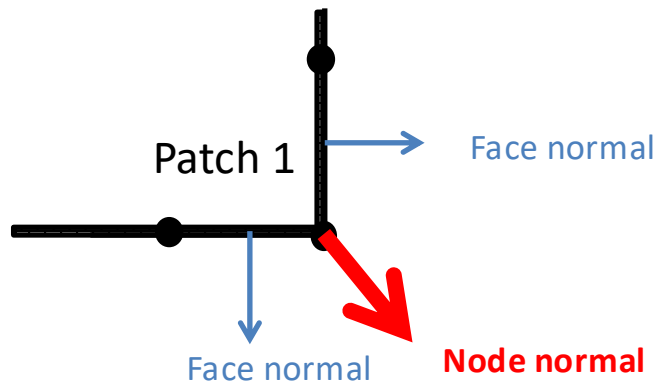
- The first treatment (patch 1) implies that the **velocity normal** to 1 is **zero**.
- The second treatment (patch 2) implies that the **velocity normal** to 2 is **zero**.

➔ $\vec{U} = \vec{0}$ at the corner (●), which is physical.

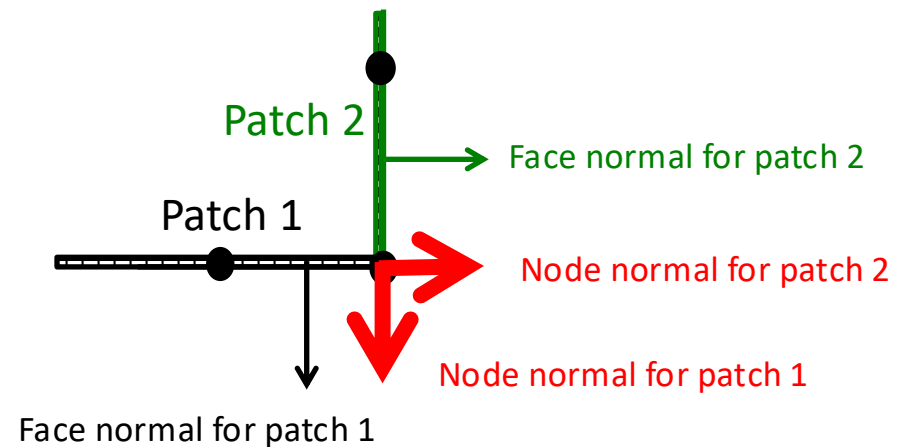


Partial conclusion on corners

One single patch



Separated patches



➔ Recommended solution : **free-corner** approach

```
!-----  
patch_name = wall_carter  
boundary_condition = WALL_LAW_ADIAB  
free_corner_min_angle = 60.0D00  
!-----
```



Theoretical insights

- Mathematical background
- Numerical methods

Description of AVBP schemes

- Lax Wendroff
- Two-step Taylor Galerkin schemes
- Properties of AVBP schemes

Practical elements

- Wiggles
- Issues at corners
- Artificial viscosity
- Shock sensors



Artificial viscosity

- Numerical schemes in AVBP are **centred schemes**. They are designed to **be low-dissipative**
- They however suffer from **dispersion errors** (as any numerical scheme)
- When Reynolds number increases, **dissipation operators may not allow to damp spurious errors**

➔ Solution adopted in AVBP: **artificial viscosity**.



Artificial viscosity

Artificial viscosity in AVBP combines **two different types of operators** :

$$\frac{d\mathbf{U}_j}{dt} = \mathbf{R}_j + \mathbf{D}_j^{(2*)} + \mathbf{D}_j^{(4*)}$$

residual

2nd order artificial
viscosity

4th order artificial
viscosity

2nd and 4th order operators rely on **sensors**

➔ Let's have a look in these operators, and sensors...

$$\frac{d\mathbf{U}_j}{dt} = \mathbf{R}_j + \mathbf{D}_j^{(2*)} + \mathbf{D}_j^{(4*)}$$

Second order AV

- 2nd order viscosity is based on a pseudo-Laplacian formulation
- Allow to damp **non-linearities**, stiff gradients

Defined in the run.params file
(jameson, colin, etc...)

