

# Thapar Institute of Engineering & Technology – Patiala

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**THAPAR INSTITUTE**  
OF ENGINEERING & TECHNOLOGY  
(Deemed to be University)

# *Overview of the course*



## Mechanical Engineering Department

Course coordinator  
Dr. Vishal Gupta  
Assistant Professor  
Mechanical Engineering Department

Course Co-coordinator  
Dr. Sachin Singh  
Assistant Professor  
Mechanical Engineering Department

## Electronics and Communication Engineering Department

Course Coordinator  
Dr. Poonam Verma  
Assistant Professor  
Electronics and Communication Engineering Department

# ENGINEERING DESIGN PROJECT-I

## UTA016

### Lecture - 3

## How Structures Fail



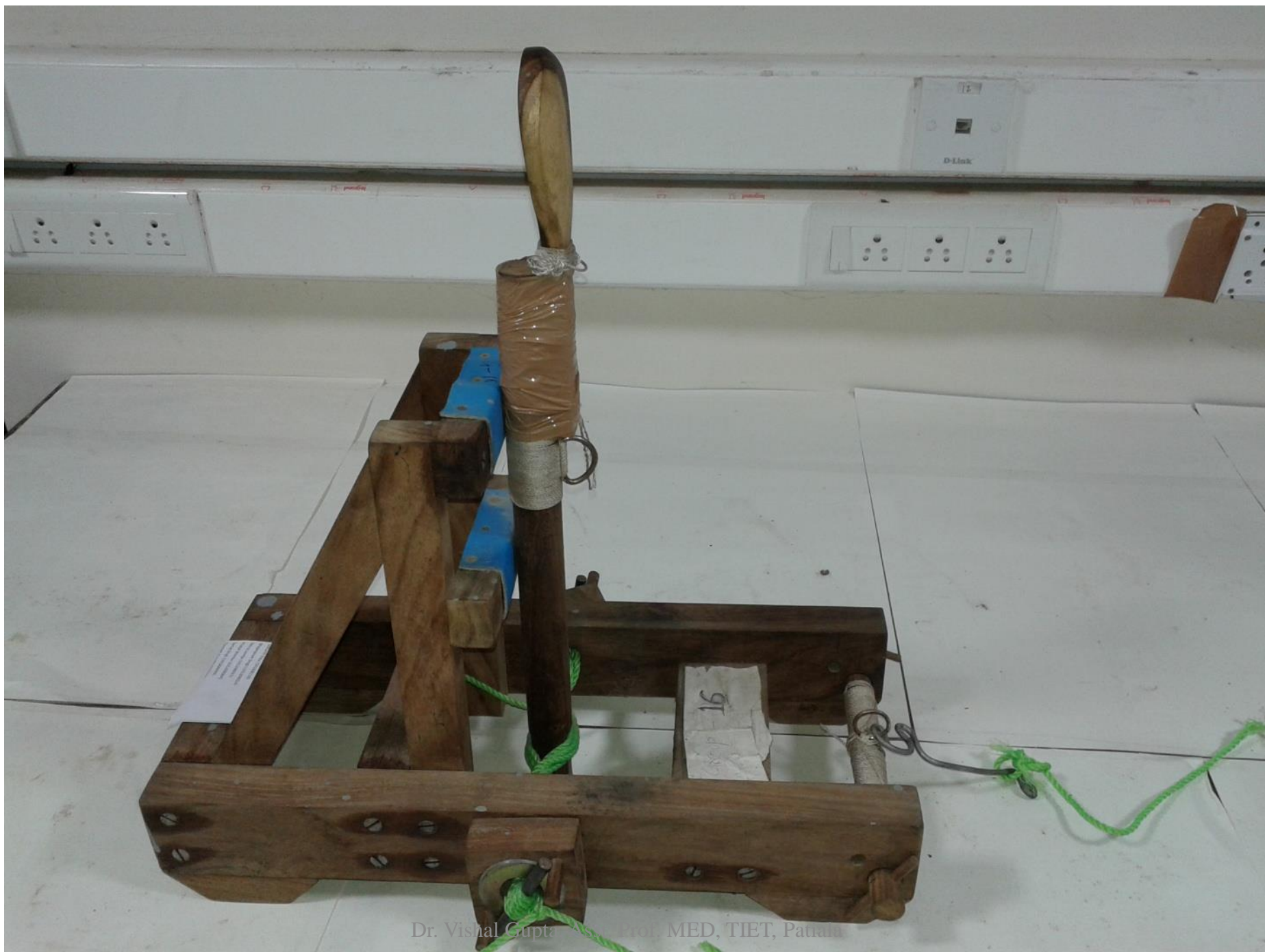
# Instructional objective

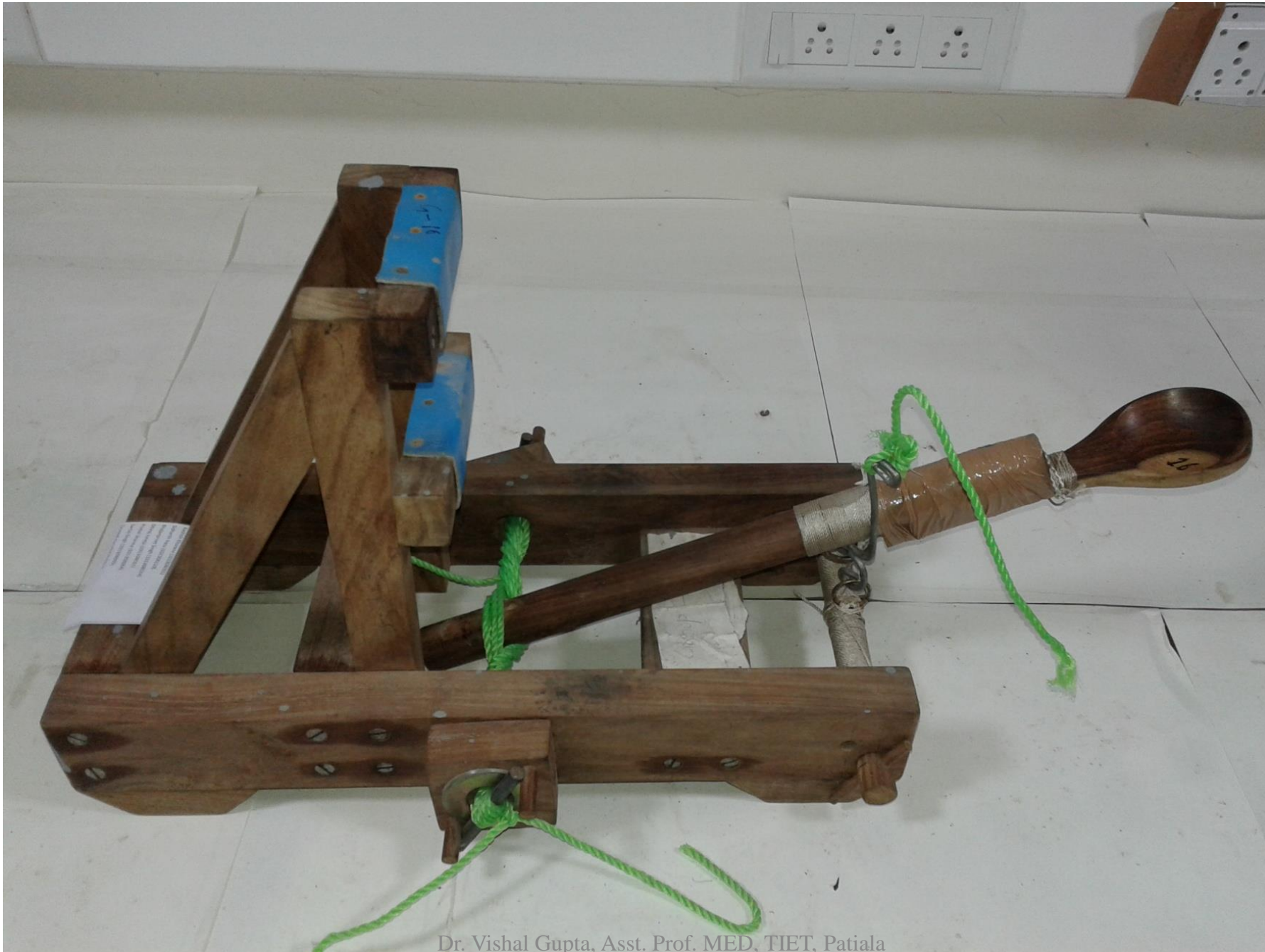


- Failure Modes – Axial members
- Failure Modes-Beams
- Factor of safety
- Failure Modes (TORSION)



