

① Consider the following boundary-value problem:

$$U_{xx} - P(P-1)x^{(P-2)} = 0, \quad 0 < x < 1$$

$$\left. \begin{aligned} U_{x}(0) &= 0 \\ U(1) &= 1 \end{aligned} \right\} \text{constraints / BC's}$$

Where P is a given constant.

Note:

$$\frac{d^2U}{dx^2} = U_{xx}$$

$$\frac{dU}{dx} = U_x$$

② Obtain the exact solution to this problem for $P=5$. Sketch.

$$U_{xx} - P(P-1)x^{(P-2)} = 0$$

$$U_{xx} - 5(5-1)x^{5-2} = 0 \rightarrow U_{xx} - 20x^3 = 0 \rightarrow \frac{d^2U}{dx^2} - 20x^3 = 0$$

$$U_{xx} = (20x^3)$$

$$U_x = \int 20x^3 dx = \frac{20}{4}x^4 + C_1 = (5x^4 + C_1)$$

$$U = \int 5x^4 + C_1 = \frac{5}{5}x^5 + \frac{C_1}{1}x + C_2 = (x^5 + C_1x + C_2)$$

$$U = x^5 + C_1x + C_2 \rightarrow \text{apply the BC's}$$

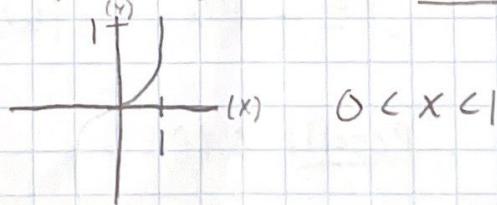
$$\underline{-U_x(0) = 0} \rightarrow U_x(0) = 0$$

$$U_x = 5x^4 + C_1 \rightarrow U_x(0) = (5(0)^4 + C_1) \rightarrow U_x(0) = C_1 \rightarrow \underline{C_1 = 0}$$

$$\underline{U(1) = 1}$$

$$U = x^5 + C_1x + C_2 \rightarrow U(1) = (1)^5 + (0)(1) + C_2 = 1 + C_2 \rightarrow \underline{C_2 = 0}$$

exact solution: $(U(x) = x^5)$



(b) State the weak formulation of the problem and show its equivalence with the strong form.

*Note: (S) $U_{xx} + f = 0$ is Ω & $U(1) = g$, & $U_{xx}(0) = -h$

Dirichlet BC Neumann BC

$$(W) \int_0^1 w_{xx} u_{xx} dx = \int_0^1 wf dx + w(0)h$$

$$a(w, v) = (w, f) + w(0)h \quad / \quad \begin{matrix} \text{trial solution space} \\ \text{trial solutions} \end{matrix}$$

$$\boxed{\text{Find } U \in S, \quad w \in V :} \quad S : \left\{ U \mid U \in H^1, \quad U(1) = g \right.$$

$$\text{test function space } V : \left\{ w \mid w \in H^1, \quad w(1) = 0 \right.$$

$$\left. \begin{array}{l} U_{xx} = 20x^3 \\ U_x = 5x^4 \\ U = x^5 \end{array} \right\} w / \quad \text{BC's} \quad \left[\begin{array}{l} -U_x(0) = 0 \\ U(1) = 1 \end{array} \right]$$

Equivalence b/w the Strong Form (S) and Weak Form (W)

↳ Assume that U is a solution of S , then show that U is a solution of (W) as well

$$(W) \quad a(w, v) = (w_{xx}, v_{xx}) = \int_0^1 w_{xx} v_{xx} dx = \int_0^1 wf dx + w(0)h$$

$$\int_0^1 w_{xx} v_{xx} dx = \int_0^1 wf dx + w(0)h$$

$$0 = - \int_0^1 w_{xx} U_{xx} dx + \int_0^1 wf dx + w(0)h \quad \text{where } U_{xx} = 5x^4$$

$$0 = -a(w, v) + (w, f) + w(0)h \quad \text{where } h = -U_{xx}(0)$$

$$0 = -a(w, v) + (w, f) - w(0)U_{xx}(0)$$

$$\text{Recall: } w(1) = 0 \longrightarrow [wU_{xx}]_0^1 \quad \downarrow \quad \rightarrow \text{integrate this term from 0 to 1}$$

$$[\omega u_{,x}]_0^1 = \int_0^1 [\omega u_{,x}]_{,x} dx$$

$$0 = - \underbrace{\int_0^1 \omega_{,x} u_{,x} dx}_{\text{can be combined}} + \int_0^1 w f dx + \underbrace{\int_0^1 [\omega u_{,x}]_{,x} dx}_{\text{Integration by parts}}$$

Int by parts:

$$\int_0^1 \omega u_{,xx} dx = \int_0^1 (u_{,x})_{,x} dx - \int_0^1 \omega_{,x} u_{,x} dx$$

$$\int u dv = uv - \int v du$$

$$\int_0^1 \omega u_{,xx} dx = \omega u_{,x} - \int_0^1 \omega_{,x} u_{,x} dx$$

$$\int_0^1 \omega (20x^3) dx = \omega 5x^4 - \int_0^1 \omega_{,x} 5x^4 dx$$

$$0 = \int_0^1 \omega u_{,xx} dx + \int_0^1 w f dx = \int_0^1 (u_{,xx} + f) \omega dx$$

divide by ω & differentiate

$$0 = \left(\frac{d}{dx} \int_0^1 (u_{,xx} + f) \omega dx \right) h$$

(S) $0 = u_{,xx} + f$ — Strong form

$$0 = 20x^3 + f$$

(W) $\int_0^1 \omega_{,x} u_{,x} dx = \int_0^1 w f dx + \omega(0) h$

$$\int_0^1 \omega_{,x} 5x^4 dx = \int_0^1 w f dx + \omega(0) h$$

> Weak form

① State the Galerkin and Matrix formulations

*Note: (G) Introduce grid size: h

finite element spaces

\mathcal{S}^h : Finite dimensional trial solution space
 \mathcal{V}^h : finite dimensional test function space

