

$$Q = \int (\vec{v} \cdot \vec{n}) dA \quad \dot{m} = \rho Q$$

• Bernoulli:

$$p_1 + \rho g z_1 + \frac{1}{2} \rho v_1^2 = p_2 + \rho g z_2 + \frac{1}{2} \rho v_2^2$$

## FORMULÁRIO

### Forma integral das equações de conservação

Massa:  $\frac{d}{dt} \int_{V_c} \rho \, dV + \int_{S_c} \rho (\vec{v}_r \cdot \vec{n}) \, dS = 0$

Quantidade de movimento:  $\frac{d}{dt} \int_{V_c} \rho \vec{v} \, dV + \int_{S_c} \rho \vec{v} (\vec{v}_r \cdot \vec{n}) \, dS = \Sigma \vec{F}_{\text{act. no-}V_c / \text{fluido}}$

Energia:  $\frac{d}{dt} \int_{V_c} \rho \left( e + \frac{1}{2} V^2 + gz \right) dV + \int_{S_c} \rho \left( e + \frac{1}{2} V^2 + gz \right) (\vec{v}_r \cdot \vec{n}) \, dS = \dot{Q} - \dot{W}_m - \dot{W}_p - \dot{W}_v$

Momento angular:  $\frac{d}{dt} \int_{V_c} \rho (\vec{r}_0 \times \vec{v}) \, dV + \int_{S_c} \rho (\vec{r}_0 \times \vec{v}) (\vec{v}_r \cdot \vec{n}) \, dS = \Sigma \vec{M}_0$

### Equações de Navier-Stokes

Equações do movimento para um escoamento incompressível de um fluido newtoniano com propriedades constantes em coordenadas cartesianas:

Continuidade:

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0$$

Transporte de quantidade de movimento:

$$\rho \left[ \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right] = \rho g_x - \frac{\partial p}{\partial x} + \mu \left[ \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right]$$

$$\rho \left[ \frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} \right] = \rho g_y - \frac{\partial p}{\partial y} + \mu \left[ \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right]$$

$$\rho \left[ \frac{\partial w}{\partial t} + u \frac{\partial w}{\partial x} + v \frac{\partial w}{\partial y} + w \frac{\partial w}{\partial z} \right] = \rho g_z - \frac{\partial p}{\partial z} + \mu \left[ \frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right]$$

Componentes do tensor das tensões viscosas:

$$\tau_{xx} = 2\mu \frac{\partial u}{\partial x} \quad \tau_{xz} = \tau_{zx} = \mu \left[ \frac{\partial w}{\partial x} + \frac{\partial u}{\partial z} \right]$$

$$\tau_{yy} = 2\mu \frac{\partial v}{\partial y} \quad \tau_{yz} = \tau_{zy} = \mu \left[ \frac{\partial w}{\partial y} + \frac{\partial v}{\partial z} \right]$$

$$\tau_{zz} = 2\mu \frac{\partial w}{\partial z}$$

$$\tau_{xy} = \tau_{yx} = \mu \left[ \frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} \right]$$

Continuidade:

$$\frac{1}{r} \frac{\partial (r v_r)}{\partial r} + \frac{1}{r} \frac{\partial (v_\theta)}{\partial \theta} + \frac{\partial (v_z)}{\partial z} = 0$$

$$\begin{aligned} & \rho \left[ \frac{\partial v_r}{\partial t} + v_r \frac{\partial v_r}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_r}{\partial \theta} + v_z \frac{\partial v_r}{\partial z} - \frac{v_\theta^2}{r} \right] \\ &= \rho g_r - \frac{\partial p}{\partial r} + \mu \left[ \frac{\partial}{\partial r} \left( \frac{1}{r} \frac{\partial (r v_r)}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_r}{\partial \theta^2} + \frac{\partial^2 v_r}{\partial z^2} - \frac{2}{r^2} \frac{\partial v_\theta}{\partial \theta} \right] \end{aligned}$$

$$\begin{aligned} & \rho \left[ \frac{\partial v_\theta}{\partial t} + v_r \frac{\partial v_\theta}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_\theta}{\partial \theta} + v_z \frac{\partial v_\theta}{\partial z} + \frac{v_r v_\theta}{r} \right] \\ &= \rho g_\theta - \frac{1}{r} \frac{\partial p}{\partial \theta} + \mu \left[ \frac{\partial}{\partial r} \left( \frac{1}{r} \frac{\partial (r v_\theta)}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_\theta}{\partial \theta^2} + \frac{\partial^2 v_\theta}{\partial z^2} + \frac{2}{r^2} \frac{\partial v_r}{\partial \theta} \right] \end{aligned}$$

$$\begin{aligned} & \rho \left[ \frac{\partial v_z}{\partial t} + v_r \frac{\partial v_z}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_z}{\partial \theta} + v_z \frac{\partial v_z}{\partial z} \right] = \\ & \rho g_z - \frac{\partial p}{\partial z} + \mu \left[ \frac{1}{r} \frac{\partial}{\partial r} \left( r \frac{\partial v_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_z}{\partial \theta^2} + \frac{\partial^2 v_z}{\partial z^2} \right] \end{aligned}$$

$$\tau_{rr} = 2\mu \frac{\partial v_r}{\partial r} \quad \tau_{\theta\theta} = 2\mu \left[ \frac{1}{r} \frac{\partial v_\theta}{\partial \theta} + \frac{v_r}{r} \right] \quad \tau_{zz} = 2\mu \frac{\partial v_z}{\partial z}$$

