In the name of God, the merciful, the compassionate



HOMEWORK 8 (TRACKING, DISTURBANCE AND LQR)

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15 Jan. 24

Problem 1 Description

Theory and Simulation

Solve Friedland Problem 8.4 Inverted pendulum on cart: compensator design.
 Bonus: Implement the compensator in Simulink. Consider initial conditions as [0, 0, 10°, 0]^T (ref = 0). Compare the system responses and control efforts and explain your results. Also, implement the compensator on the nonlinear system and compare the results.

Solution

1. Solve Friedland problem 8.4. Compensator design

$$\hat{\mathcal{A}} = A\hat{\mathcal{H}} + Bu + L(y - c\hat{\mathcal{H}})$$
 $u = k\hat{\mathcal{H}}$
 $\hat{\mathcal{A}} = (A - Bk - Lc)\hat{\mathcal{H}} + Ly - \hat{\mathcal{H}}(S) = (SI - A + Bk + Lc)^T L Y(S)$
 $U(S) = -k(SI - A + Bk + Lc)^T L Y(S) = -D(S)Y(S)$
 $D(S) = K(SI - A + Bk + Lc)^T L$
 $K = [-326.53 - 226.3265 - 812.0906 - 242.3265]^T$
 $L = [-11.936 - 394.51.3 - 452.0.58 - 143.6188]^T$
 $\Rightarrow D(S) = \frac{7.65 \times 1.5}{5.5} \frac{S^3 + 2.15 \times 1.7}{5.96 \times 1.065} \frac{7}{5} + 2.04 \times 1.5$
 $S^4 + 21.065 \leq \frac{3}{4} + 232.655 \leq \frac{2}{3} \cdot 81 \leq -1.2 \times 1.6$

Bonus

Systems are implemented in MATLAB Simulink and the results for both linear and nonlinear system is as follows,

1

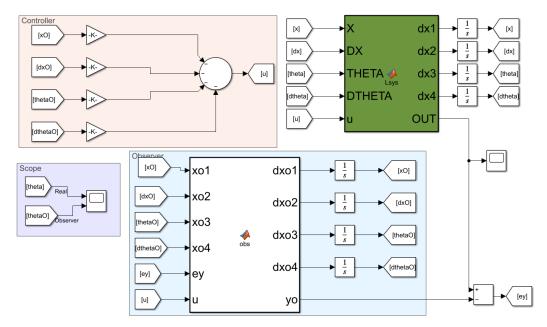


Figure 1 Linear system with compensator.

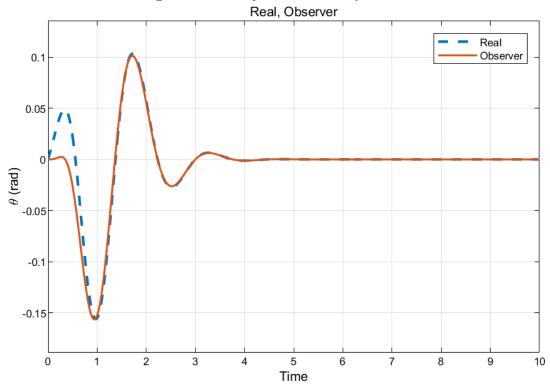


Figure 2 Linear system result.

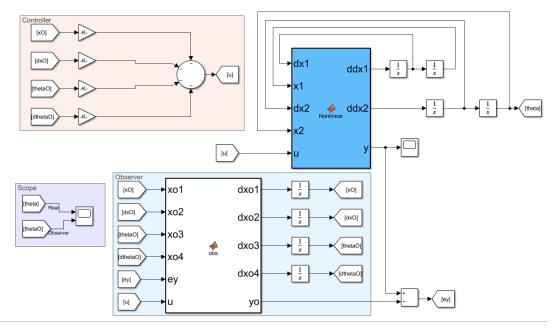


Figure 3 Nonlinear system with compensator.

