

Linear Control Design II

ASSIGNMENT

Based on your state space model for the electro-mechanical system (A, B, C e D), build a simulink model with the parameters given during the class and answer the questions:

- 1) How the structure behaves when s_0 (air gap) is varied from 5 mm to 1 mm. Simulate three different values of $s_0=5$ mm , $s_0=3$ mm ; $s_0= 1$ mm.
- 2) What is the maximum displacement experienced by the masses when a perturbation equivalent to an initial condition of velocity of 5 mm/s acts on mass 3?
- 3) Sending a step function $u(t)$ to the electromagnets how the masses will vibrate, i.e. amplitudes and frequencies.

Assignment – Mechanical & Mathematical Models

**Physical
System**

**Electro-Mechanical
Model**

**Mathematical
Model**

**Assumptions
(simplifications)**

**Newton, Euler, D'Alembert,
Lagrange, Hamilton, Jourdain ...**

**Coulomb, Ampère, Ohm,
Kirchhoff, Faraday, Lorentz,
Maxwell ...**

(principles & axioms)



State Space Matrix **A**

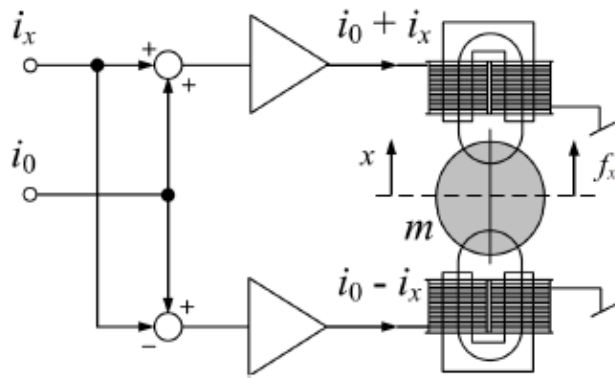
$$\mathbf{A} = \begin{bmatrix} -R/L & 0 & -K_i/L & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 & 0 \\ K_i/m_1 & -(k_1+k_2+K_s)/m_1 & 0 & k_2/m_1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & k_2/m_2 & 0 & -(k_2+k_3)/m_2 & 0 & k_3/m_2 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 1 \\ 0 & 0 & 0 & k_3/m_3 & 0 & -k_3/m_3 & 0 \end{bmatrix}$$

$$\begin{aligned} \dot{\mathbf{x}}(t) &= \mathbf{A}\mathbf{x}(t) + \mathbf{B}\mathbf{u}(t) \\ \mathbf{y}(t) &= \mathbf{C}\mathbf{x}(t) + \mathbf{D}\mathbf{u}(t) \end{aligned}$$

(notes from the book)

"Magnetic Bearings – Theory, Design, and Application to Rotating Machinery"

by
Gerhard Schwitzer and Eric H. Maslen



Electromagnets are normally coupled in a counteracting pair such that positive and negative forces can be obtained. By applying a bias current i_0 , which typically is fixed, the forces acting on the steel object in one direction can be controlled by a perturbation current i_x , as shown in Fig. 1.2. The resulting force is then given as

$$f = \frac{1}{4} \mu_0 n^2 A_a \left(\frac{(i_0 + i_x)^2}{(s_0 - x)^2} - \frac{(i_0 - i_x)^2}{(s_0 + x)^2} \right) \quad (1.13)$$

The function of force can be linearized as follows

$$f(x, y) = f(a, b) + \left. \frac{\partial f(x, y)}{\partial x} \right|_{a, b} (x - a) + \left. \frac{\partial f(x, y)}{\partial y} \right|_{a, b} (y - b) \quad (1.14)$$

where $f(x, y)$ is the general function depending on x and y variables, which is the control current i_x and the displacement of the air gap x . $f(a, b)$ is the steady state force evaluated at the operating point. Assuming the displacement x and the steady state current i_x is zero, this steady state force yields zero. Thus the linearized force can be expressed as

$$F = K_i i_x + K_s x \quad (1.15)$$

The coefficient K_i N/A which is used to describe the force as function of the control current i_x is given as

$$K_i = \frac{\mu_0 N^2 A_a i_0}{x_0^2} \quad (1.16)$$

The coefficient K_s N/m which is used to describe the force as function of the displacement x is given as

$$K_s = \frac{\mu_0 N^2 A_a i_0^2}{x_0^3} \quad (1.17)$$

