

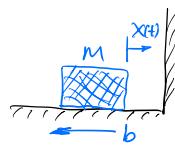
## I Control of Second-Order System

$$\text{Model: } m\ddot{x} + b\dot{x} = F$$

Objective: maintain a fixed position

$$\Rightarrow \dot{x}_d(t) = \text{Constant} \Rightarrow \ddot{x}_d(t) = \dot{\ddot{x}}_d(t) = 0$$

$$\text{Control: } F(t) = k_v(\dot{x}_d - \dot{x}) + k_p(x_d - x) + m\ddot{x}_d$$



Apply the control algorithm to the system (model) to get the error dynamics:

$$m\ddot{x} + b\dot{x} = k_v(\dot{x}_d - \dot{x}) + k_p(x_d - x) + m\ddot{x}_d$$

$$\Rightarrow m(\ddot{x}_d - \ddot{x}) + k_v(\dot{x}_d - \dot{x}) + k_p(x_d - x) = -B\dot{x}$$

$$\Rightarrow m(\ddot{x}_d - \ddot{x}) + k_v(\dot{x}_d - \dot{x}) + B(\dot{x}_d - \dot{x}) + k_p(x_d - x) = 0$$

$$\Rightarrow m(\ddot{x}_d - \ddot{x}) + (k_v + B)(\dot{x}_d - \dot{x}) + k_p(x_d - x) = 0$$

Define  $e(t) = x_d(t) - x(t)$ ,  $\dot{e}(t) = \dot{x}_d(t) - \dot{x}(t)$ ,  $\ddot{e}(t) = \ddot{x}_d(t) - \ddot{x}(t)$

$$\Rightarrow m\ddot{e}(t) + (k_v + B)\dot{e}(t) + k_p e(t) = 0 \quad \text{errordyn.}$$

$$\Rightarrow \ddot{e}(t) + \frac{k_v + B}{m}\dot{e}(t) + \frac{k_p}{m}e(t) = 0 \Rightarrow S_{h2}$$

We can choose  $k_v$  and  $k_p$  so that the pole positions are all located on the left-half of the  $s$ -plane, then  $e(t) \xrightarrow[t \rightarrow \infty]{} 0 \Rightarrow x(t) \xrightarrow[t \rightarrow \infty]{} x_d(t)$   
where  $S_{h2} = -\frac{k}{2} \pm \frac{\sqrt{k_v^2 + 4k_p}}{2} = f(k_v, k_p)$ ,  $\hat{k}_v \triangleq \frac{k_v + B}{m}$ ,  $\hat{k}_p \triangleq \frac{k_p}{m}$

## II. Control Law Partition and Trajectory Following Control

The mass-spring-friction system model:

$$m\ddot{x} + b\dot{x} + kx = f$$

The control law partition method  
partition the control into two parts.

The first part is model based, which is

$$f = \alpha f' + \beta$$

$$\text{with } \alpha = m \text{ and } \beta = b\dot{x} + kx$$

$\Rightarrow$  The error dynamics is

$$m\ddot{x} + b\dot{x} + kx = \alpha f' + \beta = mf' + b\dot{x} + kx$$

$$\Rightarrow \ddot{x} = f'$$

The 2nd-part of the control is servo part,

$$f' = \ddot{x}_d + k_v(\dot{x}_d - \dot{x}) + k_p(x_d - x)$$

$\Rightarrow$  The error dynamics now becomes

$$\ddot{x} = f' = \ddot{x}_d + k_v(\dot{x}_d - \dot{x}) + k_p(x_d - x)$$

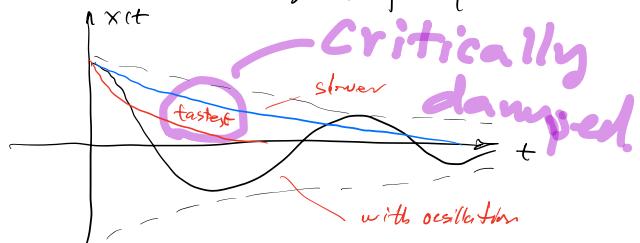
$$\Rightarrow (\ddot{x}_d - \ddot{x}) + k_v(\dot{x}_d - \dot{x}) + k_p(x_d - x) = 0$$

Define  $e(t) = x_d(t) - x(t)$ ,  $\dot{e}(t) = \dot{x}_d(t) - \dot{x}(t)$ ,  $\ddot{e}(t) = \ddot{x}_d(t) - \ddot{x}(t)$ .