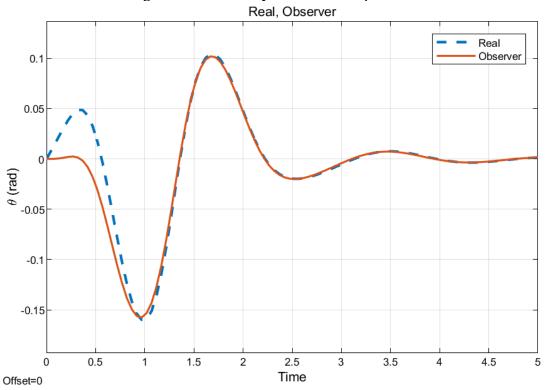


Figure 4 Nonlinear system result.

Problem 2 Description

2. Consider a system with state matrices

$$A = \begin{bmatrix} -2 & 1 \\ 0 & -3 \end{bmatrix}, \quad B = \begin{bmatrix} 1 \\ 1 \end{bmatrix}, \quad C = \begin{bmatrix} 1 & 3 \end{bmatrix}$$

2.
$$A = \begin{bmatrix} -2 & 1 \\ 0 & -3 \end{bmatrix}$$
 $B = \begin{bmatrix} 1 \\ 1 \end{bmatrix}$ $C = \begin{bmatrix} 1 & 3 \end{bmatrix}$
 $A = \begin{bmatrix} 1 & 1 \\ 0 & -3 \end{bmatrix}$ $A = \begin{bmatrix} 1$

```
C) 2-Ax+Bu=Ax+B(-Kx+NT)=(A-BK)x+BNT
   SX=(A-BK)X+NR; X=(SI-A+BK)BNR
  Y-CX -> Y-CX - C (ST A+BK) BNR
  Gu(S)= Y = C (SI A+BK) BN
Ga(6)= C(-A+BK) BN
Achie - AN= A+8A = [-2+811 1+812] - (-A+BK)-
                               -1-622
18+611+5612-5821-7622+811622-812821) -5-612
 Gd6)= 1.8 (10-3611-3612-821-822) #1.8
d) integrator
    desired >5 +75+195+21
       X=[7 19 21]
       α-a=[2 13 21]<sup>T</sup>
Q=[13 AB A<sup>2</sup>B]=
 e) = Ax+Bu -> SX = AX+BU = AX+B(-KX-K-X-)
    21-CX+r -> 5XI = CX+R -> XT = 5-C+5-R
  -> SX=AX+B(-KX-K,5-CX-K,5'R)
  -> X = (SI-A+BK+BKISTC)-(-BKISTR)
     Y=CX > Go(S)=-C (SI-A+BK+BKIST) (BK, 5")
                   = -C[(s2] - S(A+BK)+K,BC)5-17-(BK,5-)
                   =-C[s2]-S(A+BK)+KIBC]-15K15-1B
                   =-C(S2]-S(A+BK)+KIBC]-KIB
              (0) = - C(KTBC) BK+ - [7-ili-no cust
```

Bonus

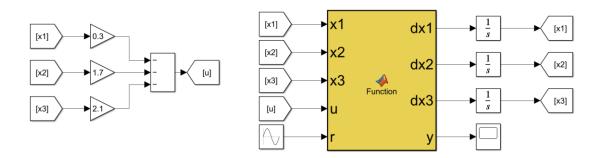


Figure 5 system with sine wave as input.

Final response of the system follows the sine wave!