Sensor
(Detect local non-linearities)

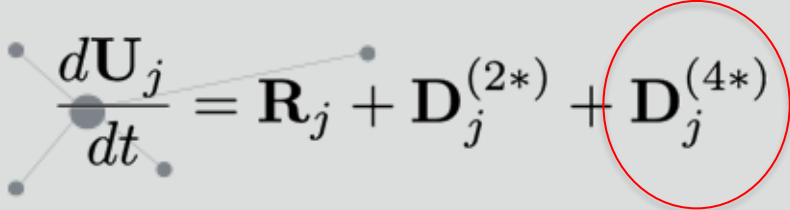
smu2

$$\mathbf{D}_j^{(2)} = \frac{1}{V_j} \sum_{e \in \mathcal{D}_j} - \frac{\epsilon^{(2)} \zeta_e V_e}{n_v^e \Delta t} (\underbrace{\bar{\mathbf{U}}_e - \mathbf{U}_j}_{\text{Difference between cell-averaged and nodal value}})$$

Nodal volume

```
artificial_viscosity_model = colin
artificial_viscosity_2nd_order = 0.01D+00
artificial_viscosity_4th_order = 0.005D+00
```


Sensor
smu2



$$\frac{d\mathbf{U}_j}{dt} = \mathbf{R}_j + \mathbf{D}_j^{(2*)} + \mathbf{D}_j^{(4*)}$$

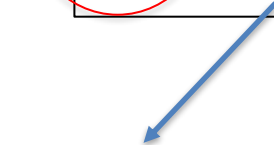
Fourth order AV

- Contrarily to 2nd order operator, 4th order operator **operates in the entire domain**
- Allows to damp **node-to-node oscillations**




$$\epsilon_e^{(4*)} = \max \left(0, \epsilon^{(4)} - \zeta_e \epsilon^{(2)} \right)$$

$$\mathbf{D}_j^{(4)} = \frac{1}{V_j} \sum_{e \in \mathcal{D}_j} \frac{\epsilon_e^{(4*)} V_e}{n_v^e \Delta t} \left[\underbrace{\left(\frac{1}{n_v^e} \sum_{k \in K_e} \vec{\nabla} \mathbf{U}_k \right) \cdot (\vec{x}_e - \vec{x}_j) - (\mathbf{U}_e - \mathbf{U}_j)}_{\text{Bi-Laplacian operator (approximates a 4th-order derivatives)}} \right]$$



 Nodal volume



 Bi-Laplacian operator
 (approximates a 4th-order derivatives)

➔ AV4 is applied everywhere, excepted in region where AV2 occurs



Sensors: Jameson

Jameson sensor at node k :

$$\zeta_k = \frac{|\Delta_1^k - \Delta_2^k|}{|\Delta_1^k| + |\Delta_2^k| + |\phi_k|}$$

With:

ϕ_k : nodal scalar quantity the sensor is based on

$$\Delta_1^k = \bar{\phi}_C - \phi_k$$

$$\Delta_2^k = \nabla \phi_k \cdot (\mathbf{x}_C - \mathbf{x}_k)$$

In AVBP, sensor applied on all conservatives variables, excepted species

Sensor at cell

$$\zeta_C^{Jameson} = \max_{k \in C} \zeta_k$$

Operators equivalence for a 1D regular mesh:

$$\Delta_1^k = \frac{\phi_{k+1} - \phi_k}{2}$$

$$\Delta_2^k = \frac{\phi_{k+1} - \phi_{k-1}}{4}$$

$$|\Delta_1^k - \Delta_2^k| = \left| \frac{1}{4} (\phi_{k+1} - 2\phi_k + \phi_{k-1}) \right|$$

- Proportional to 2nd order derivative of ϕ .
➔ Detect variations of $\vec{\nabla} \phi$ (wiggles). **Varies linearly with the perturbation**

- **However, also dissipates small scales of turbulence.**
➔ Not so adapted for LES. May be useful for initialization.



Sensors: Colin

Colin sensor:

$$\zeta_C^{Colin} = \frac{1}{2} \left(1 + \tanh \left(\frac{\Psi - \Psi_0}{\delta} \right) \right) - \frac{1}{2} \left(1 + \tanh \left(\frac{-\Psi_0}{\delta} \right) \right)$$

with

$$\begin{aligned} \Psi &= \max_{k \in C} \left(0, \frac{\Delta^k}{|\Delta^k + \epsilon_1 \phi_k|} \zeta_C^{Jameson} \right) \\ \Delta^k &= |\Delta_1^k - \Delta_2^k| - \epsilon^k \max(|\Delta_1^k|, |\Delta_2^k|) \\ \epsilon^k &= \epsilon_2 \left(1 - \epsilon_3 \frac{\max(|\Delta_1^k|, |\Delta_2^k|)}{|\Delta_1^k| + |\Delta_2^k| + |\phi_k|} \right) \end{aligned}$$

$$\begin{aligned} \Psi_0 &= 2.10^{-2} & \delta &= 1.10^{-2} \\ \epsilon_1 &= 1.10^{-2} & \epsilon_2 &= 0.95 & \epsilon_3 &= 0.5 \end{aligned}$$