If $v \in \mathcal{S}$ & $v^h \in \mathcal{S}^h$, then $v^h \in \mathcal{S}$

If $w \in V$ & $w^h \in V^h$, then $w^h \in V$

Constraints / BC's: If $v^h \in \mathcal{S}^h$, $w^h \in V^h$, then $\underbrace{v^h(0)=g, w^h(0)=0}_{\text{Dirichlet BCs}}$

(G) Given f, g, h , find $u^h = v^h + g^h$ where $v^h \in V$, $u^h \in \mathcal{S}^h$
such that $\forall w^h \in V^h$, $a(w^h, v^h) = (w^h, f) + w^h(0)h - a(w^h, g^h)$

Recall: $a(w^h, v^h) = (w^h, f) + w^h(0)h - a(w^h, g^h)$

$$u^h = v^h + g^h$$

Compare (G) against (w) where $u \in \mathcal{S}$ & $w \in V$

$$a(w, v) = \int_0^1 w_{,x} v_{,x} dx \rightarrow \text{Galerkin}$$

$$u \in \mathcal{S} \rightarrow \int_0^1 (u_{,x})^2 dx < \infty, w \in V \rightarrow \int_0^1 (w_{,x})^2 dx < \infty$$

$$\hookrightarrow u \in \mathcal{S} \text{ or } w \in V \text{ does not simplify } \int_0^1 w_{,x} u_{,x} dx < \infty$$

$$\text{but } V \in V, w \in V \rightarrow \int_0^1 (v_{,x})^2 dx < \infty \text{ & } \int_0^1 (w_{,x})^2 dx < \infty$$

$$\therefore a(w, v) = \int_0^1 (w_{,x} v_{,x}) dx < \infty$$

$$\rightarrow a(w^h, v^h) = (w^h, f) + w^h(0)h - a(w^h, g^h)$$

$$a(w^h, v^h) = \int_{\Omega} w_{,x}^h v_{,x}^h dx = a(w^h, v^h) = (w_{,x}^h, v_{,x}^h)_{\Omega}$$

Galerkin Solution: $a(w^h, v^h) = (w_{,x}^h, v_{,x}^h)_\Omega$

$$\int_0^1 w_{,x}^h v_{,x}^h dx = \int_\Omega w_{,x}^h v_{,x}^h dx$$

(M) $U^h = \sum_A c_A N_A(x)$ arbitrary constant
Shape functions

A: node number, $N_A(x) \in V^h / S^h$

For Galerkin method, U^h and w^h are obtained from the same function space

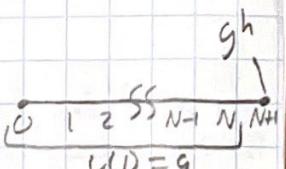
$$w^h = \sum_A c_A N_A(x), \quad c_A - \text{arbitrary constant}$$

N_A : Shape Functions / FE basis functions

$$U^h = V^h + g^h, \quad V^h \subset V^h \quad \text{test function space}$$

$$V^h = \sum_{A=0}^N c_A N_A, \quad W = \sum_{A=0}^{N+1} c_A N_A, \quad w^h = \sum_{A=0}^{N+1} c_A N_A, \quad g^h = g N_{N+1}$$

$$\begin{aligned} N_A(1) &= 0 \quad \text{for} \\ A &= 0, 1, 2, \dots, N \\ N_{N+1}(1) &= 1 \end{aligned}$$



$$N_{N+1} = \begin{cases} 1 & \text{if } x=1 \\ 0 & \text{elsewhere} \end{cases}$$

$$w^h = \sum_{A=1}^N c_A N_A + g N_{N+1} \rightarrow \text{Plug into Galerkin FEM}$$

$$a(w^h, v^h) = (w^h, f) + w^h(0)\bar{h} - a(w^h, g^h) / a(w^h, v^h) = (w^h, f) + w^h(0)\bar{h}$$

$$\sum_{A,B=0}^N c_A c_B (N_A, N_B) d\beta = \sum_{A=0}^N c_A (N_A, f) + \sum_{A=0}^N c_A N_A(0) \bar{h} - \left(\sum_{A=0}^N c_A N_A, g N_{N+1} \right)$$

$$\sum_{A=0}^N c_A \left(N_A, \sum_{B=0}^N d_B N_B, x - N_A, f - N_A(0) \bar{h} + N_A g N_{N+1} \right) = 0$$

$$\text{eqn. A} \quad \sum_{A=0}^N (C_A R_A) = 0 \quad \text{where} \quad R_A = a(N_A, \sum_{B=0}^N d_B N_B) - (N_A, f)$$