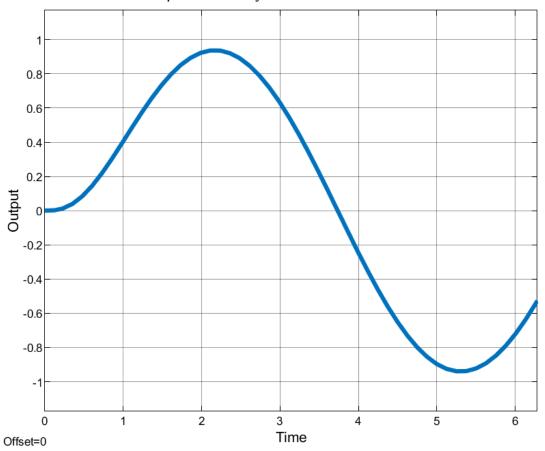


Figure 6 system result with sine wave as input.

Problem 3 Description (Bonus)

Bonus: Consider the transfer function: (you have found its full state feedback gains in HW6)

$$G(s) = \frac{-2(0.1s+1)(s+1)}{(s-2)^2(s^3+2s^2+s)}$$

$$y(t) = cx(t)$$

$$at \infty \longrightarrow o = Ax(\omega) + Bu(\omega)$$

$$\Gamma = y(\omega) = cx(\omega)$$

$$\Rightarrow x(t) = A[x(t) - x(\omega)] + B[u(t) - u(\omega)]$$

$$y(t) = -Kx(t) + u\alpha = -Kx + u(\omega) + Kx(\omega)$$

$$u(t) = -Kx(t) + [Ga(o)]^{-1}Y$$

$$u(t) = -Kx(t) + [Ga(o)]^{-1}Y$$

$$\Rightarrow K = [11, 1 29.15 27.275 11.375 1.25]^{-1}$$

$$u(t) = -Kx - KY$$

$$\Rightarrow integral control x(t) = Ax(t) + Bu(t)$$

$$y(t) = cx(t)$$

Problem 4 Description

Solve Friedland Problem 9.10 Inverted pendulum on cart: optimal gains.

$$\begin{bmatrix}
\dot{x} \\
\dot{z} \\
\dot{z}
\end{bmatrix} = \begin{bmatrix}
0 & -25 & -0.098 & 0 \\
0 & 25 & 10.178 & 0
\end{bmatrix}
\begin{bmatrix}
\chi \\
\dot{z} \\
\dot{z}
\end{bmatrix} + \begin{bmatrix}
0 & 15 \\
0 & 15
\end{bmatrix}
u$$

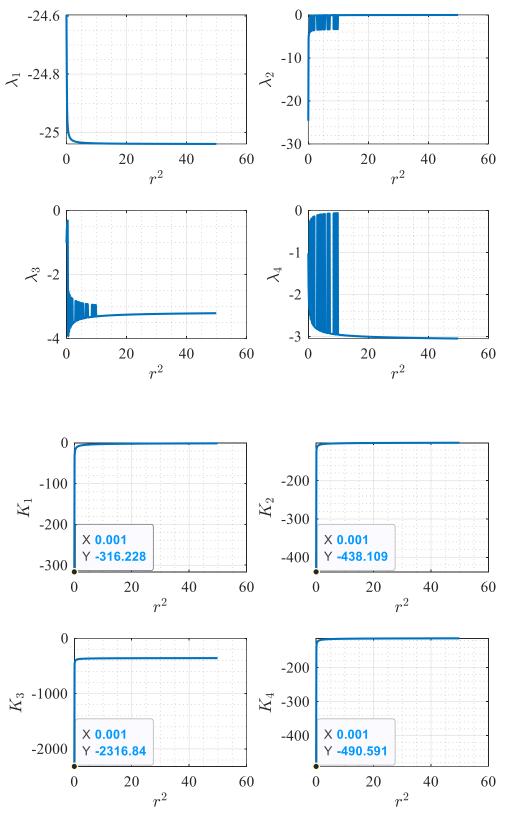
$$J = \begin{bmatrix}
0 & 1 & 0 & 0 & 0 \\
0 & 25 & 10.178 & 0
\end{bmatrix}
\begin{bmatrix}
\chi \\
\dot{z} \\
\dot{z}
\end{bmatrix} + \begin{bmatrix}
0 & 15 \\
0 & 0 & 0
\end{bmatrix}$$

