

Second Order Navier Stokes Solvers for Vortex Solutions in Lid-Driven Cavities

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- I The L_2 norms of error in flux integrals have been tabulated for various meshes in Table 1. The errors in approximation to equation 1 (in Problem Sheet) have been tabulated for a 10x10 mesh in Table 2. Spurious errors due to round-off were observed in even exact solution values. This, however, did not effect the significant digits in the error (where such spurious errors were seen at less significant digits).
- II Following are observations and relevant illustrations from the experiments on optimizing the code. The following results were achieved with $Re = 100$, on a 20x20 mesh.
- a No pressure drift was observed in the solution. The pressure solution converged for a variety of parameters and Reynolds numbers, and hence no fixed boundary condition was applied for pressure at a single ghost cell.
 - b To reduce pressure oscillations due to coupling of pressure in alternate lines of the mesh, a laplacian term ($A\Delta x\Delta y\nabla P$) of pressure was introduced in the continuity (pressure) equation. The value of A was varied to observe different results of smoothening as shown in Fig. 1, and Fig. 2 where slices of mesh with most oscillations ($i = 11$, and $j = 11$) have been plotted.
 - c The weight (β) given to transient pressure term in the continuity equation was varied to observe effects on convergence rate. Similarly successive over-relaxation coefficient (ω), and time steps were tuned to give optimum convergence. The individual effects of each of these independent of other parameters have been tabulated in Table 3.
- III The validation case of a square domain with moving top is explored here. The following results have been achieved without using the tuned parameters arrived at in the previous section. This was due to the fact that most of the parameters were mentioned explicitly in the question. Hence, $Re = 100$, $\beta = 1$, $A = 0$ have been used on a 20x20 mesh. Over-relaxation $\omega = 0$, and times step $\Delta t = 0.05$ were, however, varied as asked and have been reported accordingly. Convergence criterion used is for the L_2 norm of change in solution to be less than 10^{-6} .
- a Convergence history of solution with zero lide velocity has been plotted in Fig. 3. Alongside, steady state contour plots of pressure, and velocities have been plotted (Figures 4, 5, 6) to verify that the solution indeed reaches the stable values expected. There were minor oscillations in steady state pressure due to the lack of tuning parameter A . The average (over entire domain) values of respective fields were calculated as $1.9519992854088 \times 10^{-15}$, $-1.34855056741173 \times 10^{-17}$,

Mesh Size	Error (U_1)	Error (U_2)	Error (U_3)
10×10	0.0479469	0.195058	0.195058
20×20	0.0119834	0.0500259	0.0500259
40×40	0.0029957	0.0125859	0.0125859
80×80	0.000748916	0.00315144	0.00315144

Table 1: L_2 Norms of error in flux integral for solution with P_0 , u_0 , v_0 , and β set to 1.0, and $Re = 10$.

Cell Location	Error in U_1 ($\times 10^{-17}$)	Error in U_2 ($\times 10^{-12}$)	Error in U_3 ($\times 10^{-12}$)
10, 09	1.00018	4.99997	4.99994
09, 10	1.00018	4.99994	5.00008
10, 10	0	0.000100668	0.000100668
11, 10	-1.00018	-4.99998	-4.99987
10, 11	-1.00018	-4.99987	-4.99998

Table 2: Approximation error in change in flux integral with time step.