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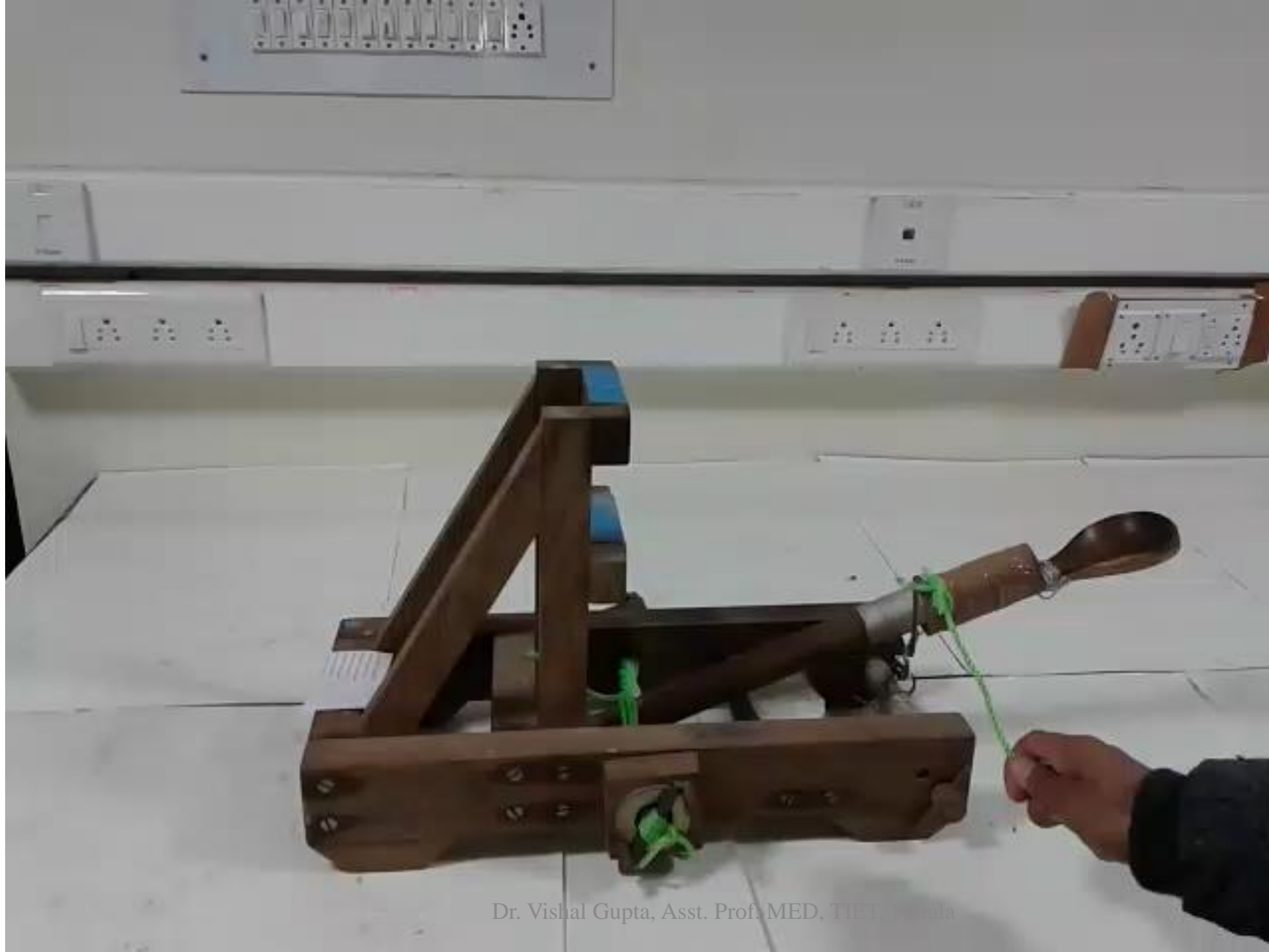




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# ***FIRING A MANGONEL***





# Failure Modes – Axial members

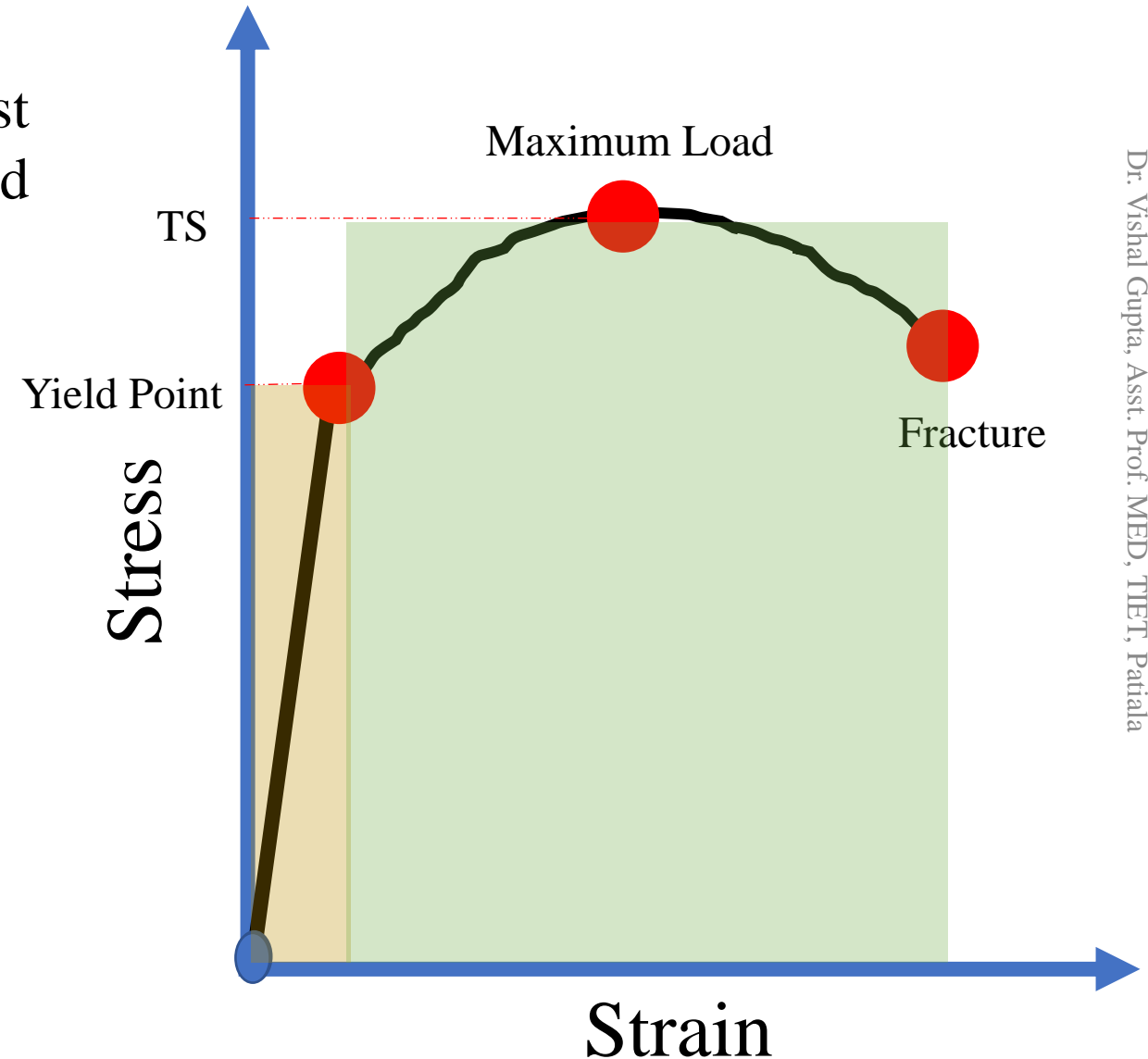
## ENGINEERING STRESS–STRAIN

The engineering stress and strain in a tensile test are defined relative to the original area and length of the test specimen.