$$J = \begin{bmatrix}
0 & 1 & 0 & 0 & 0 \\
0 & 25 & 10.178 & 0
\end{bmatrix}
\begin{bmatrix}
\chi \\
\dot{z} \\
\dot{z}
\end{bmatrix} + \begin{bmatrix}
0 & 0 & 0 & 0 \\
0 & 0 & 0 & 0
\end{bmatrix}$$

$$J = \begin{bmatrix}
0 & 1 & 0 & 0 & 0 \\
0 & 0 & 0 & 0
\end{bmatrix}$$

$$R = r^{2}$$

$$- R = r^{2$$



```
clc
clear
close all
M = 1;
```

```
m = 0.1;
1 = 1;
q = 9.8;
k = 1;
R = 100;
r = 0.02;
A = [0 \ 1 \ 0 \ 0; \dots]
0 - k^2/(M*r^2*R) - m*q/M 0;...
0 0 0 1;...
0 k^2/(M*r^2*R*1) (M+m)*g/(M*1) 0];
B = [0; k/(M*R*r); 0; -k/(M*R*r*l)];
q1 = 100;
q3 = 3000;
Q = [q1 \ 0 \ 0; 0 \ 0 \ 0; 0 \ 0 \ q3 \ 0; 0 \ 0 \ 0];
R = 0.001:0.1:50;
for i = 1:numel(R)
    K(i,:) = lqr(A,B,Q,R(i));
    Ac = A-B*K(i,:);
    landa(:,i) = real(eig(Ac));
end
figure(1)
subplot(4,1,1)
plot(R,K(:,1),'linewidth',1.6)
grid on
grid minor
xlabel('$r^{2}$','Interpreter','latex')
ylabel('$K {1}$','Interpreter','latex')
set(gca, 'FontSize', 12)
set(gca, 'fontname', 'Times New Roman')
subplot(4,1,2)
plot(R,K(:,2),'linewidth',1.6)
grid on
grid minor
xlabel('$r^{2}$','Interpreter','latex')
ylabel('$K {2}$','Interpreter','latex')
set(gca, 'FontSize', 12)
set(gca,'fontname','Times New Roman')
subplot(4,1,3)
plot(R,K(:,3),'linewidth',1.6)
grid on
grid minor
xlabel('$r^{2}$','Interpreter','latex')
ylabel('$K {3}$','Interpreter','latex')
set(gca, 'FontSize', 12)
set(gca, 'fontname', 'Times New Roman')
subplot(4,1,4)
plot(R,K(:,4),'linewidth',1.6)
grid on
grid minor
xlabel('$r^{2}$','Interpreter','latex')
ylabel('$K {4}$','Interpreter','latex')
```

```
set(gca, 'FontSize', 12)
set(gca, 'fontname', 'Times New Roman')
figure(2)
subplot(4,1,1)
plot(R, landa(1,:), 'linewidth', 1.6)
grid on
grid minor
xlabel('$r^{2}$','Interpreter','latex')
ylabel('$\lambda {1}$','Interpreter','latex')
set(gca, 'FontSize', 12)
set(gca,'fontname','Times New Roman')
subplot(4,1,2)
plot(R, landa(2,:), 'linewidth', 1.6)
grid on
grid minor
xlabel('$r^{2}$','Interpreter','latex')
ylabel('$\lambda {2}$','Interpreter','latex')
set(gca, 'FontSize', 12)
set(gca, 'fontname', 'Times New Roman')
subplot(4,1,3)
plot(R, landa(3,:), 'linewidth', 1.6)
grid on
grid minor
xlabel('$r^{2}$','Interpreter','latex')
ylabel('$\lambda {3}$','Interpreter','latex')
set(gca, 'FontSize', 12)
set(gca, 'fontname', 'Times New Roman')
subplot(4,1,4)
plot(R, landa(4,:), 'linewidth', 1.6)
grid on
grid minor
xlabel('$r^{2}$','Interpreter','latex')
ylabel('$\lambda {4}$','Interpreter','latex')
set(gca, 'FontSize', 12)
set(gca, 'fontname', 'Times New Roman')
```

