Question 1

A tube acts as a water siphon. Determine the speed of jet and the minimum pressure of water in the bend (at the point A).

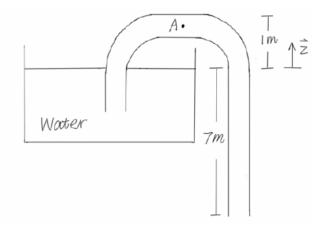


Figure 1: Tube with water siphon

Solution

Assumptions:

- Steady flow
- Incompressible flow
- Negligible viscous effects
- Negligible diameter change
- Reservoir is large enough to be considered infinite

By the Bernoulli equation, the speed of jet is given by:

$$\underbrace{\frac{P_1}{\rho} \underbrace{\frac{P_2}{\rho}}_{\text{Both exposed to atm}} + \underbrace{\frac{v_2^2}{2}}_{\text{large reservoir}} + g(z_2 - z_1) = 0$$

$$\underbrace{\frac{v_2^2}{\rho}}_{\text{large reservoir}} + g(z_2 - z_1) = 0$$

Which results in

$$v_2 = \sqrt{2g(z_1 - z_2)}$$

$$= \sqrt{2(9.81)(7)}$$

$$= 11.7 \,\mathrm{m \, s^{-1}}$$

In the pipe, by the assumptions, the velocity is constant. Therefore taking the Bernoulli equation between the pipe exit and the bend,

$$\frac{P_A}{\rho} - \frac{P_2}{\rho} + \frac{v_A^2}{2} + \frac{v_2^2}{2} + g(z_A - z_2) = 0$$
$$\frac{P_A}{\rho} - \frac{P_2}{\rho} + g(z_A - z_2) = 0$$

Rearranging for P_A ,

$$P_A = \rho g(z_2 - z_A) + P_2$$
= (1000)(9.81)(-7 - 1) + 101325
= 22.84 kPa

Question 2

A pitot-static probe is used to measure the speed of an aircraft flying at 3000 m. If the differential pressure reading is 3 kPa, determine the speed of the aircraft. (The density of the atmosphere at an elevation of 3000 m is $\rho = 0.909 \,\mathrm{kg}\,\mathrm{m}^{-3}!$)

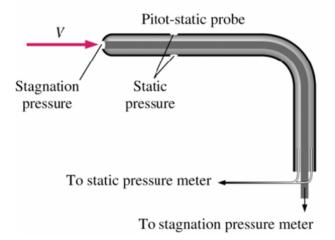


Figure 2: Pitot-static probe

Solution

Assumptions:

- Steady flow
- Incompressible flow
- Negligible viscous effects

At the stagnation point, the velocity is zero. By the Bernoulli equation,

$$\frac{P_1}{\rho} + \frac{P_2}{\rho} + \frac{v_1^2}{2} - \frac{v_2^2}{2} + g(z_1 - z_2) = 0$$
$$\frac{\Delta P}{\rho} + \frac{v_1^2}{2} = 0$$

Rearranging for v_1 ,

$$v_1 = \sqrt{2\frac{\Delta P}{\rho}}$$

$$= \sqrt{2\frac{3000}{0.909}}$$

$$= 81.2 \,\mathrm{m \, s}^{-1}$$

Question 3

Water enters a tank of diameter 1 m steadily at a mass flow rate of \dot{m} . An orifice at the bottom with diameter D_0 allows water to escape. The orifice has a rounded entrance, so the frictional losses are negligible. If the tank is initially empty, (a) determine the maximum height that the water will reach in the tank and (b) obtain a relation for water height z

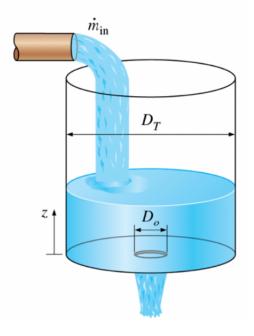


Figure 3: Water tank

Solution

(a)

Let the height of the water at the maximum height be h. At the maximum height, equilibrium is reached, so the mass flow rate in is equal to the mass flow rate out.

By the Bernoulli equation,

$$\underbrace{\frac{P_1}{\rho}}_{\text{Both at atm}} + \underbrace{\frac{v_1^2}{2}}_{\text{Sufficiently large}} - \frac{v_2^2}{2} + g(z_1 - z_2) = 0$$

$$h = \frac{v_2^2}{2g}$$

By the continuity equation,

$$\dot{m} = \rho A_0 v_2$$

$$\implies v_2 = \frac{\dot{m}}{\rho A_0}$$

$$= \frac{\dot{m}}{\rho \pi D_0^2 / 4}$$

Substituting into the Bernoulli equation,

$$h = \frac{v_2^2}{2g}$$

$$= \frac{\left(\frac{\dot{m}}{\rho \pi D_0^2/4}\right)^2}{2g}$$

$$= \frac{8\dot{m}^2}{\rho^2 \pi^2 g D_0^4}$$

(b)

Let the height of the water at time t be z(t). At a given time, the exit velocity is given by the Bernoulli equation,

$$\underbrace{\frac{P_1}{\rho}}_{\text{Both at atm}} + \underbrace{\frac{v_1^2}{2}}_{\text{Sufficiently large}} - \frac{v_2^2}{2} + g(z_1 - z_2) = 0$$

$$\implies v_2 = \sqrt{2gz(t)}$$

Accumulation of mass in the tank is given by the continuity equation,

$$\dot{V} = \dot{m} - A_0 v_2$$

Substituting expressions for V, A_0 , and v_2 ,

$$\frac{d}{dt} \left(\frac{\pi D_T^2}{4} z(t) \right) = \dot{m} - \frac{\pi D_0^2}{4} \sqrt{2gz(t)}$$
$$\frac{\pi D_T^2}{4} \dot{z}(t) = \dot{m} - \frac{\pi D_0^2}{4} \sqrt{2gz(t)}$$

Observe this is a constant coefficient, first order, non-homogeneous, non-linear differential equation. I do not know how to solve this, nor does it seem particularly useful to solve this with tools such as ODE45. As such, I will just box the implicit relationship.

$$\boxed{\frac{\pi D_T^2}{4} \dot{z}(t) = \dot{m} - \frac{\pi D_0^2}{4} \sqrt{2gz(t)}}$$

$$\theta_{wall} = A_1 \exp(-\lambda_1^2 \tau)$$

$$\implies \tau = \frac{-\ln\left(\frac{\theta_{wall}}{A_1}\right)}{\lambda_1^2}$$

$$\implies t = \frac{L^2 - \ln\left(\frac{\theta_{wall}}{A_1}\right)}{\lambda_1^2}$$