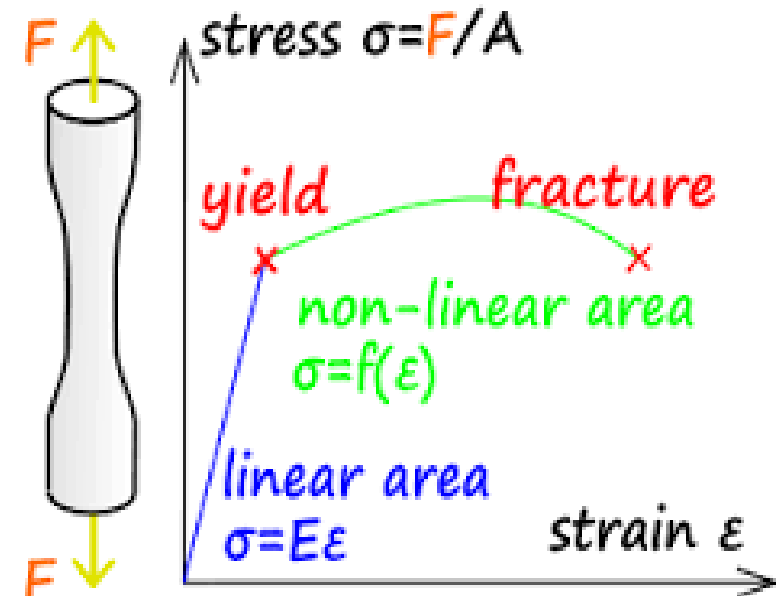
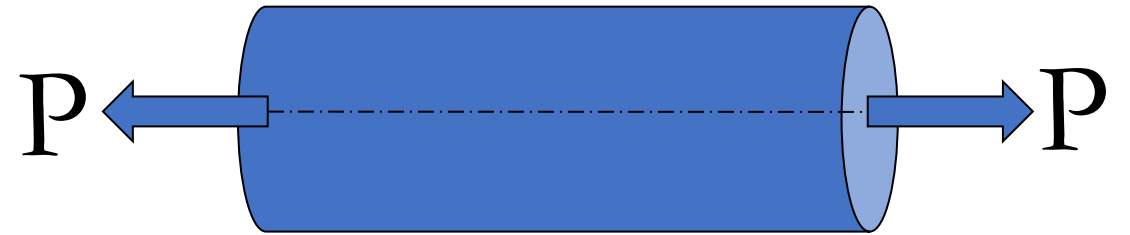
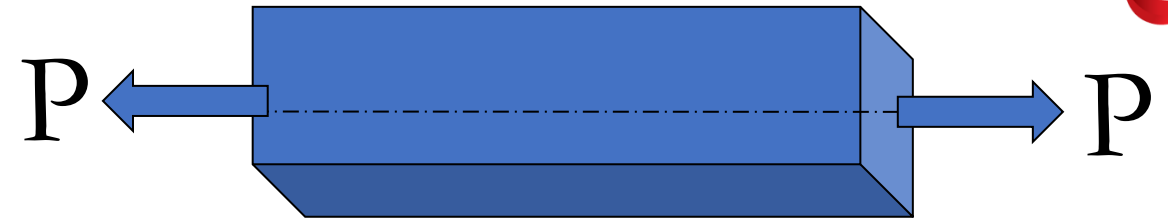
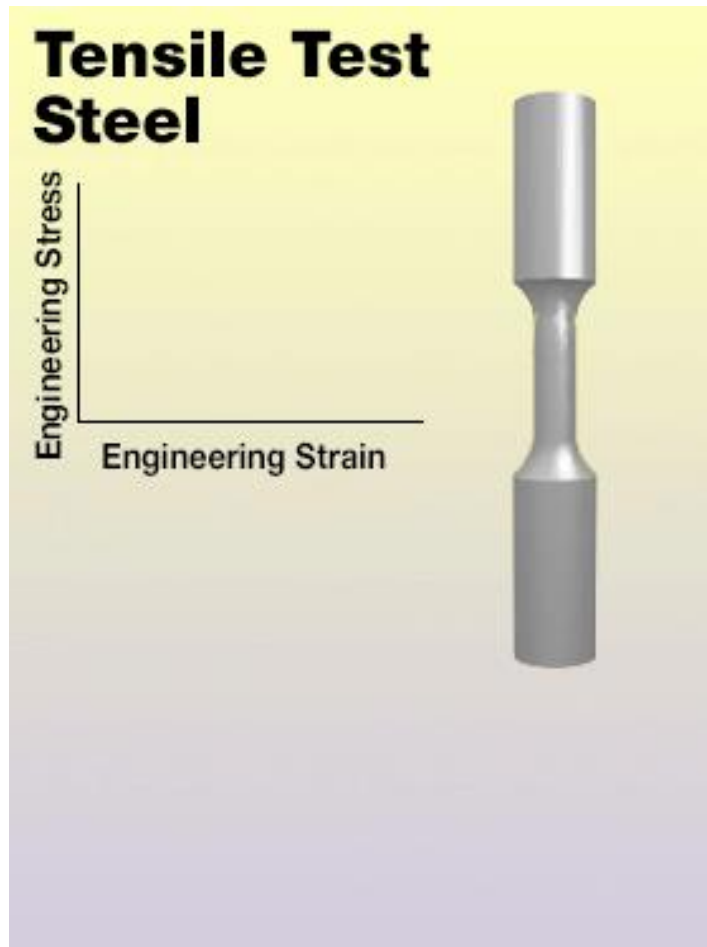
Elastic Region

