

Syllabus (Lugt & Lighthill: both good reading, especially Lugt - gives fresh Perspective on fluid mechanics)

Why study advanced fluid mechanics?

See nice write-up in Trends in Fluid Mechanics about applications

"Vortices are the muscles & sinews of fluid motion" Kuchemann. (T.P.)
→ instability (very important) transparencies

(To be able to do original research in fluid mechanics, i.e. basic research, need at least a reasonable understanding of several areas in fluid mech.: (typical course work:)

- Fluid mechanics (I)
- Viscous flow & boundary layer theory
- ↓
- Turbulence
- Compressible flow

- Kinetic theory
- Continuum theory of fluids ← --- (hydrodynamics, including interfacial hydrodynamics; present course but with emphasis on the physics of incompressible flow, including modern developments)
- Combustion
- Aerodynamics
- ...

Specialty courses

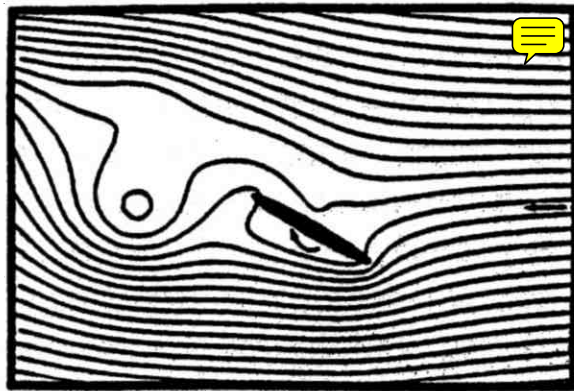
(these are all interconnected & help in clarifying the "big picture", no matter what areas of fluid mechanics you work in, be it convective htx, viscous flow, turbulence, combustion, microfluidics, etc.)

Incompressible flow: Density changes following the fluid element are negligible. (flow in the atmosphere & oceans is, for the most part, incompressible)

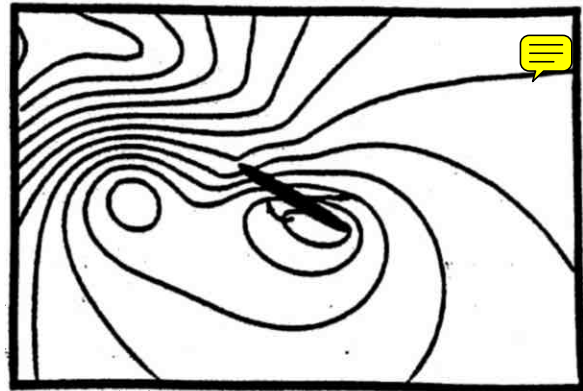
(Strictly speaking, even in a slowly flowing water stream there are density changes, but these are negligible & not dynamically important to the flow)

(Even for a gas, density (ρ) changes very little if Mach number of flow is small.)

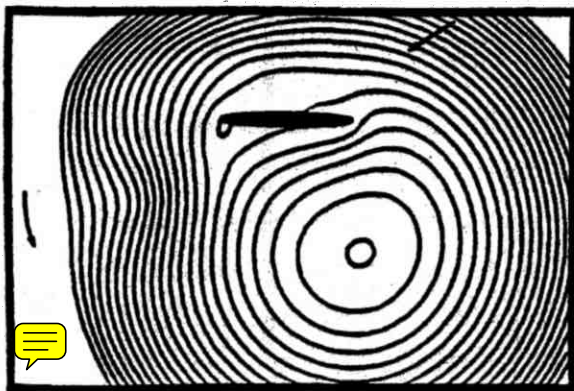
To see why) Consider the isentropic flow equations for stagnation density, ρ_0



Frame 1



Frame 2



Frame 3



Frame 4

Fig. 1.10: Computer-generated streamline patterns for the flow past a rotating elliptic cylinder (wing) in four different reference frames (Lugt and Ohring 1977).