Residue at each node

$$-N_A(0)\dot{h} + (N_A, g N_{N+1})$$

BC's

* Eqn A is satisfied for all arbitrary C_A 's $\rightarrow R_A = 0$

$$a(N_A, \sum_{B=0}^N d_B N_B) = (N_A, f) + N_A(0)\dot{h} - (N_A, N_{N+1} g)$$

$$\hookrightarrow \sum_{B=0}^N N_{A,x} d_B N_{B,x} = \sum_{B=0}^N d_B (N_{A,x}, N_{B,x}) = \sum_{B=0}^N d_B a(N_A, N_B)$$

$$\sum_{B=0}^N d_B a(N_A, N_B) = (N_A, f) + N_A(0)\dot{h} - (N_A, N_{N+1} g)$$

\downarrow

IK — (Stiffness matrix)
Unknown / node displacement

Local vector / Forcing
 $\hookrightarrow \tilde{F}(N_A)$

$$IK_{AB} d_B = F$$

(G)

$$IK_{AB} = \int_0^1 N_{A,x} N_{B,x} dx = a(N_A, N_B)$$

Stiffness matrix

Stiffness matrix

$U = \sum d_A N_A$ — given the 'coefficient matrix (IK)' and a force vector, $\tilde{F}(N_A)$ find d_B , node displacement

$$d_B = F \cdot IK_{AB}^{-1}$$

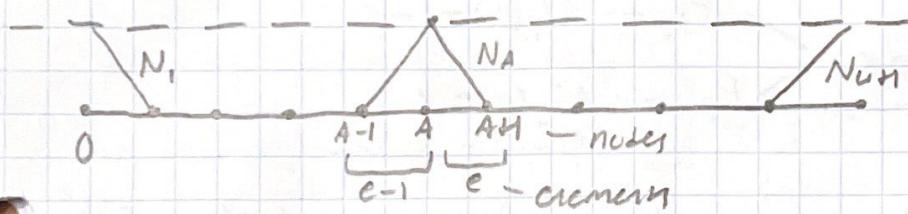
(M)

d) Solve the matrix problem for $p=5$ using piecewise linear finite element space for,

- i. Exactly one element
- ii. Two equal-length elements

Shape functions: $N_A(x) = \begin{cases} \frac{x-x_{A-1}}{h_{A-1}} & ; x_{A-1} \leq x \leq x_A \\ \frac{x_A - x}{h_A} & ; x_A \leq x \leq x_{A+1} \end{cases}$

$$h_A = h_e = x_{A+1} - x_A$$



$$K_{AB} = a(N_A, N_B) = \int_0^1 N_{A,x} N_{B,x} dx$$

we have $U_{,xx} - 20x^3 = 0$, $\underbrace{0 < x < 1}_{\text{domain}}$

BC's: $-U_{,x}(0) = 0$, $U(1) = 1$

/ /
Neumann BC Dirichlet BC

$$K_{AB} = \int_0^1 N_{A,x} N_{B,x} dx$$

$$N_1(x) = 1-x \rightarrow N_{1,x}(x) = -1$$

$$N_2(x) = x \rightarrow N_{2,x}(x) = 1$$

Calculate stiffness matrix

$$\left. \begin{array}{l} k_{11} = \int_0^1 (-1)(-1) dx = \int_0^1 dx = 1 \\ k_{12} = \int_0^1 (-1)(1) dx = \int_0^1 -dx = -1 \\ k_{21} = \int_0^1 (1)(-1) dx = \int_0^1 -dx = -1 \\ k_{22} = \int_0^1 (1)(1) dx = \int_0^1 dx = 1 \end{array} \right\} K = \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

Calculate load vector

$$F_1 = \int_0^1 20x^3 N_A(x) dx \quad \begin{cases} F_1 - N_1(x) = 1-x \\ F_2 - N_2(x) = x \end{cases}$$

$$F_1 = \int_0^1 20x^3(1-x) dx = \int_0^1 (20x^3 - 20x^4) dx = 20 \left(\frac{x^4}{4} - \frac{x^5}{5} \right) \Big|_0^1 \\ = 20 \left(\frac{1}{4} - \frac{1}{5} \right) = 5 - 4 = ①$$

$$F_2 = \int_0^1 20x^3(x) dx = \int_0^1 20x^4 dx = 20 \cdot \frac{x^5}{5} \Big|_0^1 = 4x^5 \Big|_0^1 \\ = 4(1)^5 - 4(0)^5 = ④$$

$$F = \begin{bmatrix} 1 \\ 4 \end{bmatrix}$$

$$[K_{AB}]_D = F$$

Stiffness matrix | Unknowns \ local vector

$$K \cdot v = F$$

$$\begin{pmatrix} 1 & -1 \\ -1 & 1 \end{pmatrix} \begin{pmatrix} v_1 \\ v_2 \end{pmatrix} = \begin{pmatrix} 4 \\ -4 \end{pmatrix}$$

$$\begin{aligned} v_1 - v_2 &= 1 \\ -v_1 + v_2 &= 4 \end{aligned} \quad \left. \begin{array}{l} \\ \end{array} \right\}$$

$$\underline{\text{BC's}} \quad -v_{,x}(0) = 0, \quad v(1) = 1$$

$$\begin{array}{c} / \quad | \\ \text{Neumann} \quad \text{Dirichlet} \\ x=0 \quad x=1 \end{array}$$

$$\begin{aligned} v_1 - v_2 &= 1 \rightarrow v_1 = 1 + v_2 \\ -v_1 + v_2 &= 4 \rightarrow v_2 = 4 - v_1 \end{aligned} \quad \left. \begin{array}{l} \\ \end{array} \right\} \quad v_1 = 1 + 4 - v_1$$

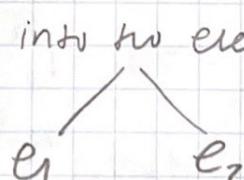
$$v(1) = 1 \rightarrow \text{fix } v_2 = 1 \quad v_x(1) = \frac{v_2 - v_1}{h} = \frac{1 - 2}{1} = -1$$

$$v_1 - v_2 = 1 \rightarrow v_1 - 1 = 1 \rightarrow v_1 = 2 \quad \text{and} \quad v_2 = 1$$

