

CE382 - HOMEWORK 3

1) For C25 & S620 $f_{cd} = 17 \text{ MPa}$, $f_{yd} = 365 \text{ MPa}$

$$j_M = 0.976, K_M = 133 \text{ mm}^2/\text{kN}$$

$$j_L = 0.86, K_L = 231 \text{ mm}^2/\text{kN}$$

K101

$$K = \frac{b d^2}{M_d} = \frac{1000 \times 460^2}{275000} = 765 > K_L \checkmark \text{ OK.}$$

$$\left. \begin{aligned} j \cdot d &= 0.9d = 414 \text{ mm} \\ j \cdot d &= d - \frac{t}{2} = 330 \text{ mm} \end{aligned} \right\} j \cdot d = 414 \text{ mm}$$

$$A_s = \frac{M_d}{f_{yd} j \cdot d} = \frac{275000}{0.365 \times 414} = 1820 \text{ mm}^2$$

$$4\phi 20 \text{ bent bars} = 1256 \text{ mm}^2$$

$$4\phi 14 \text{ straight bars} = 616 \text{ mm}^2$$

$$1872 \text{ mm}^2 > A_{s, \text{req}}$$

Support 1

$$M_d = 245 - \frac{250 \times 0.4}{3} = 211.7 \text{ kN.m}$$

$$K = \frac{300 \times 460^2}{211700} = 300 > K_L \checkmark \text{ OK}$$

$$A_s = \frac{211700}{0.365 \times 0.86 \times 460} = 1466 \text{ mm}^2$$

$$\text{available } 4\phi 20 \text{ bent bars} = 1256 \text{ mm}^2$$

$$2\phi 12 \text{ longer } = 226 \text{ mm}^2$$

$$1486 \text{ mm}^2 > A_{s, \text{req}}$$

K102

$$K = \frac{1000 \times 460^2}{100000} = 2116 \text{ mm}^2/\text{kN} > K_L$$

$$A_s = \frac{100000}{0.365 \times 414} = 662 \text{ mm}^2$$

$$2\phi 14 \text{ straight bars} = 308 \text{ mm}^2$$

$$2\phi 20 \text{ bent } = 628 \text{ mm}^2$$

$$936 \text{ mm}^2 > A_{s, \text{req}}$$

Support 2

$$M_d = 330 - \frac{270 \times 0.4}{3} = 294 \text{ kN.m}$$

$$K = \frac{300 \times 460^2}{234000} = 216 < K_L \text{ need double reinf. sect.}$$

$$M_1 = \frac{300 \times 460^2}{231} = 218.14 \text{ kN.m}$$

$$M_2 = M_d - M_1 = 75.86 \text{ kN.m}$$

$$A_{s1} = \frac{218140}{0.365 \times 0.86 \times 460} = 1511 \text{ mm}^2$$

$$A_{s2} = A_{s1} \cdot \frac{M_2}{f_{yd}(d-d')}$$

$$A_s = 1511 + 435 = 1946 \text{ mm}^2$$

$$\text{Top — available } 4\phi 20 \text{ bent bars} = 1256 \text{ mm}^2$$

$$2\phi 20 \text{ " " } = 628 \text{ mm}^2$$

$$2\phi 12 \text{ longer " } = 226 \text{ mm}^2$$

$$2110 \text{ mm}^2 > A_{s, \text{req}}$$

$$\text{Bottom — available } 4\phi 14 \text{ straight bars} = 616 \text{ mm}^2$$

$$2\phi 14 \text{ " " } = 308 \text{ mm}^2$$

$$924 \text{ mm}^2 > A_{s, \text{req}}$$

K103

$$K = \frac{1000 \times 460^2}{200000} = 1058 > K_L \checkmark \text{ OK}$$

$$A_s = \frac{200000}{0.365 \times 414} = 1324 \text{ mm}^2$$

$$4\phi 20 \text{ bent bars} = 1256 \text{ mm}^2$$

$$2\phi 14 \text{ straight " } = 308 \text{ mm}^2$$

$$1564 \text{ mm}^2 > A_{s, \text{req}}$$

Support 3

$$M_d = 300 - \frac{195 \times 0.4}{3} = 276 \text{ kN.m}$$

$$K = \frac{300 \times 460^2}{276000} = 230 < K_L \quad \text{need double reinf. sect.}$$

