

FORMULÁRIO

Forma integral das equações de conservação

$$\text{Massa: } \frac{d}{dt} \int_{V_c} \rho \, dV + \int_{S_c} \rho (\vec{v}_r \cdot \vec{n}) \, dS = 0$$

$$\text{Quantidade de movimento: } \frac{d}{dt} \int_{V_c} \rho \vec{v} \, dV + \int_{S_c} \rho \vec{v} (\vec{v}_r \cdot \vec{n}) \, dS = \Sigma \vec{F}$$

$$\text{Energia: } \frac{d}{dt} \int_{V_c} \rho \left(e + \frac{1}{2} V^2 + gz \right) dV + \int_{S_c} \rho \left(e + \frac{1}{2} V^2 + gz \right) (\vec{v}_r \cdot \vec{n}) \, dS = \dot{Q} - \dot{W}_m - \dot{W}_p - \dot{W}_v$$

$$\text{Momento angular: } \frac{d}{dt} \int_{V_c} \rho (\vec{r}_0 \times \vec{v}) \, dV + \int_{S_c} \rho (\vec{r}_0 \times \vec{v}) (\vec{v}_r \cdot \vec{n}) \, dS = \Sigma \vec{M}_0$$

Equações de Navier-Stokes

Equações do movimento para um escoamento incompressível de um fluido newtoniano com propriedades constantes em coordenadas cartesianas:

Continuidade:

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0$$

Transporte de quantidade de movimento:

$$\rho \left[\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right] = \rho g_x - \frac{\partial p}{\partial x} + \mu \left[\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right]$$

$$\rho \left[\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} \right] = \rho g_y - \frac{\partial p}{\partial y} + \mu \left[\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right]$$

$$\rho \left[\frac{\partial w}{\partial t} + u \frac{\partial w}{\partial x} + v \frac{\partial w}{\partial y} + w \frac{\partial w}{\partial z} \right] = \rho g_z - \frac{\partial p}{\partial z} + \mu \left[\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right]$$

Componentes do tensor das tensões viscosas:

$$\tau_{xx} = 2\mu \frac{\partial u}{\partial x}$$

$$\tau_{yy} = 2\mu \frac{\partial v}{\partial y}$$

$$\tau_{zz} = 2\mu \frac{\partial w}{\partial z}$$

$$\tau_{xy} = \tau_{yx} = \mu \left[\frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} \right]$$

$$\tau_{xz} = \tau_{zx} = \mu \left[\frac{\partial w}{\partial x} + \frac{\partial u}{\partial z} \right]$$

$$\tau_{yz} = \tau_{zy} = \mu \left[\frac{\partial w}{\partial y} + \frac{\partial v}{\partial z} \right]$$

Curva da Instalação

$$H = \frac{p_B - p_A}{\rho g} + z_B - z_A + \left[\frac{f \left(\frac{l}{d} \right)}{2gA^2} + \frac{\sum k}{2gA^2} \right] Q^2 \quad H_s = \frac{p_1}{\rho g} + \frac{V_1^2}{2g} - \frac{p_{vap}}{\rho g}$$

Turbomáquinas

Potência ao veio em máquinas hidráulicas: $P = \rho g Q H \eta$ ou $P = \rho g Q H / \eta$

Coeficientes adimensionais em turbomáquinas:

$$C_P = \frac{P}{\rho N^3 D^5} \quad C_Q = \frac{Q}{N D^3} \quad C_H = \frac{gH}{N^2 D^2} \quad \Omega = \frac{N \sqrt{Q}}{(gH)^{\frac{3}{4}}} \quad \Delta = \frac{D (gH)^{\frac{1}{4}}}{\sqrt{Q}} \quad S_i = \frac{N \sqrt{Q}}{(gH_{Si})^{\frac{3}{4}}}$$

Pontos dinamicamente semelhantes (mesma máquina): $\frac{Q_1}{Q_2} = \frac{N_1}{N_2} = \left(\frac{H_1}{H_2} \right)^{1/2}$

Para bombas e turbinas temos, respetivamente: $S_{i\eta} = 3,0$ e $S_{i\eta} = 4,0$

Diagrama de Cordier:

Bomba / Ventidador

$$H_s = \frac{p_{atm} - p_{vap}}{\rho g} - e_s - z_{asp}$$

Turbina

$$H_s = \frac{p_{atm} - p_v}{\rho g} - e_s$$