- ζ_C^{Colin} **small for:** low amplitude errors, or stiff gradients well resolved by the scheme
- ζ_C^{Colin} **large for:** high amplitude numerical oscillation

➔ **Colin sensor works more in a on/off way than Jameson** AV2, AV4 applied on $(\rho, \rho E, \rho Y_k)$

- **Colin sensor is better suited for LES than Jameson**
- In AVBP, variables considered for AV with Colin allows good LES of reacting flows



Artificial Viscosity, summary

Reminder on Artificial Viscosity

- Choices for the sensor (Jameson, Colin, Colin_rhou (SLK), ...)
 - Choice for the level of 2nd order and 4th order applied (smu2, smu4)
 - Choice for the variable on which AV applies
-
- **AV is required** for LES of actual configurations...
... but an **inaccurate AV can lead to physical issues** (wrong operating point, ...)
 - Do not confuse: **subgrid-scale** viscosity and **artificial** viscosity !

See AVBP website (dedicated page) for recommendations !

Available AV options in ABVP

At present time, one can chose between 8 values for the keyword `artificial_viscosity_model` :

- **no**: no A.V. ("Academic" LES or true DNS).
- **honey**: "honey" mode, sensor fully active (equal to 1), the flow is very viscous, should only be used for transients (Initialization of the computation).
- **Jameson**: A.V. for "aerodynamics" (based on Jameson sensor).
- **Jameson_species**: A.V. for "aerodynamics" (based on Jameson sensor). An additional sensor is applied for the species mass fractions.
- **Colin**: A.V. for LES and/or combustion (based on Colin sensor).
- **Colin_species**: A.V. for LES and/or combustion (based on Colin sensor). An additional sensor is applied for the species mass fractions.
- **Colin_rhou**: A.V. for LES and/or combustion (based on Colin sensor), applying besides artificial viscosity upon pu.
- **Colin_rhou_species**: A.V. for LES and/or combustion (based on Colin sensor), applying besides artificial viscosity upon pu. An additional sensor is applied for the species mass fractions.

AVBP run.params file

\$RUN-CONTROL

solver_type = ns

diffusion_scheme = FE_2delta

simulation_end_time = 1.00000000d-03

mixture_name = AIR

reactive_flow = no

combustion_model = no

two_phase_flow = no

real_gas = no

LES_model = sigma

prandtl_turb = 0.60000000D+00

schmidt_turb = 0.60000000D+00

convection_scheme = LW

CFL = 0.90000000D+00

Fourier = 0.10000000D+00

compute_chemical_timestep = no

artificial_viscosity_model = colin

artificial_viscosity_2nd_order = 0.01D+00

artificial_viscosity_4th_order = 0.005D+00

clip_species = no

\$end_RUN-CONTROL

Euler / Navier-Stokes

Diffusion scheme.

(FE_2delta only choice since AVBP_7.7)

LW / TTG4A / TTGC (+ many others !)

CFL number, if time-step is not imposed

AV Sensor model


smu2

smu4

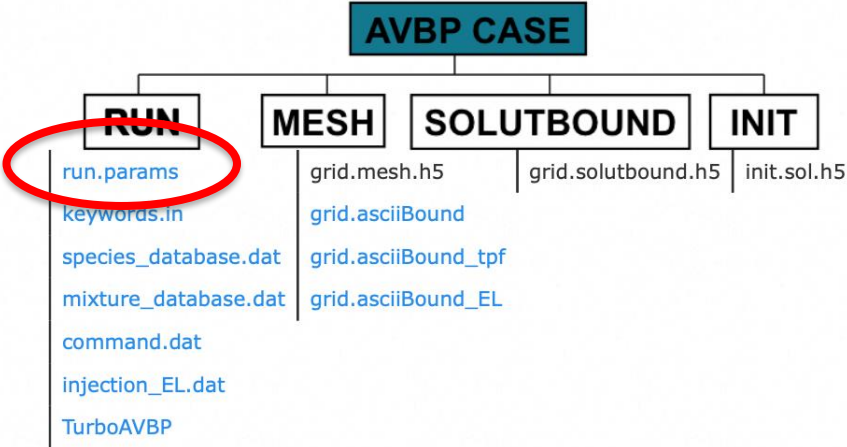
You should be able to understand, and accurately set these parameters for your case !