Solve for two equal length elements

Divide the domain $0 < x < 1$ into two elements

$$e_1 : [0, \frac{1}{2}]$$



$$e_2 : [\frac{1}{2}, 1]$$

$$\begin{array}{c} N_1: x_1 = 0 \\ \nearrow \quad \searrow \\ 3 \text{ nodes} \\ \nearrow \quad \searrow \\ N_2: x_2 = \frac{1}{2} \\ \nearrow \quad \searrow \\ N_3: x_3 = 1 \end{array}$$

two equal elements making up the domain

Basis Functions — linear

$$e_1 : [0, \frac{1}{2}] , e_2 : [\frac{1}{2}, 1]$$

$$\begin{array}{l} \\ X_1 = 0 \\ X_2 = \frac{1}{2} \\ \diagup = 1 @ x=0 \\ \diagdown = 0 @ x=\frac{1}{2} \end{array}$$

$$\begin{array}{l} \\ X_2 = \frac{1}{2} \\ X_3 = 1 \\ \diagup = 1 @ x=\frac{1}{2} \\ \diagdown = 0 @ x=1 \end{array}$$

$$\text{Node 1: } N_1(x_1) = 1 - 2x$$

$$\text{Node 2: } N_2(x_2) = 2x$$

$$\begin{array}{l} \\ \diagup = 1 @ x=\frac{1}{2} \\ \diagdown = 0 @ x=0 \end{array}$$

$$\text{Node 2: } N_2(x_2) = 2(1-x)$$

$$\text{Node 3: } N_3(x_3) = 2x - 1$$

$$\begin{array}{l} \\ \diagup = 1 @ x=1 \\ \diagdown = 0 @ x=\frac{1}{2} \end{array}$$

Compute local stiffness matrix

$$K_{AB} = \int_0^{1/2} N_{A,x} N_{B,x} dx$$

$$K_{11} = \int_0^{1/2} (-2)(-2) dx = 4 \cdot x \Big|_0^{\frac{1}{2}} = 2$$

$$K_{12} = \int_0^{1/2} (-2)(2) dx = -4 \cdot x \Big|_0^{\frac{1}{2}} = -2$$

$$K_{21} = \int_0^{1/2} (2)(-2) dx = -4 \cdot x \Big|_0^{\frac{1}{2}} = -2$$

$$K_{22} = \int_0^{1/2} (2)(2) dx = 4 \cdot x \Big|_0^{\frac{1}{2}} = 2$$

$$K = \begin{bmatrix} 2 & -2 \\ -2 & 2 \end{bmatrix}$$

$$e_1'$$

$$K_{11} = \int_{1/2}^1 (-2)(-2) dx = 4 \cdot x \Big|_{1/2}^1 = 4(1 - \frac{1}{2}) = 2$$

$$K_{12} = \int_{1/2}^1 (-2)(2) dx = -4 \cdot x \Big|_{1/2}^1 = -4(1 - \frac{1}{2}) = -2$$

$$K_{21} = \int_{1/2}^1 (2)(-2) dx = -4 \cdot x \Big|_{1/2}^1 = -4(1 - \frac{1}{2}) = -2$$

$$K_{22} = \int_{1/2}^1 (2)(2) dx = 4(1 - \frac{1}{2}) = 2$$

$$K = \begin{bmatrix} 2 & -2 \\ -2 & 2 \end{bmatrix}$$

$e_2 \rightarrow$

Compute Global Stiffness Matrix

$$K = K_1 + K_2$$

$$\begin{bmatrix} 2 & -2 & 0 \\ -2 & 2 & 0 \\ 0 & 0 & 0 \end{bmatrix} + \begin{bmatrix} 0 & 0 & 0 \\ 0 & 2 & -2 \\ 0 & -2 & 2 \end{bmatrix} = \begin{bmatrix} 2 & -2 & 0 \\ -2 & 4 & -2 \\ 0 & -2 & 2 \end{bmatrix}$$

Locd Vector

$$F_i = \int_0^1 U_{,xx} N_A(x) dx = \int_0^1 20x^3 N_A(x) dx$$

$$F_1 = \int_0^{1/2} 20x^3(1-2x) dx = \int_0^{1/2} 20x^3 - 40x^4 dx = \frac{20x^4}{4} - \frac{40x^5}{5} \Big|_0^{1/2}$$

$$= 5x^4 - 8x^5 \Big|_0^{1/2} = 5(\frac{1}{2})^4 - 8(\frac{1}{2})^5 = \frac{1}{16}$$

=====

$$F_2 = \int_0^{1/2} 20x^3(2x)dx = \int_0^{1/2} 40x^4 dx = 40 \frac{x^5}{5} \Big|_0^{1/2} = 8x^5 \Big|_0^{1/2}$$

$$= 8\left(\frac{1}{2}\right)^5 = \frac{1}{4}$$

$$F_2 = \int_{1/2}^1 20x^3(2-2x)dx = \int_{1/2}^1 40x^3 - 40x^4 dx = 40 \frac{x^4}{4} - 40 \frac{x^5}{5} \Big|_{1/2}^1$$

$$= 10x^4 - 8x^5 \Big|_{1/2}^1 = 10 - 0.625 - 8 + 0.25 = \underline{\underline{1.625}}$$

$$F_3 = \int_{1/2}^1 20x^3(2x-1)dx = \int_{1/2}^1 40x^4 - 20x^3 dx = 40 \frac{x^5}{5} - 20 \frac{x^4}{4} \Big|_{1/2}^1$$

$$= 8x^5 - 5x^4 \Big|_{1/2}^1 = 8 - 0.25 - 5 + 0.3125 = \underline{\underline{3.0625}}$$

$$F = \begin{pmatrix} \frac{1}{16} \\ \frac{1}{4} \\ \frac{13}{8} \end{pmatrix}$$