Lugt (1996)

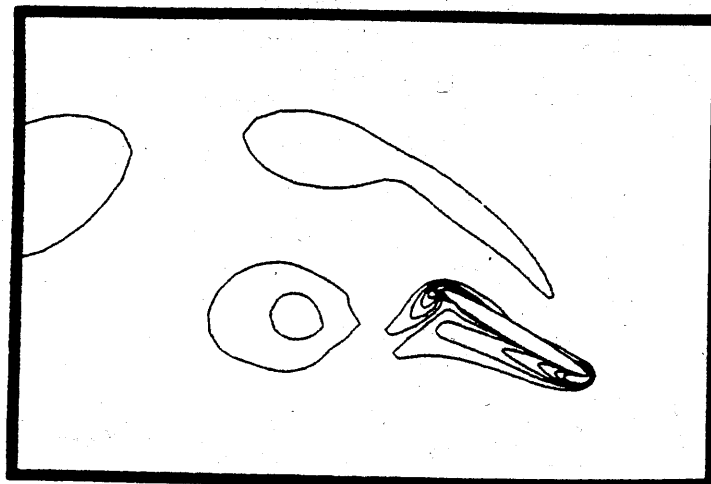


Fig. 1.11: Equivorticity lines for the flow situation displayed in Fig. 1.10 (Lugt and Ohring 1977).

$$\frac{\rho_0}{\rho} = \left(1 + \frac{\gamma-1}{2} M^2\right)^{\frac{1}{\gamma-1}}$$

where $M = \frac{V}{a}$

Mach number

V : Characteristic Velocity

a : Speed of Sound

Ratio of Constant Pressure to constant

Volume specific heat

(≈ 1.4 for diatomic gas, like air)

(eqn. applies as long as we are not doing work to our ideal gas, e.g. in a compr. or turbine ^{essentially if incompressible & can fail, but not along a nozzle})

$$\gamma = \frac{C_p}{C_v}$$

As long as flow speed is < 100 m/s (224 mph)

with corresponding $M < 0.3$ for air,

$$\frac{\Delta \rho}{\rho} < 5\% \quad \& \text{ compressibility is not very important}$$

And for liquids, speed of sound is even greater, e.g. $\gamma = 100$ m/s in

water corresponds to $M = 0.07$ (compressibility unimportant)

(not to say that this eqn applies in water!)

Note: Incompressible flow does not necessarily mean uniform

density, i.e. it can include heterogeneous flow:

Consider continuity eqn.:

$$\frac{\partial \rho}{\partial t} + \text{div}(\rho \vec{V}) = 0$$

for any fluid, compressible or otherwise

$$\Rightarrow \frac{\partial \rho}{\partial t} + \vec{V} \cdot \text{grad} \rho + \rho \text{div} \vec{V} = 0$$

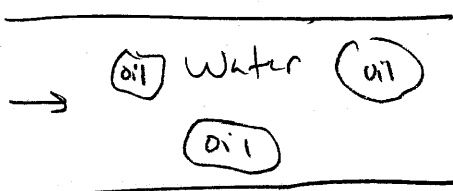
$$\frac{D\rho}{Dt} + \rho \text{div} \vec{V} = 0$$

Incompressible flow implies $\frac{D\rho}{Dt} = 0$ i.e. ^{aka ("Eulerian" or "material")} substantial derivative of density

is zero, and not always $\frac{\partial \rho}{\partial t} = 0$, since in an incompressible flow of an inhomogeneous flow, $\frac{\partial \rho}{\partial t} \neq 0$.

Example:

two-phase flow of oil & water



density at a fixed location \neq const

$$\frac{\partial \rho}{\partial t} \neq 0$$

For most of our work we'll be concerned with incompressible flow of homogeneous fluid, $\rho = \text{constant}$

● $\left(\frac{D\rho}{Dt} = \frac{\partial \rho}{\partial t} = 0 \right)$, thus continuity reduces to:

$$\text{div } \vec{V} = 0$$

(in vector notation, and in index or tensorial form:)

$$\frac{\partial u_i}{\partial x_i} = 0$$

e.g., in Cartesian coordinates:

(Summation over repeated indices is implied)

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0$$

I) Surface Waves

(See general refs. in syllabus)

Lighthill Waves in Fluids (chap 3)