Plastic Region



# TENSION IN A MEMBER

1. AXIAL STRESS
2. EXTENSION LEADING TO STRAIN



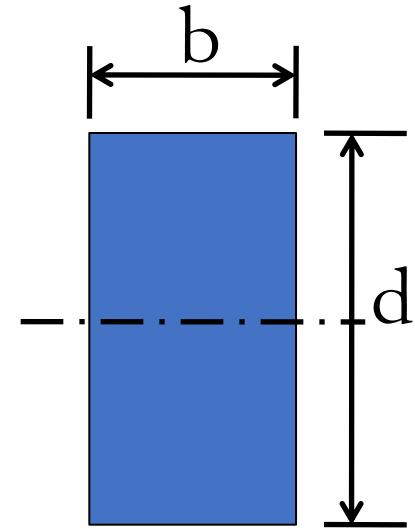
# Failure Modes – Axial members

**STRESS= FORCE /FAILURE AREA (MPa or N/mm<sup>2</sup>)**

## AXIAL STRESS

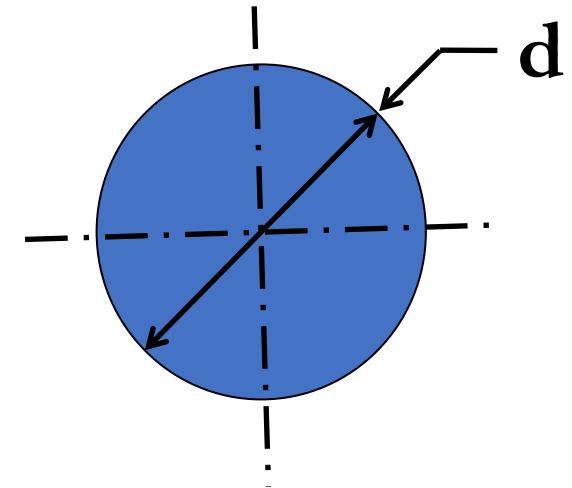
For rectangular cross-section

$$\sigma_a = \frac{P}{b \times d}$$



For circular cross-section

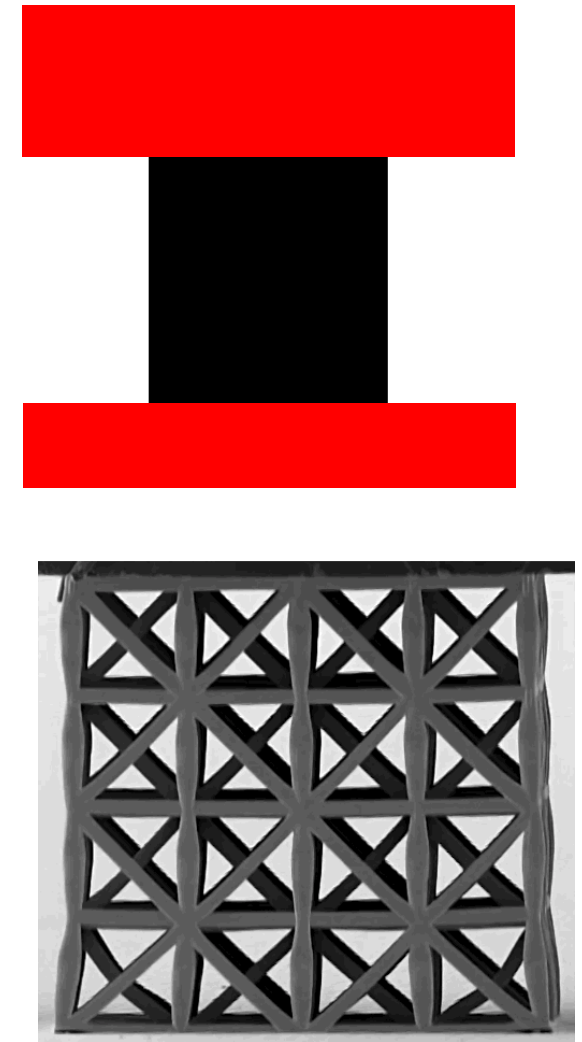
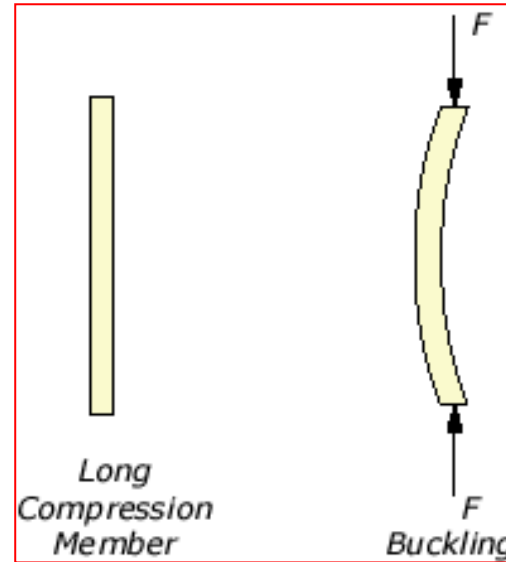
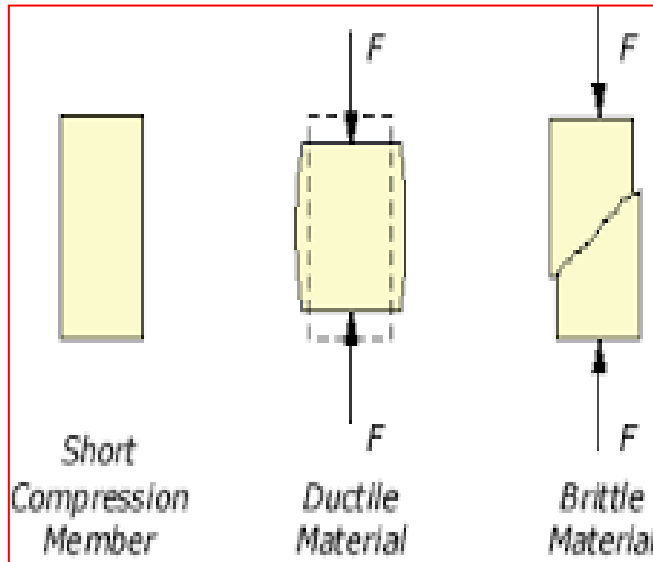
$$\sigma_a = \frac{P}{\frac{\pi}{4} d^2} = \frac{4P}{\pi d^2}$$



# BUCKLING IN COMPRESSION

- ELASTIC
- PLASTIC

$$\text{STRESS} = \text{FORCE} / \text{FAILURE AREA}$$



**SHORT COLUMN:** a column which fails in compression

$$\sigma_a = \frac{P}{b \times d}$$

**LONG COLUMN:** a column which buckles before full compression strength is reached

$$\sigma_{cr} = \frac{P_{cr}}{A}$$

Where:

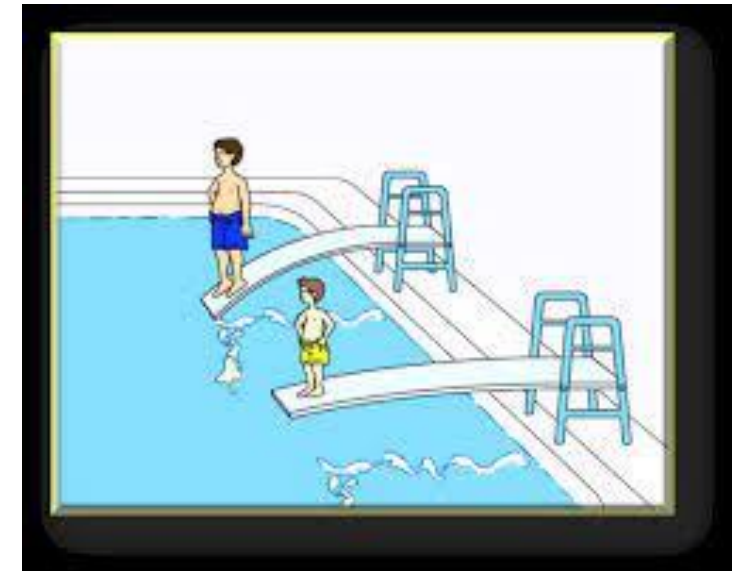
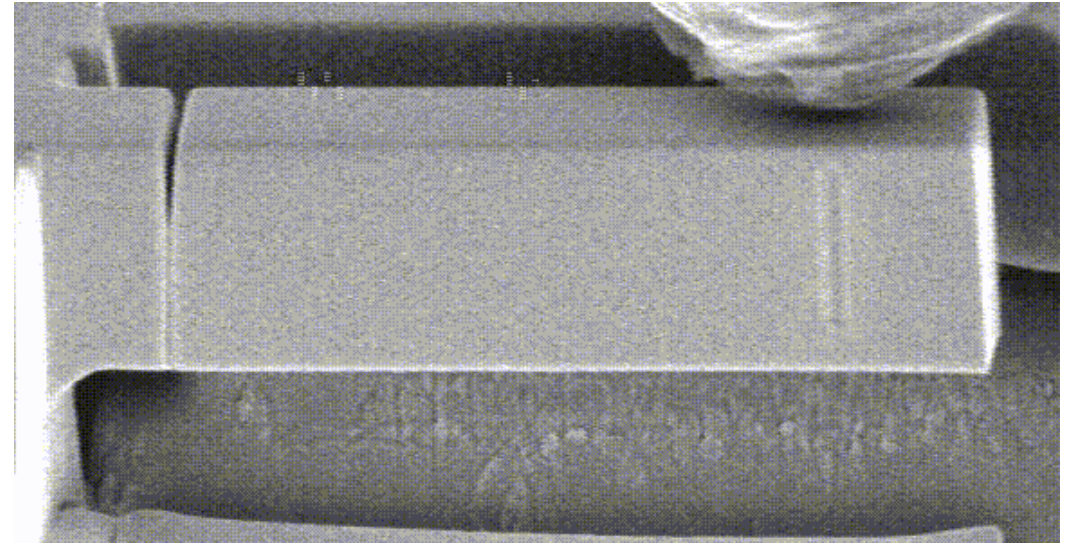
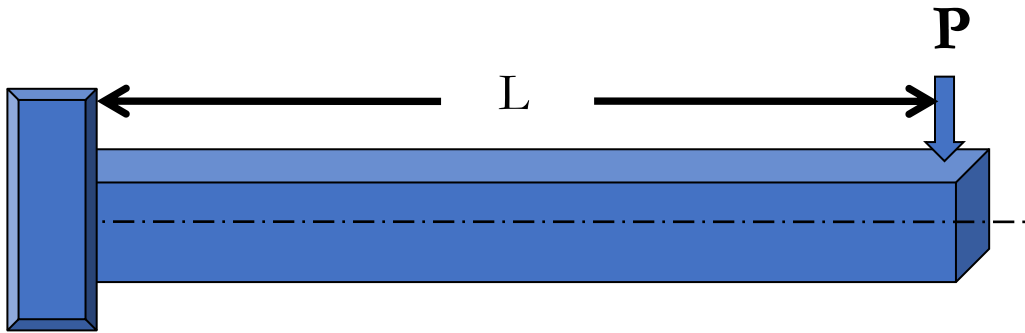
$$P_{cr} = \frac{\pi^2 EI}{L^2}$$



