## **ASEN 3112**

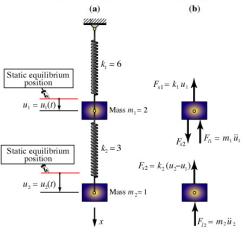
Spring 2020

**Lecture 20** 

April 2, 2020

# Modal Analysis of MDOF Unforced Undamped Systems

#### 2-DOF, Unforced, Undamped Mass-Spring Example System



#### **Matrix Equations of Motion of Example System**

$$m_1 = 2$$
,  $m_2 = 1$ ,  $c_1 = c_2 = 0$ ,  $k_1 = 6$ ,  $k_2 = 3$ ,  $p_1 = p_2 = 0$ 

$$\mathbf{M} = \begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix}, \quad \mathbf{C} = \begin{bmatrix} 0 & 0 \\ 0 & 0 \end{bmatrix}, \quad \mathbf{K} = \begin{bmatrix} 9 & -3 \\ -3 & 3 \end{bmatrix}, \quad \mathbf{p} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$\begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} \ddot{u}_1 \\ \ddot{u}_2 \end{bmatrix} + \begin{bmatrix} 9 & -3 \\ -3 & 3 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

#### **Natural Frequencies of Vibration**

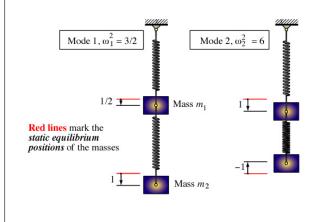
$$\begin{bmatrix} 9 & -3 \\ -3 & 3 \end{bmatrix} \begin{bmatrix} U_1 \\ U_2 \end{bmatrix} = \omega^2 \begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} U_1 \\ U_2 \end{bmatrix}$$

or 
$$\begin{bmatrix} 9 - 2\omega^2 & -3 \\ -3 & 3 - \omega^2 \end{bmatrix} \begin{bmatrix} U_1 \\ U_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$\det\begin{bmatrix} 9 - 2\omega^2 & -3 \\ -3 & 3 - \omega^2 \end{bmatrix} = 18 - 15\omega^2 + 2\omega^4 = (3 - 2\omega^2)(6 - \omega^2) = 0$$

$$\omega_1^2 = \frac{3}{2} = 1.5 \qquad \omega_2^2 = 6$$

#### **Vibration Mode Shape Pictures**



#### Mode Shape Mass Orthonormalization

$$M_{1} = \phi_{1}^{T} \mathbf{M} \phi_{1} = \begin{bmatrix} 1/2 & 1 \end{bmatrix} \begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} 1/2 \\ 1 \end{bmatrix} = 3/2$$

$$M_{2} = \phi_{2}^{T} \mathbf{M} \phi_{2} = \begin{bmatrix} 1 & -1 \end{bmatrix} \begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} 1 \\ -1 \end{bmatrix} = 3$$

$$K_{1} = \phi_{1}^{T} \mathbf{K} \phi_{1} = \begin{bmatrix} 1/2 & 1 \end{bmatrix} \begin{bmatrix} 9 & -3 \\ -3 & 3 \end{bmatrix} \begin{bmatrix} 1/2 \\ 1 \end{bmatrix} = 9/4$$

$$K_{2} = \phi_{2}^{T} \mathbf{K} \phi_{2} = \begin{bmatrix} 1 & -1 \end{bmatrix} \begin{bmatrix} 9 & -3 \\ -3 & 3 \end{bmatrix} \begin{bmatrix} 1 \\ -1 \end{bmatrix} = 18$$

$$\omega_{1}^{2} = \frac{K_{1}}{M_{1}} = \frac{9/4}{3/2} = 3/2 \qquad \omega_{2}^{2} = \frac{K_{2}}{M_{2}} = \frac{18}{3} = 6$$

To orthonormalize, divide  $\phi_i$  by the square root of  $M_i$  (i=1,2):

$$\phi_1 = \sqrt{\frac{2}{3}} \begin{bmatrix} 1/2 \\ 1 \end{bmatrix} = \begin{bmatrix} 0.4088 \\ 0.8165 \end{bmatrix}, \quad \phi_2 = \sqrt{\frac{1}{3}} \begin{bmatrix} 1 \\ -1 \end{bmatrix} = \begin{bmatrix} 0.5773 \\ -0.5773 \end{bmatrix}$$

