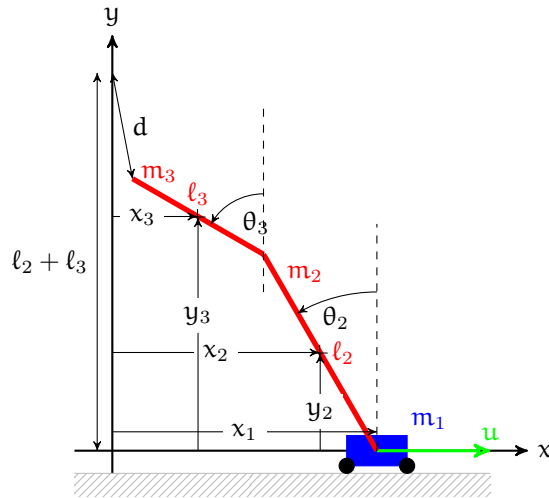


Cart and Double Pendulum Dynamics

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The cart and double pendulum dynamical system consists of a cart with mass m_1 and an attached double pendulum. The double pendulum consists of two pieces, the first of mass m_2 and length ℓ_2 and the second has mass m_3 and length ℓ_3 . Both joints are frictionless and unactuated, the double pendulum can move freely in the vertical plane. The goal position is balancing the double pendulum vertically above $x = 0$, and the discrepancy is the distance d . The system is actuated by applying a force u horizontally to the cart. The state of the system is given by the horizontal position of the cart x_1 , the two angles θ_2 and θ_3 (both measured anti-clockwise from vertically up) and the time derivatives of these three variables, \dot{x}_1 , $\dot{\theta}_2$ and $\dot{\theta}_3$.



The cart can move horizontally, with an applied external force $-20\text{N} \leq u \leq 20\text{N}$, and coefficient of friction $b = 0.1\text{Ns/m}$. The other physical values are $m_1 = m_2 = m_3 = 0.5\text{kg}$ and $\ell_2 = \ell_3 = 0.6\text{m}$ (doesn't correspond to figure). Thus, the moment of inertia around the midpoint of each part of the pendulum is $I_2 = I_3 = 0.015\text{kg m}^2$.

Equations of Motion

The geometric relationships gives the following conditions for the coordinates of the midpoint of the two penduli:

$$x_2 = x_1 - \frac{1}{2}\ell_2 \sin \theta_2, \quad y_2 = \frac{1}{2}\ell_2 \cos \theta_2, \quad (1)$$

$$x_3 = x_1 - \ell_2 \sin \theta_2 - \frac{1}{2}\ell_3 \sin \theta_3, \quad y_3 = \ell_2 \cos \theta_2 + \frac{1}{2}\ell_3 \cos \theta_3. \quad (2)$$

The squared velocities of the centres of gravity are

$$v_1^2 = \dot{x}_1^2, \quad v_2^2 = \dot{x}_1^2 - \ell_2 \dot{x}_1 \dot{\theta}_2 \cos \theta_2 + \frac{1}{4}\ell_2^2 \dot{\theta}_2^2, \quad (3)$$

$$v_3^2 = \dot{x}_1^2 - 2\ell_2 \dot{x}_1 \dot{\theta}_2 \cos \theta_2 - \ell_3 \dot{x}_1 \dot{\theta}_3 \cos \theta_3 + \ell_2^2 \dot{\theta}_2^2 + \frac{1}{4}\ell_3^2 \dot{\theta}_3^2 + \ell_2 \ell_3 \dot{\theta}_2 \dot{\theta}_3 \cos \theta_2 \cos \theta_3. \quad (4)$$

The system Lagrangian is

$$\begin{aligned} L = T - V &= \frac{1}{2}m_1 v_1^2 + \frac{1}{2}m_2 v_2^2 + \frac{1}{2}m_3 v_3^2 + \frac{1}{2}I_2 \dot{\theta}_2^2 + \frac{1}{2}I_3 \dot{\theta}_3^2 - m_2 g y_2 - m_3 g y_3 \Rightarrow \\ L &= \frac{1}{2}(m_1 + m_2 + m_3)\dot{x}_1^2 - (\frac{1}{2}m_2 + m_3)\ell_2 \dot{x}_1 \dot{\theta}_2 \cos \theta_2 + (\frac{1}{6}m_2 + \frac{1}{2}m_3)\ell_2^2 \dot{\theta}_2^2 - \frac{1}{2}m_3 \ell_3 \dot{x}_1 \dot{\theta}_3 \cos \theta_3 + \\ &\quad \frac{1}{6}m_3 \ell_3^2 \dot{\theta}_3^2 + \frac{1}{2}m_3 \ell_2 \ell_3 \dot{\theta}_2 \dot{\theta}_3 \cos(\theta_2 - \theta_3) - (\frac{1}{2}m_2 + m_3)g\ell_2 \cos \theta_2 - \frac{1}{2}m_3 g\ell_3 \cos \theta_3, \end{aligned}$$