BC's — Neumann BC — $U_x(0) = 0 \rightarrow U_1$ is free

Dirichlet BC — $U_3 = 1 \rightarrow$ fixed value at N_3

$$K \cdot \delta B = F$$

$$\begin{bmatrix} 2 & -2 \\ -2 & 4 \end{bmatrix} \begin{bmatrix} U_1 \\ U_2 \end{bmatrix} = \begin{pmatrix} \frac{1}{16} \\ \frac{1}{4} \end{pmatrix}$$

$$\begin{aligned} 2U_1 - 2U_2 &= \frac{1}{16} \\ -2U_1 + 4U_2 &= \frac{1}{4} \end{aligned}$$

$$U_1 = U_2 + \frac{1}{32}$$

$$-2\left(U_2 + \frac{1}{32}\right) + 4U_2 = \frac{1}{4} \rightarrow -2U_2 - \frac{1}{16} + 4U_2 = \frac{1}{4}$$

$$2U_2 = \frac{1}{4} + \frac{1}{16}$$

$$U_2 = \frac{5}{32}$$

$$2U_1 = 2U_2 + \frac{1}{16}$$

$$U_1 = U_2 + \frac{1}{32} = \frac{5}{32} + \frac{1}{32}$$

$$U_1 = \frac{6}{32} = \frac{3}{16}$$

$$U_3 = 1$$

$$U_X(1) \approx \frac{U_3 - U_2}{h} = \frac{1 - \frac{5}{32}}{0.5} = 1.6875$$

② Applying Galerkin FEM for ADE

Class 4
↳ derived the global nodal equation for a generic internal node by assembling the local element stiffness matrix

↳ Showed that for a uniform mesh, the nodal equation is the same as applying the central difference (CD₂) scheme to the original ADE

* in lecture we used the ('conservative' form) of the weak form of the ADE to show the equivalence with CD₂

Demonstrate that this equivalence is retained even if we use the ('convective' form) for the convection term in the weak form of the ADE.

the moment you add convection you lose the symmetric nature of the matrix

$$(W) \quad a(w, u) = \underbrace{(w, a u_{,x})_{\Omega}}_{\text{convection term}} + \underbrace{(w_{,x}, K u_{,x})_{\Omega}}_{\text{diffusion term}}$$

$$a(w, u) = (w, f) + BC's$$

(G) $a(w^h, u^h) = (w^h, f) + BC's$

$$(M) \quad u^h(x) = \sum_A d_A N_A(x)$$

$$w^h(x) = \sum_A c_A N_A(x)$$

plug into Galerkin Form

$$\underbrace{a(N_A, N_B) d_B}_{IK_{AB}} = \underbrace{(N_A, f)}_{\text{load vector}}$$

* A & B are node #'s

$w_{,x}$ stiffness matrix

$$IK_{AB} := \underbrace{(N_A, a N_{B,x})}_{\text{convection part}} + \underbrace{(N_{A,x}, K N_{B,x})}_{\text{diffusion part}}$$

diffusion coefficient

$$K_{AB} \neq K_{BA}$$

↳ convection term: $K_{AB}^{\text{conv}} := (N_A, a N_{B,x})$
 $K_{BA}^{\text{conv}} := (N_B, a N_{A,x})$] not equal

$$IK_{AB}^{\text{conv}} = -IK_{BA}^{\text{conv}}, \quad IK_{AB} = IK_{AB}^{\text{conv}} + IK_{AB}^{\text{diff}}$$

skew-symmetric symmetric

Weak Form for ADE

Find $U \in \mathcal{S}$ such that $\forall W \in V$

$$B(W, U) = F(W)$$

$$B(W, U) = \int_0^L \underbrace{(-W_{,x} a U)}_{\text{conservative form}} + \underbrace{(W_{,x} K U_{,x})}_{\text{diffusion}} dx$$

$$F(W) = \int_0^L W F dx$$

* convective form is the same as the advective form

Convective or advective form:
 $\underline{\underline{w a u_{,x}}}$

Solve the weak form for ADE using the 'convective' form for the convection term \rightarrow will show equivalence w/ CD₂ the same as the 'conservative' form did

Galerkin Form $v^h \in V$; $U^h = v^h + g^h$

$$B(W^h, V^h) = F(W^h) - B(W^h, g^h)$$

$$B(W, U) = \int_0^L \underbrace{(w a U_{,x})}_{\text{convective form}} + \underbrace{(w_{,x} K U_{,x})}_{\text{diffusion form}} dx$$

(W) with convection term

Introduce FE Basis Function \rightarrow plug into Galerkin to get matrix form

$$\underline{w^h = \sum_A c_A N_A}; \quad \underline{v^h = \sum_A d_A N_A}$$

test functions linear combination of shape functions

$$(6) \quad B(W^h, V^h) = F(W^h) - B(W^h, g^h)$$

$$\sum_{A,B} c_A B(N_A, N_B) dB = \sum_A c_A [F(N_A) - B(N_A, g)]$$

$$\sum_A c_A \int K_{AB} dB = \sum_A c_A \tilde{F}(N_A) \rightarrow \sum_A c_A \text{ cancel out}$$

