

## 7. Homogeneous Equilibrium Model

### 7.1. Governing equations

The Homogeneous Equilibrium Model (HEM) for the 1D isothermal two-fluid equations read:

$$\frac{\partial}{\partial t}(\rho_g A_g) + \frac{\partial}{\partial x}(\rho_g u_m A_g) = 0 \quad (7.1)$$

$$\frac{\partial}{\partial t}(\rho_l A_l) + \frac{\partial}{\partial x}(\rho_l u_m A_l) = 0 \quad (7.2)$$

$$\frac{\partial}{\partial t}(\rho_m u_m A) + \frac{\partial}{\partial x}(\rho_m u_m^2 A + pA) + \rho_m g_s A + T_w = 0 \quad (7.3)$$

The liquid holdup  $A_l$  and gas holdup  $A_g$  have to fill up the total area of the pipe  $A$ , which allows a direct relation between  $A_g$  and  $A_l$ :  $A_l + A_g = A$ . The last term of Equation 7.3 represents the effect of wall friction which is expressed as:

$$T_w = \frac{A}{2D} f_w \rho_m u_m |u_m| \quad (7.4)$$

Here  $f_w$  is a friction factor calculated with the Churchill correlation:

$$f_w = 8 \left( \left( \frac{8}{Re} \right)^{12} + (a + b)^{-1.5} \right)^{1/12} \quad (7.5)$$

where

$$a = \left( -2.457 \log \left( \left( \frac{7}{Re} \right)^{0.9} + 0.27 \frac{\epsilon}{D} \right) \right)^{16} \quad (7.6)$$

$$b = \left( \frac{37530}{Re} \right)^{16} \quad (7.7)$$

Here  $\epsilon$  is the roughness of the pipe and  $D$  the pipe diameter.  $Re$  is the Reynolds number which is defined as

$$Re = \frac{\rho_m u_m D}{\mu_m} \quad (7.8)$$

Here  $\rho_m$  and  $\mu_m$  are the mixture density and mixture dynamic viscosity respectively, defined as

$$\rho_m = (\rho_g A_g + \rho_l A_l) / A \quad (7.9)$$

$$\mu_m = (\mu_g A_g + \mu_l A_l) / A \quad (7.10)$$

The model is closed by defining an equation of state for the gas density  $\rho_g = \rho_g(p)$  ( $\rho_l$  is taken constant,  $\rho_l = \rho_{l0}$ ):

$$\rho_g(p) = \frac{\rho_{norm} T_{norm}}{p_{norm}} \frac{p}{T} \quad (7.11)$$

### 7.1.1. IFP case

We now specify the relevant constants we use for simulating the IFP test case described in Omgba 2004. The test case describes a ramp up of the gas mass flow rate at the inlet of a 10 km pipe filled with liquid and gas. The liquid mass flow rate is constant during the ramp up with a value of 20 kg/s. The gas mass flow rate is initially 0.2 kg/s and is then doubled to 0.4 kg/s in a time span of 10 seconds in a linear fashion. As a result, a transient will move through the pipe and a new steady state is developed. The pressure at the outlet is fixed at  $p_{outlet} = 10^6$  Pa.

The constants for this case study are defined in Table 7.1. As can be noted from the table, we consider an actual fluid temperature of  $T = 278$  K.

$\rho_{norm}$	$1 \text{ kg/m}^3$
$T_{norm}$	$300 \text{ K}$
$p_{norm}$	$10^5 \text{ Pa}$
$p_{outlet}$	$10^6 \text{ Pa}$
$T$	$278 \text{ K}$
$D$	$0.146 \text{ m}$
$L$	$10 \text{ km}$
$\epsilon$	$10^{-8} \text{ m}$
$\rho_{l0}$	$1000 \text{ kg/m}^3$
$\mu_l$	$8.9 * 10^{-4} \text{ Pa s}$
$\mu_g$	$1.8 * 10^{-5} \text{ Pa s}$

**Table 7.1:** Values of relevant constants for IFP test case.

### 7.1.2. Eigenvalue analysis

The HEM described in Equation 7.1, 7.2, and 7.3 can be written in vector notation as:

$$\frac{\partial \mathbf{U}}{\partial t} + \frac{\partial \mathbf{F}}{\partial x} + \mathbf{S} = \mathbf{0}. \quad (7.12)$$

Here  $\mathbf{U}$  is the conservative variable vector,  $\mathbf{U} = [\rho_g A_g, \rho_l A_l, \rho_m u_m A]^T$ ,  $\mathbf{F}$  is the flux vector  $\mathbf{F} = [\rho_g u_g A_g, \rho_l u_l A_l, \rho_m u_m^2 A + p A]^T$ , and  $\mathbf{S}$  is the vector containing the source terms,  $\mathbf{S} = [0, 0, \rho_m g_s A + T_w]^T$