Yih Fluid Mechanics

Lamb Hydrodynamics

Whitham Linear & nonlinear waves

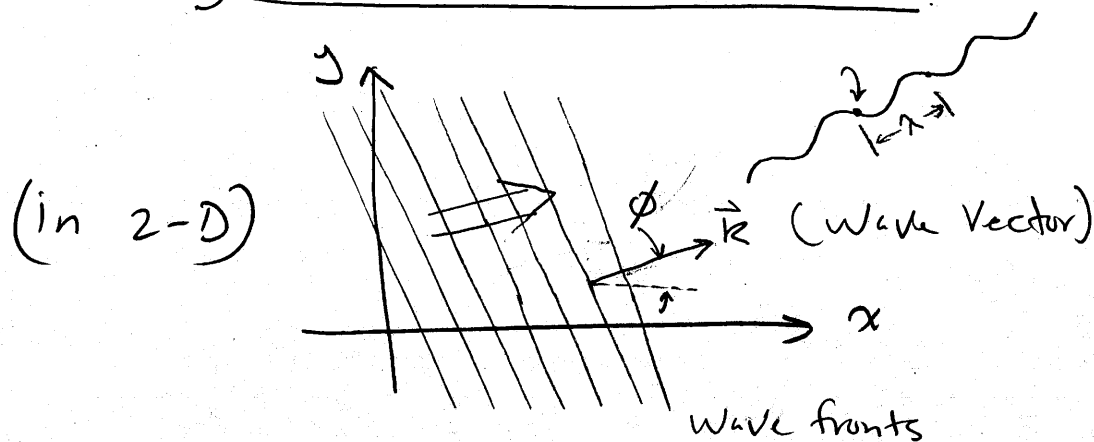
(Note that sound waves and their nonlinear counterpart, shock waves, are studied in the ^{class} compressible flow theory.

When fluid is incompressible, $a \rightarrow \infty$ and sound waves cease to be a useful concept. Here we consider surface waves, since their most important application, namely water waves, is a major topic of incompressible flow, in part because >70% of the surface of our planet is covered with water and that the transport of mass, momentum & energy between the ocean

● and the atmosphere greatly depends on water waves) & ^{of course/says} (e.g. breaking waves which cause "white caps" in ocean greatly enhance transport of CO₂)
(Water waves are in some ways more difficult than sound waves, even for the case of linear water waves, due to the

(non-dispersive nature). Similarly, non linear water waves are more difficult to analyze than nonlinear waves in the interior of a fluid, such as shock waves, again due to dispersion.

(first, some) General concepts about waves:



Wave Vector: \vec{k} Points in the direction of wave Propagation
 * its magnitude, $|\vec{k}| = \text{"Wavenumber"}$

Spatial frequency / unit length
 (# of radians) (at a fixed time)

its relation to
 Wave length, λ : $k = \frac{2\pi}{\lambda}$

(a wave can be represented by:)

$$\xi(\vec{x}, t) = A_{k, \omega} e^{i(\vec{k} \cdot \vec{x} - \omega t)}$$

location
 in 3-D
 space

time

amplitude:
 function of
 k, ω

Phase

(From Complex Variables,)
 Recall:

$$e^{i\theta} = \cos \theta + i \sin \theta \quad (\text{Can think of this as just a definition of } e^{i\theta})$$

here,

$$\Theta(\vec{x}, t) = \vec{k} \cdot \vec{x} - \omega t \equiv \text{Phase function}$$

$$\text{and } \vec{k} \cdot \vec{x} = kx \cos \phi + ky \sin \phi$$

ω is temporal frequency (# of radians/unit time)
 at a fixed location

ϕ is angle of propagation

To ride on a fixed position of wave, e.g. Crest,

$$\theta(\vec{x}, t) = \text{const.}$$

$$\Rightarrow d\theta = \vec{k} \cdot d\vec{x} - \omega dt = 0$$

In one-dim.:

$$= k dx - \omega dt = 0$$

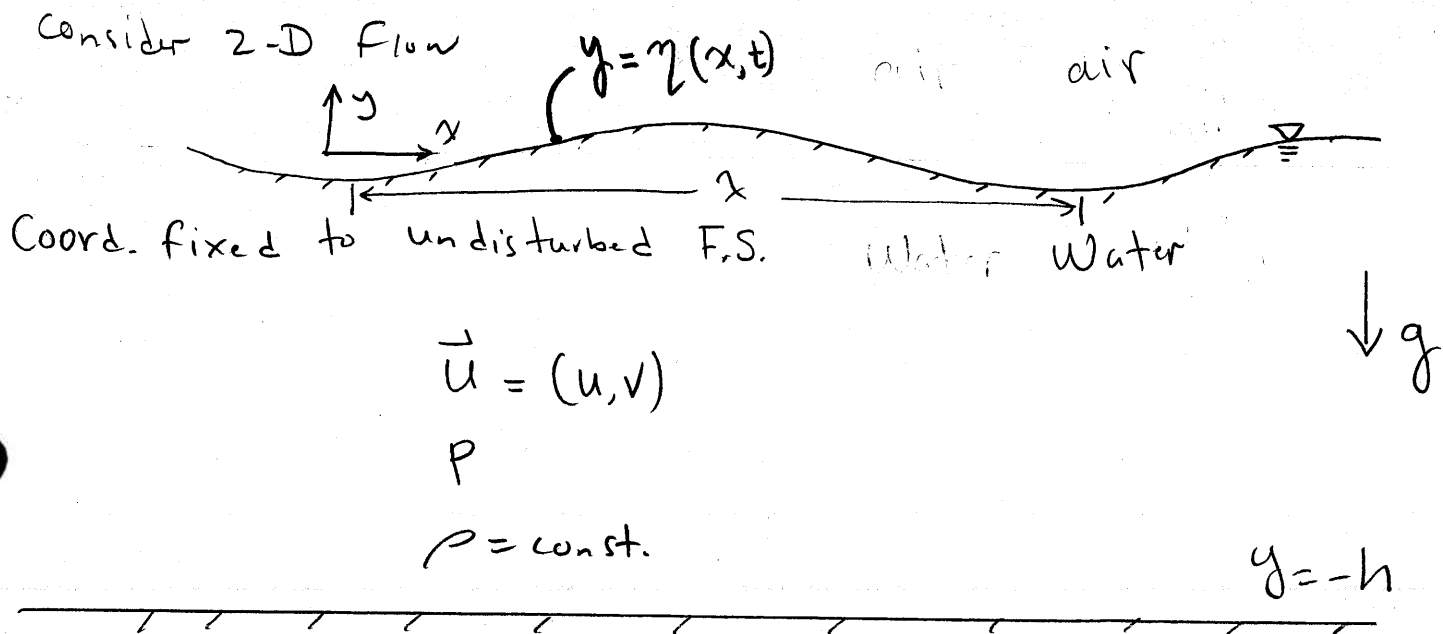
hence speed of propagation is: $\frac{dx}{dt} = \left(\frac{\omega}{k}\right) \equiv c$ "Phase speed"

Speed you have to move in the direction of the Wave Vector in order to remain at a Particular Point of the Wave.

(end of wave review)

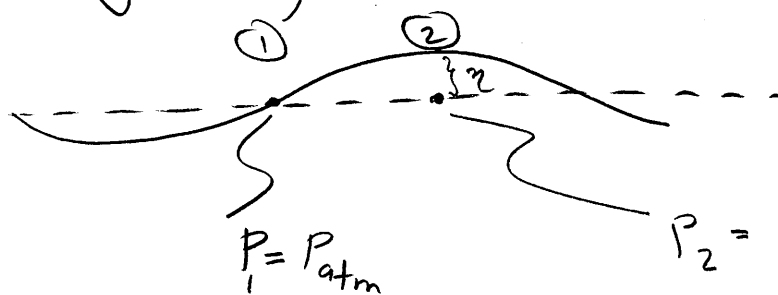
(We begin our study of surface waves by considering:)

1) Linear dispersive waves (small amplitude waves on the surface of any liquid under the influence of gravity) (we'll consider nonlinearity later)



First Q: (Why a traveling wave?)

6/1



(in absence of surface tension, considered later, or motion, considered next)

fluid at 2 will try to move to the right (sig)

(but more fundamentally, why a wave in the first place?)

(obviously, the f.s. of a liquid in equilibrium in a gravitational field is a plane. If under some external perturbation, the surface is moved from its equilibrium position at some point, motion will occur in the liquid. And like a pendulum, when perturbed, gravity does not merely return the surface back to equilibrium, but overshoots that point and creates a wave motion. (for this analysis, at least for the time being)