$$M_1 = \frac{300 \times 460^2}{231} = 218.14 \text{ kN.m}$$

$$M_2 = M_d - M_1 = 57.86 \text{ kN.m}$$

$$A_{s1} = \frac{218140}{0.365 \times 0.86 \times 460} = 1511 \text{ mm}^2$$

$$A_{s2} = A_{s1}' = \frac{57860}{0.365 \times 420} = 348 \text{ mm}^2$$

$$A_s = A_{s1} + A_{s1}' = 1889 \text{ mm}^2$$

Top — Available 2Ø20 bent bars = 628 mm²
 " 4Ø20 " " = 1256 mm²
 2Ø12 haupen " = 226 mm²

Bottom — available 2Ø14 straight bars = 308 mm² $2110 \text{ mm}^2 > A_{s, \text{req}}$
 " 2Ø14 " " = 308 mm²
 616 mm² $> A_{s1}'$

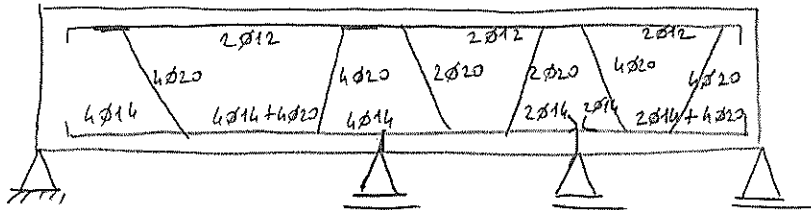
Support 4

$$M_d = 177 - \frac{170 \times 0.4}{3} = 154.3 \text{ kN.m}$$

$$K = \frac{300 \times 460^2}{154300} = 411 > K_L \quad \checkmark \text{ OK}$$

$$A_s = \frac{154300}{0.365 \times 0.86 \times 460} = 1069 \text{ mm}^2$$

available 4Ø20 bent bars = 1256 mm²
 2Ø12 haupen " = 226 mm²
 1482 mm² $> A_{s, \text{req}}$



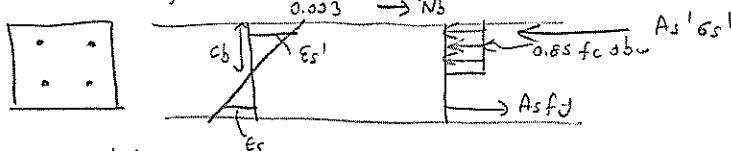
2) a) $N_{or} = 0.85 f_{cd} A_c + A_s f_{yd} = 0.85 \times 17 \times (400 \times 400) + 4 \times \frac{\pi \times 20^2}{4} \times 365 = 2972.5 \text{ kN}$

b) Tension capacity

direct tensile — $f_{ctd} = 0.35 \sqrt{17} = 1.44 \text{ MPa}$

$$N_t = f_{ctd} \times A_c + A_s \times f_{yd} = 831 \text{ kN}$$

c)



For balance case $\epsilon_s = \epsilon_y = \frac{365}{200000} = 0.00183$

$$\frac{0.003}{c_b} = \frac{0.00183}{360 - c_b} \quad \text{— } c_b = 223.6 \text{ mm}$$

$$\frac{0.003}{c_b} = \frac{\epsilon_s'}{c_b - 40} \quad \text{— } \epsilon_s' = 2.5 \times 10^{-3} > \epsilon_y \quad \text{comp. steel yields.}$$

$$N_b + A_s f_y = 0.85 f_c \sigma b_w + A_s' \sigma_s'$$

$$N_b = 1218 \text{ kN}$$

$$M_b = 2 \times (A_s f_y) \times (160) + (0.85 \times f_c \times \sigma \times b_w) \times (200 - \frac{\sigma}{2}) = 221 \text{ kN.m}$$

$$d) F_c = 0.85 \times 17 \times 0.85 \times 400 = 4913 \text{ c}$$

$$F_s = 2 \times \frac{\pi 24^2}{4} \times 365 = 330264 \text{ N}$$

$$F_s' = 2 \times \frac{\pi 24^2}{4} \times 65' = 906.8 \times 200000 \times \left(0.03 - \frac{0.12}{c}\right) = 542880 - \frac{21715200}{c}$$

$$N = 0 = F_c + F_s' - F_s \quad \text{--- } c = 48.35 \text{ mm}$$

$$F_c = 237.5 \text{ kN}, F_s = 330.2 \text{ kN}, F_s' = 93.7 \text{ kN}$$

$$M = F_c \times \left(200 - \frac{0.85 \times c}{2}\right) + F_s \times 160 + F_s' \times 160 = 110.4 \text{ kN.m}$$

