

ASEN 3112

Spring 2020

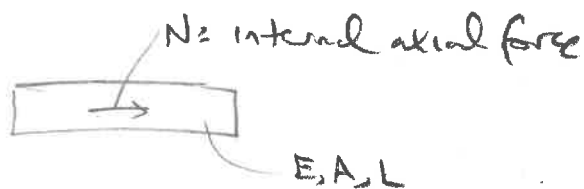
Lecture 12

Whiteboard

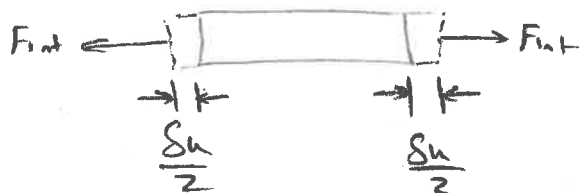
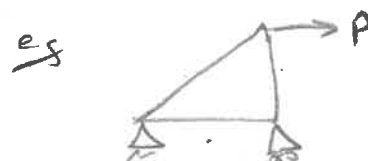
February 25, 2020

L12
1

Principle of Virtual Work (PVW) on Rod (Bar)



A bar can also serve as part of a truss



Virtual strain-displacement relation

$$\delta \epsilon_{xx} = \frac{\delta u}{L} \Rightarrow \delta u = L \delta \epsilon_{xx}$$

Real stress-strain relation

$$\sigma_{xx} = E \epsilon_{xx} \quad \left(\sigma_{xx} = \frac{N}{A} \right)$$

PVW: $\delta W_{ec} = \delta W_{ie}$

$$\delta W_{ie} = N \delta u$$

$$= \sigma_{xx} A \delta \epsilon_{xx} L$$

$$= EA \epsilon_{xx} \delta \epsilon_{xx} L \quad \left(\epsilon_{xx} = \frac{\Delta L}{L} \right)$$

$$= EA \frac{\Delta L}{L} \delta \left(\frac{\Delta L}{L} \right) L$$

$$\delta W_{ie} = \frac{EA}{L} \Delta L \delta(\Delta L)$$

corresponds to $\frac{1}{2}(\Delta L)^2$

Recall:

$$U_{rod} = \frac{1}{2} \frac{EA}{L} (\Delta L)^2$$

PVW on a Beam

$$\delta W_{ie} = \int_L EI \kappa \delta \kappa dx$$

corresponds to κ^2

Recall

$$U_{beam} = \frac{1}{2} \int EI \kappa^2 dx$$

PVW on shaft

$$\delta W_{ie} = \int_L GL \left(\frac{\partial \theta}{\partial x} \right) \delta \left(\frac{\partial \theta}{\partial x} \right) dx$$

Recall

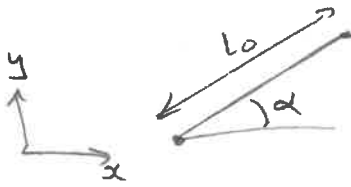
$$U_{shaft} = \frac{1}{2} \int GL \theta'^2 dx$$

