# Recent Advances in Internal Combustion Engines

4A13

#### To SI or to CI?





	SI	CI
Efficiency	lower	higher
NOx	lower	higher
PM	lower	higher
Cost	lower	higher

Is there something in-between?
Can we combine the advantages?

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## Homogeneous charge/partly mixed charge Compression Ignition (HCCI/PCCI)

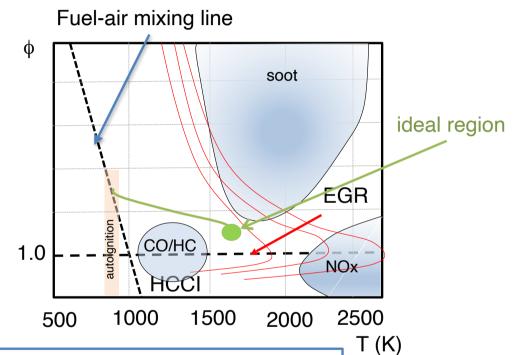
Homogeneous CCI (HCCI) (late 90s):

- premixed mixture
- CR depends of fuel to autoignite
- load limited to low values (to avoid uncontrolled ignition/high noise/vibration)

Partially premixed CI (PPCI) PREDIC, UNIBUS, LTC, using early injection or split injection

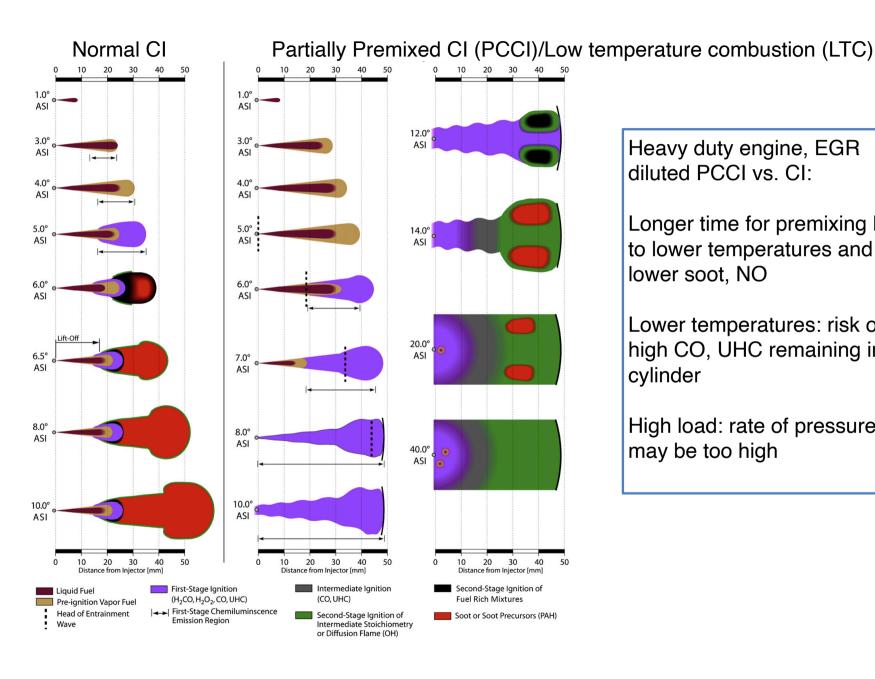
Partially premixed GDI-CI

- low octane fuel: autoignition at low temps



Goal is to reduce NO, soot emissions by increasing premixing. Constraints are:

- (a) the ability to control onset of autoignition at a given time/crankangle to maximize efficiency,
- (b) control the rate of combustion to minimize noise, vibration and harshness (NVH)
- (c) minimise reaction quench which lead to high CO,HC.



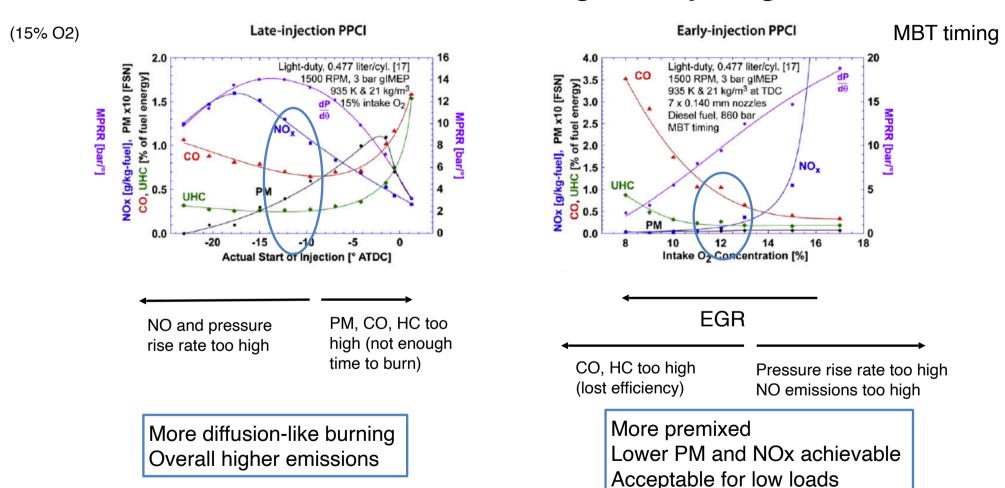
Heavy duty engine, EGR diluted PCCI vs. CI:

Longer time for premixing leads to lower temperatures and lower soot, NO

Lower temperatures: risk of high CO, UHC remaining in cylinder

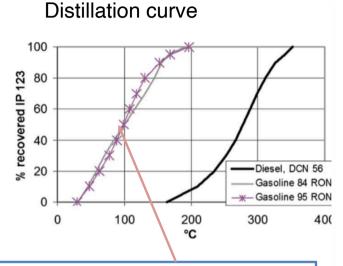
High load: rate of pressure rise may be too high

#### Emissions PPCI/LTC: Light duty engine



Musculus et al., PECS 39 (2013) 246-283

#### GDI-PCCI gasoline



Gasoline ON 84 vaporizes promptly compared to diesel: more time for premixing

Produces **much less NOx** than diesel at same load

Requires **boosting** and **intermediate CR** to autoignite

Range of delays for diesel: Longer delay for gasoline: short, so no time for premix more premixing 20 ◆Diesel, 1.1 bar Pi Gas, 84RON, 1.1 bar Pi △Gas, 95 RON, 1.5 bar Pi 16 [4/k/k] xONSI 8 4 Gas, Ref 10 1.5 bar Pi Diesel, Ref 10, 1.5 bar Pi heated intake 60 C CR = 16 $p_i = 1.5 \text{ bar}$ 0 0 50 60 Combustion delay (CA50-SOI) [CAD]

Fig. 4 ISNO<sub>x</sub> versus combustion delay for cases in Fig. 2. Also comparing with data from Ref. [10].

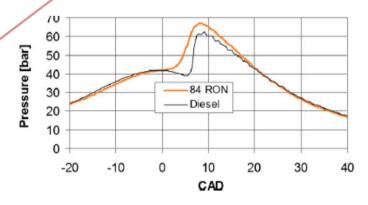


Fig. 7 Pressure and heat release rates at 3.95 bar IMEP for diesel and 84 RON gasoline

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### Pathway to 50% Brake Thermal Efficiency Using Gasoline Direct Injection Compression Ignition

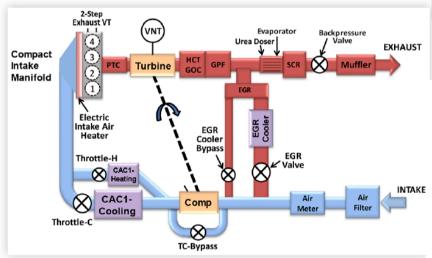
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Citation: Sellnau, M., Foster, M., Moore, W., Sinnamon, J. et al., "Pathway to 50% Brake Thermal Efficiency Using Gasoline Direct Injection Compression Ignition," SAE Int. J. Advances & Curr. Prac. in Mobility 1(4):1581-1603, 2019, doi:10.4271/2019-01-1154.

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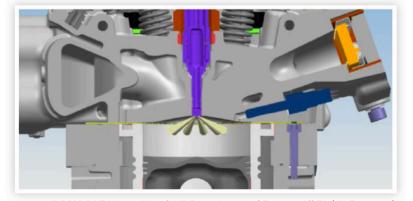
**FIGURE 2** Air system for Gen3X engine showing fast air blend system with one heating CAC, one cooling CAC, and two throttles.



**GDCI** 

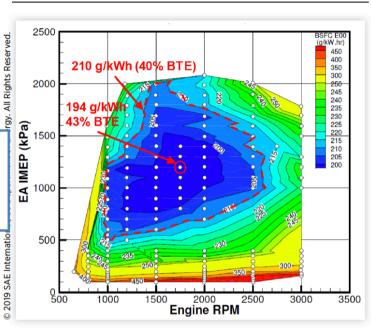
NOx less than 0.02 g/kWh and smoke less than 0.08 FSN: lower than engine-out emissions of most gasoline and diesel engines

FIGURE 4 Gen3X GDCI cylinder head.



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**FIGURE 24** Initial BSFC $_{00}$  map for Gen3X GDCI engine using a 500 bar GDi fuel system and a single-stage boost system with a new GDCI-diffusion combustion strategy for high load (full lean operation without EGR).



43% broad peak gasoline operaton low emissions

#### Summary: advanced concepts in ICEs

- Partially premixed CI (PPCI) or Low Temperature Combustion (LTC) concepts now taking IC engines to diesel like (45-50%) efficiencies at SI like emissions at CRs around 14-16
- Merging of concepts using gasoline for low emissions; operation in multimode (CI or SI) depending on load.
- Use of more volatile fuels sign as gasoline with lower soot formation tendency
- Concepts involve supercharging using electric storage and/or turbocharging
- Trend varies with vehicle application, but largely combines downsizing/ boosting with compression-ignited direct injection for low temperature, partially mixed, low PM concepts



http://to.eng.cam.ac.uk/teaching/surveys/4A13\_Lent.html。