# Failure Modes-Beams

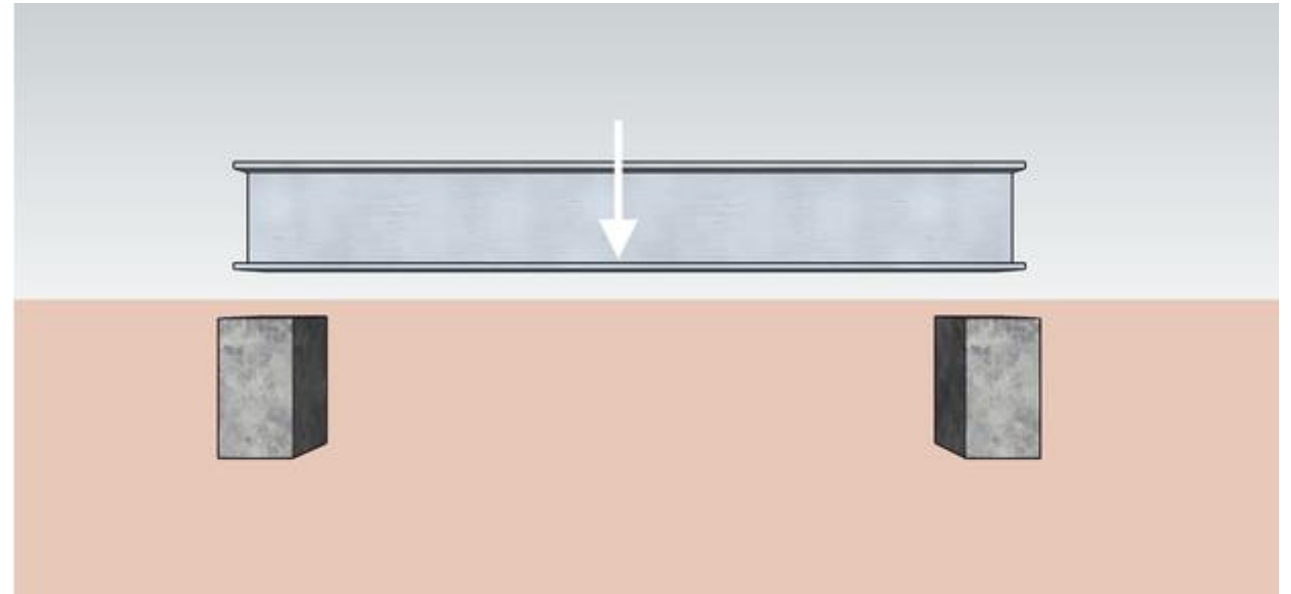
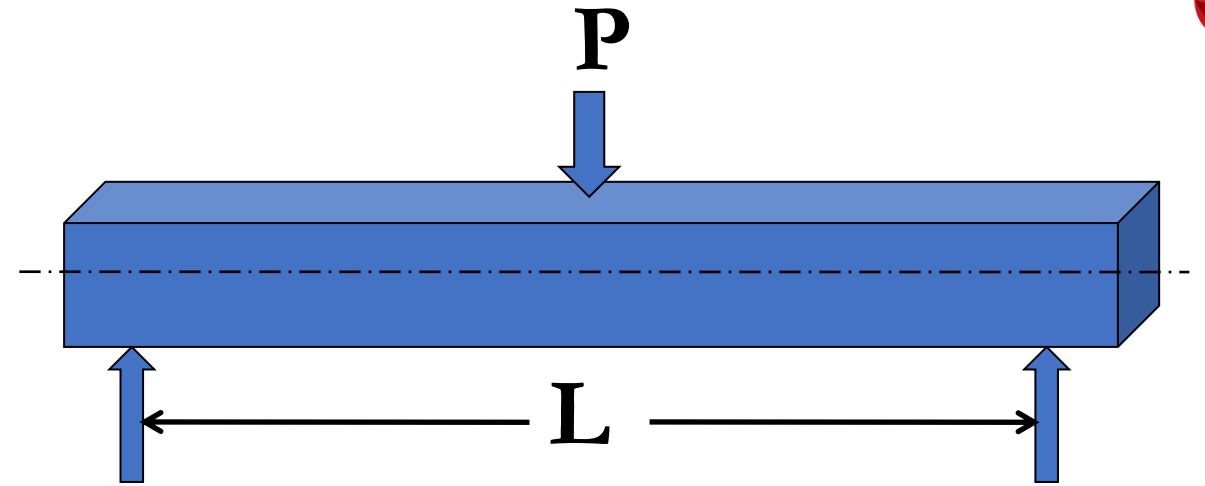
## *TYPES OF BEAMS*

### 1. CANTILEVER BEAM



# ***TYPES OF BEAMS***

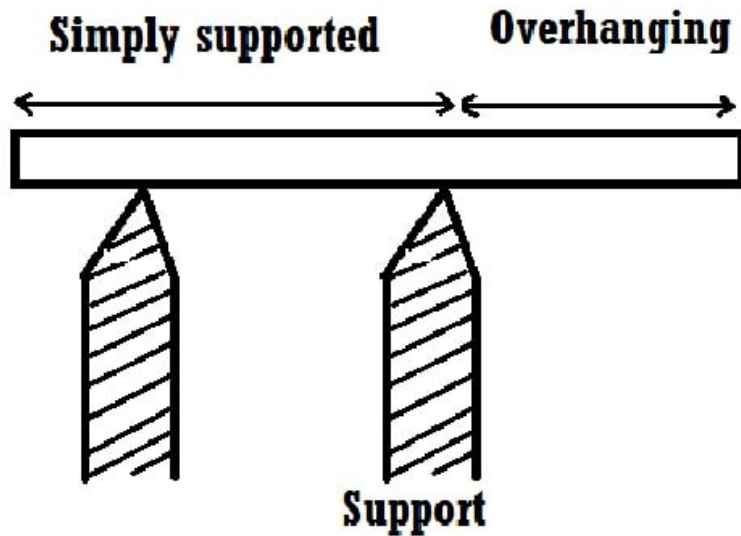
## **2. SIMPLY SUPPORTED BEAM**



# ***TYPES OF BEAMS***

## **3. OVERHANG BEAM**

### **Overhanging Beam**



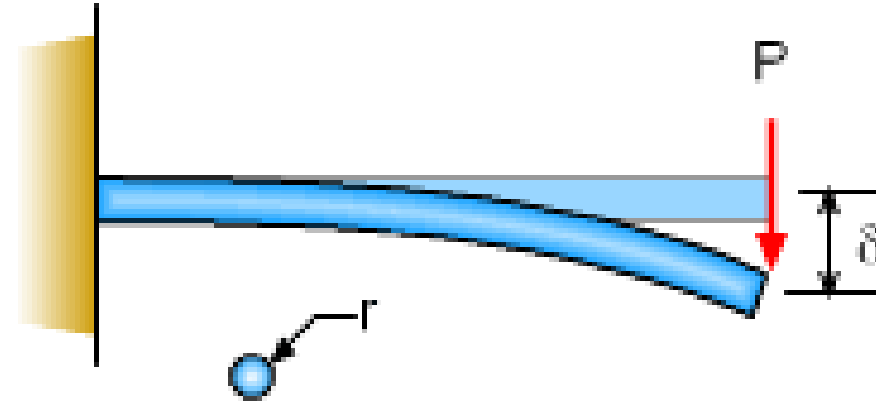
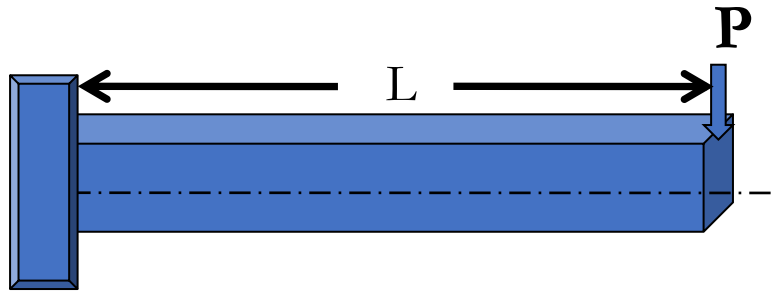