#### Mode Shape Mass Orthonormalization

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$$M_{2} = \phi_{2}^{T} \mathbf{M} \phi_{2} = \begin{bmatrix} 1 & -1 \end{bmatrix} \begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} 1 \\ -1 \end{bmatrix} = 3$$

$$K_{1} = \phi_{1}^{T} \mathbf{K} \phi_{1} = \begin{bmatrix} 1/2 & 1 \end{bmatrix} \begin{bmatrix} 9 & -3 \\ -3 & 3 \end{bmatrix} \begin{bmatrix} 1/2 \\ 1 \end{bmatrix} = 9/4$$

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#### **Superposition Through The Modal Matrix**

$$\mathbf{u} = \begin{bmatrix} u_1(t) \\ u_2(t) \end{bmatrix} \stackrel{\text{def}}{=} \phi_1 \, \eta_1(t) + \phi_2 \, \eta_2(t) = \begin{bmatrix} \phi_1 & \phi_2 \end{bmatrix} \begin{bmatrix} \eta_1(t) \\ \eta_2(t) \end{bmatrix} = \mathbf{\Phi} \, \boldsymbol{\eta}$$

$$\Phi = [\phi_1 \quad \phi_2] = \begin{bmatrix} \frac{1}{\sqrt{6}} & \frac{1}{\sqrt{3}} \\ \frac{2}{\sqrt{6}} & -\frac{1}{\sqrt{3}} \end{bmatrix} = \begin{bmatrix} 0.4082 & 0.5773 \\ 0.8165 & -0.5773 \end{bmatrix}$$

$$\mathbf{\Phi}^T \mathbf{M} \mathbf{\Phi} = \mathbf{M}_g = \mathbf{I} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

$$\mathbf{\Phi}^T \mathbf{K} \mathbf{\Phi} = \mathbf{K}_g = \mathbf{diag}[\omega_i^2] = \begin{bmatrix} \omega_1^2 & 0 \\ 0 & \omega_2^2 \end{bmatrix} = \begin{bmatrix} 3/2 & 0 \\ 0 & 6 \end{bmatrix}$$

#### **Modal Equations of Motion**

$$\mathbf{M}\ddot{\mathbf{u}} + \mathbf{K}\mathbf{u} = \mathbf{0} \implies \Phi^{T}[\mathbf{M}\ddot{\mathbf{u}} + \mathbf{K}\mathbf{u}] = \Phi^{T}\mathbf{0} = \mathbf{0}$$

$$\mathbf{u}(t) = \Phi \eta(t) \implies \dot{\mathbf{u}}(t) = \Phi \dot{\eta}(t), \quad \ddot{\mathbf{u}}(t) = \Phi \ddot{\eta}(t)$$

$$\downarrow \qquad \qquad \downarrow$$

$$\Phi^{T}\mathbf{M}\Phi \ddot{\eta} + \Phi^{T}\mathbf{K}\Phi \eta = \Phi^{T}\mathbf{0} = \mathbf{0}$$

$$\ddot{\eta} + \mathbf{K}_{g}\eta = \mathbf{0}$$

$$\begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} \ddot{\eta}_1(t) \\ \ddot{\eta}_2(t) \end{bmatrix} + \begin{bmatrix} 3/2 & 0 \\ 0 & 6 \end{bmatrix} \begin{bmatrix} \eta_1(t) \\ \eta_2(t) \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$\ddot{\eta}_1(t) + (3/2) \, \eta_1(t) = 0, \quad \ddot{\eta}_2(t) + 6 \, \eta_2(t) = 0$$

#### **Modal Initial Conditions**

$$\mathbf{u}(0) = \mathbf{u}_0 \qquad \dot{\mathbf{u}}(0) = \mathbf{v}_0$$

$$\mathbf{u}(t) = \mathbf{\Phi} \, \boldsymbol{\eta}(t) \quad \Rightarrow \quad \mathbf{u}_0 = \mathbf{u}(0) = \mathbf{\Phi} \, \boldsymbol{\eta}(0)$$

$$\dot{\mathbf{u}}(t) = \mathbf{\Phi} \, \dot{\boldsymbol{\eta}}(t) \quad \Rightarrow \quad \mathbf{v}_0 = \dot{\mathbf{u}}(0) = \mathbf{\Phi} \, \dot{\boldsymbol{\eta}}(0)$$

$$\eta_0 = \eta(0) = \Phi^{-1} \mathbf{u}_0 \quad \dot{\eta}_0 = \dot{\eta}(0) = \Phi^{-1} \mathbf{v}_0$$