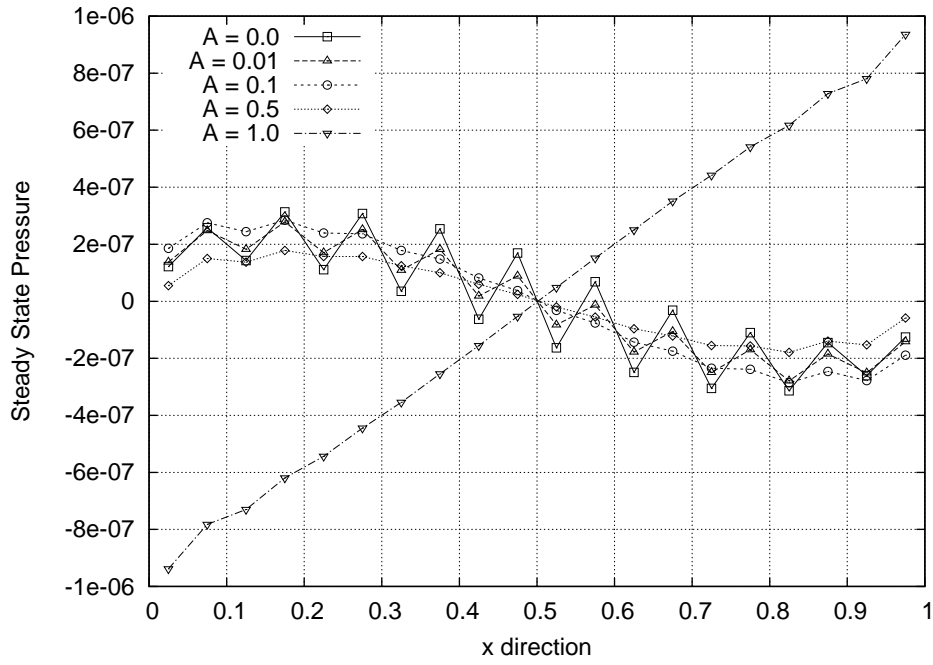


Figure 1: Pressure oscillations are damped along the x direction by adding a diffusion term. $A = 0.1$ gives best results.

β	Iterations	ω	Iterations	Δt	Iterations
0.5	232	1.0	256	0.05	256
0.6	228	1.1	246	0.10	151
0.7	224	1.2	238	0.15	123
0.8	236	1.3	231	0.20	110
0.9	249	1.4	226	0.25	103
1.0	256	1.5	224	0.30	99
1.2	281	1.6	225	0.35	98
1.3	n/a	1.7	224	0.40	98
1.4	n/a	1.8	228	0.45	100

Table 3: Convergence behaviour of solution observed on independently varying β , successive-over-relaxation parameter ω , and time step Δt , with a fixed $Re = 100$ for a mesh size of 20×20 . The optimal parameters chosen for later exercises have been highlighted.

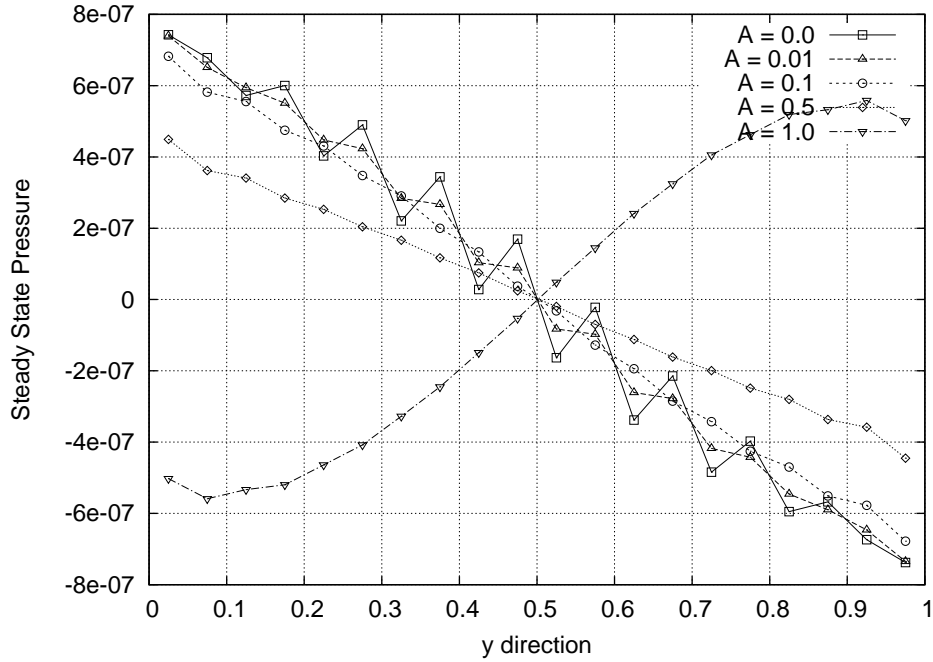


Figure 2: Pressure oscillations are damped along the y direction by adding a diffusion term. $A = 0.1$ gives best results.