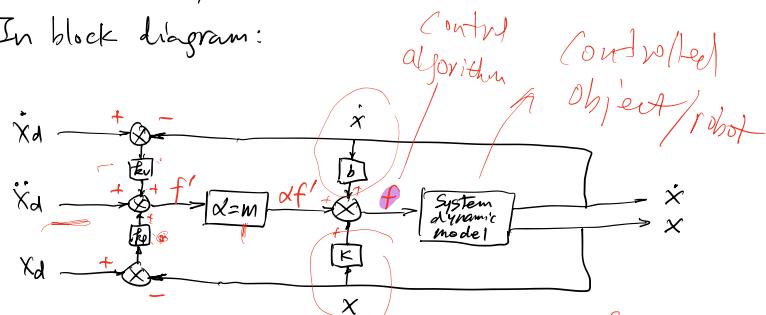
$$\Rightarrow \ddot{e}(t) + k_v\dot{e}(t) + k_p e(t) = 0 \quad \text{standard}$$

let  $k_v = 2\sqrt{k_p}$ , then the error dynamics is going to behave in **critically damped** fashion for the response  $\Rightarrow e(t) \xrightarrow{t \rightarrow \infty} 0$   $\Rightarrow x(t) \xrightarrow{t \rightarrow \infty} x_d(t)$

If  $m=1$ ,  $b=1$ ,  $k_p=16$ ,  $k_v=2\sqrt{k_p}=8$ , then  $e(t) \xrightarrow{t \rightarrow \infty} 0$  in critically damped fashion



In block diagram:



$$\left. \begin{array}{l} \text{Control law} \\ \text{Partition} \end{array} \right\} \begin{aligned} f &= \alpha f' + \beta \quad \text{model based} \\ &= M[\ddot{x}_d + k_v(\dot{x}_d - \dot{x}) + k_p(x_d - x)] + b\dot{x} + kx \end{aligned}$$

Servo - part  $f'$

## E Nonlinear Control of Robot Manipulators

A time-varying nonlinear system example.

Mass-Spring (nonlinear) - damper system:

$$m\ddot{x} + b\dot{x} + g x^3 = f$$

nonlinear

Using the control law partition method:

$$f = \alpha f' + \beta = m\ddot{f}' + b\dot{x} + g x^3$$

Exact Dyn. model

$$\text{with } f' = \ddot{x}_d + k_v(\dot{x}_d - \dot{x}) + k_p(x_d - x)$$

$$e = x_d - x \quad \dot{e} + k_v \dot{e} + k_p e = 0$$

choose  $k_v$  and  $k_p$  to make the system response in **critically damped fashion**  $\Rightarrow e(t) \xrightarrow{t \rightarrow \infty} 0, x(t) \xrightarrow{t \rightarrow \infty} x_d$

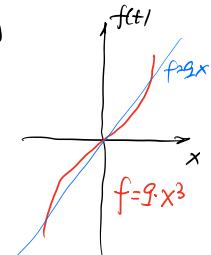
$\Rightarrow$  The control law partition works for both linear and nonlinear system but it has a significant **flaw**: It requires the **exact and precise** dynamic model to work! But in reality that is impossible!

## F MIMO Control Problem for Robot manipulators

**Computed Torque Control Method** (control law partition)

The robot manipulator dynamic model is

$$M(\theta)\ddot{\theta} + V(\theta, \dot{\theta}) + G(\theta) = \tau$$



Using the control law partition method, we have

$$\mathcal{T} = \alpha \mathcal{T}' + \beta \quad \text{where } \mathcal{T} = R^{n \times 1}$$

and  $\alpha = M(\ddot{\theta}) \in R^{n \times n}$

$$\beta = V(\dot{\theta}, \ddot{\theta}) + G(\theta) \in R^{n \times 1}$$

with the servo-part as

$$\mathcal{T}' = \ddot{\theta}_d + K_v(\dot{\theta}_d - \dot{\theta}) + K_p(\theta_d - \theta)$$