$\int K_{AB} dB = \tilde{F}(N_A)$ → each 'A' node
 Stiffness matrix Load vector

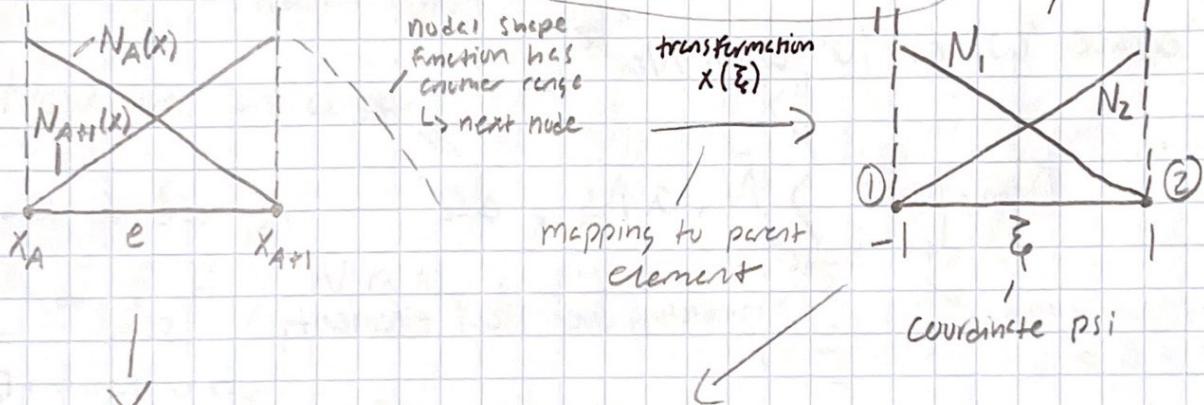
$$\boxed{\int K_{AB} dB = \sum_{e=1}^{N_e} [K_{ab}^e]} \quad \text{assembly operator over all the elements}$$

Assembly of local element wise stiffness matrix K^e

Known shape functions of the local element

How to construct K^e

Assume C^0 continuity → linear shape function



Shape functions

$$\begin{aligned} N_A(x) & \quad N_1(\xi) = \frac{1-\xi}{2} \quad N_1(-1) = 1 \\ N_{A+1}(x) & \quad N_2(\xi) = \frac{1+\xi}{2} \quad N_2(1) = 1 \end{aligned}$$

$$N_1(-1) = 0 \quad N_2(-1) = 0$$

Isoparametric Elements (or discretization)

apply to local element $U = \sum_A U_A N_A(x)$ → can compute the transformation

Position vector $X = \sum_A X_A N_A(x)$ → nodal position vector

$$X_\xi = \sum_A X_A [N_1(\xi), N_2(\xi)]$$

our integrals:

$$\int dx$$

$$dx = X_{,\xi} d\xi$$

Locally

$$X = \sum_A X_A N_A(x)$$

$$X = X_1 N_1 + X_2 N_2 \quad \text{where}$$

$$N_1 = \frac{1 - \xi}{2}$$

$$N_2 = \frac{1 + \xi}{2}$$

$$X = X_1 \left(\frac{1 - \xi}{2} \right) + X_2 \left(\frac{1 + \xi}{2} \right)$$

$$[K_{ab}^e] = K_{\text{convection}}^e + K_{\text{diffusion}}^e$$

$$K_{\text{conv}} = \int_0^L \omega_a U_x dx$$

$$K_{\text{diff}} = \int_0^L \omega_x K U_{,x} dx$$

* Replace 'ω' or 'U' with N_A → Shape Functions

$$\therefore K_{\text{conv}}^e|_{ab} = \int_{\Omega^e} N_a a N_{b,x} d\Omega^e, \quad \Omega \text{ is local } x$$

Stiffness matrix
for $a \neq b$ → integrating over local element

→ two summations → one is over A and the other B

$$K_{\text{conv}}^e|_{ab} = \int_{\Omega^e} N_a a N_{b,x} dx = \int_{-1}^1 N_a a \underbrace{(N_{b,\xi} \xi / x)}_{\text{chain rule}} \times \frac{d\xi}{dx} d\xi$$

Write everything into
change of coordinate so
that we can integrate them → in the form
of ξ (Psi)

→ U runs on B
get concnes bc of
convection

* in diffusion there are two derivatives

* in convection there is only one derivative

$$[K_{\text{conv}}^e]_{ab} = \int_{-1}^1 N_a a N_{b,\xi} d\xi - \begin{array}{l} \text{local convection} \\ \text{stiffness matrix} \\ \text{for each node } A \text{ & } B \end{array}$$

Ex: When $a=1 \neq b=1$

$$\begin{aligned} [K_{\text{conv}}^e]_{11} &= \int_{-1}^1 N_1 a N_{1,\xi} d\xi \quad \left| \begin{array}{l} N_1 = \frac{1-\xi}{2} \\ \frac{d}{d\xi} \left(\frac{1-\xi}{2} \right) = \frac{1}{2} \frac{d}{d\xi} (1-\xi) \end{array} \right. \\ &= \int_{-1}^1 \left(\frac{1-\xi}{2} \right) a \left(-\frac{1}{2} \right) d\xi \quad \left| \begin{array}{l} N_{1,\xi} = -\frac{1}{2} \\ = \frac{1}{2} \left(\frac{1}{d\xi} 1 - \frac{1}{d\xi} \xi \right) \\ = \frac{1}{2} (0 - 1) = -\frac{1}{2} \end{array} \right. \\ &= -\frac{1}{4} \int_{-1}^1 (1-\xi) a d\xi = -\frac{1}{4} \int_{-1}^1 (a - a\xi) d\xi \quad [K_{\text{conv}}^e]_{11} \\ &= -\frac{1}{4} \left(\int_{-1}^1 a d\xi - \int_{-1}^1 a\xi d\xi \right) = -\frac{1}{4} (2a - 0) = -\frac{a}{2} \end{aligned}$$