$-2.90050021523882 \times 10^{-17}$. An over-relaxation coefficient ($\omega = 1.5$) was used to attain convergence in 224 steps.

- b On a 20×20 mesh, results of using a driven lid ($U_{top} = 1$) have been shown here. The convergence history has been plotted until convergence was attained (Fig. 7, 8, 9). Velocity in x-direction has been plotted for both cases ($U_{top} = 1$ and $U_{top} = -1$), as comparison, along the vertical line of symmetry in Fig. 10. A surface plot of pressure has been provided (Fig. 11) alongside.
- c In order to guarantee symmetry in solutions of forward, and backward driven lids, a surface plot of $u_{U_{top}=1}(x, y) + u_{U_{top}=-1}(1 - x, y)$ has been constructed (Fig. 12).
- d Grid convergence has been established by successively refining the mesh, and plotting velocity (Fig. 13) along the symmetry line for each case. For a quantitative reasoning, norms of change in solution across two successive meshes at 10 different points (those from the smallest mesh) were calculated, and plotted (Fig. 14). An 80×80 mesh is sufficient for grid convergence.

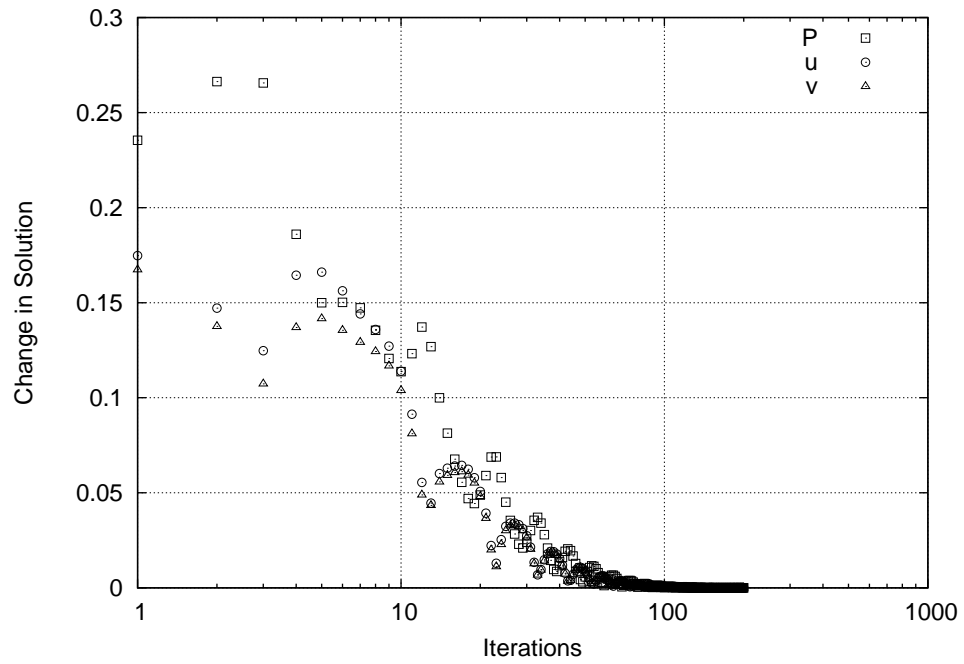


Figure 3: Change in solution (in pressure, and velocities) plotted over iterations up to 200 iterations with $\omega = 1.5$.

