

Week 9

2019年12月2日

Task 1

Use a weather forecast website, and utilize the psychrometric chart and the formula we went through in the class to determine the absolute humidity, the wet-bulb temperature and the mass of water vapour in the air in ClassRoom A (Aula A) of Piacenza campus in the moment that you are solving this exercise (provide the inputs that you utilized)

Umidità: Relative humidity, Pressione atmosferica: Air total pressure (1 hPa: 0.1 kPa), Temperatura effettiva: temperature to be utilized.

Ans:

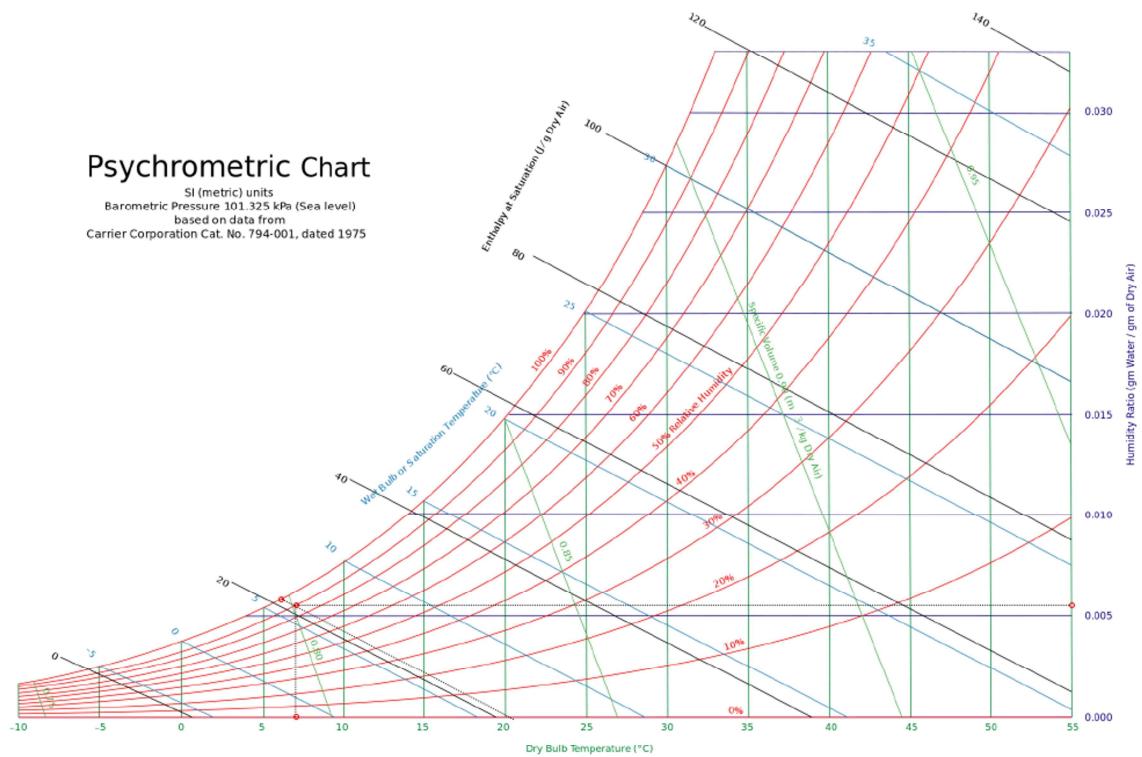
Il tempo oggi in Piacenza							
Lunedì, 02 Dicembre 2019							
	13:00	14:00	16:00	18:00	20:00	21:00	22:00
Temperatura effettiva	10°C	10°C	9°C	6°C	7°C	7°C	8°C
Temperatura percepita	10°C	10°C	8°C	5°C	7°C	6°C	7°C
Precipitazioni	0 mm						
Umidità	79 %	77 %	89 %	90 %	90 %	92 %	91 %
Pressione atmosferica	1016 hPa	1015 hPa	1016 hPa	1017 hPa	1019 hPa	1019 hPa	1020 hPa