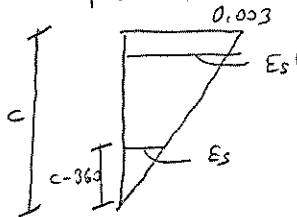
$$e) N = 0.85 N_{or} = 2527 \text{ kN}$$

$N > N_b$ — comp. failure

assume all section in compression.

$$E_s' > E_y$$

$$\frac{0.003}{c} = \frac{E_s}{c-360}$$



$$F_c = 0.85 \times 17 \times 0.85 \times 400 = 4913 \text{ c}$$

$$F_s' = 365 \times 2 \times \frac{\pi 24^2}{4} = 330252$$

$$F_s = 2 \times \frac{\pi 24^2}{4} \times 200000 \times \left(0.003 - \frac{1.08}{c}\right) = 542880 - \frac{195436800}{c}$$

$$N = 2527000 = F_c + F_s' + F_s \quad \text{--- } c = 429.3 \text{ mm}$$

$c > h$ assumption is correct ✓

$$F_c = 2103 \text{ kN}$$

$$F_s = 877 \text{ kN}$$

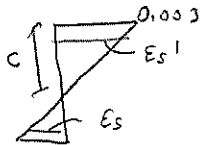
$$F_s' = 330.2 \text{ kN}$$

$$M = F_c \times \left(200 - \frac{c}{2}\right) + F_s \times 160 + F_s' \times 160 = 103.9 \text{ kN.m}$$

$$f) N = 0.45 N_{or} = 1337.6 \text{ kN}$$

$N > N_b$ — comp. failure

assume bottom steel is under tension.



$$\frac{0.003}{c} = \frac{E_s}{360-c}$$

$$F_c = 4913 \text{ c}$$

$$F_s' = 330252$$

$$F_s = \frac{195436800}{c} - 542880$$

$$N = F_c + F_s' - F_s$$

$$c = 252.2 \text{ mm}$$

$c < h$ ✓ assumption is correct

$$F_c = 1233 \text{ kN}$$

$$F_s = 330.2 \text{ kN}$$

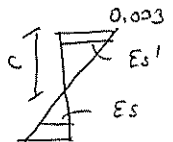
$$F_s' = 232 \text{ kN}$$

$$M = F_c \times \left(200 - \frac{c}{2}\right) + 160 \times F_s + F_s' \times 160 = 205 \text{ kN.m}$$

$$g) N = 0.15 N_{or} = 446 \text{ kN}$$

$N < N_b$ — tension failure

assume $\epsilon_s = f_y$ & $\epsilon_s' < f_y$



$$\frac{0.003}{c} = \frac{E_s'}{c-40}$$

$$E_s' = 0.003 - \frac{0.12}{c}$$

$$F_c = 4913 \text{ c}$$

$$F_s = 330264$$

$$F_s' = E_s' \times 200000 \times A_s' = 542880 - \frac{21715200}{c}$$

$$N = F_c + F_s - F_s' \quad \text{--- } c = 75.7 \text{ mm}$$

from compatibility $\left\{ \begin{array}{l} E_s' = 0.0014 < E_y \\ E_s = 0.011 > E_y \end{array} \right\}$ assumption is correct.

$$\left. \begin{aligned} F_c &= 371.8 \text{ kN} \\ F_s &= 330.2 \text{ kN} \\ F_s' &= 256 \text{ kN} \end{aligned} \right\}$$

$$M = F_c * \left(200 - \frac{d}{2}\right) + 160 * F_s + F_s' * 160 = 156.2 \text{ kN.m}$$