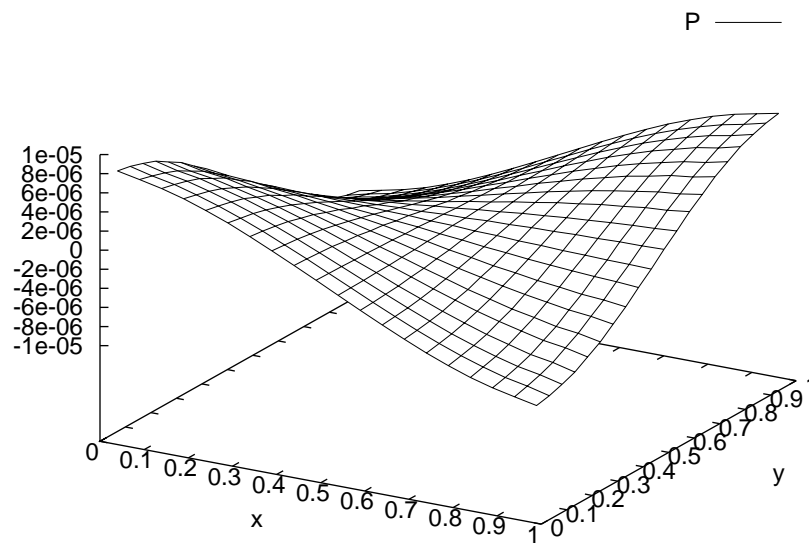


Figure 4: Steady state pressure contour plot for a 20x20 mesh with no lid velocity.

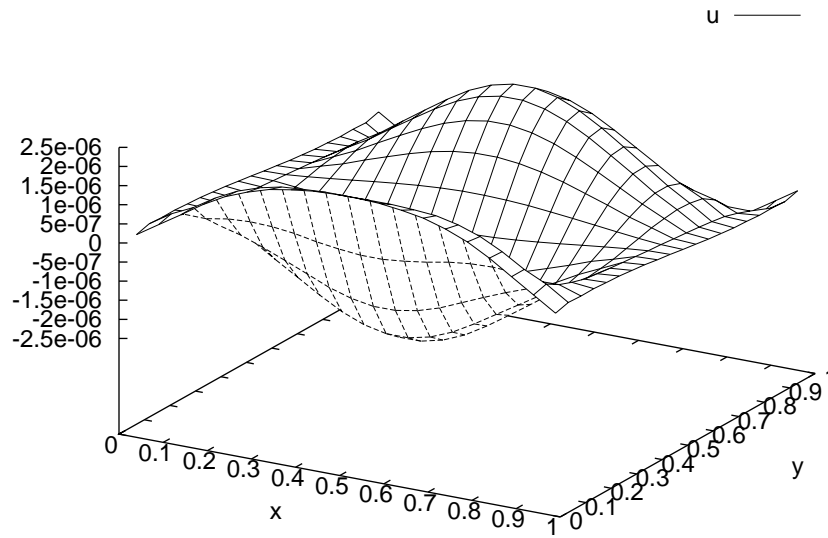


Figure 5: Steady state x velocity (u) contour plot for a 20x20 mesh with no lid velocity.

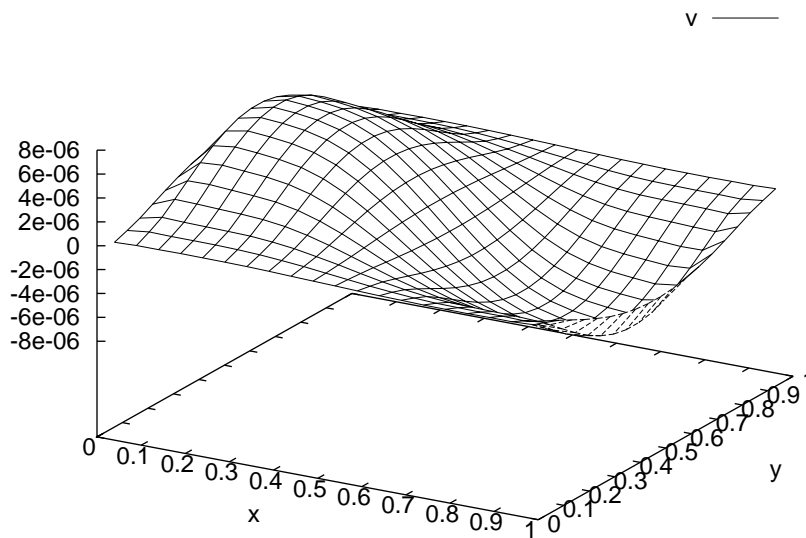


Figure 6: Steady state y velocity (v) contour plot for a 20x20 mesh with no lid velocity.