## ***BENDING OF A BEAM***

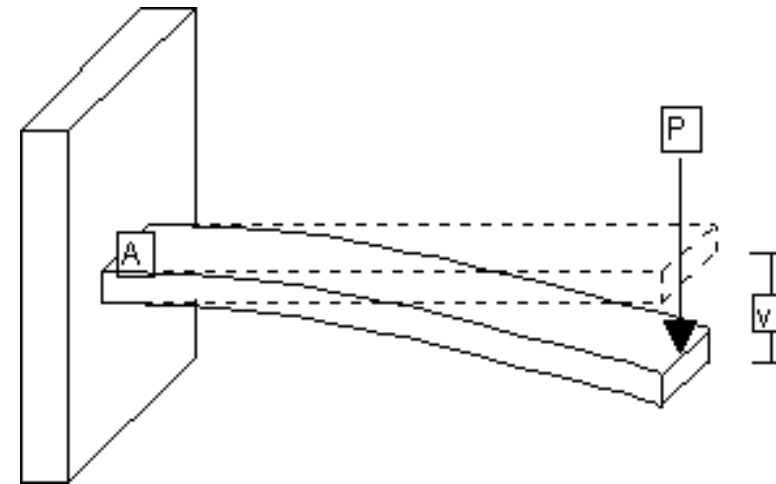
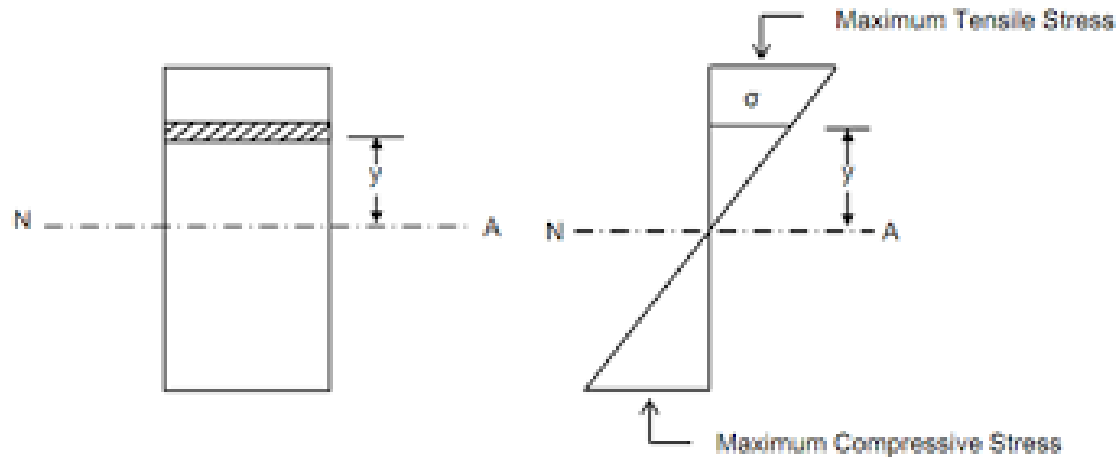
1. BENDING STRESS
2. TENSION AND COMPRESSION
3. SECOND MOMENT OF AREA
4. DEPTH V/S WIDTH
5. SECTION MODULUS



# 1. CANTILEVER BEAM



Beam Cross Section



$$M = P \times L \text{ (Cantilever beam)}$$

## EULER'S FLEXURE FORMULA

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$$\frac{M}{I} = \frac{\sigma_b}{y} = \frac{E}{\rho}$$

# BENDING OF A BEAM

## BENDING STRESS

$$\sigma_b = \frac{M.y}{I}$$

$$M = P \times L \text{ (Cantilever beam)}$$

WHERE:

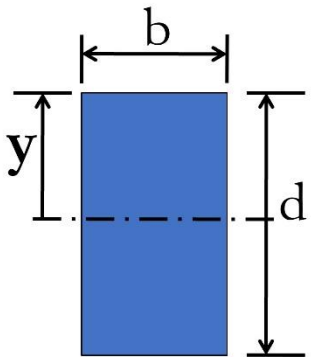
$\sigma_b$  = THE BENDING STRESS,

M= BENDING MOMENT,

y = DISTANCE FROM THE NEUTRAL AXIS

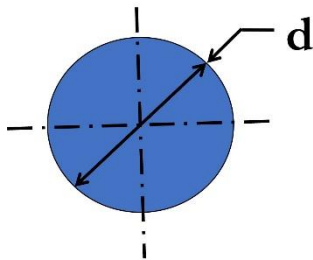
I = SECOND MOMENT OF AREA

## SECOND MOMENT OF AREA



$$y = d/2$$

$$I = \frac{bd^3}{12}$$



$$I = \frac{\pi d^4}{64}$$

## SECTION MODULUS

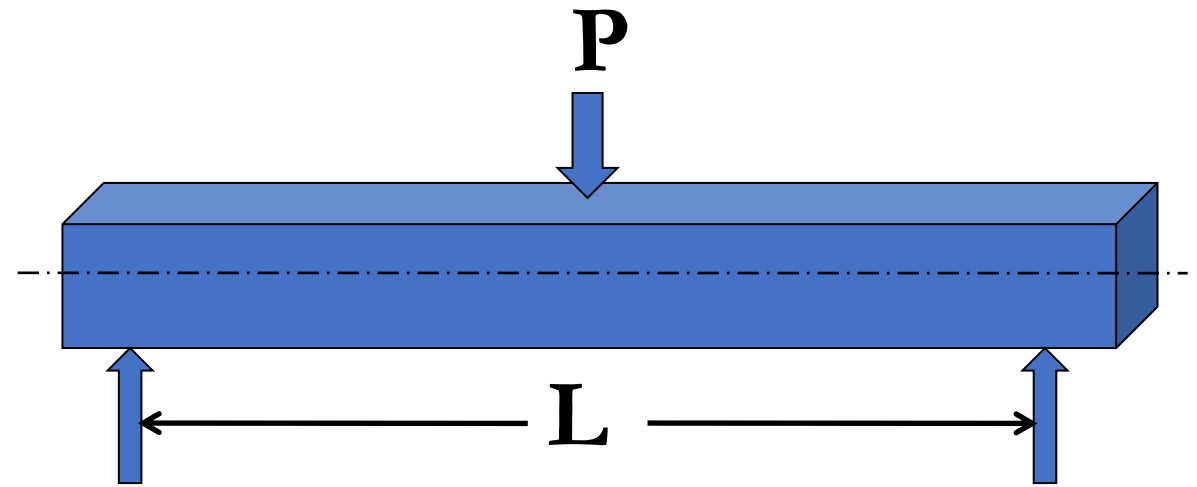
$$Z = \frac{I}{y}$$

## BENDING STRESS

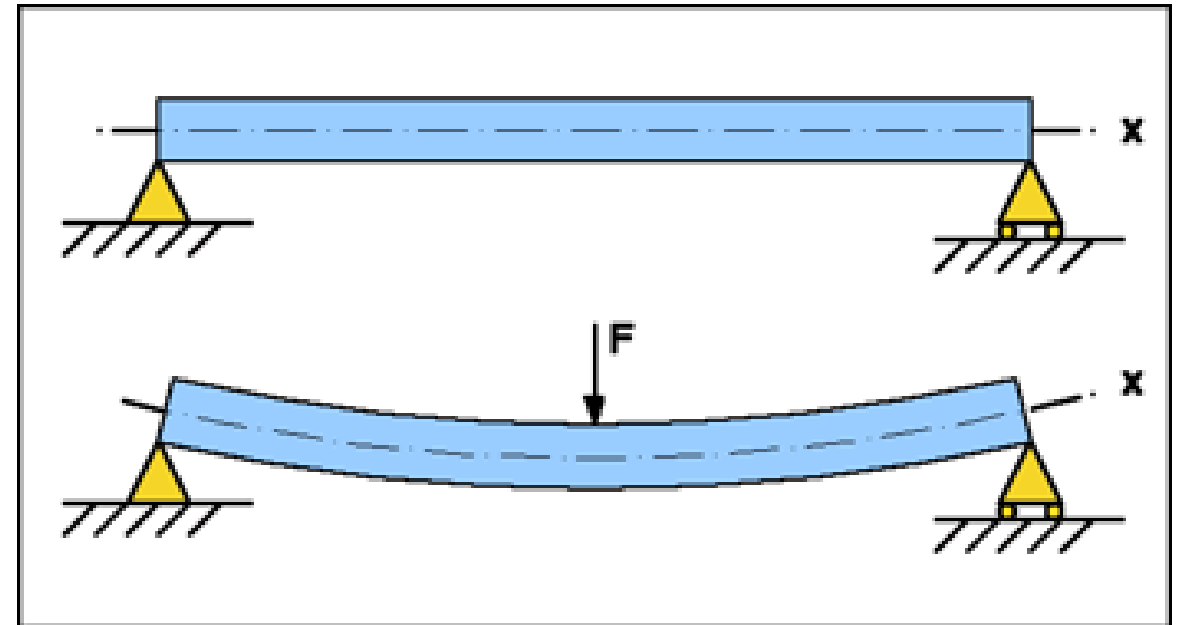
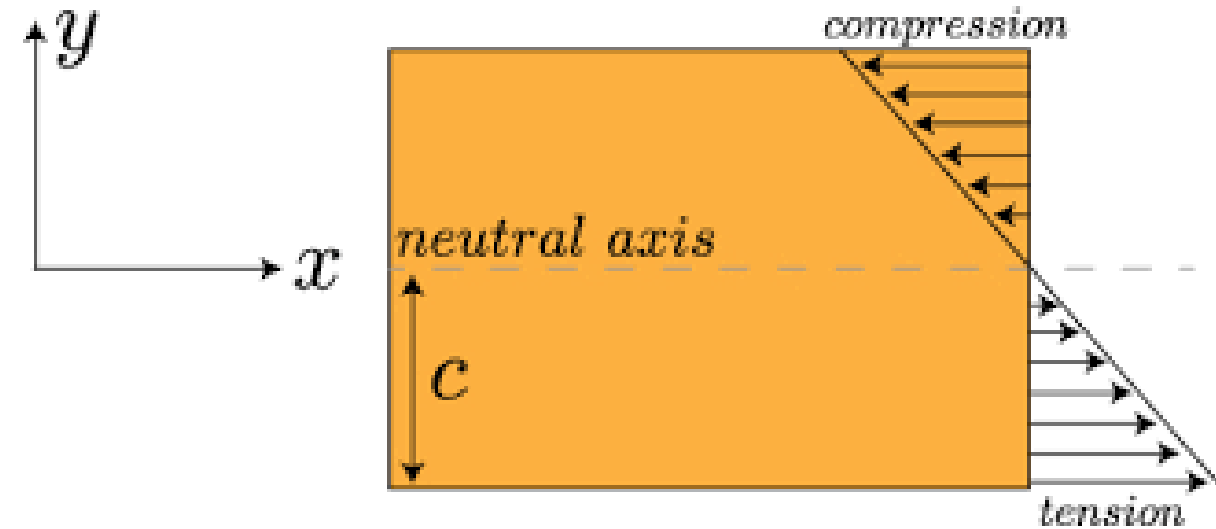
$$\sigma_b = \frac{M}{Z}$$

## 2. SIMPLY SUPPORTED BEAM

SIMPLY SUPPORTED WITH  
LOAD AT THE MIDDLE



$$M = PL/4$$

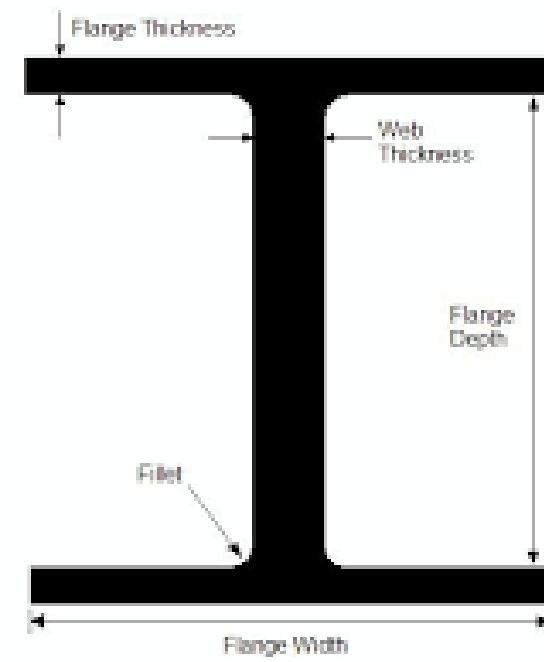




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# ***BEAM CROSS-SECTION***



# Factor of safety

- While designing a component or a structural member, it is necessary to keep sufficient reserve strength in case of an accident.
- This is achieved by taking a suitable factor of safety (FOS)

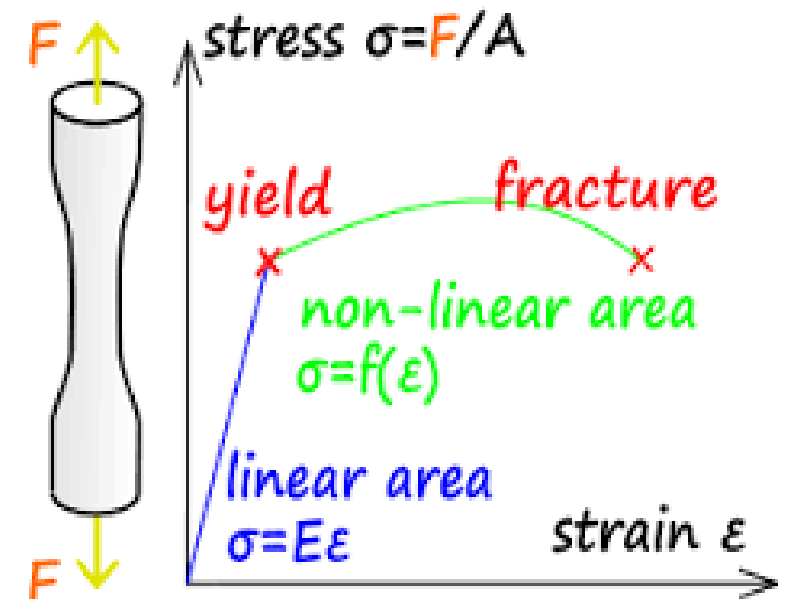
$$\text{FOS} = \frac{\text{FAILURE STRENGTH (STRESS)}}{\text{ACTUAL / ALLOWABLE STRESS}}$$

$$\text{FOS} = \frac{\text{FAILURE STRESS}}{\text{ALLOWABLE STRESS}}$$

$$\text{FOS} = \frac{\text{FAILURE LOAD}}{\text{WORKING LOAD}}$$

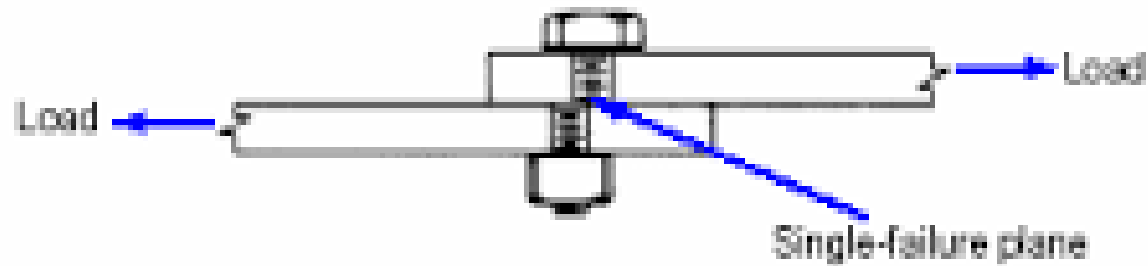
$$\text{FOS} = \frac{S_{ut} / S_{yt}}{\sigma}$$