The time now is 20:00, from the data given in the website <https://www.meteo-oggi.it/italia/regione-emilia-romagna/tempo-piacenza/>

umidità: 90%, i.e., the relative humidity $\phi = 90\%$;

pressione atmosferica: 1019 hPa, i.e., the total air pressure $P = 101.9 \text{ kPa}$;

temperatura effettiva: 7 °C, i.e., the temperature in Kelvin temperature scale $T = 230 \text{ K}$



Utilize the psychrometric chart, we can see,

the humidity ratio, i.e., the absolute humidity $\omega = 0.0055$

the web-bulb temperature $T_{wb} = 6^\circ C$

$$\begin{aligned}\therefore \omega &= \frac{0.622P_v}{P_a} = \frac{0.622P_v}{P - P_v} = 0.0055, \text{ introduce } P \\ &= 101.9 \text{ kPa into this equation, and solve it,}\end{aligned}$$

$$P_v \approx 0.893 \text{ kPa}$$

$$\text{autem, } \phi = \frac{m_v}{m_g} = 90\% \dots \dots (1)$$

for any ideal gas, $m =$

$$\frac{PV}{R_{sp}T}, \text{ during the class we were told that for water vapour, } R_{sp} = 0.4615$$

introduce the pressure of water vapor $P_v = 0.893 \text{ kPa}$, and define the volume of aula A is V , here we have:

$$m_v = \frac{0.893V}{0.4615 \times 230} \approx 8.41 \times 10^{-3}V$$

subordinate this value to equation (1), calculate the maximum water vapour m_g ,

$$m_g = \frac{m_v}{90\%} \approx 9.34 \times 10^{-3}V$$

Task 2

Utilize the same methodology we went through in the class and determine the sensible and latent load corresponding to internal gains, the ventilation, and the infiltration in a house with a good construction quality and with the same geometry (height of 2.5 m, considering two occupants and one bed room calculate, and a conditioned floor area of 200 m^2 and wall area is 144 m^2 , calculate the internal gains, infiltration, and ventilation loads) as that of the example which is located in Brindisi, Italy.

BRINDISI, Italy														WMO#: 163200											
Lat: 40.65N Long: 17.95E Elev: 10 StdP: 101.2				Time Zone: 1.00 (EUW)				Period: 86-10		WBAN: 99999															
Annual Heating and Humidification Design Conditions																									
Coldest Month	Heating DB			Humidification DP/MCDB and HR						Coldest month WS/MCDB															
	99.6%	99%		99.6%	DP	HR	MCDB	99%		0.4%	1%			MCWS/PCWD to 99.6% DB											
(1)	(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)										
(1)	2	2.9	4.1	-5.1	2.5	7.2	-3.0	3.0	7.4	13.4	10.2	12.4	10.6	3.4	250										
Annual Cooling, Dehumidification, and Enthalpy Design Conditions																									
Hottest Month	Hottest Month DB Range	Cooling DB/MCWB						Evaporation WB/MCDB						MCWS/PCWD to 0.4% DB											
		0.4%	1%	2%	0.4%	1%	2%	0.4%	1%	2%	0.4%	1%	2%	MCWS	PCWD										
(2)	(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)										
(2)	8	7.1	32.8	23.6	31.1	24.3	29.9	24.3	27.2	29.7	26.3	29.0	25.6	28.3	4.2										
Dehumidification DP/MCDB and HR																									
DP	HR	MCDB	DP	HR	MCDB	DP	HR	MCDB	Enth	MCDB	Enth	MCDB	Enth	MCDB	Hours 8 to 4 & 12.8/20.6										
(3)	(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)										
(3)	26.3	21.8	29.2	25.4	20.7	28.5	24.7	19.7	27.9	86.0	30.1	82.2	29.1	78.5	28.3										
Extreme Annual Design Conditions																									
Extreme Annual WS			Extreme Max WB	Extreme Annual DB						n-Year Return Period Values of Extreme DB															
1%	2.5%	5%		Mean	Standard deviation					n=5 years	n=10 years	n=20 years	n=50 years												
(4)	(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)										
(4)	11.3	9.9	8.7	31.4	0.4	37.3	1.4	3.0	-0.6	39.4	-1.4	41.1	-2.2	42.8	-3.2										
(4)																									