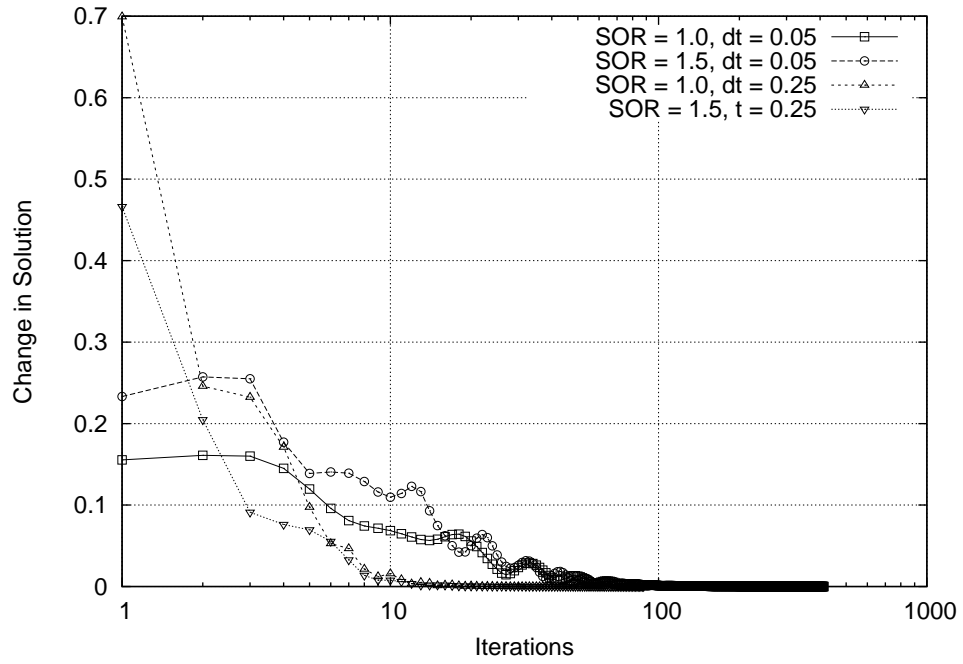


Figure 7: Convergence history of pressure for a forward driven lid case in a square mesh for different combinations of Δt , and ω .

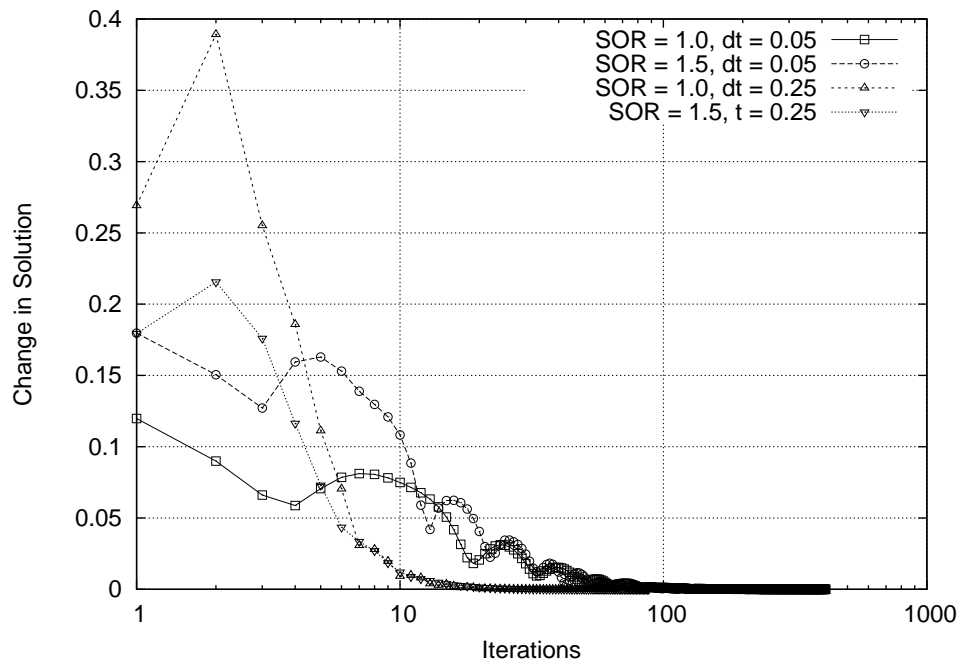


Figure 8: Convergence history of x velocity for a forward driven lid case in a square mesh for different combinations of Δt , and ω .