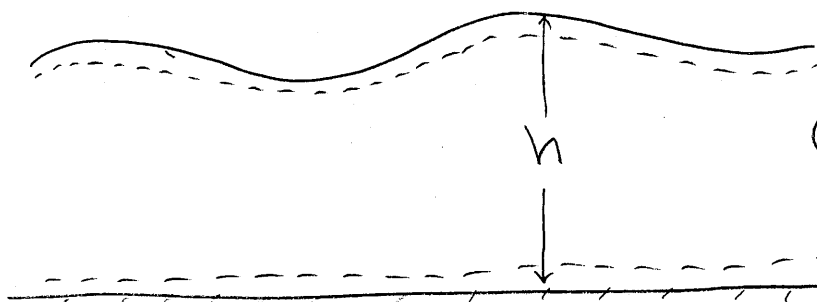
↓ 1-23-11

⇒ Assume $\mu=0$. (To be exact) when Reynolds number, $Re (= \frac{U h}{\nu})$, where $\nu = \frac{\mu}{\rho}$

is large enough, the viscous effect

is concentrated at 2, boundary regions:

recall dim. less NS: $\frac{\partial \vec{u}^*}{\partial t} + (\vec{u}^* \cdot \nabla) \vec{u}^* = -\nabla p^* + \frac{1}{Re} \nabla^2 \vec{u}^*$ (T.P.)



this BL exists since the shear stresses have to become zero, if clean (we'll discuss this limitation more later)

this BL is due to oscillatory motion next to a no-slip surface and does not grow with x .

(Similar to Stokes' 2nd Problem)

(must have)

$$Re = \frac{U h}{\nu} \gg 1$$

What to use for U ?

(turns out that)

Particle Velocity will give a deceptively low Re .

Instead, take a different point of view: one that makes the flow steady states

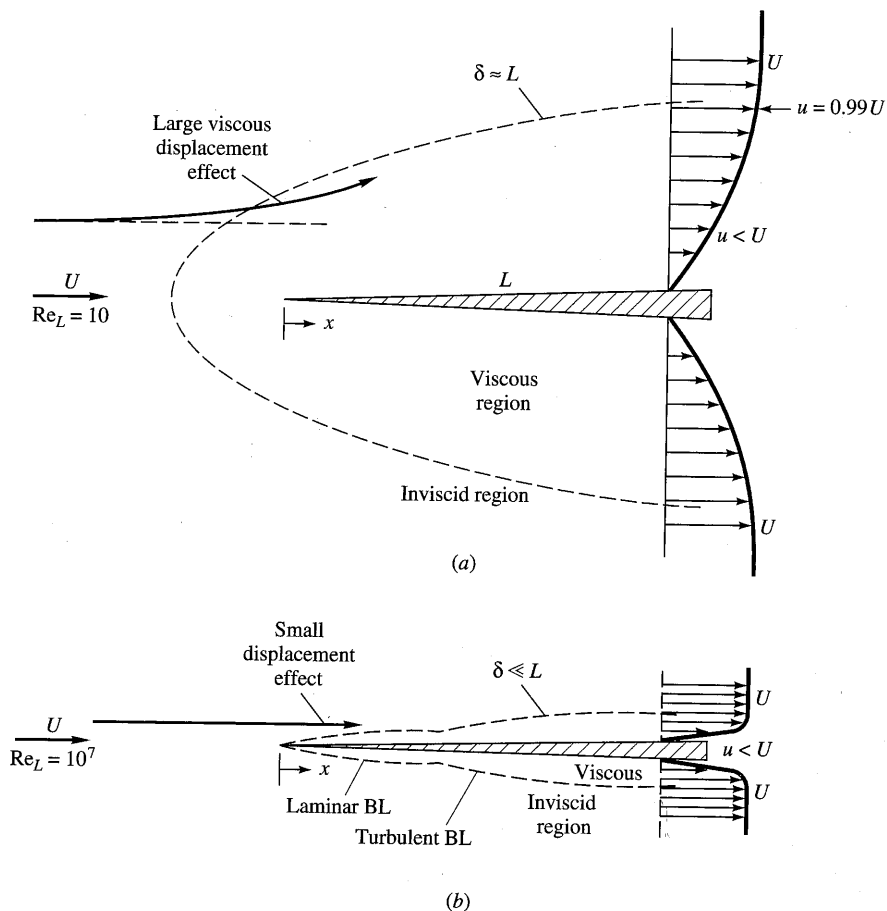


Fig. 7.1 Comparison of flow past a sharp flat plate at low and high Reynolds numbers: (a) laminar, low-Re flow; (b) high-Re flow.