Problem 5 Description

Consider the system below with $\omega_n = 2$, $\zeta = 0.5$:

$$A = \begin{bmatrix} 0 & 1 \\ -\omega_n^2 & -2\zeta\omega_n \end{bmatrix}, \qquad B = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

$$J = \int_0^\infty [y(t)^T Q y(t) + u(t) R u(t)] dt, \qquad Q = \begin{bmatrix} q & 0 \\ 0 & q \end{bmatrix}, \qquad R = r$$

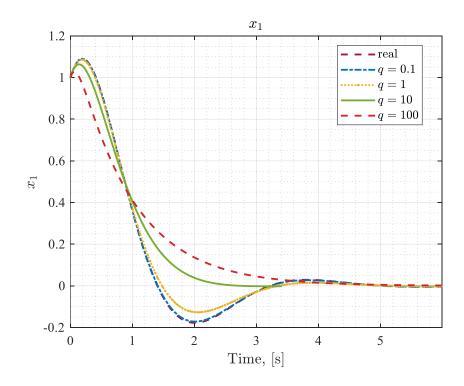
- a. Design an optimal feedback controller for the system with q=0.1,1,10,100, and r=1. Plot the system response (states) and control effort and explain your results.
- b. Design an optimal feedback controller for the system with r = 0.1, 1, 10, 100, and q = 1. Plot the system response (states) and control effort and explain your results.
- c. How can we change the cost function to design an optimal feedback controller for tracking a constant value y_d ?

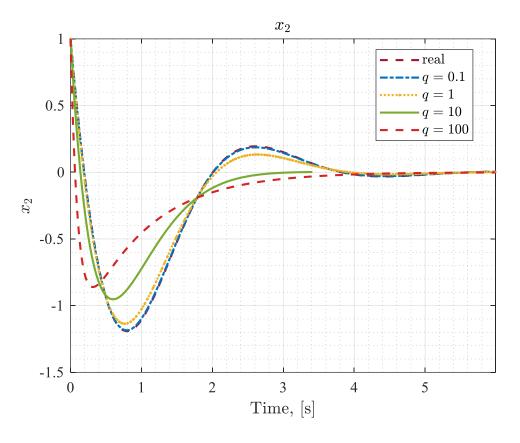
This link explains how you can implement LQR controller in MATLAB.

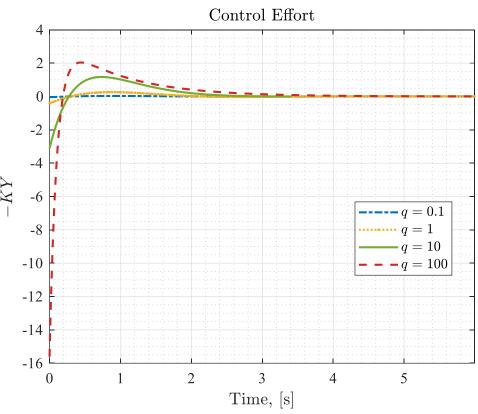
https://nl.mathworks.com/help/control/ref/lti.lqr.html#d126e97612

Solution

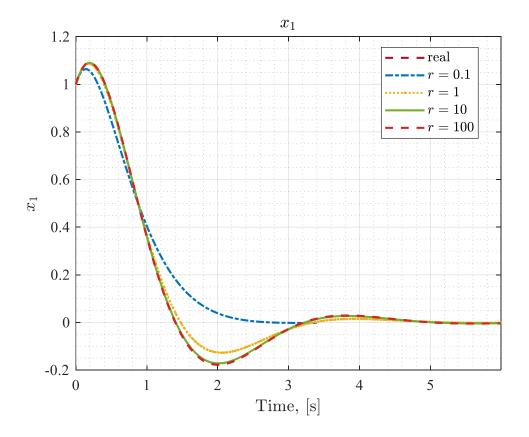
a) Increasing q value, results in increasing the control effort, as placing the poles far from the origin.