We now adopt as primitive variable vector  $\mathbf{W} = [Al, p, u_m]^T$  and define Jacobian matrix  $\mathbf{F}_\mathbf{W} = \frac{\partial \mathbf{F}}{\partial \mathbf{W}}$  and  $\mathbf{U}_\mathbf{W} = \frac{\partial \mathbf{U}}{\partial \mathbf{W}}$ :

$$\mathbf{F}_\mathbf{W} = \begin{pmatrix} -u_m \rho_g(p) & A_g u_m \frac{\partial}{\partial p} \rho_g(p) & A_g \rho_g(p) \\ u_m \rho_l(p) & A_l u_m \frac{\partial}{\partial p} \rho_l(p) & A_l \rho_l(p) \\ u_m^2 \frac{\partial}{\partial A_l} \rho_m(p, A_l) A & \left( u_m^2 \frac{\partial}{\partial p} \rho_m(p, A_l) + 1 \right) A & 2 u_m \rho_m(p, A_l) A \end{pmatrix} \quad (7.13)$$

$$\mathbf{U}_\mathbf{W} = \begin{pmatrix} -\rho_g(p) & A_g \frac{\partial}{\partial p} \rho_g(p) & 0 \\ \rho_l(p) & A_l \frac{\partial}{\partial p} \rho_l(p) & 0 \\ u_m \frac{\partial}{\partial A_l} \rho_m(p, A_l) A & u_m \frac{\partial}{\partial p} \rho_m(p, A_l) A & \rho_m(p, A_l) A \end{pmatrix} \quad (7.14)$$

Making use of the definition of the Jacobians, we multiply Equation 7.12 by  $\mathbf{U}_\mathbf{W}^{-1}$  and obtain the HEM in quasi linear form:

$$\boxed{\frac{\partial \mathbf{W}}{\partial t} + \mathbf{A}(\mathbf{W}) \frac{\partial \mathbf{W}}{\partial x} + \mathbf{U}_\mathbf{W}^{-1} \mathbf{S} = \mathbf{0}.} \quad (7.15)$$

where  $\mathbf{A}(\mathbf{W}) = \mathbf{U}_\mathbf{W}^{-1} \mathbf{F}_\mathbf{W}$ . We can calculate  $\mathbf{A}(\mathbf{W})$  analytically, as an analytic expression for  $\mathbf{U}_\mathbf{W}^{-1}$  exists. We will now substitute the relevant equations of state described in Section 7.1, and obtain for  $\mathbf{A}(\mathbf{W})$ :

$$\mathbf{A}(\mathbf{W}) = \begin{pmatrix} u_m & 0 & A_l \\ 0 & u_m & \frac{p A}{A_g} \\ 0 & \frac{c_g^2 A}{A_l \rho_{10} c_g^2 + A_g p} & u_m \end{pmatrix} \quad (7.16)$$

The eigenvalues and corresponding eigenvectors of  $\mathbf{A}$ , that is  $\mathbf{A} = \mathbf{R} \mathbf{\Lambda} \mathbf{R}^{-1}$ , which are used for characteristic boundary treatment, can now readily be calculated. Here  $\mathbf{\Lambda}$  contains the eigenvalues  $[\lambda_1, \lambda_2, \lambda_3]^T$  on the diagonal and  $\mathbf{R}$  contains the right eigenvectors of  $\mathbf{A}$ .

$$\mathbf{\Lambda} = \begin{pmatrix} \frac{A_g^{\frac{3}{2}} p u_m - A c_g \sqrt{p} \sqrt{A_l \rho_{10} c_g^2 + A_g p} + \sqrt{A_g} A_l c_g^2 \rho_{10} u_m}{A_g^{\frac{3}{2}} p + \sqrt{A_g} A_l c_g^2 \rho_{10}} & 0 & 0 \\ 0 & \frac{A_g^{\frac{3}{2}} p u_m + A c_g \sqrt{p} \sqrt{A_l \rho_{10} c_g^2 + A_g p} + \sqrt{A_g} A_l c_g^2 \rho_{10} u_m}{A_g^{\frac{3}{2}} p + \sqrt{A_g} A_l c_g^2 \rho_{10}} & 0 \\ 0 & 0 & u_m \end{pmatrix} \quad (7.17)$$

$$\mathbf{R} = \begin{pmatrix} -\frac{\sqrt{A_g} A_l \sqrt{A_l \rho_{10} c_g^2 + A_g p}}{A c_g \sqrt{p}} & \frac{\sqrt{A_g} A_l \sqrt{A_l \rho_{10} c_g^2 + A_g p}}{A c_g \sqrt{p}} & 1 \\ -\frac{\sqrt{p} \sqrt{A_l \rho_{10} c_g^2 + A_g p}}{\sqrt{A_g} c_g} & \frac{\sqrt{p} \sqrt{A_l \rho_{10} c_g^2 + A_g p}}{\sqrt{A_g} c_g} & 0 \\ 1 & 1 & 0 \end{pmatrix} \quad (7.18)$$

$$\mathbf{R}^{-1} = \begin{pmatrix} 0 & -\frac{\sqrt{A_g} c_g}{2 \sqrt{p} \sqrt{A_l \rho_{10} c_g^2 + A_g p}} & \frac{1}{2} \\ 0 & \frac{\sqrt{A_g} c_g}{2 \sqrt{p} \sqrt{A_l \rho_{10} c_g^2 + A_g p}} & \frac{1}{2} \\ 1 & -\frac{A_g A_l}{A p} & 0 \end{pmatrix} \quad (7.19)$$