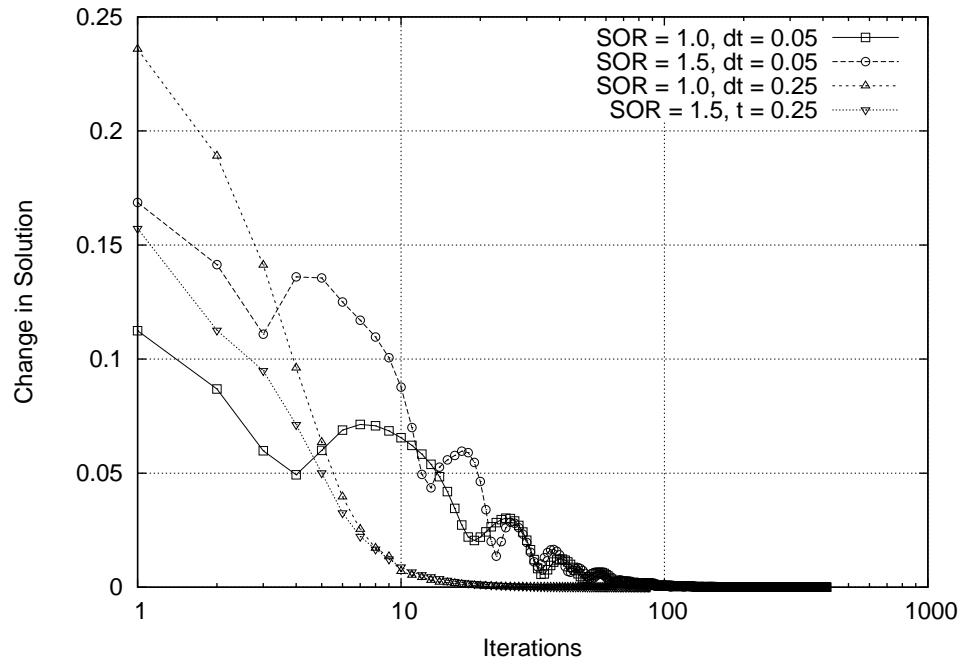


Figure 9: Convergence history of y velocity for a forward driven lid case in a square mesh for different combinations of Δt , and ω .

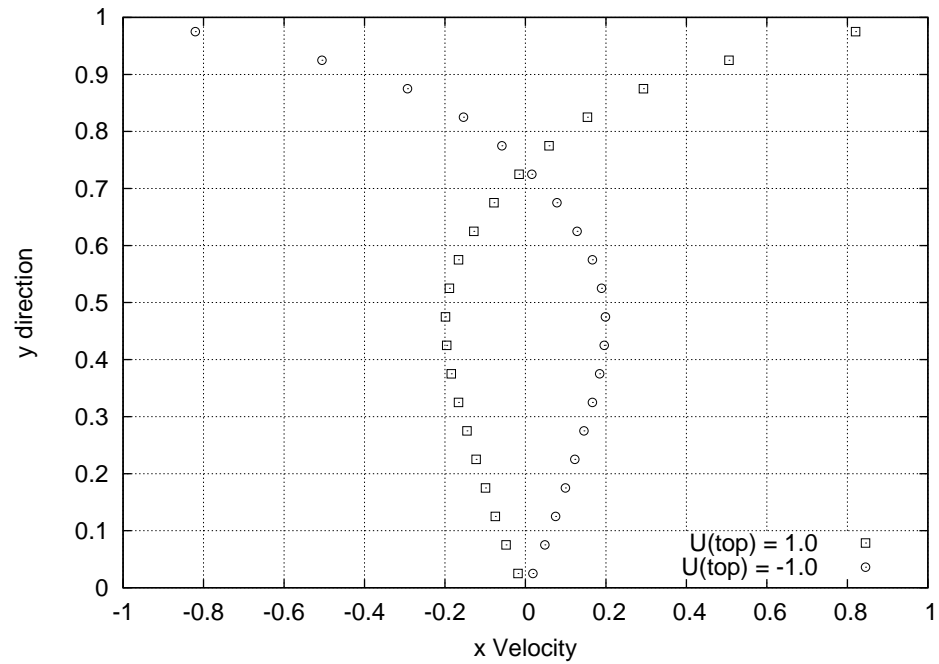


Figure 10: Plot of u along the vertical line of symmetry for both forward, and backward driven cases.