$$\tau_{r\theta} = \tau_{\theta r} = \mu \left[ r \frac{\partial}{\partial r} \left( \frac{v_\theta}{r} \right) + \frac{1}{r} \frac{\partial v_r}{\partial \theta} \right]$$

$$\tau_{\theta z} = \tau_{z\theta} = \mu \left[ \frac{\partial v_\theta}{\partial z} + \frac{1}{r} \frac{\partial v_z}{\partial \theta} \right] \quad \tau_{rz} = \tau_{zr} = \mu \left[ \frac{\partial v_z}{\partial r} + \frac{\partial v_r}{\partial z} \right]$$

$$\frac{1}{\sqrt{f}} = -1,8 \log \left[ \frac{6,9}{Re} + \left( \frac{\epsilon/D}{3,7} \right)^{1,11} \right]$$

$$Re = \frac{\bar{v} D}{\nu} \quad \nu = \frac{\mu}{\rho}$$

$\downarrow$   
 $m^2/s$

$\epsilon \rightarrow$  Rugosidade

### Curva da Instalação

$$H = \frac{p_B - p_A}{\rho g} + z_B - z_A + \left[ \frac{f(l/d)}{2gA^2} + \frac{\sum k}{2gA^2} \right] Q^2 \quad H_s = \frac{p_1}{\rho g} + \frac{V_1^2}{2g} - \frac{p_{vap}}{\rho g}$$

### Turbomáquinas

Potência ao veio em máquinas hidráulicas:  $P = \rho g Q H \eta$  ou  $P = \rho g Q H / \eta$

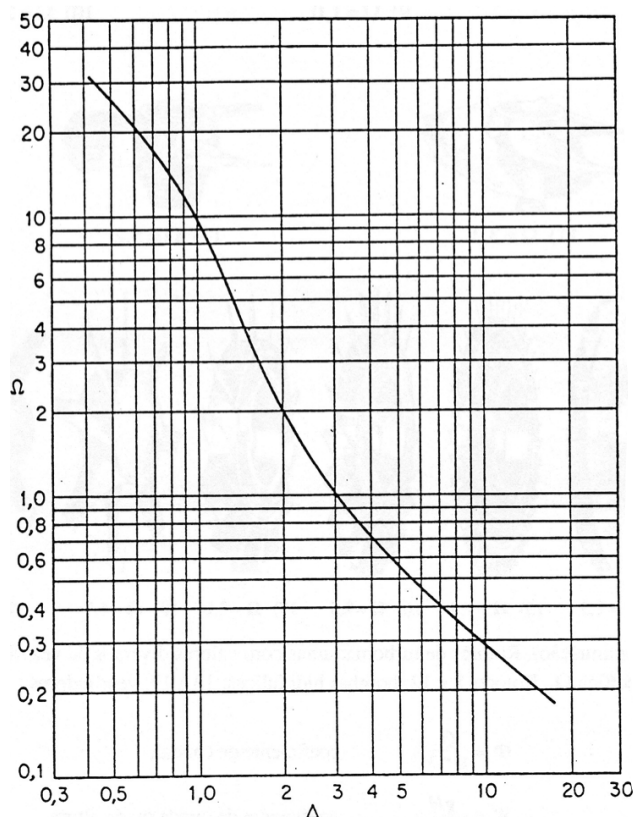
Coefficientes adimensionais em turbomáquinas:

$$C_P = \frac{P}{\rho N^3 D^5} \quad C_Q = \frac{Q}{ND^3} \quad C_H = \frac{gH}{N^2 D^2} \quad \Omega = \frac{N\sqrt{Q}}{(gH)^{3/4}} \quad \Delta = \frac{D(gH)^{1/4}}{\sqrt{Q}} \quad S_i = \frac{N\sqrt{Q}}{(gH_{Si})^{3/4}}$$

Pontos dinamicamente semelhantes (mesma máquina):  $\frac{Q_1}{Q_2} = \frac{N_1}{N_2} = \left( \frac{H_1}{H_2} \right)^{1/2}$

Para bombas e turbinas temos, respetivamente:  $S_{i\eta} = 3,0$  e  $S_{i\eta} = 4,0$

Diagrama de Cordier:



- Altura aspiração disponível, bombas e turbinas:

$$H_s = \frac{p_{atm} - p_{vap}}{\rho g} - e_s - z_{asp} \quad H_s = \frac{p_{atm} - p_{vap}}{\rho g} - e_s$$

$$P \approx N \eta_{max}$$

$$\text{Gráfico} = (Q_{\eta max})_1 \cdot (H_{\eta max})_1 \quad | \quad H_{\eta max} = \frac{H_1}{Q_1^2} Q^2$$

$$H_{\eta max} = H_{inst} \Leftrightarrow Q_{funcionando} \quad | \quad \text{(*)}$$

- Altura max bomba:

$$H_{Si} \text{ por } S_i \quad | \quad H_s > H_{Si} \text{ em ordem a } e_s$$

- Diâmetro bomba

$$\Omega = \frac{N\sqrt{Q}}{(gH)^{3/4}} \quad | \quad \Delta = \frac{D}{AL} \quad | \quad D = \frac{\Delta \sqrt{Q_{\eta max}}}{(gH_{\eta max})^{1/4}}$$