## WEEK 9 (ASSIGNMENT 9)

**Task 1:** Use a weather forecast website, and utilize the psychrometric chart and the formula we went through in the class to determine the absolute humidity, the wet-bulb temperature and the mass of water vapour in the air in ClassRoom A (Aula A) of Piacenza campus in the moment that you are solving this exercise (provide the inputs that you utilized)

Umidità: **Relative humidity**, Pressione atmosferica: **Air total pressure** (1 hPa: 0.1 kPa), Temperatura effettiva: **temperature to be utilized**.

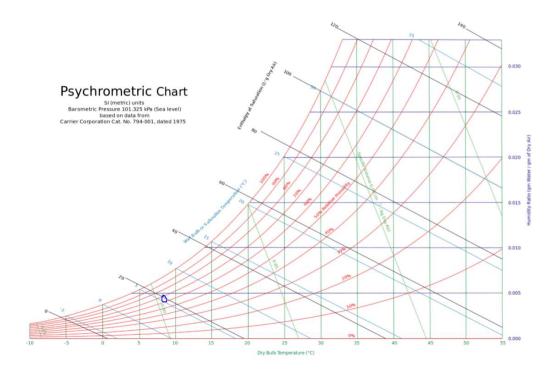
Il tempo oggi in Piacenza Martedì, 03 Dicembre 2019												
	13:00	14:00	16:00	18:00	20:00	21:00	22:00					
	LightCloud	LightCloud	PartlyCloud	Light Cloud	Sun	Sun	Sun					
Temperatura effettiva	9°C	10°C	8°C	6°C	4°C	2°C	2°C					
Temperatura percepita	7°C	10°C	6°C	4°C	2°C	0°C	0°C					
Precipitazioni	0 mm	<b>0</b> mm	0 mm	0 mm	0 mm	0 mm	<b>0</b> mm					
Umidità	67 %	<b>65</b> %	69 %	<b>70</b> %	75 %	83 %	87 %					
Pressione atmosferica	1025 hPa	1025 hPa	1025 hPa	1026 hPa	1027 hPa	1027 hPa	1028 hPa					

The time now is 18:00, from the data given in the website: https://www.meteo-oggi.it/italia/regione-emilia-romagna/tempo-piacenza/

umidità: 70%, i.e., the relative humidity  $\varphi = 70\%$ ;

pressione atmosferica: 1026 hPa, i.e., the total air pressure P =102.6 kPa;

temperatura effttiva: 6 °C; the temperature in Kelvin temperature scale T =279.15 K



Utilize the psychrometric chart, we can see,

the humidity ratio, i.e., the absolute humidity m = 0.0045

The web-bulb temperature Twb =  $8 \, ^{\circ}$ C

$$GO = \frac{0.622Pv}{Pa} = \frac{0.622Pv}{P-Pv}$$
,  $GO = 0.0045$ 

introduce P = 102.6 kPa into this equation and solve it:

$$P_v = 0.737 \text{ kPa}$$

autem, 
$$\varphi = \frac{mv}{mg} = 70\% (1)$$

ideal gas 
$$m = \frac{PV}{Rsp T}$$

from class water vapour -  $R_{sp}$  = 0.4615 , introduce the pressure of water  $0.737\ kPa$  and define the volume of aula A is V, than we have :

$$m_v = \frac{_{0.737V}}{_{0.4615~*279,15}} = \frac{_{0.737V}}{_{128,83}} = 0.00572~V = 5.72~x~10^{\text{-}3}~V$$

put this volume to equation (1) and calculate the maximum water vapour

$$m_g = \frac{mv}{70\%} = 0.00817 \text{ V} = 8.17 \text{ x } 10^{-3} \text{ V}$$

**Task 2:** Utilize the same methodology we went through in the class and determine the sensible and latent load corresponding to internal gains, the ventilation, and the infiltration in a house with a good construction quality and with the same geometry (height of 2.5 m, considering two occupants and one bed room calculate, and a conditioned floor area of 200 N2 and wall area is 144 N2, calculate the internal gains, infiltration, and ventilation loads) as that of the example which is located in Brindisi, Italy.