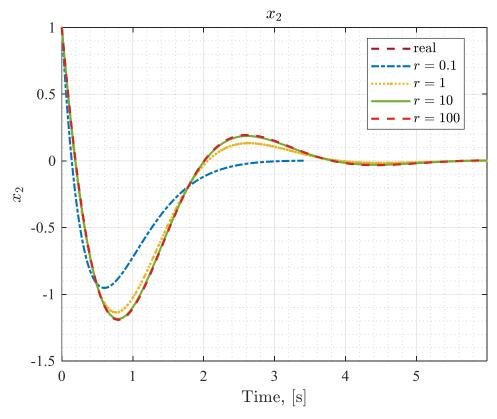


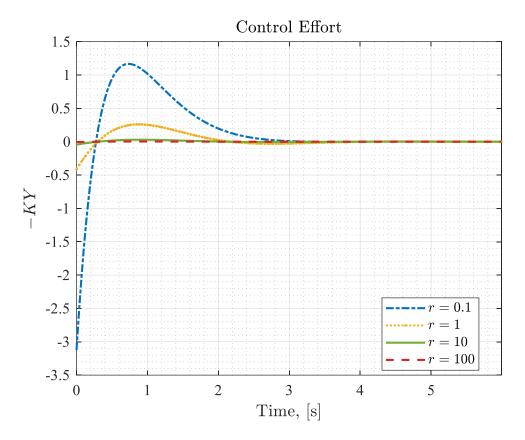




b) Increasing r value, results in decreasing the control effort, as placing the poles near the origin.







c) For feedback control, usually an integral controller would be added by increasing the states of the system, consequently, an increase in Q, matrix is inevitable! Q can be diagonally $1/\max(states)$.

Problem 6 Description

- For the system shown, determine the feedback matrix K based on EESA, which assigns the closed-loop eigenvalues.
 - a. at {-4,-5,-6} with two different choice of Eigen vectors.
 - b. at {-1+j,-1-j,-2}

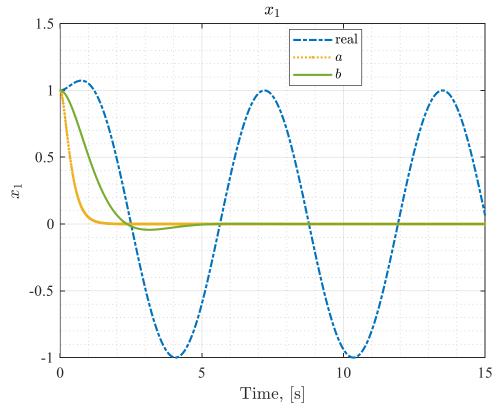
$$\dot{x} = \begin{bmatrix} -2 & 1 & 2 \\ -1 & -2 & 2 \\ -2 & 0 & 2 \end{bmatrix} x + B = \begin{bmatrix} 0 & 0 \\ 0 & 1 \\ 1 & 0 \end{bmatrix} u$$

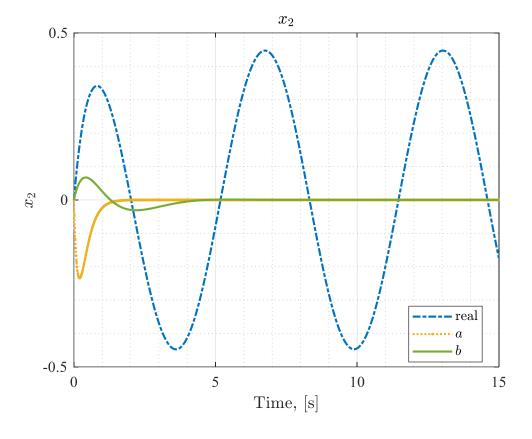
Plot the time response of all three systems with zero input, and $x(0) = \begin{bmatrix} 1 & 0 & 1 \end{bmatrix}^T$.

