# Week 9- MONDRAGON, ALEJANDRA

domingo, 1 de diciembre de 2019 11:15 p. m.

#### TASK 1-

Use a weather forecast website, and utilize the psychrometric chart and the formula we went through in the class to determine the absolute humidity, the wet-bulb temperature and the mass of water vapour in the air in ClassRoom A (Aula A) of Piacenza campus in the moment that you are solving this exercise (provide the inputs that you utilized)

Umidità: Relative humidity, Pressione atmosferica: Air total pressure (1 hPa: 0.1 kPa), Temperatura effettiva: temperature to be utilized.

II tempo oggi in Piacenza Lunedi, 02 Dicembre 2019									
	05:00	07:00	10:00	14:00	18:00	19:00	21:00		
	δ δ LightRain	∆ ∆ LightRain	PartlyCloud	Sun	LightCloud	PartlyCloud	Cloud		
Temperatura effettiva	70	7°C	8°C	11°C	6°C	7°C	7°C		
Temperatura percepita	5°C	5°C	6°C	11°C	5°C	7°C	7°C		
Precipitazioni	1 mm	0 mm	<b>0</b> mm	<b>0</b> mm	<b>0</b> mm	0 mm	<b>0</b> mm		
Umidità	93 %	93 %	88 %	74 %	90 %	90 %	88 %		
Pressione atmosferica	1015 hPa	1015 hPa	1016 hPa	1015 hPa	1016 hPa	1017 hPa	1019 hPa		
Intensità del vento	11 km/h	10 km/h	12 km/h	8 km/h	6 km/h	4 km/h	4 km/h		
Direzione del vento	<b>□</b>	$\hookrightarrow$	<b>\</b>	<b>\</b>	✓	<b>✓</b>	>		
	0	0	NO	NO	SW	SW	SE		
Probabilità di nebbia	0 %	0 %	0 %	0 %	0 %	0 %	0 %		
Punto di rugiada	6°C	6°C	6°C	6°C	5°C	5°C	5°C		
Nuvole	100 %	00 %	59 %	10 %	34 %	70 %	88 %		
Nuvole basse	57 %	<b>45</b> %	27 %	10 %	27 %	16 %	87 %		
Nuvole medie	100 %	98 %	22 %	0 %	25 %	70 %	68 %		
Nuvole alte	95 %	95 %	48 %	0 %	0 %	0 %	0 %		
25 ,									

## **ABSOLUTE HUMIDITY**

$$\phi = \frac{m_v}{m_g}$$

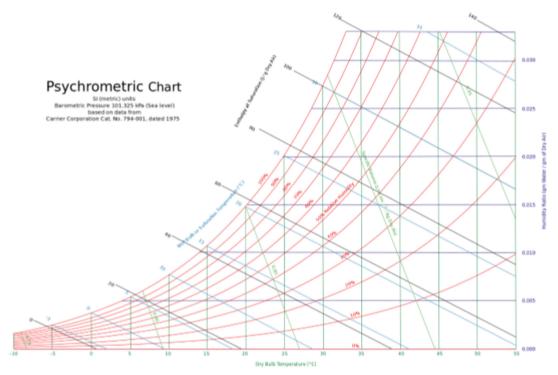
$$7 C = 1.0729 \text{ kPa}$$

$$\phi = \frac{m_v}{m_g} = \frac{P_v}{P_g} \longrightarrow P_g = P_{sat} 7 \text{ °C} = 1.0729 \text{ kPa}$$

$$\phi = \frac{P_v}{P_g} \rightarrow P_V = \phi \times P_g = 0.93*1.0729 = 0.997 \ kPa$$
 partial pressure of dry air:  $P_a = P_v - P_v = 0.1 \ kPa - 0.997 \ kPa = -0.897 \ kPa$ 

$$\omega = 0.622 \frac{P_v}{P_a} = 0.622 \frac{0.997}{-0.897} = 0.439 \frac{Kg_{vapour}}{kg_{dryAir}}$$

$$\begin{split} m &= \frac{PV}{R_{sp.}T} \\ m_a &= \frac{P_a V_a}{R_a T} \qquad R_{sp.} = \frac{R_{global}}{M_{gas}} \end{split}$$



Ra= 0.4615

$$m_v = \frac{0.897 * (5 * 10 * 3)}{0.4615 * (273 + 7)} = 0.487 \ kg$$

$$m_{wv} = \frac{0.487}{0.93} = 0.523 \ kg$$

$$h_a = 1.005 * T = 1.005 * 7 = 7.035 \frac{kJ}{kg_{dryAir}} * T in °C$$

$$h_v = 2501.3 + 1.82 * 7 = 2510.12 \frac{kJ}{kg_{water}}$$

$$h_{\Box} = h_a + \omega h_v = 7.035 + 0.439 * 2510.12 = 1108.97 \frac{kJ}{kg_{dryAir}}$$