$$\sigma = \frac{S_{ut}}{\text{FOS}}$$

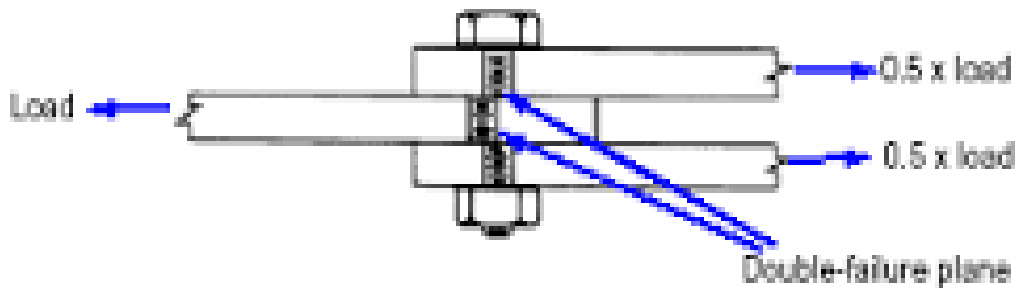


## ***SHEAR FAILURE***

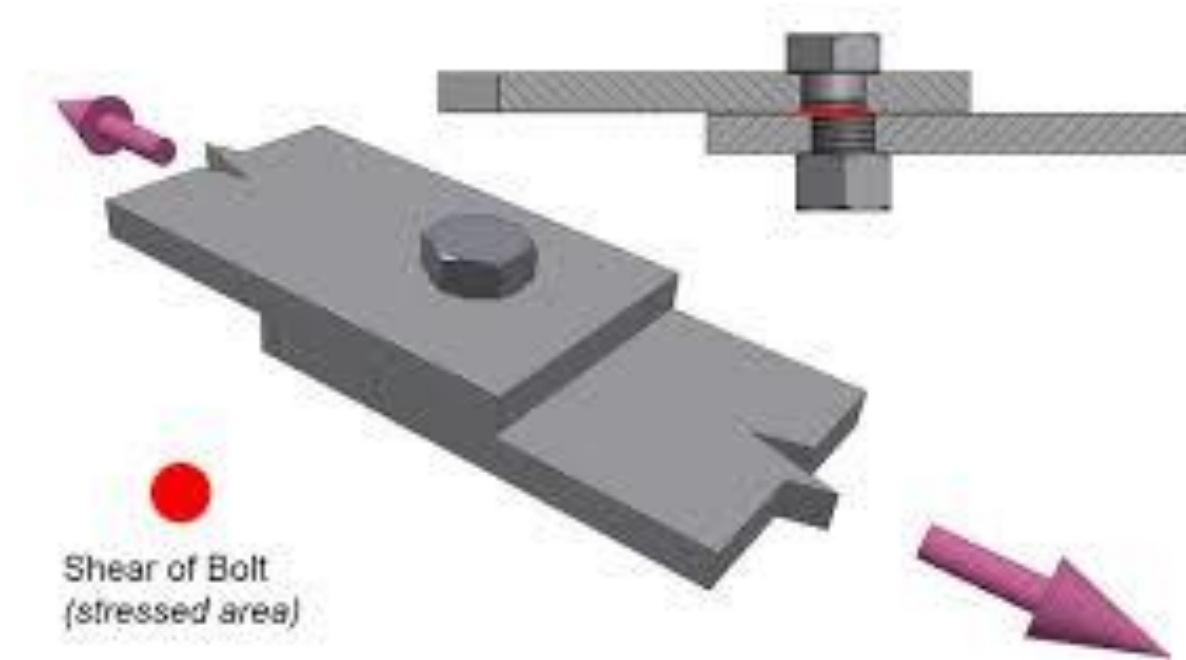
### BOLT FAILURE



A. Single-Shear Failure



B. Double-Shear Failure



**(Single Shear)**

## ***SHEAR STRESS***

$$\tau = \frac{P}{\left(\frac{\pi}{4} d^2\right)}$$

**(Single Shear)**

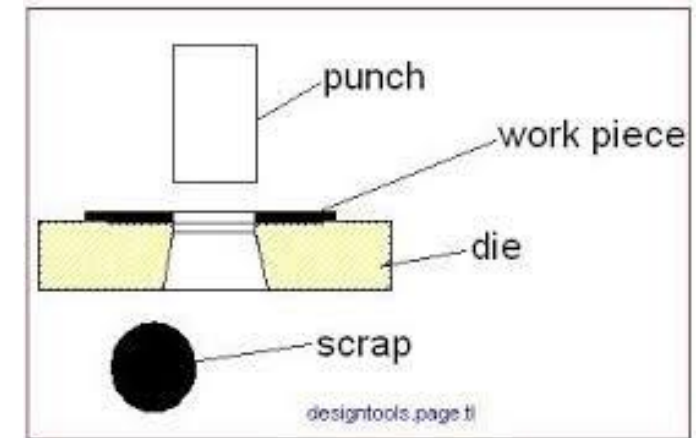
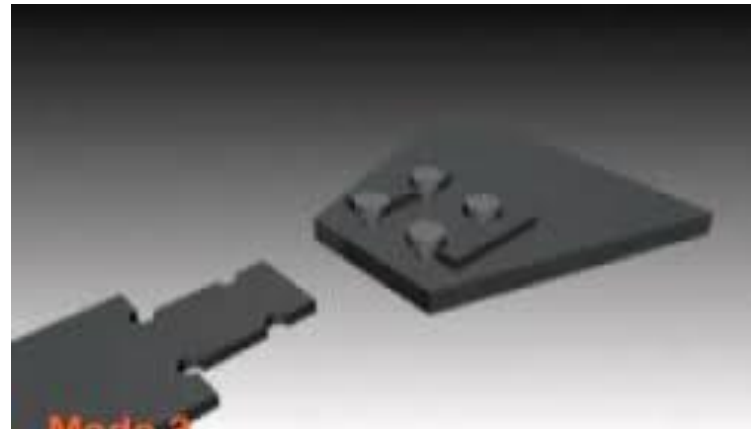
$$\tau = \frac{P}{2 \left(\frac{\pi}{4} d^2\right)}$$

**(Double Shear)**

where 'd' is the diameter of bolt

## ***SHEAR FAILURE***

OTHER EXAMPLES OF SHEAR  
PLATE FAILURE

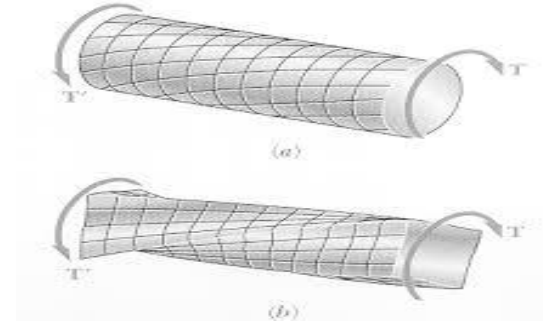
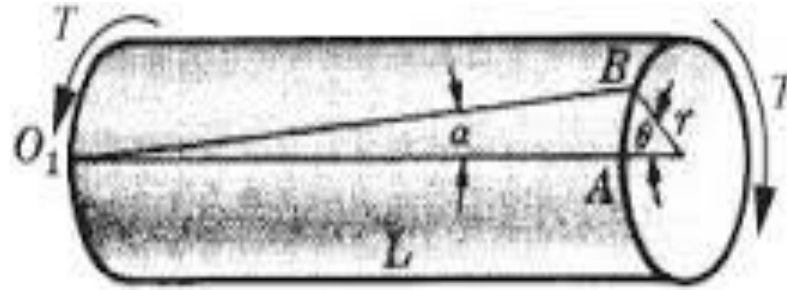




# Failure Modes (TORSION)

## SHAFT

$$\frac{T}{I_p} = \frac{\tau}{r} = \frac{G\phi}{L}$$



WHERE  $T$ ,  $I_p$ ,  $\tau$ ,  $r$ ,  $G$ ,  $\phi$  AND  $L$  ARE, RESPECTIVELY, THE TWISTING MOMENT, POLAR MOMENT OF INERTIA, SHEAR STRESS, RADIUS OF THE SHAFT, SHEAR MODULUS, ANGLE OF TWIST AND LENGTH OF SHAFT

$$\tau = \frac{T \times r}{I_p}$$

where

$$I_p = \frac{\pi}{32} d^4$$

## OTHER MODES

- CYCLIC - FATIGUE
- IMPACT
- CORROSION
- THERMAL MOVEMENT

***Thanks for attending this lecture***



***Save Electricity Save World***