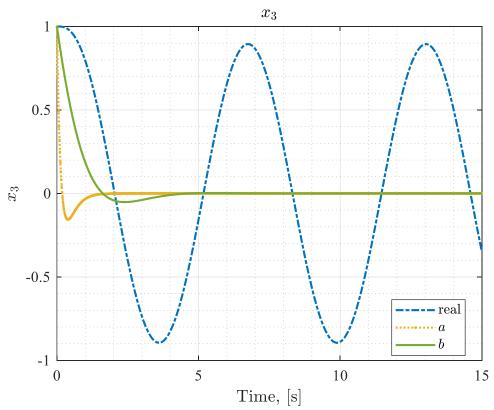
با دستور null متلب نیز بررسی شد، که جوابها متفاوت بود:)

```
clc
clear
A = [-2 \ 1 \ 2; -1 \ -2 \ 2; -2 \ 0 \ 2];
B = [0 \ 0; 0 \ 1; 1 \ 0];
m1 = [-4 -5 -6].';
A1 = [A - m1(1) * eye(3) B];
A1=null(A1)
q1=real(A1(4:end,1))
v1=real(A1(1:3,1))
A2 = [A - m1(2) * eye(3) B];
A2=null(A2)
q2=real(A2(4:end,2))
v2=real(A2(1:3,2))
A3 = [A - m1(3) * eye(3) B];
A3=null(A3)
q3=real(A3(4:end,1))
v3=real(A3(1:3,1))
```

```
k=-[q1 q2 q3]*[v1 v2 v3]^-1
place(A,B,m1)
응응 2
A = [-2 \ 1 \ 2; -1 \ -2 \ 2; -2 \ 0 \ 2];
B = [0 \ 0; 0 \ 1; 1 \ 0];
m1 = [-1+1i; -1-1i; -2].';
A1=[A-m1(1)*eye(3) B];
A1=null(A1)
q1=real(A1(4:end,1))
v1=real(A1(1:3,1))
A2=[A-m1(2)*eye(3) B];
A2=null(A2)
q2=imag(A1(4:end,1))
v2 = imag(A1(1:3,1))
A3=[A-m1(3)*eye(3) B];
A3=null(A3)
q3=real(A3(4:end,2))
v3=real(A3(1:3,2))
k=-[q1 q2 q3]*inv([v1 v2 v3])
place(A,B,m1)
```

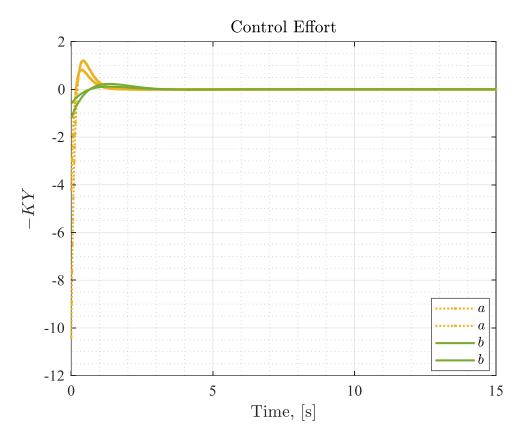






در حالت **a**، که قطبها دورتر از مرکز هستند، جواب زودتر به حالت پایدار خودش بازمیکردد، زیرا تلاش کنترلی آن، همانطور که نشان داده شده است، بیشتر خواهد بود.

در حالت real به دلیل وجود قطب در روی محور موهومی، جواب به صورت marginally stable هست.



به دلیل وجود دو سطر (دو ضریب کنترلی)، دو مقدار برای هر قسمت محاسبه شده است.