L12
2

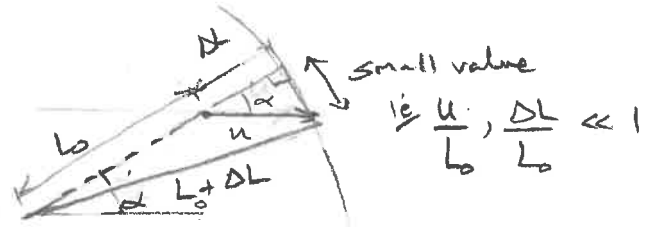
Example: bar

Truss: Elongation of a rod (bar) due to tip displacement

Undeformed



Deformed

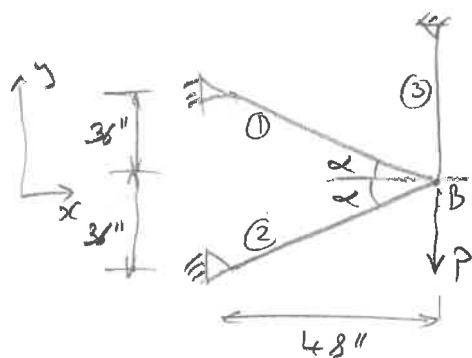


$$\Delta L = u \cos \alpha$$

L12
3

Example

Three-bar truss



① E_1, A_1, L

② E_2, A_2, L

③ E_3, A_3, L_3

$$L_1 = L_2 = 60" = L$$

$$\cos \alpha = \frac{48}{60} = \frac{4}{5}$$

$$\sin \alpha = \frac{36}{60} = \frac{3}{5}$$

Find at B: u_B, v_B

IPVW

$$\delta W_e = \delta W_{ie}$$

Unknown: u_B, v_B (real)

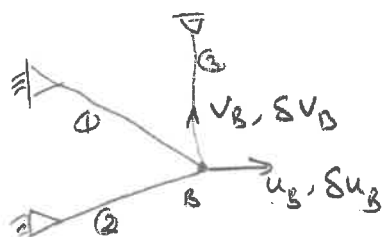
$\delta u_B, \delta v_B$ (virtual)

$$\delta W_e^u = 0 \delta u_B \rightarrow \delta W_{ie}^u ?$$

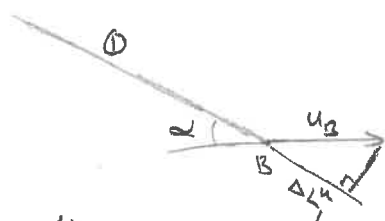
$\underbrace{\hspace{1cm}}_{\text{zero force in x-direction}}$

$$\delta W_e^v = -P \delta v_B \rightarrow \delta W_{ie}^v ?$$

Recall: $\delta W_{ie_{rod}} = \frac{EA}{L} \Delta L (\delta \Delta L)$

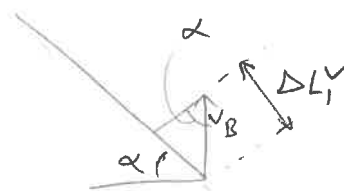


$$\delta \Delta L^u \quad \delta \Delta L^v$$



$$\Delta L_1^u = \cos \alpha u_B = \frac{4}{5} u_B$$

$$(\delta \Delta L_1^u = \frac{4}{5} \delta u_B)$$



$$\Delta L_1^v = -\sin \alpha v_B = -\frac{3}{5} v_B$$

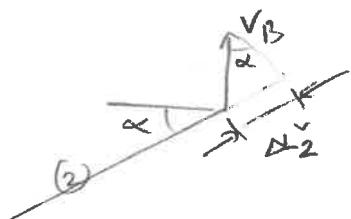
$$(\delta \Delta L_1^v = -\frac{3}{5} \delta v_B)$$

$$\rightarrow \Delta L_1 = \Delta L_1^u + \Delta L_1^v = \frac{4}{5} u_B - \frac{3}{5} v_B$$

L12

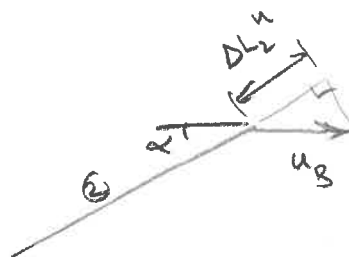
4

$$E_1 = E_2 = E$$



$$\Delta L_2^v = \sin \alpha v_B = \frac{3}{5} v_B$$

$$(\delta \Delta L_2^v = \frac{3}{5} \delta v_B)$$



$$\Delta L_2^u = \cos \alpha u_B = \frac{4}{5} u_B$$

$$(\delta \Delta L_2^u = \frac{4}{5} \delta u_B)$$

$$\rightarrow \Delta L_2 = \Delta L_2^u + \Delta L_2^v = \frac{4}{5} u_B + \frac{3}{5} v_B$$

$$\delta W_{ie}^{rod} = \frac{EA}{L} \Delta L (\delta \Delta L)$$

$$[u:] \quad \delta \Delta L_1^u = \frac{4}{5} \delta u_B, \quad \delta \Delta L_2^u = \frac{4}{5} \delta u_B$$