To avoid inverting  $\Phi$  use this linear algebra trick: postmultiply both sides of  $\Phi^T \mathbf{M} \Phi = \mathbf{I}$  by  $\Phi^{-1}$  to get  $\Phi^{-1} = \Phi^T \mathbf{M}$ , and replace

$$\eta_0 = \mathbf{\Phi}^T \mathbf{M} \, \mathbf{u}_0 \qquad \dot{\boldsymbol{\eta}}_0 = \mathbf{\Phi}^T \mathbf{M} \, \mathbf{v}_0$$

#### Sample IC: Unit Velocity At Mass 1, Else Zero

$$\mathbf{u}_0 = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \qquad \mathbf{v}_0 = \begin{bmatrix} 1 \\ 0 \end{bmatrix}$$

$$\begin{bmatrix}
\boldsymbol{\eta}_0 = \begin{bmatrix} \eta_{10} \\ \eta_{20} \end{bmatrix} = \boldsymbol{\Phi}^T \mathbf{M} \mathbf{u}_0 = \begin{bmatrix} \frac{1}{\sqrt{6}} & \frac{2}{\sqrt{6}} \\ \frac{1}{\sqrt{3}} & -\frac{1}{\sqrt{3}} \end{bmatrix} \begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} 0 \\ 0 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \\
\dot{\boldsymbol{\eta}}_0 = \begin{bmatrix} \dot{\eta}_{10} \\ \dot{\eta}_{20} \end{bmatrix} = \boldsymbol{\Phi}^T \mathbf{M} \mathbf{v}_0 = \begin{bmatrix} \frac{1}{\sqrt{6}} & \frac{2}{\sqrt{6}} \\ \frac{1}{\sqrt{3}} & -\frac{1}{\sqrt{3}} \end{bmatrix} \begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} 1 \\ 0 \end{bmatrix} = \begin{bmatrix} \frac{2}{\sqrt{6}} \\ \frac{2}{\sqrt{3}} \end{bmatrix}$$

### Solve Modal Equations And Combine Via Modal Matrix

$$\eta_1(t) = \frac{\dot{\eta}_{10}}{\omega_1} \sin \omega_1 t = \frac{2}{3\sqrt{3}} \sin(\sqrt{6}t) = \frac{2}{3} \sin(\sqrt{3/2}t) = 0.6667 \sin(1.2247t),$$

$$\eta_2(t) = \frac{\dot{\eta}_{20}}{\omega_2} \sin \omega_2 t = \frac{2/\sqrt{3}}{\sqrt{6}} \sin(\sqrt{6}t) = \frac{2}{3\sqrt{2}} \sin(\sqrt{6}t) = 0.4714 \sin(2.4495t).$$

$$\begin{bmatrix} u_1(t) \\ u_2(t) \end{bmatrix} = \begin{bmatrix} \frac{1}{\sqrt{6}} & \frac{1}{\sqrt{3}} \\ \frac{2}{\sqrt{6}} & -\frac{1}{\sqrt{3}} \end{bmatrix} \begin{bmatrix} \frac{2}{3} \sin(\sqrt{3/2}t) \\ \frac{2}{3\sqrt{2}} \sin(\sqrt{6}t) \end{bmatrix} = \begin{bmatrix} \frac{1}{3}\sqrt{\frac{2}{3}} \left( \sin(\sqrt{3/2}t) + \sin(\sqrt{6}t) \right) \\ \frac{1}{3}\sqrt{\frac{2}{3}} \left( 2\sin(\sqrt{3/2}t) - \sin(\sqrt{6}t) \right) \end{bmatrix}$$
$$= \begin{bmatrix} 0.2722 \left( \sin(1.2247t) + \sin(2.4495t) \right) \\ 0.2722 \left( 2\sin(1.2247t) - \sin(2.4495t) \right) \end{bmatrix}$$

$$\begin{bmatrix} \dot{u}_1(t) \\ \dot{u}_2(t) \end{bmatrix} = \begin{bmatrix} \frac{1}{3} \left( \cos(\sqrt{3/2}t) + 2\cos(\sqrt{6}t) \right) \\ \frac{2}{3} \left( \cos(\sqrt{3/2}t) - \cos(\sqrt{6}t) \right) \end{bmatrix} = \begin{bmatrix} 0.3333 \left( \cos(1.2247t) + 2\cos(2.4495t) \right) \\ 0.6667 \left( \cos(1.2247t) - \cos(2.4495t) \right) \end{bmatrix}$$

#### Response to Unit Initial Velocity on Mass 1