Define error as:  $E(t) = \theta_d - \theta$

$$\dot{E}(t) = \dot{\theta}_d - \dot{\theta}$$

$$\ddot{E}(t) = \ddot{\theta}_d - \ddot{\theta}$$

Then the error dynamics is

$$M(\ddot{\theta})\ddot{\theta} + V(\dot{\theta}, \ddot{\theta}) + G(\theta) = M(\ddot{\theta})\mathcal{T}' + V(\dot{\theta}, \ddot{\theta}) + G(\theta)$$

$$\Rightarrow \ddot{\theta} = \mathcal{T}' = \ddot{\theta}_d + K_v(\dot{\theta}_d - \dot{\theta}) + K_p(\theta_d - \theta)$$

$$\Rightarrow \ddot{E} + K_v \dot{E} + K_p E = 0$$

where  $K_v$  and  $K_p$  are matrices. Choose  $K_v$  and  $K_p$  as diagonal matrices, thus for  $j$ -th row:

$$\ddot{e}_j + k_{v,j} \dot{e}_j + k_{p,j} e_j = 0$$

$$\text{Choose } k_{v,j} = 2\sqrt{k_{p,j}} \Rightarrow e_j(t) \xrightarrow[t \rightarrow \infty]{\text{critically damped}} 0, \quad \theta \xrightarrow[t \rightarrow \infty]{\text{critically damped}} \theta_d$$

For  $j=1, 2, \dots, n$ , then  $\ddot{\theta}(t) \xrightarrow[t \rightarrow \infty]{\text{critically damped}} \theta_d(t)$

Then the control law becomes:

$$\mathcal{T} = \alpha \mathcal{T}' + \beta = M \mathcal{T}' + V + G$$

If let  $\mathcal{T}' \triangleq \ddot{\theta}^*$ , then the above control law becomes

$$\mathcal{T} = M(\ddot{\theta}) \cdot \ddot{\theta}^* + V(\dot{\theta}, \ddot{\theta}) + G(\theta)$$

$$\mathcal{T} = M(\ddot{\theta})(\ddot{\theta}^*) + V(\dot{\theta}, \ddot{\theta}) + G(\theta)$$

Control law dynamic model

→ This control algorithm has the same form as the robot dynamic model except that  $\ddot{\theta}$  is replaced by  $\ddot{\theta}^*$ . So this control law is called "Computed Torque Method"

where  $\ddot{\theta}^* = \mathcal{T}' = \ddot{\theta}_d + K_v(\dot{\theta}_d - \dot{\theta}) + K_p(\theta_d - \theta)$

## Servo-Part.

II. Present Industrial Robot Control Systems

a. Individual Joint PID Control

Using control law partition method,

Let  $\alpha = I_n$  identity matrix, no dynamic model  
 $\beta = 0 \Rightarrow$  no model needed.

and  $\tau' = \ddot{\theta}_d + K_v(\dot{\theta}_d - \dot{\theta}) + K_p(\theta_d - \theta)$

**Servo:** Derivative term Proportional term

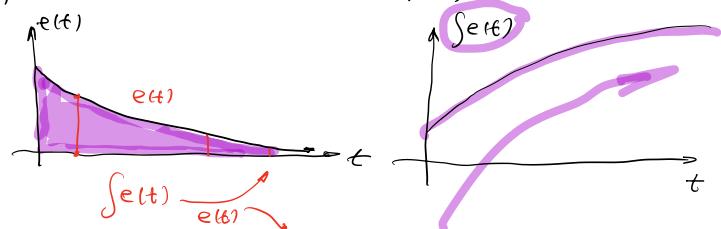
If we add an integral term:

$$\tau' = \ddot{\theta}_d + K_p(\theta_d - \theta) + K_I \cancel{\int (\theta_d - \theta) dt} + K_v(\dot{\theta}_d - \dot{\theta})$$

**P**      **I**      **D**

If no integral term, it is called PD control.

Why do we need the I (integral) term in PID?



$$\tau' = \ddot{\theta}_d + K_p E + K_I \int E dt + K_v \dot{E}$$

as  $E \rightarrow 0$

Small error is eliminated by the integral term?

The integral term works on the error signal, when the error signal becomes very small, the integral term becomes larger and larger so it drives the **Steady-state** error to zero, eventually.

Features of Industrial PID/PD control:

- ① No dynamic model is needed.
- ② Choose  $K_v$  and  $K_p$  and sometimes  $K_I$  as diagonal matrices to decouple the effect, then this becomes individual joint control

$$\tau_j = k_{v,j} \dot{\theta}_j + k_{p,j} \theta_j + k_{i,j} \int \theta_j dt, j=1,2,\dots,n$$

- ③ The performance of the control depends on the values of the gains ( $K_v, K_p, K_I$ ). Normally better performance needs higher gain values  $\Rightarrow$  more powerful motors, which are limited by physical actuator/motor capacities  $\Rightarrow$  limited performance of the robot manipulator, especially in steady-state.