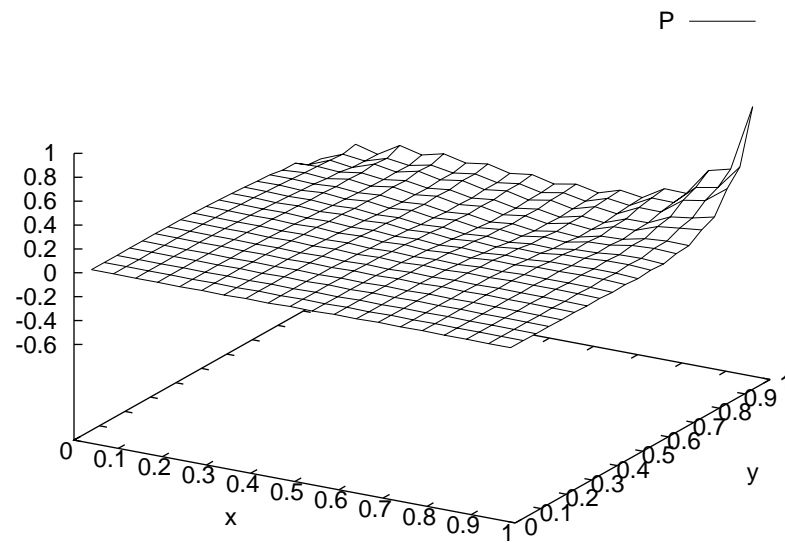


Figure 11: Surface plot of pressure for the forward driven case.

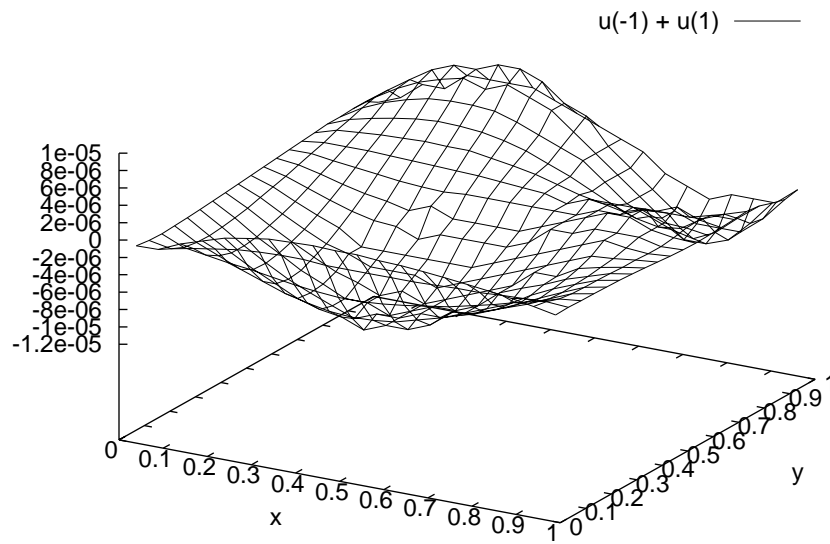


Figure 12: Surface plot of pressure for the forward driven case. Values do not attain exact zeros due to lack of complete iterative convergence.