AVBP WEBSITE

 **AVBP website**

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[QPF table](#)
[HIP](#)
[Chemistry](#)
[Movie](#)
[Gallery](#)
[Bug store](#)
[Training](#) **2**

Input files
Home > General help > Input files
This is what a typical AVBP simulation looks like:


```
graph TD
    AC[AVBP CASE] --> RUN[RUN]
    AC --> MESH[MESH]
    AC --> SOLUTBOUND[SOLUTBOUND]
    AC --> INIT[INIT]
    RUN --> RP[run.params]
    RUN --> KI[keywords.in]
    RUN --> SD[species_database.dat]
    RUN --> MD[mixture_database.dat]
    RUN --> C[command.dat]
    RUN --> IE[injection_EL.dat]
    RUN --> TAV[TurboAVBP]
    MESH --> GMH[grid.mesh.h5]
    MESH --> GAB[grid.asciiBound]
    MESH --> GABT[grid.asciiBound_tpf]
    MESH --> GABEL[grid.asciiBound_EL]
    SOLUTBOUND --> GSH[grid.solutbound.h5]
    INIT --> ISH[init.sol.h5]
```

Several files are mandatory to perform a run with AVBP, others are optional. The following sections and links describe how to generate or fill in these files.



The parameter files


Four parameter files are needed to parametrize a computation with AVBP (two are dedicated to Euler-Lagrange simulation) :

- keywords.in** : this file is the rule file. It specifies the name of the main input file of your computation (default name = 'run.params') and it also contains a list of keyw all the options available for a computation in AVBP. This file is specific to a given version of AVBP and may be retrieved for each distribution of AVBP in the directory `$AVBP_HOME/WORK/INPUT`. For a technical description of this file, see [this page](#).



AVBP WEBSITE

<code>convection_scheme.boundary_terms</code>	character	closed unclosed	<p>Controls the boundary term treatment. Only valid for convective schemes based on a Lax-Wendroff-like method (total discretization). Default values are:</p> <ul style="list-style-type: none">• closed if <code>convection_scheme</code> = 'TTGC' or 'TTG4A'.• unclosed if <code>convection_scheme</code> = 'LW' or 'LW_FE'.• Not used otherwise. <p>More information is available in this document .</p>
<code>diffusion_scheme</code>	character	FV_4delta FE_2delta	<p>The diffusion scheme:</p> <ul style="list-style-type: none">• FV_4delta : the original finite volume 4Δ diffusion scheme.• FE_2delta : the finite element 2Δ diffusion scheme developed by O. Colin. <p>More information is available in this document .</p> <p>Default: FE_2delta if <code>solver_type</code> = 'NS'. Not used otherwise.</p>
<code>CFL</code>	double	-	<p>The CFL number used to compute the convective time-step.</p>

Very interesting pdf files are available on the website trough [this document](#)  !



Good readings

PhD thesis

- AVBP-Website
- AVBP Handbook
- AVBP QPF
- N. Lamarque
- L.M Segui-Troth
- B. Martin

<https://elearning.cerfacs.fr>

Books:

- Hirsch, C. (1990) Numerical Computation of Internal and External Flows. Wiley, Hoboken.
- G. Allaire. Analyse numérique et optimisation. Les Editions de l'Ecole Polytechnique, 2012
- J. Donea and A. Huerta. *Finite Element Methods for Flow Problems*. Wiley–Blackwell, 2003
- T. Poinso and D. Veynante. Theoretical and Numerical Combustion. R.T. Edwards Inc., 2005. Ch. 9
- R. Vichnevetsky and J. Bowles. Fourier analysis of numerical approximations of hyperbolic equations. SIAM, 1982.
- B. Després, F. Dubois. Systèmes hyperboliques de lois de conservation. Les Editions de l'Ecole Polytechnique, 2005
- Godlewski, E., and Raviart, P. A., *Numerical Approximation of Hyperbolic Systems of Conservation Laws*, Springer-Verlag, New York, 1996
- *Numerical Recipes in FORTRAN: The Art of Scientific Computing*