

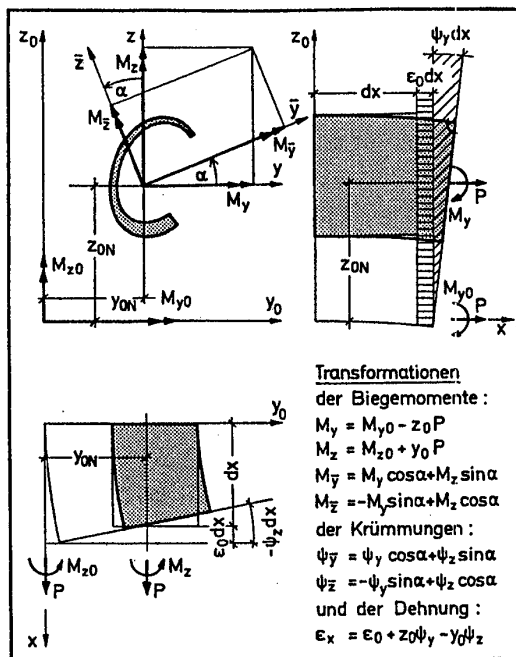
# Normal and Bending Stresses in Simple Cross Sections

## Theory

- closed polygon
- counterclockwise
- must not intersect itself

## Symbols

$y_0, z_0$	arbitrary (reference) axis system
$y_0C, z_0C$	coordinates of section centroid
$y, z$	axes parallel to $y_0, z_0$ , origin in section centroid
$\bar{y}, \bar{z}$	principal axis system, $\bar{y}$ major, origin in centroid
$I_{y0}, I_{z0}, I_{yz0}$	moments of inertia in reference system
$I_y, I_z, I_{yz}$	moments of inertia in centroid axis system
$I_1, I_2$	principal moments of inertia, $I_1 \geq I_2$
$I_{\bar{y}}, I_{\bar{z}}$	principal moments of inertia, $I_{\bar{y}} > I_{\bar{z}}$
$\phi_1$	angle between $y$ and $\bar{y}$ , direction of major principal axis
$\phi_2$	direction of minor principal axis
$A$	area of cross section
$r_1, r_2$	radii of gyration



**Bild 3.1/1.** Elementare Biegetheorie des Stabes. Schnittlasten und Verformungen am Stabelement. Koordinatentransformation

## Properties in Reference System

Area

$$A = \int_{\Omega} dy_0 dz_0$$

Centroid

$$y_{0C} = \frac{1}{A} \int_{\Omega} y_0 dy_0 dz_0$$

$$z_{0C} = \frac{1}{A} \int_{\Omega} z_0 dy_0 dz_0$$

Moments of Inertia

$$I_{y0} = \int_{\Omega} z_0^2 dy_0 dz_0$$

$$I_{z0} = \int_{\Omega} y_0^2 dy_0 dz_0$$

$$I_{yz0} = \int_{\Omega} y_0 z_0 dy_0 dz_0$$

## Properties in Centroid System

$$I_y = I_{y0} - z_{0C}^2 A$$

$$I_z = I_{z0} - y_{0C}^2 A$$

$$I_{yz} = I_{yz0} - y_{0C} z_{0C} A$$

## Principal Axes System

Principal moments of inertia.  $I_u$  is the larger one.

$$I_{\bar{y}} = \frac{1}{2} \left[ (I_y + I_z) + \sqrt{(I_y - I_z)^2 + 4I_{yz}^2} \right]$$

$$I_{\bar{z}} = \frac{1}{2} \left[ (I_y + I_z) - \sqrt{(I_y - I_z)^2 + 4I_{yz}^2} \right]$$

Radii of gyration

$$r_{\bar{y}} = \sqrt{\frac{I_{\bar{y}}}{A}}$$

$$r_{\bar{z}} = \sqrt{\frac{I_{\bar{z}}}{A}}$$

Direction of Principal Axes

$$\tan 2\phi = \frac{2I_{yz}}{I_z - I_y}$$

Here add some remarks how to find which direction belongs to major axis and which to the minor.

## Applied Loads

Normal force  $N$  and bending moments  $M_{yP}$ ,  $M_{zP}$  applied at some load application point P. P is given in reference system:  $y_{0P}$ ,  $z_{0P}$ .

Moments with respect to centroid:

$$M_y = M_{yP} - (z_{0C} - z_{0P})N$$

$$M_z = M_{zP} + (y_{0C} - y_{0P})N$$

Moments with respect to principal axes:

$$M_{\bar{y}} = M_y \cos \phi + M_z \sin \phi$$

$$M_{\bar{z}} = -M_y \sin \phi + M_z \cos \phi$$

# Stresses

$$\sigma(\bar{y}, \bar{z}) = \frac{N}{A} + \bar{z} \frac{M_{\bar{y}}}{I_{\bar{y}}} - \bar{y} \frac{M_{\bar{z}}}{I_{\bar{z}}}$$

For that, we need to transform point coordinates into principal axis system. Given some point V ( $y_{0V}$ ,  $z_{0V}$ ), what are coordinates  $\bar{y}_V$ ?

Point in centroid system y,z:

$$\begin{aligned} y_V &= y_{0V} - y_{0C} \\ z_V &= z_{0V} - z_{0C} \end{aligned}$$

Point in principal axes system:

$$\begin{aligned} \bar{y}_V &= y_V \cos \phi + z_V \sin \phi \\ \bar{z}_V &= -y_V \sin \phi + z_V \cos \phi \end{aligned}$$