Ans:

Internal gains,

Calculate the sensible cooling load from internal gains,

$$q_{ig,s} = 136 + 2.2A_{cf} + 22N_{oc} = 136 + 2.2 * 200 + 22 * 2 = 620 \text{ W}$$

Calculate the latent cooling load from internal gains,

$$q_{ig,l} = 20 + 0.22A_{cf} + 12N_{oc} = 20 + 0.22 * 200 + 12 * 2 = 88 \text{ W}$$

Infiltration,

for a house with a good construction quality, unit leakage area $A_{ul} = 1.4 \text{ cm}^2/\text{m}^2$

and the exposed surface $A_{es} = A_{wall} + A_{roof} = 200 + 144 = 344 \text{ m}^2$

thus, $A_L = A_{es} * A_{ul} = 344 * 1.4 = 481.6 \text{ cm}^2$

Define the cooling temperature $T_{cooling} = 24^\circ\text{C}$, and heating temperature $T_{heating} = 20^\circ\text{C}$

in Brindisi,

$$\Delta T_{cooling} = 31.1^\circ\text{C} - 24^\circ\text{C} = 7.1^\circ\text{C} = 7.1 \text{ K}$$

$$\Delta T_{heating} = 20^\circ C - (4.1^\circ C) = 24.1^\circ C = 15.9 K$$

$$DR = 7.1^\circ C = 7.1 K$$

$$Given that IDF_{heating} = 0.073 \frac{L}{s * cm^2},$$

$$IDF_{cooling} = 0.033 \frac{L}{s * cm^2},$$

Calculate infiltration airflow rate,

$$Q_{i,heating} = A_L * IDF_{heating} = 481.6 * 0.073 \approx 35.157 \frac{L}{s}$$

$$Q_{i,cooling} = A_L * IDF_{cooling} = 481.6 * 0.033 \approx 15.893 \frac{L}{s}$$

The required minimum whole-building ventilation rate is

$$Q_v = 0.05A_{cf} + 3.5(N_{br} + 1) = 0.05 * 200 + 3.5 * (1 + 1) = 17 \frac{L}{s}$$

thus,

$$Q_{i-v,heating} = Q_{i,heating} + Q_v \approx 35.157 + 17 = 52.157 \frac{L}{s}$$

$$Q_{i-v,cooling} = Q_{i,cooling} + Q_v \approx 15.893 + 17 = 32.893 \frac{L}{s}$$

Given that $C_{sensible} = 1.23$, $C_{latent} = 3010$, $\Delta\omega_{cooling} = 0.0039$

$$\begin{aligned} \dot{q}_{inf-ventilation_{cooling,sensible}} &= C_{sensible} Q_{i-v,cooling} \Delta T_{cooling} \approx 1.23 * 32.893 * 7.1 \\ &\approx 287.25 W \end{aligned}$$

$$\begin{aligned} \dot{q}_{inf-ventilation_{cooling,latent}} &= C_{latent} Q_{i-v,cooling} \Delta\omega_{cooling} \approx 3010 * 32.893 * 0.0039 \\ &\approx 386.13 W \end{aligned}$$

$$\begin{aligned} \dot{q}_{inf-ventilation_{heating,g,sensible}} &= C_{sensible} Q_{i-v,heating} \Delta T_{heating} \approx 1.23 * 52.157 * 15.9 \\ &\approx 1020.034 W \end{aligned}$$