where $g = 9.82 \text{m/s}^2$ is the acceleration of gravity.

The equations of motion are

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{q}_i} - \frac{\partial L}{\partial q_i} = Q_i,$$

where Q_i are the non-conservative forces. In our case

$$\frac{\partial L}{\partial \dot{x}_1} = (m_1 + m_2 + m_3)\dot{x}_1 - (\frac{1}{2}m_2 + m_3)\ell_2 \dot{\theta}_2 \cos \theta_2 - \frac{1}{2}m_3 \ell_3 \dot{\theta}_3 \cos \theta_3, \quad (5)$$

$$\frac{\partial L}{\partial x} = 0, \quad (6)$$

$$\frac{\partial L}{\partial \dot{\theta}_2} = (\frac{1}{3}m_2 + m_3)\ell_2^2 \dot{\theta}_2 - (\frac{1}{2}m_2 + m_3)\ell_2 \dot{x}_1 \cos \theta_2 + \frac{1}{2}m_3 \ell_2 \ell_3 \dot{\theta}_3 \cos(\theta_2 - \theta_3), \quad (7)$$

$$\frac{\partial L}{\partial \theta_2} = (\frac{1}{2}m_2 + m_3)\ell_2(g + \dot{x}_1 \dot{\theta}_2) \sin \theta_2 - \frac{1}{2}m_3 \ell_2 \ell_3 \dot{\theta}_2 \dot{\theta}_3 \sin(\theta_2 - \theta_3), \quad (8)$$

$$\frac{\partial L}{\partial \dot{\theta}_3} = \frac{1}{3}m_3 \ell_3^2 \dot{\theta}_3 - \frac{1}{2}m_3 \ell_3 \dot{x}_1 \cos \theta_3 + \frac{1}{2}m_3 \ell_2 \ell_3 \dot{\theta}_2 \cos(\theta_2 - \theta_3), \quad (9)$$

$$\frac{\partial L}{\partial \theta_3} = \frac{1}{2}m_3 \ell_3(g + \dot{x}_1 \dot{\theta}_3) \sin \theta_3 + \frac{1}{2}m_3 \ell_2 \ell_3 \dot{\theta}_2 \dot{\theta}_3 \sin(\theta_2 - \theta_3), \quad (10)$$

leading to the equations of motion

$$\begin{aligned} (m_1 + m_2 + m_3)\ddot{x}_1 - (\frac{1}{2}m_2 + m_3)\ell_2(\ddot{\theta}_2 \cos \theta_2 - \dot{\theta}_2^2 \sin \theta_2) - \frac{1}{2}m_3 \ell_3(\ddot{\theta}_3 \cos \theta_3 - \dot{\theta}_3^2 \sin \theta_3) &= u - b\dot{x}_1, \\ (\frac{1}{3}m_2 + m_3)\ell_2 \ddot{\theta}_2 - (\frac{1}{2}m_2 + m_3)(\ddot{x}_1 \cos \theta_2 + g \sin \theta_2) + \frac{1}{2}m_3 \ell_3[\ddot{\theta}_3 \cos(\theta_2 - \theta_3) + \dot{\theta}_3^2 \sin(\theta_2 - \theta_3)] &= 0, \\ \frac{1}{3}\ell_3 \ddot{\theta}_3 - \frac{1}{2}(\ddot{x}_1 \cos \theta_3 + g \sin \theta_3) + \frac{1}{2}\ell_2[\ddot{\theta}_2 \cos(\theta_2 - \theta_3) - \dot{\theta}_2^2 \sin(\theta_2 - \theta_3)] &= 0. \end{aligned}$$

