

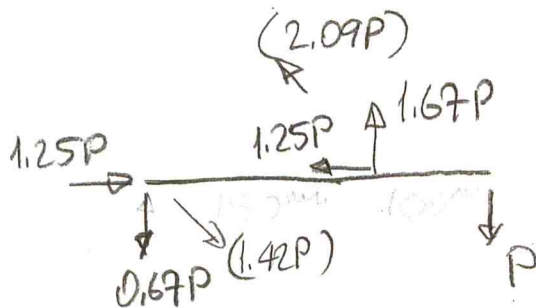
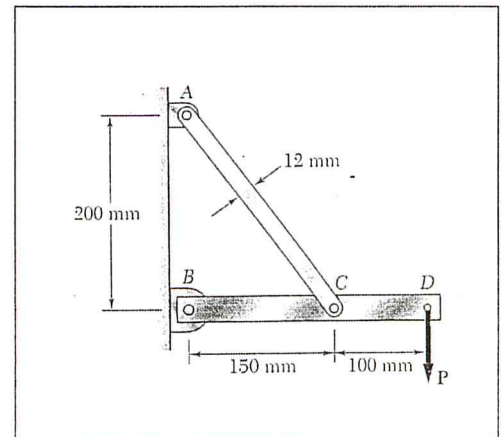
Name: **SOLUTION**

Signature:

Member AC has a 6x12mm uniform rectangular cross section and the maximum normal stress allowed on this member is 150 MPa. Member AC is connected to a support at A and to a rigid member BCD at C by 10 mm diameter pins. Member BCD is connected to the support at B by an 8 mm diameter pin. All of the pins are made of steel with a maximum allowable shearing stress of 60 MPa.

Determine the largest load **P** that can safely be applied at D without exceeding given the stress limits.

Please show all your calculations clearly!



Tension in AC

$$\frac{2.09P}{6 \times (12 - 10)} = 150 \text{ MPa} \Rightarrow P = \underline{\underline{861 \text{ N}}}$$

Pins @ A & C

$$\frac{2.09P}{\pi \times \frac{10^2}{4}} = 60 \text{ MPa} \Rightarrow P = \underline{\underline{2254 \text{ N}}}$$

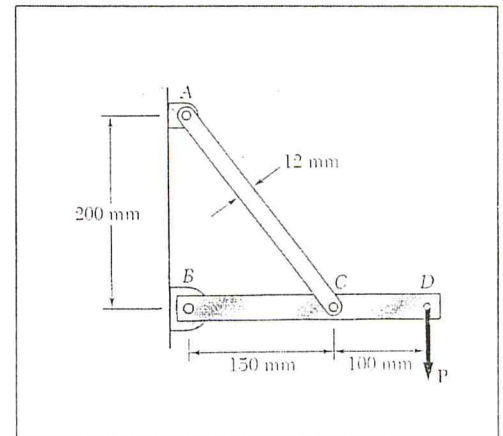
Pin @ B

$$\frac{1.42P}{\pi \times \frac{8^2}{4}} = 60 \text{ MPa} \Rightarrow P = \underline{\underline{2124 \text{ N}}}$$

$$\therefore P_{\max} = \underline{\underline{861 \text{ N}}}$$

METU Civil Engineering Department	CE 224 Mechanics of Materials Summer 2014	Quiz #1
Name: SOLUTION	Signature:	Time: 20 mins

Member AC has a 6x12mm uniform rectangular cross section and the maximum normal stress allowed on this member is 150 MPa. Member AC is connected to a support at A and to a rigid member BCD at C by 10 mm diameter pins. Member BCD is connected to the support at B by an 8 mm diameter pin. All of the pins are made of steel with a maximum allowable shearing stress of 60 MPa. All pins are in single shear.



Determine the largest load **P** that can safely be applied at D without exceeding given the stress limits.

Please show all your calculations clearly!

$$\begin{aligned} (F_{AC})_{\max} &= 150 \text{ MPa} \times (6 \times (12 \text{ mm} - 10 \text{ mm})) = 1800 \text{ N} \\ (F_{AC})_{\max} &= 60 \text{ MPa} \times \left(\pi \times \frac{10^2}{4} \right) = 4712 \text{ N} \\ (F_B)_{\max} &= 60 \text{ MPa} \times \left(\pi \times \frac{8^2}{4} \right) = 3016 \text{ N} \end{aligned} \quad \Rightarrow (F_{AC})_{\max} = 1800 \text{ N}$$

$$\begin{aligned} R_{bx} &= 1080 \text{ N} \rightarrow \\ R_{by} &= 576 \text{ N} \downarrow \\ P &= 864 \text{ N} \downarrow \\ (F_{AC}) &= (F_{AC})_{\max} \end{aligned}$$

$$R_b = \sqrt{576^2 + 1080^2} = 1224 \text{ N} < (F_B)_{\max} \checkmark$$

$$\therefore P_{\max} = 864 \text{ N}$$

Name: SOLUTION

Signature:

Time: 20 mins

Rigid bar ABCD is pinned at point C and supported by rods AE and DF.

A concentrated load $P=40$ kN is applied at point B.