(note: no contribution for member 3
because $\Delta L_3^u = 0 \Rightarrow \delta \Delta L_3^u = 0$)

$$\delta W_{ie}^{rod} = \frac{EA_1}{L} \underbrace{\left(\frac{4}{5} u_B - \frac{3}{5} v_B \right)}_{\Delta L_1} \underbrace{\left(\frac{4}{5} \delta u_B \right)}_{\delta \Delta L_1^u}$$

$$+ \frac{EA_2}{L} \underbrace{\left(\frac{4}{5} u_B + \frac{3}{5} v_B \right)}_{\Delta L_2} \underbrace{\left(\frac{4}{5} \delta u_B \right)}_{\delta \Delta L_2^u}$$

$$\delta W_{ie}^{rod} = \frac{4E}{25L} \left[4(A_1 + A_2)u_B + 3(A_2 - A_1)v_B \right] \delta u_B = 0$$

L12
5

V:

$$\delta DL_1^v = -\frac{3}{5} v_B, \quad \delta DL_2^v = \frac{3}{5} v_B$$

$$\delta W_{ic}^v = \frac{EA_1}{L} \underbrace{\left(\frac{4}{5} u_B - \frac{3}{5} v_B \right)}_{\Delta L_1} \underbrace{\left(-\frac{3}{5} \delta v_B \right)}_{\delta DL_1^v}$$

$$+ \frac{EA_2}{L} \underbrace{\left(\frac{4}{5} u_B + \frac{3}{5} v_B \right)}_{\Delta L_2} \underbrace{\left(\frac{3}{5} \delta v_B \right)}_{\delta DL_2^v}$$

$$+ \frac{EA_3}{L} (-v_B) (-\delta v_B)$$

$$= \frac{3E}{25L} \left[4(A_2 - A_1) u_B + 3(A_1 + A_2) v_B \right] \delta v_B$$

$$+ \frac{EA_3}{L_3} v_B \delta v_B$$

$$\delta W_c^v = -\delta W_{ic}^v$$

$$-P \delta v_B = \frac{3E}{25L} \left[4(A_2 - A_1) u_B + 3(A_1 + A_2) v_B \right] + \frac{EA_3}{L_3} v_B$$

Combine u & v:

Two equations in two unknowns

$$4(A_1 + A_2) u_B + 3(A_2 - A_1) v_B = 0$$

$$4(A_2 - A_1) u_B + 3(A_1 + A_2) v_B + \frac{25L}{3L_3} A_3 v_B = -\frac{25PL}{3E}$$

Solve
for
 u_B, v_B

L12
6

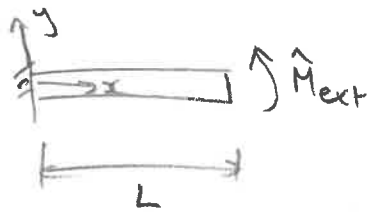
Note: The notes has treatment of virtual work.
by considering virtual force
— optional reading.