Now we can compute $[K_{\text{conv}}^e]_{12}$; $[K_{\text{conv}}^e]_{21}$; $[K_{\text{conv}}^e]_{22}$

$$\begin{aligned} [K_{\text{conv}}^e]_{12} &= \int_{-1}^1 N_1 a N_{2,\xi} d\xi \quad \left| \begin{array}{l} N_1 = \frac{1-\xi}{2} \rightarrow N_{1,\xi} = -\frac{1}{2} \\ N_2 = \frac{1+\xi}{2} \rightarrow N_{2,\xi} = \frac{1}{2} \end{array} \right. \\ &= \int_{-1}^1 \left(\frac{1-\xi}{2} \right) a \left(\frac{1}{2} \right) d\xi = \frac{a}{2} \end{aligned}$$

$$\begin{aligned} [K_{\text{conv}}^e]_{21} &= \int_{-1}^1 N_2 a N_{1,\xi} d\xi = \int_{-1}^1 \left(\frac{1+\xi}{2} \right) a \left(-\frac{1}{2} \right) d\xi \\ &= \left(-\frac{1}{4} \right) \int_{-1}^1 (1+\xi) a d\xi = \left(-\frac{1}{4} \right) \left(\int_{-1}^1 a d\xi + \int_{-1}^1 a\xi d\xi \right) \\ &= -\frac{a}{2} \end{aligned}$$

$$[K_{\text{conv}}^e]_{22} = \int_{-1}^1 N_2 a N_{2,\xi} d\xi = \int_{-1}^1 \left(\frac{1+\xi}{2}\right) a \left(\frac{1}{2}\right) d\xi$$

$$= \left(\frac{1}{4}\right) \int_{-1}^1 (1+\xi) a d\xi = \left(\frac{1}{4}\right) \int_{-1}^1 a d\xi + \int_{-1}^1 \cancel{a \xi} d\xi = \frac{a}{2}$$

$$[K_{\text{conv}}^e] = \begin{bmatrix} -\frac{a}{2} & \frac{a}{2} \\ -\frac{a}{2} & \frac{a}{2} \end{bmatrix}$$

$$dx = x, \xi d\xi$$

$$[K_{\text{diff}}^e]_{ab} = \int_{-1}^1 N_{a,x} K N_{b,x} dx \quad | \quad K_{\text{diff}} = \int_0^L w_{,x} K v_{,x} dx$$

$$= \int_{-1}^1 \underbrace{(N_{a,\xi} \xi_{,x})}_{\text{chain rule}} \underbrace{K(N_{b,\xi} \xi_{,x})}_{\text{chain rule}} \underbrace{\cancel{x, \xi} d\xi}_{dx} \quad | \quad \begin{aligned} x, \xi &= \sum N_{a,\xi} x_a \\ \xi_{,x} &= (x, \xi)^{-1} \end{aligned}$$

$$= \int_{-1}^1 N_{a,\xi} K N_{b,\xi} \xi_{,x} d\xi$$

$$x = \sum_A x_A N_A(x) = x_1 N_1 + x_2 N_2 \rightarrow x, \xi = x_1 N_{1,\xi} + x_2 N_{2,\xi} = x_1 \left(-\frac{1}{2}\right) + x_2 \left(\frac{1}{2}\right)$$

$$x, \xi = \frac{h^e}{2}, \quad \xi_{,x} = \frac{2}{h^e}$$

Solve when $a=1 \neq b=1$

$$[K_{\text{diff}}^e]_{11} = \int_{-1}^1 \underbrace{N_{1,\xi} K N_{1,\xi}}_{\left(-\frac{1}{2}\right)\left(-\frac{1}{2}\right)=\frac{1}{4}} \frac{2}{h^e} d\xi = \frac{1}{4} \int_{-1}^1 K \frac{2}{h^e} d\xi$$

$$\int_{-1}^1 d\xi = \xi \Big|_{-1}^1 = 1 - (-1) = 2$$

$$= \left(\frac{1}{4}\right) K \frac{2}{h^e} (2) = \frac{K}{h^e}$$

$$\begin{bmatrix} K_{\text{diff}}^e \end{bmatrix} = \begin{bmatrix} \frac{K}{h^e} & -\frac{K}{h^e} \\ -\frac{K}{h^e} & \frac{K}{h^e} \end{bmatrix}$$

$$\begin{bmatrix} K_{\text{conv}}^e \end{bmatrix} = \begin{bmatrix} -\frac{a}{2} & \frac{a}{2} \\ -\frac{a}{2} & \frac{a}{2} \end{bmatrix}$$

$$\begin{bmatrix} K_{\text{diff}}^e \end{bmatrix}_A^{A+1} = \begin{bmatrix} \frac{K}{h^e} & -\frac{K}{h^e} \\ -\frac{K}{h^e} & \frac{K}{h^e} \end{bmatrix}$$

$$\begin{bmatrix} K_{\text{conv}}^e \end{bmatrix}_A^{A+1} = \begin{bmatrix} -\frac{a}{2} & \frac{a}{2} \\ -\frac{a}{2} & \frac{a}{2} \end{bmatrix}$$

$$\begin{bmatrix} K_{\text{ab}}^e \end{bmatrix} = K_{\text{convection}}^e + K_{\text{diffusion}}^e$$

$$\begin{aligned} A & \quad A+1 \\ A & \begin{bmatrix} \frac{K}{h^e} - \frac{a}{2} & \frac{a}{2} - \frac{K}{h^e} \\ -\frac{K}{h^e} - \frac{a}{2} & \frac{a}{2} + \frac{K}{h^e} \end{bmatrix} \\ A+1 & \end{aligned}$$