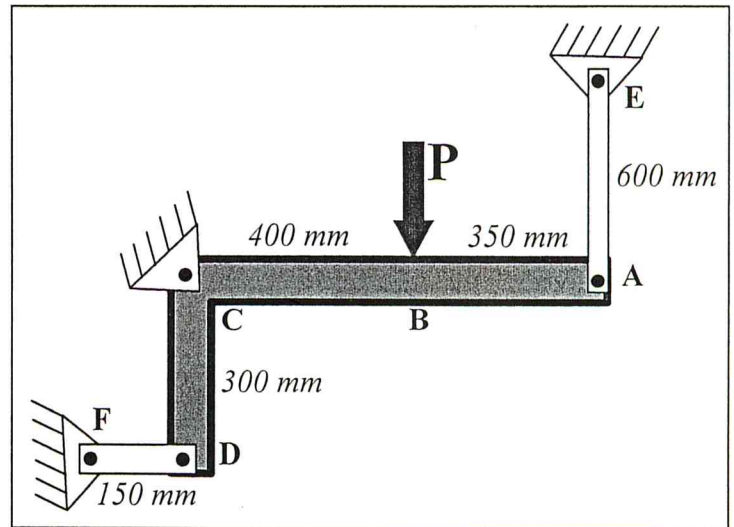
Determine:

- Force in rod AE
- Force in rod DF
- Vertical displacement at point A

Please show all your calculations clearly!

Rod AE: $A=70 \text{ mm}^2$ $E=200 \text{ GPa}$

Rod DF: $A=625 \text{ mm}^2$ $E=100 \text{ GPa}$



40 kN

400 mm 350 mm F_{AE}

300 mm

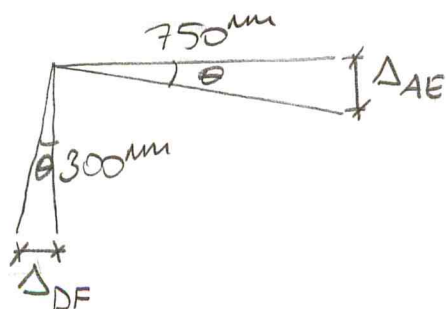
F_{DF}

$$F_{AE} \times 750 \text{ mm} + F_{DF} \times 300 \text{ mm} = 40 \text{ kN} \times 400 \text{ mm}$$

$$\Delta_{AE} = \frac{F_{AE} \times 600 \text{ mm}}{70 \text{ mm}^2 \times 200 \text{ GPa}} = 0.043 F_{AE}$$

$$\Delta_{DF} = \frac{F_{DF} \times 150 \text{ mm}}{625 \text{ mm}^2 \times 100 \text{ GPa}} = 0.0024 F_{DF}$$

Compatibility condition:



$$\frac{\Delta_{DF}}{300 \text{ mm}} = \frac{\Delta_{AE}}{750 \text{ mm}}$$

$$\frac{0.0024 F_{DF}}{300 \text{ mm}} = \frac{0.043 F_{AE}}{750 \text{ mm}}$$

$$F_{DF} = 7.17 F_{AE}$$

$$\Rightarrow (F_{AE}) \times (750 \text{ mm}) + (7.17 F_{AE}) \times (300 \text{ mm}) = 40 \text{ kN} \times 400 \text{ mm}$$

$$\Delta_{AE} = 0.043 \times 5.5 = 0.24 \text{ mm}$$

$$F_{AE} = 5.5 \text{ kN}$$

$$F_{DF} = 39.5 \text{ kN}$$

$$\Delta_{DF} = 0.0024 \times 39.5 = 0.09 \text{ mm}$$

METU Civil Engineering Department	CE 224 Mechanics of Materials Summer 2014	Quiz #3
Name: <i>SOLUTION</i>	Signature:	Time: 30 mins

Two shafts, each of 22 mm diameter, are connected by the gears shown. The shafts are made of a material that has a shear modulus of $G=77$ GPa. The rotation of shaft DF is fixed at point F. Determine the total rotation of point A when a torque of 130 N-m is applied at this point.

Please show all your calculations clearly!

$$\tau = \frac{T \cdot r}{J} \quad \phi = \frac{T \cdot L}{J \cdot G} \quad J_{\text{circle}} = \frac{\pi}{2} \cdot r^4$$

$$T_{AB} = 130 \text{ N-m}$$

$$T_{EF} = \frac{130 \text{ N-m}}{110 \text{ mm}} \cdot 150 \text{ mm} = 177.3 \text{ N-m}$$

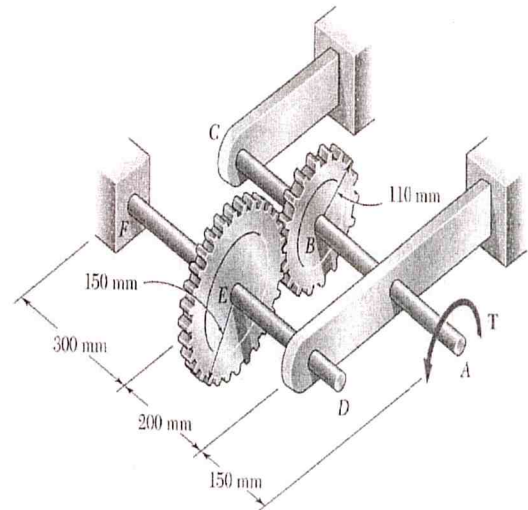
$$\theta_{A/B} = \frac{130,000 \cdot 350}{23,000 \cdot 77,000} = 0,0267 \text{ rad} = 1.47 \text{ deg.}$$

$$\theta_{E/F} = \frac{177,300 \cdot 300}{23,000 \cdot 77,000} = 0,03 \text{ rad} = 1,72 \text{ deg.}$$

$$\Rightarrow \theta_A = 1,72^\circ \cdot \frac{150 \text{ mm}}{110 \text{ mm}} + 1,47^\circ$$

$$= 2,34^\circ + 1,47^\circ \quad (0,04 \text{ rad})$$

$$\theta_A = 3,81^\circ \quad (0,066 \text{ rad})$$



$$J = \frac{\pi}{2} \cdot 11^4 = 23000 \text{ mm}^4$$