L12

7

Example

Beam

 E, I constant

$$\delta W_{ie}^{\text{beam}} = \int_L EI v'' \delta v'' dx$$

Solve for $v(x)$

$$\rightarrow \boxed{\delta W_e = \delta W_{ie}} \quad (1)$$

AsideBeam theory: $EI v'' = M$

$$v(x) = ax^2 + bx + c$$

$$BC: v(x=0) = 0 \rightarrow c = 0$$

$$v'(x=0) = 0 \rightarrow b = 0$$

(Need to find "a")

$$v(x) = ax^2$$

$$\delta v(x) = x^2 \delta a$$

$$v'(x) = 2ax$$

$$\delta v'(x) = 2x \delta a$$

$$v''(x) = 2a$$

$$\delta v''(x) = 2 \delta a$$

$$\delta W_e = \hat{M}_{ext} \delta v \Big|_{x=L} = \hat{M}_{ext} (2x \delta a) \Big|_{x=L} = \hat{M}_{ext} 2 \delta a L$$

$$\delta W_{ie} = EI 2a 2 \delta a L$$

Substitute into (1):

$$\hat{M}_{ext} 2 \cancel{\delta a} = EI 4a \cancel{\delta a} \Rightarrow a = \frac{\hat{M}_{ext}}{2EI}$$

$$\boxed{v(x) = \frac{\hat{M}_{ext}}{2EI} x^2}$$

Clicker Question 1

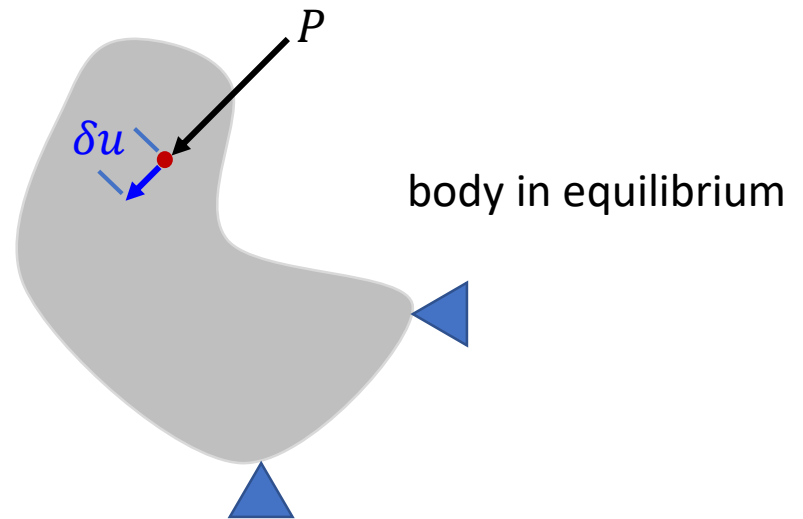
What is the correct expression of virtual work δW_e for the case below?

(a) $P\delta u$

(b) $u\delta P$

(c) $\frac{1}{2}P\delta u$

(d) $\frac{1}{2}u\delta P$



Clicker Question 2

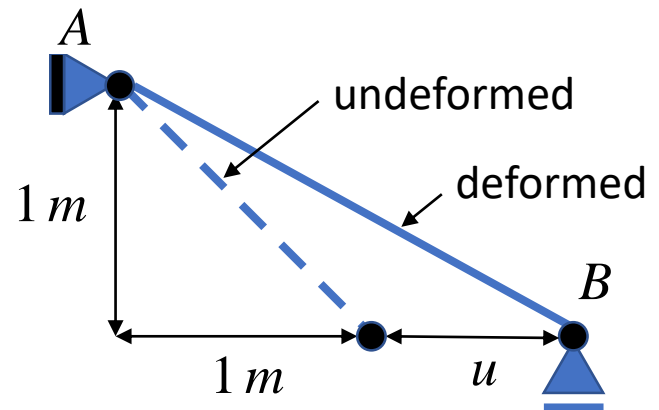
What is the elongation of the bar shown below if a horizontal displacement u is applied to joint B?

(a) u

(b) $\sqrt{2} u$

(c) $\frac{\sqrt{2}}{2} u$

(d) none of the above



Clicker Question 2

What is the elongation of the bar shown below if a horizontal displacement is applied to joint B?

(a) u

(b) $\sqrt{2} u$

(c) $\frac{\sqrt{2}}{2} u$

(d) none of the above