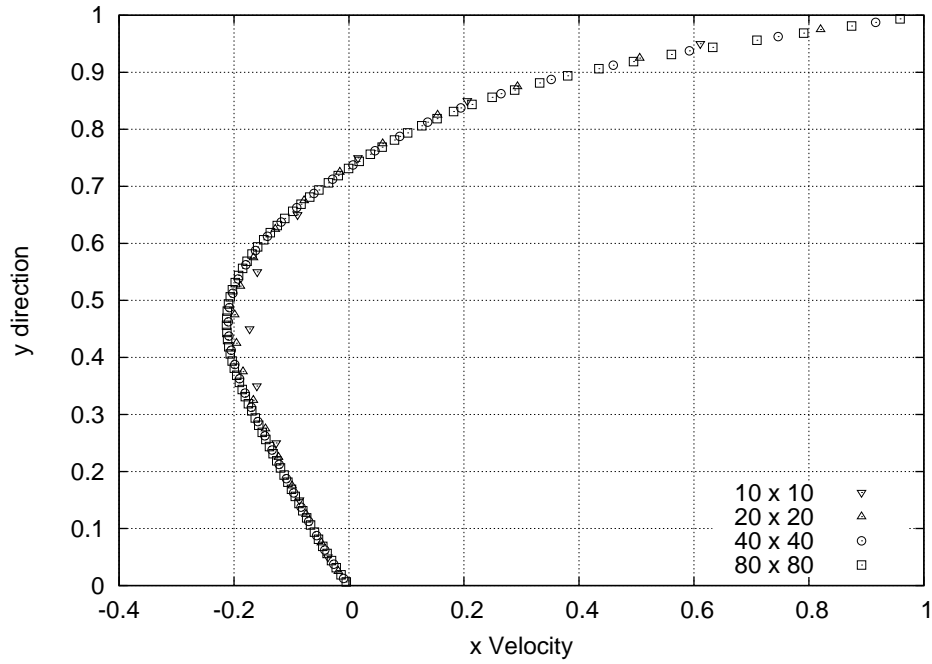


Figure 13: Plot of velocity along vertical line of symmetry for different meshes.

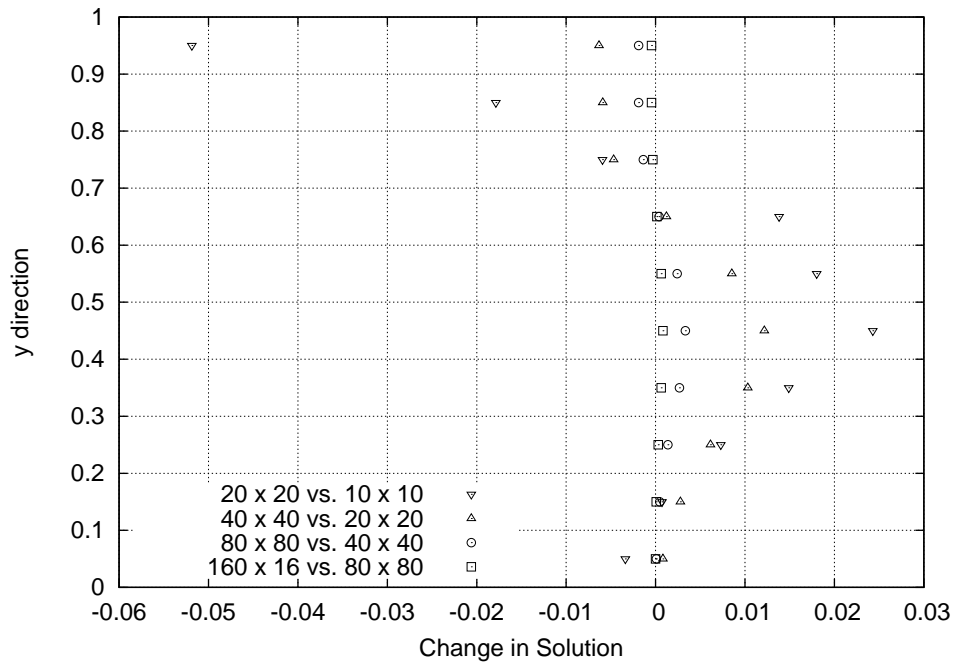


Figure 14: Plot of change in velocity estimate along vertical line of symmetry between different meshes. The L_2 norms of these changes were calculated as 0.0210647, 0.00686623, 0.00188125, and 0.000458044. The changes are diminishingly small, and also smaller by a factor of 4 for the 80x80 mesh onwards.