**Task 2** Determine the sensible and latent load corresponding to internal gains, the ventilation, and the infiltration in a house with a good construction quality and with the same geometry as that of the example which is located in Brindisi, Italy

Lat: 40.65N Long: 17.95E 101.2 Time Zone: 1.00 (EUW) Period: 86-10 WBAN: 99999 Annual Hea ing and Humidificat Coldest Month Heating DB 99% HR HR MCDB PCWD (b) (c) (d) (0) (f)(g) (h) 2 2.9 4.1 -5.1 2.5 7.2 -3.0 3.0 7.4 13.4 10.2 12.4 10.6 3.4 250 Evaporation WB/MCDE Month Month DB MCWB WB MCDE B Range MCWB (a) (b) (p) 32.8 24.3 24.3 27.2 29.7 26.3 29.0 25.6 4.2 (2) 7.1 180 (2) on DP/M Hours DB and HF Enthalpy/MCDB 8 to 4 & DP MCDB 12.8/20.6 HR HR HR Enth (a) (b) (d) (g) (p) 82.2 28.3 21.8 29.2 20.7 28.5 24.7 19.7 27.9 86.0 30.1 78.5 1236 26.3 25.4 29.1 ual Des years Max WB Max (a) (b) (c) (d) (e) (f)(g) (h) (i)(i) (k) (1)(0) (p)

41.1

-1.4

-2.2

42.8

-3.2

44.9

### Internal gains

11.3

9.9

8.7

31.4

$$\begin{array}{l} \dot{Q}_{ig_{sensible}} = 136 \, + \, 2.2 A_{cf} \, + \, 22 N_{oc} \, = \, 136 \, + \, 2.2 \, x \, 200 \, + \, 22 \, x \, 2 \, = \, 620 \, W \\ \dot{Q}_{ig_{latent}} = 20 \, + \, 0.22 A_{cf} \, + \, 12 N_{oc} \, = \, 20 \, + \, 0.22 \, x \, 200 \, + \, 12 \, x \, 2 \, = \, 88 \, W \\ \end{array}$$

0.4

37.3

1.4

3.0

-0.6

39.4

#### Infiltration

$$\begin{array}{l} A_{ul} = 1.4 \; \frac{cm^2}{m^2} \\ A_{es} = 200 \; + \; 144 \; = \; 344 \; m^2 \\ A_L = A_{es} \; x \; A_{ul} = \; 344 \; x \; 1.4 \; = \; 481.6 \; cm^2 \\ Qi = \Delta L \; IDF \\ IDF_{heating} = 0.065 \; \frac{L}{s. \; cm^2} \\ IDF_{cooling} = 0.032 \; \frac{L}{s. \; cm^2} \end{array}$$

$$T_{cooling} = 31.1 \degree C$$
  
 $T_{heating} = 4.1 \degree C$ 

$$\dot{V}_{i_{heating}} = A_L x IDF = 481.6 x 0.065 = 31.30 \frac{L}{s}$$
  
 $\dot{V}_{i_{cooling}} = A_L x IDF = 481.6 x 0.032 = 15.41 \frac{L}{s}$ 

# Ventilation

$$\dot{Q}_v = 0.05A_{cf} + 3.5(N_{br} + 1) = 0.05 \times 200 + 3.5 \times 2 = 17 \frac{L}{s}$$

$$\dot{Q}_{inf-vent\ heat} = 31.30 + 17 = 48.30 \frac{L}{s}$$

$$\dot{Q}_{inf-vent\ cool} = 15.41 + 17 = 32.41 \frac{L}{s}$$

$$\dot{Q}_{inf-vent\ coool\ senst} = C_{sensible}\dot{V}\Delta T_{cooling} = 1.23\ x\ 32.41\ x\ 7.1 = 283.04\ W$$

$$\dot{Q}_{inf-vent\;cool\;lat} = C_{latent} \dot{V} \Delta \omega_{cooling} = 3010\;x\;32.41\;x\;0.0045 = 438.99\;W$$

$$\dot{Q}_{inf-vent\ heat\ sens} = C_{sensible} \dot{V} \Delta T_{heating} = 1.23\ x\ 48.30\ x\ 15.9 = 944.60\ W$$

$$\dot{Q}_{inf-vent\;heat\;lat}$$
 =  $C_{latent}\dot{V}\Delta\omega_{heating}$  = 3010 x 48.30 x 0.0046 = 668.76 W