

## Question 1

The distribution of stress in an aluminum machine component is given by:

$$\sigma = \begin{bmatrix} \sigma_x & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_y & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_z \end{bmatrix} = \begin{bmatrix} y + 2z^2 & 3z^2 & 2y^2 \\ 3z^2 & x + z & x^2 \\ 2y^2 & x^2 & 3x + y \end{bmatrix} \text{ MPa}$$

Calculate the state of strain of a point positioned at  $(1, 2, 4)$ . Use  $E = 70 \text{ GPa}$  and  $\nu = 0.3$ .

### Solution

From the generalized Hooke's law, the stress-strain relation is given by:

$$\begin{aligned} \epsilon_x &= \frac{1}{E}(\sigma_x - \nu(\sigma_y + \sigma_z)) \\ \epsilon_y &= \frac{1}{E}(\sigma_y - \nu(\sigma_x + \sigma_z)) \\ \epsilon_z &= \frac{1}{E}(\sigma_z - \nu(\sigma_x + \sigma_y)) \\ \gamma_{xy} &= \frac{1}{2G}\tau_{xy} \\ \gamma_{yz} &= \frac{1}{2G}\tau_{yz} \\ \gamma_{xz} &= \frac{1}{2G}\tau_{xz} \end{aligned}$$

Where

$$G = \frac{E}{2(1 + \nu)} = \frac{70}{2(1 + 0.3)} = 26.923 \text{ GPa}$$

Evaluate the stress state at  $(1, 2, 4)$ :

$$\sigma = \begin{bmatrix} 2 + 2(4)^2 & 3(4)^2 & 2(2)^2 \\ 3(4)^2 & 1 + 4 & 1^2 \\ 2(2)^2 & 1^2 & 3(1) + 2 \end{bmatrix} = \begin{bmatrix} 34 & 48 & 8 \\ 48 & 5 & 1 \\ 8 & 1 & 5 \end{bmatrix} \text{ MPa}$$

Using the stress-strain relations, the strain state is given by:

$$\begin{aligned}
 \epsilon &= \begin{bmatrix} \frac{1}{E}(\sigma_x - \nu(\sigma_y + \sigma_z)) & \frac{1}{2G}\tau_{xy} & \frac{1}{2G}\tau_{xz} \\ \frac{1}{2G}\tau_{yx} & \frac{1}{E}(\sigma_y - \nu(\sigma_x + \sigma_z)) & \frac{1}{2G}\tau_{yz} \\ \frac{1}{2G}\tau_{zx} & \frac{1}{2G}\tau_{zy} & \frac{1}{E}(\sigma_z - \nu(\sigma_x + \sigma_y)) \end{bmatrix} \\
 &= \begin{bmatrix} \frac{1}{70 \times 10^3}(34 - 0.3(5 + 5)) & \frac{1}{2(26.923) \times 10^3}48 & \frac{1}{2(26.923) \times 10^3}8 \\ \frac{1}{2(26.923) \times 10^3}48 & \frac{1}{70 \times 10^3}(5 - 0.3(34 + 5)) & \frac{1}{2(26.923) \times 10^3}1 \\ \frac{1}{2(26.923) \times 10^3}8 & \frac{1}{2(26.923) \times 10^3}1 & \frac{1}{70 \times 10^3}(5 - 0.3(34 + 5)) \end{bmatrix} \\
 &= \begin{bmatrix} 4.43 \times 10^{-4} & 8.91 \times 10^{-4} & 1.49 \times 10^{-4} \\ 8.91 \times 10^{-4} & -9.57 \times 10^{-5} & 1.86 \times 10^{-5} \\ 1.49 \times 10^{-4} & 1.86 \times 10^{-5} & -9.57 \times 10^{-5} \end{bmatrix}
 \end{aligned}$$

## Question 2

The aluminum rectangular parallelepiped ( $E = 70$  GPa and  $\nu = \frac{1}{3}$ ) shown in Fig. 1 has dimensions  $a = 150$  mm,  $b = 100$  mm, and  $c = 75$  mm. It is subjected to tri-axial stresses  $\sigma_x = 70$  MPa,  $\sigma_y = -30$  MPa, and  $\sigma_z = -15$  MPa acting on the  $x$ ,  $y$ , and  $z$  faces, respectively. Determine,

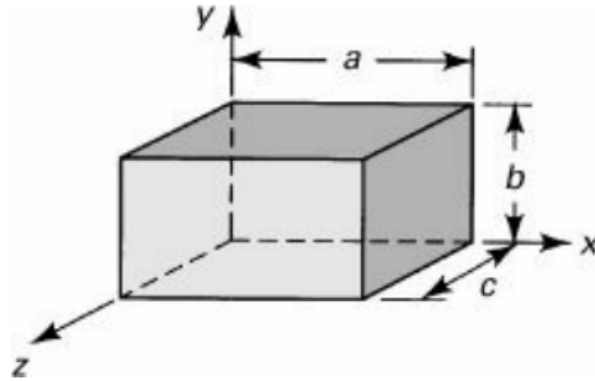


Figure 1: Rectangular parallelepiped subjected to tri-axial stresses.