Collecting the six state variables $z = (\dot{x}_1, \dot{\theta}_2, \dot{\theta}_3, x_1, \theta_2, \theta_3)$ the equations of motion can be conveniently expressed as six coupled differential equations

$$\frac{dz}{dt} = \begin{matrix} A^{-1} & B \\ I & 0 \end{matrix}$$

Linearized Dynamics

Linearizing the dynamics around the goal state, we can write the following approximation

$$\frac{dz}{dt} \simeq \mathbf{q}^{-1}(\mathbf{A}z + \mathbf{b}u), \quad \mathbf{q} = 2\ell_2\ell_3(\mathbf{m}_1(4\mathbf{m}_2 + 3\mathbf{m}_3) + \mathbf{m}_2(\mathbf{m}_2 + \mathbf{m}_3)), \quad \mathbf{b} = \begin{bmatrix} 2\ell_2\ell_3(4\mathbf{m}_2 + 3\mathbf{m}_3) \\ 6\ell_3(2\mathbf{m}_2 + \mathbf{m}_3) \\ -6\ell_2\mathbf{m}_2 \\ 0 \\ 0 \\ 0 \end{bmatrix},$$

and

$$\mathbf{A} = \begin{bmatrix} -2\ell_2\ell_3(4\mathbf{m}_2 + 3\mathbf{m}_3)\mathbf{b} & 0 & 0 & 0 & 3\ell_2\ell_3(\mathbf{m}_2 + 2\mathbf{m}_3)(2\mathbf{m}_2 + \mathbf{m}_3)g & -3\ell_2\ell_3\mathbf{m}_2\mathbf{m}_3g \\ -6\ell_3(2\mathbf{m}_2 + \mathbf{m}_3)\mathbf{b} & 0 & 0 & 0 & 3\ell_3(\mathbf{m}_2 + 2\mathbf{m}_3)(4\mathbf{m}_1 + 4\mathbf{m}_2 + \mathbf{m}_3)g & -9\ell_3\mathbf{m}_3(2\mathbf{m}_1 + \mathbf{m}_2)g \\ 6\ell_2\mathbf{m}_2\mathbf{b} & 0 & 0 & 0 & -9\ell_2(\mathbf{m}_2 + 2\mathbf{m}_3)(2\mathbf{m}_1 + \mathbf{m}_2)g & 3\ell_2(4\mathbf{m}_1(\mathbf{m}_2 + 3\mathbf{m}_3) + \mathbf{m}_2(\mathbf{m}_2 + 4\mathbf{m}_3))g \\ \mathbf{q} & 0 & 0 & 0 & 0 & 0 \\ 0 & \mathbf{q} & 0 & 0 & 0 & 0 \\ 0 & 0 & \mathbf{q} & 0 & 0 & 0 \end{bmatrix}.$$

Loss Function

The instantaneous loss is given by

$$F = 1 - \exp\left(-\frac{d^2}{2a^2}\right),$$

where d^2 is the squared distance between the tip of the pendulum and the point at distance $\ell_2 + \ell_3$ above the origin, and a is the width parameter. Note that the instantaneous loss does not depend on the speed variables, \dot{x}_1 , $\dot{\theta}_2$ and $\dot{\theta}_3$. The squared distance is

$$d^2 = (x_1 - \ell_2 \sin \theta_2 - \ell_3 \sin \theta_3)^2 + (\ell - \ell_2 \cos \theta_2 + \ell_3 \cos \theta_3)^2 = (\tilde{z} - \mu)^\top Q (\tilde{z} - \mu),$$

where \tilde{z} is z augmented by four coordinates $\sin \theta_2$, $\sin \theta_3$ and $\cos \theta_2$, $\cos \theta_3$ and

$$Q = C^\top C, \quad C = \begin{bmatrix} 0 & 0 & 0 & 1 & 0 & 0 & -\ell_1 & 0 & -\ell_2 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & \ell_1 & 0 & \ell_2 \end{bmatrix}, \quad \mu = [0 \quad 0 \quad 0 \quad 0 \quad 0 \quad 0 \quad 0 \quad 0 \quad 1 \quad 0 \quad 1]^\top.$$

The expected loss, averaging over the possibly uncertain states is therefore

$$\mathbb{E}[F(\tilde{z})] = 1 - \int F(\tilde{z})p(\tilde{z})d\tilde{z},$$

which can be evaluated in closed form for Gaussian $p(\tilde{z})$, see reward.pdf. Since the augmented state \tilde{z} will not generally be Gaussian even if z is Gaussian, we project on to the closest Gaussian by matching first and second moments.