			BRINDISI, Italy													163200	
	Lat	40.65N	Long:	17.95E	Elev:	10	StdP:	101.2		Time Zone	1.00 (EU	W)	Period:	86-10	WBAN:	99999	
	Annual He	ating and H	lumidificat	ion Design C	onditions												
					Urumi	differtion D	DAACDD and	NO.		Coldest month WS/MCDB MCWS/PCV							
	Coldest Heating DB			Humidification DP/MCDB and HR 99.6% 99%			0.4% 1%					to 99.6% DB					
	Month	99.6%	99%	DP	HR	MCDB	DP	HR	MCDB	WS	MCDB	ws	MCDB	MCWS	PCWD		
	(0)	(b)	(c)	(d)	(0)	(f)	(g)	(h)	(i)	(j)	(k)	(1)	(m)	(n)	(0)		
(1)	2	2.9	4.1	-5.1	2.5	7.2	-3.0	3.0	7.4	13.4	10.2	12.4	10.6	3.4	250		(1)
	Annual Co	olina. Dehu	midification	on, and Entha	alov Design	Condition	s										
		<b>J</b> ,			, ,		•										
	Hottest	Hottest			Cooling DB/MCWB						Evaporation WB/MCDB					PCWD	
	Month	Month	DB DB	0.4% 1% B MCWB DB MCWB						0.4% 1% MCDB WB MCDB		2%		to 0.4% DB			
	(0)	DB Range			DB	MCWB	DB	MCWB	WB	MCDB	(k)		WB	MCDB	MCWS	PCWD	
401	(a) 8	7.1	(c) 32.8	(d) 23.6	(e) 31.1	(f) 24.3	(g) <b>29.9</b>	(h) 24.3	(i) 27.2	(j) <b>29.7</b>	26.3	29.0	(m) 25.6	(n) 28.3	(o) 4.2	(p) 180	(0)
(2)		7.1	32.0					24.3	21.2	29.1	20.3			20.3	4.2		(2)
				Dehumidific	ation DP/M0	DB and HF	2			Enthalpy/MCDB 0.4% 1%						Hours	
	DP	0.4% HR	MCDB	DP	1% HR	MCDB	DP	2% HR	MCDB		0.4% 11 Enth MCDB Enth		% MCDB	Enth 2	% MCDB	8 to 4 & 12.8/20.6	
	(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(I)	(m)	(n)	(o)	(p)	
(3)	26.3	21.8	29.2	25.4	20.7	28.5	24.7	19.7	27.9	86.0	30.1	82.2	29.1	78.5	28.3	1236	(3)
(3)					20.7	20.5	24.7	15.7	21.5	00.0	30.1	02.2	25.1	70.5	20.5	1230	(9)
	Extreme A	innual Desig	gn Conditie	ons													
				Extreme		Eutromo	Assuel DD				a Vaar Da	him Dorina	Values of E	-tromo DD			
	Extreme Annual WS Max			Extreme Annual DB Mean Standard deviation			n-Year Return Period Values of n=5 years n=10 years n=2					0 years n=50 years					
		WB	Min	Max	Min	Max	Min	Max	Min	Max	Min	Max	Min	Max			
	(0)	(b)	(c)	(d)	(0)	(f)	(g)	(h)	(i)	(j)	(k)	(1)	(m)	(n)	(0)	(p)	
(4)	11.3	9.9	8.7	31.4	0.4	37.3	1.4	3.0	-0.6	39.4	-1.4	41.1	-2.2	42.8	-3.2	44.9	(4)

• Internal gains,

Calculate the sensibile cooling load from internal gains,

$$q_{ig,s} = 136 + 2.2A_{cf} + 22N_{oc} = 136 + 2.2 \times 200 + 22 \times 2 = 620 \text{ W}$$

Calculate the latent cooling load from internal gains,

$$q_{ig,l} = 20 + 0.22 A_{cf} + 12 N_{oc} = 20 + 0.22 \times 200 + 12 \times 2 = 88 \text{ W}$$

• Infiltration,

for a house with a good construction quality, unit leakage area  $A_{ul}\,=1.4\text{cm}^2/\text{m}^2$ 

and the exposed surface 
$$A_{es}\,=A_{wall}\,+A_{roof}\,=200+144=344~m^2$$

$$A_{L} = A_{es} \times A_{ul} = 344 \times 1.4 = 481.6 \text{ cm}^2$$

• Define the cooling temperature T<sub>cooling</sub> =24 °C, and heating temperature T<sub>heating</sub> =20 °C in Brindisi

$$\Delta T_{cooling} = 31.1 \text{ °C } ---24 \text{ °C} = 7.1 \text{ °C} = 7.1 \text{ K}$$

$$\Delta T_{\text{heating}} = 20 \text{ °C} - (-4.1 \text{ °C}) = 24.1 \text{ °C} = 24.1 \text{ K}$$

$$DR = 7.1 \text{ }^{\circ}C = 7.1 \text{ K}$$

• Given that

$$IDF_{heating} = 0.073 \frac{L}{s * cm2}$$

$$IDF_{cooling} = 0.033 \frac{L}{s * cm2}$$

• Calculate infiltration airflow rate,

$$Q_{i,heating} = A_L \times IDF_{heating} = 481.6 \times 0.073 = 35.157 \frac{L}{s}$$
 $Q_{i,cooling} = A_L \times IDF_{cooling} = 481.6 \times 0.033 = 15.893 \frac{L}{s}$ 

• The required miminum whole-building vetilation rate is

$$\begin{aligned} Qv &= 0.05 Acf + 3.5 (N_{br} + 1) = 0.05 \times 200 + 3.5 \times (1 + 1) = 17 \, \frac{L}{s} \\ \\ Q_{i-v,heating} &= Q_{i,heating} + Q_v = 35.157 + 17 = 52.157 \, \frac{L}{s} \\ \\ Q_{i-v,cooling} &= Q_{i,cooling} + Q_v = 15.893 + 17 = 32.893 \, \frac{L}{s} \end{aligned}$$

• Given that  $C_{sensible} = 1.23$  ,  $C_{latent} = 3010$ ,  $\Delta$   $GO_{Cooling} = 0.0039$ 

$$\dot{q}inf_{ventilation_{cooling}} = C_{sensible}Q_{i-v,cooling} \Delta T_{Cooling} = 1.23 \times 32.893 \times 7.1 = 287.25 \text{ W}$$

$$\dot{q}inf_{ventilation_{cooling}} = C_{latent}Q_{i-v,cooling} \Delta G_{Cooling} = 3010 \times 32.893 \times 0.0039 = 287.25$$

 $\dot{q}inf_{ventilation_{heating}} = C_{sensible} Q_{i-v,heating} \Delta T_{heating} = 1.23 \times 52.157 \times 24.1 = 1546.09$ W