$\xrightarrow{A \cdot A+1}$
 $\xrightarrow{(A+1)(A+1)}$

A-1 e-1 A e A+1

$$\begin{bmatrix} K^{e-1} \end{bmatrix}$$

$$\begin{bmatrix} K^e \end{bmatrix}$$

need to assemble both contributions to get the total contribution at Note A

$$\begin{array}{c} \begin{array}{ccc} A-1 & A & A+1 \end{array} \\ \begin{array}{c} A-1 \\ A \\ A+1 \end{array} \end{array} \quad \begin{array}{c} \begin{array}{ccc} A-1 & A & A+1 \end{array} \\ \begin{array}{c} O \\ O \\ O \end{array} \end{array}$$

$\xrightarrow{\quad}$

$$\begin{array}{c} \begin{array}{ccc} A-1 & A & A+1 \end{array} \\ \begin{array}{c} O \\ O \\ O \end{array} \end{array} + \begin{array}{c} \begin{array}{ccc} A-1 & A & A+1 \end{array} \\ \begin{array}{c} O \\ O \\ O \end{array} \end{array}$$

$\xrightarrow{\quad}$

$$\begin{array}{c} \begin{array}{ccc} A-1 & A & A+1 \end{array} \\ \begin{array}{c} \frac{K}{h^{e-1}} - \frac{a}{2} & \frac{a}{2} - \frac{K}{h^{e-1}} & 0 \\ -\frac{K}{h^{e-1}} - \frac{a}{2} & \frac{a}{2} + \frac{K}{h^{e-1}} & 0 \\ 0 & 0 & 0 \end{array} \end{array}$$

$$\begin{array}{c} \begin{array}{ccc} A-1 & A & A+1 \end{array} \\ \begin{array}{c} O \\ O \\ O \end{array} \end{array} + \begin{array}{c} \begin{array}{ccc} A-1 & A & A+1 \end{array} \\ \begin{array}{c} O \\ O \\ O \end{array} \end{array}$$

$\xrightarrow{\quad}$

$$\begin{array}{c} \begin{array}{ccc} A-1 & A & A+1 \end{array} \\ \begin{array}{c} O \\ O \\ O \end{array} \end{array}$$

$\xrightarrow{\quad}$

$$\begin{array}{c} \begin{array}{ccc} A-1 & A & A+1 \end{array} \\ \begin{array}{c} \frac{K}{h^e} - \frac{a}{2} & \frac{a}{2} - \frac{K}{h^e} & 0 \\ -\frac{K}{h^e} - \frac{a}{2} & \frac{a}{2} + \frac{K}{h^e} & 0 \\ 0 & -\frac{K}{h^e} - \frac{a}{2} & \frac{a}{2} + \frac{K}{h^e} \end{array} \end{array}$$

$\xrightarrow{\quad}$

$$\begin{array}{c} \begin{array}{ccc} A-1 & A & A+1 \end{array} \\ \begin{array}{c} \frac{K}{h^{e-1}} - \frac{a}{2} & \frac{a}{2} - \frac{K}{h^{e-1}} & 0 \\ -\frac{K}{h^{e-1}} - \frac{a}{2} & \frac{a}{2} + \frac{K}{h^{e-1}} & 0 \\ 0 & 0 & 0 \end{array} \end{array}$$

$\xrightarrow{\quad}$

$$\begin{array}{c} \begin{array}{ccc} A-1 & A & A+1 \end{array} \\ \begin{array}{c} K^{e-1} \\ K^e \end{array} \end{array}$$

how you construct the whole stiffness matrix

Now that we have slightly expanded the matrix, we can add them
 $[3 \times 3]$

Don't normally do this all the time \rightarrow normally we write a nice assembly subroutine trying to identify all the node numbers and make sure that residue, stiffness matrix, gets added to its correct location

Ath Node:

$$\left[-\frac{K}{h^{e-1}} - \frac{a}{2} \right] \quad \left[\frac{K}{h^e} + \frac{K}{h^{e-1}} \right] \quad \left[\frac{a}{2} - \frac{K}{h^e} \right] \quad \text{Want to write this as an equation}$$

$$\left[-\frac{K}{h^{e-1}} - \frac{a}{2} \right] u_{A-1} + \left[\frac{K}{h^e} + \frac{K}{h^{e-1}} \right] u_A + \left[\frac{a}{2} - \frac{K}{h^e} \right] u_{A+1} = \tilde{F}_A \cdot h \quad \begin{matrix} \downarrow \\ \text{load vector} \end{matrix}$$

multiply K by the unknown vector

this term is the interaction b/w A-1 & A

* Assume Uniform mesh, $h^{e-1} = h^e = h$

$$\left[-\frac{K}{h} - \frac{a}{2} \right] u_{A-1} + \left[\frac{K}{h} + \frac{K}{h} \right] u_A + \left[\frac{a}{2} - \frac{K}{h} \right] u_{A+1} = \tilde{F}_A \cdot h$$

* Rearrange and divide by h, assume F=0

$$-\frac{K}{h^2} u_{A-1} - \underbrace{\frac{a}{2h} u_{A-1}}_{\alpha \frac{(u_{A+1} - u_{A-1})}{2h}} + \underbrace{\frac{2K}{h^2} u_A}_{\alpha} + \underbrace{\frac{a}{2h} u_{A+1}}_{\alpha} - \frac{K}{h^2} u_{A+1} = 0$$

$$-\frac{K}{h^2} u_{A-1} + \frac{2K}{h^2} u_A - \frac{K}{h^2} u_{A+1} = -\frac{K(u_{A+1} - 2u_A + u_{A-1})}{h^2}$$

$$\alpha \frac{(U_{A+1} - U_{A-1})}{2h} - K \frac{(U_{A+1} - 2U_A + U_{A-1})}{h^2} = 0$$

C \hookrightarrow central differencing (CD) scheme

Galerkin FEM is exactly same as CD
for uniform mesh