- the changes  $\Delta a$ ,  $\Delta b$ , and  $\Delta c$  in the dimensions of the block.
- the change  $\Delta V$  in the volume.

**Solution**

**(a)**

From the generalized Hooke's law, the stress-strain relation is given by:

$$\begin{aligned}\epsilon_x &= \frac{1}{E}(\sigma_x - \nu(\sigma_y + \sigma_z)) \\ \epsilon_y &= \frac{1}{E}(\sigma_y - \nu(\sigma_x + \sigma_z)) \\ \epsilon_z &= \frac{1}{E}(\sigma_z - \nu(\sigma_x + \sigma_y))\end{aligned}$$

From the definition of strain,

$$\begin{aligned}\Delta x &= \epsilon_x a \\ \Delta y &= \epsilon_y b \\ \Delta z &= \epsilon_z c\end{aligned}$$

By direct substitution,

$$\begin{aligned}\Delta x &= \frac{1}{E}(\sigma_x - \nu(\sigma_y + \sigma_z))a \\ &= \frac{1}{70 \times 10^3}(70 - \frac{1}{3}(-30 - 15))150 \\ &= \boxed{0.182 \text{ mm}} \\ \Delta b &= \frac{1}{E}(\sigma_y - \nu(\sigma_x + \sigma_z))b \\ &= \frac{1}{70 \times 10^3}(-30 - \frac{1}{3}(70 - 15))100 \\ &= \boxed{-0.0690 \text{ mm}} \\ \Delta c &= \frac{1}{E}(\sigma_z - \nu(\sigma_x + \sigma_y))c \\ &= \frac{1}{70 \times 10^3}(-15 - \frac{1}{3}(70 - 30))75 \\ &= \boxed{-0.0304 \text{ mm}}\end{aligned}$$

**(b)**

The change in volume is given by:

$$\Delta V = eV_0$$

Where dilation  $e$  is given by:

$$\begin{aligned}e = \epsilon_x + \epsilon_y + \epsilon_z &= \frac{1 - 2\nu}{E}(\sigma_x + \sigma_y + \sigma_z) \\ &= \frac{1 - \frac{2}{3}}{70 \times 10^3}(70 - 30 - 15) \\ &= 1.1905 \times 10^{-4}\end{aligned}$$

Therefore,

$$\begin{aligned}
 \Delta V &= eV_0 \\
 &= (1.1905 \times 10^{-4})(150)(100)(75) \\
 &= \boxed{134 \text{ mm}^3}
 \end{aligned}$$

### Question 3

A rectangular plate is subjected to uniform tensile stress  $\sigma$  along its upper and lower edges as shown in Fig. 2. Determine the displacements  $u$  and  $v$  in terms of  $x$ ,  $y$ , and material properties ( $E$ ,  $\nu$ ) using Eqns. (1) and (2) and the appropriate conditions at the origin.

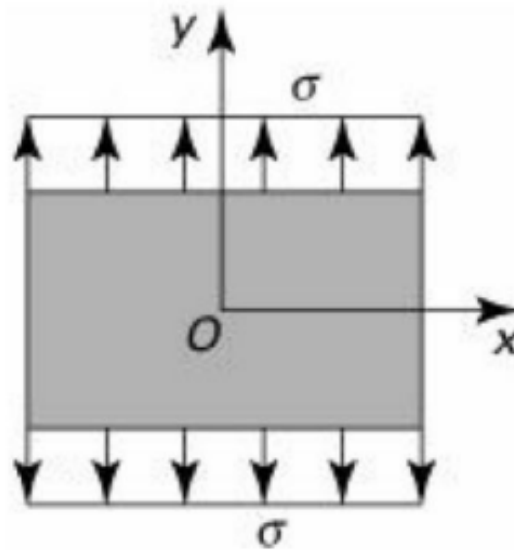


Figure 2: Rectangular plate subjected to uniform tensile stress.

$$\epsilon_x = \frac{\nabla u}{\nabla x}, \quad \epsilon_y = \frac{\nabla v}{\nabla y} \quad (1)$$

$$\gamma_{xy} = \alpha_x - \alpha_y = \frac{\nabla u}{\nabla y} + \frac{\nabla v}{\nabla x} \quad (2)$$