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## 1 Steady State Solutions for unforced SDOF Systems

 $\rm MEC~E~451~Formula~Sheet$ 

Table 1: Steady State Solutions for unforced SDOF Systems

System	Steady State Solution	
Undamped Spring Mass	$\frac{v_0}{p}\sin(pt) + x_0\cos(pt)$	
Damped Spring Mass	$e^{-\zeta pt} \left[ \frac{v_0 + \zeta p x_0}{\sqrt{1 - \zeta^2} p} \sin(\sqrt{1 - \zeta^2} pt) + x_0 \cos(\sqrt{1 - \zeta^2} pt) \right]$	

## 2 Steady State Solutions for forced SDOF Systems

Table 2: Steady State Solutions for forced SDOF Systems

System	Steady State	DMF	Transmissibility	
		(or Amplitude Response)		
Undamped Forced Spring Mass				
Forced Spring Mass	$\left(\frac{F_0}{k}\right) \left(\frac{1}{1 - \left(\frac{\omega}{p}\right)^2}\right) \sin(\omega t)$	$rac{\mathbb{X}}{\delta_{ ext{ST}}} = rac{1}{\left 1-\left(rac{\omega}{p} ight)^2 ight }$	$rac{1}{\left 1-\left(rac{\omega}{p} ight)^2 ight }$	
Rotating Imbalance	$\frac{\tilde{m}e\omega^2/k}{\left 1-\left(\frac{\omega}{p}\right)^2\right }\sin(\omega t-\phi)$	$rac{M\mathbb{X}}{ ilde{m}e} = rac{\left(rac{\omega}{p} ight)^2}{\left 1 - \left(rac{\omega}{p} ight)^2 ight }$	$rac{1}{\left 1-\left(rac{\omega}{p} ight)^2 ight }$	
Base Excitation	$a\left(\frac{1}{1-\left(\frac{\omega}{p}\right)^2}\right)\sin(\omega t)$	$\frac{\mathbb{X}}{\delta_{\mathrm{ST}}} = \frac{1}{\left 1 - \left(\frac{\omega}{p}\right)^2\right }$	$\frac{\left(\frac{\omega}{p}\right)^2}{\left 1-\left(\frac{\omega}{p}\right)^2\right }$	
	Dampeo	d Forced Spring Mass		
Forced Spring Mass		$\frac{\mathbb{X}}{\delta_{\text{ST}}} = \frac{1}{\sqrt{\left[1 - \left(\frac{\omega}{p}\right)^2\right]^2 + \left(2\zeta\frac{\omega}{p}\right)^2}}$		
Rotating Imbalance		$\frac{M\mathbb{X}}{\tilde{m}e} = \frac{\left(\frac{\omega}{p}\right)^2}{\sqrt{\left[1 - \left(\frac{\omega}{p}\right)^2\right]^2 + \left(2\zeta\frac{\omega}{p}\right)^2}}$		
Base Excitation	$\frac{a\sqrt{k^2 + (c\omega)^2}}{\sqrt{(k - m\omega^2)^2 + (c\omega)^2}}$	$\frac{\mathbb{X}}{a} = \frac{\sqrt{1 + \left(2\zeta\frac{\omega}{p}\right)^2}}{\sqrt{\left[1 - \left(\frac{\omega}{p}\right)^2\right]^2 + \left(2\zeta\frac{\omega}{p}\right)^2}}$	$\frac{\left(\frac{\omega}{p}\right)^2 \sqrt{1 + \left(2\zeta \frac{\omega}{p}\right)^2}}{\sqrt{\left[1 - \left(\frac{\omega}{p}\right)^2\right]^2 + \left(2\zeta \frac{\omega}{p}\right)^2}}$	
		$\frac{\mathbb{Z}}{a} = \frac{\left(\frac{\omega}{p}\right)^2}{\sqrt{\left[1 - \left(\frac{\omega}{p}\right)^2\right]^2 + \left(2\zeta\frac{\omega}{p}\right)^2}}$		

General solution for

$$m_{\text{eff}}\ddot{x} + c_{\text{eff}}\dot{x} + k_{\text{eff}}x = F_0\sin(\omega t - \alpha)$$

is given by

$$x(t) = \mathbb{X}\sin(\omega t - \alpha - \phi)$$

where

$$\mathbb{X} = \frac{F_0}{\sqrt{\left(k_{\text{eff}} - m_{\text{eff}}\omega^2\right)^2 + \left(c_{\text{eff}}\omega\right)^2}}$$

## 3 Steady State Solutions for Forced Non-Harmonic SDOF Systems

The general steady state solution for a periodic  $(\tau = 2\pi/\omega)$  forced damped SDOF system is given by:

$$x(t) = \frac{a_0}{2k} + \sum_{j=1}^{\infty} \frac{a_j/k}{\sqrt{\left[1 - \left(\frac{\omega}{p}\right)^2\right]^2 + \left(2\zeta\frac{\omega}{p}\right)^2}} \cos(j\omega t - \phi_j)$$
$$+ \sum_{j=1}^{\infty} \frac{b_j/k}{\sqrt{\left[1 - \left(\frac{\omega}{p}\right)^2\right]^2 + \left(2\zeta\frac{\omega}{p}\right)^2}} \sin(j\omega t - \phi_j)$$

where

$$\phi_j = \tan^{-1} \left[ \frac{2\zeta \frac{\omega}{p}}{1 - \left(\frac{j\omega}{p}\right)^2} \right]$$

$$a_0 = \frac{2}{\tau} \int_0^{\tau} F(t) dt = 2F_{\text{avg}}$$

$$a_j = \frac{2}{\tau} \int_0^{\tau} F(t) \cos(j\omega t) dt, \quad j = 1, 2, 3, \dots$$

$$b_j = \frac{2}{\tau} \int_0^{\tau} F(t) \sin(j\omega t) dt, \quad j = 1, 2, 3, \dots$$

Todo: add common solutions such as step, ramp,

## 4 Transient Response of SDOF Systems